

STRENGTH OF SHIPS - ON THE NEUTRAL
AXIS LOCATION

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by

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ABSTRACT

This work is an attempt at determining the optimum neutral axis location in the midship section of a vessel, using as a model for the calculations a box girder, subject to several different conditions of sagging and hogging bending moments; several values of design stress are also considered and in the end what happens when lateral hydrostatic load is considered along with different values assigned to the design stress in tension and in compression is also analysed.

Thesis Supervisor: J. Harvey Evans

Title: Professor of Naval Architecture

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LIST OF SYMBOLS

a	- distance between transverse stiffening members
AL	- length of box girder or vessel
B	- beam
BOT	- thickness of the plating on the bottom
BM	- bending moment
D	- depth
DIBOT	- distance of the bottom from the neutral axis
DITOP	- distance of the top from the neutral axis
DSC	- design stress in compression
DST	- design stress in tension; also design stress without discrimination when DSC=DST
H	- head of sea water (lateral load)
HB	- head of sea water on the bottom
HM	- hogging bending moment
HT	- head of sea water on the top
I	- moment of inertia of the cross section with respect to the neutral axis (GMI in the program)
LBS	- longitudinal bending stress
S	- spacing of longitudinal stiffening members
SBCRIT	- critical stress on bottom
SBH	- stress on bottom due to hogging
SBS	- stress on bottom due to sagging
SM	- sagging bending moment
STCRIT	- critical stress on top
STH	- stress on top due to hogging
STR	- stress

STR1 - primary stress = LBS
STR2 - secondary stress
STR3 - tertiary stress
STR3B - tertiary stress on bottom
STR3T - tertiary stress on top
STRCR - critical stress
STS - stress on top due to sagging
t - thickness of the plate
T - draft
TOP - thickness of the plating on the top
TS - thickness of the side plating
TSTH - total stress on top in hogging
TSTS - total stress on top in sagging
TSBH - total stress on bottom in hogging
TSBS - total stress on bottom in sagging

INTRODUCTION

It is frequently found in the literature related with the strenght of ships such statements as:

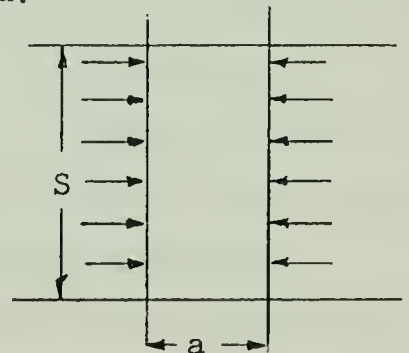
"In general the neutral axis is expected to be at 40% of the depth (D) above the base line ... a preliminary estimate of the thickness required for the bottom plating (BOT) can be obtained from

$$1.875 \left(\frac{0.4D}{0.6D} \right) 2240 \sqrt[3]{AL} = \frac{40300}{1 + \frac{1}{950} \left(\frac{S}{BOT} \right) 1.75}$$

where the Montgomerie's expression is being considered to determine the critical stress (SBCRIT psi) and the short edges are assumed far apart and simply supported.

$$SBCRIT = \frac{40300}{1 + \frac{1}{950} \left(\frac{S}{BOT} \right) 1.75}$$

$$S \geq 4a$$



a- distance between transverse stiffening members (inches)

1.875 accounts for a factor of safety

Tobin's expression was considered in the above formula to give the maximum permissible bending stress (LBS, psi)

$$LBS = 2240 \sqrt[3]{AL}$$

We see thus that this procedure allows for an estimate of the bottom thickness to be obtained after knowing the length and assuming a longitudinal frame spacing. What may be questioned here is the reason why one assumes the neutral axis $0.4D$ above the base line and also the reasons behind the choice of a frame spacing.

In the following a procedure to estimate the scantlings of longitudinal members is described; this procedure can also be found frequently in the literature dealing with the subject.

..."For a particular set of principal dimensions , general arrangement and frame spacing, plate panel thicknesses for one after another of the amidships longitudinally continuous elements are selected for an assumed position of the neutral axis at 40% of the depth above the keel. The composite midship section assemblage is then tested for the adequacy of its section modulus when ranged alongside some standard value, such as given by the Load Line Regulations or some function of it. Thus the modulus is judged sufficient to limit ship bending stresses to the value previously set and incorporated into the design equations.

For all but the deepest ships the modulus will undoubtedly need to be increased and a small arbitrary increment is added to the upper deck or bottom whichever happens still to be farther from the true neutral axis location found after the first design cycle. The shift of neutral axis resulting from the scantling increment requires a compensating increment in the opposite flange-

ge with the obvious outcome that the neutral axis is shifted in reverse with attendant effect on the scantling chosen first; a cyclic process is then begun. More trials will be necessary before the match of available to required section modulus is achieved."

From this transcript, we may again note the initial statement "...for a particular set of principal dimensions, general arrangement and frame spacing..." and question whether or not the frame spacing should be chosen as an input or rather be left as a variable and be obtained as a result of a structural design procedure. It is here acknowledged though that in these procedures an acceptable solution can be worked out and each new cycle is an improvement over the precedent ones. What may be questionable though is whether the first cycle, due to the assumptions made at its beginning, has or not a strong influence in the outcome leading to a solution among the various possible ones which in the end does not lead to "the optimum design" whatever its definition may be.

It seems that this trial procedure can be looked at as a "weakness" of the structural design procedure for the determination of scantlings and it is perhaps reasonable to make some effort in order to work out a process that could give the scantlings in a direct manner and be at the same time in agreement with what can be considered "the optimum design", (usually based on economic considerations).

Nowadays much concern is given to economics in ship design and naturally the strength calculations of a ship are also made with economical optimization in mind. With this respect, we can realize that in ships structures there is a desirability to minimize structural weight and some define structural efficiency as being the least weight structure capable of resisting the loads for which they are designed.

It should be clear that the cost of construction and of maintenance and repair are inter related factors together with least weight, to be considered when bringing economics into ship structural design. Also other matters such as accessibility, weldability and availability of material must be considered as well.

In the following sections an attempt is made to obtain without any interactions the scantlings for the plating of a box girder. It is hoped that through this simple model some helpful conclusions can be made in view of their application to a vessel's structure. Each of the different situations analyzed will be exhaustively explained in the following sections.

PRELIMINARY CALCULATIONS

The model used in this section and also throughout this work with only very small additional considerations is a box girder which cross section is pictured in figure 1.

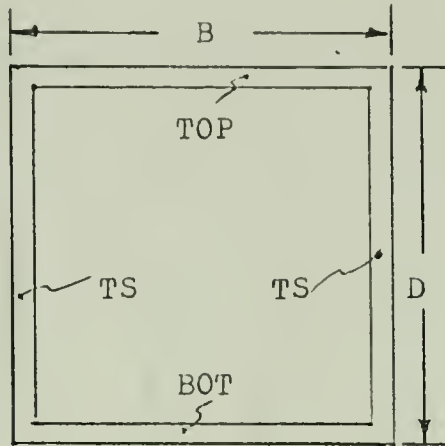


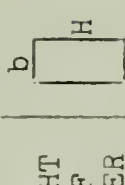
Fig. 1

As will be seen in the calculations of the following sections, the dimensional parameters used are of the order of magnitude of those considered in ship design.

This model was chosen due to its simplicity for the performance of the following calculations as can be understood immediately from its geometrically symmetrical cross section.

The objective of this section is to determine the necessary formulation to be used in future sections.

The formulation dealt with in this section is related only to the particular model chosen and the geometry of its cross section.

MEMBER	SCANTLINGS inches (in)	AREA A ² (in)	DISTANCE TO BOTTOM (ft)	A*d ² (in*ft)	A*d*d ² (in*ft) ²	WIDTH of member b(in)	HEIGHT OF MEMBER H(ft)	 I = b * H ³ (in*ft) ²
TOP	B*TOP	B*TOP	D/12	B*TOP*D/12	B*TOP*(D/12) ²	B	-	-
SIDE	2*TS*D	2*TS*D	D/(2*12)	TS*D*D/12	TS*D*(D/12)/12	2*TS	D/12	2*TS*(D/12) ³
BOTTOM	B*BOT	B*BOT	-	-	-	b	-	-

Σ I =

Σ(A*d*d)
+ Σ I =

Σ(A*d) =

Σ A =

I_{BOT} =

- Σ A * (DIBOT)² =

$$DIBOT = \frac{\sum(A*d)}{\sum A} * 12$$

$$DIBOT' = \frac{\sum(A*d)}{\sum A}$$

$$\sum A = B*TOP + 2*TS*D + B*BOT = 2*D*TS + B*(TOP+BOT)$$

DIBOT = Neutral axis location above bottom (in)

DIBOT' = " " " " " (ft)

I_{BOT} = Moment of inertia with respect to the bottom

I = " " " " " " " neutral axis

$$\sum(A*d) = (D/12) * (B*TOP + D*TS)$$

$$I = \sum(A*d*d) + \sum I - (\sum(A*d))^2 / (\sum A)$$

$$DIBOT = D * (B*TOP + D*TS) / (2*D*TS + B*(TOP+BOT))$$

This formula will give DIBOT in inches if D and B are expressed in inches; it will give DIBOT in feet in the event that D and B are expressed in feet.

$$DITOP = D - DIBOT$$

$$DITOP = D * (B*BOT + D*TS) / (2*D*TS + B*(TOP+BOT))$$

DITOP in units of inches or feet as explained for DIBOT

The following formula will give I in units of (in*ft)² for B and D given in inches.

$$I = \frac{(D/12)^2 * (3 * B * B * TOP * BOT + (D * TS)^2 + 2 * B * D * TS * (TOP + BOT))}{3 * (2 * D * TS + B * (TOP + BOT))}$$

Next formula gives I in units of (in*ft)² for B and D given in feet.

$$I = \frac{4 * D * D * (3 * B^2 * TOP * BOT + (D * TS)^2 + 2 * B * D * TS * (TOP + BOT))}{2 * D * TS + B * (TOP + BOT)}$$

These expressions will be used in most of the following sections with B and D expressed in feet.

NOTE: TS which is defined as the thickness of the side plating is considered throughout this work to be uniform and will always be equal to the larger of the thicknesses obtained for the top and for the bottom plating.

FORMULAS USED IN THE ANALYSIS

BEAM EQUATION

Assuming pure bending, the stresses in the cross section of the box girder are given by

$$\text{STR} = \frac{\text{BM} \cdot y}{I}$$

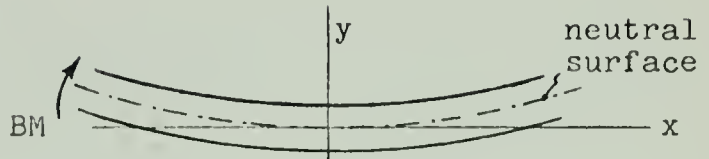


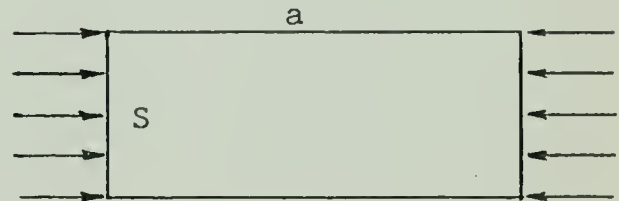
Fig. 2

where y is the distance from the neutral axis to the point where the stress is being determined.

The neutral surface indicated in fig. 2 is composed of all the points which are not subjected to any stress in pure bending.

BRYAN EQUATION - Plate under inplane load

$$\text{STRCR} = K \cdot \frac{(\pi)^2 \cdot E \cdot (t)^2}{12 \cdot (1 - (\text{POI})^2) \cdot (S)^2}$$



When $a/S > 2$ then $K=4$

STRCR given in psi for t and S expressed in inches and E in psi.

$$\pi = 3.1416$$

PLATE UNDER LATERAL LOAD

$$\text{STR3} = 0.5 * K * \rho * H * \left(\frac{S}{t}\right)^2 * \frac{1}{144}$$

STR3 - tertiary stress

ρ - density of sea water (64 lb/ft³)

H - head of sea water (ft)

a - long dimension of plate (in)

S - short dimension of plate (in) = distance between longitudinal stiffeners

t - plate thickness (in)

The stresses are more severe at points A and B but we are particularly interested in the stresses at point B because there they will combine with the stresses resulting from bending also called primary stresses.

Note that for a panel under lateral load, the direction of the stresses is, at the edges, perpendicular to them, so at B they are directed "longitudinally".

For $a/S > 2$ we have $K = 0.685$

Replacing all the numbers in the above formula, we obtain

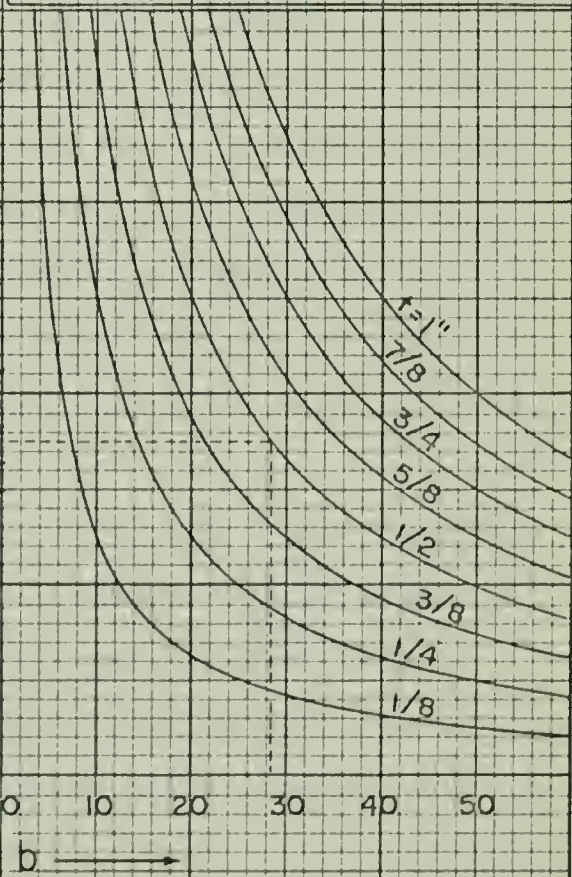
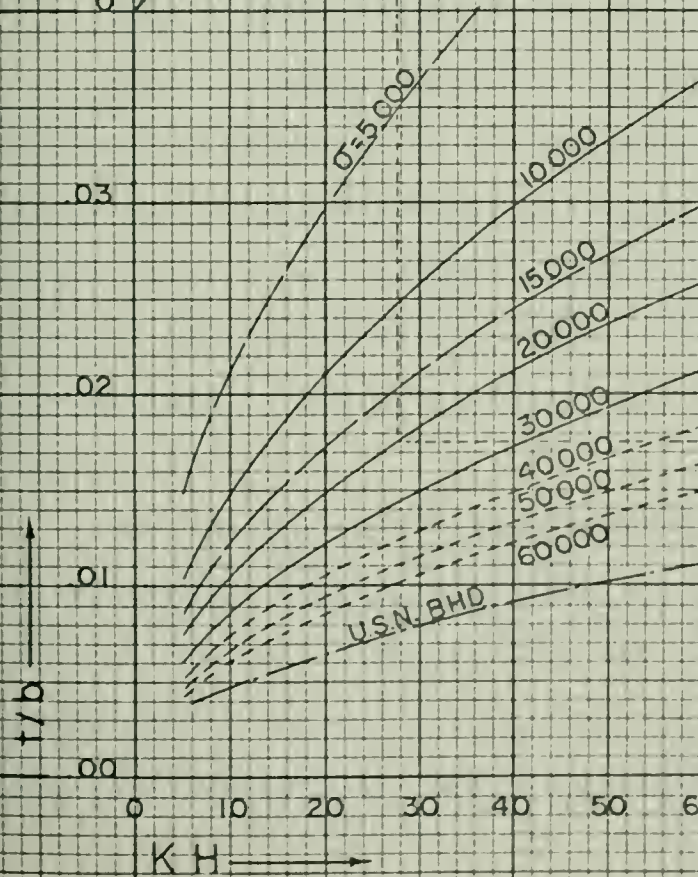
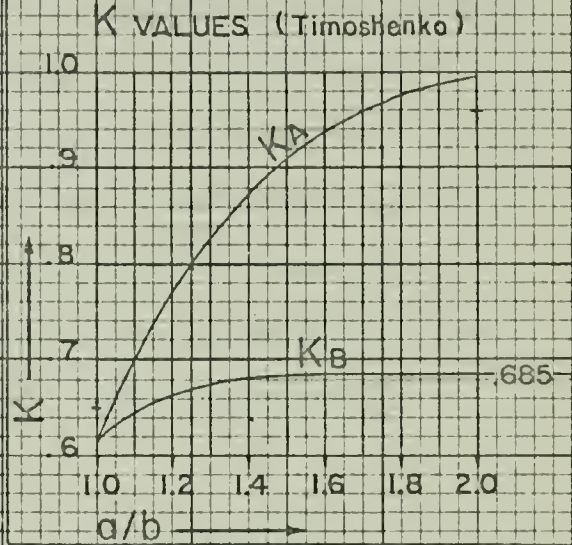
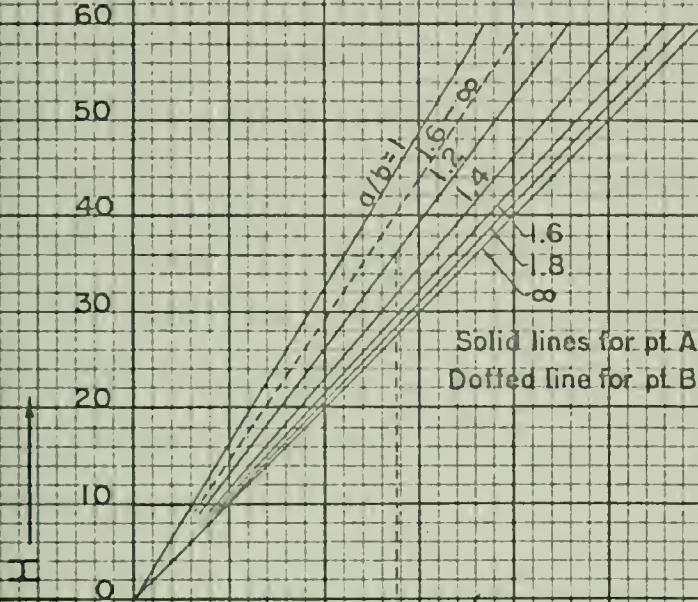
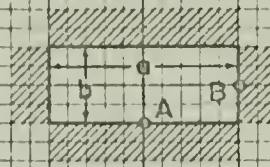
$$\text{STR3} = 0.152222 * H * \left(\frac{S}{t}\right)^2$$

BENDING STRESSES IN FLAT PLATES

WITH CLAMPED EDGES UNDER UNIFORM HYDROSTATIC LOAD

- σ = Bending stress (lbs/sq in)
- ρ = Density of sea water (64 lbs/cu ft)
- H = Head of sea water (feet)
- a = Long dimension of plate panel (inches)
- b = Short dimension of plate panel (inches)
- t = Plate thickness (inches)

Formula:
$$\sigma = \frac{1}{2} K \rho H \left(\frac{b}{t}\right)^2 \frac{1}{144}$$



The following stress definitions and figures 3 and 4 will help understanding the reasons behind the calculations made in many of the following sections.

STRESS DEFINITIONS

STR1 - SHIP BENDING STRESS or PRIMARY STRESS

Since this stress is directly proportional to distance from the ship's neutral axis, it is substantially constant at any point on the cross section of the bottom or top plating when the vessel is in an upright attitude. For distinction, this stress may be termed a primary stress or an area stress in this location and under such a condition. While the longitudinal direction is implied in the foregoing comments, they are equally applicable transversely.

STR2 - GIRDER BENDING STRESS or SECONDARY STRESS

Girder bending stress arises from the reaction of a plating stiffener combination to a loading normal to the plating such as may be due to water pressure. Following the usual concept, the plating acts as one flange of the composite girder and so is stressed. Due to shear lag, this secondary stress in the plating is a maximum in way of the stiffener and diminishes with distance from it.

For the previous case of bottom plating and vessel upright, the transverse variation of fore and aft stress will be sensibly constant for some distance from the stiffener but will vary significantly through the plating thickness as the plate thickness is a substantial part of the distance to the neutral axis

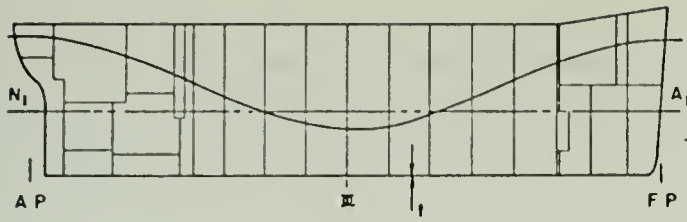
of the plate stiffener combination. For convenience such a stress may be termed a line stress although for a wide spacing of stiffeners with consequently greater deviation from uniform distribution some inaccuracy of concept is introduced by using this term.

For girders continuous over several supports, such as transverse bulkheads, maximum bending moments occur at the supports and will readily be in the order of twice the field moment.

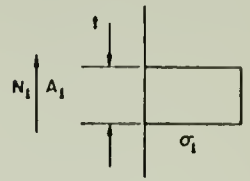
STR3 - PLATE BENDING STRESS - TERTIARY STRESS

Proceeding in turn to ever more elemental portions of the ship's structure concludes with the simple plate panel supported on its four edges. In such a plate under lateral or normal loads, bending stresses are created which are variable from point to point on the panel and through its thickness with maxima occurring at one or the other surface. Since any such stress is effective at a point only, it may be called a point stress or tertiary stress.

Each elemental panel of ship's plating is generally one of a repeating pattern. Quite apart from any rotational restraint provided by the panel supports consisting of floors, frames, longitudinals, etc., there is the restraint afforded by the continuity of the plating and its loading. For practical purposes then, a condition of edge clamping exists with maximum stresses being found at the midlength of the longer panel edges and secondary maxima at the midlength of the shorter edges. In both cases these stresses are normal to their respective edges.

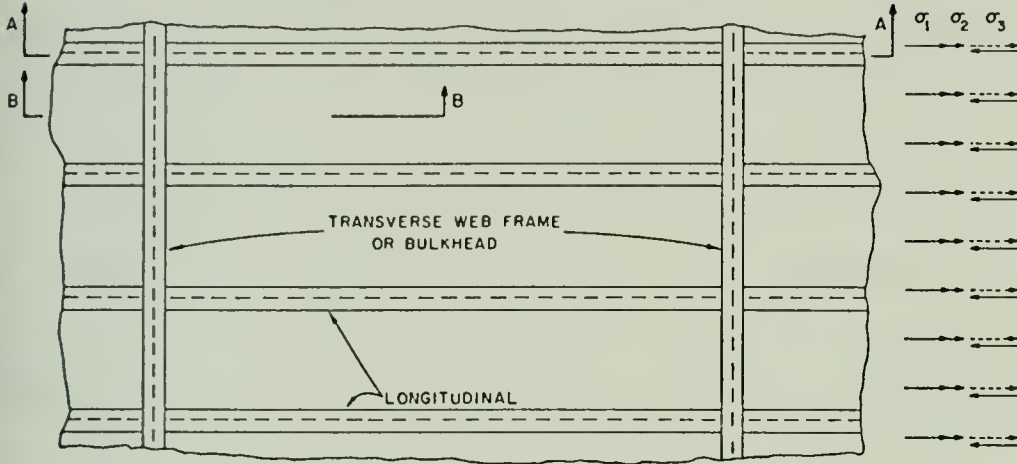


STRESS DISTRIBUTION IN BOTTOM SHELL AT TRANSVERSE WEB FRAME

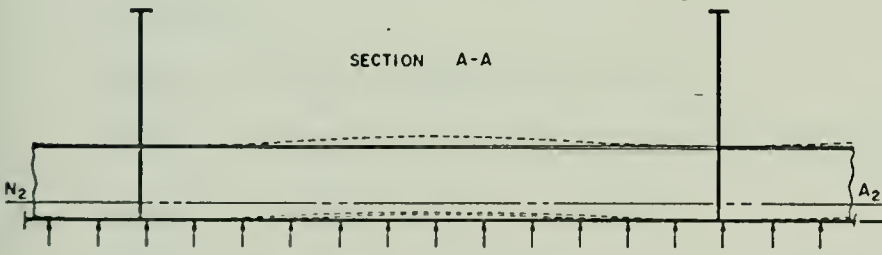


HULL BENDING STRESS

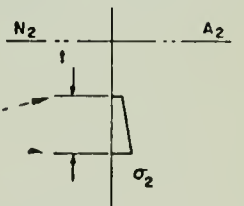
TYPICAL BOTTOM STRUCTURE OF A LONGITUDINALLY FRAMED SHIP



SECTION A-A



STIFFENER BENDING STRESS



SECTION B-B

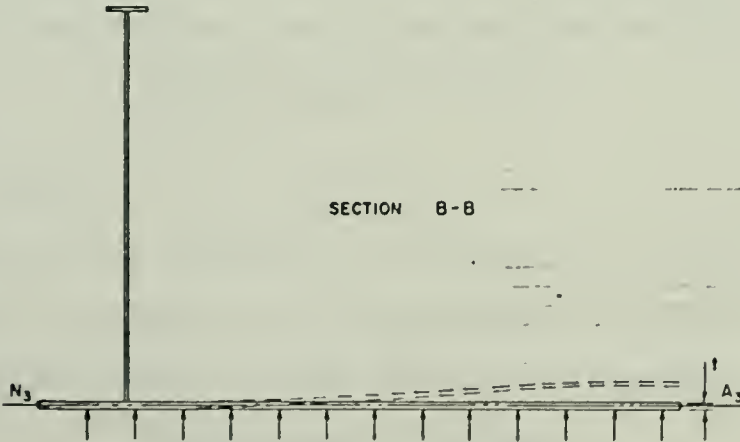


PLATE BENDING STRESS

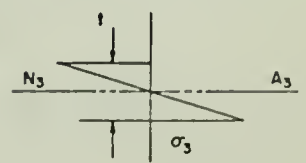


FIGURE 3

From the previous definitions and figure 3 we may summarize in figure 4 the effect of STR1, STR2 and STR3.

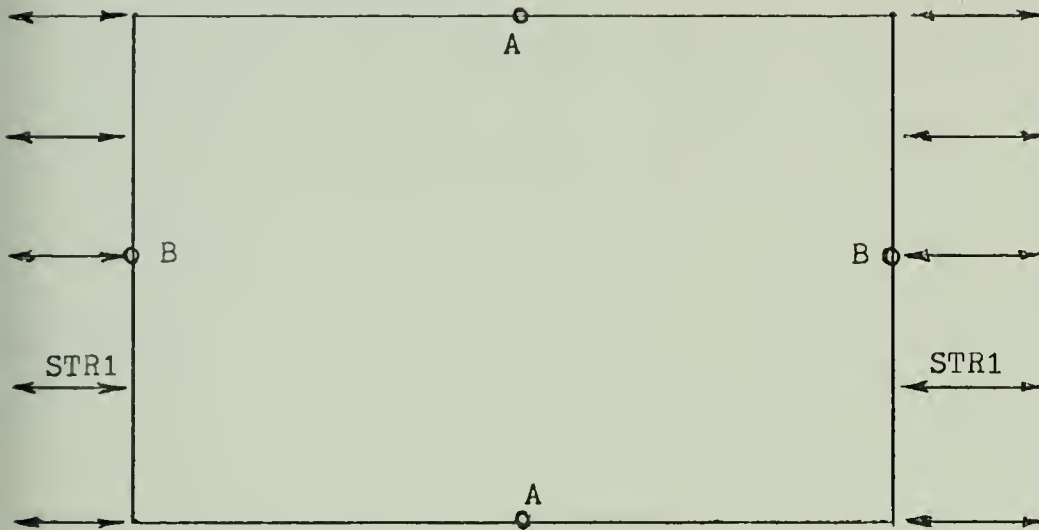


Fig. 4

At point A

STR2 is larger than at point B

STR2 is acting in the longitudinal direction

STR3 is larger than at point B

STR3 is acting in the transverse direction

STR1 and STR2 are here combined acting both in the longitudinal direction

At point B

STR2 may be acting in the transverse direction

STR3 is acting in the longitudinal direction

STR1 and STR3 are here combined acting both in the longitudinal direction.

Usually STR2 is attributed a constant value and can be taken into account by automatically decreasing the design stress by the value assigned to STR2.

THE BENDING MOMENT EQUATION

The bending moment equation used is derived from

$$BM = \Delta * AL / 35 \quad \text{ton*ft}$$

knowing that

$$\Delta = \nabla / 35 \quad \text{ton}$$

and that

$$\nabla = B * T * AL * C_b$$

and using

$$C_b = 0.75$$

we obtain

$$BM = \frac{0.75}{35 * 35} * B * T * AL * AL \quad \text{ton*ft}$$

where

AL = is the length in ft

B = is the beam in ft

T = is the draft in ft

C_b = is the block coefficient

or multiplying by 2240 we obtain

$$BM = 1.37143 * B * T * AL * AL \quad \text{lb*ft}$$

Note:

∇ is the displacement volume and Δ is the corresponding displacement weight.

MOMENT ACTING IN ONE DIRECTION

NO RESTRICTION IMPOSED ON THE COMPRESSION STRESS

Considering the box girder subjected to a bending moment acting in hogging BM, we will observe as a consequence that the top will be subjected to tension and the bottom to compression. These stresses are given by the already introduced formula

$$STR = \frac{BM * Y}{I}$$

or in this case, being interested on the top and bottom,

$$STH = \frac{BM * DI * TOP}{I}$$

$$SBH = \frac{BM * DI * BOT}{I}$$

In this introductory study it was decided not to limit the value of the stress in compression. Only the value of the stress in tension, here STH will be requested to be equal to the value chosen for the design stress in tension (or smaller).

SBH will be a consequence of the value of DST and also of the S value chosen.

So, we will basically have 3 equations

$$DST = STH \leftrightarrow TOP \quad \text{Bending equation}$$

$$S \leftrightarrow SBH \leftrightarrow BOT \quad \text{Bending equation + Bryan formula}$$

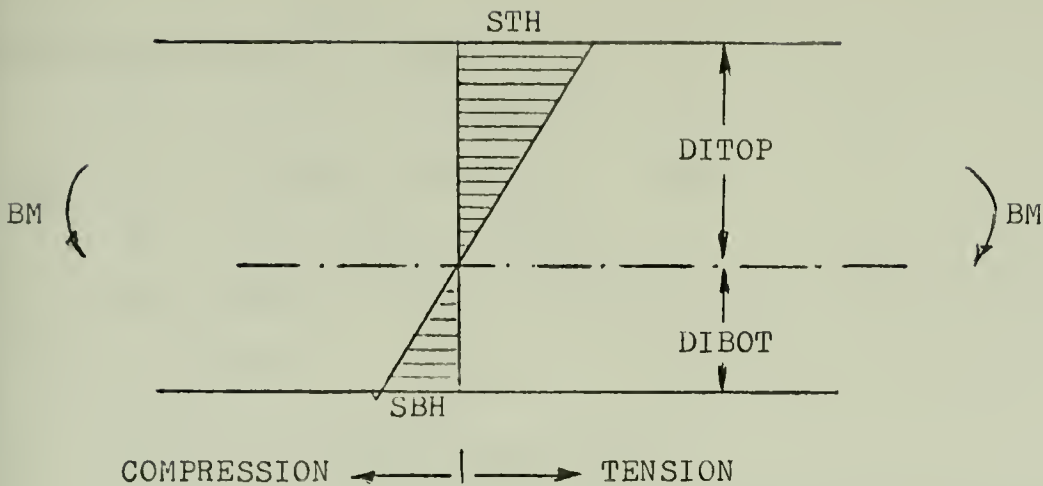


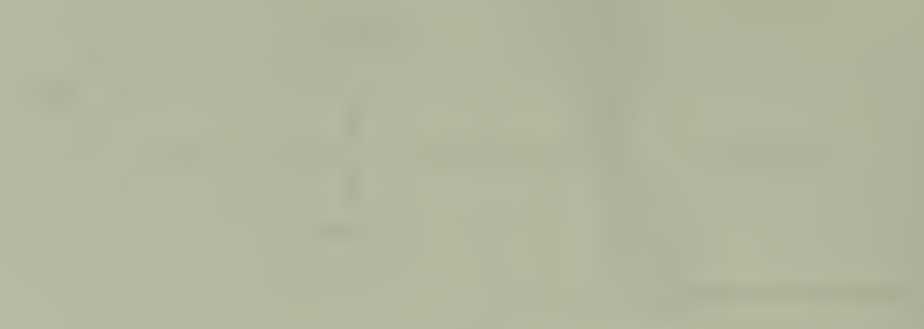
Fig. 5

The three equations that can be obtained will allow one to determine the three unknowns which are BOT, SBH, TOP.

By replacing in the equations already mentioned the expressions derived previously for DIBOT, DITOP and I and solving for BOT, TOP and SBH we obtain after a lengthy but simple process the equations in their final format as we will see.

Note here that in order to solve the three equations, the thickness of the side plates TS was made equal to the larger TOP or BOT; since TOP and BOT were not known and had to be determined after deciding on TS, the equations had to be solved twice. First the equations are solved assuming that $TOP > BOT$ this simply implying that the neutral axis is going to be chosen already closer to the top and $TS = TOP$. Second, the equations are solved assuming $BOT > TOP$ thus making $TS = BOT$.

As will be seen, only one solution is possible, the other being rejected by giving a result that does not agree with the



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This section of text is also illegible due to blurriness. It seems to contain several lines of text, possibly a continuation of the list or description from the previous section. The content is not discernible.

The final section of text is illegible. It appears to be a concluding paragraph or a list of items. The text is too faint to read and does not provide any meaningful information.

assumption just mentioned.

TOP , BOT , S (inches)

B , D , DITOP , DIBOT , AL (ft)

DST , STH , SBH , E (psi)

BM (ton*ft)

I (in*ft)²

$$DST \gg STH = \frac{560 * BM * (B * BOT + D * TS)}{D * (3 * B * B * TOP * BOT + 2 * B * D * TS * (TOP + BOT) + (D * TS)^2)}$$

$$SBH = \frac{560 * BM * (B * TOP + D * TS)}{\text{Same denominator}}$$

$$BOT = \sqrt{G} \sqrt{SBH}$$

$$G = 0.30396 * \frac{(S)^2 * (1 - (POI)^2)}{E}$$

$$\frac{DST}{SBH} = \frac{B * BOT + D * TS}{B * TOP + D * TS}$$

Making

$$A = \frac{9 * B * B * D}{140 * BM}$$

$$C = \frac{3 * B * D * D}{70 * BM}$$

$$F = \frac{3 * D * D * D}{140 * BM}$$

we will obtain, by assuming first TOP > BOT

$$DIBOT = \frac{D*(B+D)*TOP}{(2*D+B)*TOP+B*BOT}$$

$$DITOP = \frac{D*(B*BOT+D*TOP)}{\text{Same denominator}}$$

$$I = 4*D*D* \frac{(2*B*D+D*D)*TOP*TOP+(2*B*D+3*B*B)*TOP*BOT}{\text{Same denominator}}$$

$$TOP = \frac{B*(BOT)^3}{(B+D)*G*DST-D*(BOT)^2}$$

$$BOT = \frac{TOP*(12*D-(C+F)*DST*TOP)}{DST*(A+C)*TOP-12*B}$$

Before going to the equations obtained when $BOT > TOP$ is assumed, we call the attention of the reader to the fact that the values of BOT and TOP have to be obtained from the two above equations. Substitution of one into the other would result in an equation of degree higher than at least 5th degree, unsuitable for direct solution. As will be later exemplified, the solution is here obtained by assigning values to either one of BOT and TOP and through the two equations, we obtain the respective values for TOP and BOT . The plotting of these values BOT vs TOP will give one line for each of the two equations. The intersection of the two lines thus obtained allows the solution to be read from the plot.

This solution will be acceptable if the intersection lies in the sector $TOP > BOT$ otherwise the intersection must be rejected.

Assuming now $BOT > TOP$ we make $TS = BOT$ and obtain:

$$DIBOT = \frac{D*(B*TOP+D*BOT)}{(2*D+B)*BOT+B*TOP}$$

$$DITOP = \frac{D*(B+D)*BOT}{\text{Same denominator}}$$

$$I = 4*D*D* \frac{(2*B*D+D*D)*(BOT)^2+(2*B*D+3*B*B)*TOP*BOT}{\text{Same denominator}}$$

$$TOP = \frac{(B+D)*(BOT)^3 - D*G*DST*BOT}{B*G*DST}$$

$$BOT = \frac{12*(B+D) - (A+C)*DST*TOP}{(C+F)*DST}$$

In this form the equations can be used in a very simple computer program that will give the set of values TOP vs BOT suitable for plotting. At this stage we are hoping that if the intersection obtained in the situation where we assumed $TOP > BOT$ is acceptable, the two lines resulting from the case $BOT > TOP$ will not intersect or if they do that the intersection must be rejected due to the fact that they will violate the assumption from which they were derived

As can be observed from the several cases run in the computer, the case where two acceptable solutions exist never happened and it can only happen if $TOP = BOT$ where then both solutions are acceptable and coincide as in the case of one of the following runs.

In the following pages the computer printouts are included. The first run is explained in detail to make the interested reader familiar with the problems encountered in determining the intersections and the respective thicknesses.

The other runs are here presented as well to serve as proof that a simple solution can always be obtained and also to give an idea of the range of thicknesses expected in each case.

In all runs only a bending moment acting in hogging was considered.

The program used is a direct application of the equations here derived to a particular set of values chosen as the input. From the results obtained, some conclusions were drawn that allowed for the improvements made in following sections.

No further explanation on the program shall here be given because it is self explanatory in the comment statements included.

STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 1

```

2024 C *AL* IS THE LENGHT
2024 AL=507.
2024 R=AL/8.
2018 C *DRA* IS THE DRAFT
2018 D=8/3.
2024 D=AL/9.
2030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
2030 FOR STEEL
2030 POI=0.3
2038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
2038 FOR STEEL
2038 E=37.*10.**6
2040 C *DST* IS THE DESIGN STRESS
2040 DST=24072.
2054 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
2054 S=12.
2060 C *BM* IS THE BENDING MOMENT
2060 BM=(B*DRA*AL*AL*2.75)/(25.*35.)
2068 C G=7.37396*S*S*(1.-POI*POI)/E
2068 A=2.*R*S*D/(147.*BM)
2068 C G=3.*S*D*D/(72.*BM)
2068 F=2.*D*D*D/(14.*BM)
2080 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
2080 WRITE(5,17)
2082 17 FORMAT('1',20X,'TOP ASSUMED LARGER THAN BOTTOM')
2084 C *TOP* IS THE THICKNESS OF THE TOP PLATING
2084 *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
2084 BOT1=2.1
2076 11 TOP1=(R*BOT1*BOT1*BOT1)/((B+D)*G*DST-D*BOT1*BOT1)
2084 WRITE(5,20)TOP1,BOT1
2084 20 FORMAT('1',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
2084 BOT1=BOT1*2.225
2084 IF(BOT1.GT.2.2)GO TO 30
2084 GO TO 11
2084 30 TOP2=2.1
2084 31 BOT2=TOP2*(12.*D*(C+F)*DST*TOP2)/(DST*(A+C)*TOP2-12.*B)
2084 WRITE(5,40)TOP2,BOT2
2084 40 FORMAT('1',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
2084 TOP2=TOP2*2.25
2084 IF(TOP2.GT.1.)GO TO 50
2084 GO TO 31
2084 50 CONTINUE
2084 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
2084 WRITE(5,60)
2084 60 FORMAT('1',20X,'BOTTOM ASSUMED LARGER THAN TOP')
2084 BOT3=2.1
2084 61 TOP3=((B+D)*BOT3*BOT3*BOT3-D*G*DST*BOT3)/(B*G*DST)
2084 WRITE(5,70)TOP3,BOT3
2084 70 FORMAT('1',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
2084 BOT3=BOT3*2.25
2084 IF(BOT3.GT.1.)GO TO 80
2084 GO TO 61

```


STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 2

```

0402      80 TOP4=0.1
040A      81 BOT4=(12.*(R+D)-(A+C)*DST*TOP4)/((C+F)*DST)
044E      WRITE(5,90)TOP4,BOT4
0472      90 FORMAT(' ',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
049A      TOP4=TOP4+.05
04A6      IF(TOP4.GT.1.)GO TO 100
04B8      GO TO 81
04BC      100 CONTINUE
04BC      END
0400(S)  .U      0400(V)  AL      0408(V)  B      04D0(V)  DRA      04D8(V)  D
0404(V)  POI      04E8(V)  E      0400(S)  .R      04F8(V)  DST      0500(V)  S
0404(V)  RM      0518(V)  G      0524(V)  A      0520(V)  C      0534(V)  F
0404(L)  12      0200(S)  01      0538(V)  BOT1      0176(L)  11      0540(V)  TOP1
0402(L)  22      0220(L)  30      0554(V)  TOP2      0234(L)  31      0552(V)  BOT2
0404(L)  40      02FF(L)  50      0312(L)  67      0560(V)  BOT3      0348(L)  61
0404(V)  TOP3      0388(L)  70      0402(L)  80      0574(V)  TOP4      040A(L)  81
0404(V)  BOT4      0472(L)  90      04BC(L)  100      0400(S)  .V
0400(S)  .XEQ      L
0400(S)

```

PROGRAM LABELS:

2070 *MAIN*	2070 .V	2A30 .COMP	2C1A 0I	25EC .R	2B04 .ZE
2A62 \$6	2620 .A	2B60 .MES	29DE .W	27E6 EXP	296C AI
26DE ALOG	2AA4 .RARG	2B02 .5	27E6 AEXP	2B08 .ERCNT	2AD6 \$8
2B08 .0	2BDE .U	3036			

ENTRY-POINTS:

25EC .R	2620 .A	26DE ALOG	27E6 AEXP	27E6 EXP	296C AI
29DE .W	2A30 .COMP	2A62 \$6	2AA4 .RARG	2AD6 \$8	2B00 .5
2B04 .ZERO	2B08 .ERCNT	2B2A .0	2B60 .MES	2BFA .U	2C10 .V
2C1A 0I					

COMMON-BLOCKS:

NONE

UNDEFINED SUBROUTINES:

NONE

TRANSFER ADDRESS: 2070

EXECUTION BEGINS:

TOP	ASSUMED	LARGER THAN	BOTTOM
TOP1=	0.2385		BOT1= 0.1000
TOP1=	0.2462		BOT1= 0.1050
TOP1=	0.2553		BOT1= 0.1100
TOP1=	0.2659		BOT1= 0.1150
TOP1=	0.2784		BOT1= 0.1200
TOP1=	0.2933		BOT1= 0.1250
TOP1=	0.1109		BOT1= 0.1300
TOP1=	0.1321		BOT1= 0.1350
TOP1=	0.1576		BOT1= 0.1400
TOP1=	0.1889		BOT1= 0.1450
TOP1=	0.2276		BOT1= 0.1500
TOP1=	0.2763		BOT1= 0.1550
TOP1=	0.3392		BOT1= 0.1600
TOP1=	0.4225		BOT1= 0.1650
TOP1=	0.5374		BOT1= 0.1700
TOP1=	0.7043		BOT1= 0.1750
TOP1=	0.9669		BOT1= 0.1800
TOP1=	1.4359		BOT1= 0.1850
TOP1=	2.5207		BOT1= 0.1900
TOP1=	7.1867		BOT1= 0.1950
TOP1=	-11.2527		BOT1= 0.2000
TOP2=	0.1000		BOT2= -0.1038
TOP2=	0.1500		BOT2= -0.1758
TOP2=	0.2000		BOT2= -0.2808
TOP2=	0.2500		BOT2= -0.4765
TOP2=	0.3000		BOT2= -1.1361
TOP2=	0.3500		BOT2= 2.8493
TOP2=	0.4000		BOT2= 0.5282
TOP2=	0.4500		BOT2= 0.2263
TOP2=	0.5000		BOT2= 0.4925
TOP2=	0.5500		BOT2= 0.2287
TOP2=	0.6000		BOT2= -0.2534
TOP2=	0.6500		BOT2= -0.1244
TOP2=	0.7000		BOT2= -0.1487
TOP2=	0.7500		BOT2= -0.1888
TOP2=	0.8000		BOT2= -0.2260
TOP2=	0.8500		BOT2= -0.2612
TOP2=	0.9000		BOT2= -0.2950
TOP2=	0.9500		BOT2= -0.3276
TOP2=	1.0000		BOT2= -0.3593

BOTTOM ASSUMED LARGER THAN TOP

TOP3=	0.2135	BOT3=	0.1000
TOP3=	0.2124	BOT3=	0.1500
TOP3=	0.6417	BOT3=	0.2000
TOP3=	1.3783	BOT3=	0.2500
TOP3=	2.4990	BOT3=	0.3000
TOP3=	4.0807	BOT3=	0.3500
TOP3=	6.2001	BOT3=	0.4000
TOP3=	8.9341	BOT3=	0.4500
TOP3=	12.2596	BOT3=	0.5000
TOP3=	16.5533	BOT3=	0.5500
TOP3=	21.5920	BOT3=	0.6000
TOP3=	27.5527	BOT3=	0.6500
TOP3=	34.5121	BOT3=	0.7000
TOP3=	42.5470	BOT3=	0.7500
TOP3=	51.7342	BOT3=	0.8000
TOP3=	62.1507	BOT3=	0.8500
TOP3=	73.8731	BOT3=	0.9000
TOP3=	86.9784	BOT3=	0.9500
TOP3=	101.5434	BOT3=	1.0000
TOP4=	0.1000	BOT4=	0.0061
TOP4=	0.1500	BOT4=	0.0031
TOP4=	0.2000	BOT4=	0.0101
TOP4=	0.2500	BOT4=	0.0170
TOP4=	0.3000	BOT4=	0.0240
TOP4=	0.3500	BOT4=	0.0310
TOP4=	0.4000	BOT4=	0.0379
TOP4=	0.4500	BOT4=	0.0449
TOP4=	0.5000	BOT4=	0.0519
TOP4=	0.5500	BOT4=	0.0589
TOP4=	0.6000	BOT4=	0.0658
TOP4=	0.6500	BOT4=	-0.0272
TOP4=	0.7000	BOT4=	-0.1202
TOP4=	0.7500	BOT4=	-0.2133
TOP4=	0.8000	BOT4=	-0.3063
TOP4=	0.8500	BOT4=	-0.3993
TOP4=	0.9000	BOT4=	-0.4923
TOP4=	0.9500	BOT4=	-0.5854
TOP4=	1.0000	BOT4=	-0.6784

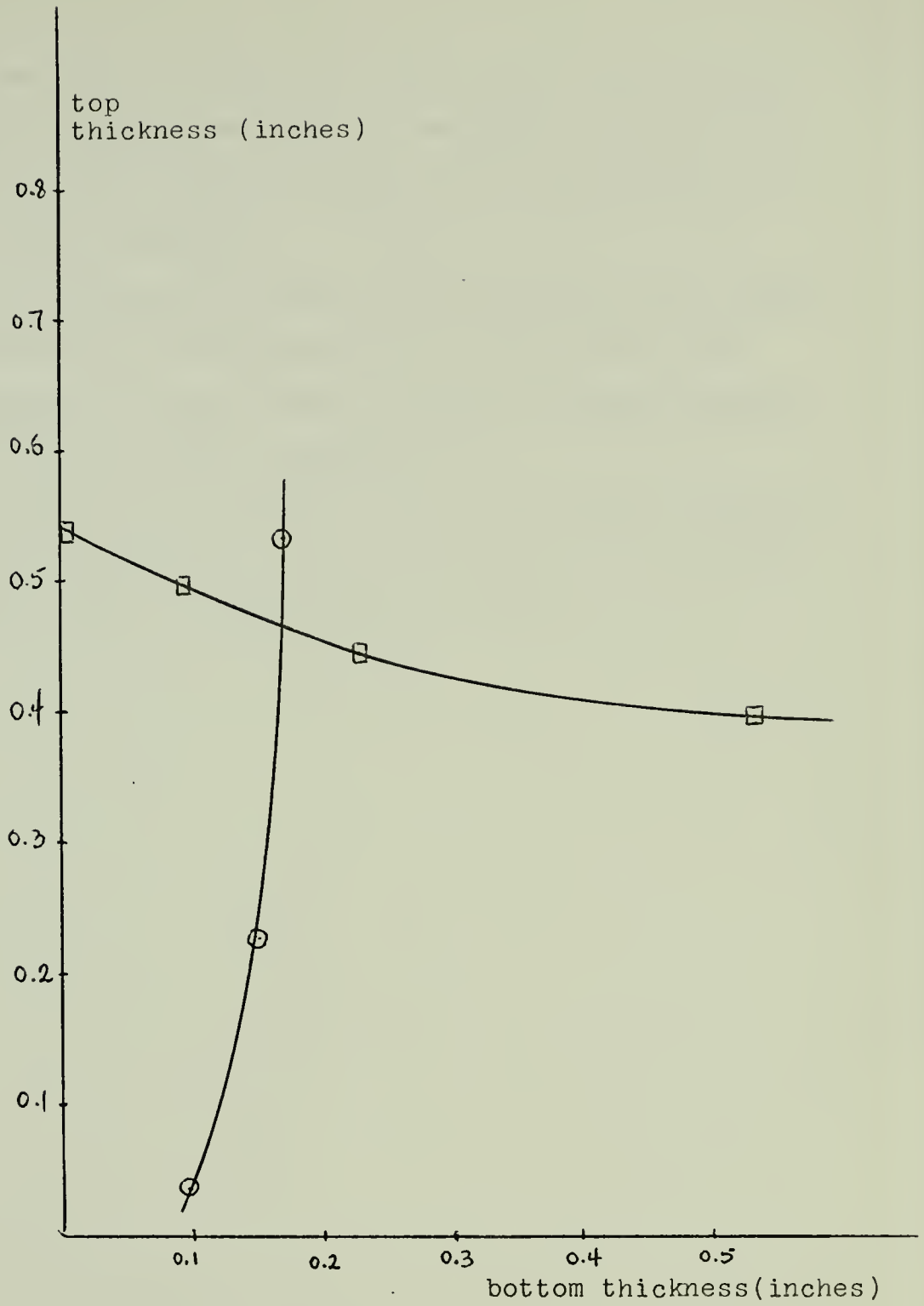
END

CP VIC OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

JOB HANDLING CHARGE	\$.35 / JOB	.35
175 LINES PRINTED PR2	\$ 1.25 / K LN	.22
67 CARDS READ	\$ 1.50 / K CD	.10
00 PLOTTER VECTORS	\$.25 / 1000	.00
15 MODEL 70 SECONDS	\$25.00 / HOUR	.10
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE	\$.77

PREIR 490 14731 LOGGED OUT 05/06/74 15:14. \$ 14.10 LEFT AFTER 17 LOGINS.

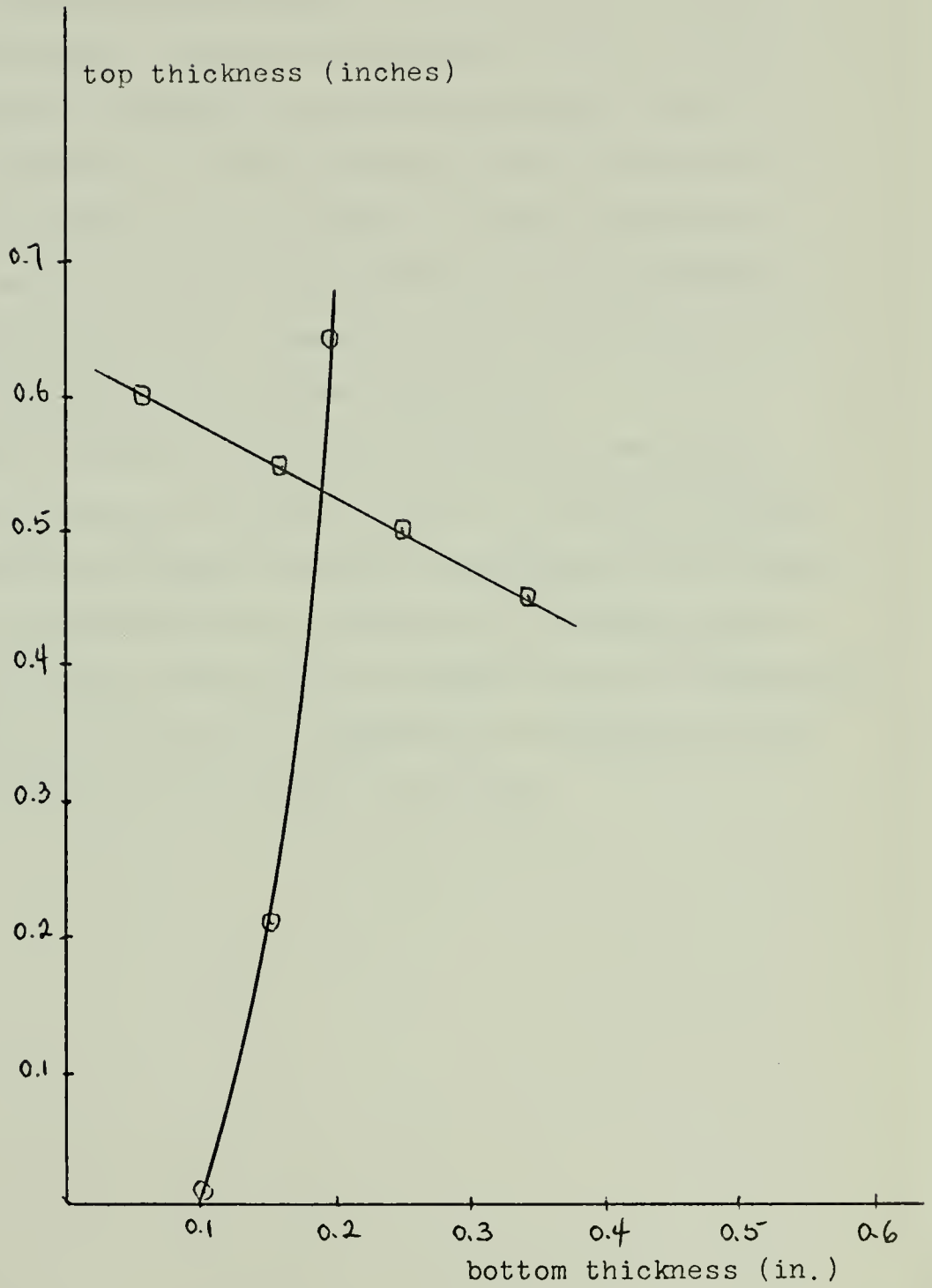
From these lists of thickness values printed by the computer, we can see that the acceptable ones are those marked with a \parallel sign. Plotting these corresponding pairs of values properly, as is next exemplified, allows for one to read from the intersection of the two lines, which falls within the range set by the assumption that originated it, the values of thicknesses at the top and at the bottom which satisfy the model we started with.



We can see that the thickness values are easily read from the intersection obtained and they are acceptable because they fall within the range assumed to derive the formulas that originated them namely $TOP > BOT$.

Next will be plotted the case $BOT > TOP$ and as can be seen, again an intersection is obtained which in this case is not acceptable because it occurs within the range $TOP > BOT$ contrary to the assumed $BOT > TOP$ that originated it.

With a little more experience, this can be seen immediately from the computer printouts, thus avoiding loss of time doing a plot that will lead to a nonacceptable result.



Now having obtained the values for the thicknesses from the plots, we verify them through the use of the next program based on the formulation previously derived. As can be noted, we obtain an acceptable value for the stress at the top (since we are considering a hogging bending moment) which we said before in the derivation of the equations should not exceed the design stress in tension. In this case the design stress was $DST=20000$ psi and we see that the stress at the top $STH=20037.5$ the 37.5 coming possibly from inaccuracy when reading the thicknesses from the plot, and computer roundoffs.

