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## WORKS OF PROF. C. E. GREENE

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」 MANLAL FOR IIESIGJERS, AND A TEIT-BUOK FOR SCIENTIFIC SCHOOLS

## TRUSSES AND ARCHES

## ANALYZED AND DISCUSSED BY GRAPHICAL METHODS.

BY
CHARLES E. GREENE, A.M.. PROFESSOK OF CIVIL ENGINEERING, CNIVERSITY OF MICHIGAN.

> IN THREE PARTS.
I.

ROOF-TRESSES: Diagrans for Steady Loal, Snow, and Wind.
II.

BRIDGE-TRUSSES: Single, Continlous, And Draw Spans; Single And Multiple Systems; Straight and Inclined Chords.

## III.

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## PREFACE TO PART III.

The curved lines of arches are pleasing to the eye, and may often be introduced with adrantage in constructions. An arch may furnish, under some circumstances, a very economical way of spaming an opening; and arched ribs are employed in other cases, at conspicuous locations, where beauty of design is regarded, or where ample and uninterrupted space beneath a roof is desired. Stone arches have been built for many centuries: at the present time, wood, iron, and steel are also used as materials. If the principles which enable one to ascertain the forces acting in all parts of an arched structure are clearly understood, designs of this type will be nore common than they now are; and it is with the desire to do what he can toward shedding some light upon this subject, as well as to give the ability to intelligently design an arch to those who are not familiar with the ligher mathematics, that the author submits the following pages to the public.

Most persons experience difficulty in mastering the principles which govern the action of an arch as they have hitherto been presented. Even one who hat successfully worked through the mathematical theory, ats he finds it in the text-books, maty sometimes lose sight of the actual meaning of each step in the
process; so that there is a certain mystery about the application of the formulæ to a specific example, although one may feel confident that the results are reliable. To many constructors a treatise on the arch, as usually written, is a sealed book, and the whole subject is veiled in obscurity. Empirical rules, copying of existing examples, and guesswork have been the refuge of many. While such practice may answer for masonry structures, where the factor of safety as regards strength is very large, the introduction of iron skeleton structures, where the pieces occupy definite lines of force, and the sharp rivalry for economical disposition of the material, render a better practice desirable. It is hoped that the graphical method developed in the following pages will enable the reader to understand as clearly the effect of applied forces on an arch, as it has, through the explanations of Parts I. and II., enabled him to analyze trussed roofs and bridges.

From the bending moment, direct thrust, and shear, here obtained at successive sections of the arched rib, the stresses in the chords or flanges, and bracing or web, are derived as if the structure were a simple truss. In finding the resultant stresses in the pieces, the method of Part I. will sometimes be preferred to that of Part II. So far as possible, the formulæ of the text have been obtained by direct and easy ways ; and, while it has been convenient to arrive at some of the definite results by the use of the calculus, such results have been obtained from the diagrams, and can in all cases be verified by the reader, for any specific example, by the most simple means.

After the subject is once mastered, the resulting formulie and applications will, naturally, alone be referred to in working out designs: the author has therefore thought it best to place the results, \&c., in direct connection with the explanatory
statements, and to have the analytical or mathematical demonstrations follow in smaller type. One who simply desires working-material may omit the matter printed in small type, without losing any of the facts, but must then take some statements for granted.

A distinctive notation for the figures, introduced in Parts I. and II., - capitals for structures and moment diagrams, small letters for the shear diagrams, and numerals for the stress diagrams, - has been generally adhered to. While an acquaintance with Parts I. and II. will aid the reader in understanding more readily the graphical constructions here given, it has been the aim of the author to enter sufficiently into detail to make this part intelligible by itself: hence a few explanations are repeated here.

It is believed that many things offered in these pages will be new to most readers. The work is almost entirely the result of independent investigation. A portion of the material was once printed in the "Engineering News," but it has been entirely revised since that time: over one-half of this part is now in type for the first time. The device of increasing the breadth of the parabolic rib, or the thickness of the flanges, from the crown to the springing, while the depth remains constant, which device will be found in Rankine's "Civil Engineering," - enables the summation of ordinates to be made across the span, as for a beam, remlering the treatment simple. On the other hand, the depth and breadth of the circular rib are supposed to be constant, and the summation is made along the curve. Herein the treatment differs from that of some authors. It is shown that the direct thrust on a right section is not equal to the product of the horizontal thrust by the secant of the inclination of the rib at the section to the horizon, as some
writers assume, unless the equilibrium curve is parallel to the axis of the rib. Other points of difference in treatment amd result will be found by readers who are familiar with the literature on this subject. The discussion, in Chapter VII., of the action of the wind on an arehed roof, will, it is hoped, be foumd timely and serviceable; the effect of change of temperatme, and the change of form under stress (Chapter XI.), are often ignored by writers ; an example of a stone arch of considerable magnitude is worked out in detail; the methods of stiffening suspension bridges are discossed and compared: on some of these points very little has heretofore been given.

C. E. G.

Ans Arbor, Mich., July, 1879.