Note that the stress at the bottom, since there was no limit placed on it except the Bryan formula to resist buckling caused by compressive inplane loading, is attaining a too large value which is unacceptable and unrealistic. Here again there should be no difference between SBH and SBCRIT and the difference that can be noted is also due to inaccuracy in reading the thicknesses from the plot and computer round-offs.


```

0024 C *ZL* IS THE DEPTH
0024 ZL=91.7
0030 F=AL/ZL
0031 C *ZL* IS THE DEPTH
0031 D=L/ZL
0034 F=L/ZL
0035 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0035 FOR STEEL
0035 POI=.3
0036 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0036 FOR STEEL
0036 E=29.0E6
0040 C *ST* IS THE DESIGN STRESS
0040 ST=20.0
0054 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0054 S=10.
0060 C *K* IS THE CORRECTION FACTOR
0060 K=(.5+ZL/AL+.75)/(25+.85)
0063 F=.5*E*POI*(1+.5*ZL/AL)/E
0064 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0064 T=1.4
0064 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0064 B=.1
0064 Z=91.7*(.5*POI+D*TOP)
0064 DE=*.5*E*Z*(.5*POI+.75*TOP*(TOP+BOT)+(D*TOP*TOP))
0070 C *STH* IS THE STRESS ON TOP DUE TO BOWING
0070 STH=Z/DE
0074 F=ITH*(S/AL)*STH
0074 C FORMAT(' ',2X,IS TH='E11.3)
0074 F=560.0*(F*TOP+Z*DE)
0074 C *SHE* IS THE STRESS ON BOT DUE TO BOWING
0074 SHE=C/D*F
0074 C WRITE(5,*)SHE
0082 C FORMAT(' ',2X,IS HE='E11.3)
0082 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
0082 SBCRIT=PI*E*BOT/ZL
0082 C WRITE(5,*)SBCRIT
0084 110 FORMAT(' ',12X,IS CRIT='E11.3)
0084 P=2.0*F*TH*(3*(TOP+BOT)
0084 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0084 DITOP=(D*BOT+Z*TOP)/E
0084 C WRITE(5,*)DITOP
0084 40 C FORMAT(' ',18X,IS DITOP='E12.4)
0084 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0084 DIBOT=(D*TOP+Z*BOT)/E
0084 C WRITE(5,*)DIBOT
0084 50 C FORMAT(' ',18X,IS DIBOT='E12.4)
0084 C
0084 60 C
0084 70 C
0084 80 C
0084 90 C
0084 100 C
0084 110 C
0084 120 C
0084 130 C
0084 140 C
0084 150 C
0084 160 C
0084 170 C
0084 180 C
0084 190 C
0084 200 C
0084 210 C
0084 220 C
0084 230 C
0084 240 C
0084 250 C
0084 260 C
0084 270 C
0084 280 C
0084 290 C
0084 300 C
0084 310 C
0084 320 C
0084 330 C
0084 340 C
0084 350 C
0084 360 C
0084 370 C
0084 380 C
0084 390 C
0084 400 C
0084 410 C
0084 420 C
0084 430 C
0084 440 C
0084 450 C
0084 460 C
0084 470 C
0084 480 C
0084 490 C
0084 500 C
0084 510 C
0084 520 C
0084 530 C
0084 540 C
0084 550 C
0084 560 C
0084 570 C
0084 580 C
0084 590 C
0084 600 C
0084 610 C
0084 620 C
0084 630 C
0084 640 C
0084 650 C
0084 660 C
0084 670 C
0084 680 C
0084 690 C
0084 700 C
0084 710 C
0084 720 C
0084 730 C
0084 740 C
0084 750 C
0084 760 C
0084 770 C
0084 780 C
0084 790 C
0084 800 C
0084 810 C
0084 820 C
0084 830 C
0084 840 C
0084 850 C
0084 860 C
0084 870 C
0084 880 C
0084 890 C
0084 900 C
0084 910 C
0084 920 C
0084 930 C
0084 940 C
0084 950 C
0084 960 C
0084 970 C
0084 980 C
0084 990 C
0084 1000 C

```

```

0084 [S] * 0224 [V] AL 0226 [V] B 0334 [V] DRA 0330 [V] D
0340 [V] F 0336 [V] .R 0350 [V] DST 0364 [V] S
0370 [V] G 0338 [V] TOP 0360 [V] BOT 0398 [V] A
0388 [V] ITH 0388 [V] STH 0374 [V] DE 0390 [V] K 0280 [V] C

```


STEP OUT OF SEQUENCE THE PROLOGUE AXIS LOCATION

IC (V) 5.00	182 (U) 11	204 (V) 5.00	02.3 (U) 11	MRC (V) P
BC (V) 1.00	283 (U) 47	215 (V) 5.00	03.6 (U) 57	274 (V) 1.0

PROLOGUE LABELS:

270	274	278	272	244	250
274	278	282	286	290	294
298	302	306	310	314	318
322	326	330	334	338	342

INTERMEDIATE:

244	274	282	286	290	294
298	302	306	310	314	318
322	326	330	334	338	342
346	350	354	358	362	366

COMMENTARY:

C/E

DEFINITION SUBROUTINES:

C/F

RANGE & ADDRESS 2070

EQUATION 1015:

SIZE = 227,449
 GRAB = 1434,045
 SIZELIM = 20814,172
 SIZELIM = 20,000
 SIZELIM = 33,521

END

CF MID OPERATING SYSTEM (FERG) REVISION 12 3/24/74 GENERATED 5/16/74

100	1.25	1.00	.38
85	1.25	1.00	.11
54	1.25	1.00	.27
20	1.25	1.00	.20
13	1.25	1.00	.20
20	1.25	1.00	.20
		TOTAL	.62

REIP 402 14731 LOGGED OUT 5/16/74 15:43. B 11.79 LEFT AFTER 20 LOGINS.

SUMMARY FOR THE PRECEDING CASE

Length	= 500 ft
Beam	= AL/8
Draft	= B/3
Depth	= AL/9
DST	= 20000 psi
S	= 10 inches
Material	- STEEL
TOP	= 0.468 inches
BOT	= 0.166 inches
STH	= 20037.45
SBH	= 30434.89
SBCRIT	= 29886.80
DITOP	= 22.06 ft
DIBOT	= 33.50 ft

We can already notice here that the stress in the bottom is attaining a too large value due to the fact that, since no limit was imposed on it, it is dependent only on the chosen value for S and the respective capacity to resist buckling provided by S and BOT while at the same time stressing the top plate to its maximum allowed DST.

In the following two cases only the spacing S between the longitudinal stiffening members will be allowed to change and the results obtained are simply stated as previously exemplified and were determined in a similar way.

From the results obtained we can note the effect caused by the change of S.


```

0224 C *AL* IS THE LENGHT
0224 AL=522.
022C B=AL/8.
0218 C *DRA* IS THE DRAFT
0218 DRA=B/3.
0224 D=AL/9.
0232 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0232 FOR STEEL
0232 POI=0.3
0232 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0232 FOR STEEL
0232 E=30.*10**6
0240 C *DST* IS THE DESIGN STRESS
0240 DST=20000.
0254 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0254 S=24.
025C *BM* IS THE BENDING MOMENT
025C M=(B*DRA*AL*AL*2.75)/(35.*35.)
0258 G=0.30396*S*S*(1.-POI*POI)/E
0258 A=9.*B*B*D/(142.*BM)
0258 C=3.*B*D/(70.*BM)
0258 F=3.*D*D/(142.*BM)
026C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
026C WRITE(5,17)
026C 10 FORMAT('1',25X,'TOP ASSUMED LARGER THAN BOTTOM')
0265 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0265 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0265 BOT1=0.1
0274 11 TOP1=(M*BOT1*BOT1*BOT1)/((B+D)*G*DST-D*BOT1*BOT1)
0274 WRITE(5,20)TOP1,BOT1
0274 20 FORMAT('1',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
0274 BOT1=BOT1+.25
0274 IF(BOT1.GT.1.)GO TO 30
0274 GO TO 11
0274 30 TOP2=0.1
0274 31 BOT2=TOP2*(12.*D-(C+F)*DST*TOP2)/(DST*(A+C)*TOP2-12.*B)
0274 WRITE(5,40)TOP2,BOT2
0274 40 FORMAT('1',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
0274 TOP2=TOP2+.25
0274 IF(TOP2.GT.1.)GO TO 50
0274 GO TO 31
0274 50 CONTINUE
0274 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
0274 WRITE(5,60)
0274 60 FORMAT('1',25X,'BOTTOM ASSUMED LARGER THAN TOP')
0274 BOT3=0.1
0274 61 TOP3=((B+D)*BOT3*BOT3*BOT3-D*G*DST*BOT3)/(B*G*DST)
0274 WRITE(5,70)TOP3,BOT3
0274 70 FORMAT('1',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
0274 BOT3=BOT3+.25
0274 IF(BOT3.GT.1.)GO TO 80
0274 GO TO 61

```


STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

```

0402      80 TOP4=0.1
040A      81 BOT4=(12.*(B+D)-(A+C)*DST*TOP4)/((C+F)*DST)
044E      WRITE(5,90)TOP4,BOT4
0472      90 FORMAT(' ',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
049A      TOP4=TOP4+0.75
04A6      IF(TOP4.GT.1.)GO TO 100
04B8      GO TO 81
04BC      100 CONTINUE
04BC      END
0400(S)  .U      0400(V) AL      0402(V) B      0400(V) DRA      0408(V) D
0402(V) POI      0408(V) E      0402(S) .F      0408(V) DST      0500(V) S
0404(V) BM      0518(V) G      0524(V) A      0520(V) C      0534(V) F
0402(L) 12      0220(S) @I      0532(V) BOT1      0176(L) 11      0542(V) TOP1
0402(L) 20      0220(L) 30      0550(V) TOP2      0234(L) 31      0554(V) BOT2
0404(L) 40      02FF(L) 60      0312(L) 62      0564(V) BOT3      0348(L) 61
0406(V) TOP3      0388(L) 70      0402(L) 82      0560(V) TOP4      040A(L) 81
0406(V) BOT4      0472(L) 90      0480(L) 120      0000(S) .V
0406      L
0406      OF
    
```

PROGRAM LABELS:

2070 *MAIN*	2004 .V	2A34 .COMP	2C12 @I	25E4 .R	2AFC .ZF
2A5A \$6	2624 .A	2B58 .MES	29D6 .W	27DE EXP	2964 AIM
26D6 ALOG	2A9C .RARG	2AF8 .5	27DE AEXP	2B00 .ERCNT	2ACE \$8
2870 .0	2BD6 .U	3D2E			

ENTRY-POINTS:

25E4 .R	2624 .A	26D6 ALOG	27DE AEXP	27DE EXP	2964 AIM
29D6 .W	2A34 .COMP	2A5A \$6	2A9C .RARG	2ACE \$8	2AF8 .5
2AFC .ZF	2B00 .ERCNT	2B72 .0	2B58 .MES	2BE2 .U	2C08 .V
2C12 @I					

COMMON-BLOCKS:

ONE

UNDEFINED SUBROUTINES:

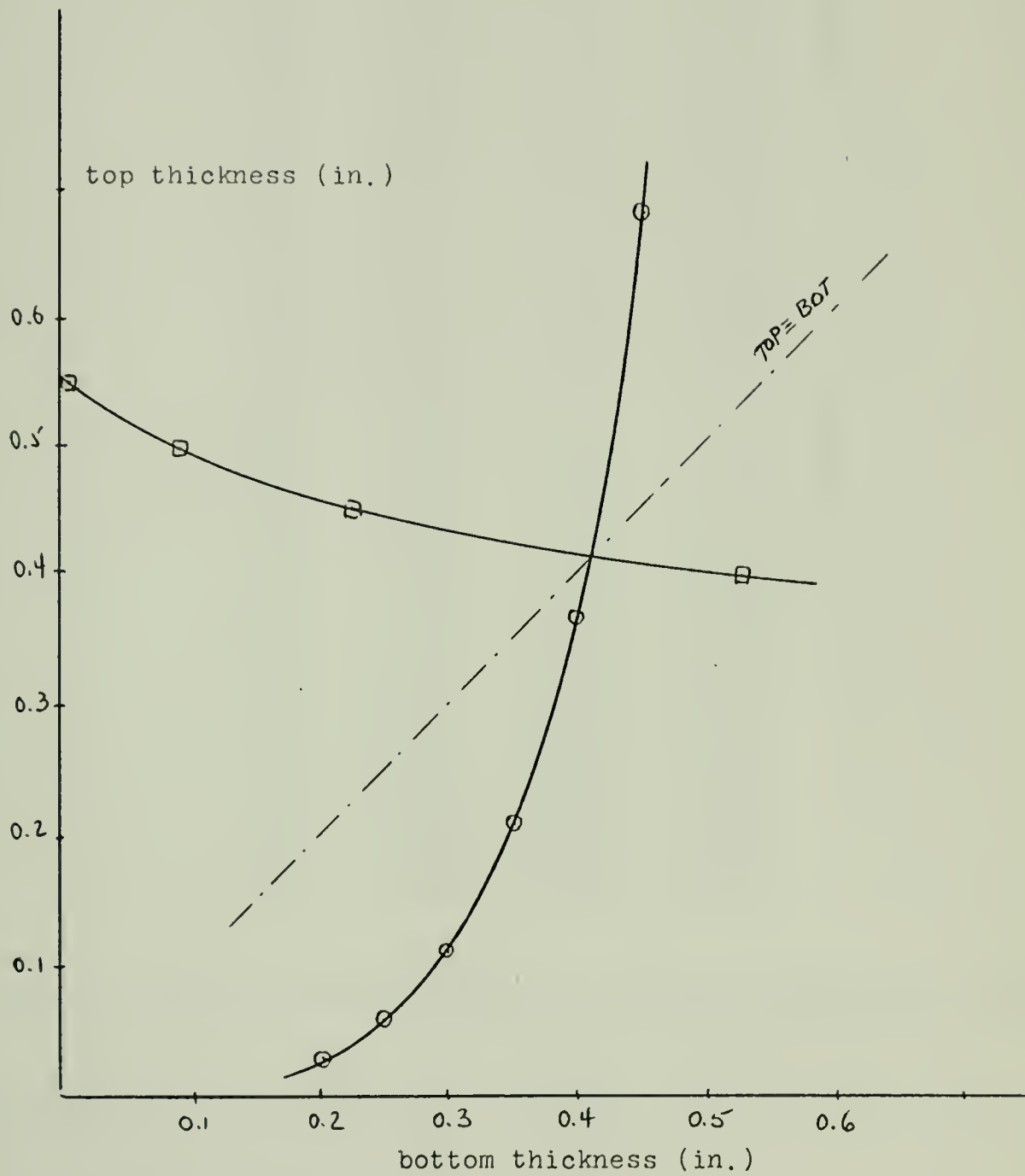
ONE

TRANSFER ADDRESS 2072

EXECUTION BEGINS:

TOP ASSUMED LARGER THAN BOTTOM

TOP1=	2.2033	BOT1=	0.1200
TOP1=	2.2115	BOT1=	0.1500
TOP1=	2.2288	BOT1=	0.2200
TOP1=	2.2626	BOT1=	0.2500
TOP1=	2.1156	BOT1=	0.3200
TOP1=	2.2096	BOT1=	0.3500
TOP1=	2.3737	BOT1=	2.4200
TOP1=	2.6827	BOT1=	2.4500
TOP1=	1.3697	BOT1=	0.5200
TOP1=	3.7376	BOT1=	2.5500
TOP1=	-32.1585	BOT1=	2.6200
TOP1=	-4.4245	BOT1=	2.6500
TOP1=	-2.8099	BOT1=	2.7200
TOP1=	-2.2619	BOT1=	0.7500
TOP1=	-2.0047	BOT1=	2.8200
TOP1=	-1.8681	BOT1=	2.8500
TOP1=	-1.7933	BOT1=	2.9200
TOP1=	-1.7543	BOT1=	2.9500
TOP1=	-1.7379	BOT1=	1.0000
TOP2=	2.1700	BOT2=	-2.1238
TOP2=	2.1500	BOT2=	-2.1758
TOP2=	2.2700	BOT2=	-2.2808
TOP2=	2.2500	BOT2=	-2.4765
TOP2=	2.3200	BOT2=	-1.1361
TOP2=	2.3500	BOT2=	2.8493
TOP2=	2.4200	BOT2=	2.5222
TOP2=	2.4500	BOT2=	2.2263
TOP2=	2.5200	BOT2=	2.0925
TOP2=	2.5500	BOT2=	2.2287
TOP2=	2.6200	BOT2=	-2.2534
TOP2=	2.6500	BOT2=	-2.1244
TOP2=	2.7000	BOT2=	-2.1487
TOP2=	2.7500	BOT2=	-2.1888
TOP2=	2.8000	BOT2=	-2.2260
TOP2=	2.8500	BOT2=	-2.2612
TOP2=	2.9000	BOT2=	-2.2950
TOP2=	2.9500	BOT2=	-2.3276
TOP2=	1.0000	BOT2=	-2.3593



BOTTOM	ASSUMED	LARGER THAN	TOP
TOP3=	-0.2775	BOT3=	0.1000
TOP3=	-0.0949	BOT3=	0.1500
TOP3=	-2.2867	BOT3=	0.2000
TOP3=	-2.2444	BOT3=	2.2500
TOP3=	0.2406	BOT3=	2.3000
TOP3=	0.1769	BOT3=	2.3500
TOP3=	0.3729	BOT3=	2.4000
TOP3=	2.6371	BOT3=	2.4500
TOP3=	2.9782	BOT3=	2.5000
TOP3=	1.4047	BOT3=	2.5500
TOP3=	1.9250	BOT3=	2.6000
TOP3=	2.5478	BOT3=	2.6500
TOP3=	3.2816	BOT3=	2.7000
TOP3=	4.1349	BOT3=	2.7500
TOP3=	5.1162	BOT3=	2.8000
TOP3=	6.2340	BOT3=	2.8500
TOP3=	7.4970	BOT3=	2.9000
TOP3=	8.9137	BOT3=	2.9500
TOP3=	10.4925	BOT3=	1.0000
TOP4=	0.1000	BOT4=	0.9961
TOP4=	0.1500	BOT4=	0.9931
TOP4=	0.2000	BOT4=	0.9901
TOP4=	0.2500	BOT4=	0.9871
TOP4=	0.3000	BOT4=	0.9841
TOP4=	0.3500	BOT4=	0.9811
TOP4=	0.4000	BOT4=	0.9781
TOP4=	0.4500	BOT4=	0.9751
TOP4=	0.5000	BOT4=	0.9721
TOP4=	0.5500	BOT4=	0.9691
TOP4=	0.6000	BOT4=	0.9661
TOP4=	0.6500	BOT4=	0.9631
TOP4=	0.7000	BOT4=	0.9601
TOP4=	0.7500	BOT4=	0.9571
TOP4=	0.8000	BOT4=	0.9541
TOP4=	0.8500	BOT4=	0.9511
TOP4=	0.9000	BOT4=	0.9481
TOP4=	0.9500	BOT4=	0.9451
TOP4=	1.0000	BOT4=	0.9421

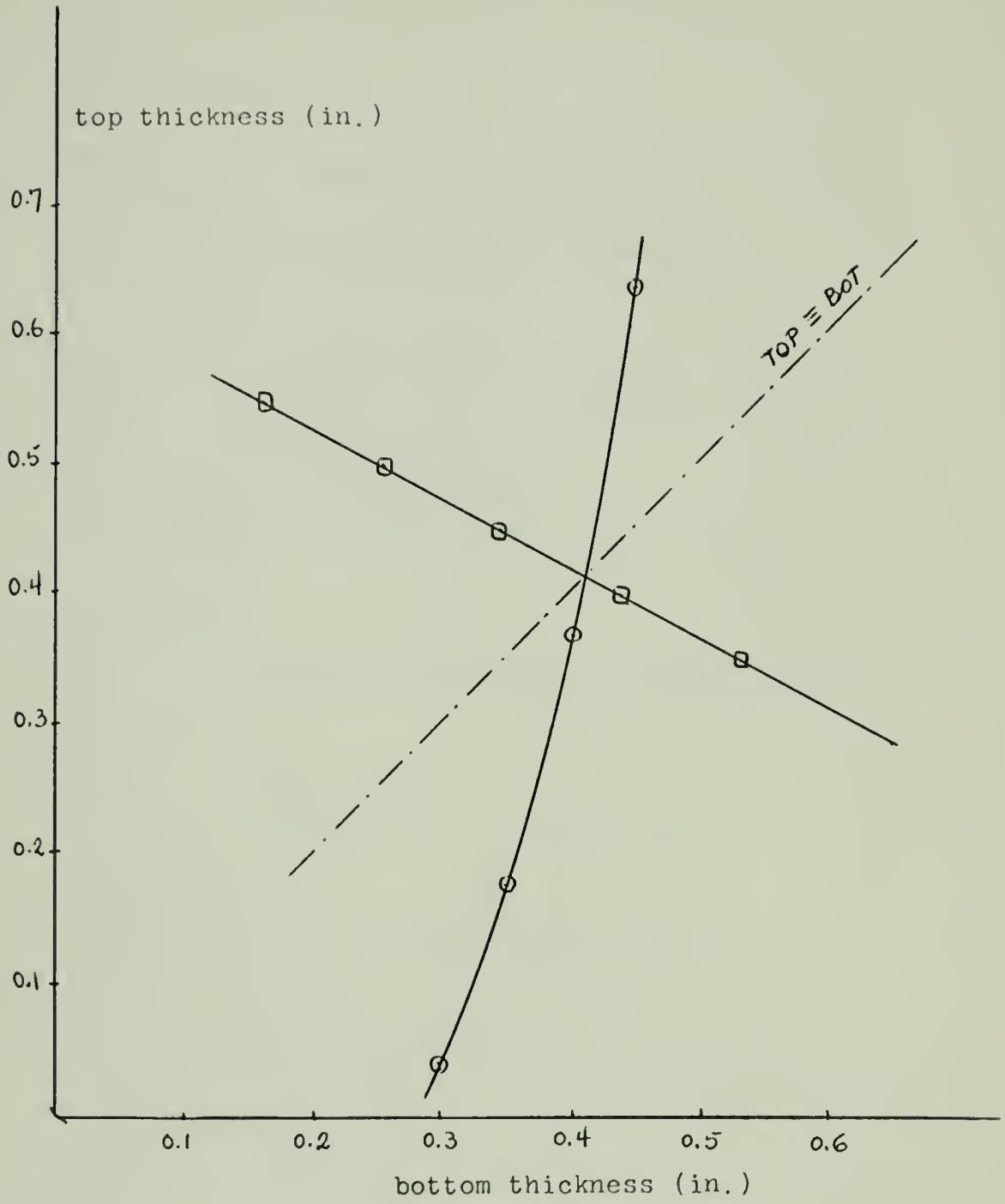


END

FVIC OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74 0

JOB HANDLING CHARGE	\$.35 / JOB	.35
173 LINES PRINTED PR2	\$ 1.25 / K LN	.22
67 CARDS READ	\$ 1.50 / K CD	.10
00 PLOTTER VECTORS	\$.25 / 1000	.00
20 MODEL 70 SECONDS	\$25.00 / HOUR	.14
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE \$.81

REIR 490 14731 LOGGED OUT 25/26/74 15:20. \$ 13.29 LEFT AFTER 18 LOGINS.




```

0224 C *AL* IS THE LENGHT
0224 AL=500.
0270 C *A* IS THE DRAFT
0270 A=AL/8.
0218 C *DRA* IS THE DRAFT
0218 DRA=B/3.
0224 C *D* IS THE POISSON'S RATIO OF THE MATERIAL
0232 C FOR STEEL
0232 POI=0.3
0238 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0238 E=30.*10.**+6
0240 C *DST* IS THE DESIGN STRESS
0240 DST=20000.
0254 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0254 S=37.
0250 C *M* IS THE BENDING MOMENT
0250 M=(B*DRA*AL*AL*7.75)/(35.*35.)
0254 C *G* IS THE THICKNESS OF THE TOP PLATING
0254 G=7.30396*S*S*(1.-POI+POI)/E
0264 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0264 TOP=0.4125
0270 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0270 BOT=7.4125
0274 C *DEN* IS THE DENOMINATOR
0274 DEN=D*(3.*B*B*TOP*BOT+2.*B*D*TOP*(TOP+BOT)+(D*D*TOP*TOP))
0280 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
0280 STH=A/DEN
0284 WRITE(5,60)STH
0274 60 FORMAT(' ',20X,'STH='F10.3)
0284 C *SBH* IS THE STRESS ON BOT DUE TO HOGGING
0284 SBH=C/DEN
0284 WRITE(5,90)SBH
0284 90 FORMAT(' ',20X,'SBH='F10.3)
0290 C *SPCRIT* IS THE CRITICAL STRESS ON BOT
0290 SPCRIT=BOT*BOT/G
0294 WRITE(5,110)SPCRIT
0284 110 FORMAT(' ',17X,'SPCRIT='F10.3)
0284 B=2.*D*TOP+B*(TOP+BOT)
0284 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0284 DITOP=D*(B+BOT+D*TOP)/B
0284 WRITE(5,42)DITOP
0284 40 FORMAT(' ',18X,'DITOP='F10.4)
0284 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0284 DIBOT=D*(B+TOP+D*TOP)/B
0284 WRITE(5,54)DIBOT
0284 50 FORMAT(' ',18X,'DIBOT='F10.4)
0284 END
0284 .U 0324 [V] AL 0320 [V] B 0334 [V] DRA 0330 [V] D
0284 .V 0340 [V] E 0200 [S] .R 0350 [V] DST 0364 [V] S
0284 .V 0370 [V] G 0388 [V] TOP 0390 [V] BOT 0394 [V] A
0284 .V 0384 [V] ST- 0174 [L] 62 0000 [S] @I 0388 [V] C

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 2

C(V) SBH C1E2(L) 92 030(V) SBCKIT 0228(L) 110 0304(V) P
 X(V) DITOP 02A8(L) 42 0300(V) DIBOT 0304(L) 50 0000(S) .V
 XEQ L
 F

OGRIN LABELS:

2070 MAIN	2A60 .V	2890 .COMP	2A6E .I	2440 .R	2958 .ZE
2896 \$6	2480 .A	2984 .MES	2832 .W	263A EXP	2700 AIN
2532 ALOG	28F8 .RARG	2954 .5	263A AEXP	295C .ERCNT	292A \$8
2950 .0	2A32 .U	3B8A			

TRY-POINTS:

2440 .R	2480 .A	2532 ALOG	263A AEXP	263A EXP	2700 AIN
2832 .W	2890 .COMP	28B6 \$6	28F8 .RARG	292A \$8	2954 .5
2950 .ZERO	295C .ERCNT	295F .0	2984 .MES	2A3E .U	2A64 .V
2A6E .I					

NAME-BLOCKS:

E

DEFINED SUBROUTINES:

E

TRANSFER ADDRESS 2070

ROUTINE BEGINS:

STH= 2037.113
 SBH= 2037.113
 SBCKIT= 20505.348
 DITOP= 27.7778
 DIBOT= 27.7778

END

F VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74 0

JOB HANDLING CHARGE	\$.35 / JOB	.35
85 LINES PRINTED PR2	\$ 1.25 / K LN	.11
54 CARDS READ	\$ 1.50 / K CD	.08
20 PLOTTER VECTORS	\$.25 / 1000	.00
13 MODEL 70 SECONDS	\$25.00 / HOUR	.09
20 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE \$.63

FEIR 490 14731 LOGGED OUT 05/06/74 15:46. \$ 11.16 LEFT AFTER 21 LOGINS.

SUMMARY

Length	=	500 ft
Beam	=	AL/8
Draft	=	B/3
Depth	=	AL/9
DST	=	20000 psi
S	=	30"
Material	-	STEEL
TOP	=	0.4125"
BOT	=	0.4125"
STH	=	20037.11
SBH	=	20037.11
SBCRIT	=	20505.43
DITOP	=	27.78'
DIBOT	=	27.78'

Note that this is the case where two acceptable solutions exist although they end up being the same because TOP=BOT.

Note that the neutral axis is located half way between top and bottom and the stress is the same at the top and at the bottom, as it was expected beforehand.


```

0004 C *AL* IS THE LENGHT
0004 AL=500.
0008 C *DRA* IS THE DRAFT
0008 DRA=B/3.
0012 C *AL* IS THE LENGHT
0012 AL=500.
0016 C *DRA* IS THE DRAFT
0016 DRA=B/3.
0020 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0020 FOR STEEL
0024 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0024 FOR STEEL
0028 E=37.*10.**6
0032 C *DST* IS THE DESIGN STRESS
0032 DST=20700.
0036 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0036 S=120.
0040 C *BM* IS THE BENDING MOMENT
0040 BM=(B*DRA*AL*AL*2.75)/(35.*35.)
0044 G=2.30396*S*S*(1.-POI*POI)/E
0048 A=9.*B*B*D/(147.*BM)
0052 C=3.*B*D*D/(72.*BM)
0056 F=3.*D*D*D/(147.*BM)
0060 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
0060 WRITE(5,17)
0064 10 FORMAT('1',25X,'TOP ASSUMED LARGER THAN BOTTOM')
0068 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0072 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0076 BOT1=1.
0080 11 TOP1=B*BOT1*BOT1*BOT1/(B+D)*G*DST+D*BOT1*BOT1)
0084 WRITE(5,20)TOP1,BOT1
0088 20 FORMAT('1',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
0092 BOT1=BOT1+.25
0096 IF(BOT1.GT.2.)GO TO 30
0100 GO TO 11
0104 30 TOP2=.02
0108 31 BOT2=TOP2*(12.*D-(C+F)*DST*TOP2)/(DST*(A+C)*TOP2+12.*B)
0112 WRITE(5,40)TOP2,BOT2
0116 40 FORMAT('1',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
0120 TOP2=TOP2+.02
0124 IF(TOP2.GT.1.)GO TO 50
0128 GO TO 31
0132 50 CONTINUE
0136 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
0140 WRITE(5,60)
0144 60 FORMAT('1',25X,'BOTTOM ASSLMED LARGER THAN TOP')
0148 BOT3=1.
0152 61 TOP3=(B+D)*BOT3*BOT3*BOT3-D*G*DST*BOT3)/(B*G*DST)
0156 WRITE(5,70)TOP3,BOT3
0160 70 FORMAT('1',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
0164 BOT3=BOT3+.25
0168 IF(BOT3.GT.2.)GO TO 80
0172 GO TO 61

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 2

```

0402      80 TOP4=0.02
040A      81 BOT4=(12.*(B+D)-(A+C)*DST*TOP4)/((C+F)*DST)
044E      WRITE(5,90)TOP4,BOT4
0472      90 FORMAT(' ',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
049A      TOP4=TOP4+0.22
04A6      IF(TOP4.GT.1.)GO TO 100
04B8      GO TO 81
04BC      100 CONTINUE
04BC      END
00[S] .U      04C0[V] AL      04C8[V] B      04D0[V] DRA      04D8[V] D
00[V] POI      04E8[V] E      0200[S] .R      04F8[V] DST      0500[V] S
08[V] BM      0510[V] G      0528[V] A      0530[V] C      0538[V] F
00[L] 10      0000[S] @I      0530[V] BOT1      0176[L] 11      0540[V] TOP1
02[L] 20      0220[L] 30      0554[V] TOP2      0234[L] 31      0550[V] BOT2
04[L] 40      02F5[L] 50      0312[L] 60      0560[V] BOT3      0348[L] 61
02[V] TOP3      03B8[L] 70      0402[L] 80      0574[V] TOP4      040A[L] 81
08[V] BOT4      0472[L] 90      0480[L] 100      0000[S] .V
XEQ      L
DF

```

PROGRAM LABELS:

2070 *MAIN*	2000 .V	2A30 .COMP	2C1A @I	25EC .R	2B04 .ZE
2A62 \$6	2620 .A	2B60 .MES	29DE .W	27E6 EXP	296C AI
26DE ALOG	2AA4 .RARG	2B00 .5	27E6 AEXP	2B08 .ERCNT	2AD6 \$8
2B08 .0	2BDE .U	3D36			

ENTRY-POINTS:

25EC .R	2620 .A	26DE ALOG	27E6 AEXP	27E6 EXP	296C AI
29DE .W	2A30 .COMP	2A62 \$6	2AA4 .RARG	2AD6 \$8	2B00 .5
2B04 .ZERO	2B08 .ERCNT	2B0A .0	2B60 .MES	2BEA .U	2C10 .V
2C1A @I					

COMMON-BLOCKS:

ONE

UNDEFINED SUBROUTINES:

ONE

TRANSFER ADDRESS 2070

EXECUTION BEGINS:

TOP ASSUMED LARGER THAN BOTTOM

TOP1=	0.2423	BOT1=	1.0000
TOP1=	0.2868	BOT1=	1.0500
TOP1=	0.3378	BOT1=	1.1000
TOP1=	0.3960	BOT1=	1.1500
TOP1=	0.4626	BOT1=	1.2000
TOP1=	0.5385	BOT1=	1.2500
TOP1=	0.6253	BOT1=	1.3000
TOP1=	0.7245	BOT1=	1.3500
TOP1=	0.8382	BOT1=	1.4000
TOP1=	0.9688	BOT1=	1.4500
TOP1=	1.1191	BOT1=	1.5000
TOP1=	1.2929	BOT1=	1.5500
TOP1=	1.4947	BOT1=	1.6000
TOP1=	1.7305	BOT1=	1.6500
TOP1=	2.0078	BOT1=	1.7000
TOP1=	2.3366	BOT1=	1.7500
TOP1=	2.7305	BOT1=	1.8000
TOP1=	3.2081	BOT1=	1.8500
TOP1=	3.7959	BOT1=	1.9000
TOP1=	4.5327	BOT1=	1.9500
TOP1=	5.4783	BOT1=	2.0000
TOP2=	0.0200	BOT2=	-0.0182
TOP2=	0.0400	BOT2=	-0.0375
TOP2=	0.0600	BOT2=	-0.0579
TOP2=	0.0800	BOT2=	-0.0799
TOP2=	0.1000	BOT2=	-0.1038
TOP2=	0.1200	BOT2=	-0.1301
TOP2=	0.1400	BOT2=	-0.1595
TOP2=	0.1600	BOT2=	-0.1932
TOP2=	0.1800	BOT2=	-0.2328
TOP2=	0.2000	BOT2=	-0.2808
TOP2=	0.2200	BOT2=	-0.3417
TOP2=	0.2400	BOT2=	-0.4234
TOP2=	0.2600	BOT2=	-0.5421
TOP2=	0.2800	BOT2=	-0.7376
TOP2=	0.3000	BOT2=	-1.1361
TOP2=	0.3200	BOT2=	-2.4815
TOP2=	0.3400	BOT2=	10.9162
TOP2=	0.3600	BOT2=	1.6081
TOP2=	0.3800	BOT2=	0.8255
TOP2=	0.4000	BOT2=	0.5282
TOP2=	0.4200	BOT2=	0.3680
TOP2=	0.4400	BOT2=	0.2655
TOP2=	0.4600	BOT2=	0.1926
TOP2=	0.4800	BOT2=	0.1371
TOP2=	0.5000	BOT2=	0.0925
TOP2=	0.5200	BOT2=	0.0553
TOP2=	0.5400	BOT2=	0.0233
TOP2=	0.5600	BOT2=	-0.0050
TOP2=	0.5800	BOT2=	-0.0303
TOP2=	0.6000	BOT2=	-0.0534
TOP2=	0.6200	BOT2=	-0.0748
TOP2=	0.6400	BOT2=	-0.0948
TOP2=	0.6600	BOT2=	-0.1136
TOP2=	0.6800	BOT2=	-0.1315

TCP2=	0.7000	BOT2=	-0.1487
TCP2=	0.7200	BOT2=	-0.1651
TCP2=	0.7400	BOT2=	-0.1810
TCP2=	0.7600	BOT2=	-0.1964
TCP2=	0.7800	BOT2=	-0.2114
TCP2=	0.8000	BOT2=	-0.2260
TCP2=	0.8200	BOT2=	-0.2403
TCP2=	0.8400	BOT2=	-0.2543
TCP2=	0.8600	BOT2=	-0.2681
TCP2=	0.8800	BOT2=	-0.2816
TCP2=	0.9000	BOT2=	-0.2950
TCP2=	0.9200	BOT2=	-0.3081
TCP2=	0.9400	BOT2=	-0.3211
TCP2=	0.9600	BOT2=	-0.3340
TCP2=	0.9800	BOT2=	-0.3467
TCP2=	1.0000	BOT2=	-0.3593

Note that in this case by simple inspection of the coordinates of the points to be plotted we can see that the intersection will occur outside the acceptable range.

Note also that in order to obtain the coordinates of the points within a range of interest where the intersection may occur, different ranges and increments are considered in each program. This could be improved if at that time these printouts were available and a better idea of the thicknesses that should be expected existed.

BOTTOM	ASSUMED	LARGER THAN	TOP
TOP3=	-0.1776	BOT3=	1.0000
TOP3=	-0.1099	BOT3=	1.0500
TOP3=	-0.0310	BOT3=	1.1000
TOP3=	0.0596	BOT3=	1.1500
TOP3=	0.1625	BOT3=	1.2000
TOP3=	0.2782	BOT3=	1.2500
TOP3=	0.4072	BOT3=	1.3000
TOP3=	0.5501	BOT3=	1.3500
TOP3=	0.7074	BOT3=	1.4000
TOP3=	0.8797	BOT3=	1.4500
TOP3=	1.0674	BOT3=	1.5000
TOP3=	1.2711	BOT3=	1.5500
TOP3=	1.4914	BOT3=	1.6000
TOP3=	1.7287	BOT3=	1.6500
TOP3=	1.9836	BOT3=	1.7000
TOP3=	2.2567	BOT3=	1.7500
TOP3=	2.5484	BOT3=	1.8000
TOP3=	2.8594	BOT3=	1.8500
TOP3=	3.1901	BOT3=	1.9000
TOP3=	3.5410	BOT3=	1.9500
TOP3=	3.9128	BOT3=	2.0000
TOP4=	0.0200	BOT4=	1.1000
TOP4=	0.0400	BOT4=	1.1078
TOP4=	0.0600	BOT4=	1.1155
TOP4=	0.0800	BOT4=	1.1233
TOP4=	0.1000	BOT4=	0.9961
TOP4=	0.1200	BOT4=	0.9589
TOP4=	0.1400	BOT4=	0.9217
TOP4=	0.1600	BOT4=	0.8845
TOP4=	0.1800	BOT4=	0.8473
TOP4=	0.2000	BOT4=	0.8101
TOP4=	0.2200	BOT4=	0.7728
TOP4=	0.2400	BOT4=	0.7356
TOP4=	0.2600	BOT4=	0.6984
TOP4=	0.2800	BOT4=	0.6612
TOP4=	0.3000	BOT4=	0.6240
TOP4=	0.3200	BOT4=	0.5868
TOP4=	0.3400	BOT4=	0.5496
TOP4=	0.3600	BOT4=	0.5124
TOP4=	0.3800	BOT4=	0.4752
TOP4=	0.4000	BOT4=	0.4379
TOP4=	0.4200	BOT4=	0.4007
TOP4=	0.4400	BOT4=	0.3635
TOP4=	0.4600	BOT4=	0.3263
TOP4=	0.4800	BOT4=	0.2891
TOP4=	0.5000	BOT4=	0.2519
TOP4=	0.5200	BOT4=	0.2147
TOP4=	0.5400	BOT4=	0.1775
TOP4=	0.5600	BOT4=	0.1403
TOP4=	0.5800	BOT4=	0.1030
TOP4=	0.6000	BOT4=	0.0658
TOP4=	0.6200	BOT4=	0.0286
TOP4=	0.6400	BOT4=	0.0000
TOP4=	0.6600	BOT4=	0.0000
TOP4=	0.6800	BOT4=	0.0000



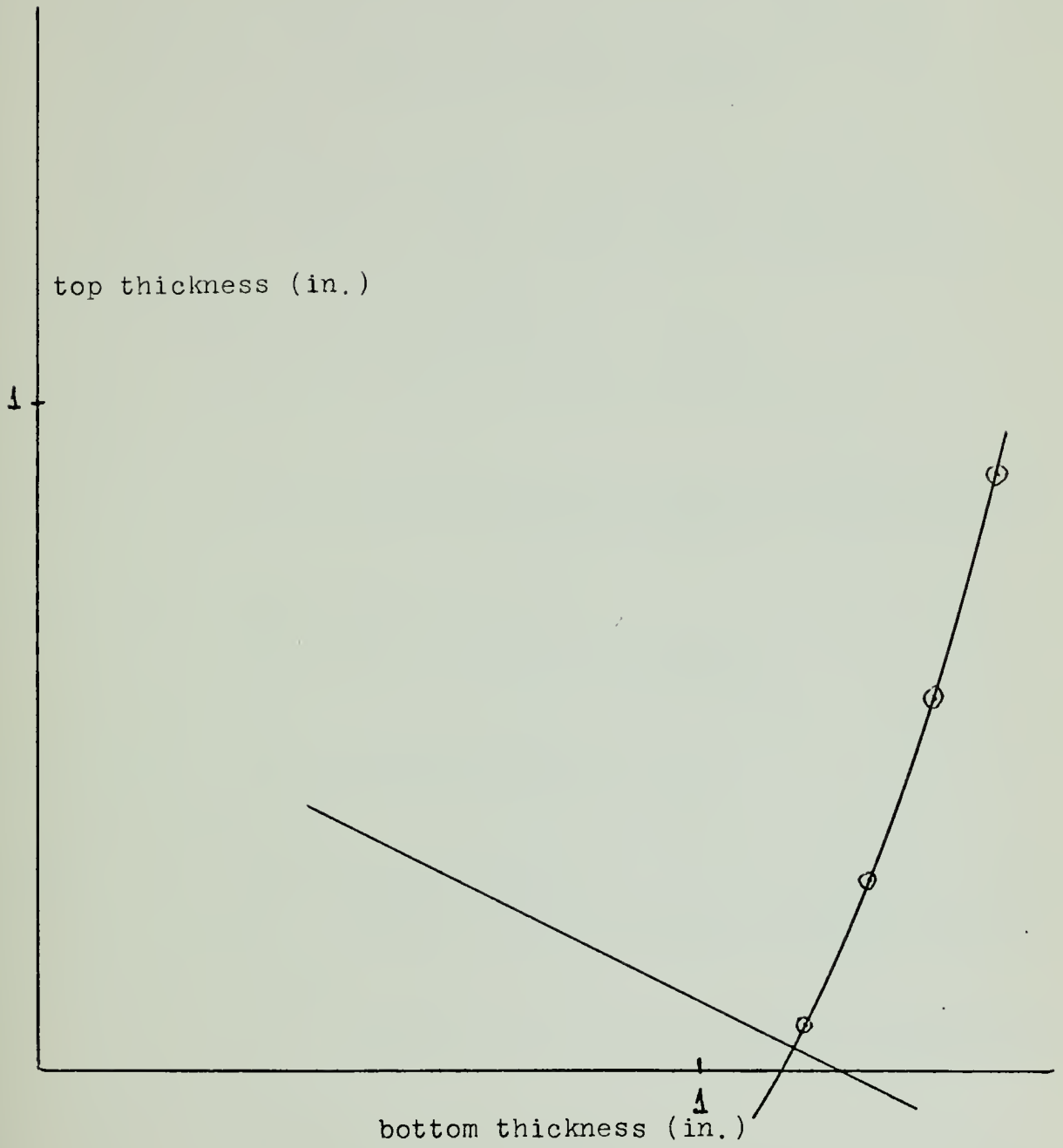
TOP4 =	2.7227	BO4 =	-2.1282
TOP4 =	2.7227	BO4 =	-2.1574
TOP4 =	2.7427	BO4 =	-2.1246
TOP4 =	2.7627	BO4 =	-2.2219
TOP4 =	2.7827	BO4 =	-2.2191
TOP4 =	2.8027	BO4 =	-2.3263
TOP4 =	2.8227	BO4 =	-2.3435
TOP4 =	2.8427	BO4 =	-2.3977
TOP4 =	2.8627	BO4 =	-2.4179
TOP4 =	2.8827	BO4 =	-2.4551
TOP4 =	2.9027	BO4 =	-2.4923
TOP4 =	2.9227	BO4 =	-2.5296
TOP4 =	2.9427	BO4 =	-2.5668
TOP4 =	2.9627	BO4 =	-2.6042
TOP4 =	2.9827	BO4 =	-2.6412
TOP4 =	1.0027	BO4 =	-2.6784

END

CPM VID OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74 0

JOB HANDLING CHARGE	\$.35 / JOB	.35
239 LINES PRINTED PR2	\$ 1.25 / K LN	.30
67 CARDS READ	\$ 1.50 / K CD	.12
00 PLOTTER VECTORS	\$.25 / 1000	.00
17 MODEL 70 SECONDS	\$25.00 / HOUR	.12
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE \$.87

REFIR 490 14731 LOGGED OUT 25/06/74 15:25. \$ 12.42 LEFT AFTER 19 LOGINS.




```

04 C *A) IS THE LENGTH
04 AL=L/3
06 C *B) IS THE WIDTH
06 BL=B/3
10 C *C) IS THE DEPTH
10 CL=L/3
14 C *D) IS THE POISSON'S RATIO OF THE MATERIAL
14 FOR STEEL
14 DDI=.3
18 C *E) IS THE YOUNG'S MODULUS OF THE MATERIAL
18 FOR STEEL
18 E=30.*10**6
22 C *F) IS THE DESIGN STRESS
22 FST=2.*10**6
26 C *G) IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
26 G=12.
30 C *H) IS THE AREA MOMENT OF INERTIA
30 I=(L**3)/AL*(L/3+.75)/(36.*36.)
34 C *I) IS THE AREA MOMENT OF INERTIA OF THE TOP PLATING
34 ITOE=L**3*(3*E+30*(1-DDI)*E)
38 C *J) IS THE THICKNESS OF THE TOP PLATING
38 JOE=.1
42 C *K) IS THE AREA MOMENT OF INERTIA OF THE BOTTOM PLATING
42 KTOE=L**3*(3*E+30*(1-DDI)*E)
46 C *L) IS THE THICKNESS OF THE BOTTOM PLATING
46 LOE=.1
50 C *M) IS THE DESIGN STRESS ON TOP DUE TO HOGGING
50 STT=(FST*(1-DDI)*KTOE+(FST-D)*E*(TOE+KTOE)+(D*E*KTOE*BOT))
54 C *N) IS THE STRESS ON TOP DUE TO SAGGING
54 STB=(FST*(1-DDI)*KTOE+(FST-D)*E*(TOE+KTOE)+(D*E*KTOE*BOT))
58 C *O) IS THE CRITICAL STRESS ON BOT
58 CRIIT=(FST-D)*E*(TOE+KTOE)+(D*E*KTOE*BOT)
62 C *P) IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
62 CTOE=(L**3*(3*E+30*(1-DDI)*E))/.
66 C *Q) IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
66 CBOE=(L**3*(3*E+30*(1-DDI)*E))/.

```

[S]	..	024 [V]	320 [y]	0324 [V]	0330 [V]
[V]	ROI	0340 [V]	110 [c]	0350 [V]	0364 [V]
[V]	...	0380 [V]	000 [y]	0394 [V]	0390 [V]
[V]	...	0300 [V]	174 [c]	0412 [S]	0300 [V]

STRENGTH OF STRAINING THE CIRCULAR AXIS LOCATION

(V) SDR	182(1) 50	308(V) SECRET	(208(1) 110	3300(V) P
(V) MICO	145(1) 40	304(V) DIRECT	(314(1) 50	2000(S) .V

GRAV LABEL :

170	2461 .1	2130 .000P	2176 .1	2448 .R	2760 .71
200E .A	2442 .1	2130 .000P	2176 .1	2442 EXP	2708 AT
33A .110	290 .000P	2150 .0	2142 EXP	2964 .ERCNT	2132 .3
44 .0	2474 .1	2130 .000P	2176 .1		

EXP-100-100 :

448 .1	2442 .1	2130 .000P	2176 EXP	2442 EXP	2708 AT
53A .1	280 .000P	2130 .0	2142 .000P	2932 .R	2950 .R
67 .070	2964 .000P	2150 .0	2130 .000P	2442 .1	2460 .1
475 .1					

OFFICE OF THE DIRECTOR

SECRET

SECRET

SECRET = 19774.573
 SECRET = 19774.573
 SECRET = 19774.573
 SECRET = 19774.573
 SECRET = 19774.573

OFFICE OF THE DIRECTOR (REVISION) 12 3/24/74 GENERATED (5/16/74

01	SECRET	1.35	/	100	.25
05	SECRET	1.35	/	100	.11
04	SECRET	1.35	/	100	.20
06	SECRET	1.35	/	100	.1
10	SECRET	1.35	/	100	.11
11	SECRET	1.35	/	100	.01
	TOTAL				.6

SECRET 19774.573 (5/16/74 15:40) 11.51 LEFT AFTER 22 LOGINS.

SUMMARY

Length	=	500'
Beam	=	AL/8
Draft	=	B/3
Depth	=	AL/9
DST	=	20000 psi
S	=	120"
Material	-	STEEL
TOP	=	0.03"
BOT	=	1.14"
STH	=	19771.88
SBH	=	9579.87
SBCRIT	=	9788.37
DITOP	=	37.42'
DIBOT	=	18.13'

Note in this case the large value chosen for S and associated with it the large value required for the thickness at the bottom to avoid buckling which nevertheless does not resist stresses larger than 9788 psi in compression.

If we look now at these three cases, all for the same material and set of dimensions (same bending moment) we can see how the thicknesses and neutral axis position changed as the value of S was changed.

CASE	S inches	TOP inches	BOT inches	SBH psi	SBCRIT psi	DITOP ft	DIBOT ft
1	10	0.468	0.166	30000	30000	22.06	33.05
2	30	0.4125	0.4125	20000	20000	27.78	27.78
3	120	0.03	1.14	9700	9700	37.43	18.13

In this table, we can more readily make conclusions on the influence of the value chosen for S on the neutral axis location, and also a quick look at the resulting values for the thicknesses may give an idea of the weight associated with each case.

In the following cases the dimension parameters are changed (increased) and consequently the bending moment and the thicknesses of plating will be higher.


```

024 C *AL* IS THE LENGHT
024 AL=1000.
020 B=AL/5.75
018 C *DRA* IS THE DRAFT
018 DRA=B/3.3
024 D=AL/14.
030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
030 FOR STEEL
030 POI=0.3
038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
038 FOR STEEL
038 E=30.*10.**6
040 C *DST* IS THE DESIGN STRESS
040 DST=18000.
054 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
054 S=38.
050 C *BM* IS THE BENDING MOMENT
050 BM=(B*DRA*AL*AL*0.75)/(35.*35.)
058 G=B*30396*S*S*(1.-POI*POI)/E
058 A=B*B*B*D/(147.*BM)
070 C=3.-B*D*D/(70.*BM)
070 F=3.*D*D*D/(147.*BM)
080 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
080 WRITE(5,10)
080 10 FORMAT('1',25X,'TOP ASSUMED LARGER THAN BOTTOM')
085 C *TOP* IS THE THICKNESS OF THE TOP PLATING
085 *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
085 BOT1=0.1
076 11 TOP1=B*BOT1*BOT1*BOT1/((B+D)*G*DST-D*BOT1*BOT1)
085 WRITE(5,20)TOP1,BOT1
085 20 FORMAT('1',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
085 BOT1=BOT1+0.25
085 IF(BOT1.GT.2.)GO TO 30
085 GO TO 11
085 30 TOP2=1.
085 31 BOT2=TOP2*(12.*D*(C+F)*DST*TOP2)/(DST*(A+C)*TOP2-12.*B)
085 WRITE(5,40)TOP2,BOT2
085 40 FORMAT('1',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
085 TOP2=TOP2+0.5
085 IF(TOP2.GT.15.)GO TO 50
085 GO TO 31
085 50 CONTINUE
085 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
085 WRITE(5,60)
085 60 FORMAT('1',25X,'BOTTOM ASSUMED LARGER THAN TOP')
085 BOT3=0.1
085 61 TOP3=((B+D)*BOT3*BOT3*BOT3-D*G*DST*BOT3)/(B*G*DST)
085 WRITE(5,70)TOP3,BOT3
085 70 FORMAT('1',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
085 BOT3=BOT3+0.25
085 IF(BOT3.GT.3.)GO TO 80
085 GO TO 61

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

```

02      80 TOP4=1.
04      81 BOT4=(12.*(R+D)-(A+C)*DST*TOP4)/((C+F)*DST)
06      WRITE(5,90)TOP4,BOT4
08      90 FORMAT(' ',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
10      TOP4=TOP4+.5
12      IF(TOP4.GT.15.)GO TO 100
14      GO TO 81
16      100 CONTINUE
18      END

```

(S) .U	0400(V) AL	0408(V) B	04D0(V) DRA	04D8(V) D
(V) POI	04F8(V) E	0000(S) .R	04F8(V) DST	0500(V) S
(V) BM	0518(V) G	0524(V) A	0530(V) C	0530(V) F
(L) 10	0200(S) @I	0540(V) BOT1	0176(L) 11	0548(V) TOP1
(L) 20	0220(L) 30	0550(V) TOP2	0234(L) 31	0560(V) BOT2
(L) 40	02FE(L) 50	0312(L) 60	0578(V) BOT3	0348(L) 61
(V) TOP3	0308(L) 70	0402(L) 80	0580(V) TOP4	040A(L) 81
(V) BOT4	0472(L) 90	048C(L) 100	0200(S) .V	

EQ L

GRAM LABELS:

07 .MAIN*	2C18 .V	2A48 .COMP	2C26 @I	25F8 .R	2B10 .ZE
06E \$6	2638 .A	2B6C .MES	29EA .W	27F2 EXP	2978 AIM
06EA ALOG	2AB0 .RARG	2B0C .5	27F2 AEXP	2B14 .ERCNT	2AE2 \$8
0B14 .0	2BEA .U	3D42			

BY-POINTS:

05F8 .R	2638 .A	26EA ALOG	27F2 AEXP	27F2 EXP	2978 AIM
09EA .W	2A48 .COMP	2A6E \$6	2AB0 .RARG	2AE2 \$8	2B0C .5
0B10 .ZERO	2B14 .ERCNT	2B16 .0	2B6C .MES	2BF6 .U	2C1C .V
0C26 @I					

MON-BLOCKS:

E

DEFINED SUBROUTINES:

E

TRANSFER ADDRESS 2070

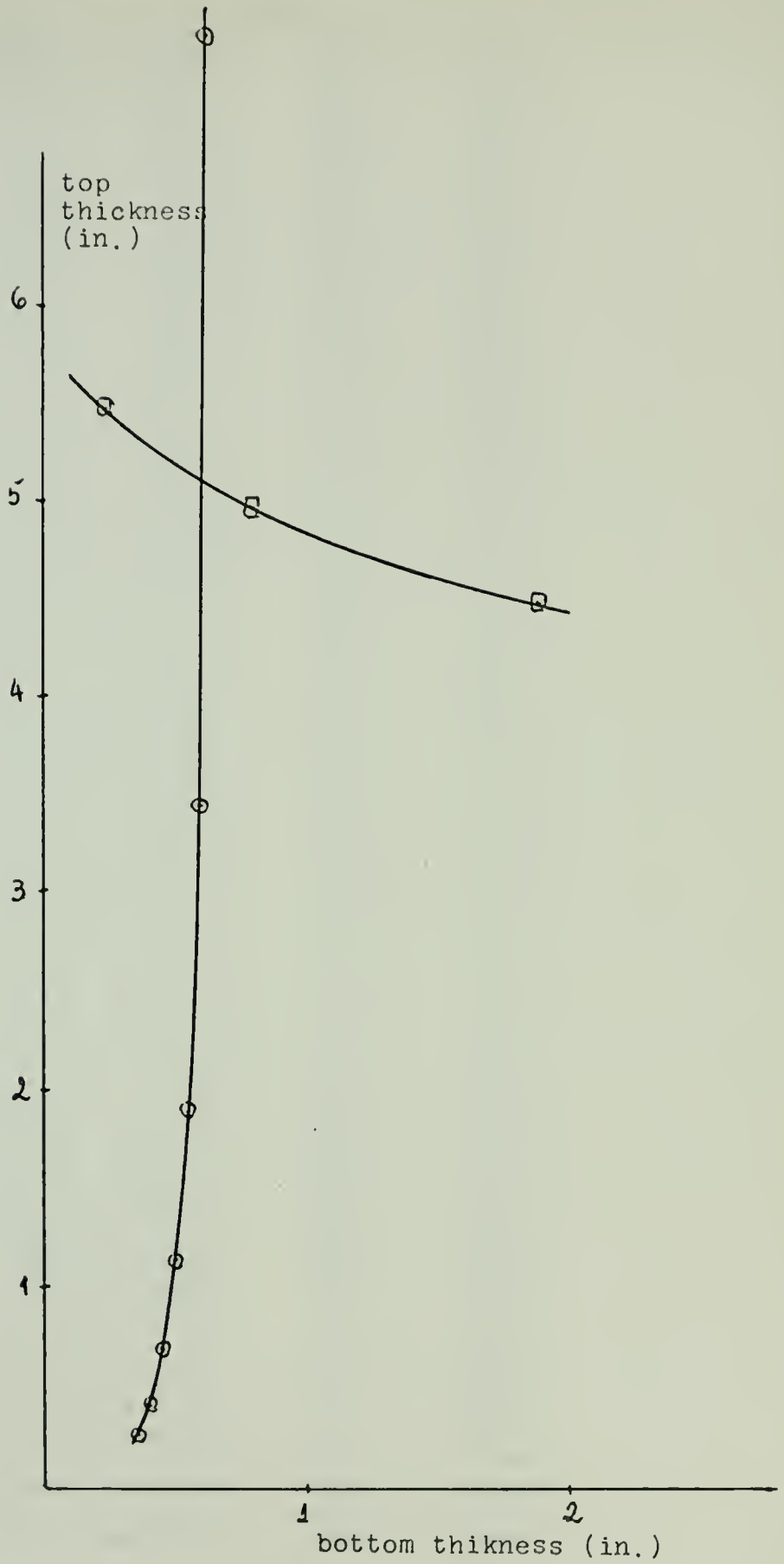
LOCATION BEGINS:

TOP ASSUMED LARGER THAN BOTTOM

TOP1=	2.0048	BOT1=	0.1000
TOP1=	0.0168	BOT1=	0.1500
TOP1=	0.0412	BOT1=	0.2000
TOP1=	0.0844	BOT1=	0.2500
TOP1=	0.1554	BOT1=	0.3000
TOP1=	0.2673	BOT1=	0.3500
TOP1=	0.4414	BOT1=	0.4000
TOP1=	0.7145	BOT1=	0.4500
TOP1=	1.1570	BOT1=	0.5000
TOP1=	1.9240	BOT1=	0.5500
TOP1=	3.4364	BOT1=	0.6000
TOP1=	7.3849	BOT1=	0.6500
TOP1=	36.2430	BOT1=	0.7000
TOP1=	-20.7688	BOT1=	0.7500
TOP1=	-9.8191	BOT1=	0.8000
TOP1=	-7.1387	BOT1=	0.8500
TOP1=	-5.9771	BOT1=	0.9000
TOP1=	-5.3601	BOT1=	0.9500
TOP1=	-5.0000	BOT1=	1.0000
TOP1=	-4.7816	BOT1=	1.0500
TOP1=	-4.6498	BOT1=	1.1000
TOP1=	-4.5747	BOT1=	1.1500
TOP1=	-4.5388	BOT1=	1.2000
TOP1=	-4.5313	BOT1=	1.2500
TOP1=	-4.5450	BOT1=	1.3000
TOP1=	-4.5748	BOT1=	1.3500
TOP1=	-4.6173	BOT1=	1.4000
TOP1=	-4.6700	BOT1=	1.4500
TOP1=	-4.7309	BOT1=	1.5000
TOP1=	-4.7986	BOT1=	1.5500
TOP1=	-4.8720	BOT1=	1.6000
TOP1=	-4.9502	BOT1=	1.6500
TOP1=	-5.0325	BOT1=	1.7000
TOP1=	-5.1183	BOT1=	1.7500
TOP1=	-5.2071	BOT1=	1.8000
TOP1=	-5.2986	BOT1=	1.8500
TOP1=	-5.3924	BOT1=	1.9000
TOP1=	-5.4883	BOT1=	1.9500
TOP1=	-5.5860	BOT1=	2.0000
TOP2=	1.0000	BOT2=	-0.4673
TOP2=	1.5000	BOT2=	-0.7727
TOP2=	2.0000	BOT2=	-1.1829
TOP2=	2.5000	BOT2=	-1.8315
TOP2=	3.0000	BOT2=	-3.2461
TOP2=	3.5000	BOT2=	-11.8971
TOP2=	4.0000	BOT2=	5.8810
TOP2=	4.5000	BOT2=	1.8850
TOP2=	5.0000	BOT2=	0.8127
TOP2=	5.5000	BOT2=	0.2578
TOP2=	6.0000	BOT2=	-0.1140
TOP2=	6.5000	BOT2=	-2.4000
TOP2=	7.0000	BOT2=	-0.6389
TOP2=	7.5000	BOT2=	-0.8492
TOP2=	8.0000	BOT2=	-1.0408
TOP2=	8.5000	BOT2=	-1.2195



TOP2=	9.0000	BOT2=	-1.3890
TOP2=	9.5000	BOT2=	-1.5516
TOP2=	10.0000	BOT2=	-1.7290
TOP2=	10.5000	BOT2=	-1.9223
TOP2=	11.0000	BOT2=	-2.1324
TOP2=	11.5000	BOT2=	-2.3599
TOP2=	12.0000	BOT2=	-2.6052
TOP2=	12.5000	BOT2=	-2.8687
TOP2=	13.0000	BOT2=	-3.1507
TOP2=	13.5000	BOT2=	-3.4514
TOP2=	14.0000	BOT2=	-3.7711
TOP2=	14.5000	BOT2=	-4.1098
TOP2=	15.0000	BOT2=	-4.4677



BOTTOM	ASSUMED	LARGER THAN	TOP
TOP3=	0.0316	BOT3=	0.1000
TOP3=	0.0297	BOT3=	0.1500
TOP3=	0.0066	BOT3=	0.2000
TOP3=	0.449	BOT3=	0.2500
TOP3=	0.1318	BOT3=	0.3000
TOP3=	0.2612	BOT3=	0.3500
TOP3=	0.4442	BOT3=	0.4000
TOP3=	0.6758	BOT3=	0.4500
TOP3=	0.9752	BOT3=	0.5000
TOP3=	1.3455	BOT3=	0.5500
TOP3=	1.7926	BOT3=	0.6000
TOP3=	2.3262	BOT3=	0.6500
TOP3=	2.9524	BOT3=	0.7000
TOP3=	3.6764	BOT3=	0.7500
TOP3=	4.5071	BOT3=	0.8000
TOP3=	5.4511	BOT3=	0.8500
TOP3=	6.5155	BOT3=	0.9000
TOP3=	7.7074	BOT3=	0.9500
TOP3=	9.0339	BOT3=	1.0000
TOP3=	10.5021	BOT3=	1.0500
TOP3=	12.1130	BOT3=	1.1000
TOP3=	13.8917	BOT3=	1.1500
TOP3=	15.8273	BOT3=	1.2000
TOP3=	17.9320	BOT3=	1.2500
TOP3=	20.2157	BOT3=	1.3000
TOP3=	22.6826	BOT3=	1.3500
TOP3=	25.3427	BOT3=	1.4000
TOP3=	28.1972	BOT3=	1.4500
TOP3=	31.2591	BOT3=	1.5000
TOP3=	34.5324	BOT3=	1.5500
TOP3=	38.0274	BOT3=	1.6000
TOP3=	41.7430	BOT3=	1.6500
TOP3=	45.7024	BOT3=	1.7000
TOP3=	49.8976	BOT3=	1.7500
TOP3=	54.3477	BOT3=	1.8000
TOP3=	59.0388	BOT3=	1.8500
TOP3=	63.9990	BOT3=	1.9000
TOP3=	69.2283	BOT3=	1.9500
TOP3=	74.7339	BOT3=	2.0000
TOP3=	80.5228	BOT3=	2.0500
TOP3=	86.6022	BOT3=	2.1000
TOP3=	92.9790	BOT3=	2.1500
TOP3=	99.6674	BOT3=	2.2000
TOP3=	106.6534	BOT3=	2.2500
TOP3=	113.9652	BOT3=	2.3000
TOP3=	121.6029	BOT3=	2.3500
TOP3=	129.5735	BOT3=	2.4000
TOP3=	137.9842	BOT3=	2.4500
TOP3=	146.5416	BOT3=	2.5000
TOP3=	155.5534	BOT3=	2.5500
TOP3=	164.9265	BOT3=	2.6000
TOP3=	174.6679	BOT3=	2.6500
TOP3=	184.7845	BOT3=	2.7000
TOP3=	195.2838	BOT3=	2.7500
TOP3=	206.1726	BOT3=	2.8000

TOP3=	217.4581	BOT3=	2.8500
TOP3=	229.1475	BOT3=	2.9200
TOP3=	241.2473	BOT3=	2.9500
TOP3=	253.7654	BOT3=	3.0200
TOP4=	1.0000	BOT4=	16.1639
TOP4=	1.5000	BOT4=	14.2342
TOP4=	2.0000	BOT4=	12.3244
TOP4=	2.5000	BOT4=	10.3746
TOP4=	3.0000	BOT4=	8.4448
TOP4=	3.5000	BOT4=	6.5150
TOP4=	4.0000	BOT4=	4.5852
TOP4=	4.5000	BOT4=	2.6554
TOP4=	5.0000	BOT4=	0.7256
TOP4=	5.5000	BOT4=	-1.2042
TOP4=	6.0000	BOT4=	-3.1340
TOP4=	6.5000	BOT4=	-5.0638
TOP4=	7.0000	BOT4=	-6.9936
TOP4=	7.5000	BOT4=	-8.9234
TOP4=	8.0000	BOT4=	-10.8532
TOP4=	8.5000	BOT4=	-12.7829
TOP4=	9.0000	BOT4=	-14.7127
TOP4=	9.5000	BOT4=	-16.6425
TOP4=	10.0000	BOT4=	-18.5723
TOP4=	10.5000	BOT4=	-20.5021
TOP4=	11.0000	BOT4=	-22.4319
TOP4=	11.5000	BOT4=	-24.3617
TOP4=	12.0000	BOT4=	-26.2915
TOP4=	12.5000	BOT4=	-28.2213
TOP4=	13.0000	BOT4=	-30.1511
TOP4=	13.5000	BOT4=	-32.0808
TOP4=	14.0000	BOT4=	-34.0106
TOP4=	14.5000	BOT4=	-35.9404
TOP4=	15.0000	BOT4=	-37.8702

ND

VIC OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

08 HANDLING CHARGE	\$.35 / JOB	.35
53 LINES PRINTED PR2	\$ 1.25 / K LN	.32
67 CARDS READ	\$ 1.50 / K CD	.10
02 PLOTTER VECTORS	\$.25 / 1227	.00
17 MODEL 70 SECONDS	\$25.00 / HOUR	.12
02 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE	\$.89

DIR 490 14731 LOGGED OUT 25/06/74 17:26. \$ 9.62 LEFT AFTER 23 LOGINS.

STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 1

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0004 C *AL* IS THE LENGHT
0104 AL=1000.
020C B=AL/5.75
0318 C *DRA* IS THE DRAFT
0418 DRA=B/3.3
0524 D=AL/14.
0632 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0732 C FOR STEEL
0832 POI=0.3
0938 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
1038 C FOR STEEL
1138 E=30.*10.**6
124C C *DST* IS THE DESIGN STRESS
134C DST=18000.
1454 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
1554 S=30.
165C C *BM* IS THE BENDING MOMENT
175C BM=(B*DRA*AL*AL*0.75)/(35.*35.)
188A G=0.30396*S*S*(1.-POI*POI)/E
1984 C *TOP* IS THE THICKNESS OF THE TOP PLATING
2084 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
2184 TOP=5.12
228C BOT=0.62
23C4 A=560.*BM*(B*BOT+D*TOP)
24F0 DEN=D*(3.*B*B*TOP*BOT+2.*B*D*TOP*(TOP+BOT)+(D*D*TOP*TOP))
254C C *STH* IS THE STRESS ON TOP DUE TO HOGGING
264C STH=A/DEN
2758 WRITE(5,60)STH
2874 60 FORMAT(' ',20X,'STH='F10.3)
298E C=560.*BM*(B*TOP+D*TOP)
3084 C *SBH* IS THE STRESS ON BOT DUE TO HOGGING
3184 SBH=C/DEN
32C6 WRITE(5,90)SBH
33E2 90 FORMAT(' ',20X,'SBH='F10.3)
34FC C *SBCRIT* IS THE CRITICAL STRESS ON BOT
35FC SBCRIT=BOT*BOT/G
362C WRITE(5,110)SBCRIT
3728 110 FORMAT(' ',17X,'SBCRIT='F10.3)
3844 P=2.*D*TOP+B*(TOP+BOT)
3968 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
4068 DITOP=D*(B*BOT+D*TOP)/P
418C WRITE(5,40)DITOP
428A 40 FORMAT(' ',18X,'DITOP='F10.4)
43C4 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
44C4 DIBOT=D*(B*TOP+D*TOP)/P
45E8 WRITE(5,50)DIBOT
4624 50 FORMAT(' ',18X,'DIBOT='F10.4)
4720 END
48(S) *U 0324(V) AL 032C(V) B 0334(V) DRA 033C(V) D
49(V) POI 034C(V) E 0000(S) .R 035C(V) DST 0364(V) S
50(V) BM 037C(V) G 0388(V) TOP 0300(V) BOT 0398(V) A
49(V) DEN 038C(V) STH 0174(L) 60 0000(S) .I 03C0(V) C

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 2

4(V) SBH 01E2(L) 90 03C8(V) SBCRIT 0228(L) 110 03CC(V) P
 0(V) DITOP 02A8(L) 40 03D4(V) DIBOT 0304(L) 50 0000(S) .V
 VEO L
 OF

PROGRAM LABELS:

2070 *MAIN*	2A68 .V	2898 .COMP	2A76 01	2448 .R	2960 .Z
288E \$6	2488 .A	298C .MES	283A .W	2642 EXP	27C8 AI
253A ALOG	2900 .RARG	295C .5	2642 AEXP	2964 .ERCNT	2932 \$8
2964 .0	2A3A .U	3B92			

TRY-POINTS:

2448 .R	2488 .A	253A ALOG	2642 AEXP	2642 EXP	27C8 AI
283A .W	2898 .COMP	288E \$6	2900 .RARG	2932 \$8	295C .5
2960 .ZERO	2964 .ERCNT	2966 .0	298C .MES	2A46 .U	2A6C .V
2A76 01					

COMMON-BLOCKS:

NE

DEFINED SUBROUTINES:

NE

TRANSFER ADDRESS 2070

CUTION BEGINS:

STH= 18085.023
 SBH= 47973.711
 SBCRIT= 46323.625
 DITOP= 19.5551
 DIBOT= 51.8734

END

F VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

JOB HANDLING CHARGE	\$.35 / JOB	.35
85 LINES PRINTED PR1	\$ 1.25 / K LN	.11
54 CARDS READ	\$ 1.50 / K CD	.08
00 PLOTTER VECTORS	\$.25 / 1020	.00
13 MODEL 70 SECONDS	\$25.00 / HOUR	.09
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE	\$.63

REIR 490 14731 LOGGED OUT 25/06/74 17:36. \$ 8.11 LEFT AFTER 25 LOGINS.

SUMMARY

Length	= 1000 ft
Beam	= AL/5.75
Draft	= B/3.3
Depth	= AL/14
DST	= 18000 psi
S	= 30"
Material	- STEEL
TOP	= 5.12"
BOT	= 0.62"
STH	= 18085.02
SBH	= 47973.71
SBCRIT	= 46323.82
DITOP	= 19.55'
DIBOT	= 51.87'

We might say here that the value chosen for S is small and this accounts for the compression at the bottom to reach such a high value.

In the next case, the value chosen for S is increased to 60 inches.


```

0704 C *AL* IS THE LENGHT
0704 AL=1000.
0704 B=AL/5.75
0718 C *DRA* IS THE DRAFT
0718 DRA=B/3.3
0724 D=AL/14.
0730 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0730 FOR STEEL
0730 POI=0.3
0738 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0738 FOR STEEL
0738 E=30.*10.**6
0740 C *DST* IS THE DESIGN STRESS
0740 DST=18000.
0754 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0754 S=60.
0750 C *BM* IS THE BENDING MOMENT
0750 BM=(B*DRA*AL*AL*0.75)/(35.*35.)
0758 G=0.30396*S*S*(1.-POI*POI)/E
0764 A=9.*B*B*D/(140.*BM)
0764 C=3.*B*D*D/(70.*BM)
0764 F=3.*D*D*D/(140.*BM)
0764 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
0764 WRITE(5,10)
0764 10 FORMAT('1',25X,'TOP ASSUMED LARGER THAN BOTTOM')
0764 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0764 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0764 BOT1=0.02
0764 11 TOP1=B*BOT1*BOT1*BOT1/((B+D)*G*DST-D*BOT1*POI1)
0764 WRITE(5,20)TOP1,BOT1
0764 20 FORMAT(' ',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
0764 BOT1=BOT1+0.02
0764 IF(BOT1.GT.2.)GO TO 30
0764 GO TO 11
0764 30 TOP2=1.
0764 31 BOT2=TOP2*(12.*D*(C+F)*DST*TOP2)/(DST*(A+C)*TOP2-12.*B)
0764 WRITE(5,40)TOP2,BOT2
0764 40 FORMAT(' ',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
0764 TOP2=TOP2+0.2
0764 IF(TOP2.GT.6. ) GO TO 50
0764 GO TO 31
0764 50 CONTINUE
0764 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
0764 WRITE(5,60)
0764 60 FORMAT('1',25X,'BOTTOM ASSUMED LARGER THAN TOP')
0764 BOT3=0.4
0764 61 TOP3=((B+D)*BOT3*BOT3*BOT3-D*G*DST+BOT3)/(B*G*DST)
0764 WRITE(5,70)TOP3,BOT3
0764 70 FORMAT(' ',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
0764 BOT3=BOT3+0.02
0764 IF(BOT3.GT..85)GO TO 80
0764 80 TOP4=1.

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 2

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0406      81 BOT4=(12.*(B+D)-(A+C)*DST*TOP4)/((C+F)*DST)
0444      WRITE(5,90)TOP4,BOT4
046E      90 FORMAT(' ',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
0496      TOP4=TOP4*0.2
04A2      IF(TOP4.GT.5.5)GO TO 100
04B4      GO TO 81
04B8      100 CONTINUE
04B8      END
04(S)  .U      048C(V) AL      04C4(V) B      04CC(V) DRA      04D4(V) D
04(V)  POI     04E4(V) E      0000(S) .R      04F4(V) DST      04FC(V) S
04(V)  BM      0518(V) G      0524(V) A      0530(V) C      053C(V) F
04(L)  10      0000(S) @I     0540(V) BOT1    0176(L) 11      0548(V) TOP1
04(L)  20      022C(L) 30      0558(V) TOP2    0234(L) 31      055C(V) BOT2
04(L)  40      02FE(L) 50      0312(L) 60      0574(V) BOT3    0348(L) 61
04(V)  TOP3    0388(L) 70      03FE(L) 80      0584(V) TOP4    0406(L) 81
04(V)  BOT4    046E(L) 90      0488(L) 100     0000(S) .V
04E0      L
04F

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PROGRAM LABELS:

2070 *MAIN*	2020 .V	2A50 .COMP	2C2E @I	2600 .R	2B18 .7
2A76 \$6	2640 .A	2B74 .MES	29F2 .W	27FA EXP	2980 AI
26F2 ALOG	2AB8 .RARG	2B14 .5	27FA AEXP	2B1C .ERCNT	2AEA \$8
2B1C .0	2BF2 .U	3D4A			

TRY-POINTS:

2600 .R	2640 .A	26F2 ALOG	27FA AEXP	27FA EXP	2980 AI
29F2 .W	2A50 .COMP	2A76 \$6	2AB8 .RARG	2AEA \$8	2B14 .5
2B18 .ZEPO	2B1C .ERCNT	2B1E .0	2B74 .MES	2BF2 .U	2C24 .V
2C2E @I					

COMMON-BLOCKS:

NONE

DEFINED SUBROUTINES:

NONE

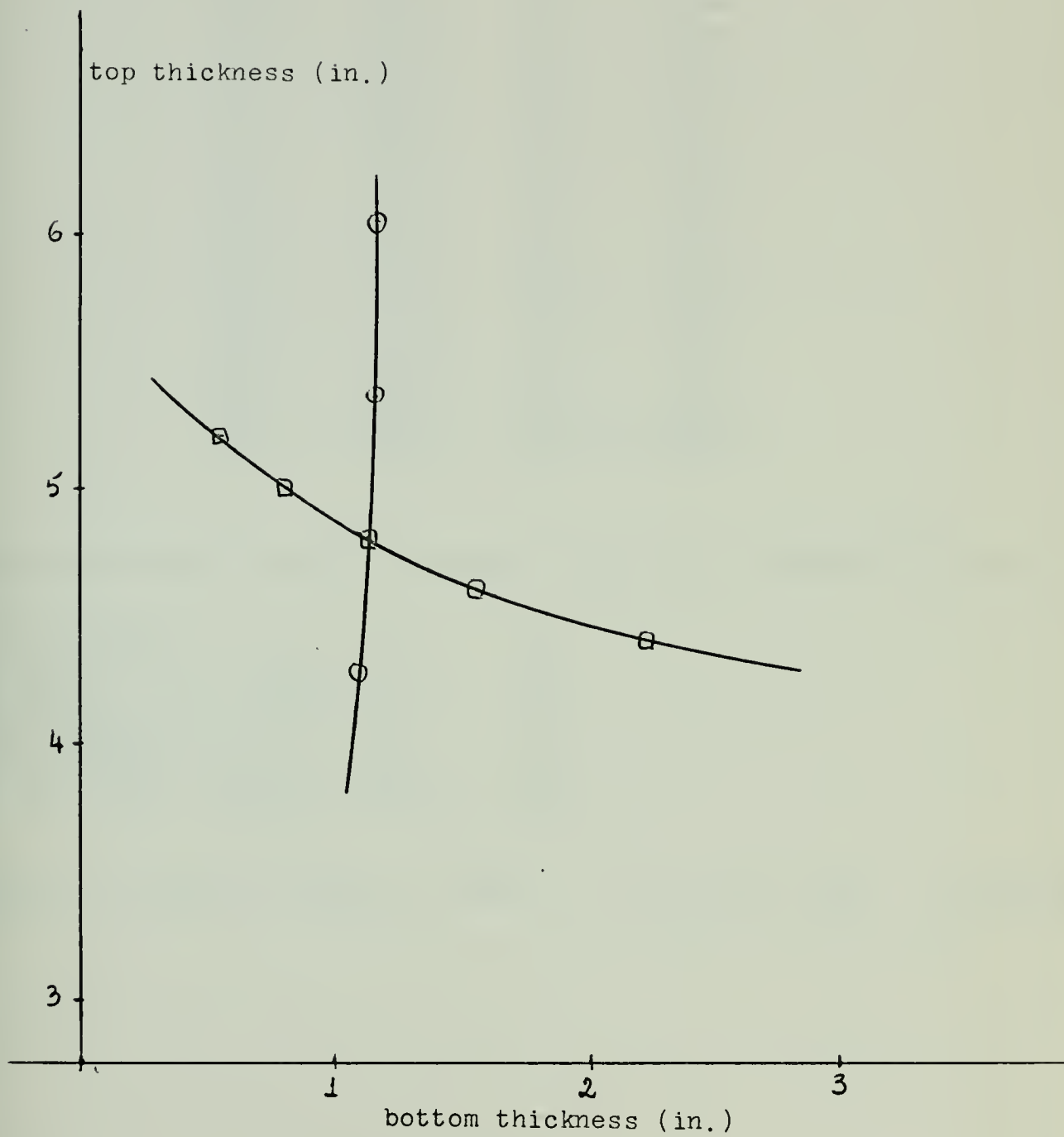
TRANSFER ADDRESS 2070

CUTION BEGINS:

TOP ASSUMED LARGER THAN BOTTOM			
TOP1=	0.0000	BOT1=	0.0200
TOP1=	0.0001	BOT1=	0.0400
TOP1=	0.0003	BOT1=	0.0600
TOP1=	0.0006	BOT1=	0.0800
TOP1=	0.0012	BOT1=	0.1000
TOP1=	0.0021	BOT1=	0.1200
TOP1=	0.0033	BOT1=	0.1400
TOP1=	0.0049	BOT1=	0.1600
TOP1=	0.0070	BOT1=	0.1800
TOP1=	0.0097	BOT1=	0.2000
TOP1=	0.0129	BOT1=	0.2200
TOP1=	0.0169	BOT1=	0.2400
TOP1=	0.0216	BOT1=	0.2600
TOP1=	0.0271	BOT1=	0.2800
TOP1=	0.0325	BOT1=	0.3000
TOP1=	0.0409	BOT1=	0.3200
TOP1=	0.0494	BOT1=	0.3400
TOP1=	0.0591	BOT1=	0.3600
TOP1=	0.0700	BOT1=	0.3800
TOP1=	0.0824	BOT1=	0.4000
TOP1=	0.0962	BOT1=	0.4200
TOP1=	0.1116	BOT1=	0.4400
TOP1=	0.1288	BOT1=	0.4600
TOP1=	0.1478	BOT1=	0.4800
TOP1=	0.1689	BOT1=	0.5000
TOP1=	0.1921	BOT1=	0.5200
TOP1=	0.2178	BOT1=	0.5400
TOP1=	0.2459	BOT1=	0.5600
TOP1=	0.2769	BOT1=	0.5800
TOP1=	0.3108	BOT1=	0.6000
TOP1=	0.3479	BOT1=	0.6200
TOP1=	0.3886	BOT1=	0.6400
TOP1=	0.4330	BOT1=	0.6600
TOP1=	0.4816	BOT1=	0.6800
TOP1=	0.5346	BOT1=	0.7000
TOP1=	0.5925	BOT1=	0.7200
TOP1=	0.6558	BOT1=	0.7400
TOP1=	0.7248	BOT1=	0.7600
TOP1=	0.8003	BOT1=	0.7800
TOP1=	0.8828	BOT1=	0.8000
TOP1=	0.9730	BOT1=	0.8200
TOP1=	1.0717	BOT1=	0.8400
TOP1=	1.1799	BOT1=	0.8600
TOP1=	1.2985	BOT1=	0.8800
TOP1=	1.4289	BOT1=	0.9000
TOP1=	1.5724	BOT1=	0.9200
TOP1=	1.7306	BOT1=	0.9400
TOP1=	1.9054	BOT1=	0.9600
TOP1=	2.0920	BOT1=	0.9800
TOP1=	2.3141	BOT1=	1.0000
TOP1=	2.5537	BOT1=	1.0200
TOP1=	2.8218	BOT1=	1.0400
TOP1=	3.1229	BOT1=	1.0600
TOP1=	3.4626	BOT1=	1.0800
TOP1=	3.8480	BOT1=	1.1000

TOP1=	4.2878	BOT1=	1.1200
TOP1=	4.7931	BOT1=	1.1400
TOP1=	5.3786	BOT1=	1.1600
TOP1=	6.0632	BOT1=	1.1800
TOP1=	6.8726	BOT1=	1.2000
TOP1=	7.8419	BOT1=	1.2200
TOP1=	9.0212	BOT1=	1.2400
TOP1=	10.4833	BOT1=	1.2600
TOP1=	12.3398	BOT1=	1.2800
TOP1=	14.7689	BOT1=	1.3000
TOP1=	18.0762	BOT1=	1.3200
TOP1=	22.8310	BOT1=	1.3400
TOP1=	30.2304	BOT1=	1.3600
TOP1=	43.2943	BOT1=	1.3800
TOP1=	72.4629	BOT1=	1.4000
TOP1=	114.7422	BOT1=	1.4200
TOP1=	-339.7163	BOT1=	1.4400
TOP1=	-95.4322	BOT1=	1.4600
TOP1=	-57.1123	BOT1=	1.4800
TOP1=	-41.5440	BOT1=	1.5000
TOP1=	-33.1158	BOT1=	1.5200
TOP1=	-27.8412	BOT1=	1.5400
TOP1=	-24.2356	BOT1=	1.5600
TOP1=	-21.6199	BOT1=	1.5800
TOP1=	-19.6394	BOT1=	1.6000
TOP1=	-18.0908	BOT1=	1.6200
TOP1=	-16.8492	BOT1=	1.6400
TOP1=	-15.8337	BOT1=	1.6600
TOP1=	-14.9894	BOT1=	1.6800
TOP1=	-14.2780	BOT1=	1.7000
TOP1=	-13.6717	BOT1=	1.7200
TOP1=	-13.1500	BOT1=	1.7400
TOP1=	-12.6974	BOT1=	1.7600
TOP1=	-12.3021	BOT1=	1.7800
TOP1=	-11.9546	BOT1=	1.8000
TOP1=	-11.6475	BOT1=	1.8200
TOP1=	-11.3750	BOT1=	1.8400
TOP1=	-11.1322	BOT1=	1.8600
TOP1=	-10.9151	BOT1=	1.8800
TOP1=	-10.7223	BOT1=	1.9000
TOP1=	-10.5453	BOT1=	1.9200
TOP1=	-10.3875	BOT1=	1.9400
TOP1=	-10.2452	BOT1=	1.9600
TOP1=	-10.1165	BOT1=	1.9800
TOP1=	-10.0001	BOT1=	2.0000
TOP2=	1.0000	BOT2=	-0.4673
TOP2=	1.2400	BOT2=	-0.5810
TOP2=	1.4000	BOT2=	-0.7055
TOP2=	1.6000	BOT2=	-0.8440
TOP2=	1.8000	BOT2=	-1.0009
TOP2=	2.0000	BOT2=	-1.1829
TOP2=	2.2000	BOT2=	-1.4002
TOP2=	2.4000	BOT2=	-1.6693
TOP2=	2.6000	BOT2=	-2.0190
TOP2=	2.8000	BOT2=	-2.5045
TOP2=	3.0000	BOT2=	-3.2460

TOP2=	3.2000	BOT2=	-4.5652
TOP2=	3.4000	BOT2=	-7.7108
TOP2=	3.6000	BOT2=	-26.8087
TOP2=	3.8000	BOT2=	16.3226
TOP2=	4.0000	BOT2=	5.8811
TOP2=	4.2000	BOT2=	3.3945
TOP2=	4.4000	BOT2=	2.2560
TOP2=	4.6000	BOT2=	1.5888
TOP2=	4.8000	BOT2=	1.1408
TOP2=	5.0000	BOT2=	0.8127
TOP2=	5.2000	BOT2=	0.5572
TOP2=	5.4000	BOT2=	0.3490
TOP2=	5.6000	BOT2=	0.1733
TOP2=	5.8000	BOT2=	0.0210
TOP2=	6.0000	BOT2=	-0.1140



BOTTOM	ASSUMED	LARGER THAN	TOP
TOP3=	-0.0132	BOT3=	0.4000
TOP4=	1.2000	BOT4=	16.1639
TOP4=	1.2000	BOT4=	15.3920
TOP4=	1.4000	BOT4=	14.6201
TOP4=	1.6000	BOT4=	13.8482
TOP4=	1.8000	BOT4=	13.0763
TOP4=	2.0000	BOT4=	12.3044
TOP4=	2.2000	BOT4=	11.5324
TOP4=	2.4000	BOT4=	10.7605
TOP4=	2.6000	BOT4=	9.9886
TOP4=	2.8000	BOT4=	9.2167
TOP4=	3.0000	BOT4=	8.4448
TOP4=	3.2000	BOT4=	7.6729
TOP4=	3.4000	BOT4=	6.9010
TOP4=	3.6000	BOT4=	6.1290
TOP4=	3.8000	BOT4=	5.3571
TOP4=	4.0000	BOT4=	4.5852
TOP4=	4.2000	BOT4=	3.8133
TOP4=	4.4000	BOT4=	3.0414
TOP4=	4.6000	BOT4=	2.2695
TOP4=	4.8000	BOT4=	1.4975
TOP4=	5.0000	BOT4=	0.7256
TOP4=	5.2000	BOT4=	-0.0463
TOP4=	5.4000	BOT4=	-0.8182

END

VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

JOB HANDLING CHARGE	\$ 0.35 / JOB	0.35
46 LINES PRINTED PR1	\$ 1.25 / K LN	0.31
66 CARDS READ	\$ 1.50 / K CD	0.10
02 PLOTTER VECTORS	\$ 0.25 / 1000	0.00
17 MODEL 72 SECONDS	\$25.00 / HOUR	0.12
00 MODEL 80 SECONDS	\$12.50 / HOUR	0.00
TOTAL CHARGE	\$	0.88

PEIR 490 14731 LOGGED OUT 05/06/74 17:28. \$ 8.74 LEFT AFTER 24 LOGINS.


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04 C *AL* IS THE LENGTH
04 AL=1000.
20 C B=AL/5.75
18 C *DRA* IS THE DRAFT
18 DRA=B/3.3
24 D=AL/14.
30 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
30 FOR STEEL
32 POI=0.3
38 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
38 FOR STEEL
38 E=30.*10.**6
40 C *DST* IS THE DESIGN STRESS
40 DST=18000.
54 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
54 S=60.
50 C *BM* IS THE BENDING MOMENT
50 BM=(B*DRA*AL*AL*0.75)/(35.*35.)
82 G=0.30396*S*S*(1.-POI+POI)/E
84 C *TOP* IS THE THICKNESS OF THE TOP PLATING
84 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
84 TOP=4.79
80 BOT=1.15
84 A=560.*BM*(B*BOT+D*TOP)
84 DEN=D*(3.*B*TOP*BOT+2.*B*D*TOP*(TOP+BOT)+(D*D*TOP*TOP))
40 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
40 STH=A/DEN
58 WRITE(5,60)STH
74 60 FORMAT(' ',20X,'STH='F10.3)
8E C=560.*BM*(B*TOP+D*TOP)
8A C *SBH* IS THE STRESS ON BOT DUE TO HOGGING
8A SBH=C/DEN
86 WRITE(5,90)SBH
82 90 FORMAT(' ',20X,'SBH='F10.3)
8C C *SBCRIT* IS THE CRITICAL STRESS ON BOT
8C SBCRIT=BOT*BOT/G
8C WRITE(5,110)SBCRIT
82 110 FORMAT(' ',17X,'SBCRIT='F10.3)
84 P=2.*D*TOP+B*(TOP+BOT)
62 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
62 DITOP=D*(B*BOT+D*TOP)/P
80 WRITE(5,40)DITOP
8A 40 FORMAT(' ',18X,'DITOP='F10.4)
8C C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
8C DIBOT=D*(B*TOP+D*TOP)/P
8E WRITE(5,50)DIBOT
82 50 FORMAT(' ',18X,'DIBOT='F10.4)
82 END
(S) *U 0324(V) AL 032C(V) B 0334(V) DRA 033C(V) D
(V) POI 034C(V) E 0000(S) .R 035C(V) DST 0364(V) S
C(V) BM 0382(V) G 038C(V) TOP 0394(V) BOT 039C(V) A
C(V) DEN 03C0(V) STH 0174(L) 60 0000(S) .I 03C4(V) C

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 2

02(V) SRH 01E2(L) 90 03CC(V) SBCRIT 0228(L) 110 03D0(V) P
 04(V) DITOP 02A8(L) 40 03D8(V) DIBOT 0304(L) 50 0000(S) .V
 XED L
 OF

PROGRAM LABELS:

2070 *MAIN*	2A6C .V	289C .COMP	2A7A 0I	244C .R	2964 .Z
28C2 \$6	248C .A	29C0 .MES	283E .W	2646 EXP	27CC AI
253E ALOG	2904 .RARG	2960 .5	2646 ΔEXP	2968 .ERCNT	2936 \$8
2968 .0	2A3E .U	3896			

ENTRY-POINTS:

244C .R	248C .A	253E ALOG	2A46 ΔEXP	2646 EXP	27CC AI
283E .W	289C .COMP	28C2 \$6	2904 .RARG	2936 \$8	2960 .5
2964 .ZERO	2968 .ERCNT	296A .0	29C0 .MES	2A4A .U	2A70 .V
2A7A 0I					

COMMON-BLOCKS:

VE

DEFINED SUBROUTINES:

VE

TRANSFER ADDRESS 2070

CUTION BEGINS:

STH= 18017.242
 SBH= 39055.434
 SBCRIT= 39843.250
 DITOP= 22.5492
 DIBOT= 48.8793

END

VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

JOB HANDLING CHARGE	\$.35 / JOB	.35
85 LINES PRINTED PRI	\$ 1.25 / K LN	.11
54 CARDS READ	\$ 1.50 / K CD	.08
20 PLOTTER VECTORS	\$.25 / 1000	.00
13 MODEL 70 SECONDS	\$25.00 / HOUR	.09
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE	\$.63

FEIR 490 14731 LOGGED OUT 05/06/74 17:37. \$ 7.48 LEFT AFTER 26 LOGINS.

SUMMARY

Length	=	1000'
Beam	=	AL/5.75
Draft	=	B/3.3
Depth	=	AL/14
DST	=	18000 psi
S	=	60"
Material	-	STEEL
TOP	=	4.79"
BOT	=	1.15"
STH	=	18017.24
SBH	=	39055.43
SBCRIT	=	39843.44
DITOP	=	22.55'
DIBOT	=	48.88'

Since sometimes it is desirable to use aluminum alloys in the structure of ships, due to their light weight, high corrosion resistance and non-magnetic characteristics, in the following runs, the material considered is aluminum. The necessary change is made in the input to account for the properties of this material.

When considering aluminum as the structural material, we must keep in mind that the aluminum hull will be more flexible than a corresponding steel ship. The modulus of elasticity of aluminum is about one third that of steel, and if scantlings were identical, the deflection of an aluminum ship, under a given load, would be three times those of a steel ship.

However, to obtain a factor of safety on ultimate strength equal to that in steel, the area of an aluminum member in direct stress, or the section modulus of an aluminum member in bending would be about 1.5 times that of the corresponding steel member (based on an ultimate strength of about 60000 psi in medium steel and of about 40000 psi across the welds of a welded aluminum structure).

A section modulus of 1.5 times that in steel, with the same depth being considered in both cases, would result in a deflection in aluminum about twice that in steel. However, the lower modulus of elasticity of aluminum reduces its ability to resist buckling, and for the same factor of safety both the section modulus and the moment of inertia in aluminum would in

general be somewhat more than 1.5 times those in steel, so that the deflection of the aluminum structure would in general be somewhat less than twice that of the steel structure.

The greater deflection would reduce the natural frequency of hull vibration, and would affect the interaction of slamming stress and bending moment stress. It could also affect shaft bearing pressures. It has been proposed as an arbitrary basis of design that aluminum ships be designed for a deflection about 1.5 times that of steel ships. This would require a moment of inertia of twice the value of that in the steel hull girder, and would automatically result in a section modulus substantially more than 1.5 times that in steel, and a factor of safety correspondingly greater than in steel.


```

0004 C *AL* IS THE LENGTH
0004 AL=1000.
0004 B=AL/5.75
0018 C *DRA* IS THE DRAFT
0018 DRA=B/3.3
0024 D=AL/14.
0030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0030 FOR ALUMINUM
0032 POI=0.33
0038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0038 FOR ALUMINUM
0038 E=10.**10.**6
0040 C *DST* IS THE DESIGN STRESS
0040 DST=18000.
0054 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0054 S=10.
0060 C *BM* IS THE BENDING MOMENT
0060 BM=(B*DRA*AL*AL*0.75)/(35.**35.)
0068 G=0.30296*S*S*(1.-POI/POI)/E
0074 A=9.**B**D/(140.**BM)
0080 C=3.**B*(D/(70.**BM)
0084 F=3.**D*(D/(140.**BM)
0120 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
0120 WRITE(5,10)
0140 10 FORMAT('1',25X,'TOP ASSUMED LARGER THAN BOTTOM')
0160 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0160 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0160 BOT1=0.2
0176 11 TOP1=B*BOT1*BM/((B+D)*G*DST-D*BOT1*BOT1)
0180 WRITE(5,20)TOP1,BOT1
0182 20 FORMAT('1',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
0194 BOT1=BOT1+0.21
0200 IF(BOT1.GT.2.4)GO TO 30
0208 GO TO 11
0220 30 TOP2=1.
0234 31 BOT2=TOP2*(12.*D-(C+F)*DST*TOP2)/(DST*(A+C)*TOP2-12.*B)
0240 WRITE(5,40)TOP2,BOT2
0248 40 FORMAT('1',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
0254 TOP2=TOP2+0.2
0260 IF(TOP2.GT.5. ) GO TO 50
0268 GO TO 31
0276 50 CONTINUE
0280 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
0280 WRITE(5,60)
0292 60 FORMAT('1',25X,'BOTTOM ASSUMED LARGER THAN TOP')
0300 BOT3=0.1
0308 61 TOP3=((B+D)*BOT3*BM/((B+D)*G*DST+BOT3)/DST)
0314 WRITE(5,70)TOP3,BOT3
0320 70 FORMAT('1',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
0328 BOT3=BOT3+0.21
0336 IF(BOT3.GT..24)GO TO 80
0344 GO TO 61

```


STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

```

2402      20 TOP4=1.
2404      31 BOT4=(12.*(R+S)-(A+C)+DST*TOP4)/((C+F)*DST)
2406      WRITE(5,90)TOP4,BOT4
2407      30 FORMAT(' ',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
2408      TOP4=TOP4+0.2
2409      IF(TOP4.GT.5.5)GO TO 100
2410      GO TO 81
2411      10 CONTINUE
2412      END
2413      (S)  .U      2400[V] AL      0400[V] B      0400[V] DRA      0408[V] D
2414      E(V)  20I    2408[V] E      0400[V] .R      0404[V] DST      04FC[V] S
2415      (V)  30      2514[V] G      0520[V] A      052C[V] C      0538[V] F
2416      (L)  12      2420[S]  0I      053C[V] BOT1    0176[L] 11      0544[V] TOP1
2417      (L)  27      222C[L] 30      2550[V] TOP2    0234[L] 31      055C[V] BOT2
2418      (L)  42      22FE[L] 50      2312[L] 60      0570[V] BOT3    0348[L] 61
2419      (V)  TOP3    2335[L] 70      2400[L] 80      0580[V] TOP4    040A[L] 81
2420      (V)  BOT4    2472[L] 90      04BC[L] 100     0000[S] .V
2421      EC      L
    
```

GRAPH LABELS:

2470 .A.I	2C1C .V	2A4C .COMP	2C2A .I	25FC .R	2B14 .Z
2A72 .S	263C .A	2B70 .MES	29EF .W	27F6 EXP	297C AI
26FE ALUG	2AB4 .RARG	2B10 .5	27F6 AEXP	2B18 .ERCNT	2AE6 \$8
2B18 .C	2BEE .U	3D46			

ARRAY-POINTS:

25FC .A	263C .A	26FE ALUG	27F6 AEXP	27F6 EXP	297C AI
2BEE .V	2A4C .COMP	2A72 \$6	2AB4 .RARG	2AE6 \$8	2B10 .5
2B14 .7FF0	2B18 .ERCNT	2B1A .0	2B70 .MES	2BFA .U	2C20 .V
2C2A .T					

COMMON-BLOCKS:

0 E

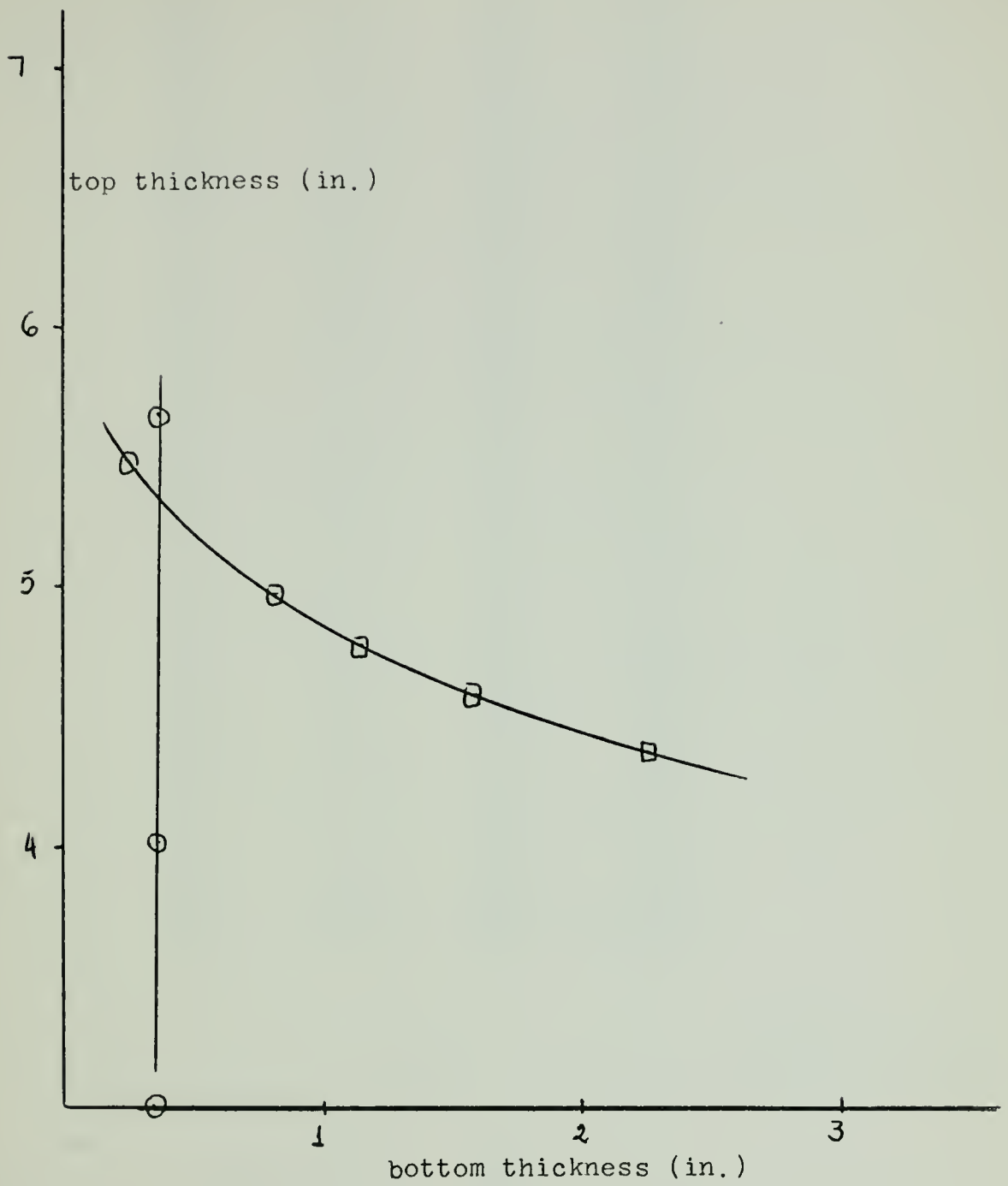
REFINE-SUBROUTINES:

0 E

TRANSFER ADDRESS 2270

EXECUTION BEGINS:

TOP ASSUMED LARGER THAN BOTTOM			
TOP1=	0.1558	BOT1=	0.2000
TOP1=	0.1858	BOT1=	0.2100
TOP1=	0.2177	BOT1=	0.2200
TOP1=	0.2506	BOT1=	0.2300
TOP1=	0.2824	BOT1=	0.2400
TOP1=	0.3624	BOT1=	0.2500
TOP1=	0.4225	BOT1=	0.2600
TOP1=	0.5028	BOT1=	0.2700
TOP1=	0.6021	BOT1=	0.2800
TOP1=	0.7123	BOT1=	0.2900
TOP1=	0.8427	BOT1=	0.3000
TOP1=	1.0124	BOT1=	0.3100
TOP1=	1.2223	BOT1=	0.3200
TOP1=	1.4941	BOT1=	0.3300
TOP1=	1.8452	BOT1=	0.3400
TOP1=	2.3248	BOT1=	0.3500
TOP1=	3.0023	BOT1=	0.3600
TOP1=	4.0353	BOT1=	0.3700
TOP1=	5.7951	BOT1=	0.3800
TOP1=	9.4046	BOT1=	0.3900
TOP1=	20.8821	BOT1=	0.4000
TOP2=	1.0000	BOT2=	-0.4673
TOP2=	1.2000	BOT2=	-0.5810
TOP2=	1.4000	BOT2=	-0.7055
TOP2=	1.6000	BOT2=	-0.8440
TOP2=	1.8000	BOT2=	-1.0009
TOP2=	2.0000	BOT2=	-1.1829
TOP2=	2.2000	BOT2=	-1.4002
TOP2=	2.4000	BOT2=	-1.6693
TOP2=	2.6000	BOT2=	-2.0190
TOP2=	2.8000	BOT2=	-2.5045
TOP2=	3.0000	BOT2=	-3.2460
TOP2=	3.2000	BOT2=	-4.5652
TOP2=	3.4000	BOT2=	-7.7108
TOP2=	3.6000	BOT2=	-26.8087
TOP2=	3.8000	BOT2=	16.3226
TOP2=	4.0000	BOT2=	5.8811
TOP2=	4.2000	BOT2=	3.3945
TOP2=	4.4000	BOT2=	2.2560
TOP2=	4.6000	BOT2=	1.5888
TOP2=	4.8000	BOT2=	1.1408
TOP2=	5.0000	BOT2=	0.8127
TOP 2=	5.5000	BOT 2=	0.2578



BOTTOM	ASSUMED	LARGER THAN	TOP
TOP3=	-0.0121	BOT3=	0.1000
TOP3=	-0.0247	BOT3=	0.1100
TOP3=	0.0327	BOT3=	0.1200
TOP3=	0.0412	BOT3=	0.1300
TOP3=	0.0519	BOT3=	0.1400
TOP3=	0.0660	BOT3=	0.1500
TOP3=	0.0828	BOT3=	0.1600
TOP3=	0.0923	BOT3=	0.1700
TOP3=	0.0948	BOT3=	0.1800
TOP3=	0.1204	BOT3=	0.1900
TOP3=	0.1403	BOT3=	0.2000
TOP3=	0.1617	BOT3=	0.2100
TOP3=	0.2177	BOT3=	0.2200
TOP3=	0.2576	BOT3=	0.2300
TOP3=	0.3014	BOT3=	0.2400
TOP4=	1.0100	BOT4=	16.1639
TOP4=	1.2000	BOT4=	15.3920
TOP4=	1.4100	BOT4=	14.6201
TOP4=	1.6000	BOT4=	13.8482
TOP4=	1.8000	BOT4=	13.0763
TOP4=	2.0000	BOT4=	12.3044
TOP4=	2.2000	BOT4=	11.5324
TOP4=	2.4000	BOT4=	10.7605
TOP4=	2.6000	BOT4=	9.9886
TOP4=	2.8000	BOT4=	9.2167
TOP4=	3.0000	BOT4=	8.4448
TOP4=	3.2000	BOT4=	7.6729
TOP4=	3.4000	BOT4=	6.9010
TOP4=	3.6000	BOT4=	6.1290
TOP4=	3.8000	BOT4=	5.3571
TOP4=	4.0000	BOT4=	4.5852
TOP4=	4.2000	BOT4=	3.8133
TOP4=	4.4000	BOT4=	3.0414
TOP4=	4.6000	BOT4=	2.2695
TOP4=	4.8000	BOT4=	1.4975
TOP4=	5.0000	BOT4=	0.7256
TOP4=	5.2000	BOT4=	-0.0463
TOP4=	5.4000	BOT4=	-0.8182

END

F VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

JOB HANDLING CHARGE	\$ 0.35 / JOB	0.35
177 LINES PRINTED PR1	\$ 1.25 / K LN	0.22
67 CARDS READ	\$ 1.50 / K Ch	0.10
20 PLOTTER VECTORS	\$ 0.25 / 1000	0.00
22 MODEL 70 SECONDS	\$25.00 / HOUR	0.14
22 MODEL 80 SECONDS	\$12.50 / HOUR	0.00
TOTAL CHARGE \$		0.81

FEIR 403 14731 LOGGED OUT 05/26/74 19:37. \$ 6.67 LEFT AFTER 27 LOGINS.


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0204 C *AL* IS THE LENGHT
0204 AL=1001.
0204 *L=AL/5.75
0218 C *DRA* IS THE DRAFT
0218 DRA=3/3.3
0224 D=AL/14.
0230 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0230 FOR ALUMINUM
0230 POI=.33
0234 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0234 FOR ALUMINUM
0234 E=10.**10.**6
0240 C *DST* IS THE DESIGN STRESS
0240 DST=18000.
0254 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0254 S=10.
0260 C *BM* IS THE BENDING MOMENT
0260 BM=(B*DRA*AL**2*L**0.75)/(35.**35.)
0260 G=.30394*S*S*(1.-POI+POI)/E
0264 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0264 *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0264 TOP=5.35
0264 BOT=0.343
0264 A=560.*BM*(B*BOT+D*TOP)
0264 DEN=D*(3.*B*B*TOP*BOT+2.*B*D*TOP*(TOP+BOT)+(D*D*TOP*TOP))
0270 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
0270 STH=A/DEN
0274 WRITE(5,60)STH
0274 60 FORMAT(' ',22Y,'STH='F10.3)
0274 C=560.*BM*(B*TOP+D*BOT)
0274 C *SBH* IS THE STRESS ON BOT DUE TO HOGGING
0274 SBH=C/DEN
0274 WRITE(5,90)SBH
0274 90 FORMAT(' ',22Y,'SBH='F10.3)
0280 C *SCRIT* IS THE CRITICAL STRESS ON BOT
0280 SCRIT=(BOT*BOT)/G
0280 WRITE(5,110)SCRIT
0280 110 FORMAT(' ',17Y,'SCRIT='F10.3)
0284 E=2.*D*TOP*B*(TOP+BOT)
0284 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0284 DITOP=L*(B*BOT+D*TOP)/E
0284 WRITE(5,42)DITOP
0284 40 FORMAT(' ',18Y,'DITOP='F10.4)
0284 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0284 DIBOT=L*(B*TOP+D*TOP)/E
0284 WRITE(5,54)DIBOT
0284 50 FORMAT(' ',18Y,'DIBOT='F10.4)
0284 END
(S) .H 0324 (V) AL 0320 (V) B 0334 (V) DRA 0330 (V) D
(V) P=I 0340 (V) E 0342 (S) .R 0358 (V) DST 0360 (V) S
(V) S= 0378 (V) G 0384 (V) TOP 0380 (V) BOT 0394 (V) A
(V) DEN 0388 (V) STH 0174 (L) 60 0000 (S) @I 0380 (V) C

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STRENGTH OF SHIPS ON THE CENTRAL AXIS LOCATION

(V) SQ4 , 1E2(L) 92 304(V) SBCRIT 0228(L) 110 0308(V) P
 (V) DITOP 248(L) 40 302(V) DIBOT 0304(L) 50 0000(S) .V
 EQ L

GRAM LABELS:

17V 74111	2464 .V	2894 .COMP	2A72 0I	2444 .R	2950 .Z1
48A 3A	2484 .A	2988 .MES	2936 .W	263E EXP	27C4 AI
536 ALCC	28FC .MARG	2958 .S	263E AEXP	2960 .ERCNT	292E \$8
482 .	2736 .U	288E			

XY-POINTS:

444 .0	2484 .A	2536 AL06	263E AEXP	263E EXP	27C4 AI
236 .	2504 .COMP	248A 4A	28FC .MARG	292E \$8	2958 .S
2FC .75E6	296V .ERCNT	2968 .	2988 .MES	2A42 .U	2A68 .V
472 XY					

NON-BLOCKS:

DEFINED SYMBOLS:

TRANSFER ADDRESS 2270
 POSITION REGIONS:

STR= 18111.109
 SBF= 53387.980
 SBCRIT= 48640.414
 DITOP= 18.0932
 DIBOT= 53.3353

VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

08 HANDLING CHARGE	\$.25 / JOB	.35
25 LINES PRINTED PR1	\$ 1.25 / K LN	.11
54 CARDS READ	\$ 1.25 / K Ch	.38
20 PLOTTER VECTORS	\$.25 / 1272	.00
17 MODEL 70 SECONDS	\$25.20 / HOUR	.12
00 MODEL 80 SECONDS	\$12.50 / HOUR	.20
	TOTAL CHARGE \$.66

TEIP 427 14731 LOGGED OUT 25/06/74 19:56. \$ 3.43 LEFT AFTER 31 LOGINS.