## NOTE TO THIRD EDITION.

In the present edition the parabolic rib with a hinge at the erown only has leeen treated. The solution will be found on page 192. The method of locating the equilibrimm polygon by trial, on a rib of any ontline, has been emphasized. Illustrations of three notable arehed bridges are added-one over the Marlem river at New York, two over the Niagara river at Niagara Falls, N. Y. In designing two of these structures, if not all three, the method of analysis developed in this book was applied.

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## ARCHES.

## CHAP'TER I.

1. Arches. - An arch may be considered to he any structure which, under the action of vertical forces. exerts horizontal or inclined forces against its supports or abutments. Such a definition will include not only the roof of two simple rafters, but also the suspension bridge: and we see no objection to so including them. The case of two rafters we need not tomeh upon: the suspension bridge only comes incilentally within the scope of this part. until we take up, the means of stiffening such a structure under a moving and partial loat.
2. Funicular Polygon applied to a Curved Rib. - Suppese that a curved rib A C E B. Fig. 1, uf any material which possesses stiffues, for instance iron, is attached by a pin, on which it can turn freely, to cach of the points of support $A$ and $B$, and has suspended from it certain known weights. represented by $\mathbb{W}_{1}$. $W_{2}$, de.. at known points. The weight of the curved rib) itself is mot at present comsidered. The rib, if flexible. as a cord or chain is flexible. will temd to assume the shape of the fimicular, or equilibrium polygon. proper to these weights in their respective positions. If we lay off the load line -1 . to any seale, space off on it the weights in succession, assume any convenient point 0 , draw radiating lines from that point to the
points of division and to the extremities of the load line, and then, starting from $\Lambda$, or any other point in the vertical through that point of support, draw lines. successively parallel to the lines radiating from 0 , and limited by the verticals through the weights, one sueh equilibrium polygon will be found.

This polygon was discussed in Part II., "Bridges," § 2. By moving the point 0 of the stress diagram, the place where the equilibrium polygon strikes the vertical drawn through B will be changed ; and. if $O$ is horizontally opposite the point which divides the load line into the two supporting forces, the polygon, drawn from $A$ as a ruint of beginning, will strike $B$. But 0 may move on a horizontal line, and II will then have any value we please. H is therefore, at present, an unknown quantity: but we will suppose that A K I B is the desired equilibrium polygon for this given case, - an imaginary line, the weights being attached to the arch.
3. Relation between Equilibrium Polygon and Bending Moments. - If the rib) is made of a rigid material, the tendeney to take a shape other than the one to which it was first formed will cause a bending action or moment at different points. Thus, between A and C the rib will flatten somewhat, moving towards the straight line A C, and from C to B it will become slightly more convex. At C , where the rib eoincides with the equilibrium polygon, there will be no tendeney to bend. The bending moments on either side of a point where the equilibrium polygon crosses the rib will therefore be of eontrary kinds or signs. It is necessary to know the value of the bending moments at all points, in order to so design the crosssection of the rib that it shall be able to resist them. The point C is not necessarily the crown of the arch: it happens to come near it in our figure. If the arched ril) is free to turn at its supporting points, no bending moments can exist there ; if it is jointed or hinged at any place, as, for example, the middle or crown, no bending moment will be found there: the equilibrium polygon must therefore pass through all such points. The rib may be so fastened at A and B that it camiot turn in a
vertical plane; and there will then be lending moments at those points, as in the amalogons case of a beam fixed at both ends. except for such a distribution of the load as makes the equilibrimm polygon coincile with the arch at it., ends.

If the rib is hinged at three points, that is, at the ends and middle, the equilibrimm polygon is immediately fixed in position by the necessity of passing through these three points, and the problem of finding the stresses in the rib becomes very simple. as will be seen later.
4. Value of Bending Moment. - Let us suppose, at first, that the rib of Fig. 1 is jointed, and free to turn at its ends only. The stress diagram, $01 \xrightarrow[2]{2}$, and the imaginary equilibrium polygon, having been constructed, and the horizontal line H from 0 drawn, it will he seen that this line will divide the load line into two forces, the rertical components of the abutment reactions, as proved in Part II., $\S 6$. The arrows in the figure denote these components: and we will call the rertical ones, analogous to the supporting forces of a beam, $\mathrm{P}_{1}$ and $\mathrm{P}_{2}$, as marked. We have here the usimal closed polygon of external forces.

Let an imaginary vertical section be made at D F : from the theorem of moments, as equilibrium exists in this loaded arch, the moments of all the external forces must balance around any point, for instance the point E , where the plane of section cuts the rib. If the sum of the moments around E equals zero, the moments on one side of the plane of section must equal those on the other ; and, as E is in the section of the rib, these moments can only neutralize one another through the moment of resistance of the section : consequently, the sum of the moments on either side must equal the bending moment at E . Then at E , if $\mathrm{P}_{2}$ and H are the rectangular components of the reaction at B. and ごW. L denotes the smm of the products of each weight by its horizontal distance L from E. the bending moment will be

$$
\mathrm{M}=\mathrm{P}_{2} . \mathrm{DB}-\Sigma \mathrm{E}^{\mathrm{W}} . \mathrm{L}-\mathrm{H} . \mathrm{D} \mathrm{E} .
$$