SUMMARY

Length	=	1000'
Beam	=	AL/5.75
Draft	=	B/3.3
Depth	=	AL/14
DST	=	18000 psi
S	=	10"
Material	=	ALUMINUM
TOP	=	5.35"
BOT	=	0.363"
STH	=	18111.11
SBH	=	53387.98
SBCRIT	=	48648.64
DITOP	=	18.09'
DIBOT	=	53.34'


```

004 C *AL* IS THE LENGTH
004 AL=1000.
008 B=AL/5.75
018 C *DRA* IS THE DRAFT
018 DRA=B/3.3
024 D=AL/14.
032 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
032 FOR ALUMINUM
032 POI=0.35
038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
038 FOR ALUMINUM
038 E=10.**10.**6
040 C *DST* IS THE DESIGN STRESS
040 DST=18000.
054 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
054 S=30.
060 C *BM* IS THE BENDING MOMENT
060 BM=(B*DRA*AL*AL*(5.75)/(35.*35.))
068 G=0.30386*S*S*(1.-POI*POI)/E
074 A=9.*B**D/(140.*BM)
080 C=3.*B*D/D/(70.*BM)
084 F=3.*D*D/D/(140.*BM)
100 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
100 WRITE(5,10)
104 10 FORMAT('1',25Y,'TOP ASSUMED LARGER THAN BOTTOM')
106 C *TOP* IS THE THICKNESS OF THE TOP PLATING
106 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
106 BOT1=0.2
106 11 TOP1=(B*BOT1*BOT1*BOT1)/((B+D)*G*DST-D*BOT1*POT1)
106 WRITE(5,20)TOP1,BOT1
106 20 FORMAT(' ',25Y,'TOP1='F9.4,5X,'BOT1='F9.4)
106 BOT1=BOT1+0.1
106 IF(BOT1.GT.2.5)GO TO30
106 GO TO 11
106 30 TOP2=1.
106 31 BOT2=TOP2*(12.*D-(C+F)*DST*TOP2)/(DST*(A+C)*TOP2-12.*B)
106 WRITE(5,40)TOP2,BOT2
106 40 FORMAT(' ',25Y,'TOP2='F9.4,5X,'BOT2='F9.4)
106 TOP2=TOP2+0.2
106 IF(TOP2.GT.6. ) GO TO 50
106 GO TO 31
106 50 CONTINUE
106 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
106 WRITE(5,60)
106 60 FORMAT('1',25Y,'BOTTOM ASSUMED LARGER THAN TOP')
106 BOT3=0.1
106 61 TOP3=((B+D)*BOT3*BOT3*BOT3-D*G*DST-BOT3)/(B*G*DST)
106 WRITE(5,70)TOP3,BOT3
106 70 FORMAT(' ',25Y,'TOP3='F9.4,5X,'BOT3='F9.4)
106 BOT3=BOT3+0.01
106 IF(BOT3.GT.1.5)GO TO 80
106 GO TO 61

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STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

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-12      90 TOP4=1.
-13      31 BOT4=(12.*(B+n)-(A+C)*DST*TOP4)/((C+E)*DST)
-14      WRITE(5,90)TOP4,BOT4
-17      90 FORMAT('1',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
-19      TOP4=TOP4+0.2
-21      IF(TOP4*GT.5.5)GO TO 100
-22      GO TO 81
-23      100 CONTINUE
-24      END
(S) 01 0404(V) AL 0408(V) B 0410(V) DPA 0408(V) D
(V) P01 0458(V) F 0460(V) .R 0464(V) DST 0460(V) S
(V) 84 0518(V) G 0524(V) A 0530(V) C 0530(V) F
(L) 10 0540(S) AI 0540(V) BOT1 0176(L) 11 0548(V) TOP1
(L) 20 0520(L) 31 0550(V) TOP2 0234(L) 31 0560(V) BOT2
(L) 40 0288(L) 50 0312(L) 60 0574(V) BOT3 0348(L) 61
(V) TOP3 0348(L) 70 0402(L) 80 0584(V) TOP4 0408(L) 81
(V) BOT4 0472(L) 90 0490(L) 100 0000(S) .V
E6 L

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GRAM LABELS:

170 *MAIN*	2028 .V	2A50 .COMP	2C2E .I	2600 .P	2B18 .78
A76 84	2640 .A	2A74 .MES	29F2 .W	27FA EXP	2980 AIN
652 ALCO	2AB3 .RARG	2B14 .5	27FA AEXP	2B1C .ERCNT	2AEA 88
810 .0	2BFB .U	304A			

HY-POINTS:

810 .R	2640 .A	26F2 ALCO	27FA AEXP	27FA EXP	2980 AIN
9F2 .7	2A50 .COMP	2A74 84	2A88 .RARG	2AEA 88	2B14 .5
618 .7EEN	2B1C .ERCNT	2B1E .0	2B74 .MES	2BFE .U	2C24 .7
02E 91					

MON-BLOCKS:

E

DEFINED SUBROUTINES:

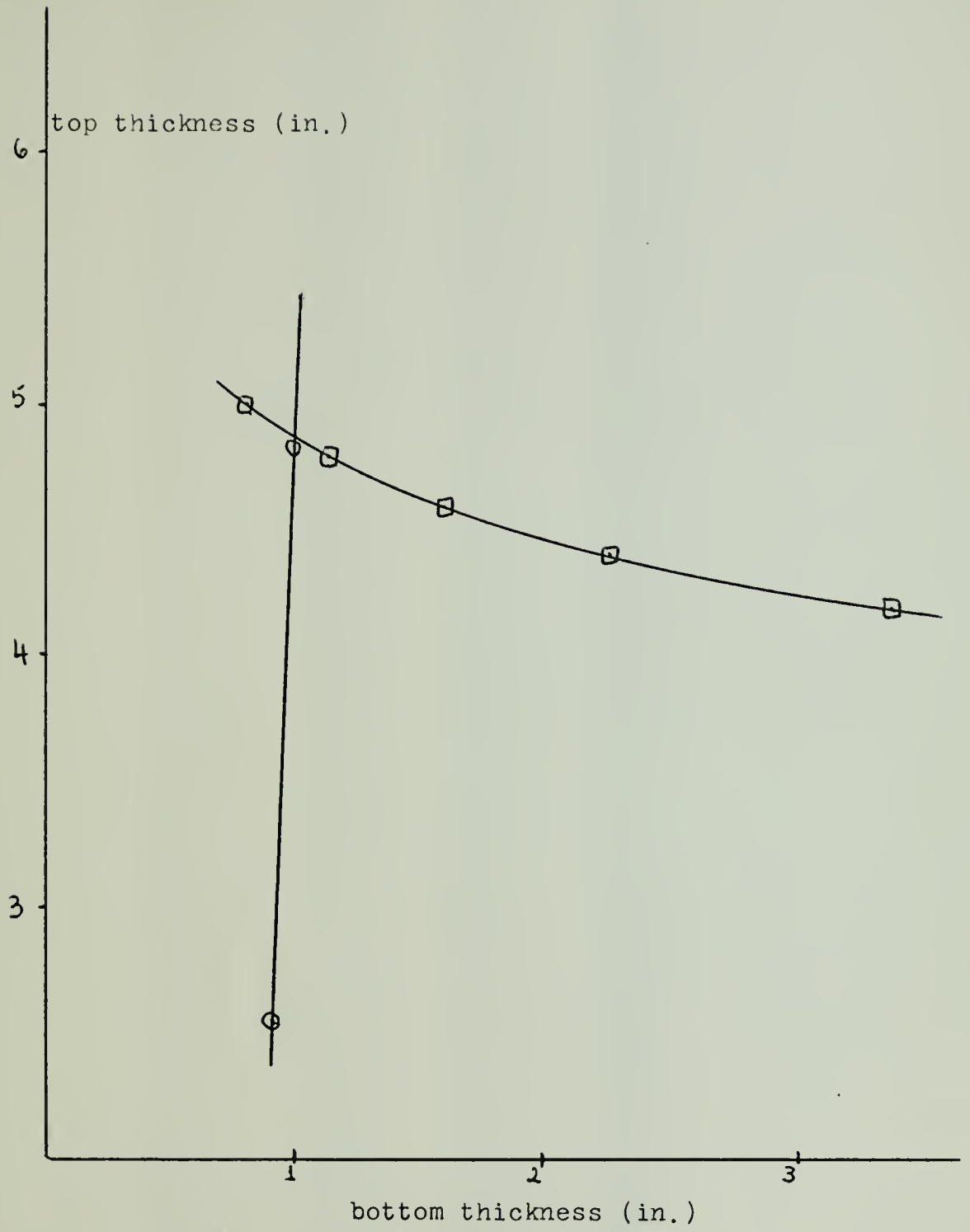
E

TRANSFER ADDRESS 2070

LOCATION BEGINS:

TOP ASSUMED LARGER THAN BOTTOM

TOP1=	2.0123	BOT1=	0.2000
TOP1=	2.2464	BOT1=	0.3000
TOP1=	4.1127	BOT1=	0.4000
TOP1=	3.2421	BOT1=	2.5000
TOP1=	1.4524	BOT1=	0.6000
TOP1=	0.2210	BOT1=	0.7000
TOP1=	1.4376	BOT1=	0.8000
TOP1=	2.5460	BOT1=	0.9000
TOP1=	4.8228	BOT1=	1.0000
TOP1=	10.9155	BOT1=	1.1000
TOP1=	62.6441	BOT1=	1.2000
TOP1=	-29.2267	BOT1=	1.3000
TOP1=	-14.7228	BOT1=	1.4000
TOP1=	-11.7622	BOT1=	1.5000
TOP1=	-9.4724	BOT1=	1.6000
TOP1=	-8.6524	BOT1=	1.7000
TOP1=	-8.1925	BOT1=	1.8000
TOP1=	-7.3417	BOT1=	1.9000
TOP1=	-7.8127	BOT1=	2.0000
TOP1=	-7.7677	BOT1=	2.1000
TOP1=	-7.7728	BOT1=	2.2000
TOP1=	-7.8311	BOT1=	2.3000
TOP1=	-7.9123	BOT1=	2.4000
TOP1=	-8.0212	BOT1=	2.5000
TOP2=	1.2220	BOT2=	-0.4673
TOP2=	1.2220	BOT2=	-0.5810
TOP2=	1.4220	BOT2=	-0.7055
TOP2=	1.6220	BOT2=	-7.8440
TOP2=	1.2220	BOT2=	-1.0009
TOP2=	2.0020	BOT2=	-1.1829
TOP2=	2.2220	BOT2=	-1.4002
TOP2=	2.4220	BOT2=	-1.6693
TOP2=	2.6220	BOT2=	-2.0190
TOP2=	2.8220	BOT2=	-2.5045
TOP2=	3.2220	BOT2=	-3.2460
TOP2=	3.2420	BOT2=	-4.5652
TOP2=	3.4220	BOT2=	-7.7102
TOP2=	3.6220	BOT2=	-26.8087
TOP2=	3.8020	BOT2=	16.3226
TOP2=	4.0220	BOT2=	5.8811
TOP2=	4.2220	BOT2=	3.3945
TOP2=	4.4220	BOT2=	2.2560
TOP2=	4.6220	BOT2=	1.5888
TOP2=	4.2220	BOT2=	1.1408
TOP2=	5.0220	BOT2=	0.8127
TOP2=	5.2220	BOT2=	0.5572
TOP2=	5.4220	BOT2=	0.3490
TOP2=	5.6220	BOT2=	0.1733
TOP2=	5.8220	BOT2=	0.0210
TOP2=	6.0220	BOT2=	-0.1140



BOTTOM	ASSIGNED	LARGER THAN	TOP
TCP3=	-0.0349	BOT3=	0.1000
TCP3=	-0.0409	BOT3=	0.1100
TCP3=	-0.0427	BOT3=	0.1200
TCP3=	-0.0463	BOT3=	0.1300
TCP3=	-0.0497	BOT3=	0.1400
TCP3=	-0.0528	BOT3=	0.1500
TCP3=	-0.0525	BOT3=	0.1600
TCP3=	-0.0540	BOT3=	0.1700
TCP3=	-0.0552	BOT3=	0.1800
TCP3=	-0.0560	BOT3=	0.1900
TCP3=	-0.0564	BOT3=	0.2000
TCP3=	-0.0565	BOT3=	0.2100
TCP3=	-0.0561	BOT3=	0.2200
TCP3=	-0.0563	BOT3=	0.2300
TCP3=	-0.0561	BOT3=	0.2400
TCP3=	-0.0524	BOT3=	0.2500
TCP3=	-0.0503	BOT3=	0.2600
TCP3=	-0.0476	BOT3=	0.2700
TCP3=	-0.0444	BOT3=	0.2800
TCP3=	-0.0407	BOT3=	0.2900
TCP3=	-0.0364	BOT3=	0.3000
TCP3=	-0.0315	BOT3=	0.3100
TCP3=	-0.2261	BOT3=	0.3200
TCP3=	-0.2200	BOT3=	0.3300
TCP3=	-0.2153	BOT3=	0.3400
TCP3=	-0.2109	BOT3=	0.3500
TCP3=	0.2021	BOT3=	0.3600
TCP3=	0.2109	BOT3=	0.3700
TCP3=	0.2203	BOT3=	0.3800
TCP3=	0.2305	BOT3=	0.3900
TCP3=	0.2415	BOT3=	0.4000
TCP3=	0.2522	BOT3=	0.4100
TCP3=	0.2627	BOT3=	0.4200
TCP3=	0.2700	BOT3=	0.4300
TCP3=	0.2822	BOT3=	0.4400
TCP3=	0.2921	BOT3=	0.4500
TCP3=	0.3040	BOT3=	0.4600
TCP3=	0.3148	BOT3=	0.4700
TCP3=	0.3224	BOT3=	0.4800
TCP3=	0.3270	BOT3=	0.4900
TCP3=	0.3365	BOT3=	0.5000
TCP3=	0.3470	BOT3=	0.5100
TCP3=	0.3505	BOT3=	0.5200
TCP3=	0.3610	BOT3=	0.5300
TCP3=	0.3745	BOT3=	0.5400
TCP3=	0.3800	BOT3=	0.5500
TCP3=	0.3946	BOT3=	0.5600
TCP3=	0.4013	BOT3=	0.5700
TCP3=	0.4101	BOT3=	0.5800
TCP3=	0.4180	BOT3=	0.5900
TCP3=	0.4200	BOT3=	0.6000
TCP3=	0.4702	BOT3=	0.6100
TCP3=	0.5116	BOT3=	0.6200
TCP3=	0.5451	BOT3=	0.6300
TCP3=	0.5709	BOT3=	0.6400
TCP3=	0.6159	BOT3=	0.6500

TOP3=	0.6522	BOT3=	0.6600
TOP3=	0.6918	BOT3=	0.6700
TOP3=	0.7316	BOT3=	0.6800
TOP3=	0.7728	BOT3=	0.6900
TOP3=	0.8152	BOT3=	0.7000
TOP3=	0.8581	BOT3=	0.7100
TOP3=	0.9013	BOT3=	0.7200
TOP3=	0.9509	BOT3=	0.7300
TOP3=	0.9999	BOT3=	0.7400
TOP3=	1.0463	BOT3=	0.7500
TOP3=	1.0902	BOT3=	0.7600
TOP3=	1.1315	BOT3=	0.7700
TOP3=	1.1753	BOT3=	0.7800
TOP3=	1.2166	BOT3=	0.7900
TOP3=	1.2575	BOT3=	0.8000
TOP3=	1.2969	BOT3=	0.8100
TOP3=	1.4369	BOT3=	0.8200
TOP3=	1.4974	BOT3=	0.8300
TOP3=	1.5605	BOT3=	0.8400
TOP3=	1.6253	BOT3=	0.8500
TOP3=	1.6917	BOT3=	0.8600
TOP3=	1.7508	BOT3=	0.8700
TOP3=	1.8205	BOT3=	0.8800
TOP3=	1.9009	BOT3=	0.8900
TOP3=	1.9711	BOT3=	0.9000
TOP3=	2.0420	BOT3=	0.9100
TOP3=	2.1256	BOT3=	0.9200
TOP3=	2.2040	BOT3=	0.9300
TOP3=	2.2842	BOT3=	0.9400
TOP3=	2.3663	BOT3=	0.9500
TOP3=	2.4521	BOT3=	0.9600
TOP3=	2.5358	BOT3=	0.9700
TOP3=	2.6234	BOT3=	0.9800
TOP3=	2.7129	BOT3=	0.9900
TOP3=	2.8043	BOT3=	1.0000
TOP3=	2.8976	BOT3=	1.0100
TOP3=	2.9928	BOT3=	1.0200
TOP3=	3.0902	BOT3=	1.0300
TOP3=	3.1802	BOT3=	1.0400
TOP3=	3.2925	BOT3=	1.0500
TOP3=	3.3927	BOT3=	1.0600
TOP3=	3.4900	BOT3=	1.0700
TOP3=	3.6043	BOT3=	1.0800
TOP3=	3.7127	BOT3=	1.0900
TOP3=	3.8273	BOT3=	1.1000
TOP3=	3.9429	BOT3=	1.1100
TOP3=	4.0567	BOT3=	1.1200
TOP3=	4.1747	BOT3=	1.1300
TOP3=	4.2948	BOT3=	1.1400
TOP3=	4.4171	BOT3=	1.1500
TOP3=	4.5417	BOT3=	1.1600
TOP3=	4.6625	BOT3=	1.1700
TOP3=	4.7975	BOT3=	1.1800
TOP3=	4.9228	BOT3=	1.1900
TOP3=	5.0624	BOT3=	1.2000
TOP3=	5.1924	BOT3=	1.2100
TOP3=	5.3326	BOT3=	1.2200

TOP3=	5.4772	BOT3=	1.2300
TOP3=	5.6222	BOT3=	1.2400
TOP3=	5.7626	BOT3=	1.2500
TOP3=	5.9124	BOT3=	1.2600
TOP3=	6.0626	BOT3=	1.2700
TOP3=	6.2123	BOT3=	1.2800
TOP3=	6.3714	BOT3=	1.2900
TOP3=	6.5220	BOT3=	1.3000
TOP3=	6.6822	BOT3=	1.3100
TOP3=	6.8518	BOT3=	1.3200
TOP3=	7.0170	BOT3=	1.3300
TOP3=	7.1848	BOT3=	1.3400
TOP3=	7.3522	BOT3=	1.3500
TOP3=	7.5221	BOT3=	1.3600
TOP3=	7.7027	BOT3=	1.3700
TOP3=	7.8819	BOT3=	1.3800
TOP3=	8.0628	BOT3=	1.3900
TOP3=	8.2424	BOT3=	1.4000
TOP3=	8.4326	BOT3=	1.4100
TOP3=	8.6216	BOT3=	1.4200
TOP3=	8.8123	BOT3=	1.4300
TOP3=	9.0078	BOT3=	1.4400
TOP3=	9.2021	BOT3=	1.4500
TOP3=	9.4021	BOT3=	1.4600
TOP3=	9.6020	BOT3=	1.4700
TOP3=	9.8127	BOT3=	1.4800
TOP3=	10.0223	BOT3=	1.4900
TOP3=	10.2327	BOT3=	1.5000
TOP4=	1.2420	BOT4=	16.1639
TOP4=	1.2720	BOT4=	15.3920
TOP4=	1.4120	BOT4=	14.6201
TOP4=	1.6120	BOT4=	13.8482
TOP4=	1.8120	BOT4=	13.0763
TOP4=	2.0220	BOT4=	12.3044
TOP4=	2.2420	BOT4=	11.5324
TOP4=	2.4720	BOT4=	10.7605
TOP4=	2.6420	BOT4=	9.9886
TOP4=	2.8120	BOT4=	9.2167
TOP4=	3.0220	BOT4=	8.4448
TOP4=	3.2220	BOT4=	7.6729
TOP4=	3.4720	BOT4=	6.9010
TOP4=	3.6720	BOT4=	6.1290
TOP4=	3.8220	BOT4=	5.3571
TOP4=	4.0220	BOT4=	4.5852
TOP4=	4.2420	BOT4=	3.8133
TOP4=	4.4220	BOT4=	3.0414
TOP4=	4.6720	BOT4=	2.2695
TOP4=	4.9220	BOT4=	1.4975
TOP4=	5.2220	BOT4=	0.7256
TOP4=	5.2420	BOT4=	-0.4463
TOP4=	5.4220	BOT4=	-0.8182

CR HANDLING CHARGE	\$.35 / JOB	.35
11 LINES PRINTED PR1	\$ 1.25 / K LN	.39
67 CARDS READ	\$ 1.50 / K CD	.17
87 PLOTTER VECTORS	\$.25 / 1000	.02
22 MODEL 7: SECONDS	\$25.20 / HOUR	.15
80 MODEL 8: SECONDS	\$12.50 / HOUR	.08
	TOTAL CHARGE \$.99

EIP 404 14731 LOGGED OUT 05/26/74 19:42. \$ 5.68 LEFT AFTER 28 LOGINS.


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04 C *AL* IS THE LENGTH
04 AL=100.7
08 C *DRA* IS THE DRAFT
08 DRA=H/3.3
12 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
12 FOR ALUMINUM
12 POI=0.33
16 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
16 FOR ALUMINUM
16 E=10.**10.**6
20 C *DST* IS THE DESIGN STRESS
20 DST=15000.
24 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
24 S=30.
28 C *M* IS THE BENDING MOMENT
28 BM=(B*DRA*AL*AL*0.75)/(35.**35.)
32 C *TOP* IS THE THICKNESS OF THE TOP PLATING
32 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
32 TOP=4.84
32 BOT=1.
36 C *A* IS THE AREA OF THE SECTION
36 A=560.**M*(E*BOT+D*TOP)
40 C *CEN* IS THE CENTROID OF THE SECTION
40 CEN=D*(B*B*P*TOP*POI+2.*B*D*TOP*(TOP+BOT)+(D*D*TOP*TOP))
44 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
44 STH=A/DEN
48 WRITE(5,60)STH
52 FORMAT(' ',20Y,'STH='F10.3)
56 C *SBH* IS THE STRESS ON BOT DUE TO HOGGING
56 SBH=C/DEN
60 WRITE(5,90)SBH
64 FORMAT(' ',20Y,'SBH='F10.3)
68 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
68 SBCRIT=BOT*BOT/G
72 WRITE(5,110)SBCRIT
76 FORMAT(' ',17Y,'SBCRIT='F10.3)
80 P=2.*D*TOP+B*(TOP+BOT)
84 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
84 DITOP=D*(B*BOT+D*TOP)/P
88 WRITE(5,40)DITOP
92 FORMAT(' ',12Y,'DITOP='F10.4)
96 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
96 DIBOT=D*(B*TOP+D*TOP)/P
100 WRITE(5,50)DIBOT
104 FORMAT(' ',18Y,'DIBOT='F10.4)
108 END
110
(S) *U 0324(V) AL 0320(V) B 0334(V) DRA 0330(V) D
(V) POI 0340(V) E 0327(S) *R 0358(V) DST 0360(V) S
(V) BM 0370(V) G 0388(V) TOP 0390(V) BOT 0394(V) A
(V) DEN 0388(V) STH 0174(L) 60 0000(S) @I 0380(V) C

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STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

(V) SAH 21F2(L) 92 1304(V) SBCRIT 0228(L) 110 0308(V) P
 (V) DITOP 02A8(L) 42 1300(V) DIPOT 0304(L) 50 0002(S) .V
 DEL L

GRAM LABELS:

2770 *SAH*	2A64 .V	2894 .COMP	2A72 AI	2444 .R	295C .Z
282A SA	2484 .A	2888 .MES	2436 .W	263F EXP	27C4 AI
236 AL0G	28FC .RARG	2858 .S	263E AEXP	2960 .ERCNT	292E \$8
260 .D	2A26 .D	288F			

KEY-POINTS:

444 .	2484 .A	2836 AL0G	263E AEXP	263E EXP	27C4 AI
236 .	2894 .COMP	2884 .S	28FC .RARG	292E \$8	2958 .5
550 .7-80	2960 .ERCNT	2862 .D	2888 .MES	2A42 .U	2468 .V
472 .T					

CON-BLOCKS:

E

DEFINED SUBROUTINES:

E

TRANSFER ADDRESS 2070

LOCATION REGISTERS:

SIH= 18062.194
 SHF= 41334.957
 SBCRIT= 41021.625
 DITOP= 21.7217
 DIPOT= 49.7269

END

VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

JOB HANDLING CHARGE	\$.35 / JOB	.35
85 LINES PRINTED PR1	\$ 1.25 / K LN	.11
54 CARDS READ	\$ 1.50 / K CD	.08
77 PLOTTER VECTORS	\$.25 / 1770	.00
16 MODEL 70 SECONDS	\$25.00 / HOUR	.11
02 MODEL 80 SECONDS	\$12.50 / HOUR	.00
	TOTAL CHARGE \$.65

FEIR 40 14731 LOGGED OUT 05/26/74 19:58. \$ 2.78 LEFT AFTER 32 LOGINS.

SUMMARY

Length	=	1000'
Beam	=	AL/5.75
Draft	=	B/3.3
Depth	=	AL/14
DST	=	18000 psi
S	=	30"
Material	=	ALUMINUM
TOP	=	4.86"
BOT	=	1.00"
STH	=	18063.18
SBH	=	41334.96
SBCRIT	=	41021.80
DITOP	=	21.72'
DIBOT	=	49.71'

STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

PAGE 1

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0084 C *AL* IS THE LENGTH
0084 AL=1000.
0090 B=AL/5.75
0090 C *DRA* IS THE DRAFT
0090 DRA=B/3.3
0090 D=AL/14.
0090 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0090 FOR ALUMINUM
0090 POI=7.35
0090 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0090 FOR ALUMINUM
0090 E=10.**11.**6
0090 C *DST* IS THE DESIGN STRESS
0090 DST=18000.
0090 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0090 S=60.
0090 C *BM* IS THE BENDING MOMENT
0090 BM=(B*D*A*AL*AL*.75)/(35.*35.)
0090 G=V.30386*S*S*(1.+POI*POI)/E
0090 A=9.*B.*D/(140.*BM)
0090 C=3.*E.*D/(70.*A)
0090 F=3.*D.*D/(140.*B)
0090 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
0090 WRITE(5,17)
0090 1) FORMAT(' ',25X,'TOP ASSUMED LARGER THAN BOTTOM')
0090 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0090 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0090 BOT1=1.
0090 1) TOP1=(B*BOT1*BOT1*BOT1)/((B+D)*G*DST-D*BOT1*BOT1)
0090 WRITE(5,20)TOP1,BOT1
0090 2) FORMAT(' ',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
0090 BOT1=BOT1+0.1
0090 IF(BOT1.GT.2.5)GO TO 30
0090 GO TO 11
0090 3) TOP2=3.5
0090 31 BOT2=(TOP2*(12.*D-(C+F)*DST*TOP2)/(DST*(A+C)*TOP2-12.*B)
0090 WRITE(5,40)TOP2,BOT2
0090 4) FORMAT(' ',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
0090 TOP2=TOP2+0.2
0090 IF(TOP2.GT.6. ) GO TO 52
0090 GO TO 31
0090 5) CONTINUE
0090 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
0090 WRITE(5,60)
0090 4) FORMAT(' ',25X,'BOTTOM ASSUMED LARGER THAN TOP')
0090 BOT3=0.7
0090 61 TOP3=((B+D)*BOT3*BOT3*BOT3-D*G*DST*BOT3)/(B*G*DST)
0090 WRITE(5,70)TOP3,BOT3
0090 7) FORMAT(' ',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
0090 BOT3=BOT3+0.05
0090 IF(BOT3.GT.1.5)GO TO 32
0090 GO TO 61

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

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482      20 TOP4=1.
484      21 BOT4=(10.*(P+n)-(A+C)*DST*TOP4)/((C+F)*DST)
486      WRITE(5,99)TOP4,BOT4
487      9. FORMAT(' ',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
488      TOP4=TOP4+0.2
489      IF (TOP4.GT.5.) GO TO 490
490      GO TO 31
491      12. CONTINUE
492      END
    
```

(S) .H	0400 (V) AL	0405 (V) B	0407 (V) DRA	0408 (V) D
(V) BOT1	0408 (V) E	0409 (V) .R	0404 (V) DST	0406 (V) S
(V) 3V	0518 (V) G	0524 (V) A	0530 (V) C	0530 (V) F
(L) 17	0500 (S) .I	0540 (V) BOT1	0176 (L) 11	0544 (V) TOP1
(L) 27	0220 (L) 3V	0550 (V) TOP2	0224 (L) 31	0560 (V) BOT2
(L) 47	0200 (L) 3V	0312 (L) 4V	0578 (V) BOT3	0348 (L) 61
(V) TOP3	0308 (L) 7V	0400 (L) 8V	0500 (V) TOP4	0400 (L) 81
(V) BOT4	0472 (L) 9V	0400 (L) 10V	0000 (S) .V	

REG L

DATA LABELS:

2070 .MAT	2028 .V	2058 .COMP	2036 .I	2602 .R	2820 .Z
247E .A	2648 .A	2470 .MES	29FA .W	2802 EXP	2988 AI
26FA .ALOG	2ACC .RARG	2810 .S	2882 .EXP	2824 .ERCNT	2AF2 .S
2824 .S	28FA .L	3052			

TRY-POINTS:

2602 .R	2648 .A	26FA .ALOG	2802 .EXP	2802 EXP	2988 AI
29FA .W	2A58 .COMP	247E .A	2ACC .RARG	2AF2 .S	2810 .S
2820 .S	2824 .ERCNT	2824 .S	2870 .MES	2C06 .U	2C20 .V
2C36 .I					

CON-BLOCKS:

E

DEFINED SUBROUTINES:

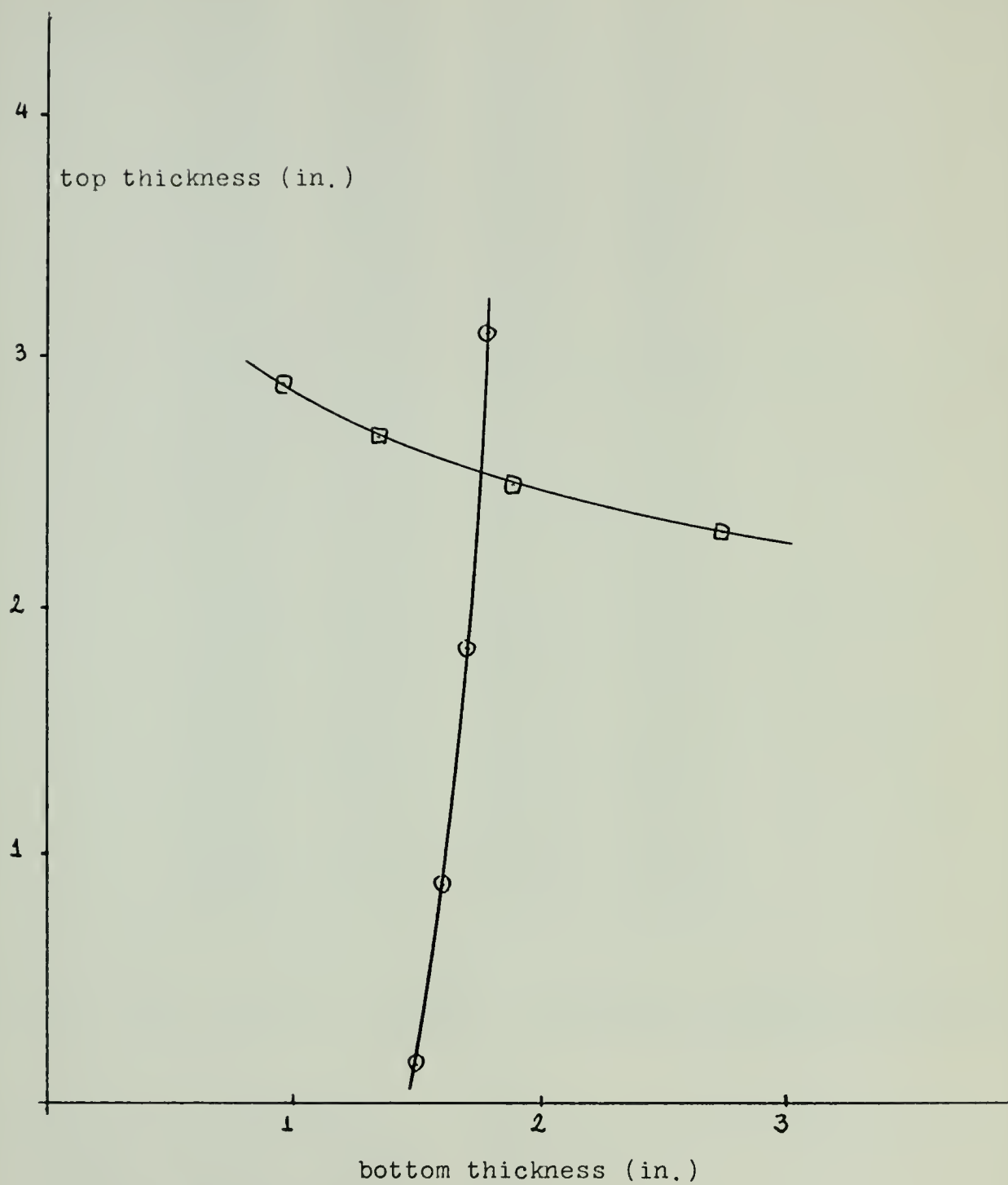
E

TRANSFER ADDRESS 2070

LOCATION REGISTERS:

TCP ASSUMED LARGER THAN BOTTOM

TCP1=	1.4842	BOT1=	1.3000
TCP1=	1.6725	BOT1=	1.1700
TCP1=	1.9149	BOT1=	1.2000
TCP1=	1.2329	BOT1=	1.3000
TCP1=	1.6421	BOT1=	1.4000
TCP1=	2.1747	BOT1=	1.5000
TCP1=	2.2751	BOT1=	1.6000
TCP1=	3.2112	BOT1=	1.7000
TCP1=	5.2919	BOT1=	1.8000
TCP1=	6.9047	BOT1=	1.9000
TCP1=	9.6145	BOT1=	2.0000
TCP1=	13.9532	BOT1=	2.1000
TCP1=	21.2125	BOT1=	2.2000
TCP1=	47.1143	BOT1=	2.3000
TCP1=	125.2754	BOT1=	2.4000
TCP1=	-171.5108	BOT1=	2.5000
TCP2=	3.5000	BOT2=	-11.8971
TCP2=	3.7000	BOT2=	91.0296
TCP2=	3.9000	BOT2=	8.7662
TCP2=	4.1000	BOT2=	4.3496
TCP2=	4.2000	BOT2=	2.7380
TCP2=	4.3000	BOT2=	1.8850
TCP2=	4.7000	BOT2=	1.3454
TCP2=	4.9000	BOT2=	0.9655
TCP2=	5.1000	BOT2=	0.6778
TCP2=	5.2000	BOT2=	0.4483
TCP2=	5.5000	BOT2=	0.2578
TCP2=	5.7000	BOT2=	0.1947
TCP2=	5.9000	BOT2=	-0.2484



BOTTOM ASSUMED	LARGER THAN	TOP
TOP3= 0.0118	BOT3=	0.7000
TOP3= 0.0310	BOT3=	0.7500
TOP3= 0.0809	BOT3=	0.8000
TOP3= 0.1445	BOT3=	0.8500
TOP3= 0.2143	BOT3=	0.9000
TOP3= 0.2909	BOT3=	0.9500
TOP3= 0.3900	BOT3=	1.0000
TOP3= 0.4902	BOT3=	1.0500
TOP3= 0.6100	BOT3=	1.1000
TOP3= 0.7501	BOT3=	1.1500
TOP3= 0.8940	BOT3=	1.2000
TOP3= 1.0524	BOT3=	1.2500
TOP3= 1.2319	BOT3=	1.3000
TOP3= 1.4200	BOT3=	1.3500
TOP3= 1.6305	BOT3=	1.4000
TOP3= 1.8548	BOT3=	1.4500
TOP3= 2.0945	BOT3=	1.5000
TOP4= 1.0000	BOT4=	16.1639
TOP4= 1.2200	BOT4=	15.3920
TOP4= 1.4400	BOT4=	14.6201
TOP4= 1.6600	BOT4=	13.8482
TOP4= 1.8800	BOT4=	13.0763
TOP4= 2.1000	BOT4=	12.3044
TOP4= 2.3200	BOT4=	11.5324
TOP4= 2.5400	BOT4=	10.7605
TOP4= 2.7600	BOT4=	9.9886
TOP4= 2.9800	BOT4=	9.2167
TOP4= 3.2000	BOT4=	8.4448
TOP4= 3.4200	BOT4=	7.6729
TOP4= 3.6400	BOT4=	6.9010
TOP4= 3.8600	BOT4=	6.1290
TOP4= 4.0800	BOT4=	5.3571
TOP4= 4.3000	BOT4=	4.5852
TOP4= 4.5200	BOT4=	3.8133
TOP4= 4.7400	BOT4=	3.0414
TOP4= 4.9600	BOT4=	2.2695
TOP4= 5.1800	BOT4=	1.4975
TOP4= 5.4000	BOT4=	0.7256

END

UNIX OPERATING SYSTEM VERSION 1 REVISION 12 03/04/74 GENERATED 05/06/74

JOB HANDLING CHARGE	\$ 0.25 / JOB	0.35
164 LINES PRINTED PR	\$ 1.25 / K LN	0.21
67 CARDS READ	\$ 1.50 / K CD	0.10
00 PLOTTER VECTORS	\$ 0.25 / 10.00	0.00
19 MODEL 7 SECONDS	\$25.00 / HOUR	0.13
02 MODEL 8 SECONDS	\$12.50 / HOUR	0.00
TOTAL CHARGE	\$	0.79

IP 402 14731 LOGGED OUT 05/06/74 19:48. \$ 4.89 LEFT AFTER 29 LOGINS.

STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 1

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0004 C *AL* IS THE LENGHT
0004 AL=1000.
0004 B=AL/5.75
0004 C *DRA* IS THE DRAFT
0004 DRA=B/3.3
0004 D=AL/14.
0004 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0004 FOR ALUMINUM
0004 POI=0.33
0004 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0004 FOR ALUMINUM
0004 E=10**10**6
0004 C *DST* IS THE DESIGN STRESS
0004 DST=18722.
0004 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0004 S=60.
0004 C *BM* IS THE BENDING MOMENT
0004 BM=(B*DRA*AL*AL*7.75)/(35.*35.)
0004 C=C*3.396*S*S*(1.+POI*POI)/E
0004 C *TOP* IS THE T-THICKNESS OF THE TOP PLATING
0004 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0004 TOP=4.52
0004 BOT=1.74
0004 A=560.*BM*(B*BOT+D*TOP)
0004 DEN=D*(2.*B*TOP*BOT+2.*B*D*TOP*(TOP+BOT)+(D*D*TOP-TOP))
0004 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
0004 STH=A/DEN
0004 WRITE(5,40)STH
0004 60 FORMAT(' ',2X,'STH='F10.3)
0004 C=C*560.*BM*(B*TOP+D*TOP)
0004 C *SBH* IS THE STRESS ON BOT DUE TO HOGGING
0004 SBH=C/DEN
0004 WRITE(5,80)SBH
0004 90 FORMAT(' ',2X,'SBH='F10.3)
0004 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
0004 SBCRIT=BOT*BOT/G
0004 WRITE(5,11)SBCRIT
0004 110 FORMAT(' ',1X,'SBCRIT='F10.3)
0004 P=2.*D*TOP+B*(TOP+BOT)
0004 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0004 DITOP=D*(B*BOT+D*TOP)/P
0004 WRITE(5,42)DITOP
0004 40 FORMAT(' ',1X,'DITOP='F10.4)
0004 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0004 DIBOT=D*(B*TOP+D*TOP)/P
0004 WRITE(5,50)DIBOT
0004 50 FORMAT(' ',1X,'DIBOT='F10.4)
0004 END
0004
0004 (G) .U 0324 [V] AL 0320 [V] B 0334 [V] DRA 0330 [V] D
0004 (V) PCI 0340 [V] E 0000 [S] .R 0358 [V] DST 0360 [V] S
0004 (V) BM 0370 [V] G 0388 [V] TOP 0390 [V] BOT 0398 [V] A
0004 (V) DEN 0380 [V] STH 0174 [L] 60 0000 [S] POI 0300 [V] C

```


STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

01E2 [L] 90	0308 [V] SB CRIT	0228 [L] 110	0300 [V] P
02A8 [L] 40	0304 [V] DIROT	0304 [L] 50	0000 [S] .V

PROGRAM LABELS:

2077 .MAIN*	2408 .V	2898 .COMP	2476 .AI	2448 .R	2960 .ZE
2438 \$6	2438 .A	2900 .MES	283A .W	2642 EXP	2708 AIM
253A ALOG	2970 .RARG	2950 .S	2642 AEXP	2964 .ERCNT	2932 \$8
2964 .O	2A3A .U	3E92			

KEY-POINTS:

2448 .R	2438 .A	253A ALOG	2642 AEXP	2642 EXP	2708 AIM
2476 .W	2898 .COMP	2880 \$4	2900 .RARG	2932 \$8	2950 .S
2964 .ZERO	2964 .ERCNT	2964 .O	2900 .MES	2446 .U	2A60 .V
2476 .AI					

MIN-BLOCKS:

DEFINED SUBROUTINES:

TRANSFER ADDRESS 2077

CUTION BEGINS:

STH= 14034.023
 SBH= 31831.496
 SB CRIT= 31767.121
 DITOP= 25.8324
 DIROT= 45.5962

END

PLC OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74 0

JOB HANDLING CHARGE	\$.35 / JOB	.35
25 LINES PRINTED PR2	\$ 1.25 / K LY	.11
54 CARDS READ	\$ 1.57 / K CD	.08
72 PLOTTER VECTORS	\$.25 / 1007	.00
14 MODEL 70 SECONDS	\$25.00 / HOUR	.10
22 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE \$.64

REIS 490 14731 LOGGED OUT 05/26/74 20:01. \$ 2.14 LEFT AFTER 33 LOGINS.

SUMMARY

Length	=	1000'
Beam	=	AL/5.75
Draft	=	B/3.3
Depth	=	AL/14
DST	=	18000 psi
S	=	60"
Material	-	ALUMINUM
TOP	=	4.53"
BOT	=	1.76"
STH	=	18034.02
SBH	=	31831.49
SBCRIT	=	31767.27
DITOP	=	25.83'
DIBOT	=	45.60'


```

04 C *AL* IS THE LENGHT
04 AL=1200*
04 B=AL/5.75
18 C *DPA* IS THE DRAFT
18 DPA=B/3*3
24 D=AL/14*
34 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
34 FOR ALUMINUM
34 POI=.33
38 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
38 FOR ALUMINUM
38 E=10**11**6
40 C *DST* IS THE DESIGN STRESS
40 DST=18000*
54 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
54 S=120*
60 C *BM* IS THE BENDING MOMENT
60 BM=(B*DPA*AL*AL*B.75)/(35*35.)
68 C (=0.30396*S*S*(1.-POI*POI))/E
74 A=9.*B**D/(140.*BM)
80 C=3.*B*D*D/(70.*BM)
84 F=3.*D*D*D/(140.*BM)
90 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
90 WRITE(5,10)
100 FORMAT('1',25X,'TOP ASSUMED LARGER THAN BOTTOM')
104 C *TOP* IS THE THICKNESS OF THE TOP PLATING
104 *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
104 BOT1=2*
110 1) TOP1=(B*BOT1*BOT1*BOT1)/(B+C)*G*DST*D*BOT1*BOT1)
110 WRITE(5,20)TOP1,BOT1
114 2) FORMAT('1',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
114 BOT1=BOT1+C*2
116 IF (TOP1.GT.7.) GO TO 30
120 GO TO 11
124 3) TOP2=3.5
124 BOT2=TOP2*(12.*E-(C+F)*DST*TOP2)/(DST*(A+C)*TOP2-12.*B)
124 WRITE(5,40)TOP2,BOT2
128 4) FORMAT('1',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
128 TOP2=TOP2+C*2
132 IF (TOP2.GT.6.) GO TO 50
136 GO TO 31
140 5) CONTINUE
144 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
144 WRITE(5,60)
148 6) FORMAT('1',25X,'BOTTOM ASSUMED LARGER THAN TOP')
148 BOT3=1*
152 6) TOP3=((B+C)*BOT3*BOT3*BOT3-D*G*DST*BOT3)/(P*G*DST)
152 WRITE(5,70)TOP3,BOT3
156 7) FORMAT('1',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
156 BOT3=BOT3+C*2
160 IF (BOT3.GT.3.5) GO TO 80
164 GO TO 61

```



```

072      *~ TOP4=1.
07A      *1 BOT4=(12.*(B+D)-(A+C)*DST*TOP4)/((C+F)*DST)
04E      *WRITE(5,90)TOP4, BOT4
072      *~ FORMAT(' ',25V,' TOP4=,F9.4,5X,' BOT4=,F9.4)
09A      TOP4=TOP4+0.2
0A6      IF(TOP4*GT.4.5)GO TO 100
0E2      GO TO 81
0EC      100 CONTINUE
0EC      END
(S)  *U      2400 [V] AL      0400 [V] B      0400 [V] DRA      0408 [V] D
(V)  *BOT    24E8 [V] E      2220 [G]  *R      04F4 [V] DST      04FC [V] S
(V)  *BY      2518 [V] G      2524 [V] A      0520 [V] C      053C [V] F
(L)  *10     2000 [S] AI     2540 [V] BOT1    0176 [L] 11      0548 [V] TOP1
(L)  *20     2220 [L] BI     255C [V] TOP2    0234 [L] 31      0564 [V] ROT2
(L)  *40     22FE [L] BI     2312 [L] 60      0578 [V] ROT3    0348 [L] 61
(V)  *TRP3   2338 [L] 70     2402 [L] 80      0580 [V] TOP4    040A [L] 81
(V)  *BOT4   2472 [L] 90     248C [L] 100     0200 [S] *V
REG      L
    
```

PROGRAM LABELS:

2770 *MAI *	2C1C *V	2A4C *COMP	2C2A *I	25FC *R	2B14 *Z
2A72 *A	263C *A	2A72 *A	26EE *W	27F6 EXP	297C AI
2EEE A LOG	2A84 *RARG	2E1A *0	27F6 AEXP	2B18 *ERCNT	2AE6 *8
2E13 *0	2E9E *C	2E46			

KEY-POINTS:

2EFC *B	263C *A	26EE A LOG	27F6 AEXP	27F6 EXP	297C AI
2EEE *V	2A4C *COMP	2A72 *A	2A84 *RARG	2AE6 *8	2B10 *5
2E14 *7ERC	2E18 *ERCNT	2E1A *0	2B70 *MES	2BFA *U	2C20 *V
2CA *T					

COMMON-BLOCKS:

E

DEFINED SUBROUTINES:

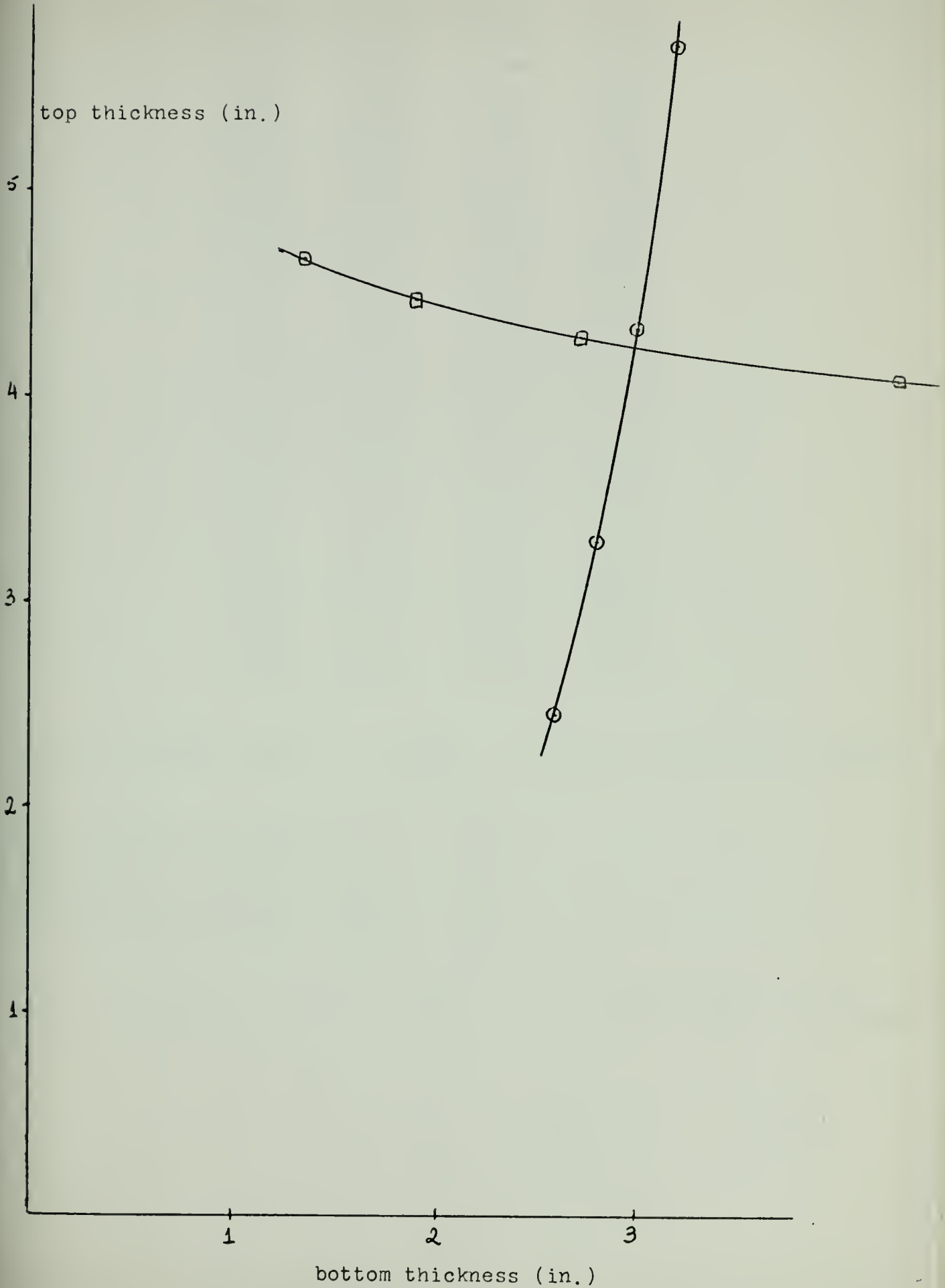
E

TRANSFER ADDRESS 2A70

SECTION BEGINS:

TOP ASSUMED LARGER THAN BOTTOM

TOP1=	3.0684	BOT1=	2.2000
TOP1=	1.2481	BOT1=	2.2000
TOP1=	1.8328	BOT1=	2.4000
TOP1=	2.4688	BOT1=	2.6000
TOP1=	3.2542	BOT1=	2.8000
TOP1=	4.3604	BOT1=	3.0000
TOP1=	5.7523	BOT1=	3.2000
TOP1=	7.6225	BOT1=	3.4000
TOP1=	11.1159	BOT1=	3.6000
TOP1=	13.8105	BOT1=	3.8000
TOP1=	19.2522	BOT1=	4.0000
TOP1=	27.8640	BOT1=	4.2000
TOP1=	43.2722	BOT1=	4.4000
TOP1=	32.2119	BOT1=	4.6000
TOP1=	252.5808	BOT1=	4.8000
TOP1=	-243.7314	BOT1=	5.0000
TOP1=	-117.7282	BOT1=	5.2000
TOP1=	-75.9876	BOT1=	5.4000
TOP1=	-59.1147	BOT1=	5.6000
TOP1=	-49.8724	BOT1=	5.8000
TOP1=	-44.2425	BOT1=	6.0000
TOP1=	-40.8128	BOT1=	6.2000
TOP1=	-37.8224	BOT1=	6.4000
TOP1=	-35.0977	BOT1=	6.6000
TOP1=	-34.2015	BOT1=	6.8000
TOP1=	-33.5500	BOT1=	7.2000
TOP2=	3.6000	BOT2=	-11.8971
TOP2=	3.7200	BOT2=	91.0296
TOP2=	3.8400	BOT2=	2.7662
TOP2=	4.1000	BOT2=	4.3496
TOP2=	4.2200	BOT2=	2.7380
TOP2=	4.5200	BOT2=	1.8850
TOP2=	4.7000	BOT2=	1.3454
TOP2=	4.9000	BOT2=	0.9655
TOP2=	5.1000	BOT2=	0.6778
TOP2=	5.3000	BOT2=	0.4483
TOP2=	5.5000	BOT2=	0.2578
TOP2=	5.7000	BOT2=	0.0947
TOP2=	5.9200	BOT2=	-0.0484



BOTTOM ASSIGNED	LARGER THAN	TOP
TOP3= 1.2000	BOT3=	1.2000
TOP3= 1.4000	BOT3=	1.2000
TOP3= 1.6000	BOT3=	1.4000
TOP3= 1.8000	BOT3=	1.6000
TOP3= 2.0000	BOT3=	1.8000
TOP3= 2.2000	BOT3=	2.0000
TOP3= 2.4000	BOT3=	2.2000
TOP3= 2.6000	BOT3=	2.4000
TOP3= 2.8000	BOT3=	2.6000
TOP3= 3.0000	BOT3=	2.8000
TOP3= 3.2000	BOT3=	3.0000
TOP3= 3.4000	BOT3=	3.2000
TOP4= 16.1639	BOT4=	16.1639
TOP4= 15.3920	BOT4=	15.3920
TOP4= 14.6201	BOT4=	14.6201
TOP4= 13.8482	BOT4=	13.8482
TOP4= 13.0763	BOT4=	13.0763
TOP4= 12.3044	BOT4=	12.3044
TOP4= 11.5324	BOT4=	11.5324
TOP4= 10.7605	BOT4=	10.7605
TOP4= 9.9886	BOT4=	9.9886
TOP4= 9.2167	BOT4=	9.2167
TOP4= 8.4448	BOT4=	8.4448
TOP4= 7.6729	BOT4=	7.6729
TOP4= 6.9010	BOT4=	6.9010
TOP4= 6.1290	BOT4=	6.1290
TOP4= 5.3571	BOT4=	5.3571
TOP4= 4.5852	BOT4=	4.5852
TOP4= 3.8133	BOT4=	3.8133
TOP4= 3.0414	BOT4=	3.0414

TWO

F VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

CP HANDLING CHARGE	\$.35 / JOB	.35
67 LINES PRINTED PR1	\$ 1.25 / K LN	.21
67 CARDS READ	\$ 1.50 / K Ch	.10
20 PLOTTER VECTORS	\$.25 / 1000	.00
20 MODEL 72 SECONDS	\$25.00 / HOUR	.14
20 MODEL 80 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE	\$.80

EIP 404 14731 LOGGED OUT 05/06/74 19:52. \$ 4.09 LEFT AFTER 30 LOGINS.


```

04 C *AL* IS THE LENGTH
06 AL=1000.
10 B=AL/5.75
14 C *DRA* IS THE DRAFT
16 DRA=4/3.5
20 D=AL/14.
24 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
26 FOR ALUMINUM
28 POI=0.33
32 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
34 FOR ALUMINUM
36 E=10.*10.**6
40 C *DST* IS THE DESIGN STRESS
42 DST=11200.
46 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
48 S=12.
52 C *DE* IS THE BENDING MOMENT
54 DE=(D*DRA*AL*(1.75))/(35.*35.)
56 DE=.31396*S**2*(1.-POI/EUIZE)
60 C *TOP* IS THE THICKNESS OF THE TOP PLATING
62 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
64 TOP=4.27
66 BOT=2.08
68 A=560.*D*(D*(D+D+TOP)
70 DE=D*(3.*D**2*(E*(D**2)*B*D*TOP*(TOP+BOT)+(D*D*TOP*TOP))
74 C *STH* IS THE STRESS ON TOP DUE TO BOWING
76 STH=A/DEH
78 WRITE(5,60)STH
80 FORMAT(' ',2F10.3)
82 C=560.*D*(B*(D+D+TOP)
84 C *SBH* IS THE STRESS ON BOT DUE TO BOWING
86 SBH=C/DEH
88 WRITE(5,80)SBH
90 FORMAT(' ',2F10.3)
94 C *SCRIT* IS THE CRITICAL STRESS ON BOT
96 SCRIT=BOT*BOT/E
98 WRITE(5,100)SCRIT
100 FORMAT(' ',1F10.3)
104 110 DE=2.*D*TOP+d*(TOP+BOT)
106 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
108 DITOP=D*(B*BOT+d*TOP)/D
110 WRITE(5,104)DITOP
112 FORMAT(' ',1F10.4)
114 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
116 DIBOT=D*(B*TOP+d*TOP)/D
118 WRITE(5,114)DIBOT
120 FORMAT(' ',1F10.4)
122 50 EN0
124
126 (S) *H 0324 (V) AL 0320 (V) B 0334 (V) DRA 0330 (V) D
128 (V) POI 0340 (V) E 0360 (V) .R 0358 (V) DST 0364 (V) S
130 (V) DRA 0370 (V) G 0368 (V) TOP 0352 (V) BOT 0398 (V) A
132 (V) DEH 0380 (V) STH 0374 (V) 60 0000 (S) @I 0300 (V) C

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STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

(V) SRH	0112(L) 90	300(V) SPCIT	0228(L) 110	0300(V) P
(V) TITOP	2745(L) 40	300(V) DIPOT	0304(L) 50	0000(S) .V

GRAM LABELS:

70 MATI	2448 .V	2498 .COMP	2476 .I	2448 .R	2960 .Z
EE	2448 .A	2980 .YES	283A .W	2642 EXP	2708 AI
SA ALOC	2920 .FARG	2950 .S	2642 EXP	2964 .ERCNT	2932 .S
44 .O	273A .L	2992			

KEY-PATHS:

44 .R	2498 .A	2937 .ALOG	2642 EXP	2642 EXP	2708 AI
28A .	2898 .COMP	2998 .A	2940 .RARG	2932 .S	2950 .S
92 .-FED	2964 .ERCNT	2966 .O	2980 .YES	2746 .U	2800 .V

MINI-LOCATIONS:

DEFINITION LINES:

MAPPER ADDRESS 2270
LOCATION BEGINS:

SH= 17950.723
 SE= 22852.766
 SPCIT= 22760.008
 TITOP= 31.4316
 DIPOT= 39.9071

MINI OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 5/06/74

08 HANDLING CHARGE	\$.25 / JOB	.35
05 LINES PRINTED PRI	\$ 1.25 / LINE	.11
04 CARDS READ	\$ 1.50 / 1000	.08
02 PLOTTER VECTORS	\$.25 / 1000	.01
16 MODEL 70 SECONDS	\$25.00 / HOUR	.11
02 MODEL 80 SECONDS	\$12.50 / HOUR	.02
	TOTAL CHARGE \$.65

DIR 49 14731 LOGGED OUT 25/06/74 20:03. \$ 1.49 LEFT AFTER 34 LOGINS.

SUMMARY

Length = 1000'
Beam = AL/5.75
Draft = B/3.3
Depth = AL/14
DST = 18000 psi
S = 120"
Material - ALUMINUM
TOP = 4.27"
BOT = 2.98"
STH = 17958.79
SBH = 22852.76
SBCRIT = 22768.12
DITOP = 31.43'
DIBOT = 39.99' or 40.00'

Looking at the following table we can also observe for the aluminum the effect of changing S on the neutral axis location and the compressive stresses resulting on the bottom plating.

CASE	S inches	TOP inches	BOT inches	SBH psi	SBCRIT psi	DITOP ft	DIBOT ft
1	10	5.35	0.363	53000	49000*	18.09	53.34
2	30	4.86	1.00	41000	41000	21.72	49.71
3	60	4.53	1.76	32000	32000	25.83	45.60
4	120	4.27	2.98	23000	23000	31.43	40.00

(*) This difference may result from inaccuracy in reading the plot.

Here again the values of TOP and BOT may allow for a quick evaluation of the weight relationship between the four cases.

One run was also made in the case of the aluminum for AL=500 ft and the results are here included.


```

004 C *AL* IS THE LENGTH
004 AL=500.
006 P=AL/2.
008 C *DRA* IS THE DRAFT
008 DRA=6/3.
010 L=AL/9.
012 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
012 FOR ALUMINUM
014 POI=0.33
016 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
016 FOR ALUMINUM
018 E=1.0*10**6
020 C *DST* IS THE DESIGN STRESS
020 DST=10**6.
022 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
022 S=30.
024 C *M* IS THE BENDING MOMENT
024 M=(D+DRA*AL*AL*0.75)/(35.*35.)
026 G=.37396*S*S*(1.-POI*POI)/E
028 A=9.0**3*D/(147.*BM)
030 C=3.*E*D/(7.*M)
032 F=3.*D*M/(127.*BM)
034 C THICKNESS OF TOP ASSUMED LARGER THAN BOTTOM THICKNESS
034 WRITE(5,10)
036 10 FORMAT('1',25X,'TOP ASSUMED LARGER THAN BOTTOM')
038 C *TOP* IS THE THICKNESS OF THE TOP PLATING
040 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
042 BOT1=0.1
044 11 TOP1=(M*BOT1*BOT1*BOT1)/((B+D)*G*DST-D*BOT1*BOT1)
046 WRITE(5,20)TOP1,BOT1
048 20 FORMAT('1',25X,'TOP1='F9.4,5X,'BOT1='F9.4)
050 BOT1=BOT1+0.25
052 IF (BOT1.GT.1.)GO TO 30
054 GO TO 11
056 30 TOP2=0.1
058 31 BOT2=(TOP2*(12.*E-(C+F)*DST*TOP2))/(DST*(A+C)*TOP2-12.*B)
060 WRITE(5,40)TOP2,BOT2
062 40 FORMAT('1',25X,'TOP2='F9.4,5X,'BOT2='F9.4)
064 TOP2=TOP2+0.25
066 IF (TOP2.GT.1.)GO TO 50
068 GO TO 31
070 50 CONTINUE
072 C THICKNESS OF BOTTOM ASSUMED LARGER THAN TOP
074 WRITE(5,60)
076 60 FORMAT('1',25X,'BOTTOM ASSUMED LARGER THAN TOP')
078 BOT3=0.1
080 61 TOP3=((B+D)*BOT3*BOT3*BOT3-D*G*DST*BOT3)/(B*G*DST)
082 WRITE(5,70)TOP3,BOT3
084 70 FORMAT('1',25X,'TOP3='F9.4,5X,'BOT3='F9.4)
086 BOT3=BOT3+0.25
088 IF (BOT3.GT.1.)GO TO 80
090 GO TO 61

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STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

```

100 10P4=8.1
101 BOT4=(12.*(H+D)-(A+C).DST*TOP4)/((C+F)*DST)
102 WRITE(9,33)TOP4,BOT4
103 FORMAT(' ',25X,'TOP4='F9.4,5X,'BOT4='F9.4)
104 TOP4=TOP4+(.025)
105 IF(TOP4*.3T.1.)GO TO 100
106 GO TO 31
107 CONTINUE
108 END

```

(5) 0.0	0400 [V] AL	0400 [V] B	0400 [V] DRA	0408 [V] D
(6) BOT	0408 [V] E	0408 [V] .R	0404 [V] DST	04FC [V] S
(7) 0.0	0518 [V] C	0524 [V] A	0520 [V] C	0534 [V] F
(8) 0.0	0600 [V] AI	0632 [V] BOT1	0676 [L] 11	0644 [V] TOP1
(9) 0.0	0800 [V] 30	0852 [V] TOP2	0924 [L] 31	0854 [V] BOT2
(10) 0.0	0800 [L] 50	0912 [L] 60	0924 [V] BOT3	0948 [L] 61
(11) 0.0	0900 [L] 70	0912 [L] 80	0924 [V] TOP4	0948 [L] 81
(12) 0.0	0972 [L] 90	0912 [L] 100	0924 [S] .V	

PARAM LABELS:

2004 .V	2A34 .COMP	2012 .I	25E4 .R	2AFC .Z
2024 .V	2A50 .MES	20D6 .J	27DF EXP	2964 AI
2A5C .RARG	2AFA .R	27DF AEXP	2B00 .FRONT	2ACE #8
2016 .V	202E			

KEY POINTS:

2624 .V	26D6 /LOG	27DE /EXP	27DE EXP	2964 AI
2A34 .COMP	2A5A #4	2A9C .RARG	2ACE #8	2AF8 .5
2B00 .FRONT	2E32 .R	2058 .MES	2BE2 .U	2C78 .V
012 AI				

NON-BLOCKS:

E

DEFINED SUBROUTINES:

E

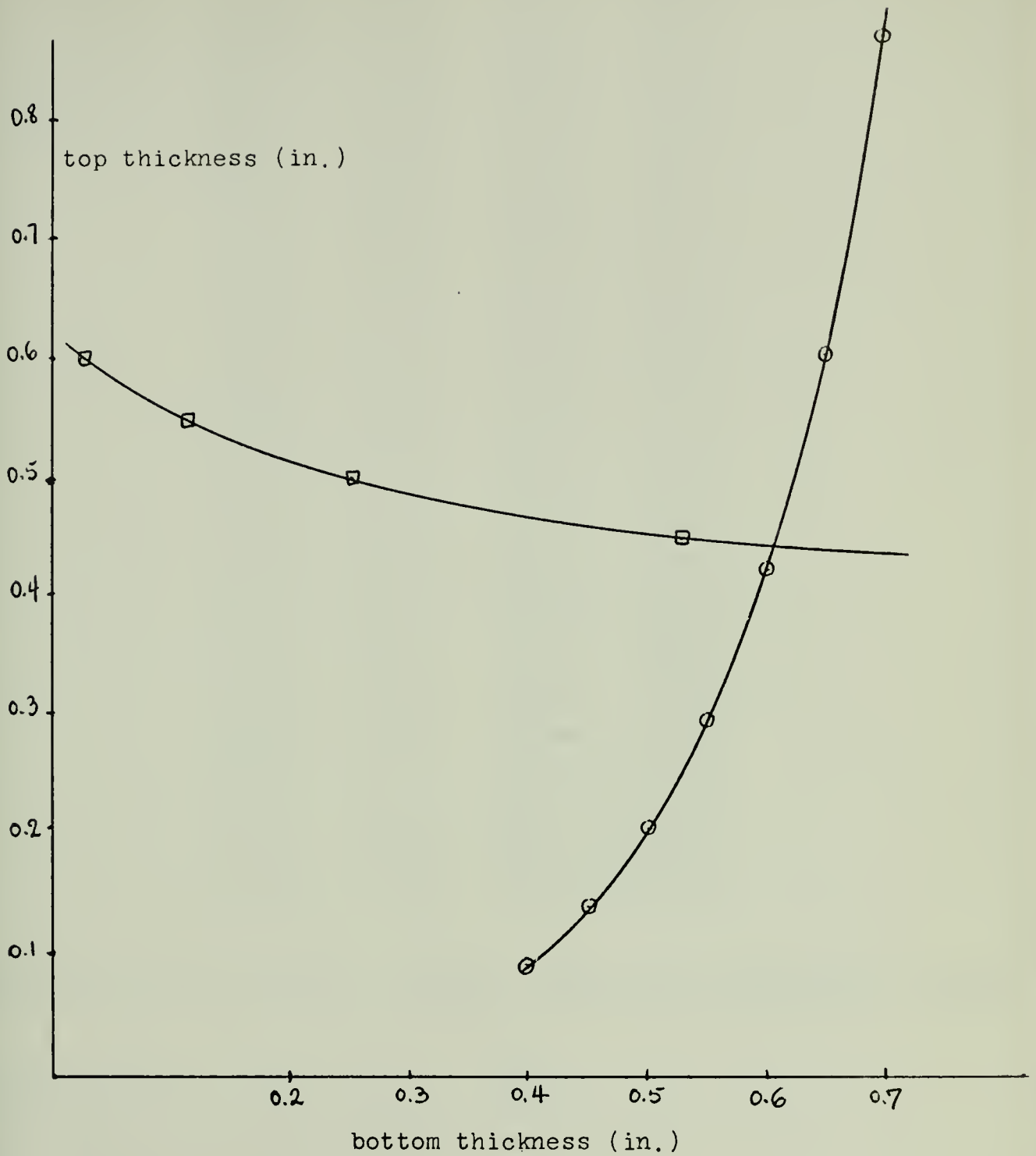
TRANSFER ADDRESS 2070

SECTION BEGINS:

TOP ASSUMED LARGER THAN BOTTOM

TOP1=	4.0042	BOT1=	0.1000
TOP1=	4.0142	BOT1=	0.1500
TOP1=	4.0161	BOT1=	0.2000
TOP1=	4.0262	BOT1=	0.2500
TOP1=	4.0321	BOT1=	0.3000
TOP1=	4.0566	BOT1=	0.3500
TOP1=	4.0852	BOT1=	0.4000
TOP1=	4.1414	BOT1=	0.4500
TOP1=	4.2161	BOT1=	0.5000
TOP1=	4.2871	BOT1=	0.5500
TOP1=	4.4245	BOT1=	0.6000
TOP1=	4.6169	BOT1=	0.6500
TOP1=	4.8722	BOT1=	0.7000
TOP1=	4.8829	BOT1=	0.7500
TOP1=	4.9667	BOT1=	0.8000
TOP1=	5.2969	BOT1=	0.8500
TOP1=	6.6463	BOT1=	0.9000
TOP1=	22.2169	BOT1=	0.9500
TOP1=	-16.6511	BOT1=	1.0000
TOP2=	4.1160	BOT2=	-0.1017
TOP2=	4.1564	BOT2=	-0.1687
TOP2=	4.2167	BOT2=	-0.2587
TOP2=	4.2867	BOT2=	-0.3997
TOP2=	4.3767	BOT2=	-0.6955
TOP2=	4.4767	BOT2=	-2.1234
TOP2=	4.4867	BOT2=	1.7868
TOP2=	4.4867	BOT2=	0.5333
TOP2=	4.5167	BOT2=	0.2515
TOP2=	4.5767	BOT2=	0.1143
TOP2=	4.6167	BOT2=	0.0258
TOP2=	4.6567	BOT2=	-0.0403
TOP2=	4.7167	BOT2=	-0.0944
TOP2=	4.7867	BOT2=	-0.1413
TOP2=	4.8167	BOT2=	-0.1835
TOP2=	4.8567	BOT2=	-0.2224
TOP2=	4.9167	BOT2=	-0.2591
TOP2=	4.9567	BOT2=	-0.2941
TOP2=	1.0167	BOT2=	-0.3277





Intersection not accepted; assumed Top > Bot ended up with Bot > Top.

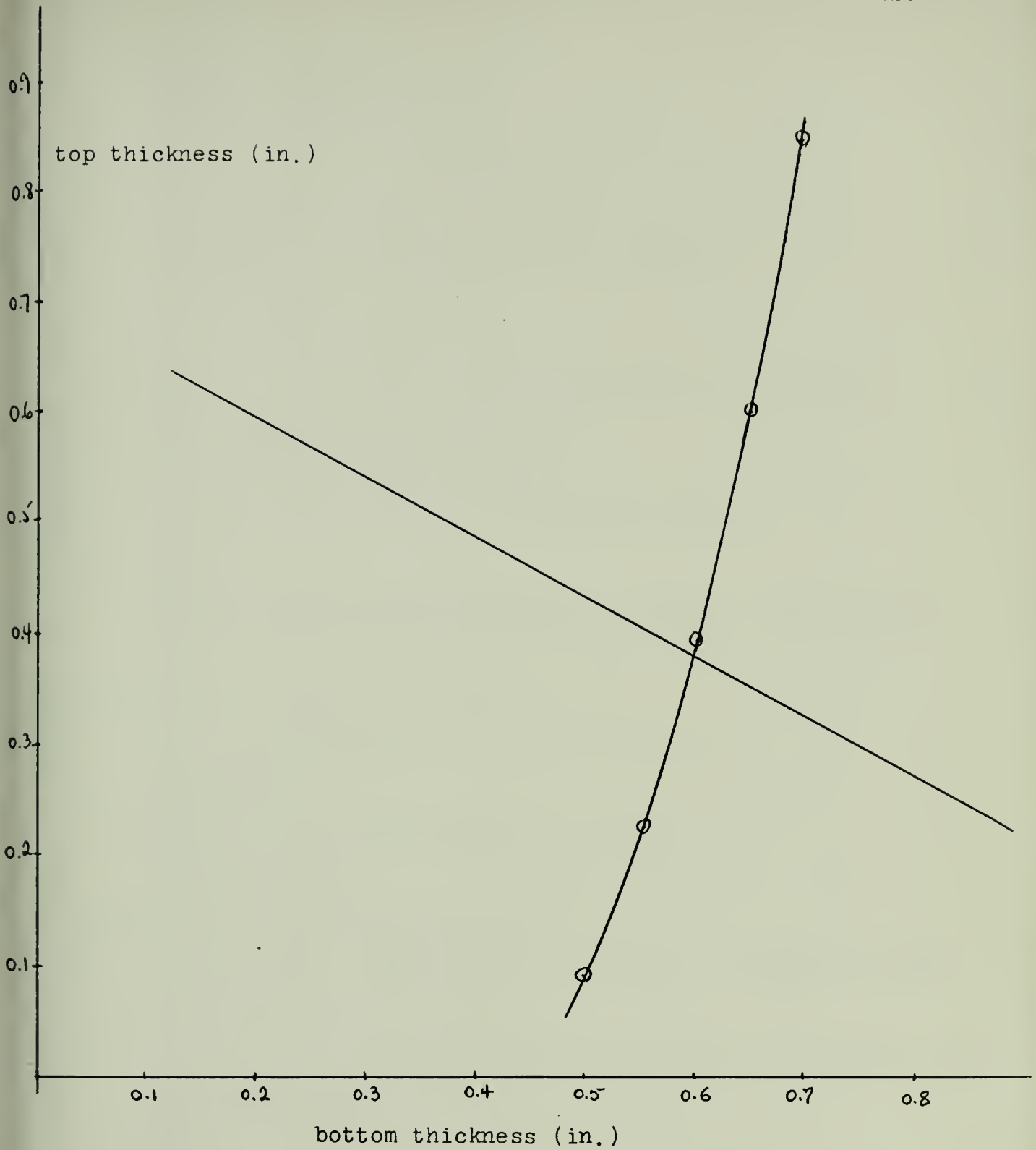
BOTTOM	ASSUMED	LARGER THAN	TOP
TOP3=	-0.0000	BOT3=	0.1000
TOP3=	-0.1100	BOT3=	0.1500
TOP3=	-0.1400	BOT3=	0.2000
TOP3=	-0.1500	BOT3=	0.2500
TOP3=	-0.1800	BOT3=	0.3000
TOP3=	-1.1000	BOT3=	0.3500
TOP3=	-3.0000	BOT3=	0.4000
TOP3=	-0.0077	BOT3=	0.4500
TOP3=	0.0056	BOT3=	0.5000
TOP3=	1.0073	BOT3=	0.5500
TOP3=	0.0048	BOT3=	0.6000
TOP3=	0.6044	BOT3=	0.6500
TOP3=	0.8543	BOT3=	0.7000
TOP3=	1.1404	BOT3=	0.7500
TOP3=	1.4900	BOT3=	0.8000
TOP3=	1.8001	BOT3=	0.8500
TOP3=	2.0001	BOT3=	0.9000
TOP3=	2.0043	BOT3=	0.9500
TOP3=	3.4108	BOT3=	1.0000
TOP4=	0.1000	BOT4=	1.1275
TOP4=	0.1500	BOT4=	1.2344
TOP4=	0.2000	BOT4=	0.9414
TOP4=	0.2500	BOT4=	0.2484
TOP4=	0.3000	BOT4=	0.7554
TOP4=	0.3500	BOT4=	0.6623
TOP4=	0.4000	BOT4=	0.5693
TOP4=	0.4500	BOT4=	0.4763
TOP4=	0.5000	BOT4=	0.3832
TOP4=	0.5500	BOT4=	0.2902
TOP4=	0.6000	BOT4=	0.1972
TOP4=	0.6500	BOT4=	0.1042
TOP4=	0.7000	BOT4=	0.0111
TOP4=	0.7500	BOT4=	-0.0819
TOP4=	0.8000	BOT4=	-0.1749
TOP4=	0.8500	BOT4=	-0.2680
TOP4=	0.9000	BOT4=	-0.3610
TOP4=	0.9500	BOT4=	-0.4540
TOP4=	1.0000	BOT4=	-0.5470

END

F VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/06/74

JOB HANDLING CHARGE	\$ 0.35 / JOB	0.35
173 LINES PRINTED PR1	\$ 1.05 / K LM	0.22
67 CARDS READ	\$ 1.50 / K CD	0.1
30 PLOTTER VECTORS	\$ 0.25 / 1000	0.07
22 MODEL 70 SECONDS	\$25.00 / HOUR	0.14
22 MODEL 80 SECONDS	\$12.50 / HOUR	0.00
TOTAL CHARGE	\$	0.81

FEIR 460 14731 LOGGED OUT 05/06/74 20:09. \$ 0.68 LEFT AFTER 05 LOGINS.



this intersection is acceptable Bot > Top


```

04 C *AL* IS THE LENGTH
04 AL=500.
08 F=AL/8.
12 C *DRA* IS THE DRAFT
18 DRA=F/3.
24 D=AL/9.
32 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
32 FOR ALUMINUM
34 POI=.33
38 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
38 FOR ALUMINUM
39 E=10.**10.**6
40 C *DST* IS THE DESIGN STRESS
40 DST=12000.
54 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
54 S=30.
60 C *BM* IS THE BENDING MOMENT
60 BM=(B*DRA*AL*AL*(0.75))/(35.**35.)
68 G=.36326*S*S*(1.-POI+POI)/E
74 C *TOP* IS THE THICKNESS OF THE TOP PLATING
74 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
74 TOP=.38
80 BOT=.6
84 A=562.**BM*(B*BOT+D*BOT)
90 DEN=G*(B.**B*B*TOP*BOT+2.**B*D*BOT*(TOP+BOT)+(D*D*BOT*BOT))
94 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
94 C STH=A/DEN
98 WRITE(5,42)STH
104 FORMAT(' ',20,' ',STH='F10.3)
108 C=562.**BM*(B*TOP+D*BOT)
114 C *SBH* IS THE STRESS ON BOT DUE TO HOGGING
114 C SBH=C/DEN
118 WRITE(5,90)SBH
124 FORMAT(' ',20,' ',SBH='F10.3)
130 C *SCRIT* IS THE CRITICAL STRESS ON BOT
130 SCRIT=BOT*BOT/G
134 WRITE(5,110)SCRIT
140 FORMAT(' ',17,' ',SCRIT='F10.3)
144 P=2.**D*BOT+B*(TOP+BOT)
150 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
150 DITOP=D*(B*BOT+D*BOT)/P
154 WRITE(5,40)DITOP
160 FORMAT(' ',18,' ',DITOP='F10.4)
164 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
164 DIBOT=D*(B*TOP+D*BOT)/P
168 WRITE(5,50)DIBOT
174 FORMAT(' ',18,' ',DIBOT='F10.4)
180 END
[5] *II 2324 [V] AL 2320 [V] B 2334 [V] DRA 2330 [V] D
[V] POI 2340 [V] E 2000 [G] .R 2358 [V] DST 2360 [V] S
[V] BV 2370 [V] G 2388 [V] TOP 2390 [V] BOT 2398 [V] A
[V] DEN 2388 [V] STH 2174 [L] 60 2000 [S] @I 2380 [V] C

```


STATEMENT OF SHIPS-ON-THE-NEUTRAL-AXIS LOCATION

(V) S04 W1F2(L) 90 0304(L) SBCRIT 0228(L) 110 0308(V) P
 (V) DITOP 22A8(L) 40 0300(V) DIFOT 0304(L) 50 0000(S) .V
 EU L

GRAM LABELS:

2444 .V	2894 .COMP	2172 @I	2444 .R	2950 .ZE
2424 .A	2982 .MES	2936 .W	263F EXP	2704 AII
25FC .DARG	2958 .S	263E AEXP	2960 .ERCNT	292E \$8
2A36 .L	298E			

BY-PORT TR:

2444 .S	2894 .COMP	2536 A1 06	263E AEXP	263F EXP	2704 AII
2936 .	2982 .MES	298A \$6	29FC .RARG	292E \$8	2958 .S
2960 .S	2962 .S	2938 .MES	2A42 .U	2A68 .V	

MINIMUMS:

DEFINES & PROCLINES:

STEP ADDRESS 2070
 TION BEGINS:

STH= 18980.664
 SBH= 14570.137
 SBCRIT= 14767.777
 DITOP= 34.7137
 DIFOT= 24.7119

VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/26/74

06 HANDLING CHARGE	\$.25 / JOB	.35
08 LINES PRINTED PPI	\$ 1.00 / K L	.11
04 CARDS READ	\$ 1.00 / K C	.08
04 PLOTTER VECTORS	\$.25 / 1000	.00
16 MODEL 70 SECONDS	\$25.00 / HOUR	.11
04 MODEL 80 SECONDS	\$12.00 / HOUR	.00
TOTAL CHARGE \$.65

IF 451 14731 LOGGED OUT 05/26/74 20:13. \$.03 LEFT AFTER 36 LOGINS.

SUMMARY

Length	=	500'
Beam	=	AL/8
Draft	=	B/3
Depth	=	AL/9
DST	=	18000 psi
S	=	30"
Material	-	ALUMINUM
TOP	=	0.38"
BOT	=	0.6"
STH	=	18089.66
SBH	=	14578.14
SBCRIT	=	14767.85
DITOP	=	30.76"
DIBOT	=	24.79"

HOGGING BENDING MOMENT=SAGGING BENDING MOMENT

DESIGN STRESS IN TENSION=DESIGN STRESS IN COMPRESSION

From the preceding section we can understand the need to put some limit on the value of the compressive stresses, and perhaps also the inefficiency of considering the value of S (the spacing between longitudinal stiffening members) as an input to the problem.

In this section we will consider a bending moment of the same magnitude in both hogging and in sagging and the stress in either tension or compression will have as upper limit the design stress.

Since the bending moment is the same, it is intuitive that the thickness required to resist tension on top and compression on bottom (buckling) when the hogging moment is acting will also be respectively the thicknesses to resist tension on bottom and buckling on top when we consider the sagging moment. We would end up then with a thickness on top required by the hogging moment $TOPH$, a thickness on bottom required by the hogging moment, as well as a thickness on top required by the sagging moment $TOPS$ and a thickness on bottom required by the sagging moment $BOTS$.

Since $HM = SM$ we must obtain

$TOPS = BOTH = t_1$ and $BOTS = TOPH = t_2$

and we will end up considering the largest t_1 or t_2 , to be the thickness of the top and bottom simultaneously.

Assume for instance that $t_1 > t_2$ then if we decide on t_1 for

thickness of the top as well as at the bottom, we will verify that we have overdesigned because TOPS = t1 corresponds to BOTS = t2 which plugged into the expression for I gives a smaller value than using TOPS = t1 with BOTH = t1 that will give a larger value for I; this larger I value when used in

$$STR = \frac{BM*Y}{I}$$

will give a smaller STR value than the design stress and we could use a smaller thickness, may be between t1 and t2 for both the top and bottom and satisfy the DST requirement.

This means that we must incorporate in this analysis the fact already known that when BM (hogging) is equal to BM (sagging) we will have to consider the neutral axis half way between top and bottom which means that the thickness of the top shall be equal to the thickness of the bottom for a geometrically symmetrical cross section.

Let's determine the formulas that will best serve this case then:

$$HM = SM = BM$$

$$DITOP = DIBOT = D*(B*t+D*t)/(2*D*t+B*(2*t))$$

$$t = TOP = BOT = TS$$

$$I = 2*D*D*t* \frac{3*B*B+D*D+4*B*D}{D+B}$$

Now we can write

$$STR = \frac{BM*DITOP}{I} = \frac{BM*DIBOT}{I} = DST$$

and the instability formula

$$t = S \sqrt{0.30396 \frac{1 - \text{POI} * \text{POI}}{E} * \text{DST}}$$

Where S is the spacing between longitudinals.

We see thus that we cannot have more than two equations and so we must seek for two variables. A quick inspection and one can see that knowing BM, B, D, the material properties and the design stress DST we have left to determine what should be the thickness of the plating (t) and the corresponding frame spacing (S). Solving the equations with this in mind, we obtain

$$\text{BM} = 1.37143 * \text{B} * \text{T} * \text{AL} * \text{AL} \quad \text{lb*ft}$$

and from $\text{STR} = \frac{\text{BM} * \text{Y}}{\text{I}}$ we obtain

$$\text{DST} = \frac{\text{BM} * (\text{D} + \text{B})}{4 * \text{D} * \text{t} * (3 * \text{B} * \text{B} + \text{D} * \text{D} + 4 * \text{B} * \text{D})}$$

which gives

$$t = \frac{\text{BM} * (\text{D} + \text{B})}{4 * \text{D} * \text{DST} * (3 * \text{B} * \text{B} + \text{D} * \text{D} + 4 * \text{B} * \text{D})}$$

and also

$$S = \frac{t}{\sqrt{0.30396 \frac{(1 - \text{POI} * \text{POI}) * \text{DST}}{E}}}$$

These formulas will give for a prescribed set of dimensions (ft) and a particular material subject to bending BM (lb*ft) the thickness t (inches) and the spacing of longitudinal stiffening members S (inches) required.

HOGGING MOMENT AND SAGGING MOMENT OF
DIFFERENT MAGNITUDES
THE DESIGN STRESS IS THE SAME IN TENSION AND
IN COMPRESSION

Let's consider now the case where the magnitude of HM is different than the value for SM.

We have in this case:

$$DITOP = \frac{D*(B*BOT+D*TS)}{2*D*TS+B*(TOP+BOT)}$$

$$DIBOT = \frac{D*(B*TOP-D*TS)}{\text{SAME DENOMINATOR}}$$

$$I = 4*D*D* \frac{3*B*B*TOP*BOT+(D*TS)^2+2*B*D*TS*(TOP+BOT)}{\text{SAME DENOMINATOR}}$$

$$STH = \frac{HM*DITOP}{I}$$

$$STS = \frac{SM*DITOP}{I}$$

$$SBH = \frac{HM*DIBOT}{I}$$

$$SBS = \frac{SM*DIBOT}{I}$$

$$BOT = \sqrt{G} \sqrt{SBH}$$

$$TOP = \sqrt{G} \sqrt{STS}$$

where $G = 0.30396* \frac{S^2*(1-POI*POI)}{E}$

Looking at these equations, we know that the result obtained for the thicknesses is going to determine the value of DITOP, DIBOT and I. If furthermore we restrain the stresses from being larger than DST we will undoubtedly want them to end up being DST otherwise if they are smaller we are not using the material efficiently.

We thus have

$$DST = STH = HM * K1 \quad DST = STS = SM * K1 \quad \text{where } K1 = DITOP/I$$

$$DST = SBH = HM * K2 \quad DST = SBS = SM * K2 \quad \text{where } K2 = DIBOT/I$$

This tells us that if for instance $HM > SM$

we should be considering $DST = STH = HM * K1$

$$\text{and } DST = SBH = HM * K2$$

because the other two equations will determine lower stresses and are thus within the requirement of not exceeding the design stress, being automatically satisfied.

Now from these equations we can see that if the design stress required for the top, (in tension for the case of $HM > SM$; note that for the case $SM > HM$ the other two equations should be considered and the conclusions are similar), is going to be the same one as required for the bottom (in compression), we must have $K1 = K2$ and so $DITOP = DIBOT$ and the neutral axis will again end up being located halfway between top and bottom for a geometrically symmetrical cross section.

We can here use the same two equations derived in the previous section to determine (t) and (S). The moment to be considered here will be the larger of HM or SM instead of BM.

HOGGING MOMENT LARGER THAN SAGGING MOMENT

DESIGN STRESS IN COMPRESSION LARGER THAN DESIGN STRESS IN TENSION

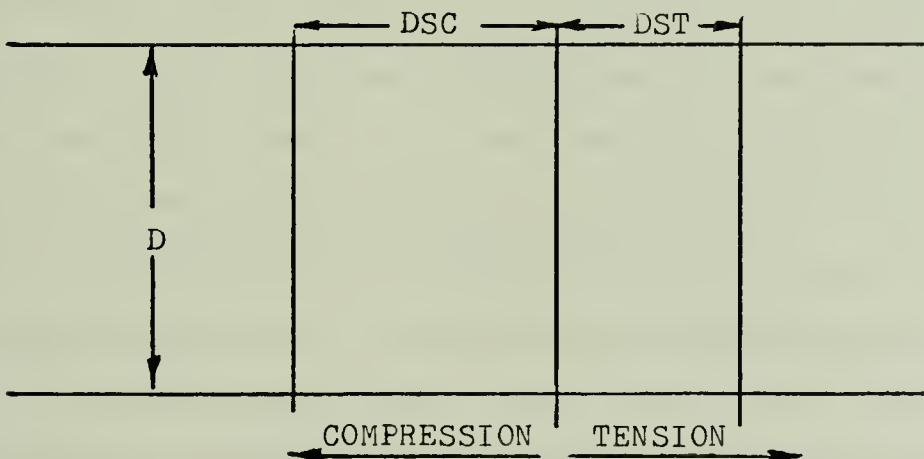
The situation we are analysing in this section

$$HM > SM$$

$$DSC > DST$$

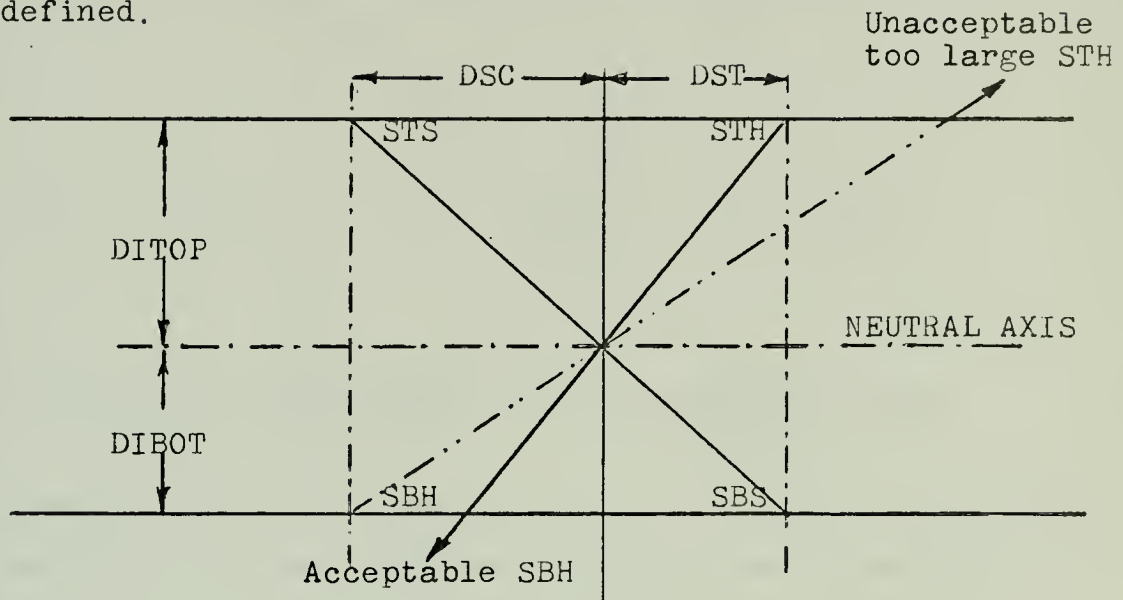
is perhaps more interesting than the previous ones because its study provided us with a much simpler method for determining the thicknesses and the neutral axis location; we call here the attention of the reader to fully understand the steps of this section because the following sections till the end of this work will use this same method of analysis applied to their own particular case.

Indicated in the next diagram are the ranges within which we may stress the material in order to satisfy the requirement of not exceeding DSC or DST.



Let's consider now that we will try to determine the best position for the neutral axis by first gaining some insight into

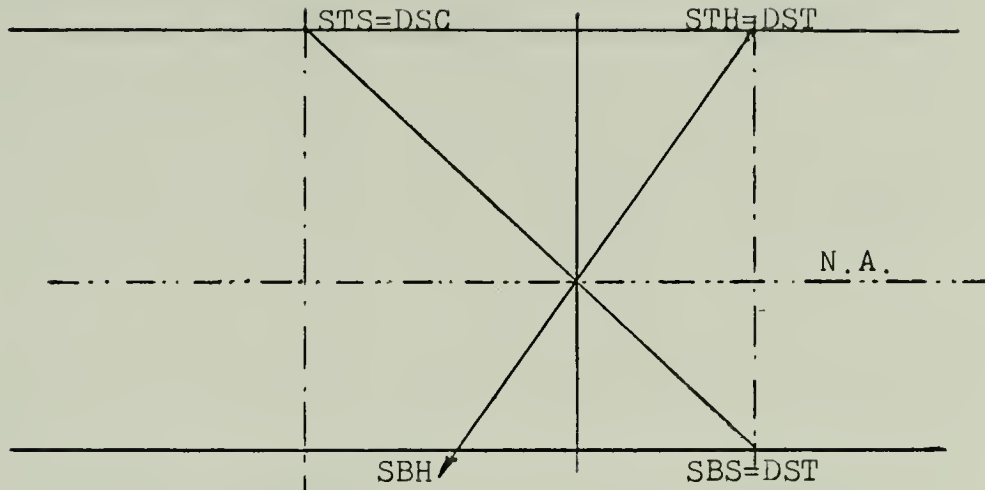
the geometry of the stress distributions and the stress limits already defined.



From the previous diagram we may easily understand that if we decide to stress the material to its maximum in tension and compression when under the effect of a sagging bending moment (SBS - STS) we end up with a neutral axis closer to the bottom due to the fact that $DST < DSC$ and we see that when considering the hogging moment the only possible alternative left is to limit the stress caused by this moment in tension on top $STH = DST$; in this case the compressive stress on the bottom SBH would automatically fall inside the acceptable range.

If for the hogging moment we choose to allow the stress in compression on the bottom to attain its maximum value $SBH = DSC$ we see from the diagram that we are forcing the STH to fall outside of the acceptable range. We are left in this situation with an acceptable solution so far and we must investigate it further.

At this stage we have



We may see from the diagram that we are asking for $SBS=STH=DST$ to have the same value and at the same time to have the neutral axis located closer to the bottom $DITOP > DIBOT$.

We note that the equations that give STH and SBS are

$$STH = HM \cdot DITOP / I$$

and $SBS = SM \cdot DIBOT / I$

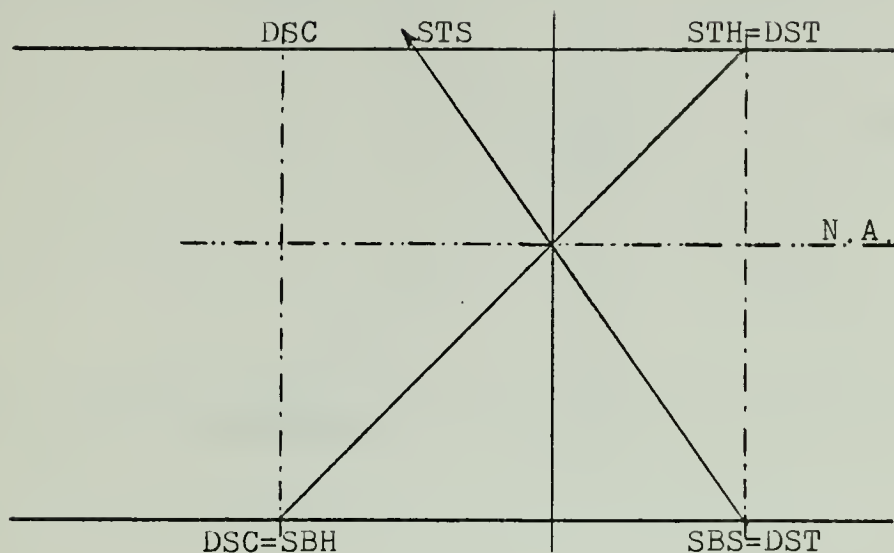
and since I is the same STH can never be of the same magnitude as SBS because

$$STH = HM \cdot DITOP / I > SBS = SM \cdot DIBOT / I$$

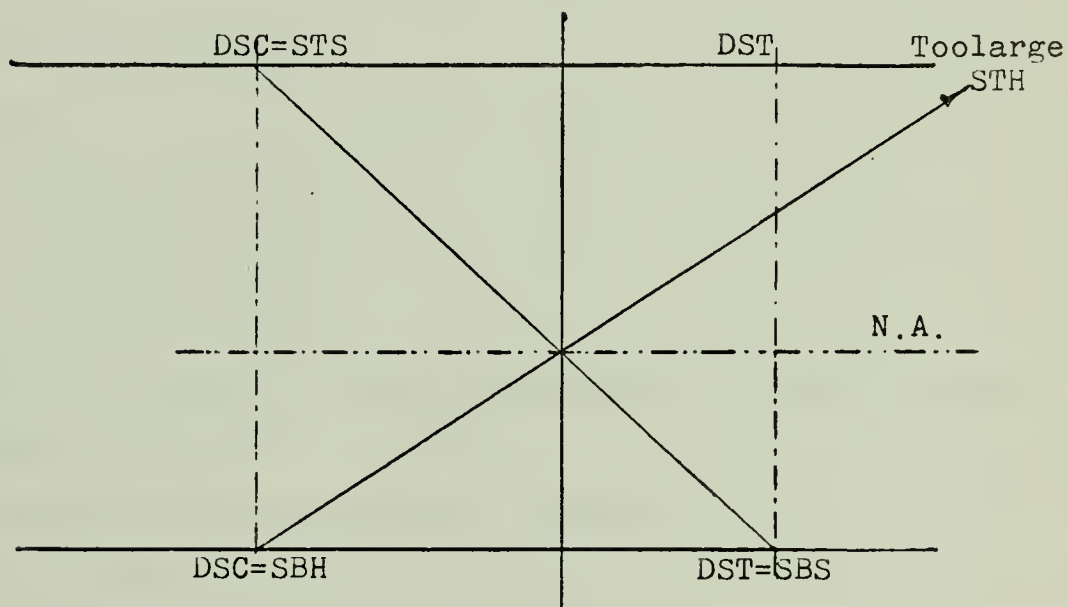
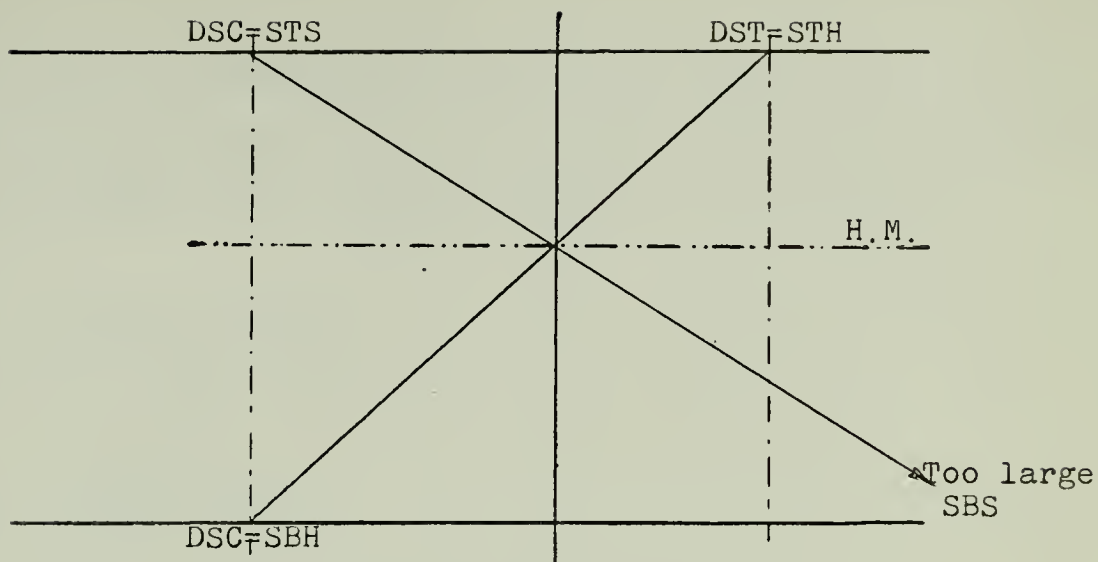
both because $DITOP$ is being set larger than $DIBOT$ and also because we are considering $HM > SM$ to begin with. Thus we are left with the alternative of making $DIBOT$ larger than $DITOP$ to make it possible to satisfy

$$STH = HM \cdot DITOP / I = SBS = SM \cdot DIBOT / I$$

The situation would then be



Besides this is only natural because what we are doing here is to make the material to be stressed to its allowable limit while resisting the larger of the two moments; hogging in this case. This situation will be worked out here as an example and it seems to be quite clear that similarly, cases where $SM > HM$ or $DST > DSC$ or other situation like $DST > DSC$ with $HM > SM$ or $DST < DSC$ with $SM > HM$ could be solved applying a similar reasoning. So, having $HM > SM$ and $DSC > DST$ we should consider the smaller design stress, in this case DST , and have the material stressed to this upper limit. Why?, because if the larger is considered, DSC in this case, there is no way we can have the material stressed to this higher limit in both hogging and sagging while considering the smaller we may attempt to do so. This can be better understood from the two next diagrams.



So, considering the lower design stress, in this case DST, we will stress the material to this maximum in both hogging and sagging thus giving

$$STH = HM \cdot DITOP / I = DST$$

$$SBS = SM \cdot DIBOT / I = DST$$

Note here that we already know that the neutral axis will

be closer to the top $DITOP < DIBOT$ which means that we must obtain a thickness for the plating at the top larger than at the bottom and so $TS=TOP$ which means that we will be using the following formulas

$$DITOP = \frac{D*(B*BOT+D*TOP)}{P}$$

$$DIBOT = \frac{D*TOP*(B+D)}{P}$$

$$I = \frac{4*D^2*TOP*((3*B^2+2*B*D)*BOT+(D^2+2*B*D)*TOP)}{P}$$

where $P = (2*D+B)*TOP+B*BOT$

we have also

$$STH = DST = HM*DITOP/I$$

$$SBS = DST = SM*DIBOT/I$$

$$SBH = DSC = HM*DIBOT/I$$

From these three equations we may combine the two first to obtain

$$HM*DITOP = SM*DIBOT$$

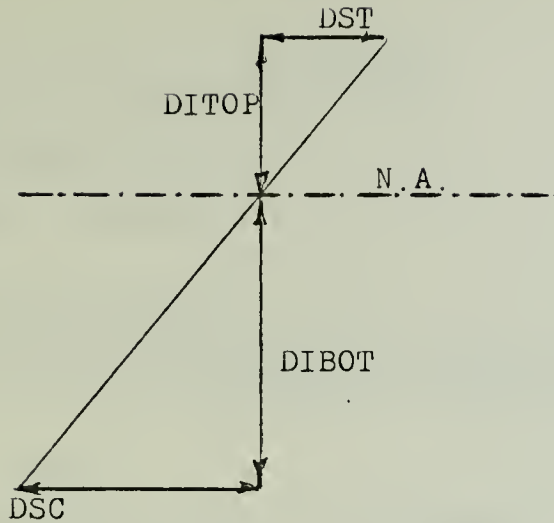
and combining the second and third we obtain

$$DST*HM = DSC*SM$$

After combining these two equations we obtain

$$DST*DIBOT = DSC*DITOP$$

This equation could have been obtained more easily just by looking at the diagram we are considering and notice the similar triangles



Note also that what we obtain from this equation is to determine the N.A. location by making DITOP and DIBOT proportional to DST And DSC.

Proceeding with this equation we obtain:

$$DST*(D*TOP*(B+D))/P + DSC*(D*B*BOT+D^2*TOP)/P$$

$$TOP*(DST*B+DST*D-DSC*D) = DSC*B*BOT$$

and making

$$AC = DST/DSC+D*(DST-DSC)/(DSC*B)$$

we obtain

$$BOT = AC*TOP$$

We need now an equation to stress the material to its design limit and this can be done by either using

$$STH = DST = HM*DITOP/I$$

or $SBH = DSC = HM*DIBOT/I$

From these two equations I prefer to use the second especially because it will simplify the calculations that follow as can be understood from the simpler expression that gives DIBOT as

compared with the one that gives DITOP.

We will then consider here

$$SBH = DSC = HM * DIBOT / I$$

which gives

$$DSC = \frac{HM * D * TOP * (B + D)}{4 * D^2 * TOP * ((3 * B^2 + 2 * B * D) * BOT + (D^2 + 2 * B * D) * TOP)}$$

Simplifying we obtain

$$TOP = \frac{HM * (B + D)}{DSC * 4 * D * (D^2 + 2 * B * D)} - \frac{3 * B^2 + 2 * B * D}{D^2 + 2 * B * D} * BOT$$

or making

$$A = HM * (B + D) / (DSC * 4 * D * (D^2 + 2 * B * D))$$

$$AB = (3 * B^2 + 2 * B * D) / (D^2 + 2 * B * D)$$

we have

$$TOP = A - AB * BOT$$

replacing here

$$BOT = AC * TOP$$

we obtain

$$TOP = \frac{A}{1 + AB * AC}$$

Having already found TOP and BOT we may use Bryan's formula to determine S; since the thickness of plating is for this case smaller at the bottom we must consider this thickness and along with it DSC because this will be the larger compressive stress the bottom will be subjected to.

$$\text{From } DSC = \frac{E}{0.30396*(1-POI*POI)} * \frac{BOT^2}{S^2}$$

We obtain

$$S = \frac{1}{\sqrt{0.30396 * \frac{1-POI*POI}{E} * DSC}} * BOT$$

or making

$$Z = \frac{1}{\sqrt{0.30396 * \frac{1-POI*POI}{E} * DSC}}$$

we have

$$S = Z * BOT$$

A computer program was written and applied to two particular sets of dimensions and two different materials namely steel and aluminum. The results are presented next in the computer printouts.

STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE

```

0004 C *AL* IS THE LENGTH
0004 AL=500.
0008 C *R* IS THE RADIUS
0008 R=AL/8.
0012 C *DRA* IS THE DRAFT
0012 DRA=B/3.
0016 C *D* IS THE DEPTH OF THE HULL
0016 D=AL/9.
0020 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0020 FOR STEEL
0024 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0024 FOR STEEL
0028 C *F* IS THE DESIGN STRESS IN TENSION
0028 F=30.*10.**6
0032 C *DST* IS THE DESIGN STRESS IN TENSION
0032 DST=25000.
0036 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
0036 DSC=30000.
0040 C *HM* IS THE HOGGING BENDING MOMENT
0040 HM=1.37143*B*DRA*AL*AL
0044 C *SM* IS THE SAGGING BENDING MOMENT
0044 SM=B*DRA*AL*AL
0048 C *W* IS THE SECTION MODULUS
0048 W=0.30396*(1.-POI*POI)*DSC/E
0052 C *H* IS THE HULL HEIGHT
0052 H=SQRT(W)
0056 C *Z* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE TOP OF THE HULL
0056 Z=1./U
0060 C *A* IS THE AREA OF THE HULL SECTION
0060 A=HM*(R+D)/(DSC*4.*D*(D*D+2.*R*D))
0064 C *AB* IS THE AREA OF THE TOP PLATING
0064 AB=(3.*R*B+2.*B*D)/(D*D+2.*R*D)
0068 C *AC* IS THE AREA OF THE BOTTOM PLATING
0068 AC=DST/DSC+D*(DST-DSC)/(DSC*B)
0072 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0072 TOP=A/(1.+AB*AC)
0076 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0076 BOT=AC*TOP
0080 C *WRITE(5,10) TOP
0080 WRITE(5,10) TOP
0084 C *FORMAT('1',22Y,1TOP='E10.4)
0084 FORMAT('1',22Y,1TOP='E10.4)
0088 C *WRITE(5,20) BOT
0088 WRITE(5,20) BOT
0092 C *FORMAT('1',22Y,1BOT='E10.4)
0092 FORMAT('1',22Y,1BOT='E10.4)
0096 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0096 S=Z*BOT
0100 C *WRITE(5,30) S
0100 WRITE(5,30) S
0104 C *FORMAT('1',22Y,1S='E10.4)
0104 FORMAT('1',22Y,1S='E10.4)
0108 C *P* IS THE PERIOD OF VIBRATION
0108 P=2.*D*TOP+B*(TOP+BOT)
0112 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0112 DITOP=D*(B*BOT+D*TOP)/P
0116 C *WRITE(5,40) DITOP
0116 WRITE(5,40) DITOP
0120 C *FORMAT('1',18Y,1DITOP='E10.4)
0120 FORMAT('1',18Y,1DITOP='E10.4)
0124 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0124 DIBOT=D*(B*TOP+D*BOT)/P
0128 C *WRITE(5,50) DIBOT
0128 WRITE(5,50) DIBOT
0132 C *FORMAT('1',18Y,1DIBOT='E10.4)
0132 FORMAT('1',18Y,1DIBOT='E10.4)
0136 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.
0136 GMI=4.*D*D*((3.*R*R+2.*R*D)*TOP*BOT+(D*D+2.*R*D)*TOP*TOP)/P
0140 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
0140 *STS* IS THE STRESS ON TOP DUE TO SAGGING
0140 *SRH* IS THE STRESS ON BOT DUE TO HOGGING