If the weights had been attached to the cord, or equilibrium
pulygon, we should have had, for moments on the right of and about F ,

$$
\mathrm{P}_{2} . \text { I) } \mathrm{B}-\mathrm{E} \mathrm{~W} . \mathrm{L}-\mathrm{II} . \mathrm{I} \mathrm{~F}
$$

But a corl, being flexible, can resist no lending moment. As this cord is the equilibrimm polygon, there can be no tendency to move or no bending moment at any point of it, and expression (2.) must reduce to zero, or

$$
P_{2} \cdot D B-\Sigma W \cdot L=I I . D F .
$$

Substitute this value in (1.), and it becomes

$$
\mathrm{M}=\mathrm{H} \cdot \mathrm{D} \mathrm{~F}-\mathrm{H} \cdot \mathrm{DE}=\mathrm{II} \cdot \mathrm{EF} ;(3 .)
$$

which siguifies that the bending moment at any point of an arched rib, under any rertical load, is equal to the product of the vertical ordinate from that point to the proper equilibrium polyyon, multiplied by H from the stress diagram.
5. Remarks. - It will be noticed that, to the left of C', D F - D E will change sign, becoming negative, and therefore that the bending moment will change in direction, as stated before. If the rib beeomes straight and horizontal, the point E moves up to D, and the bending moment becomes equal to II . D F, which is its value for a beam supported at both ends.

The relation of the equilibrime polygon to the arch, or the fact that the bencting moment equals H. E F, as just proved, may he readily explained in another way. Suppose that the arch $\Lambda^{\prime} B^{\prime}$ of Fig . 14 has a single weight placed upon it in a certain position: it will thrust horizontally against the abutments an amount II. Let the equilibrium polygon for this weight, and having the same H, be AFB. The ordinates to this equilibrium polygon will be proportional to the bending moments due to the weight on a beam or truss of span AB; the moments will all be positive, and equal to II.D F. But the thrist II of the arch, which actually carries the weight, acting in the line $A^{\prime} B^{\prime}$, will exert negative hending moments equal to H.DE at all sections of the arel. The resultant hending moment at any point, when the equilibrimu polygon is superimposed on the arch, will be the product of HI by the
difference of these two ordinates, or II (D F - D E) = II . E F , at some places negative, and at others positive. Thus we see that, while we have for a given system of weights an equilibriun polygon exactly similar to those treated in Part II., "Bridges," the arch, by reason of its horizontal thrust which eanses negative bending moments as above, ammuls or cuts off a portion of the area of the equilibrium polygon, and the portion of the ordinate in excess or deficient at any point measures the existing bending moment. It is only necessary that the arch and polygon should have the same value of $H$. The arch, in its capacity of frame, as it were, earries a portion, more or less, of the forces which would otherwise canse bending moments and shears.

Such an arrangement of weights might be devised, contimuously distributed along the rib, that there would be no tendency to change the shape of the arch at any point. The equilibrimen polygon, becoming a curve for a continnous load, would then coincide with the centre line of the arch, and we should have what is termed an equilibrated rib. And, on the other hand, a rib can be designed for any given distribution of load, of such a shape as to be in equilibrium. This faet can sometimes he made use of when the load is definite, that is, not a moving load, and we shall refer to it again in the sequel.
6. Condition to determine H; Invariability of Span. - It may he noticed that in $\S 4$ we nsed the term proper equilibrium polygm. It has been stated that it is easy to draw, between A and B , any number of funicular polygons, which have their angles on the verticals let fall from the weights, by simply moving the point 0 horizontally in the stress diagram, and thms altering the value of II, the horizontal component of the tension. But the actual rib, under a given system of weights, must have a fixed value of II, and definite bending moments at all points: there is therefore but one funicular polygon which will be the proper equilibrimm polygon. Some condition must be imposed; and a sufficient one is, that, supposing the points $A$ and $B$ to be fixed in position relatively to one
another, the distance 1 B. or the spren of the rith, s.7ull be unchanyed. An arch between two myielding abutments satisfies this condition. If the curve $A C$ is flattened by the pull upon it, or by the bending moments by which it is mged towards the straight line $A C$, the point $C$ will move a little to the right, while the portion between $C$ and $B$ will become slightly more convex. The morement of the point $B$, however, with reference to $A$, must be zero.
7. Formula for this Condition. - Consider the arched rib as discomnected from its fixed points of support. but suspended in the air by the forces which were but now the reactions at those points. Equilibrium will still exist. The bending moment H.EF at E. fiom it. effect on the particles at that section, causing an elongation of the fibres on one side and a compression of the fibres on the other side, produces what may be called an exceedingly small angle in the rib. or, better, a chonge of inclination, at E , moving the free end B , so far as this change alone is concerned, a very small distance in a direction perpendicular to a straight line from E to B . The amount of this displacement will depend upon the distance E B, and upon the change of inclination at $E$, which change has just been shown to depend upon the bending moment II . E F. The amount. B R, of this movement, is greatly exaggerated in the figure. But the horizontal component, or projection, B S, of the displacement, which alone affects the horizontal distance of $B$ from $A$. will manifestly, from the proportionality of the sides of the two right-angled triangles $B$ R S and E B D, he to BR as D E is to E B , or BS will be proportional to D E .