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

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008 C *SBS* IS THE STRESS ON BOT DUE TO SAGGING
009 C *STCRIT* IS THE CRITICAL STRESS ON TOP
010 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
011 STH=HM*DITOP/GMI
012 STS=SM*DITOP/GMI
013 SBH=HM*DIBOT/GMI
014 SBS=SM*DIBOT/GMI
015 STCRIT=TOP*TOP*F/(0.30396*S*S*(1.-POI*POI))
016 SBCRIT=BOT*BOT*F/(0.30396*S*S*(1.-POI*POI))
017 WRITE(5,40)STH
018 60 FORMAT(' ',20Y,'STH='F10.3)
019 WRITE(5,70)STS
020 70 FORMAT(' ',20Y,'STS='F10.3)
021 WRITE(5,80)STCRIT
022 80 FORMAT(' ',17Y,'STCRIT='F10.3)
023 WRITE(5,90)SBH
024 90 FORMAT(' ',20Y,'SBH='F10.3)
025 WRITE(5,100)SBS
026 100 FORMAT(' ',20Y,'SBS='F10.3)
027 WRITE(5,110)SBCRIT
028 110 FORMAT(' ',17Y,'SBCRIT='F10.3)
029 END

```

[S] •U	0584 [V] AL	0580 [V] R	0504 [V] DPA	0500 [V] D
[V] POT	0500 [V] F	0000 [S] .R	0500 [V] DST	0504 [V] DBC
[V] HM	0624 [V] SM	0600 [V] W	0614 [V] U	0000 [S] SORT
[V] Z	0610 [V] A	0640 [V] AR	0644 [V] AC	0648 [V] TOP
[V] BOT	0100 [L] 10	0000 [S] @I	0200 [L] 20	0650 [V] 0
[L] 30	0654 [V] P	0650 [V] DITOP	0200 [L] 40	0650 [V] DIBOT
[L] 50	0660 [V] GMT	0660 [V] STH	0670 [V] STS	0674 [V] SBH
[V] SBS	0670 [V] STCRIT	0680 [V] SBCRIT	0404 [L] 60	0484 [L] 70
[L] 80	0528 [L] 90	0550 [L] 100	0504 [L] 110	0020 [S] .V

L

PARAM LABELS:

0170 *MAIN*	208A •V	288A •COMP	28DC ALDG	2098 @I	26F4 .R
0182 •ZERO	28E0 \$6	2734 •A	20DF .MES	28E0 •W	20E4 EV
0184 AINT	27E6 SORT	2022 •BARG	2 7F .5	2054 \$8	2036 .F
0186 •0	2050 •U	3EB4			

KEY-POINTS:

0184 •0	2734 •A	27E6 SORT	28DC ALDG	28E4 EXP	28FA AI
0185 •W	288A •COMP	28E0 \$6	2 22 .BARG	2054 \$8	207E .5
0182 •ZERO	20E6 •FRONT	2036 •0	20DE .MES	2068 •U	203E .V
0188 @T					

COMMON-BLOCKS:

DEFINED SUBROUTINES:

TOP= 0.3464
 BOT= 2.2374
 S= 14.2732
 DITOP= 25.2525
 DIBOT= 31.3030
 STH= 25032.000
 STS= 18220.152
 STCRIT= 63932.609
 SRH= 30032.012
 SRS= 21874.092
 SRCRIT= 29990.961

08 HANDLING CHARGE	\$.35 / JOB	.35
18 LINES PRINTED PRI	\$ 1.25 / K LN	.1
80 CAPDS READ	\$ 1.50 / K CD	.1
00 PLOTTER VECTORS	\$.25 / 1000	.02
19 MODEL 70 SECONDS	\$25.00 / HOUR	.1
00 MODEL 80 SECONDS	\$12.50 / HOUR	.02
	TOTAL CHARGE \$.7

EIR 400 14731 LOGGED OUT 04/21/74 20:22. \$ 9.05 LEFT AFTER 34 LOGINS.

STRENGTH OF SHIP ON THE NEUTRAL AXIS LOCATION

PAGE 1

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04 C *AL* IS THE LENGTH
04 AL=500.
04 C *B* IS THE DRAFT
04 B=AL/3.
04 C *DRA* IS THE DRAFT
04 DRA=1/3.
04 D=AL/3.
04 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
04 FOR ALUMINUM
04 POI=.33
04 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
04 FOR ALUMINUM
04 E=1.0E+10
04 C *DST* IS THE DESIGN STRESS IN TENSION
04 DST=20000.
04 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
04 DSC=30000.
04 C *M* IS THE HOGGING BENDING MOMENT
04 M=1.0E7/3*(3+DRA*AL*AL)
04 C *SM* IS THE SAGGING BENDING MOMENT
04 SM=1.0E7*(3+AL*AL)
04 F=.9*(3*(1.-POI-POI)*DSC/E)
04 U=SM/F(1.)
04 Z=1./U
04 A=M*(1.+U)/(DSC*4.*F*(D+D+2.*B*D))
04 AB=(3.*M*U+3.*M*U)/(D*D+2.*B*D)
04 AC=DST/DSC*(1.+U)/(DSC*B)
04 C *TOP* IS THE THICKNESS OF THE TOP PLATING
04 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
04 TOP=A/(1.+1/3*AC)
04 BOT=AC*TOP
04 WRITE(5,1)TOP
04 10 FORMAT(' ',20X,'TOP='F12.4)
04 WRITE(5,2)BOT
04 20 FORMAT(' ',20X,'BOT='F12.4)
04 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
04 S=Z*BOT
04 WRITE(5,3)S
04 30 FORMAT(' ',20X,'S='F12.4)
04 P=2.*F*TOP*(1.+TOP*BOT)
04 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
04 DITOP=D*(1.+BOT*TOP)/P
04 WRITE(5,4)DITOP
04 40 FORMAT(' ',10X,'DITOP='F12.4)
04 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
04 DIBOT=D*(1.+TOP*BOT)/P
04 WRITE(5,5)DIBOT
04 50 FORMAT(' ',10X,'DIBOT='F12.4)
04 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.
04 GMI=4.*F*D*(3.*B*B+2.*B*D)*TOP*BOT+(D*D+2.*B*D)*TOP*TOP/P
04 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
04 C *STS* IS THE STRESS ON TOP DUE TO SAGGING
04 C *SBH* IS THE STRESS ON BOT DUE TO HOGGING

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STRENGTH OF SHEAR AT THE NEUTRAL AXIS LOCATION

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300 C *SBS* IS THE STRESS ON FOOT DUE TO SAGGING
310 C *STCRIT* IS THE CRITICAL STRESS ON TOP
320 C *SCCRIT* IS THE CRITICAL STRESS ON BOT
330 STH=H*Z/IT+Z/G*I
340 STS=H*Z/IT+Z/G*I
350 SBH=H*Z/IT+Z/G*I
360 SBS=H*Z/IT+Z/G*I
370 STCRIT=I*E*Z/P*E/(.30396*S*S*(1.+FOI*POI))
380 SCCRIT=I*E*Z/P*E/(.30396*S*S*(1.+POI*POI))
390 WRITE(7,*)STH
400 FORMAT(1,'2 X','STH='F11.3)
410 WRITE(7,*)STS
420 FORMAT(1,'2 X','STS='F11.3)
430 WRITE(7,*)STCRIT
440 FORMAT(1,'17 X','STCRIT='F11.3)
450 WRITE(7,*)SBH
460 FORMAT(1,'2 X','SBH='F11.3)
470 WRITE(7,*)SBS
480 FORMAT(1,'2 X','SBS='F11.3)
490 WRITE(7,*)SCCRIT
500 FORMAT(1,'17 X','SCCRIT='F11.3)
510 END

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U	2524 [V]	DL	0.80 [V]	B	2504 [V]	DFR	0.500 [V]	D
FOI	0.000 [V]	F	0.00 [S]	*R	25E8 [V]	DS1	0.5E [V]	DSC
AF	0.614 [V]	AF	0.604 [V]	W	2614 [V]	U	0.00 [S]	SOFT
Z	0.415 [V]	A	0.430 [V]	AF	2644 [V]	AC	0.644 [V]	TOP
FOI	0.100 [V]	14	0.00 [S]	@I	0.202 [L]	20	0.640 [V]	S
SV	0.657 [V]	F	0.444 [V]	DITOP	0.201 [L]	40	2.654 [V]	DIRECT
SV	0.590 [V]	0.1	0.466 [V]	STH	0.660 [V]	STS	2.67 [V]	SBH
SBS	0.672 [V]	STCRIT	0.470 [V]	SCCRIT	0.484 [L]	60	0.484 [L]	70
SV	0.528 [L]	0.1	0.550 [L]	110	0.594 [L]	110	0.20 [S]	*V

L

LABELS:

7	*MAI *	2056 *V	2886 *COMP	2808 ALCG	2094 @I	26E0 *R
7	*ZER	2810 *F	2782 *A	20DA *MES	2858 *W	29E0 EXP
7	AINT	27E2 SOFT	201E *RABG	207A *S	2050 *R	2032 *EF
0	*O	2058 *L	3E80			

CINTS:

7	*R	2734 *W	27E2 SOFT	2808 ALCG	29E0 EXP	2AE6 AIT
7	*W	2886 *COMP	2800 *F	201E *RABG	2050 *R	207A *S
7	*ZER	2032 *EFCIT	2084 *C	20DA *MES	2064 *U	208A *V
7	@I					

PLUCKS:

DEFINED SHEAR ROUTINES:

TOP= .2464
 FOT= .2374
 S= .2270
 DITOP= 25.2525
 DIFOT= 3.223
 STOP= 1 228.15
 STOFOT= 1 228.15
 SOTOP= 2 228.15
 SOTOFOT= 2 228.15
 SOTOP= 2 228.15
 SOTOFOT= 2 228.15

01	HANDLING CHARGE	1.35	/	JOB	.35
02	INES PRINTED	1.25	/	K LI	.15
03	APDS READ	1.5	/	K CI	.12
04	LOTT n VECTORS	1.25	/	1.0	.08
05	CDPL 70 SECONDS	5.0	/	HOUP	.10
06	CDPL 80 SECONDS	12.5	/	HOUP	.06
	TOTAL	19.45			.72

499 14731 LOGOUT UT 4/1/74 20:20. \$ 9.80 LEFT AFTER 33 LOGINS.


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004 C *AL* IS THE LENGTH
004 AL=1000.
008 C *R* IS THE RADIUS
008 R=AL/5.75
012 C *DRA* IS THE DRAFT
012 DRA=R/3.3
016 C *I* IS THE MOMENT OF INERTIA OF THE PLATING
016 I=AL/14.
020 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
020 FOR STEEL
024 C *NU* IS THE POISSON'S RATIO OF THE MATERIAL
024 FOR STEEL
028 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
028 FOR STEEL
032 C *F* IS THE YOUNG'S MODULUS OF THE MATERIAL
032 FOR STEEL
036 C *DST* IS THE DESIGN STRESS IN TENSION
036 DST=25000.
040 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
040 DSC=30000.
044 C *HM* IS THE HOGGING BENDING MOMENT
044 HM=1.37143*R*DRA*AL*I
048 C *SM* IS THE SAGGING BENDING MOMENT
048 SM=R*DRA*AL*AL
052 C *W* IS THE WEIGHT OF THE PLATING
052 W=0.30396*(1.-POI*POI)*DSC/E
056 C *H* IS THE HOGGING BENDING MOMENT
056 H=SQRT(W)
060 C *Z* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE TOP PLATING
060 Z=1./U
064 C *A* IS THE AREA OF THE TOP PLATING
064 A=HM*(R+D)/(DSC*4.*D*(D+D+2.*R*D))
068 C *B* IS THE AREA OF THE BOTTOM PLATING
068 B=(3.*R*B+2.*R*D)/(D+D+2.*R*D)
072 C *AC* IS THE AREA OF THE TOP PLATING
072 AC=DST/DSC+D*(DST-DSC)/(DSC*B)
076 C *TOP* IS THE THICKNESS OF THE TOP PLATING
076 TOP=A/(1.+AB*AC)
080 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
080 BOT=AC*TOP
084 C *TOP* IS THE THICKNESS OF THE TOP PLATING
084 WRITE(5,10)TOP
088 C *TOP* IS THE THICKNESS OF THE TOP PLATING
088 10 FORMAT(' ',20X,'TOP=1E10.4)
092 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
092 WRITE(5,20)BOT
096 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
096 20 FORMAT(' ',20X,'BOT=1E10.4)
100 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
100 S=Z*BOT
104 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
104 WRITE(5,30)S
108 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
108 30 FORMAT(' ',22X,'S=1E10.4)
112 C *P* IS THE PERIOD OF VIBRATION
112 P=2.*D*TOP+B*(TOP+BOT)
116 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
116 DITOP=D*(R*BOT+D*TOP)/P
120 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
120 WRITE(5,40)DITOP
124 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
124 40 FORMAT(' ',18X,'DITOP=1E10.4)
128 C *DIROT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
128 DIROT=D*(R*TOP+D*TOP)/P
132 C *DIROT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
132 WRITE(5,50)DIROT
136 C *DIROT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
136 50 FORMAT(' ',18X,'DIROT=1E10.4)
140 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT M.A.
140 GMI=4.*R*D*((.5.*R*B+2.*R*D)*TOP*BOT+(D*D+2.*R*D)*TOP*TOP)/P
144 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
144 *STH* IS THE STRESS ON TOP DUE TO HOGGING
148 C *STS* IS THE STRESS ON TOP DUE TO SAGGING
148 *STS* IS THE STRESS ON TOP DUE TO SAGGING
152 C *SRH* IS THE STRESS ON BOT DUE TO HOGGING
152 *SRH* IS THE STRESS ON BOT DUE TO HOGGING

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

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0008 C *SBS* IS THE STRESS ON BOT DUE TO SAGGING
0008 C *STCRIT* IS THE CRITICAL STRESS ON TOP
0008 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
0008 CTH=HM*DITOP/GMI
0008 CTS=SM*DITOP/GMI
0008 CCBH=HM*DIBOT/GMI
0008 CBS=SM*DIBOT/GMI
0008 CTCRIT=TOP*TOP*F/(0.30306*S*S*(1.-POI*POI))
0008 CBSCRIT=BOT*BOT*F/(0.30306*S*S*(1.-POI*POI))
0008 CWRITE(5,60)STH
0008 60 FORMAT(' ',20Y,'STH='F10.3)
0008 CWRITE(5,70)STS
0008 70 FORMAT(' ',20Y,'STS='F10.3)
0008 CWRITE(5,80)STCRIT
0008 80 FORMAT(' ',17Y,'STCRIT='F10.3)
0008 CWRITE(5,90)SBH
0008 90 FORMAT(' ',20Y,'SBH='F10.3)
0008 CWRITE(5,100)SBS
0008 100 FORMAT(' ',20Y,'SBS='F10.3)
0008 CWRITE(5,110)SBCRIT
0008 110 FORMAT(' ',17Y,'SBCRIT='F10.3)
0008 CEND

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[S] •H	2584 [V]	AL	0580 [V]	B	0504 [V]	DRA	0500 [V]	D
[V] POI	2500 [V]	F	0470 [V]	.R	0500 [V]	DST	0554 [V]	DSC
[V] HM	2604 [V]	SM	0670 [V]	W	0614 [V]	U	0630 [S]	SOPT
[V] Z	0610 [V]	A	0640 [V]	AB	0648 [V]	AC	0640 [V]	TOP
[V] BOT	0100 [L]	10	0000 [V]	AI	0202 [L]	20	0654 [V]	S
[L] 30	0658 [V]	P	0650 [V]	DITOP	0200 [L]	40	0660 [V]	DIBOT
[L] 50	0664 [V]	GMI	0670 [V]	STH	0674 [V]	STS	0678 [V]	SBH
[V] SBS	0680 [V]	STCRIT	0684 [V]	SBCRIT	0404 [L]	60	0688 [L]	70
[L] 80	0528 [L]	90	0550 [L]	100	0504 [L]	110	0700 [S]	.V

L

PROGRAM LABELS:

2770 *MAIN*	208E •V	288E •COMP	28E0 AL0G	2D9C @I	26F8 •P
2CR6 •ZERO	28E4 \$F	2738 •A	2CE2 .MES	2860 •U	29E8 FX
2AEE ATNT	27EA SORT	2C26 •RARG	2C82 .S	2C58 \$R	2C8A •E
2CRA •0	2D60 •U	2E88			

TRY-POINTS:

26F8 •P	2738 •A	27EA SORT	28E0 AL0G	29E8 EXP	2AEE AT
2860 •U	28E4 \$F	28E4 \$F	2C26 .RARG	2C58 \$R	2C82 .S
2CR6 •ZERO	2C8A •ERCNT	2C8C •0	2CE2 .MES	2D60 •U	2D92 •V
2D9C @I					

COMMON-BLOCKS:

E

DEFINED SUBROUTINES:

TOP= 3.0399
 BOT= 2.3252
 S= 139.8065
 DITOP= 32.4675
 DIROT= 38.9610
 STH= 25000.000
 STS= 18220.152
 STCRIT= 51278.250
 SBH= 29990.996
 SBS= 21874.084
 SRCRIT= 29990.961

JOB HANDLING CHARGE	\$.25 / JOB	.25
18 LINES PRINTED PR1	\$ 1.25 / K LN	.15
80 CARDS READ	\$ 1.50 / K CP	.12
20 PLOTTER VECTORS	\$.25 / 1000	.02
14 MODEL 70 SECONDS	\$25.00 / HOUR	.1
00 MODEL 80 SECONDS	\$12.50 / HOUR	.0
	TOTAL CHARGE \$.72

EIR 400 14731 LOGGED OUT 04/21/74 20:27. \$ 7.58 LEFT AFTER 26 LOGINS.


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004 C *AL* IS THE LENGTH
004 AL=1000.
004 R=AL/5.75
018 C *DRA* IS THE DRAFT
018 DRA=R/3.3
024 D=AL/14.
030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
030 FOR ALUMINUM
030 POI=0.33
038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
038 FOR ALUMINUM
038 E=10.*10.**6
040 C *DST* IS THE DESIGN STRESS IN TENSION
040 DST=25000.
054 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
054 DSC=30000.
050 C *HM* IS THE HOGGING BENDING MOMENT
050 HM=1.37143*B*DRA*AL*Aj
074 C *SM* IS THE SAGGING BENDING MOMENT
074 SM=B*DRA*AL*Aj
088 W=0.30396*(1.-POI*POI)*DSC/E
088 U=SQRT(W)
088 Z=1./U
088 A=HM*(B+D)/(DSC*4.*D*(D+2.*B*D))
088 AB=(3.*B*B+2.*B*D)/(D+D+2.*B*D)
088 AC=DST/DSC+D*(DST-DSC)/(DSC*B)
080 C *TOP* IS THE THICKNESS OF THE TOP PLATING
080 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
080 TOP=A/(1.+AB*AC)
104 BOT=AC*TOP
108 WRITE(5,10)TOP
100 10 FORMAT('1',22Y,'TOP='F10.4)
106 WRITE(5,20)BOT
102 20 FORMAT('1',22Y,'BOT='F10.4)
110 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
110 S=Z*BOT
128 WRITE(5,30)S
104 30 FORMAT('1',22Y,'S='F10.4)
150 R=2.*D*TOP+B*(TOP+BOT)
180 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
180 DITOP=D*(B*BOT+D*TOP)/R
184 WRITE(5,40)DITOP
200 40 FORMAT('1',18Y,'DITOP='F10.4)
200 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
200 DIBOT=D*(B*TOP+D*BOT)/R
208 WRITE(5,50)DIBOT
210 50 FORMAT('1',18Y,'DIBOT='F10.4)
238 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.
238 GMI=4.*D*D*((3.*B*B+2.*B*D)*TOP*BOT+(D+D+2.*B*D)*TOP*TOP)/P
248 C *STH* IS THE STRESS ON TOP DUE TO HOGGING
248 C *STS* IS THE STRESS ON TOP DUE TO SAGGING
248 C *SBH* IS THE STRESS ON BOT DUE TO HOGGING

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

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0008 C *SRS* IS THE STRESS ON BOT DUE TO SAGGING
0008 C *STCRIT* IS THE CRITICAL STRESS ON TOP
0008 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
0008 CTH=HM*DITOP/GMI
0008 STS=SM*DITOP/GMI
0008 CPH=HM*DIBOT/GMI
0008 CBS=SM*DIBOT/GMI
0008 STCRIT=TOP*TOP*F/(0.30396*S*S*(1.-FOI*POI))
0008 SBCRIT=BOT*BOT*F/(0.30396*S*S*(1.-FOI*POI))
0008 WRITE(5,60)STG
0008 60 FORMAT(' ',23Y,'STH='F10.3)
0008 WRITE(5,70)STG
0008 70 FORMAT(' ',23Y,'STS='F10.3)
0008 WRITE(5,80)STCRIT
0008 80 FORMAT(' ',17Y,'STCRIT='F10.3)
0008 WRITE(5,90)SRH
0008 90 FORMAT(' ',23Y,'SRH='F10.3)
0008 WRITE(5,100)SRB
0008 100 FORMAT(' ',23Y,'SRB='F10.3)
0008 WRITE(5,110)SBCRIT
0008 110 FORMAT(' ',17Y,'SBCRIT='F10.3)
0008 END

```

[S] •H	0584 [V] AL	0580 [V] B	0504 [V] DPA	0500 [V] D
[V] POI	0500 [V] F	0000 [G] .R	0503 [V] DST	0500 [V] DSC
[V] HM	0600 [V] SM	0604 [V] W	0610 [V] U	0000 [S] SORT
[V] Z	0618 [V] A	0630 [V] AR	0604 [V] AC	0648 [V] TOP
[V] BOT	0100 [L] 10	0000 [G] 0I	0202 [L] 20	0650 [V] S
[L] 30	0654 [V] P	0658 [V] DITOP	0200 [L] 40	0650 [V] DIBOT
[L] 50	0660 [V] GMI	0660 [V] STH	0600 [V] STS	0674 [V] SRH
[V] SRS	0670 [V] STCRIT	0680 [V] SBCRIT	0404 [L] 60	0480 [L] 70
[L] 80	0528 [L] 90	0550 [L] 100	0504 [L] 110	0000 [S] .V

L

PROGRAM LABELS:

0070 *MAIN*	208A •V	208A •COMP	20DC ALDG	2098 0I	26F4 •0
0082 •7ERD	20E0 \$4	2734 •A	20DF .MES	2050 •W	20E4 FY
00FA ATNT	27E6 SORT	2022 •RARG	207E .5	2054 48	2086 •E
0086 •0	2050 •U	2584			

TRY-POINTS:

06F4 •0	2734 •A	27E6 SORT	20DC ALDG	29F4 FYP	20EA AT
00FC •W	208A •COMP	20E0 \$4	2022 .RARG	2054 48	207E .5
0082 •7ERD	2086 •FRONT	2088 •0	20DE .MES	2068 •H	208E •I
0098 0I					

COMMON-BLOCKS:

E

DEFINED SUBROUTINES:

TOP= 3.0799
 BOT= 2.3252
 S= 81.5487
 DJTOP= 32.4475
 DIROT= 32.9410
 STU= 25070.200
 STS= 18220.152
 STCRIT= 51270.223
 SBH= 20990.096
 SBS= 21874.084
 SBCRIT= 32000.035

00 HANDLING CHARGE	\$.25 / JOB	.25
18 LINES PRINTED PR1	\$ 1.25 / K LINE	.15
80 CARDS READ	\$ 1.50 / K CARD	.12
00 PLOTTER VECTORS	\$.25 / 1000	.0
19 MODEL 70 SECONDS	\$25.00 / HOUR	.15
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
	TOTAL CHARGE \$.70

FIR 400 14731 LOGGED OUT 04/21/74 20:24. \$ 8.30 LEFT AFTER 35 LOGINS.

HOGGING MOMENT AND SAGGING MOMENT OF EQUAL MAGNITUDE
DESIGN STRESS IN TENSION AND DESIGN STRESS IN COMPRESSION OF
EQUAL VALUE; LATERAL LOAD ALSO CONSIDERED

The lateral load being here considered will be a "head" of salt water of eight feet on the top plating (HTOP=8 ft) and a "head" of salt water equal to the draft on the bottom plating (HBOT=DRA).

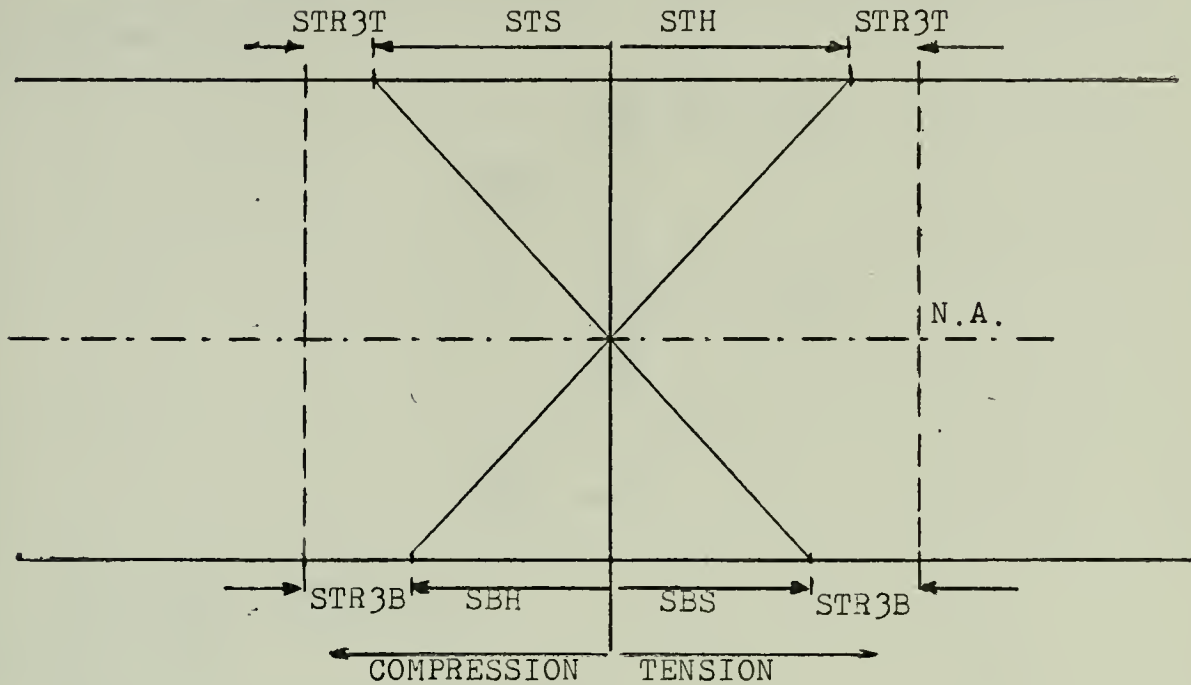
We will use here the simplified approach as suggested in the following note; this simplified approach is at least conservative.

Note: Bleich, from his experiments and theoretical studies on the effect of normal pressure in the buckling strength of thin panels, concluded that the buckling strength is increased when combined with a sufficiently large normal load, and that the increase is greater for the simply supported edges situation than with fixed edges. For heavier plating, where the deflection does not exceed half of the plating thickness, the increase in buckling strength resulting from normal load is negligible. It was therefore suggested, for simplicity, that the critical stress be computed for in plane loading only.

The tertiary stresses we are interested in considering here are those exerted at points B directed longitudinally; these will combine with the longitudinal bending stresses or primary stresses as described in a previous section.

Since now we have $DST=DSC$ and $HM=SM=BM$ we may think of

a diagram for the stress distributions as indicated next figure:



From this diagram we may understand that the neutral axis should be located closer to the bottom which means that the thickness at the bottom will be larger than at the top $BOT > TOP$. So, we may already write the following formulas where TS was replaced by BOT

$$DITOP = D*(B+D)*BOT/P$$

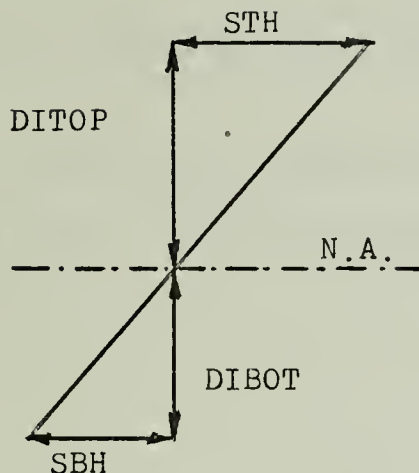
$$DIBOT = D*(B*TOP+D*BOT)/P$$

$$I = 4*D*D*((3*B^2+2*B*D)*TOP+(D^2+2*B*D)*BOT)*BOT/P$$

where

$$P = 2*D*BOT+B*(TOP+BOT)$$

We may now think of the following relationships to determine TOP and BOT.



$$STH * DIBOT = SBH * DITOP$$

or knowing that

$$STR3T = 0.152222 * HTOP * \left(\frac{S}{TOP}\right)^2$$

$$STR3B = 0.152222 * HBOT * \left(\frac{S}{BOT}\right)^2$$

and anticipating

$$S = Z * TOP$$

where

$$Z = 1 / \left(\sqrt{0.30396 * \frac{1 - POI * POI}{E} * DST} \right)$$

we have

$$Z^2 = \left(\frac{S}{TOP}\right)^2$$

If we now make $\left(\frac{S}{BOT}\right)^2 = Z^2$

in order to simplify the equations, we see that when we do this, since

$$\left(\frac{S}{\text{BOT}}\right)^2 < \left(\frac{S}{\text{TOP}}\right)^2$$

we are increasing the value of STR3B and this means that since

$$\text{DST} = \text{SBH} + \text{STR3B}$$

we are decreasing "slightly" the value of SBH that could be allowed to reach a larger value; so , the substitution

$$\left(\frac{S}{\text{BOT}}\right)^2 = Z^2$$

is on the safe side and has the advantage of simplifying the equations considerably.

So we may write

$$\text{STH} = \text{DST} - \text{STR3T}$$

$$\text{SBH} = \text{DST} - \text{STR3B}$$

$$\text{STR3T} = 0.152222 * \text{HTOP} * Z^2$$

$$\text{STR3B} = 0.152222 * \text{HBOT} * Z^2$$

$$\text{A} = \text{DST} - \text{STR3T}$$

$$\text{AB} = \text{DST} - \text{STR3B}$$

so

$$\text{A} * \text{DIBOT} = \text{AB} * \text{DITOP}$$

and we obtain replacing DITOP and DIBOT by their equations, and simplifying

$$\text{TOP} = \frac{\text{AB} * (\text{B} + \text{D}) - \text{A} * \text{D}}{\text{A} * \text{B}} * \text{BOT}$$

or making

$$AC = (AB*(B+D)-A*D)/(A*B)$$

we finally obtain

$$TOP = AC*BOT$$

Now in order to stress the material to its design stress we may either consider STH, STS, SBH, SBS.

Due to ease of calculation we may either choose STS or STH; the expressions will be exactly the same. I will consider here

$$STH = BM*DITOP/I$$

and

$$STH = DST-STR3T = A$$

Substituting for DITOP and I we obtain

$$A = \frac{BM*D*(B+D)*BOT}{4*D^2*BOT*((3*B^2+2*B*D)*TOP+(D^2+2*B*D)*BOT)}$$

and after simplifying we arrive at

$$BOT = \frac{BM*(B+D)}{A*4*D*(D^2+2*B*D)} - \frac{3*B^2+2*B*D}{D^2+2*B*D} *TOP$$

Making

$$AD = BM*(B+D)/(A*4*D*(D^2+2*B*D))$$

$$AE = (3*B^2+2*B*D)/(D^2+2*B*D)$$

we obtain

$$BOT = AD-AE*TOP$$

substituting

$$TOP = AC * BOT$$

we obtain

$$BOT = \frac{AD}{1 + AC * AE}$$

Considering now the instability formula (BRYAN) we notice that since the top will have the thinner plating we will consider TOP stressed to the maximum allowed DST.

So we start with

$$Z = 1 / \left(\sqrt{0.30396 * \frac{1 - POI * POI}{E} * DST} \right)$$

$$S = Z * TOP$$

The total stresses will then be given by

$$TSTH = BM * DITOP / I + STR3T$$

$$TSTS = BM * DITOP / I + STR3T$$

$$TSBH = BM * DIBOT / I + 0.152222 * HBOT * S^2 / BOT^2$$

$$TSBS = BM * DIBOT / I + 0.152222 * HBOT * S^2 / BOT^2$$

The computer printouts for the same cases analysed in the previous section are presented next.

STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 1

```

0004 C *AL* IS THE LENGHT
0004 AL=500.
000C B=AL/8.
0018 C *DRA* IS THE DRAFT
0018 DRA=B/3.
0024 D=AL/9.
0030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0030 C FOR STEEL
0030 POI=0.3
0038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0038 C FOR STEEL
0038 E=30.**12.**6
004C C *DST* IS THE DESIGN STRESS
004C DST=25000.
0054 C *BM* IS THE BENDING MOMENT
0054 BM=1.37143*B*DRA*AL*AL
006C W=0.30396*(1.-POI*POI)*DST/E
008C U=SQRT(W)
0098 Z=1./U
00A4 *HTOP* IS THE HEAD OF WATER ON DECK
00A4 HTOP=8.
00AC *HBOT* IS THE HEAD OF WATER ON BOTTOM
00AC HBOT=DRA
00B4 STR3T=0.152222*HTOP*Z*Z
00C8 STR3B=0.152222*HBTOT*Z*Z
00DC A=DST-STR3T
00E8 AB=DST-STR3B
00F4 AC=(AB*(B+D)-A*D)/(A*B)
012C AD=BM*(B+D)/(A**4.*D*(D*D+2.*B*D))
0178 AE=(3.*B*B+2.*B*D)/(D*D+2.*B*D)
01C4 C *TOP* IS THE THICKNESS OF THE TOP PLATING
01C4 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
01C4 BOT=AD/(1.+AC*AE)
01DC TOP=AC*BOT
01E8 WRITE(5,10)TOP
0204 10 FORMAT('1',20X,'TOP='F10.4)
021E WRITE(5,20)BOT
023A 20 FORMAT(' ',20X,'BOT='F10.4)
0254 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0254 S=Z*TOP
0260 WRITE(5,30)S
027C 30 FORMAT(' ',22X,'S='F10.4)
0294 P=2.*D*BOT+B*(TOP+BOT)
02B8 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
02B8 DITOP=D*(B*BOT+D*BOT)/P
02DC WRITE(5,40)DITOP
02F8 40 FORMAT(' ',18X,'DITOP='F10.4)
0314 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0314 DIBOT=D*(B*TOP+D*BOT)/P
0338 WRITE(5,50)DIBOT
0354 50 FORMAT(' ',18X,'DIBOT='F10.4)
0370 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.

```


STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

```

0270      GMI=4.*D*D*((3.*R*B+2.*B*D)*TOP*BOT+(D*D+2.*B*D)*BOT*BOT)/P
03E0      C      *TSTH* IS THE STRESS ON TOP DUE TO HOGGING
03E0      C      *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
03E0      C      *TSBH* IS THE STRESS ON BOT DUE TO HOGGING
03E0      C      *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
03E0      C      *STCRIT* IS THE CRITICAL STRESS ON TOP
03E0      C      *SBCRIT* IS THE CRITICAL STRESS ON BOT
05E0      TSTH=BM*DITOP/GMI+STR3T
03F4      TSTS=BM*DITOP/GMI+STR3T
0408      STCRIT=TOP*TOP*E/(0.30396*S*S*(1.-POI*POI))
0448      WRITE(5,60)TSTH
0464      60  FORMAT(' ',19X,'TSTH='F10.3)
047E      WRITE(5,70)TSTS
049A      70  FORMAT(' ',19X,'TSTS='F10.3)
04B4      WRITE(5,80)STCRIT
04D0      80  FORMAT(' ',17X,'STCRIT='F10.3)
04EC      TSBH=BM*DIBOT/GMI+0.152222*HBOT*S*S/(BOT*BOT)
0528      TSBS=BM*DIBOT/GMI+0.152222*HBOT*S*S/(BOT*BOT)
0564      SBCRIT=BOT*BOT*E/(0.30396*S*S*(1.-FOI*FOI))
05A4      WRITE(5,90)TSBH
05C0      90  FORMAT(' ',19X,'TSBH='F10.3)
05DA      WRITE(5,100)TSBS
05F6      100 FORMAT(' ',19X,'TSBS='F10.3)
0610      WRITE(5,110)SBCRIT
062C      110 FORMAT(' ',17X,'SBCRIT='F10.3)
0648      END
07[S]  •U      064C[V] AL      0654[V] B      065C[V] DRA      0664[V] D
08[V]  POI     0674[V] E      0000[S] •R      0684[V] DST      068C[V] BM
09[V]  W       06A0[V] U      0000[S] SQRT     06A4[V] Z      06A8[V] HTOP
10[V]  HBOT    06B0[V] STR3T   06B8[V] STR3B    06BC[V] A      06C0[V] AB
11[V]  AC      06D8[V] AD      06EC[V] AE      06F0[V] BOT     06F4[V] TOP
12[L]  10     0000[S] 0I      023A[L] 20      06F8[V] S      027C[L] 30
13[V]  P       0700[V] DITOP   02F8[L] 40      0704[V] DIBOT   0354[L] 50
14[V]  GMI     0714[V] TSTH     0718[V] TSTS     071C[V] STCRIT  0464[L] 60
15[L]  70     04D0[L] 80      0720[V] TSBH    0724[V] TSBS     0728[V] SBCRIT
16[L]  90     05F6[L] 100    062C[L] 110    0000[S] •V
07EQ      L

```

PROGRAM LABELS:

02070 *MAIN*	2EB2 •V	2CE2 •COMP	2984 ALOG	2EC0 0I	279C •F
020AA •ZERO	2D08 \$6	27DC •A	2E06 •MES	2C84 •W	2A8C E
02C12 AINT	288E SQRT	2D4A •RARG	2DA6 •5	2A8C AEXP	2DAE •F
02D7C \$8	2DAE •0	2E84 •U	3FDC		

TRY-POINTS:

279C •R	27DC •A	288E SQRT	2984 ALOG	2A8C AEXP	2A8C E
02C12 AINT	2C84 •W	2CE2 •COMP	2D08 \$6	2D4A •RARG	2D7C \$
02DA6 •5	020AA •ZERO	02DAE •ERCNT	2080 •0	2E06 •MES	2E90 •I
02EB6 •V	2EC0 0I				

TOP= 0.1671
 BOT= 0.8883
 S= 11.0049
 DITOP= 35.3820
 DIBOT= 20.1736
 TSTH= 25000.012
 TSTS= 25000.012
 STCRIT= 25000.000
 TSBH= 11728.668
 TSBS= 11728.668
 SBCRIT=706635.250

END

F VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 04/27/74

JOB HANDLING CHARGE	\$.35 / JOB	.35
125 LINES PRINTED PR1	\$ 1.25 / K LN	.16
84 CARDS READ	\$ 1.50 / K CD	.13
00 PLOTTER VECTORS	\$.25 / 1000	.00
15 MODEL 70 SECONDS	\$25.00 / HOUR	.10
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
	TOTAL CHARGE \$.74

PEIR 490 14731 LOGGED OUT 04/29/74 18:35. \$ 33.19 LEFT AFTER 42 LOGINS.

Note here that because we have in this run BOT>> TOP the approximation made to simplify the formulas, namely

$$Z = \frac{S}{\text{BOT}}$$

becomes very inaccurate compared to what the actual value should be

$$Z = \frac{S}{\text{TOP}}$$

this is the cause for obtaining such low stresses at the bottom. We might refine the result for this case as is done for the "1000 ft steel case" also included in this section which presented still another difficulty. Since all steps to have the bottom stressed to DST are explained in detail for such case they will be omitted here.

STIFFNESS OF SHIPS ON THE NEUTRAL AXIS LOCATION

PAGE 1

```

0104 C *AL* IS THE LENGTH
0134 AL=500.
0170 B=AL/8.
0212 C *DPA* IS THE DRAFT
0214 DP=5.23.
0224 D=AL/9.
0230 C *NU* IS THE POISSON'S RATIO OF THE MATERIAL
0232 NU=.3.
0234 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0236 E=10.*10**8.
0238 C *US* IS THE DESIGN STRESS
0240 US=240.
0242 C *M* IS THE BENDING MOMENT
0244 M=1.37143*E*DP*AL*AL
0246 M=1.3669*(1.-NU)*M*SIZE
0248 M=8910.
0250 Z=1.70
0252 C *HTOP* IS THE HEAD OF WATER ON DECK
0254 HTOP=8.
0256 C *HBOT* IS THE HEAD OF WATER ON BOTTOM
0258 HBOT=0.
0260 STB3I=.152226*HTOP**2**2
0262 STB3e=.152226*HBOT**2**2
0264 A=US*STB3I
0266 AB=US*STB3e
0268 AC=(AB*(1+NU)+A**2)/(A*B)
0270 AD=-A*(1+NU)/(A**2+AB*(1+2*NU))
0272 AE=(B**2+2*NU*AB)/(NU*A+2*NU*B)
0274 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0276 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0278 BOT=AD/(1+AC*AE)
0280 TOP=AC*BOT
0282 WRITE(5,1)TOP
0284 10 FORMAT('1',22Y,'TOP='F10.4)
0286 WRITE(5,2)BOT
0288 20 FORMAT('1',22Y,'BOT='F10.4)
0290 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0292 S=7*TOP
0294 WRITE(5,3)S
0296 30 FORMAT('1',22Y,'S='F10.4)
0298 R=2.*D*BOT+B*(TOP+BOT)
0300 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0302 DITOP=D*(B*BOT+R*BOT)/R
0304 WRITE(5,4)DITOP
0306 40 FORMAT('1',18Y,'DITOP='F10.4)
0308 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0310 DIBOT=D*(B*TOP+R*BOT)/R
0312 WRITE(5,5)DIBOT
0314 50 FORMAT('1',18Y,'DIBOT='F10.4)
0316 C *IGI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.

```


STRENGTH OF SHIPS-ON THE CENTRAL AXIS LOCATION

```

4670      GMI=4.*D*(2.*E+2.*I*D)*TOP*POI+(D*D+2.*B*D)*BOT*BOT)/P
4680      *ISTH*IS THE STRESS ON TOP DUE TO BOWING
4690      *TSPH*IS THE STRESS ON TOP DUE TO SAGGING
4700      *ISPH*IS THE STRESS ON BOT DUE TO BOWING
4710      *ISBS*IS THE STRESS ON BOT DUE TO SAGGING
4720      *SCRIT* IS THE CRITICAL STRESS ON TOP
4730      *SBCRIT* IS THE CRITICAL STRESS ON BOT
4740      ISTH=(M*BITOP/3)+ST*BT
4750      ISPH=3*M*BITOP/2*I+ST*BT
4760      SICRIT=M*B*TOP*E/(1.3-3*B*S*(1.-POI*POI))
4770      WRITE(5,60)ISTH
4780      FORMAT(11,12,13,14,15,16,17)
4790      WRITE(5,70)ISPH
4800      FORMAT(11,12,13,14,15,16,17)
4810      WRITE(5,80)SICRIT
4820      FORMAT(11,12,13,14,15,16,17)
4830      ISBH=(M*BITOP/3)+2.152222*H*BOT*S*(BOT*BOT)
4840      ISBS=(M*BITOP/3)+2.152222*H*BOT*S*(BOT*BOT)
4850      SBCRIT=M*B*BOT*E/(1.3-3*B*S*(1.-POI*POI))
4860      WRITE(5,90)ISBH
4870      FORMAT(11,12,13,14,15,16,17)
4880      WRITE(5,100)ISBS
4890      FORMAT(11,12,13,14,15,16,17)
4900      WRITE(5,110)SBCRIT
4910      FORMAT(11,12,13,14,15,16,17)
4920      END

```

[S] .1	1640 [V] AL	1654 [V] B	1660 [V] DRA	1664 [V] D
[V] BOT	1674 [V] E	1680 [V] .R	1684 [V] DST	1688 [V] RM
[V] .	1690 [V] G	1696 [V] SDRF	1699 [V] Z	1704 [V] HTOP
[V] .	1710 [V] STBT	1714 [V] STRB	1718 [V] A	1720 [V] AB
[V] AC	1734 [V] AD	1738 [V] AE	1740 [V] BOT	1742 [V] TOP
[L] 1	1750 [V] I	1754 [V] 20	1758 [V] S	1760 [L] 30
[V] F	1760 [V] HITOP	1764 [V] 40	1768 [V] DJPOT	1770 [L] 50
[V] G	1770 [V] ISTH	1774 [V] TSTS	1778 [V] STCRIT	1780 [L] 60
[L] 70	1780 [L] 80	1784 [V] TSPH	1788 [V] TSPS	1790 [V] SBCRIT
[L] 9	1796 [L] 100	1800 [L] 110	1804 [S] .V	

PROGRAM LABELS:

```

2170 *INITIAL 2078 .V 2082 .CCMP 2086 .ALOG 2090 .@I 2798 .
20A6 .ZFRD 2004 .S 2708 .A 2002 .MES 2080 .W 2A88 E
200E .AJNT 208A .S-T 204A .RARG 20A2 .5 2A8E .AEXP 20AA .
2078 .S 207A . 2038 .H 2008

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TRY-POINTS:

```

2798 . 2708 .A 203A .S-T 2080 .ALOG 2A82 .AEXP 2A88 E
200E .AJNT 2004 . 200E .CCMP 204A .6 204E .RARG 2078 .S
20A2 .5 20A6 .ZFRD 20AA .S-T 20AC .0 2002 .MES 2080 .
2002 .V 2000 .01

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WORK-BOOKS:

TOP= 4.4217
 POT= 1.4225
 S= 12.3632
 CITOP= 29.6132
 CIPTOP= 26.9353
 TSTP= 25721.000
 TSTG= 25721.000
 STCRIT= 25721.000
 TSPH= 23256.451
 TSPG= 23256.451
 STCRIT= 42705.652

VIO OPERATING SYSTEM VERSION J REVISION 012 03/04/74 GENERATED 04/27/74

08	HANDLING CHARGE	4	0.25	/	JOE		0.35
25	LINE PRINTER PRN	1	1.25	/	K L		0.13
84	CARD READER	4	1.50	/	K G		0.13
00	PLATTER FACTORS		0.25	/	1000		0.40
15	MODEL 70 SECONDS	425	0.20	/	FOUR		0.1
00	MODEL 70 SECONDS	112	0.50	/	FOUR		0.17
	TOTAL CHARGE \$						0.74

FIR # 14731 LOGGED OUT 04/29/74 18:05 \$ 26.17 LEFT AFTER 38 LOGINS.

This case gave a result close to the correct solution with the bottom stressed to almost DST.

Next is presented the "1000 ft steel case" where still another difficulty was encountered and a procedure to have the bottom stressed to DST is exemplified.

STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE 1

```

04 C *AL* IS THE LENGHT
04 AL=1000.
08 C *B* IS THE BREADTH
08 B=AL/5.75
18 C *DRA* IS THE DRAFT
18 DRA=B/3.3
24 C *L* IS THE LENGTH OF THE SHIP
24 L=AL/14.
30 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
30 FOR STEEL
30 POI=0.3
38 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
38 FOR STEEL
38 E=30.*10.**6
40 C *DST* IS THE DESIGN STRESS
40 DST=25000.
54 C *EM* IS THE BENDING MOMENT
54 BM=1.37143*B*DRA*AL*AL
60 C *W* IS THE WEIGHT OF THE SHIP
60 W=0.34396*(1.-POI*POI)*DST/E
80 C *U* IS THE VELOCITY OF THE SHIP
80 U=SQRT(W)
98 C *Z* IS THE DISTANCE FROM THE NEUTRAL AXIS
98 Z=1./U
A4 C *HTOP* IS THE HEAD OF WATER ON DECK
A4 HTOP=8.
AC C *HBOT* IS THE HEAD OF WATER ON BOTTOM
AC HBOT=DRA
B4 C *STR3T* IS THE STRESS AT THE TOP OF THE SHIP
B4 STR3T=0.152222*HTOP*Z*Z
C8 C *STR3B* IS THE STRESS AT THE BOTTOM OF THE SHIP
C8 STR3B=0.152222*HBOT*Z*Z
DC C *A* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE TOP OF THE SHIP
DC A=DST-STR3T
E8 C *AB* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE BOTTOM OF THE SHIP
E8 AB=DST-STR3B
F4 C *AC* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE CENTER OF GRAVITY
F4 AC=(AB*(B+D)-A*D)/(A*B)
20 C *AD* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE CENTER OF GRAVITY
20 AD=BM*(B+D)/(A*4.*D*(D*D+2.*B*D))
78 C *AE* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE CENTER OF GRAVITY
78 AE=(3.*B*B+2.*B*D)/(D*D+2.*B*D)
C4 C *TOP* IS THE THICKNESS OF THE TOP PLATING
C4 *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
C4 BOT=AD/(1.+AC*AE)
DC TOP=AC*BOT
E8 WRITE(5,10)TOP
04 10 FORMAT('1',20X,'TOP='F10.4)
1E WRITE(5,20)BOT
04 20 FORMAT('1',20X,'BOT='F10.4)
04 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
04 S=Z*TOP
08 WRITE(5,30)S
08 30 FORMAT('1',22X,'S='F10.4)
08 P=2.*D*BOT+B*(TOP+BOT)
08 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
08 DITOP=D*(B*BOT+D*BOT)/P
08 WRITE(5,40)DITOP
08 40 FORMAT('1',18X,'DITOP='F10.4)
08 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
08 DIBOT=D*(B*TOP+D*BOT)/P
08 WRITE(5,50)DIBOT
08 50 FORMAT('1',18X,'DIBOT='F10.4)
08 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.

```



```

170      GMI=4.*D*D*((3.*R*B+2.*B*D)*TOP*BOT+(D*D+2.*B*D)*BOT*BOT)/P
E0      C      *TSTH* IS THE STRESS ON TOP DUE TO HOGGING
E0      C      *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
E0      C      *TSBH* IS THE STRESS ON BOT DUE TO HOGGING
E0      C      *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
E0      C      *STCRIT* IS THE CRITICAL STRESS ON TOP
E0      C      *SBCRIT* IS THE CRITICAL STRESS ON BOT
E0      TSTH=BM*DITOP/GMI+STR3T
F4      TSTS=BM*DI TOP/GMI+STR3T
28      STCRIT=TOP*TOP*E/(0.30396*S*S*(1.-POI*POI))
48      WRITE(5,60)TSTH
64      60  FORMAT(' ',19X,'TSTH='F10.3)
7E      WRITE(5,70)TSTS
9A      70  FORMAT(' ',19X,'TSTS='F10.3)
84      WRITE(5,80)STCRIT
D0      80  FORMAT(' ',17X,'STCRIT='F10.3)
EC      TSBH=BM*DIBOT/GMI+0.152222*HBOT*S*S/(BOT*BOT)
28      TSBS=BM*DI BOT/GMI+0.152222*HBOT*S*S/(BOT*BOT)
64      SBCRIT=BOT*BOT*E/(0.30396*S*S*(1.-POI*POI))
A4      WRITE(5,90)TSBH
C0      90  FORMAT(' ',19X,'TSBH='F10.3)
DA      WRITE(5,100)TSBS
F6      100 FORMAT(' ',19X,'TSBS='F10.3)
10      WRITE(5,110)SBCRIT
2C      110 FORMAT(' ',17X,'SBCRIT='F10.3)
48      END
(S)  .U      0640 [V] AL      0654 [V] B      0650 [V] DRA      0664 [V] D
(V)  POI      0674 [V] E      0700 [S] .R      0684 [V] DST      0680 [V] BM
(V)  W      06A0 [V] U      0700 [S] SQRT      06A4 [V] Z      06A8 [V] H TOP
(V)  HBOT      0684 [V] STR3T      0680 [V] STR3B      06C0 [V] A      06C4 [V] AB
(V)  AC      06D0 [V] AD      06F0 [V] AE      06F8 [V] BOT      06FC [V] TOP
(L)  10      0700 [S] @I      223A [L] 20      0700 [V] S      0270 [L] 30
(V)  P      0708 [V] DITOP      02F8 [L] 4A      0700 [V] DIBOT      0354 [L] 50
(V)  GMI      0710 [V] TSTH      0720 [V] TSTS      0724 [V] STCRIT      0464 [L] 60
(L)  70      04D0 [L] 80      0728 [V] TSBH      0720 [V] TSBS      0730 [V] SBCRIT
(L)  90      05F6 [L] 100      0620 [L] 110      0000 [S] .V
10      L

```

GRAM LABELS:

```

170 *MAIN*      2E3A .V      2C6A .COMP      298C ALOG      2E48 @I      27A4 .R
132 .ZERO      2C90 $6      27E4 .A      2D8E .MES      2C0C .W      2A94 EXP
9A AINT      2896 SQRT      2CD2 .RARG      2D2E .5      2D04 $8      2D36 .E
36 .0      2E0C .U      3F64

```

Y-POINTS:

```

A4 .R      27E4 .A      2896 SQRT      298C ALOG      2A94 EXP      2B9A AIN
0C .W      2C6A .COMP      2C90 $6      2CD2 .RARG      2D04 $8      2D2E .5
32 .ZERO      2D36 .ERCNT      2D38 .0      2D8E .MES      2E18 .U      2E3E .V
48 @I

```


TOP= 6.1748
 BOT= -5.5523
 S= 406.7124
 DITOP= 142.8614
 DIBOT= -72.6328
 TSTH= 24999.992
 TSTS= 24999.992
 STCRIT= 24999.988
 TSBH= 33241.842
 TSBS= 33241.842
 SBCRIT= 2 213.252

ND

VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 04/30/74

08 HANDLING CHARGE	\$.35 / JOB	.35
25 LINES PRINTED PR2	\$ 1.25 / K LN	.16
34 CARDS READ	\$ 1.50 / K CD	.13
40 PLOTTER VECTORS	\$.25 / 1000	.02
21 MODEL 70 SECONDS	\$25.00 / HOUR	.15
40 MODEL 80 SECONDS	\$12.50 / HOUR	.02
	TOTAL CHARGE \$.79

RRIR 490 14731 LOGGED OUT 04/30/74 21:26. \$ 27.07 LEFT AFTER 50 LOGINS.

It does not come as a surprise that the program does not work for the "1000 ft steel case" since in this situation we obtain STR3B DST which cannot be accepted and also this program is not prepared to handle.

$$AL = 1000$$

$$B = AL/5.75 = 173.913$$

$$DRA = B/3.3 = 52.7$$

$$POI = 0.3$$

$$E = 30 \times 10^6$$

$$DST = 25000$$

$$BM = 1.37143 * B * DRA * AL^2 = 12569.45 * 10^6$$

$$W = 0.30396 * (1 - POI^2) * DST / E = 230.503 * 10^{-6}$$

$$U = W = 15.182 * 10^{-3}$$

$$Z = 1/U = 65.867$$

$$HBOT = DRA = 52.7$$

$$STR3B = 0.152222 * HBOT * Z^2 = 34803.57 \quad DST$$

Since we understand that the program should work for $DST > STR3B$ we make the next run with $DST = 35000$ to prove that in such case a solution is obtained, and after proceed to obtain the actual solution for the case we were considering with $DST = 25000$.


```

04 C *AL* IS THE LENGHT
04 AL=1000.
08 C *B* IS THE BREADTH
08 B=L/5.75
18 C *DRA* IS THE DRAFT
18 DRA=B/3.3
24 C *D* IS THE DEPTH
24 D=L/14.
30 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
30 C FOR STEEL
30 POI=0.3
38 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
38 C FOR STEEL
38 E=30.*10.**6
40 C *DST* IS THE DESIGN STRESS
40 DST=35000.
54 C *BM* IS THE BENDING MOMENT
54 BM=1.37143*B*DRA*AL*AL
60 C *W* IS THE WEIGHT
60 W=0.37396*(1.-POI*P/I)*DST/E
80 C *U* IS THE DEFLECTION
80 U=SQRT(W)
98 C *Z* IS THE SECTION MODULUS
98 Z=I./U
A4 C *HTOP* IS THE HEAD OF WATER ON DECK
A4 HTOP=8.
AC C *HBOT* IS THE HEAD OF WATER ON BOTTOM
AC HBOT=DRA
B4 C *STRBT* IS THE STRESS AT THE BOTTOM
B4 STRBT=0.152222*HTOP*Z*Z
C8 C *STRBT* IS THE STRESS AT THE TOP
C8 STRBT=0.152222*HBOT*Z*Z
DC C *A* IS THE AREA OF THE SECTION
DC A=DST-STRBT
E8 C *AB* IS THE AREA OF THE SECTION
E8 AB=DST-STRBT
F4 C *AC* IS THE AREA OF THE SECTION
F4 AC=(AB*(B+D)-A*D)/(A*B)
20 C *AD* IS THE AREA OF THE SECTION
20 AD=BM*(B+D)/(A*4.*D*(D*D+2.*B*D))
78 C *AE* IS THE AREA OF THE SECTION
78 AE=(3.*B*B+2.*B*D)/D*(D+2.*B*D)
C4 C *TOP* IS THE THICKNESS OF THE TOP PLATING
C4 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
C4 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
C4 BOT=AD/(1.+AC/AE)
DC C *TOP* IS THE THICKNESS OF THE TOP PLATING
DC TOP=AC*BOT
08 C *WRITE* IS THE WRITE STATEMENT
08 WRITE(5,10)TOP
14 C *FORMAT* IS THE FORMAT STATEMENT
14 10 FORMAT(' ',22X,'TOP='F10.4)
E C *WRITE* IS THE WRITE STATEMENT
E WRITE(5,20)BOT
1A C *FORMAT* IS THE FORMAT STATEMENT
1A 20 FORMAT(' ',22X,'BOT='F10.4)
14 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
14 S=Z*TOP
20 C *WRITE* IS THE WRITE STATEMENT
20 WRITE(5,30)S
24 C *FORMAT* IS THE FORMAT STATEMENT
24 30 FORMAT(' ',22X,'S='F10.4)
28 C *P* IS THE PERIOD OF VIBRATION
28 P=2.*D*BOT+B*(TOP+BOT)
08 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
08 DITOP=D*(B*BOT+D*BOT)/P
08 C *WRITE* IS THE WRITE STATEMENT
08 WRITE(5,40)DITOP
08 C *FORMAT* IS THE FORMAT STATEMENT
08 40 FORMAT(' ',18X,'DITOP='F10.4)
04 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
04 DIBOT=D*(B*TOP+D*BOT)/P
08 C *WRITE* IS THE WRITE STATEMENT
08 WRITE(5,50)DIBOT
04 C *FORMAT* IS THE FORMAT STATEMENT
04 50 FORMAT(' ',18X,'DIBOT='F10.4)
08 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.

```



```

70      GMI=4.*D*D*( (3.*R+R/2.*B*D)*TOP*BOT+(D*D+2.*B*D)*BOT*BOT)/P
E0      C      *TSTH* IS THE STRESS ON TOP DUE TO HOGGING
E0      C      *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
E0      C      *TSBH* IS THE STRESS ON BOT DUE TO HOGGING
E0      C      *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
E0      C      *STCRIT* IS THE CRITICAL STRESS ON TOP
E0      C      *SBCRIT* IS THE CRITICAL STRESS ON BOT
E0      TSTH=BM*DITOP/GMI+STRBT
F4      TSTS=BM*DI*TOP/GMI+STRBT
08      STCRIT=TOP*TOP*E/(0.30396*S*S*(1.-POI*POI))
48      WRITE(5,60)TSTH
64      60  FORMAT(' ',19X,'TSTH='F10.3)
7E      WRITE(5,70)TSTS
9A      70  FORMAT(' ',19X,'TSTS='F10.3)
B4      WRITE(5,80)STCRIT
D0      80  FORMAT(' ',17X,'STCRIT='F10.3)
EC      TSBH=BM*DIBOT/GMI+0.152222*HBOT*S*S/(BOT*BOT)
E8      TSBS=BM*DI*BOT/GMI+0.152222*HBOT*S*S/(BOT*BOT)
64      SBCRIT=BOT*BOT*E/(0.30396*S*S*(1.-POI*POI))
A4      WRITE(5,90)TSBH
C0      90  FORMAT(' ',19X,'TSBH='F10.3)
DA      WRITE(5,100)TSBS
F6      100 FORMAT(' ',19X,'TSBS='F10.3)
E0      WRITE(5,110)SBCRIT
EC      110 FORMAT(' ',17X,'SBCRIT='F10.3)
E8      END
[S]  .U      0640 [V]  AL      0654 [V]  B      0650 [V]  DRA      0664 [V]  D
[V]  POI      0674 [V]  E      0600 [S]  .R      0684 [V]  DST      0680 [V]  BM
[V]  W      0640 [V]  U      0620 [S]  SQRT      06A4 [V]  Z      0645 [V]  HTOP
[V]  HBOT      0684 [V]  STRBT      0630 [V]  STR3B      06C0 [V]  A      06C4 [V]  AB
[V]  AC      06DC [V]  AD      06E [V]  AE      06F8 [V]  BOT      06FC [V]  TOP
[L]  10      0000 [S]  @I      0224 [L]  20      0700 [V]  S      0270 [L]  30
[V]  P      0708 [V]  DITOP      02E0 [L]  40      0720 [V]  DIBOT      0354 [L]  50
[V]  GMI      0710 [V]  TSTH      0720 [V]  TSTS      0724 [V]  STCRIT      0464 [L]  60
[L]  70      0400 [L]  80      0728 [V]  TSBH      0720 [V]  TSBS      0730 [V]  SBCRIT
[L]  90      05F6 [L]  100      0620 [L]  110      0000 [S]  .V
3
L

```

RAM LABELS:

70 *MAIN*	2EBA .V	2CEA .COMP	298C ALOG	2EC8 @I	27A4 .R
B2 .ZERO	2D10 \$6	27E4 .A	2E0E .MES	2C8C .W	2A94 EXP
1A AINT	2896 SQRT	2D52 .RARG	2DAE .5	2A94 AEXP	2DB6 .E
84 \$8	2DB6 .C	2E8C .U	3FE4		

XY-POINTS:

1A4 .R	27E4 .A	2896 SQRT	298C ALOG	2A94 AEXP	2A94 EXP
1A AINT	2C8C .W	2CEA .COMP	2D10 \$6	2D52 .RARG	2DB6 \$8
2AE .5	2DB2 .ZERO	2DB6 .ERCNT	2DB8 .0	2E0E .MES	2E98 .U
2BE .V	2EC8 @I				

TCP= 0.4626
 BCT= 9.7569
 S= 25.7509
 DITCP= 53.9186
 DIBCT= 17.5099
 TSIH= 34999.977
 TSTS= 34999.977
 STCRIT= 35000.016
 TSPH= 1 106.547
 TSES= 1 106.547

SBCRIT=***** (TOO HIGH OUTSIDE FORECAST RANGE)

ND

VIC OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 04/27/74

08 HANDLING CHARGE	\$.35 / JOB	.35
25 LINES PRINTED PR2	\$ 1.25 / K LN	.16
34 CARDS READ	\$ 1.50 / K CD	.13
60 PLOTTER VECTORS	\$.25 / 1000	.00
15 MODEL 70 SECONDS	\$25.20 / HOUR	.10
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
	TOTAL CHARGE \$.74

FEIR 490 14731 LOGGED OUT 04/29/74 20:46. \$ 32.45 LEFT AFTER 43 LOGINS.

Now let's consider our situation with $DST = 25000$. Once the situation is such that due to the approximation made in

$$\frac{S}{BOT} = Z$$

we end up with a $STR3B > DST$ we must consider a value $Z1$ lower than Z to bring $STR3B$ down. This can be also understood since it is acceptable to make here

$$\frac{S}{TOP} = Z$$

but because we obtain with the program as is $TOP \ll BOT$, making the approximation

$$\frac{S}{BOT} = Z$$

makes Z attain a much larger value than it actually should.

We must then consider $Z1 < Z = 65.8$ and may start by trying

$$Z1 = 35$$

The computer printout in this case is as follows.

STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

PAGE 1

```

04 C *AL* IS THE LENGTH
04 AL=142.*
08 R=AL/5.75
12 C *L* IS THE DEPTH
12 [L=1/23.3
24 R=L/14.*
30 C *EOL* IS THE POISSON'S RATIO OF THE MATERIAL
30 FOR STEEL
30 EOL=0.3
38 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
38 FOR STEEL
38 E=30.*10.**6
40 C *AST* IS THE DESIGN STRESS
40 DST=5000.*
54 C *BI* IS THE BENDING MOMENT
54 BI=1.37143*B*H*AL*AL
60 W=2.26396*(1.+EOL*EOL)*DST/E
70 W=5*BI*(.)
98 Z=1./W
04 *HTOP* IS THE HEAD OF WATER ON DECK
04 HTOP=5.*
08 *HBTM* IS THE HEAD OF WATER ON BOTTOM
08 HBTM=0.*
14 STRBT=0.152222*HTOP*Z*Z
18 Z1=35.*
22 STRSF=2.152222*HBTM*Z1*Z1
34 A=DST-STRBT
38 P=DST-STRSF
42 L=(2*B*(B+D)-A*L)/Z*(YD)
50 L'=B*D*(B+D)/(2.*4.*D*(D*D+2.*B*D))
54 L''=(2.*L*B+2.*B*D)/(D*D+2.*B*D)
60 C *TOP* IS THE THICKNESS OF THE TOP PLATING
60 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
60 BOT=1/D/(1.+AOL*AL)
64 TOP=AOL*BOT
68 WRITE(5,10)TOP
70 C 10 FORMAT('1',20Y,'TOP='F10.4)
74 WRITE(5,20)BOT
76 C 20 FORMAT('1',20Y,'BOT='F10.4)
80 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
80 S=2*TOP
88 WRITE(5,30)S
94 C 30 FORMAT('1',20Y,'S='F10.4)
98 D=2.*L*BOT+B*(TOP+BOT)
02 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
02 DITOP=D*(B*BOT+D*BOT)/P
04 WRITE(5,40)DITOP
06 C 40 FORMAT('1',18X,'DITOP='F10.4)
10 C *DIROT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
10 DIROT=D*(B*TOP+D*BOT)/P
12 WRITE(5,50)DIROT
14 C 50 FORMAT('1',18X,'DIROT='F10.4)

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STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATI

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178 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT Y=0.
179 C I1=4.*B.*D*(.25.*B+.25.*B*D)*TOP*BOT+(D*D+2.*B*D)*BOT*BT)/8
180 C *TSTH* IS THE STRESS ON TOP DUE TO BOWING
181 C *TSTB* IS THE STRESS ON TOP DUE TO SAGGING
182 C *TSPH* IS THE STRESS ON BOT DUE TO BOWING
183 C *TSPB* IS THE STRESS ON BOT DUE TO SAGGING
184 C *STCRIT* IS THE CRITICAL STRESS ON TOP
185 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
186 C ISTR=I1*D1100/26*I1+STRBT
187 C TSTR=I1*D1100/8*I1+STRBT
188 C ICSB1=TOP*TOP*B/2*(.25*396*B*B*(1.-FOI*FOI))
189 C *ITE (5,62)ISTR
190 C FEMT(T(1,1,19Y,ISTR)=F10.3)
191 C *ITE (5,70)TSTR
192 C FEMT(T(1,1,19Y,TSTR)=F110.3)
193 C *ITE (5,70)STRBT
194 C FEMT(T(1,1,17Y,STRBT)=F10.3)
195 C TSPH=I1*D1BOT/26*I1+.152222*H*BOT*B*B/2*(BOT*BOT)
196 C TSPB=I1*D1BOT/26*I1+.152222*H*BOT*B*B/2*(BOT*BOT)
197 C SBCRIT=BOT*BOT*B/2*(.25*396*B*B*(1.-OI*POI))
198 C *ITE (5,62)TSPH
199 C FEMT(T(1,1,19Y,TSPH)=F110.3)
200 C *ITE (5,100)TSPB
201 C FEMT(T(1,1,19Y,TSPB)=F10.3)
202 C *ITE (5,110)SBCRIT
203 C FEMT(T(1,1,17Y,SBCRIT)=F10.3)
204 C END

```

[S]	.	2464 [V]	AL	1.650 [V]	R	0.674 [V]	DRA	0.660 [V]	D
[V]	POI	2670 [V]	E	1.100 [S]	.6	0.690 [V]	DST	0.694 [V]	RM
[V]	*	2648 [V]	U	1.000 [S]	SRFT	0.670 [V]	Z	0.687 [V]	HTOP
[V]	HBOT	2540 [V]	STRBT	604 [V]	Z1	1.600 [V]	STR30	1.600 [V]	Z
[V]	AV	2608 [V]	AC	650 [V]	AD	0.700 [V]	AE	0.728 [V]	BOT
[V]	TBF	2270 [V]	10	0.000 [S]	BI	0.242 [L]	20	0.710 [V]	S
[L]	30	0.714 [V]	P	0.718 [V]	DITOP	0.300 [L]	40	0.710 [V]	DIBOT
[L]	50	0.720 [V]	GMI	0.720 [V]	TSTH	0.720 [V]	TSTS	0.734 [V]	STCRIT
[L]	60	0.442 [L]	70	0.408 [L]	80	0.728 [V]	TSPH	0.730 [V]	TSPB
[V]	SBCRIT	0.602 [L]	90	0.550 [L]	100	0.634 [L]	110	0.690 [S]	.V

GRAM LABELS:

270 *MAIN*	2804 *V	20FA *COMP	2990 *LOG	2ED8 *I	27B4 *?
002 *ZERO	2D24 *0	27F4 *A	201E *MES	2090 *W	2AA4 *E
02A *INT	28A6 *SRFT	2062 *RARG	20BE *5	2AA4 *AEXP	20C6 *E
094 *8	2D06 *0	2E9C *U	3FF4		

KEY-POINTS:

7B4 *0	27F4 *A	28A6 *SRFT	2990 *LOG	2AA4 *AEXP	2AA4 *EX
02A *INT	2090 *W	20FA *COMP	2022 *6	2D62 *RARG	2D94 *8
DRE *5	2D02 *ZERO	20C6 *FRONT	2D02 *0	2E1E *YES	2EA8 *0
E0E *V	2E0E *01				

TOP# 3.4224
 EOT# 5.7711
 S# 225.4135
 LITCR# 47.3658
 LIFCIT# 31.7423
 TSTLR# 24995.088
 TSTLR# 24995.088
 STCRIT# 24995.088
 TSC# 31022.045
 TSC# 31022.045
 SCORIT# 54897.211

NO

VIO OPERATING SYSTEM VERSION 1 REVISION 12 3/04/74 GENERATED 4/27/74

03	HANDLING CHARGE	\$.25 / JOB	.30
26	CHARGES ASSOCIATED PR1	\$ 1.25 / 1 LN	.14
25	CARDS READ	\$ 1.50 / 1 CD	.13
02	PLOTTER VECTORS	\$.55 / 1 JOB	.00
21	MODEL 7 SECONDS	\$25.00 / HOUR	.15
07	MODEL 21 SECONDS	\$12.50 / HOUR	.00
	TOTAL CHARGE \$.70

FEIR 400 14731 LOGGED OUT 4/29/74 21:59. \$ 0.06 LEFT AFTER 40 LOGINS.

This time we obtained a realistic but still unacceptable result; we notice that the bottom is being stressed to 31000 psi when it can only go up to 25000 psi; besides we assumed

$$Z1 = \frac{S}{BOT} = 35$$

and end up with

$$\frac{S}{BOT} = \frac{225.4}{5.07} = 44.46$$

This means that the Z1 value should be higher.

We may now try Z1 = 40; the computer results for this case are shown next.



STRENGTH OF SHIPS-OF THE NEUTRAL AXIS LOCATION

PAGE 1

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24 C *L* IS THE LENGTH
24 AL=L*1001.
26 B=AL/5.75
18 C *DRA* IS THE DRAFT
18 DRA=D/3.3
24 D=AL/14.
30 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
32 FOR STEEL
32 POI=.3
32 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
32 FOR STEEL
32 E=37.*10**6
40 C *DST* IS THE DESIGN STRESS
40 DST=25300.
54 C *BM* IS THE BENDING MOMENT
54 M=1.37143*B*DRA*AL*AL
60 U=0.34336*(1.+POI*POI)*DST/E
60 U=SQRT(U)
66 Z=1./U
74 *HTOP* IS THE HEAD OF WATER ON DECK
74 HTOP=R.
80 *HBOT* IS THE HEAD OF WATER ON BOTTOM
80 HBOT=DRA
84 STRST=3.152222*HTOP*Z+Z
86 Z1=40.
86 STR3B=3.152222*HBOT*Z1*Z1
88 A=DST-STRST
88 AB=DST-STR3B
90 AC=(AB*(B+D)-A*D)/(A*B)
90 AD=B*(B+D)/(A*4.*D*(D+2.*B+D))
90 AE=(3.*B*B+2.*B*D)/(D+D+2.*B*D)
100 C *TOP* IS THE THICKNESS OF THE TOP PLATING
100 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
100 PDT=AD/(1.+AC*AE)
104 TOP=AC*PDT
106 WRITE(5,10)TOP
110 10 FORMAT(' ',20X,'TOP='F10.4)
112 WRITE(5,20)BOT
114 20 FORMAT(' ',20X,'BOT='F10.4)
120 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
120 S=Z*TOP
126 WRITE(5,30)S
128 30 FORMAT(' ',20X,'S='F10.4)
130 D=2.*D*BOT+B*(TOP+BOT)
132 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
132 DITOP=(D*BOT+D*BOT)/D
134 WRITE(5,40)DITOP
136 40 FORMAT(' ',18X,'DITOP='F10.4)
140 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
140 DIBOT=D*(B*TOP+D*BOT)/D
144 WRITE(5,50)DIBOT
146 50 FORMAT(' ',18X,'DIBOT='F10.4)

```



STRESS OF SHEETS ON THE NEUTRAL AXIS LOCATION

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070 C *GMI* IS THE MOMENT OF INERTIA OF THE Y SECTION ABOUT 'A.
074 GMI=4.*D*D*(C.*R*R+2.*R*D)*IOT*ROT+(D*D+2.*R*D)*ROT*B*T)/2
080 C *TSTH* IS THE STRESS ON TOP DUE TO BOWING
084 C *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
088 C *TSPH* IS THE STRESS ON BOT DUE TO BOWING
092 C *TSPS* IS THE STRESS ON BOT DUE TO SAGGING
096 C *STCRIT* IS THE CRITICAL STRESS ON TOP
100 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
104 TSTH=RY*D(IOT/2)*I+STH2I
108 TSTS=-RY*D(IOT/2)*I+STH3I
112 STCRIT=TCP*IC2*E/(C*.37396*S*S*(1.-FOI*POI))
116 WRITE(5,40)TSTH
120 FORMAT(' ',19X,'TSTH='F10.3)
124 WRITE(5,70)TSTS
128 FORMAT(' ',19X,'TSTS='F10.3)
132 WRITE(5,80)STCRIT
136 FORMAT(' ',17X,'STCRIT='F10.3)
140 TSPH=RY*D(IOT/2)*I+2*.162222*H*ROT*S*E/(ROT*POI)
144 TSPS=-RY*D(IOT/2)*I+2*.162222*H*ROT*S*E/(ROT*POI)
148 SBCRIT=BOT*IC2*E/(C*.37396*S*S*(1.-FOI*POI))
152 WRITE(5,90)TSPH
156 FORMAT(' ',19X,'TSPH='F10.3)
160 WRITE(5,100)TSPS
164 FORMAT(' ',19X,'TSPS='F10.3)
168 WRITE(5,110)SBCRIT
172 FORMAT(' ',17X,'SBCRIT='F10.3)
176 END

```

[S]	0654 [V]	AL	0650 [V]	B	0674 [V]	DRA	0660 [V]	D
[V]	0670 [V]	E	0672 [S]	.R	0670 [V]	DST	0694 [V]	BM
[V]	0684 [V]	I	0670 [S]	SQRT	0670 [V]	Z	0680 [V]	HTOP
[V]	0680 [V]	STHBT	0604 [V]	Z1	0670 [V]	STF3B	0600 [V]	A
[V]	0608 [V]	AC	0650 [V]	AD	0720 [V]	AE	0708 [V]	ROT
[V]	0220 [L]	10	0222 [S]	0I	0222 [L]	20	0710 [V]	S
[L]	0714 [V]	P	0710 [V]	DIOTOP	0300 [L]	42	0710 [V]	DIOT
[L]	0720 [V]	GMI	0720 [V]	TSTH	0720 [V]	TSTS	0734 [V]	STCRIT
[L]	0442 [L]	70	0400 [L]	80	0703 [V]	TSPH	0730 [V]	TSPS
[V]	0508 [L]	92	0550 [L]	100	0604 [L]	110	0000 [S]	.V

PROGRAM LABELS:

0670 *MAIN*	20CA .V	20FA .CONF	2990 ALOG	2ED8 @I	2784 .A
2002 *ZERR	2020 \$6	27F4 .A	2E1E .MES	2C9C .W	2AA4 E
202A ATNT	28A6 SQRT	2062 .PARG	208E .S	2AA4 AEXP	2DC6 .E
2094 .S	20C6 .C	2E9C .U	3FF4		

TRY-POINTS:

27E4 .A	27F4 .A	28A6 SQRT	2990 ALOG	2AA4 AEXP	2AA4 E
202A ATNT	2C9C .W	20FA .CONF	2020 \$6	2D62 .PARG	2D94 \$
208E .S	2002 *ZERR	20C6 *ERRCNT	2DC8 .O	2E1E .MES	2EA8 .
20CE .V	2ED8 @I				

TOP= 3.2289
 BOT= 6.5253
 S= 199.4004
 CITOP= 44.1743
 CIROT= 27.2523
 TSTH= 24990.004
 TSTS= 24990.004
 SICRIT= 24990.004
 TSBH= 19514.004
 TSSS= 19514.004
 SECRTT=118337.001

ND

M VIO OPERATING SYSTEM VERSION 1 REVISION 012 (3/04/74 GENERATED 4/27/74

JOB HANDLING CHARGE	1	.35	/	JOB	.35
126 LINES PRINTED PER	1	1.25	/	K LN	.14
85 CARDS READ	1	1.50	/	K CD	.13
2 PLATTER VECTORS	1	.35	/	10.0	.00
15 MODEL 20 SECONDS	1	25.00	/	40.00	.10
20 MODEL 50 SECONDS	1	12.50	/	40.00	.00
TOTAL CHARGE					.74

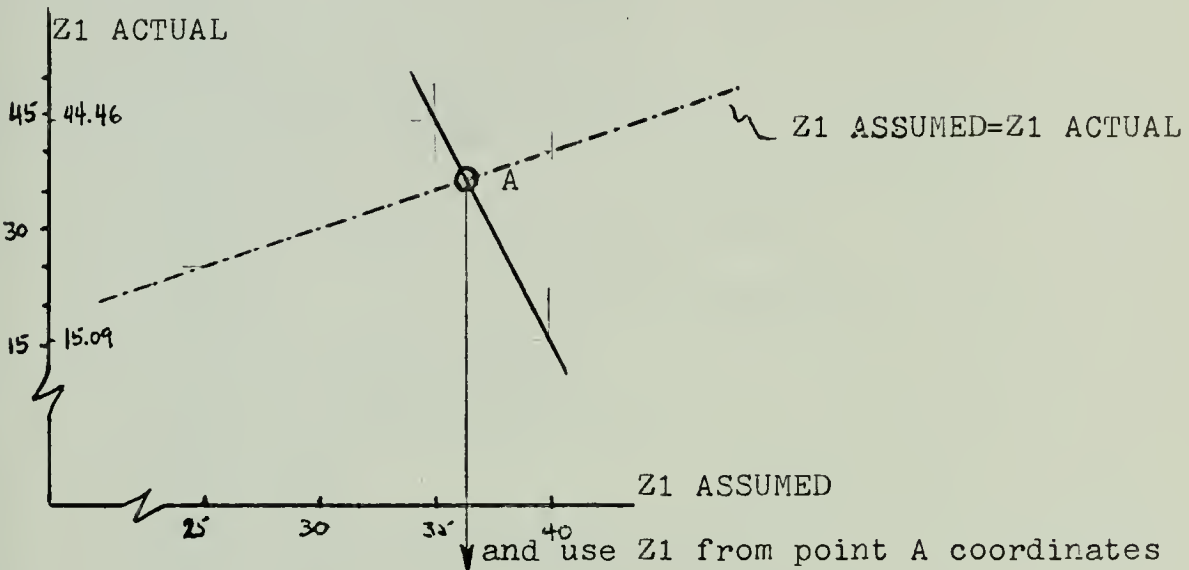
REIR 434 14731 LOGGED OUT 04/29/74 21:22. 6 28.60 LEFT AFTER 28 LOGINS.

In this case we assumed $Z1 = 40$ and ended up with

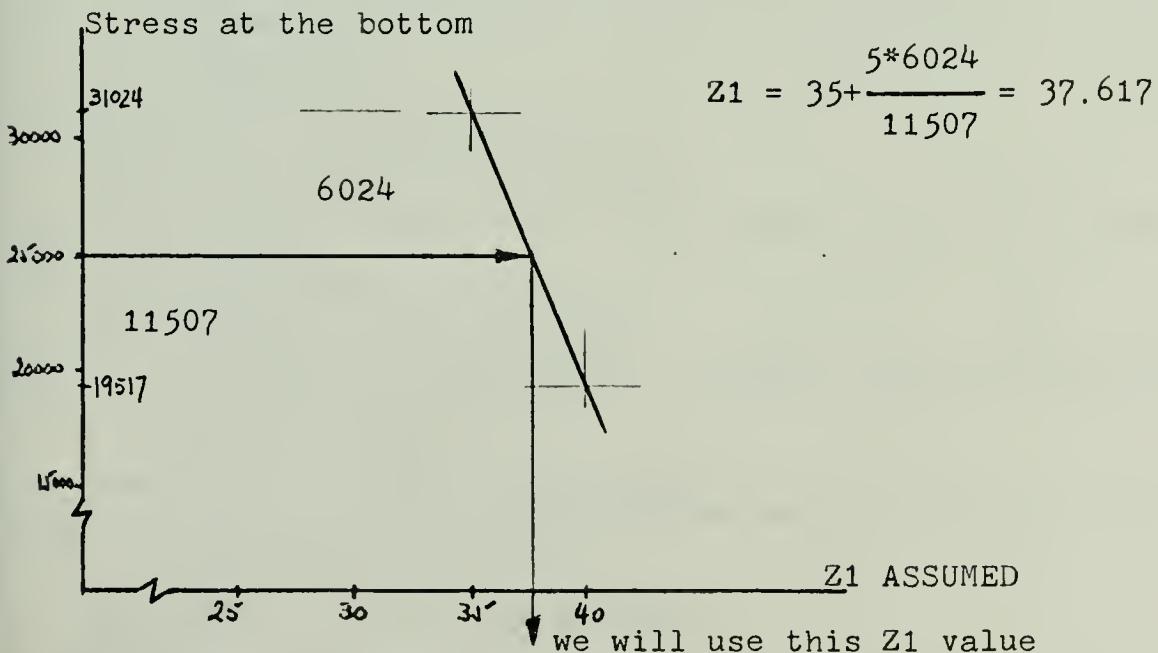
$$\frac{S}{BOT} = \frac{199.5}{6.59} = 15.09$$

and a stress at the bottom of only 19000 psi which is too small.

We may now try to use a better $Z1$ value and can do the following:



or which is "the same":



we will use this Z1 value

The results obtained with this Z1 value are the following:


```

14 C *AL* IS THE LENGTH
14 AL=1000.
17 C B=AL/5.75
18 C *DRA* IS THE DRAFT
18 DRA=B/3.3
24 C D=AL/14.
30 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
30 FOR STEEL
32 C POI=0.3
38 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
38 FOR STEEL
38 E=30.*10.**6
40 C *DST* IS THE DESIGN STRESS
40 DST=25000.
54 C *BM* IS THE BENDING MOMENT
54 BM=1.37143*B*DRA*AL*AL
60 C W=0.3396*(1.-POI+POI)*DST/E
80 U=SQRT(W)
92 Z=1./U
14 *HTOP* IS THE HEAD OF WATER ON DECK
14 HTOP=2.
16 *HROT* IS THE HEAD OF WATER ON BOTTOM
16 HROT=DRA
84 STR3T=7.152222+HTOP*Z*Z
88 Z1=35.+5.*424./11517.
90 STR3B=7.152222+HROT*Z1*Z1
92 A=DST-STR3T
94 AB=DST-STR3B
96 AC=(AB*(R+D)-A*D)/(A*R)
98 AD=BM*(R+D)/(A*4.*D*(D*D+2.*B*D))
100 AE=(3.*B*R+2.*B*D)/(D*D+2.*B*D)
102 C *TOP* IS THE THICKNESS OF THE TOP PLATING
102 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
102 BOT=AD/(1.+AC*AE)
104 TOP=AC*BOT
106 WRITE(5,10)TOP
108 10 FORMAT(' ',20X,'TOP='F10.4)
110 WRITE(5,20)BOT
112 20 FORMAT(' ',20X,'BOT='F10.4)
114 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
114 S=Z*TOP.
116 WRITE(5,30)S
118 30 FORMAT(' ',20X,'S='F10.4)
120 P=2.*D*BOT+B*(TOP+BOT)
122 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
122 DITOP=D*(B+BOT+D*BOT)/P
124 WRITE(5,40)DITOP
126 40 FORMAT(' ',18X,'DITOP='F10.4)
128 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
128 DIBOT=D*(B-TOP+D*BOT)/P
130 WRITE(5,50)DIBOT
132 50 FORMAT(' ',18X,'DIBOT='F10.4)

```



```

384 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.
384 GMI=4.*D*D*((3.*B*B+2.*B*D)*TOP*BOT+(D*D+2.*B*D)*BOT*BOT)/P
3F4 C *TSTH* IS THE STRESS ON TOP DUE TO HOGGING
3F4 C *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
3F4 C *TSBH* IS THE STRESS ON BOT DUE TO HOGGING
3F4 C *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
3F4 C *STCRIT* IS THE CRITICAL STRESS ON TOP
3F4 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
3F4 TSTH=BM*DITOP/GMI+STRBT
408 TSTS=BM*DI TOP/GMI+STRBT
410 STCRIT=TOP*TOP*E/(0.30396*S*S*(1.-POI*POI))
450 WRITE(5,60)TSTH
478 60 FORMAT(' ',19X,'TSTH='F10.3)
492 WRITE(5,70)TSTS
4AE 70 FORMAT(' ',19X,'TSTS='F10.3)
4C8 WRITE(5,80)STCRIT
4E4 80 FORMAT(' ',17X,'STCRIT='F10.3)
500 TSBH=BM*DI BOT/GMI+0.152222*H BOT*S*S/(BOT*BOT)
530 TSBS=BM*DI BOT/GMI+0.152222*H BOT*S*S/(BOT*BOT)
578 SBCRIT=BOT*BOT*E/(0.30396*S*S*(1.-POI*POI))
588 WRITE(5,90)TSBH
604 90 FORMAT(' ',19X,'TSBH='F10.3)
6EE WRITE(5,100)TSBS
60A 100 FORMAT(' ',19X,'TSBS='F10.3)
624 WRITE(5,110)SBCRIT
640 110 FORMAT(' ',17X,'SBCRIT='F10.3)
650 END
[S] •U 0660[V] AL 0668[V] B 0670[V] DRA 0678[V] D
[V] POI 0688[V] E 0690[S] •R 0698[V] DST 06A0[V] BM
[V] W 0684[V] U 0690[S] SQRT 0688[V] Z 0680[V] HTOP
[V] H BOT 06C8[V] STRBT 06D0[V] Z1 06E4[V] STR3B 06F0[V] A
[V] AB 06F0[V] AC 0704[V] AD 0718[V] AE 0720[V] BOT
[V] TOP 0218[L] 10 0720[S] @I 024E[L] 20 0728[V] S
[L] 30 0720[V] P 0732[V] DITOP 0300[L] 40 0734[V] DI BOT
[L] 50 0738[V] GMI 0744[V] TSTH 0748[V] TSTS 0740[V] STCRIT
[L] 60 04AE[L] 70 04E4[L] 80 0750[V] TSBH 0754[V] TSBS
[V] SBCRIT 05D4[L] 90 0604[L] 100 0640[L] 110 0600[S] •V
EQ L

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PROGRAM LABELS:

070 *MAIN*	2EE2 •V	2D12 •COMP	29B4 AL0G	2EF0 @I	27CC •R
0DA •ZERO	2D38 \$6	282C •A	2E36 •MES	2CB4 •W	2AFC EXP
042 AINT	28BE SQRT	2D7A •RARG	2DD6 •5	2ABC AEXP	2DDE •E
0AC \$8	2DDE •0	2E64 •U	402C		

KEY-POINTS:

7CC •R	280C •A	28BE SQRT	29B4 AL0G	2ABC AEXP	2AFC EXP
042 AINT	2CB4 •W	2D12 •COMP	2D38 \$6	2D7A •RARG	2D7C \$8
0D6 •5	2DDA •ZERO	2DDE •ERCNT	2DE0 •0	2E36 •MES	2E0A •U
EE6 •V	2EF0 @I				

TOP= 3.2486
 BOT= 5.7418
 S= 213.9701
 DITOP= 42.2107
 DIBOT= 29.2179
 TSTH= 25000.000
 TSTS= 25000.000
 STCRIT= 24999.992
 TSBH= 24788.316
 TSBS= 24788.316
 SBCKIT= 72100.937

END

VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 04/27/74 1

08	HANDLING CHARGE	\$.35 / JOB	.35
26	LINES PRINTED PR2	\$ 1.25 / K LN	.16
85	CARDS READ	\$ 1.50 / K CD	.13
00	PLOTTER VECTORS	\$.25 / 1000	.00
15	MODEL 70 SECONDS	\$25.00 / HOUR	.10
00	MODEL 80 SECONDS	\$12.50 / HOUR	.00
	TOTAL CHARGE \$.74

EIR 490 14731 LOGGED OUT 04/29/74 21:30. \$ 27.86 LEFT AFTER 40 LOGINS.

The results show that we have obtained

$$Z1 = \frac{S}{BOT} = \frac{213.9701}{5.7418} = 37.265$$

while it was assumed

$$Z1 = 37.617$$

We could further correct the result until $Z1$ assumed and $Z1$ actual would coincide but it seems that there is no need to refine more the solution for the purpose here involved.

Note that this procedure should also be followed for the other cases in order to bring the stress at the bottom to its maximum allowed DST for such cases as well.

STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

PAGE 1

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04 C *AL* IS THE LENGTH
04 L=10.0*
10 L=AL/3.7*
14 C *DRA* IS THE DRAFT
14 DRA=4/3.0*
18 D=AL/14*
24 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
24 C FOR ALUMINUM
27 POI=.28
33 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
33 C FOR ALUMINUM
33 E=10.0*10.0*6
40 C *DST* IS THE DESIGN STRESS
40 DST=9800.0*
44 C *BM* IS THE BENDING MOMENT
44 BM=1.37143*B*DRA*AL*AL
44 W=3.2286*(1.-POI*POI)*DST/E
48 W=1./W
54 C *HTOP* IS THE HEAD OF WATER ON DECK
54 *TOP*.
54 C *HROT* IS THE HEAD OF WATER ON BOTTOM
54 *HROT*.
60 C *STRB*
60 STRB=.15*222*HTOP*Z*Z
64 C *STRR*
64 STRR=.15*222*HROT*Z*Z
68 C *D*
68 D=HROT-STRR
74 C *D*
74 D=(AL*(B+1)-A*D)/(A*3)
78 C *D*
78 D=WM*(B+D)/(A*4.*D+(D*D+2.*B*D))
82 C *D*
82 D=(B.*D*B+2.*B*D)/(D*D+2.*B*D)
86 C *TOP* IS THE THICKNESS OF THE TOP PLATING
86 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
86 BOT=D/(1.+AC*AF)
90 C *TOP*
90 TOP=AC*BOT
94 C *WRITE(5,1)*TOP
94 *WRITE(5,2)*BOT
98 C *FORMAT(1,1),2XX,*TOP='F12.4)
98 *WRITE(5,2)*BOT
102 C *FORMAT(1,1),2XX,*BOT='F12.4)
102 *WRITE(5,3)*S
106 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
106 S=7*TOP
110 C *WRITE(5,3)*S
114 C *FORMAT(1,1),22X,*S='F12.4)
114 S=D.*D*HROT./B*(TOP+BOT)
118 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
118 DITOP=D*(B.-BOT+D*BOT)/P
122 C *WRITE(5,4)*DITOP
126 C *FORMAT(1,1),14X,*DITOP='F12.4)
130 C *DIRBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
130 DIRBOT=D*(B.-TOP+D*BOT)/P
134 C *WRITE(5,5)*DIRBOT
138 C *FORMAT(1,1),14X,*DIRBOT='F12.4)
142 C *I* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.

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2570 GMI=4.*D*D*(73.*R*R+2.*R*D)*TOP*BOT+(D*D+2.*R*D)*BOT*BOT)/P
2575 C *TOTH* IS THE STRESS ON TOP DUE TO HOGGING
2580 C *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
2585 C *TBRH* IS THE STRESS ON BOT DUE TO HOGGING
2590 C *TBSB* IS THE STRESS ON BOT DUE TO SAGGING
2595 C *STCRIT* IS THE CRITICAL STRESS ON TOP
2600 C *SBCRIT* IS THE CRITICAL STRESS ON BOT
2605 C TOTH=RM*DI+2R/GMI+G*ROT
2610 C TSTS=RM*DI+2R/GMI+G*ROT
2615 C STCRIT=TOTH*E/(1.32396*S*S*(1.-POI*POI))
2620 C WRITE(5,60)TOTH
2625 60 FORMAT(' ',10X,'TOTH=',F17.3)
2630 C WRITE(5,70)TSTS
2635 70 FORMAT(' ',10X,'TSTS=',F17.3)
2640 C WRITE(5,80)STCRIT
2645 80 FORMAT(' ',10X,'STCRIT=',F17.3)
2650 C HROT=RM*DI*(1/GMI+1.152222*HROT*S*S/(ROT*BOT))
2655 C TBRH=RM*DI*(1/GMI+1.152222*HROT*S*S/(ROT*BOT))
2660 C SBCRIT=BOT*HROT*E/(1.32396*S*S*(1.-POI*POI))
2665 C WRITE(5,90)TBRH
2670 90 FORMAT(' ',10X,'TBRH=',F17.3)
2675 C WRITE(5,100)TBSB
2680 100 FORMAT(' ',10X,'TBSB=',F17.3)
2685 C WRITE(5,110)SBCRIT
2690 110 FORMAT(' ',10X,'SBCRIT=',F17.3)
2695 C END
2700 .U 2640 [V] AL 2650 [V] R 2650 [V] DRA 0664 [V] D
2705 .POI 2674 [V] F 2674 [V] .R 2684 [V] DST 2688 [V] BM
2710 .W 2680 [V] H 2680 [V] SORT 2684 [V] Z 2684 [V] HTOP
2715 .HROT 2690 [V] STRBT 2690 [V] STRBB 2690 [V] A 2690 [V] AB
2720 .AC 2698 [V] AE 2698 [V] AE 2698 [V] ROT 2698 [V] TOP
2725 .14 2700 [V] S 2700 [V] S 2700 [V] S 2700 [V] S
2730 .R 2704 [V] DIRBT 2704 [V] DIRBT 2704 [V] DIRBT 2704 [V] DIRBT
2735 .GMI 2710 [V] TSTH 2710 [V] TSTH 2710 [V] TSTH 2710 [V] TSTH
2740 .7X 2720 [V] TBRH 2720 [V] TBRH 2720 [V] TBRH 2720 [V] TBRH
2745 .97 2728 [V] TSBS 2728 [V] TSBS 2728 [V] TSBS 2728 [V] TSBS
2750 L

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DESPAN LABELS:

2870 *MAIN*	2876 .V	2886 .COMP	2888 AL06	28C4 \$I	27A0 .R
28AE .ZERO	287C \$6	2857 .A	280A .MES	2888 .W	2A90 EX
2816 AINT	2882 SORT	284E .RARG	2DAA .5	2A90 AEXP	20B2 .E
2888 \$R	2882 .0	2888 .U	3FE0		

TRY-POINTS:

27A0 .R	2750 .A	2892 SORT	2988 AL06	2A90 AEXP	2A90 EX
2816 AINT	2888 .V	2854 .COMP	2D0C \$6	2D4E .RARG	2080 \$8
28AA .5	284E .ZERO	2812 .EPCNT	2DB4 .0	2E0A .MES	2E94 .U
288A .V	2884 \$I				

TOP# 204150
 DATE# 60212
 SS# 920360
 DT# 650880
 DT# 250420
 TST# 20000000
 TST# 25000000
 STC# 20000000
 TR# 14945.707
 TR# 10945.707
 TR# 10945.707

MIC OPERATING SYSTEM REVISION 1 REVISION 012 03/04/74 GENERATED 04/27/74

05 HANDLING CHARGE	\$.35 / JOB	.35
25 LINES PRINTED PER	\$ 1.25 / K LN	.14
44 CARDS READ	\$ 1.50 / 1 CD	.13
78 PLOTTER VECTORS	\$ 1.25 / 1220	.07
18 MODEL 72 SECONDS	\$ 5.50 / HOUR	.12
28 MODEL 87 SECONDS	\$ 2.50 / HOUR	.27
	TOTAL CHARGE \$.76

TR 490 14731 LOGGED OUT 04/27/74 18:28. \$ 34.67 LEFT AFTER 40 LOGINS.

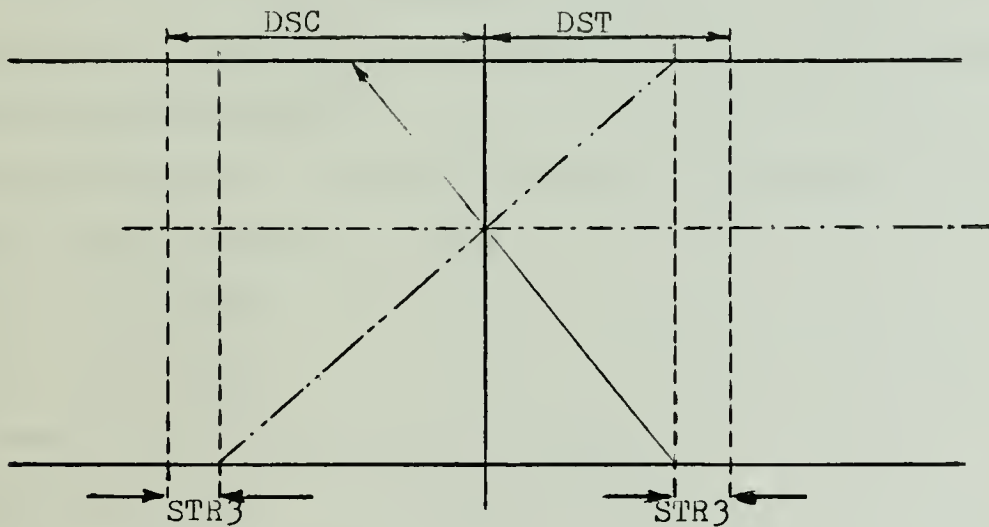
This 1000 ft aluminum case shows again a stress at the bottom much lower than DST. It should be corrected as done for "1000 ft steel" in the previous situation.

HOGGING MOMENT LARGER THAN SAGGING MOMENT

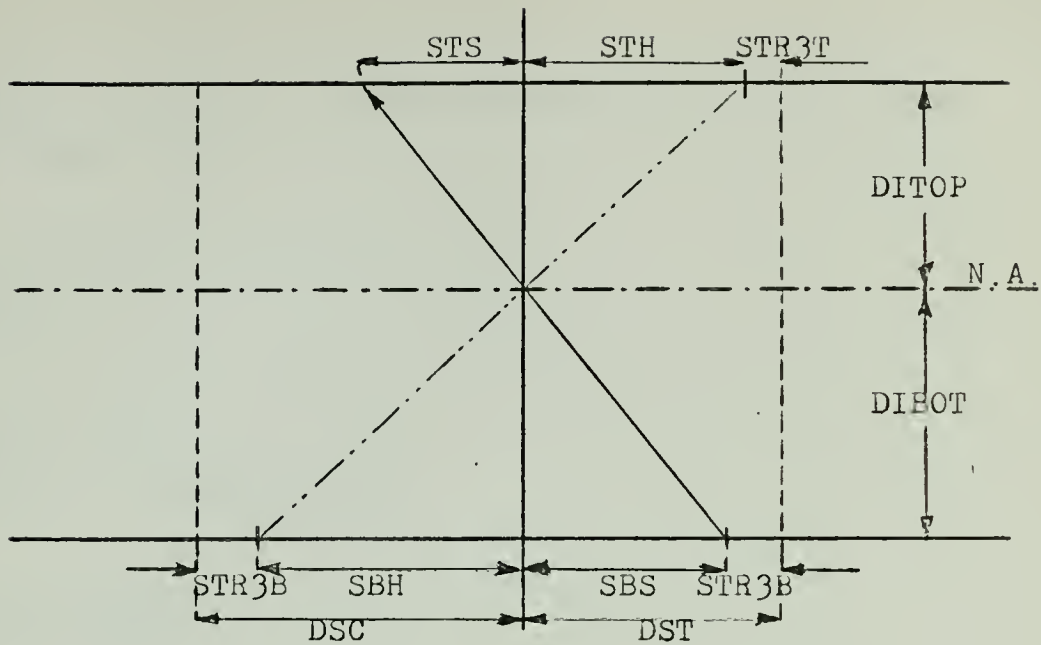
DESIGN STRESS IN COMPRESSION LARGER THAN DESIGN STRESS IN TENSION

LATERAL LOAD ALSO CONSIDERED

With the help of what was understood for the similar situation without the lateral load being included, we may here proceed at once to look at what the diagram of stress distributions might look like. Actually, the diagram that will be considered first is basically the same with only a small change due to the addition of the tertiary stresses; since these will probably be of the same magnitude on both top or bottom and assuming that their actual value does not differ considerably, what we may start looking into is a diagram as follows:



We would consider here the same basic distribution of longitudinal bending stresses thus obtaining



We will then expect to obtain $TOP > BOT$ and $DITOP < DIBOT$ and we may already consider the respective expressions for $DITOP$, $DIBOT$ and I with TS replaced by TOP

From the diagram we may consider the following equations:

$$STH * DIBOT = SBH * DITOP$$

and notice here that if we want to stress the material to the allowable design stresses we have

$$STH = DST - STR3T$$

$$SBH = DSC - STR3B$$

we may combine these equations to obtain

$$(DST - STR3T) * DIBOT = (DSC - STR3B) * DITOP$$

$STR3T$ and $STR3B$ are given by

$$STR3T = 0.152222 * H_{TOP} * \left(\frac{S}{TOP} \right)^2$$

$$STR3B = 0.152222 * H_{BOT} * \left(\frac{S}{BOT} \right)^2$$

anticipating here that $S = Z \cdot \text{BOT}$ because BOT is the thinner plating we have

$$\left(\frac{S}{\text{BOT}}\right)^2 = Z^2$$

and thus may write

$$\text{STR3B} = 0.152222 \cdot \text{HBOT} \cdot Z^2$$

Since

$$\left(\frac{S}{\text{TOP}}\right)^2 < \left(\frac{S}{\text{BOT}}\right)^2$$

because $\text{TOP} > \text{BOT}$, if we make

$$\left(\frac{S}{\text{TOP}}\right)^2 = Z^2$$

to simplify the equations that will follow we will understand that in doing so we are only increasing the value of STR3T.

This as we see from

$$\text{STH} = \text{DST} - \text{STR3T}$$

will make us have STH with a slightly lower value than the one that is allowed, so we are on the safe side when we consider

$$\left(\frac{S}{\text{TOP}}\right)^2 = Z^2$$

Now if we end up too much on the safe side and we have a TSTH too low we may bring it to a value closer to DST by considering a Z1 value as done and exemplified in the previous section and done again in this section for the case 1000 ft steel where for this particular exemple it is TSBH that needs correction as explained in detail .

We may then write

$$Z = \frac{1}{\sqrt{0.30396 * \frac{1 - POI * POI}{E} * DSC}}$$

$$STR3T = 0.152222 * HTOP * Z^2$$

$$STR3B = 0.152222 * HBOT * Z^2$$

and from

$$(DST - STR3T) * DIBOT = (DSC - STR3B) * DITOP$$

we have if we make

$$A = DST - STR3T$$

$$AB = DSC - STR3B$$

$$P = (2 * D + B) * TOP + B * BOT$$

$$A * \frac{D * TOP * (B + D)}{P} = AB * \frac{D * (B * BOT + D * TOP)}{P}$$

which gives after simplification

$$A * ((B + D) - AB * D) * TOP = AB * B * BOT$$

and making

$$AC = (A * (B + D) - AB * D) / (AB * B)$$

we have

$$BOT = AC * TOP$$

Following the suggestions of a previous section we will also use the equation that will make the bottom to be stressed the most in compression under the larger of the moments being considered (HM in this case).

We obtain

$$SBH = HM * DIBOT / I = AB$$

Replacing DIBOT and I by their expressions we obtain

$$AB = \frac{HM*D*TOP*(B+D)}{4*D^2*TOP*((3*B^2+2*B*D)*BOT+(D^2+2*B*D)*TOP)}$$

simplifying we end up with

$$TOP = \frac{HM*(B+D)}{AB*4*D*(D^2+2*B*D)} - \frac{3*B^2+2*B*D}{D^2+2*B*D} * BOT$$

now making

$$AD = HM*(B+D)/(AB*4*D*(D^2+2*B*D))$$

$$AE = (3*B^2+2*B*D)/(D^2+2*B*D)$$

we have

$$TOP = AD - AE*BOT$$

substituting here

$$BOT = AC*TOP$$

we obtain finally

$$TOP = \frac{AD}{1+AC*AE}$$

Considering here again the bottom as the location of the thinner plating (in this particular situation we are looking at), and knowing that it will be subject to a compressive stress of magnitude as large as DSC we may again write $S = Z*BOT$. We finally obtain the total stress on top and bottom in both tension and compression by adding the longitudinal bending stresses to the tertiary stresses.

$$TSTH = HM*DITOP/I + 0.152222*HTOP*S^2/TOP^2$$

$$TSTS = SM*DITOP/I + 0.152222*HTOP*S^2/TOP^2$$

$$TSBH = HM*DIBOT/I + STR3B$$

$$TSBS = SM*DIBOT/I + STR3B$$

NOTE: since usually the secondary stresses are assigned a constant value like 2000 psi or 3000 psi they can be accounted for when giving a value to the design stresses.

STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

PAGE 1

```

0004 C *AL* IS THE LENGTH
0004 AL=507.
000C R=AL/8.
0018 C *DR* IS THE DRAFT
0018 DR=A/3.
0024 B=AL/9.
0030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0030 FOR ALUMINUM
0030 POI=.33
0038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0038 FOR ALUMINUM
0038 E=12.*10**6
004C C *DST* IS THE DESIGN STRESS IN TENSION
004C DST=25000.
0054 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
0054 DSC=30000.
005C C *H* IS THE HOUSING BENDING MOMENT
005C H=1.37142*B*DR*AL*Q
0074 C *S* IS THE SHEAR BENDING MOMENT
0074 S=P*DR*AL*Q
0088 *V=0.41384*(1.-POI)*DSC/E
00A8 H=S*H/(.)
00B4 Z=1./U
00C0 C *HTOP* IS THE HEAD OF WATER ON DECK
00C0 HTOP=8.
00C8 C *HBT* IS THE HEAD OF WATER ON BOTTOM
00C8 HBT=DR.
00D0 STRBT=0.152225*HTOP*Z*Z
00E4 STRBH=0.152225*HBT*Z*Z
00F8 A=DST-STRBT
0104 AH=DSC-STRBH
0110 AC=(A*(H+D)-AH*Z)/(A*B*Z)
0148 AD=H*Z*(H+D)/(A*B*4.*D*(H+D+2.*B*D))
0194 AE=(2.*B*H+2.*B*D)/(H*D+2.*B*D)
01E0 C *TOP* IS THE THICKNESS OF THE TOP PLATING
01E0 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
01E0 TOP=AD/(1.+AC*AE)
01F8 BOT=AC*TOP
0204 *WRITE(5,12)TOP
0220 10 FORMAT('11',22X,'TOP='E11.4)
023A *WRITE(5,22)BOT
0256 20 FORMAT('11',22X,'BOT='E11.4)
0270 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0270 S=Z*BOT
027C *WRITE(5,30)S
0298 30 FORMAT('11',22X,'S='E11.4)
02B0 P=P.*D*TOP+2.*(TOP+BOT)
02D4 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
02D4 DITOP=D*(2.*BOT+3.*TOP)/3
02F8 *WRITE(5,40)DITOP
0314 40 FORMAT('11',18X,'DITOP='F12.4)
0330 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS

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STRENGTH OF SHIPS-ON THE EQUATORIAL AXIS LOCATION

DATE

```

0330      DIPOI=D*(R*TOP+*TOP)/P
0354      WRITE(5,55)DIPOI
0370      5  FORMAT(' ',1X,'DIPOI='F10.4)
0380      C      *POI* IS THE POI OF THE INERTIA OF THE Y SECTION ABOUT Z.
0380      GOI=4.*D*(D*(D*(R*Z+*Z)*L)*TOP*BOI+(D*D+2.*R*R)*TOP*(L*P))/P
03FC      C      *TSTH* IS THE STRESS ON TOP DUE TO BOWING
03FC      C      *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
03FC      C      *TSBH* IS THE STRESS ON BOT DUE TO BOWING
03FC      C      *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
03FC      C      *SCRIT* IS THE CRITICAL STRESS ON TOP
03FC      C      *SBCRI* IS THE CRITICAL STRESS ON BOT
03FC      TSTH=HY*DIPOI/3*E+*.1-2222*HTOP*S*S/(TOP*TOP)
0438      TSTS=SY*DIPOI/3*E+*.1-2222*HTOP*S*S/(TOP*TOP)
0474      SCRIT=TOP*BOI*Z/(.3-336*S*S+(1.-POI*POI))
0484      WRITE(5,60)TSTH
04D0      6  FORMAT(' ',1X,'TSTH='F10.3)
04EA      WRITE(5,70)TSTS
0526      7  FORMAT(' ',1X,'TSTS='F10.3)
0527      WRITE(5,80)SCRIT
0530      8  FORMAT(' ',1X,'SCRIT='F10.3)
0558      TSBH=HY*DIPOI/3*E+*TSTH*E
0560      TSBG=SY*DIPOI/3*E+*TSTS*E
0587      SBCRI=TOP*BOI*Z/(.3-336*S*S+(1.-POI*POI))
05C0      WRITE(5,90)TSBH
05DC      9  FORMAT(' ',1X,'TSBH='F10.3)
05F6      WRITE(5,100)TSBG
0612      10 FORMAT(' ',1X,'TSBG='F10.3)
0620      WRITE(5,110)SBCRI
0648      11  FORMAT(' ',1X,'SBCRI='F10.3)
0664      END

```

0 [S] •U	1668 [V] AL	628 [V] B	0678 [V] DRZ	068 [V] D
38 [V] POI	1680 [V] E	638 [V] •R	0690 [V] DST	16A4 [V] DSC
AC [V] H*	1684 [V] SM	638 [V] W	0694 [V] U	070 [S] SPTI
CF [V] Z	16CC [V] HTOP	600 [V] HBOT	0694 [V] STFBT	0600 [V] STBR
EF [V] A	16E4 [V] AB	0678 [V] AC	0690 [V] AD	071 [V] ZF
14 [V] TOP	0716 [V] BOT	0220 [L] 10	0700 [S] @J	0256 [L] 2.
1- [V] S	0282 [L] 30	0720 [V] P	0724 [V] DITOP	0314 [L] 4
22 [V] DIAGT	0370 [L] 50	0720 [V] GNJ	0728 [V] TSTH	0730 [V] TSTS
44 [V] STCRIT	0400 [L] 60	0584 [L] 70	0630 [L] SC	0744 [V] TSBH
44 [V] TSBS	0740 [V] SBCRI	0500 [L] 90	0612 [L] 100	0648 [L] 110
04 [S] •V				
XEQ	L			
0F				

PROGRAM LABELS:

2070 *MAIN*	2F56 •V	2084 ••OMP	20A8 /LOG	2E64 RT	2700 ••
204E •ZERR	20AC 56	2100 •A	21AA •MES	2028 •U	2A30 EX
28B6 AINT	28E2 SPT	20EE ••ARG	214A •S	2D20 68	2052 ••
2052 •0	2E28 •0	3F30			

TRY-POINTS:

27C0 •R	2800 •A	2882 SPT	29A8 /LOG	2AB0 EXP	2806 AI
---------	---------	----------	-----------	----------	---------

TOP= 1.368
 BOT= 2.270
 S= 10.2791
 DITOP= 26.333
 DIBOT= 29.2316
 TSH= 24469.754
 TSS= 18177.71
 STCHII= 45437.394
 TSP= 29999.006
 TSSS= 22931.332
 SPCRII= 30000.39

/ END

OF VIO OPERATING SYSTEM VERSION 1 REVISION 12 3/24/74 GENERATED 14/31/74

JOB HANDLING CHARGE	\$.25 / JOB	.35
130 LINES PRINTED PRI	\$ 1.25 / K Lh	.16
88 CARDS READ	\$ 1.50 / K Ch	.13
00 PLOTTER VECTORS	\$.65 / 100	.00
15 MODEL 7 SECONDS	\$25.00 / HOUR	.12
00 MODEL 8 SECONDS	\$12.50 / HOUR	.00
TOTAL CHARGE		.74

RREIR 490 14731 LOGGED OUT 4/11/74 22:19. 5 26.33 LEFT AFTER 01 LOGINS.

As we see the results obtained for the 500 ft aluminum case are acceptable and agree with all our considerations in the derivation of the formulas, namely TOP > BOT.