Perhaps this point may be brought out more plainly if stated algebraically, thus:-

$$
\begin{gathered}
\text { BR varies as EB.H.EF; } \\
B S=B R \cdot \frac{D E}{E B} \text {; therefore, } \\
B \text { S varies as } \frac{E B \cdot H \cdot E F \cdot D E}{E B} \text {, or as H.EF.DE. }
\end{gathered}
$$

Taking all the points in the rib into consideration, we see
that the total horizontal displacement of B from A will be proportional to H. - EF.DE, if $\check{y}$ is the sign of summation of all of the products E F. D E. As the span A B is to be unchanged, the above quantity must equal zero, and therefore, as H cannot be zero, we have the desired condition reduced to
ェEF.DE=0. (1.)
8. The Equilibrium Polygon determinate. - As E F changes sign when the equilibrium polygon crosses the rib, as at C, we arrive at this result for a rib free to turn, or hinged, at its ends, that the smmation of the products E F. D E for every point where the equilibrium pulygon lies on one side of the rib must equal the summation of the similar products for every point uhere the polygon lies on the other side. Only one polygon, manifestly, will satisfy this condition: for, if we draw a new polygon between A and B , we immediately increase one set of EF 's and diminish the other. An equilibrium polygon may first be drawn tentatively, ordinates be measured at intervals, and the above prolucts computed. It will then be readily seen whether the polygon should be moved up or down; to move it, change H , and draw again. We can deal thus with a rib of any outline; but, for the regular forms of arches commonly in use, we will show presently how to determine the exact equilibrium polygon without experimental trial.
9. Deflection of the Rib. - The vertical component R S, of the displacement B R, manifests itself, since B camnot move, by a slight movement of the rib at E vertically, corresponding to the deflection of a beam under transverse forces.
10. Another Value for Bending Moment. - It has been shown that the bending moment at E equals $\mathrm{H} . \mathrm{EF}$. If we draw from E a perpendicular, E N , to that side of the equilibrium polygon which passes through F , the side being prolonged if necessary, we shall form a right-angled triangle, similar to one formed in the stress diagram by H , the line parallel to the side of the polygon, and the vertical line. Thus, in Fig. 1, the triangle EF N will be similar to 025 , and we may write the proportion

$$
0-2: 0-5=\mathrm{E} \mathrm{~F}: \mathrm{E} \mathrm{~N} ;
$$

or, if T denotes the tension $0-2$ in the part of the cord which passes though F , we get, upon multiplying extremes and means,

$$
\text { II . E F }=\text { T. } \mathrm{E} \mathrm{~N} \text {; (1.) }
$$

so that the bending moment at each point is also equal to the product of the tension in the cord by the perpendieular let fall on the cord from the given point; and this is the measure of a moment, as shown in mechanics. The discussion of the bending moment might have been approached in this way.
11. Combined Effect of Bending Moment and Direct Force.-If a force T acts in the line A K, which, when we consider the curved rib, is an imaginary line, its moment with respect to the rib at E is, then, T.E N. Now, from mechanics, if we analyze the effect of a force T, Fig. 2, at any distance laterally from a point E, we may apply two equal and opposite forces. +T and -T , at this point, which is here the middle of the rib, or what would be, for flexure only, the neutral axis, without destroying the equilibrium. Hence we have at E the direct force +T , producing tension, and the couple T.EN, producing flexure. The enlarged sketches will represent the condition of the rib. 'The small arrows at $\mathrm{E}^{\prime}$ denote the magnitude or intensities of the stresses which form the moment of resistance to balance the luending moment, these intensities heing taken as uniformly varying, a supposition which is satisfied within the elastic limit: at $\mathrm{E}^{\prime \prime}$ are shown the stresses on the particles; of the section from the direct force; and the comlination of the moment and force is represented at $\mathrm{E}^{\prime \prime \prime}$, it leing understood that these several views represent one and the same section E.

The point of no stress, or the position of the nentral axis, is seen to be shifted from the middle of the section at $\mathrm{E}^{\prime}$ to one side at $\mathrm{E}^{\prime \prime \prime}$ : and it will disappear altogether when the arm of the couple or moment becomes sufficiently small, so that the entire section may be under one kind of stress of varying intensity. If we know the form of cross-section of the rilh, we
can tell from the location of the equilibrium polygon, by simple inspection, where we shall find both tension and compression, and where only one kind of stress. This matter will be tonched upon later: §§ 106-108.
12. Reversal of Figure; Movement of Rib from Equilibrium Polygon. - When an arch is under andysis, the figures thus far given will be inverted. Imagine them to be so. All of the forces will then he reversed. The polygon which was under tension will be compressed, and its sides will represent struts. It will be in unstable equilibrium, and its relation to vertical forces is not, perhaps, so readily apprehended, by one not acquainted with this subject, as is that of the funicular polygon. For this reason it was thonght best to take in inverted arch first. Hereafter the arches will be drawn above the springing line; If becomes the horizontal thrust of the rib against its abutments.

The eursed rib, between the points A and C, Fig. 1, so long as there is tension along the straight line A C, tends to move towards that line, just as the cord, if drawn towards the arch, returns to its former position; but as soon as the figure is inverted, and C is forced by compression towards A , the arch tends to move avay from the equilibrium polyyon. This fact is true of all points of the rib, and, being borne in mind, will enable one to tell at a glance the kind of moment at each point of the rib. All the bending moments are therefore reversed. Those bending moments which tend to make the arch flatter, or of less curvature, at any point, are called positive; those which tend to make it more convex are called negative.

It may aid in fixing the ideas, to take a piece of small steel wire, bend it into the are of a circle, and, placing the two ends in two notches upon a board, notice the change of shape arising from a pressure or load imposed on any portion. The movement of the wire will indicate, in a general way, where the equilibrimm eurve lies in reference to the rib.
13. Equilibrium Polygon for a Single Load. - It is now readily seen that the equilibrimm polygon for a single, concen-
trated load on an arch is composed of two straight lines which meet on the vertical drawn through the point where the load is imposed. In the case just treated, these lines will start from the two springing points of the arch. The only quantity needful to fix their position will be the distance of their point of intersection vertically from the rib; and the single condition of (1.) $\S 7$, that $\Sigma \mathrm{EF} . \mathrm{DE}=0$, will determine the unknown quantity. It will be easier to find the effect of a single load at successive points on the arch, and to combine these effects for any possible arrangements and intensities of load, than to treat at once several loads. We shall pursue this method.
14. Direct Force and Shear at a Right Section. - Since an arched rib is often composed of two flanges, and a web or connecting bracing, similar to a girder or truss, we desire, after we have found the bending moments at all points, to find that portion of the vertical force or the shear at each section which must be resisted by the web members. Shear was explained in Part II., "Bridges," § 4. In a horizontal beam, carried on two supports, we should have, in Fig. 1, $\mathrm{P}_{2}$ for the supporting force, and shear on the right of any section between $B$ and $W_{1}$; $P_{2}-W_{1}$, or $(1-5)-(3-1)$, for the shear anywhere between $W_{1}$ and $W_{2} ; P_{2}-W_{1}-W_{2}$, or (3-5) - (4-3), that is $-(5-4)$, between $W_{2}$ and $W_{3}$; and so on, subtracting each weight from the previous shear or resultant. But in a beam, or a truss with horizontal chords, the other forces, those which oppose the bending moment, are horizontal: here they are not. Supposing the rib to be inverted, the direct thrust, being in the direction of a tangent at the centre line of the rib, has a vertical component which affects the amount of shear to be resisted by the web. In short, the inclined flanges or chords act as braces; and we have, at any section, these chords as well as the web members, among which to distribute the shearing force. The action corresponds with that of the bow in a bowstring girder.

It is not probable that the thrust in the side of the equilibrium polygon will be parallel to the tangent to the curve of the centre line of the rib at a particular section, but this thrust
will be the resultant force at the section. It may then properly be resolved into two rectangular components, one perpendicular to the section, representing the direct force, and the other parallel to the plane of the section, representing the shear. The direct stress, combined with the tension and compression due to bending moment, will be resisted by the flanges or chords, and the shear by the web members, if the rib is so comstructed. If the rib is of solid seetion, like a beam, the separate consideration of shear is generally unnecessary. It will at once be seen that the direct stress at any point of the ril) is obtained by projecting the force in that side of the equilibrium polygon which passes near the point upon the tangent to the rilh. Thus, in Fig. 1, $0-3$ is the tensile force in the side I G of the equilibrium polygon, and 0-6 is drawn parallel to the tangent at U : if a perpendicular were drawn from 3 upon 0-6 prolonged, the distance from 0 to the foot of the perpendicular would be the direct stress, and the perpendicular itself wonld be the shear on a right section at C . Or, again, if $0-2$ is the force in A K , and $0-7$ is parallel to the tangent at Q, a perpendicular from 2 on $0-7$ will cut off the direct stress. and be itself the shear at Q .
15. Sign of Shear; Maximum Bending Moment at Point of Zero Shear. - The ahove points may be made more clear, if necessary, by reference to the sketch above and on the left of Fig. 8. Let A C represent a portion of an areh, and A R' a portion of the equilibrimu polygon which exerts a thrust R at A . The components of the abutment reaction will be H , the horizontal thrust, and $\mathrm{P}_{1}$, the vertical force. But R may also be decomposed, on a right section of the rilh nerir $A$. into $T$ direct thrust and $F$ shear at the section. The little sketch adjoining shows, that, as these components act on the left of the seetion, we must have the opposite shear on the right of the seetion. giving what we have been accustomed to call negative shear (see Part II., "Bridges"). When, at any right section, a line parallel to the side of the equilibrimn polygon lies above the tangent to the rib, the forces being taken on the
left of the section, as is the ease at $C$, where $T^{\prime}$ and $F^{\prime}$ are the components of $\mathrm{R}^{\prime}$, the shear will be positive. Where the side of the equilibrium polygon is parallel to the tangent to the rib) as for instance near $d$, at that point there will be no shear, and the shear will be of opposite signs on each side of such point. The direet stress there will be H multiplied by the secant of the inclination of the tangent to the horizon.