The following two runs, for 500 ft steel and 1000 ft aluminum show that the results are in disagreement with TOP > BOT used in the formulas. We will see next why this happens and derive the proper set of formulas to be used for such cases.


```

0204 C *AL* IS THE LENGTH
0204 AL=5000.
0204 B=AL/P.
0218 C *DRA* IS THE DRAFT
0218 DRA=B/3.
0224 D=AL/9.
0230 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0230 FOR STEEL
0230 POI=0.3
0238 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0238 FOR STEEL
0238 E=30.*10.**6
0240 C *DST* IS THE DESIGN STRESS IN TENSION
0240 DST=25000.
0254 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
0254 DSC=30000.
0258 C *HM* IS THE HOGGING BENDING MOMENT
0258 HM=1.37143*B*DRA*AL*AL
0274 C *SM* IS THE SAGGING BENDING MOMENT
0274 SM=B*DRA*AL*AL
0288 V=0.3/396*(1.-POI*POI)*DSC/E
0288 U=SQRT(V)
0288 Z=1./U
0290 C *HTOP* IS THE HEAD OF WATER ON DECK
0290 HTOP=8.
0298 C *HBOT* IS THE HEAD OF WATER ON BOTTOM
0298 HBOT=DRA
0300 STR3T=0.15222*HTOP*Z*Z
0304 STR3B=0.15222*HBOT*Z*Z
0308 A=DST-STR3T
0312 AB=DSC-STR3B
0316 AC=(A*(B+D)-AB*D)/(AB*B)
0320 AD=HM*(B+D)/(AB*4.*D*(D*D+2.*B*D))
0324 AE=(3.*B*D+2.*B*D)/(D*D+2.*B*D)
0328 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0328 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0332 TOP=AD/(1.+AC*AE)
0336 BOT=AC*TOP
0340 WRITE(5,10)TOP
0344 10 FORMAT(' ',20X,'TOP='F10.4)
0348 *WRITE(5,20)BOT
0352 20 FORMAT(' ',20X,'BOT='F10.4)
0356 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0356 S=Z*BOT
0360 WRITE(5,30)S
0364 30 FORMAT(' ',22X,'S='F10.4)
0368 P=2.*D*TOP+B*(TOP+BOT)
0372 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0372 DITOP=D*(B-BOT+D*TOP)/P
0376 WRITE(5,40)DITOP
0380 40 FORMAT(' ',18X,'DITOP='F10.4)
0384 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS

```



```

0330      DIBOT=D*(B*TOP+D*TOP)/P
0354      WRITE(5,50)DIBOT
0370      50  FORMAT(' ',18X,'DIBOT='F10.4)
0380      C   *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.
0380      GMI=4.*D*D*( (3.*B*B+2.*B*D)*TOP*BOT+(D*D+2.*B*D)*TOP*TOP)/P
03FC      C   *TSTH* IS THE STRESS ON TOP DUE TO HOGGING
03FC      C   *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
03FC      C   *TSBH* IS THE STRESS ON BOT DUE TO HOGGING
03FC      C   *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
03FC      C   *STCRIT* IS THE CRITICAL STRESS ON TOP
03FC      C   *SBCRIT* IS THE CRITICAL STRESS ON BOT
03FC      TSTH=HM*DITOP/GMI+0.152222*HTOP*S*S/(TOP*TOP)
0438      TSTS=SM*DITOP/GMI+0.152222*HTOP*S*S/(TOP*TOP)
0474      STCRIT=TOP*TOP*E/(0.30396*S*S*(1.-POI*POI))
0484      WRITE(5,60)TSTH
04D0      60  FORMAT(' ',19X,'TSTH='F10.3)
04EA      WRITE(5,70)TSTS
0526      70  FORMAT(' ',19X,'TSTS='F10.3)
0520      WRITE(5,80)STCRIT
0530      80  FORMAT(' ',17X,'STCRIT='F10.3)
0558      TSBH=HM*DIBOT/GMI+STR3B
0560      TSBS=SM*DIBOT/GMI+STR3B
0580      SBCRIT=BOT*BOT*E/(0.30396*S*S*(1.-POI*POI))
05C0      WRITE(5,90)TSBH
05D0      90  FORMAT(' ',19X,'TSBH='F10.3)
05F6      WRITE(5,100)TSBS
0612      100 FORMAT(' ',19X,'TSBS='F10.3)
0620      WRITE(5,110)SBCRIT
0648      110 FORMAT(' ',17X,'SBCRIT='F10.3)
0664      END
07(S)  .U      0668(V) AL      067 (V) B      0678(V) DRA      0680(V) D
08(V)  POI     0690(V) E      0000(S) .R      06A0(V) DST      06A3(V) DSC
09(V)  HM      06E8(V) SM      06B0(V) W      06C8(V) U      0000(S) SQRT
10(V)  Z      06D0(V) HTOP    06D4(V) HBOT    06D8(V) STR3T    06E0(V) STR3B
11(V)  A      06E8(V) AB      06E0(V) AC      0700(V) AD      0714(V) AE
12(V)  TOP     0710(V) BOT      0220(L) 10      0000(S) @I      0256(L) 20
13(V)  S      0298(L) 30      0724(V) P      0728(V) DITOP    0314(L) 40
14(V)  DIBOT   0370(L) 50      0730(V) GMI     0730(V) TSTH     0740(V) TSTS
15(V)  STCRIT  0400(L) 60      0500(L) 70      0530(L) 80      0748(V) TSBH
16(V)  TSBS   0750(V) SBCRIT  05D0(L) 90      0612(L) 100     0648(L) 110
17(S)  .V
18EQ
19

```

PROGRAM LABELS:

2070 *MAIN*	2E5A .V	208A .COMP	29AC ALOG	2E68 @I	27C4 .R
2052 .ZERO	20B0 \$6	2804 .A	2DAE .MES	2C2C .W	2AB4 EXP
28BA AINT	28B6 SQRT	2CF2 .RARG	2D4E .5	2D24 \$8	2D56 .E
2D56 .0	2E2C .U	3F84			

TRY-POINTS:

27C4 .R	2804 .A	28B6 SQRT	29AC ALOG	2AB4 EXP	28BA AI
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TOP= 0.3923
 BOT= 0.4748
 S= 28.5459
 DITOP= 29.2419
 DIRECT= 26.3137
 TSTH= 27045.305
 TSTS= 21466.824
 STCRIT= 27483.910
 TSBH= 37000.012
 TSBS= 24980.129
 SBCRIT= 29909.953

END

VIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 04/30/74 1

JOB HANDLING CHARGE	\$.35 / JOB	.35
30 LINES PRINTED PR2	\$ 1.25 / K LN	.16
88 CARDS READ	\$ 1.50 / K CD	.13
20 PLOTTER VECTORS	\$.25 / 1 00	.00
15 MODEL 70 SECONDS	\$25.00 / HOUR	.10
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
	TOTAL CHARGE \$.74

EIR 490 14731 LOGGED OUT 05/21/74 22:23. \$ 25.59 LEFT AFTER 02 LOGINS.


```

004 C *AL* IS THE LENGHT
004 AL=1000.
004 B=AL/5.75
018 C *DRA* IS THE DRAFT
018 DRA=B/3.3
024 D=AL/14.
030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
030 FOR ALUMINUM
030 POI=0.33
038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
038 FOR ALUMINUM
038 E=10.*12.*106
040 C *DST* IS THE DESIGN STRESS IN TENSION
040 DST=25000.
044 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
044 DSC=30000.
050 C *HM* IS THE HOGGING BENDING MOMENT
050 HM=1.37143*B*DRA*AL*AL
074 C *SM* IS THE SAGGING BENDING MOMENT
074 SM=B*DRA*AL*AL
088 W=0.30396*(1.-POI*POI)*DSC/E
088 U=SQRT(W)
084 Z=1./U
090 C *HTOP* IS THE HEAD OF WATER ON DECK
090 HTOP=8.
098 C *HBOT* IS THE HEAD OF WATER ON BOTTOM
098 HBOT=DRA
100 STPBT=0.152222*HTOP*Z*Z
104 STPBB=0.152222*HBOT*Z*Z
108 A=DST-STRBT
114 AB=DSC-STRBB
120 AC=(A*(B+D)-AB*D)/(AB*B)
128 AD=HM*(B+D)/(AB*4.**(D+D+2.*B*D))
134 AE=(3.*B*B+2.*B*D)/(D+D+2.*B*D)
140 C *TOP* IS THE THICKNESS OF THE TOP PLATING
140 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
140 TOP=AD/(1.+AC*AE)
148 BOT=AC*TOP
154 WRITE(5,10)TOP
160 10 FORMAT('1',20X,'TOP='F10.4)
164 WRITE(5,20)BOT
170 20 FORMAT('1',20X,'BOT='F10.4)
176 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
176 S=Z*BOT
180 C WRITE(5,30)S
184 30 FORMAT('1',20X,'S='F10.4)
188 P=2.*D*TOP+B*(TOP+BOT)
194 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
194 DITOP=D*(B+BOT+D*TOP)/P
200 WRITE(5,40)DITOP
204 40 FORMAT('1',10X,'DITOP='F10.4)
208 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS

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```

330      DIBOT=D*(B+TOP+D*TOP)/P
354      WRITE(5,50)DIBOT
370      50  FORMAT(' ',18X,'DIBOT='F10.4)
380      C   *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.
380      GMI=4.*D*D*(3.*B*B+2.*B*D)*TOP*BOT+(D*D+2.*B*D)*TOP*TOP)/P
3FC     C   *TSTH* IS THE STRESS ON TOP DUE TO HOGGING
3FC     C   *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
3FC     C   *TSBH* IS THE STRESS ON BOT DUE TO HOGGING
3FC     C   *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
3FC     C   *STCRIT* IS THE CRITICAL STRESS ON TOP
3FC     C   *SBCRIT* IS THE CRITICAL STRESS ON BOT
3FC     TSTH=HM*DITOP/GMI+C.152222*HTOP*S*S/(TOP*TOP)
38     TSTS=SM*DITOP/GMI+C.152222*HTOP*S*S/(TOP*TOP)
74     STCRIT=TOP*TOP*E/(0.30396*S*S*(1.-POI*POI))
84     WRITE(5,60)TSTH
80     60  FORMAT(' ',19X,'TSTH='F10.3)
80     WRITE(5,70)TSTS
86     70  FORMAT(' ',19X,'TSTS='F10.3)
80     WRITE(5,80)STCRIT
80     80  FORMAT(' ',17X,'STCRIT='F10.3)
58     TSBH=HM*DIBOT/GMI+STR3B
60     TSBS=SM*DIBOT/GMI+STR3B
80     SBCRIT=BOT*BOT*E/(0.30396*S*S*(1.-POI*POI))
80     WRITE(5,90)TSBH
80     90  FORMAT(' ',19X,'TSBH='F10.3)
86     WRITE(5,100)TSBS
80     100 FORMAT(' ',19X,'TSBS='F10.3)
80     WRITE(5,110)SBCRIT
80     110 FORMAT(' ',17X,'SBCRIT='F10.3)
84     END
(S)  .U      0668 [V] AL      0670 [V] B      0678 [V] DRA      0680 [V] D
(V)  POI     0690 [V] E      0700 [S] .R      0690 [V] DST      06A4 [V] DSC
(V)  HM      0684 [V] SM      06B8 [V] W      06C4 [V] U      0000 [S] SQRT
(V)  Z       0600 [V] HTOP    06D4 [V] HBOT    06D8 [V] STR3T    26E2 [V] STR3B
(V)  A       06E8 [V] AB      06E0 [V] AC      0700 [V] AD      0714 [V] AE
(V)  TOP     0720 [V] BOT      0720 [L] 10     0000 [S] @I     0256 [L] 20
(V)  S       0298 [L] 30     0720 [V] P      0720 [V] DITOP    0314 [L] 40
(V)  DIBOT   0370 [L] 50     0734 [V] GMI     0740 [V] TSTH     0744 [V] TSTS
(V)  STCRIT  0400 [L] 60     0500 [L] 70     0530 [L] 80     0740 [V] TSBH
(V)  TSBS    0754 [V] SBCRIT  0500 [L] 90     0612 [L] 100    0648 [L] 110
(S)  .V
0     L

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RAM LABELS:

70 *MAIN*	2E5E .V	2CPE .COMP	2980 ALOG	2E6C @I	27C8 .R
56 .ZERO	2C84 \$6	2808 .A	2DB2 .MES	2C30 .W	2AB8 EXP
BE AINT	28BA SQRT	2CF6 .RAFG	2D52 .5	2D28 \$8	2D5A .ERC
5A .0	2E30 .U	3F88			

XY-POINTS:

208 .R	2808 .A	28BA SQRT	2980 ALOG	2AB8 EXP	28BE AINT
--------	---------	-----------	-----------	----------	-----------

TOP= 3.1023
 BOT= 3.8359
 S= 134.5649
 DITOP= 38.4762
 DIBOT= 32.9524
 TSTH= 25792.578
 TSTS= 19427.625
 STCRIT= 19622.289
 TSBH= 29999.980
 TSBS= 24548.805
 SBCRIT= 30000.031

END

VIC OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 04/30/74 1

JOB HANDLING CHARGE	\$.35 / JOB	.35
30 LINES PRINTED PR2	\$ 1.25 / K LN	.15
88 CARDS READ	\$ 1.50 / K CD	.13
00 PLOTTER VECTORS	\$.25 / 1000	.00
22 MODEL 70 SECONDS	\$25.00 / HOUR	.15
00 MODEL 80 SECONDS	\$12.50 / HOUR	.00
	TOTAL CHARGE \$.79

EIR 490 14731 LOGGED OUT 25/01/74 22:26. \$ 24.80 LEFT AFTER 03 LOGINS.

STEEL

$$POI = 0.3$$

$$E = 30 \cdot 10^6$$

$$AL = 500$$

$$B = AL/8 = 62.5$$

$$DRA = B/3 = 20.833$$

$$D = AL/9 = 55.556$$

$$HM = 1.37143 \cdot B \cdot DRA \cdot AL^2 = 44642.187 \cdot 10^4$$

$$SM = B \cdot DRA \cdot AL^2 = 32551.562 \cdot 10^4$$

$$W = 0.30396 \cdot (1 - POI^2) \cdot DSC/E = 276.6 \cdot 10^{-6}$$

$$Z = 60.129$$

$$HTOP = 8$$

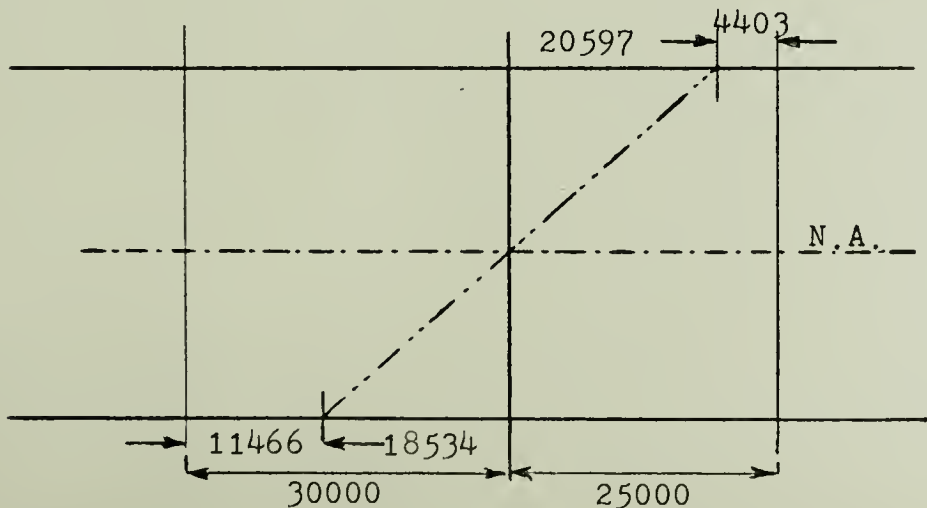
$$HBOT = 20.833$$

$$STR3T = 0.152222 \cdot HTOP \cdot Z^2 = 4402.865 \quad \text{note that this value}$$

should be corrected for the Z
approximation and actually is
slightly smaller

$$STR3B = 0.152222 \cdot HBOT \cdot Z^2 = 11465.61$$

We see from the tertiary stresses obtained that we end up with the following diagram of stress distributions:



From the diagram we see that the neutral axis will end up closer to the bottom and so we must consider BOT > TOP in the formulas.

The same type of diagram will be obtained for 1000' aluminum case since the tertiary stresses obtained in such case are

$$\text{STR3T} = 1498.73$$

$$\text{STR3B} = 9873.078$$

For the 1000' steel case the same type of diagram will occur due to the tertiary stresses being

$$\text{STR3T} = 4402.86$$

$$\text{STR3B} = 29004.$$

Besides this situation requiring us also to consider the neutral axis closer to the bottom, still another problem arises since STR3B is larger than DST=25000. This case is solved by properly correcting the inaccuracy that was introduced by the Z approximation as exemplified next.

Now let's derive the formulas capable of handling the 500' steel case, the 1000' steel case and the 1000' aluminum case.

We must consider $BOT > TOP$ and $DITOP > DIBOT$

$$DITOP = D*(B+D)*BOT/P$$

$$DIBOT = D*(B*TOP+D*BOT)/P$$

$$I = 4*D^2*BOT*((3*B^2+2*B*D)*TOP+(D^2+2*B*D)*BOT)/P$$

where

$$P = 2*D*BOT+B*(TOP+BOT)$$

Starting again with the equation

$$STH*DIBOT = SBH*DITOP$$

and making

$$A = DST-STR3T$$

$$AB = DSC-STR3B$$

we obtain

$$A*DIBOT = AB*DITOP$$

Knowing that now the top is the thinner plating we anticipate

$$S = Z*TOP$$

and may write

$$STR3T = 0.152222*HTOP*Z^2$$

$$STR3B = 0.152222*HBOT*Z^2$$

where the Z value in STR3B should be $\frac{S}{BOT}$ which has a smaller

value than Z.

We then obtain:

$$A*D*(B*TOP+D*BOT)/P = AB*D*(B+D)*BOT/P$$

Simplifying and making

$$AC = (AB*(B+D)-A*D)/(A*B)$$

we obtain

$$TOP = AC*BOT$$

Making now the top to be stressed the most in tension we have.

$$STH = HM*DITOP/I = A$$

Replacing DITOP and I by their equations we have

$$A = \frac{HM*D*(B+D)*BOT}{4*D^2*BOT*((3*B^2+2*B*D)*TOP+(D^2+2*B*D)*BOT)}$$

and obtain

$$BOT = \frac{HM*(B+D)}{A*4*D*(D^2+2*B*D)} - \frac{3*B^2+2*B*D}{D^2+2*B*D}*TOP$$

Making

$$AD = HM*(B+D)/(A*4*D*(D^2+2*B*D))$$

$$AE = (3*B^2+2*B*D)/(D^2+2*B*D)$$

the formulas become

$$BOT = AD-AE*TOP$$

and substituting

$$TOP = AC*BOT$$

we finally have

$$BOT = \frac{AD}{1+AC*AE}$$

The results obtained for the examples chosen are presented next:


```

0044 C *AL* IS THE LENGTH
0044 AL=B*Z.
0046 L=AL/Z.
0048 C *DR* IS THE DRAFT
0048 DR=B/3.
0050 D=L/4.
0052 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0052 FOR STEEL
0054 POI=.3.
0056 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0056 FOR STEEL
0058 E=30.*10.**6.
0060 C *DST* IS THE DESIGN STRESS IN TENSION
0060 DST=25000.
0062 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
0062 DSC=30000.
0064 C *HM* IS THE HOGBING BENDING MOMENT
0064 HM=1.37142*B*DR*AL*AL
0066 C *BM* IS THE SAGGING BENDING MOMENT
0066 BM=D*DR*AL*AL
0068 Z=.32222*(1.+POI)*POI)*DSC/E
0070 G=SQRT(1.)
0072 Z=1./Z
0074 *HTOP* IS THE HEAD OF WATER ON DECK
0074 HTOP=8.
0076 *HBTM* IS THE HEAD OF WATER ON BOTTOM
0076 HBTM=D*G
0078 STRBT=.152222*HTOP*Z*Z
0080 STRBB=.152222*HBTM*Z*Z
0082 A=DST-STRBT
0084 AD=DSC-STRBB
0086 AC=(A*B*(B+D)-A*Z)/(A*B)
0088 AD=H*(B+D)/(A*4.*D*(B+D+2.*B*D))
0090 AB=(B.*B+2.*D*B)/(1.+L+2.*B*D)
0092 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0092 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0094 BOT=AD/(1.+AC*AB)
0096 TOP=AC*BOT
0098 WRITE(5,10)TOP
0100 10 FORMAT('1',22X,'TOP='F10.4)
0102 WRITE(5,20)BOT
0104 20 FORMAT('1',22X,'BOT='F10.4)
0106 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0106 S=Z*TOP
0108 WRITE(5,30)S
0110 30 FORMAT('1',22X,'S='F10.4)
0112 P=2.*D*POI+B*(TOP+BOT)
0114 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0114 DITOP=D*(P+BOT+D*BOT)/P
0116 WRITE(5,40)DITOP
0118 40 FORMAT('1',18X,'DITOP='F10.4)
0120 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS

```


TOP= 1.2711
 ROI= 1.4576
 S= 22.3 57
 OI TOP= 29.2413
 OI BOT= 26.2134
 TST= 24999.100
 TSIS= 19421.510
 SICHIT= 29999.100
 TSKH= 26075.150
 TSKS= 21055.253
 SACHIT= 48627.762

SM

MIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 5/02/74

06 HANDLING CHARGE	\$	0.35	/	JO		0.35
132 LINES (50/10) PKI	\$	1.25	/	K/L		0.16
88 CARDS 25/L	\$	1.50	/	50		0.13
14 PLOTTER 10/100	\$	0.25	/	100		0.00
21 MODEL 7 50/100	\$	25.00	/	100		0.1
10 MODEL 8 50/100	\$	12.50	/	100		0.1
				TOTAL CHARGE	\$	0.79

AIR 49 147.31 LOGGED OUT 29/09/74 22:41. \$ 17.92 LEFT AFTER 12 LOGINS.

We could have obtained $TSBH = 30000$ by correcting the approximation used $Z = S/BOT$ as is done for the 1000' steel case that follows next.

In this case we already anticipate the use of a $Z1$ value as done in the previous section and according to the output obtained we will then obtain the correct result.


```

004 C *AL* IS THE LENGHT
004 AL=100.0
004 B=AL/5.75
018 C *DRA* IS THE DRAFT
018 DRA=B/3.3
024 C *L* IS THE LENGHT
024 L=AL/14.0
030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
030 FOR STEEL
030 POI=0.3
038 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
038 FOR STEEL
038 E=30.*10.**6
040 C *DST* IS THE DESIGN STRESS IN TENSION
040 DST=284.0
054 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
054 DSC=310.0
060 C *HMB* IS THE HOBGING BENDING MOMENT
060 HMB=1.27143*L*DRA*AL*AL
074 C *SMB* IS THE SAGGING BENDING MOMENT
074 SMB=L*L*L*L*AL*AL
088 C *DSC/E* IS THE DESIGN STRESS IN COMPRESSION
088 DSC/E=0.0103333*(1.-POI*POI)*DSC/E
094 C *Z* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE TOP OF THE DECK
094 Z=1.70
100 C *HTOP* IS THE HEAD OF WATER ON DECK
100 HTOP=0.
108 C *HBTM* IS THE HEAD OF WATER ON BOTTOM
108 HBTM=DRA
114 C *STRBT* IS THE STRESS IN THE BOTTOM PLATING
114 STRBT=.152222*HTOP*Z*Z
120 C *STRTP* IS THE STRESS IN THE TOP PLATING
120 STRTP=.152222*HBTM*Z1*Z1
126 C *A* IS THE AREA OF THE DECK PLATING
126 A=(B*(D+D1)-A*D1)/(A*B)
132 C *D* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE TOP OF THE DECK
132 D=(B*(D+D1)/(A*4.*D1+(D*D+2.*B*D)))
138 C *D1* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE TOP OF THE DECK
138 D1=(3.*D*B+2.*D*D)/(D*D+2.*B*D)
144 C *TOP* IS THE THICKNESS OF THE TOP PLATING
144 TOP=AD/(1.+AC*AE)
150 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
150 BOT=AD/(1.+AC*AE)
156 C *ZTOP* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE TOP OF THE DECK
156 ZTOP=(5/12)*TOP
162 C *ZBOT* IS THE DISTANCE FROM THE NEUTRAL AXIS TO THE BOTTOM OF THE DECK
162 ZBOT=(5/12)*BOT
168 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
168 S=L/24
174 C *I* IS THE MOMENT OF INERTIA OF THE DECK PLATING
174 I=(B*TOP**3)/12
180 C *I* IS THE MOMENT OF INERTIA OF THE BOTTOM PLATING
180 I=(B*BOT**3)/12
186 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
186 DITOP=D*(B*BOT+D*BOT)/P
192 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
192 DIBOT=D*(B*BOT+D*BOT)/P
198 C *I* IS THE MOMENT OF INERTIA OF THE DECK PLATING
198 I=(B*TOP**3)/12
204 C *I* IS THE MOMENT OF INERTIA OF THE BOTTOM PLATING
204 I=(B*BOT**3)/12

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STRENGTH OF CHIP ON THE NEUTRAL AXIS LOCATION

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0388 C      *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0389      *DIBOT=D*(B+TOP+D*POI)/P
0390      WRITE(5,6)DIBOT
0391 50 FORMAT(' ',19X,'DIBOT='F10.4)
0392 C      *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.
0393      *GMI=4.*(B*D*(3.*B**2+2.*B*D)*TOP*BOT+(D*D+2.*B*D)*BOT*BOT)/P
0394 C      *TSTH* IS THE STRESS ON TOP DUE TO HOGGING
0395 C      *TSTH* IS THE STRESS ON TOP DUE TO SAGGING
0396 C      *TSTB* IS THE STRESS ON BOT DUE TO HOGGING
0397 C      *TSTB* IS THE STRESS ON BOT DUE TO SAGGING
0398 C      *STR3T* IS THE CRITICAL STRESS ON TOP
0399 C      *STR3B* IS THE CRITICAL STRESS ON BOT
0400      *TSTHCRIT=TSTH/GMI+STR3T
0401      *TSTBCRIT=TSTB/GMI+STR3B
0402      *STR3T=PI*B*D**3*(1.37396+S**3*(1.-POI*POI))
0403      WRITE(5,7)TSTH
0404 60 FORMAT(' ',19X,'TSTH='F10.3)
0405      WRITE(5,7)TSTB
0406 70 FORMAT(' ',19X,'TSTB='F10.3)
0407      WRITE(5,8)STR3T
0408 80 FORMAT(' ',17X,'STR3T='F10.3)
0409      *TSRH=PI*B*D*(BOT/BOT+1.152222*H*BOT*S*S/(BOT*BOT))
0410      *TSRS=PI*B*D*(BOT/BOT+1.152222*H*BOT*S*S/(BOT*BOT))
0411      *STR3T=BOT*BOT*E/(1.37396*S**3*(1.-POI*POI))
0412      WRITE(5,9)TSRH
0413 90 FORMAT(' ',19X,'TSRH='F10.3)
0414      WRITE(5,10)TSRS
0415 100 FORMAT(' ',19X,'TSRS='F10.3)
0416      WRITE(5,11)STR3T
0417 110 FORMAT(' ',17X,'STR3T='F10.3)
0418      END
0419
0420 [U] 0670[V] AL      0679[V] B      0680[V] DRA      0688[V] D
0421 [V] POI      0698[V] E      0709[S] 0R      06A8[V] DST      06B1[V] DSC
0422 [V] HM      06C0[V] SM      06C4[V] W      06D0[V] U      0000[S] SQRT
0423 [V] Z      06D5[V] HTOP      06E1[V] HBOT      06E4[V] STR3T      06E0[V] Z1
0424 [V] STR3B      06F8[V] A      06F0[V] AR      0700[V] AC      0714[V] AD
0425 [V] AR      0730[V] BOT      0734[V] TOP      0228[L] 10      0000[S] 01
0426 [L] 00      0738[V] S      02A0[L] 30      0730[V] P      0747[V] DITOP
0427 [L] 40      0744[V] DIBOT      0375[L] 50      0748[V] GMI      0754[V] TSTH
0428 [V] TSTH      0750[V] STR3T      0485[L] 60      048E[L] 70      04F4[L] 80
0429 [V] TSBH      0764[V] TSBS      0765[V] STR3T      05E4[L] 90      061A[L] 100
0430 [L] 110      0000[S] 0V
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PROGRAM LABELS:

0070 *MAIN*	2EF2 0V	2D22 0COMP	29C4 ALOG	2F00 0I	27DC 0
2DEA 0ZERC	2D42 06	2F1C 0A	2E46 0YES	2CC4 0W	2ACC 0E
2C52 AINT	28CE SQRT	2DFA 0RARG	2DE6 05	2ACC AEXP	2DEE 0
2DBC 08	2DEE 0C	2E04 0U	421C		

TRY-POINTS:

TCP= 3.5788
 BCT= 3.6860
 S= 215.1802
 DITCP= 36.4263
 DIRECT= 36.3423
 TSTH= 21020.400
 TSTS= 1.421.525
 STCRIT= 20909.988
 TSHH= 42512.700
 TSES= 4.249.328
 SBCRIT= 31824.254

EID

CP VIC OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/02/74

JOB HANDLING CHARGE	4	.35 / JOB	.35
131 LINES PRINTED PER	4	1.25 / K LN	.16
89 CARDS READ	4	1.52 / K CD	.13
40 PLOTTER VECTORS	4	.25 / 1000	.00
15 MODEL 70 SECONDS	275.72	/ HOUR	.10
00 MODEL 80 SECONDS	212.57	/ HOUR	.02
TOTAL CHARGE			.74

PEIR 492 14731 LOGGED OUT 29/03/75 02:48. \$ 16.44 LEFT AFTER 14 LOGINS

STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

PAGE

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0204 C *AL* IS THE LENGTH
0204 AL=1002.
0206 B=AL/5.75
0218 C *DRA* IS THE DRAFT
0218 DRA=R/3.2
0224 D=AL/14.
0230 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0230 FOR STEEL
0230 POI=.3
0238 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0238 FOR STEEL
0238 E=30.*10**6
0240 C *DST* IS THE DESIG. STRESS IN TENSION
0240 DST=25.02.
0254 C *DSC* IS THE DESIG. STRESS IN COMPRESSION
0254 DSC=30.02.
0256 C *HM* IS THE HOGGING BENDING MOMENT
0256 HM=1.37143*E*(DRA*AL+Aj)
0274 C *SM* IS THE SAGGING BENDING MOMENT
0274 SM=E*(DRA*AL+Aj)
0288 W=0.3735*(1.-POI*POI)*DSC/E
0288 U=SQRT(W)
0284 Z=1./U
0290 *HTOP* IS THE HEAD OF WATER ON DECK
0290 HTOP=5.
0298 *HROT* IS THE HEAD OF WATER ON BOTTOM
0298 HROT=DRA.
0300 STRBT=-.152222*HTOP*Z*Z
0304 Z1=4.
0306 STRSB=-.152222*HROT*Z1*Z1
0310 A=DST-STRBT
0310 AB=DSC-STRSB
0318 AC=(AD*(D+D)+A*W)/(A*B)
0320 AD=H*(D+D)/(A*B*D*(D+D+2.*B*D))
0324 AE=(3.*W+2.*B*D)/(D+D+2.*B*D)
0328 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0328 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0328 BOT=AD/(1.+AC+AE)
0330 TOP=AC*BOT
0332 WRITE(5,10)TOP
0338 10 FORMAT('1',22Y,'TOP='F10.4)
0342 WRITE(5,22)BOT
0348 22 FORMAT('1',21Y,'BOT='F10.4)
0358 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0358 S=Z*TOP
0364 WRITE(5,24)S
0368 24 FORMAT('1',22Y,'S='F10.4)
0378 P=2.*D*BOT+B*(TOP+BOT)
0380 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0380 DITOP=D*(B+BOT+2.*BOT)/P
0384 WRITE(5,42)DITOP
0388 42 FORMAT('1',18Y,'DITOP='F10.4)

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STRENGTH OF SHIPSTON THE NEUTRAL AXIS LOCATION

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0338 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0338 DIBOT=R*(1+POI+POTI)/P
0350 WRITE(5,57)DIBOT
0378 B FORMAT('1',13Y,' DIBOT=IF10.4)
0394 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT Y-A.
0394 GMI=4.*B.*I*(D.*Y+2.*B*P)*TOP*POT+(D*D+2.*B*P)*POT*B.TI)/P
0404 C *TSTH* IS THE STRESS ON TOP DUE TO BOWING
0404 TSTH=H*DI(TOP/6)*I+STCRIT
0404 C *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
0404 TSTS=SM*DI(TOP/6)*I+STCRIT
0404 C *TSPH* IS THE STRESS ON BOT DUE TO BOWING
0404 TSPH=H*DI(BOT/6)*I+STCRIT
0404 C *TSSB* IS THE STRESS ON BOT DUE TO SAGGING
0404 TSSB=SM*DI(BOT/6)*I+STCRIT
0418 TSTHIT=TOP*POT+P/((.3+396*S*S*(1.-POI*POTI))
0420 TSSBIT=BOT*POT+P/((.3+396*S*S*(1.-POI*POTI))
0460 WRITE(5,66)TSTH
0488 A FORMAT('1',10Y,' TSTH=IF11.3)
04A2 WRITE(5,7)TSTS
04BE 7 FORMAT('1',10Y,' TSSB=IF12.3)
04DE WRITE(5,3)STCRIT
04F4 8 FORMAT('1',17Y,' TSTHIT=IF11.3)
0510 TSPH=H*DI(BOT/6)*I+1.162222*HBOT*S*S/(BOT*BCT)
0540 TSSB=SM*DI(BOT/6)*I+1.162222*HBOT*S*S/(BOT*BCT)
0588 SPCRIT=TOP*POT+P/((.3+396*S*S*(1.-POI*POTI))
05C8 WRITE(5,66)TSPH
05E4 9 FORMAT('1',10Y,' TSSB=IF13.3)
05FE WRITE(5,10)TSPH
061A 10 FORMAT('1',10Y,' TSSB=IF10.3)
0634 WRITE(5,11)SPCRIT
0650 11 FORMAT('1',17Y,' SPCRIT=IF10.3)
0660 END

```

0 [S]	•U	267. [V]	AL	578 [V]	A	267. [V]	DRA	267. [V]	D
90 [V]	POI	2698 [V]	E	2698 [V]	R	2698 [V]	UST	2698 [V]	POI
89 [V]	HY	2601 [V]	SM	2601 [V]	I	2601 [V]	U	2601 [V]	SPCRIT
04 [V]	Z	2602 [V]	HICH	2602 [V]	HBCT	2602 [V]	STCRIT	2602 [V]	ZI
F4 [V]	STR3	26F8 [V]	A	26F8 [V]	AB	2711 [V]	AC	2714 [V]	AD
21 [V]	AE	273. [V]	POT	2734 [V]	TOP	2228 [L]	12	2040 [S]	61
5E [L]	27	2738 [V]	S	2700 [L]	30	2720 [V]	P	2741 [V]	DTTOP
10 [L]	40	2744 [V]	DIBOT	2720 [L]	5F	2748 [V]	GMI	2754 [V]	TSTH
58 [V]	TSTS	2750 [V]	SICRIT	2488 [L]	60	241E [L]	70	24F4 [L]	81
57 [V]	TSPH	2764 [V]	TSSB	2768 [V]	SPCRIT	2514 [L]	90	2614 [L]	101
51 [L]	113	2800 [S]	•V						

•XEQ L

PROGRAM LABELS:

2070 *MATN*	28F2 •V	2022 •COMP	2404 ALOG	2F00 OT	27DC •A
2DEA •ZERO	2D48 \$E	2810 •A	2446 •YES	2CC4 •V	2ACC EX
2C52 AINT	28CE SPCRT	2581 •PARG	2DE6 •5	2ACC AEXP	2DEE •V
2DBC \$R	2DEE •0	2804 •U	4010		

TRY-POINTS:

27DC •R	2810 •A	28CE SPCRT	2404 ALOG	2ACC AEXP	2ACC EX
---------	---------	------------	-----------	-----------	---------

TOP= 3.2166
 ROT= 4.4277
 S= 203.6253
 DITOP= 32.9611
 DIRO1= 32.4674
 TSTO= 25016.111
 TSTS= 19421.123
 STCHIT= 29990.850
 TSB= 34131.155
 TSS= 29439.152
 SHCHIT= 51270.154

// END

UCF VIO OPERATING SYSTEM VERSION 1 REVISION C12 C3/14/74 GENERATED 5/ 2/74

JOB HANDLING CHARGE	0	.05	/	JOB	.05
131 LINES PRINTED PER	0	1.25	/	K L	.16
89 CARDS READ	0	1.50	/	K C	.13
00 PLOTTER VECTORS	0	.05	/	1 A	.00
20 MODEL 7 SECONDS	505.00	/	10.0	.10	
00 MODEL 3 SECONDS	112.00	/	10.0	.00	
				TOTAL CHARGE	.75

ERREIR 490 14/31 LOGGED OUT 23/ 9/75 2:5 . 15.66 LEFT AFTER 15 LOGINS

Now we see that with an assumed value $Z_1 = 35$ we obtain

$$S = 215.1808$$

$$BOT = 3.686 \quad \text{so } Z_1 \text{ actual} = 58.38$$

$$TSBH = 47512.793 \text{ (too high)}$$

$$TSBS \quad \text{also too high}$$

The Z_1 actual value obtained suggests that we should have assumed an higher Z_1 value and with an assumed value $Z_1 = 40$ we then obtain

$$S = 203.626$$

$$BOT = 4.428 \quad \text{so } Z_1 \text{ actual} = 45.98$$

$$TSBH = 34131.855 \text{ (too high)}$$

$$TSBS \quad \text{also too high}$$

This suggests an assumed Z_1 value higher than 40 but now we may interpolate with the two results already obtained and determine the Z_1 value that will bring $TSBH = DSC = 30000$ psi and done properly this new Z_1 assumed will end up being Z_1 actual.

From interpolation we obtain

$$Z_1 = 40 + \frac{(40-35) \cdot (34131.855 - 30000)}{(47512.793 - 34131.855)} = 41.544$$

and now assume this value obtaining the following result:

STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

```

0004 C *AL* IS THE LENGTH
0004 AL=1.0
0000 B=AL/5.75
0018 C *DFA* IS THE DRAFT
0018 DFA=B/3.0
0024 D=AL/14.0
0030 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0030 FOR STEEL
0030 POI=0.3
0030 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0030 FOR STEEL
0030 E=30.0*10**6
0040 C *DST* IS THE DESIGN STRESS IN TENSION
0040 DST=25000.0
0054 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
0054 DSC=24000.0
0050 C *I* IS THE BENDING INERTIAL MOMENT
0050 I=1.87145*E*DFA*AL**3
0070 C *SI* IS THE SECTION INERTIAL MOMENT
0070 SI=E*DH**3*AL
0080 I=0.30316*(1.0-POI*POI)*DSC*E
00A0 I=SI-I
00A4 Z=1.70
00C0 *HTOP* IS THE HEAD OF WATER ON DECK
00C0 HTOP=2.0
00C0 *HBOT* IS THE HEAD OF WATER ON BOTTOM
00C0 HBOT=0.0
00C0 STRG1=1.15222e+10P*Z+Z
00E0 Z1=4.0+1.0**4131.055/1.0e8*.938
00E0 STRG2=1.15222e+10P*Z1*Z1
00A0 A=DST*STRG1
0010 AB=DSC*STRG2
0020 AC=(AB*(E+D)-A*D)/(E*D)
0050 AD=D*(E+D)/(E+4.0*E*(5.0D+2.0*B*D))
00A0 AE=(3.0*E*D+2.0*B*D)/(E*D+2.0*B*D)
00F0 C *TOP* IS THE THICKNESS OF THE TOP PLATING
00F0 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
00F0 BOT=AD/(1.0+AC*AE)
0200 TOP=AC*AD
0210 WRITE(5,10)TOP
0230 10 FORMAT(11,22Y,'TOP='F10.4)
0240 WRITE(5,20)BOT
0260 20 FORMAT(11,22Y,'BOT='F10.4)
0270 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0280 S=Z*TOP
0290 WRITE(5,30)S
02AC 30 FORMAT(11,22Y,'S='F10.4)
02C0 P=2.0*D*BOT+P*(TOP+BOT)
02E0 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
02E0 DITOP=D*(B*BOT+1.0*BOT)/P
02F0 WRITE(5,40)DITOP
0320 40 FORMAT(11,18Y,'DITOP='F10.4)

```


STRENGTH OF SHIPS ON THE NEUTRAL AXIS LOCATION

```

0344 C .DIROT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS
0344 DIROT=(B*TOP+H*BOT)/2
0368 WRITE(5,57)DIROT
0384 50 FORMAT(' ',19V,'DIROT='F10.4)
0380 C *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT *A*.
0380 GMI=4.*H*I*(C.*H*E+2.*H*D)*TOP*BOT+(D*D+2.*B*E)*BOT*(D+T)/2
0410 C *ISTH* IS THE STRESS ON TOP DUE TO BOWING
0410 *ISTB* IS THE STRESS ON TOP DUE TO SAGGING
0410 *TSRH* IS THE STRESS ON BOT DUE TO BOWING
0410 *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
0410 *STCRIT* IS THE CRITICAL STRESS ON TOP
0410 *SBCRIT* IS THE CRITICAL STRESS ON BOT
0410 ISTH=H*(D+TOP)/GMI+STHBT
0424 ISTB=S*(D+TOP)/GMI+STHBT
0438 SBCRIT=TOP*TOP*(C.*H*E+2.*H*D)*S*(1.-POI*POI))
0478 WRITE(5,62)ISTH
0494 60 FORMAT(' ',19V,'ISTH='F10.3)
0494 WRITE(5,72)ISTB
0494 70 FORMAT(' ',19V,'ISTB='F10.3)
0494 WRITE(5,82)SBCRIT
0494 80 FORMAT(' ',17V,'SBCRIT='F10.3)
0494 TSRH=H*(D+TOP)/GMI+0.162222*H*BOT*S*(D+BOT+BOT)
0494 TSBS=S*(D+TOP)/GMI+0.162222*H*BOT*S*(D+BOT)
0494 SBCRIT=BOT*BOT*(C.*H*E+2.*H*D)*S*(1.-POI*POI))
0494 WRITE(5,90)TSRH
0494 90 FORMAT(' ',19V,'TSRH='F10.3)
0494 WRITE(5,100)TSBS
0494 100 FORMAT(' ',19V,'TSBS='F10.3)
0494 WRITE(5,110)SBCRIT
0494 110 FORMAT(' ',17V,'SBCRIT='F10.3)
0494 END
0494 *U 0670[V] AL 0684[V] R 0690[V] DRA 0694[V] IN
0494 *POI 0644[V] E 0650[V] .R 0654[V] DST 0658[V] D3C
0494 *HM 0600[V] SM 0604[V] W 0608[V] U 0612[V] SPT
0494 *Z 0624[V] HTOP 0628[V] HBOT 0632[V] STE3T 0636[V] Z1
0494 *ST93H 0710[V] A 0714[V] AB 0718[V] AC 0722[V] AD
0494 *AF 0744[V] BOT 0748[V] TOP 0752[V] L0 0756[V] AI
0494 *20 0750[V] S 0754[V] 20 0758[V] P 0762[V] DITOP
0494 *40 0750[V] DIROT 0754[V] 50 0758[V] GMI 0762[V] TSTH
0494 *TSTS 0774[V] STCRIT 0778[V] 60 0782[V] 70 0786[V] 80
0494 *TSBH 0770[V] TSBS 0774[V] SBCRIT 0778[V] 90 0782[V] 100
0494 *110 0780[V] .V
0494 *FC L
0494 *F

```

PROGRAM LABELS:

2070 *MAIN*	2084 .V	2098 *COMP	2112 ALOG	2126 @I	2140 .9
2082 *ZERO	2096 .R	2110 .A	2124 .MES	2138 .W	2152 E7
2064 AINT	2078 .SPT	2092 .PANG	2106 .5	2120 AEXP	2134 .5
2084 3R	2098 .L	2112 .H	4734		

TRY-POINTS:

TOTE 3.355
 BOTE 4.744
 SE 198.620
 DITOTE 40.0318
 DIBOTE 31.3947
 TSTHE 24999.008
 TSTS 19421.516
 STCHIE 29996.002
 TSH 3.191.608
 TSHGE 25817.4-4
 SBCHIE 61956.123

END

CP MIO OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 5/22/74

CR HANDLING CHARGE	1	•35	/	JOB	•35
131 LINES PRINTED PER	1	•05	/	K L	•16
89 CARDS READ	1	•02	/	K D	•10
CR PLOTTER MESSAGES	1	•05	/	1000	•00
22 MODEL 70 SECONDS	525	•00	/	4000	•15
17 MODEL 80 SECONDS	512	•00	/	4000	•00
TOTAL CHARGE					•72

AIR 490 14731 LO PER CUT 09/10/75 MR:01. 14.87 LEFT AFTER 16 LIGINS.

We see that TSBH is just a little too high and the same happens with TSBS; here it might be the case that TSBS should have been the one under consideration for the choice of a Z1 value. Nevertheless for the purpose of this work this example shows the procedure to be used along with the equation derived. The accuracy obtained can be total. Here we assumed $Z1 = 41.544$ and ended up with $Z1 \text{ actual} = 41.83$ which is close enough for our purpose and TSBH and TSBS is very close to what it should be to use the material effectively.

STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

PAGE

```

0724 C *AL* IS THE LENGTH
0724 AL=1022.
0720 B=AL/5.75
0718 C *DRA* IS THE DRAFT
0718 DRA=8/3.3
0724 D=AL/14.
0730 C *POI* IS THE POISSON'S RATIO OF THE MATERIAL
0730 FOR ALUMINUM
0730 POI=0.33
0735 C *E* IS THE YOUNG'S MODULUS OF THE MATERIAL
0735 FOR ALUMINUM
0735 E=10**7.1**6
0740 C *DST* IS THE DESIGN STRESS IN TENSION
0740 DST=25.72.
0754 C *DSC* IS THE DESIGN STRESS IN COMPRESSION
0754 DSC=31.222.
0750 C *HM* IS THE HOGGING BENDING MOMENT
0750 HM=1.37143*B*DRA*AL*AL
0774 C *SM* IS THE SAGGING BENDING MOMENT
0774 SM=B*DRA*AL*AL
0788 W=0.34296*(1.-POI+POI)*DSC/E
0788 U=SQRT(W)
0784 Z=1./U
0700 *HTOP* IS THE HEAD OF WATER ON DECK
0700 HTOP=8.
0700 *HROT* IS THE HEAD OF WATER ON BOTTOM
0700 HROT=LRA
0700 STR3T=7.15222*HTOP*Z*Z
0700 STR3B=7.15222*HROT*Z*Z
0700 A=DST-STR3T
0700 AB=DSC-STR3B
0700 AC=(AD*(B+D)-A*D)/(A*B)
0700 AD=HM*(B+D)/(A*4.*D*(D*D+2.*B*D))
0700 AE=(3.*B*B+2.*B*D)/(D*D+2.*B*D)
0700 C *TOP* IS THE THICKNESS OF THE TOP PLATING
0700 C *BOT* IS THE THICKNESS OF THE BOTTOM PLATING
0700 BOT=AD/(1.+AC*AE)
0700 TOP=AC*BOT
0704 WRITE(5,10)TOP
0720 10 FORMAT('1',20X,'TOP='F10.4)
0730 WRITE(5,20)BOT
0756 20 FORMAT('1',20X,'BOT='F10.4)
0700 C *S* IS THE SPACING OF LONGITUDINAL STIFFENING MEMBERS
0700 S=Z*TOP
0700 WRITE(5,30)S
0700 30 FORMAT('1',22X,'S='F10.4)
0700 P=2.*D*BOT+B*(TOP+BOT)
0704 C *DITOP* IS THE DISTANCE OF THE TOP FROM THE NEUTRAL AXIS
0704 DITOP=D*(B*BOT+D*HROT)/P
0708 WRITE(5,40)DITOP
0714 40 FORMAT('1',18X,'DITOP='F10.4)
0730 C *DIBOT* IS THE DISTANCE OF THE BOTTOM FROM THE NEUTRAL AXIS

```


STRENGTH OF SHIPS-ON THE NEUTRAL AXIS LOCATION

```

0330      DIBOT=0*(B*TOP+D*BOT)/P
0354      WRITE(5,52)DIBOT
0370      50 FORMAT(' ',10X,'DIBOT='F10.4)
0380      C      *GMI* IS THE MOMENT OF INERTIA OF THE X SECTION ABOUT N.A.
0380      GMI=4.*D*D*((3.*R*R+2.*R*D)*TOP*BOT+(D*D+2.*B*D)*BOT*BOT)/P
03FC      C      *TSTH* IS THE STRESS ON TOP DUE TO HOGGING
03FC      C      *TSTS* IS THE STRESS ON TOP DUE TO SAGGING
03FC      C      *TSBH* IS THE STRESS ON BOT DUE TO HOGGING
03FC      C      *TSBS* IS THE STRESS ON BOT DUE TO SAGGING
03FC      C      *STCRIT* IS THE CRITICAL STRESS ON TOP
03FC      C      *SBCRIT* IS THE CRITICAL STRESS ON BOT
03FC      TSTH=HM*DITOP/GMI+3*RG3T
0412      TSTS=SM*DITOP/GMI+3*RG3T
0424      STCRIT=TOP*TOP*E/(2.30396*S*S*(1.-POI*POI))
0464      WRITE(5,40)TSTH
0480      60 FORMAT(' ',10X,'TSTH='F10.3)
049A      WRITE(5,70)TSTS
04B6      70 FORMAT(' ',10X,'TSTS='F10.3)
04D0      WRITE(5,80)STCRIT
04EC      80 FORMAT(' ',17X,'STCRIT='F10.3)
0508      TSBH=HM*DI-BOT/GMI+2.152222*HBOT*S*S/(BOT*BOT)
0544      TSBS=SM*DI-BOT/GMI+2.152222*HBOT*S*S/(BOT*BOT)
0580      SBCRIT=BOT*BOT*E/(2.30396*S*S*(1.-POI*POI))
0600      WRITE(5,90)TSBH
06DC      90 FORMAT(' ',10X,'TSBH='F10.3)
06F6      WRITE(5,100)TSBS
0712      100 FORMAT(' ',10X,'TSBS='F10.3)
072C      WRITE(5,110)SBCRIT
0748      110 FORMAT(' ',17X,'SBCRIT='F10.3)
0764      END
0[S]      .U      0668[V] AL      0670[V] B      0678[V] DRA      0680[V] D
8[V]      POI      0690[V] F      0000[S] .R      0690[V] DST      06A4[V] DSC
C[V]      HM      06A4[V] SM      06B8[V] W      06C4[V] U      0000[S] SQRT
8[V]      Z      06CC[V] HTOP      06D4[V] HBOT      06D8[V] STR3T      06E4[V] STR3B
4[V]      A      06E8[V] AB      06EC[V] AC      0700[V] AD      0714[V] AE
C[V]      BOT      0720[V] TOP      0720[V] IZ      0000[S] @I      0256[L] 20
4[V]      S      0298[L] 30      0728[V] P      0720[V] DITOP      0314[L] 40
0[V]      DIBOT      0370[L] 50      0734[V] GMI      0740[V] TSTH      0744[V] TSTS
8[V]      STCRIT      0424[L] 60      04B4[L] 70      04EC[L] 80      0740[V] TSBH
8[V]      TSBS      0754[V] SBCRIT      05DC[L] 90      0612[L] 100      0648[L] 110
7[S]      .V
XEQ      L

```

PROGRAM LABELS:

070 *MAIN*	2FDE .V	2DME .COMP	29B0 ALOG	2EEC @I	27C8 .R
2DD6 .ZERO	2D34 \$6	2808 .A	2E32 .MES	2CB0 .W	2AB8 EXP
2C3E AINT	28BA SQRT	2D76 .FARG	2DD2 .5	2AB8 AEXP	2DDA .ER
2DAB \$8	2DDA .0	2EB0 .U	4028		

TRY-POINTS:

27C8 .P	2808 .A	28BA SQRT	29B0 ALOG	2AB8 AEXP	2AB8 EXP
---------	---------	-----------	-----------	-----------	----------

TOP= 2.9992
 BOT= 3.7608
 S= 105.2122
 DITOP= 38.4761
 DIBOT= 32.9524
 TSTH= 24999.988
 TSTS= 18635.431
 STCRIT= 3.440.444
 TSPH= 24476.447
 TSPS= 2.954.855
 SRCRIT= 47172.344

END

CP VIC OPERATING SYSTEM VERSION 1 REVISION 012 03/04/74 GENERATED 05/02/74

JOB HANDLING CHARGE	\$.35 / JOB	.35
132 LINES PRINTED PK2	\$ 1.25 / K LN	.16
88 CARDS READ	\$ 1.50 / K CD	.13
40 PLOTTER VECTORS	\$.25 / 1 72	.04
15 OCEL 72 SECONDS	\$25.00 / HOUR	.12
40 OCEL 87 SECONDS	\$12.50 / HOUR	.04
	TOTAL CHARGE \$.74

RRRIR 492 14731 LOGGED OUT 09/09/75 02:45. \$ 17.18 LEFT AFTER 13 LOGINS.

This last example shows that TSBH is low and the Z approximation used in the formulas here again should be corrected as already exemplified in order to bring it closer to the DSC value.

CONCLUSION

From this work the author is led to conclude that there is no simplified approach to the neutral axis location in the midship section, capable of handling properly all or only a majority of cases. It is advisable to look into each particular situation by first gaining some insight at what the diagrams of stress distributions may look like as exemplified in several sections. The two last sections exemplify well why this may be of some advantage and specially the last section which shows that while at first it was expected for the examples studied a neutral axis closer to the top, actually the results ended up requiring a neutral axis closer to the bottom and a new set of formulas had to be derived.

As a future development for this work it is here suggested that a more ship like cross section could be analysed as a step forward towards obtaining a method of determination of scantlings as direct as possible. This next step should consider the derivation of proper formulas for DITOP, DIBOT and I involving only the plate thicknesses as unknowns. It should be considered then, in order to simplify the problem., a standard type of frame and its scantlings should be constant throughout the midship section.

LIST OF REFERENCES

- J.H. EVANS, "A Structural Analysis and Design Integration"
Trans. S.N.A.M.E. 1958
- J.H. EVANS, & DAN KHOUSHY, "Optimized Design of Midship
Section Structure" S.N.A.M.E. 1963
- J.H. EVANS, "Optimizing Hull Structural Design With High
Strength Steels" S.N.A.M.E. 1967
- J.H. EVANS, "Structural Performance Norms in Ship Design"
S.N.A.M.E. 1970
- C.E. ROTH III, & D. LIU, "Computer Applications to the
Structural Design of a Combination Framed
Midship Section", M.I.T. Department of N.A.M.E.
Report # 66-1
- R.D. ROCKWELL, & O.P.J. JONS, "Optimization of Primary Hull
Structure with Mixed Framed Systems"
M.I.T. Department of N.A.M.E. Report # 67-4
- J.F. INCE, "Effects of Material Variations in the Computeri-
zed Design of Primary Hull Structure"
M.I.T. Department of N.A.M.E. Report # 68-3
- L. AFANASIEFF, "A Study of the Interface Locations Between
Dissimilar Materials and Other Aspects of a Com-
posite Midship Section", M.S. Thesis, M.I.T.
Department of N.A.M.E. August 1967
- J.H. EVANS, "A Weight Cost Analysis of High Strength Steels
in the Primary Structure of Ships",
M.I.T. Department of N.A.M.E. Report # 67-3

- COMSTOCK, "Principles of Naval Architecture" S.N.A.M.E. 1967



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D16175

Da Silva

Strength of ships -
on the neutral axis
location.

153771

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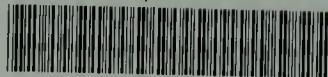
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