As the maximum ordinate between the side of the equilibrium polygon and the arch occurs where the side of the polygon is parallel to the rib, the maximum bending moments in the arch, as in a beam or truss, are fornd at points of no shear.
16. Treatment of Arch with Fixed Ends requires Three Conditions. - If the arehed rib is fixed in direction at the emis (in place of being free to turn as preciously smposerl), by being firmly bolted to the abutments, or by having sillare ends aceurately bedded upon the skewbacks, a bending moment will generally exist at the points of support when the arch is loaded. By taking the piece of easily flexible wire before mentioned, clamping the ends firmly, so as to fix the wire in the position of an arch. and then applying a load or the pressure of the finger, one can easily rerify this statement for himself; and he will see that, for many positions of the load, the bending moment at one abutment is of the olposite kind to that at the other. The points at which the equilibrimm polygon begins and ends will no longer be A and B of Fig. 1, and some new conditions must be imposed in order to determine these points.

Consider the effect of a single load upon the arched ril) 1 C $B$ of Fig. 3, which rib is fixed in direction at its ends. The equilibrimm polygon will be two straight lines, such as I N and N L: and, as there may be bending at both points of smport. it will be necessary to find the magnitudes of A I and B L, as well as of N ( a , three unknown quantities. Three conditions mnst therefore be satisfied. Such writers as, in treating the arch either graphically or mathematically, require but two conditions to be fulfilled for an arch with fixed ends, err in their
assumptions, and hence in their results. If two conditions only are imposed, where three are necessary, many polygons can be drawn, and the prohlem is left undetermined.
17. First Condition. - One condition which must be satisfied is plainly the one already used, $s s .6$ and $T$, that the change of span A B shall equal zero, or that

$$
\text { צ EF. I) } \mathrm{E}=0 .
$$

18. Second Condition: Change of Inclination between Abutments equals Zero. - As the chumge of inclination between any two contiguons points is directly proportional, in direction and magnitude, to the bending moment (for the clongation and compression of the fibres on the two sides, upper and lower, of the rib, result from this bending moment, and canse whatever change of direction or inclination the ril) may take on), and as the bending moment has been proved to be proportional simply to the ordinate E F , the change of inclination at any point is proportional to the ordinate E F from that point of the rib to the equilibrimm polygon.

The reader must distinguish between the change of inclination produced by flexure, and the original inclination of the rib, to the horizon at each point due to the curve to which the rib) is constructed. If an arch is loaded, it assumes a form slightly different from its slape when moaded. The angle, at any partienlar point, between the two tangents to the curve of the rib, before and after it is loaded, is the shange of inclinution at that point.

Starting fiom A. Fig. 3, the total change of inclination at ( will be proportional to the sum of all the ordinates between $A$ and $C$. On the other side of $C$, where the straight line crosses the rib, the bending moment heing of the opposite kind, the changes of inclination will be in the opposite direction. and. in any summation of ordinates, for instance from $A$ to E , must bo subtracted. Then, as both $A$ and $B$ are fixed in their original directions, if we sum up all of the ordinates $E \mathrm{~F}$, from $A$ to $B$, the total change of inclination between abutments is zero, and
this sum must be zero. Therefore the second condition to be realized is that

$$
\Sigma \mathrm{E} F=0
$$

or that the sum of all the ordinates between the arch and the equilibrium polyyon on the inside of the arch must equal the similar sum outside.
19. Third Condition: Deflection between Abutments equals Zero. - Fig. 3 shows that, since the displacement B R of $B$, relatively to the point E , in case B could move, has been proved, by $\$ 7$, to be proportional to H. E F . E B , the vertical component of this displacement varies as H. E F . D B ; for, by a similar proportion to the one used in that section,

$$
\begin{gathered}
S R=B R \frac{D B}{E B} \text {; therefore, } \\
S R \text { varies as } \frac{E B \cdot H \cdot E F \cdot D B}{E B} \text {, or as } H \cdot E F \cdot D B .
\end{gathered}
$$

If the products E F . D B should be summed up for all points from A to Q , for example, we should get a quantity proportional to the vertical displacement of $Q$, arising from the separate minute displacements between $A$ and $Q$. If we pass beyond $C$, we have products of an opposite sign; and it then appears, that, since the ends at $A$ and $B$ are fixed both in position and direetion, the sum of all the products between $A$ and $B$ must equal zero, or, since $H$ cannot equal zero,

$$
\Sigma \mathrm{EF} . \mathrm{DB}=0 .
$$

Therefore the third and last condition is, that the sum of the products of each ordinate, between the arch and the equilibrium poly!gon on the inside of the arch, by its distance from one springin!! point, must equal the similar sum on the outside. It is immaterial which springing is chosen, but all the D B's must bee measured to the same abutment.

20 . This Condition not applicable to Hinged Rib.- It may be expedient to dwell upon this equation a little longer; for the question will apparently arise, why this condition is not also properly applicable to an arch which is jointed or hinged at
the ends. Let a tangent A K be drawn to the rib at the point $A$, and a vertical line be dropped from it to the point $Q$. If the arch is now bent at the point $\mathrm{E}^{\prime}$, by a bending moment which is proportional to $\mathrm{E}^{\prime} \mathrm{F}$, the point Q is moved a distanco proportional to $\mathrm{E}^{\prime} \mathrm{F}$ multiplied by the distance from $\mathrm{E}^{\prime}$ to Q ; but the distance which $Q$ moves in the vertical line $Q \mathrm{~K}$ will be proportional to $\mathrm{E}^{\prime} \mathrm{F}$ multiplied by the horizontal projection of $\mathrm{E}^{\prime}(\mathrm{Q}$, ur I) 'T. and similarly for moments at all other points between $A$ and $Q$. As the tangent at $A$ is fixed in direction in this case, the morement of $Q$ away from the extremity of $K(Q$, or its movement in relation to the tangent at $A$, will be proportional to the summation of the $\mathrm{E} F \cdot \mathrm{~s}$ multiplied by the D ' T "s; and as the abutment $B$ is fixed, the distance of 13 from a tangent at $A$ must be unchanged by any load, or its displacement must he zero, as above. In the case of the rib hinged at the ends, while the above area moments give the deflection from the tangent at $A$, this tangent is not fixed, but changes in direction upon the imposition of a load, and this condition cannot be applied. If, however, one should treat an areh which was fixed at $A$ and hinged at $B$, this condition would be necessary, and all the distances D B would be measured to the hinged end; while the second condition would not apply, and would not be needed.

This third condition was first applied to the determination of the bending moments in continuous bridges and pivot draw spans, in the first edition of Part II. of this work.
21. Remarks: Abutment Reactions; Shear, \&c. - The arch of Fig. 3 is cut by the equilibrium polygon in three places. and it may be cut in four points, giving ats many places of contraflexure. The areas on opposite sides of the rib represent bending moments of opposite kinds, and of which kind is readily known if one remembers that the arch under thrust alwas: moves from the equilibrium polygon. The amount of the weight, not being contained in any of the equations of condition, does not affect the diagram; for $H$ is constant for all points of the arch for any given vertical load, and, not being
equal to zero, is thrown out of the equations. But the weight W does affect the value of H .

If 1-2 represents W in the stress diagram of Fig. 3, and 1-0 and $2-0$ are drawn parallel to N I and N L, 0-3 drawn horizontally will determine the horizontal thrust $H$, while the load-line will be divided at 3 into the two vertical components $\mathrm{P}_{1}$ and $\mathrm{P}_{2}$ of the reactions as marked. These vertical forces are not the same as would be obtained for the case previously considered, nor for a beam only supported at the ends. Such forces would be equal to the divisions of 1-2 made by a line drawn through 0, parallel to a line from I to L . If we notice the arrows drawn at the abutment $\Lambda$, we see that, supposing $P_{1}$ were at first the fraction of W ane to the position of G , or $\frac{\mathrm{C} ~}{\mathrm{~A} B} \mathrm{~B}$ W, we have also at $A$, besides the horizontal thrust $H$, a couple II . A I. There is another couple at the other aboutment, which may be of the same or opposite kind ; their algehaic sum can only be balanced ly vertical forces at the two abutments acting with a lever arm of the span: and these vertical forces must be added to one reaction, and subtracted from the other, bringing $P_{1}$ and $P_{2}$ to the amounts found by the stress diagram. The effect of the comple is the same as if $P_{1}$ had been calculated for the point where N I would meet the horizontal line. This is another example of the principle in mechanies cited in $\S 11$.

The remarks on shear in $s \leq 14,15$, apply equally well here. The direct compression in the ril) at any point is obtained, as before, by drawing a line through 0 parallel to the tangent to the ribl at the point in question, and dropping a perpendicular upon it from the extremity of the line which represents the stress in the adjacent side of the equilibrium polygon. Thus the compression at E will be the distance from 0 along $0-4$ produced to the foot of a perpendicular from 2. Recalling the three conditions just staterl, it will be evident, that, while it will be possible to adjust the two lines of the equilibrium polygon to their proper position by successive trials, it will not, as in the former case, be easy. The three ordinates, $\Lambda \mathrm{I}, \mathrm{G} \mathrm{N}$, and B L,
ean, however, be computed quite readily, and the remainder of the process is very simple. The statements so far made apply to a structure of any outline, so long as it acts as an arch, although some modification will lee called for when the eross-section and the depth vary very much, or when what is known as the moment of inertia is not practieally constant; but, for forms other than regular eurves, the application of these conditions must probably be made by trial.
21a. Shear at a Vertical Section.-The relation of the equilibrimm polygon to the areh which was pointed out in $\$ 5$, Fig. 14, shows how the shear at any vertical section of a loaded rib is affected by the curvature of the arch. In the same way that the ordinates of the ril may be superimposed on those of the triangle which represents the equilibrium polygon for a single load, the two shear diagrams may be placed on one another. Onc will have the form of aimnl, Fig. S, conforming to the load which gives the curve of Fig. 14, and fond from the amonut of rertical reaction which, combined with II, will give a direet thrust at the springing; the other will resemble ade $f g$ l, Fig. S, the usual shear diagran for a single load, which load produces the triangle of Fig. 14. The flanges of the arch take up at each point an amome equal to the ordinates from al to $i n$, and the web or bracing earries the remainder, which will be positive at some points and negative at others, as marked in the Figure. Thus we see that, through the direct thrust, the arch is relieved of a portion of the truss stresses due to both bending moment and shear.

## CHAPTER 11.

## ARCH HINGED AT THREE POINTS.

22. Three-hinged Arch. - Before taking up for treatment any arches of special curves, we will notice the simple case of a rib, of any form, hinged at both ends and the middle, or, as it is sometimes called, the "three-hinged arch." The three hinges or joints may be located anywhere, and two of them may be placed near together at one abutment. reducing the portion of arch between them to a short link or strut, which necessarily lies in the direction of the thrust at that abutment. For the ribs of this chapter it has been stated that the equilibrium polygon or curve is at once definitely located. If a single load is placed at K , on the arch A D B of Fig. 4, hinged at A, D, and B, one of the two straight lines composing the polygon must, starting from $A$, pass throngh $D$, while the other, starting from B , must meet the former on the vertical line drawn through K , as required by the principle of the funicular polygon: A CB , therefore, is the polygon. If $2-1$ represents the weight at $K$, and $2-0$ and $1-0$ are drawn parallel to C B and $\mathrm{A} \mathrm{C}, 0-3$, drawn horizontally, will give the horizontal thrust, while $1-3$ and $8-2$ will be the vertical components of the reactions at $A$ and $I$. Let it be remembered that the total reaction of the abutment at A is, and is in the direction of, $1-0$, although it is often convenient to decompose it into $P_{1}$ and $H$.

A load vertically below E will, similarly, have for its equilibrimm polygon A E B. For different positions of the weight
between $D$ and $B$, all of the vertices of the polygons will be found on the straight line D L, and the portion A D does not change for any movement of the weight on the right half of the areh. A weight on the left half will simply reverse the diagram. The dotted lines show the equilibrium polygons for a weight at such successive points as divide the half-span into five equal horizontal parts, and the corresponding changes in the value of II will be seen in the stress diagram on the left.
23. Formula for H.- If F D, the height or rise of the arch, is denoted by $k$, the half-span A $\mathrm{F},=\mathrm{F} \mathrm{B}$, by $c$, and the horizontal distance F G, from the weight to the middle of the span, by $b$, we shall have $\mathrm{A} G=c+b$, and $\mathrm{G} \mathrm{B}=c-b$. From the similarity of triangles $A \mathrm{DF}$ and 013 , we may write,

$$
: 3-0: 3-1=c: k \text {, or } \mathrm{H}: \mathrm{P}_{1}=c: k
$$

By the usual rule,

$$
\mathrm{P}_{1}=\frac{c-b}{2 c} \mathrm{~W}
$$

therefore

$$
\mathrm{H}=\frac{c-b}{2 k} \mathrm{~W} .
$$

The quantity $c-b$ is to be understood to mean the horizontal distance from the weight to the nearer abutment. $H$ is seen to decrease regularly as the weight moves from the middle of the span.
24. Stone Arches. - In the treatment of stone arches it has often been assumed by writers that the equilibrimm curve passed through either the middle of the depth of the keystone or some other arbitrary point within the middle third of its depth; and a similar assmmption would then be made for the springing-points. Such a treatment immediately reduces the stone arch to this case, and the equilibrium curve can at once be drawn. As such an assumption does not seem to be warranted, it is not thonght expedient to go into the case of the stone arch until later (Chap. IX.); but the reader who desires to look up such a morle of handling the problem is referred to a paper ber William Bell, in the Transactions of the Institute of Civil Engineer: of

Great Britain, vol. xxxiii., reprinted in Van Nostrand's "Engineering Magazine," vol. viii., March to May, 1873.
25. Example. - We will, as an example, show how to draw an equilibrium curve for an areh which is loaded miformly along its rib. Such a distribution will conform quite well to that of the steady load on an arched roof. For definiteness, let the pointed areh of Fig. 5 be of 80 feet span, 40 feet rise, the two ares having a radius of 60 feet, and let it he loaded with 500 pounds per foot of the rib. We may, if we please, divide the ril) into a convenient number of equal portions, which divisions will give us a number of equal weights to lee laid off on the load line. Otherwise we may space off a number of equal horizontal distances. In either case, the load of each space will be considered as concentrated at its centre of gravity; and, if the spaces are small enough, the centre of gravity may: without sensible error, be taken as coinciding with the middle of each space. For the sake of reducing the number of lines, so as to aroid confusion in a small figure, we have divided the half-span into four parts, of ten feet each, measured horizontally; and their centres of gravity will be assumed to be at five feet, fifteen feet, \&c., from the point of support. Draw verticals through these centres of gravity, D, E, le, and G .

To find the weight on each division: The lengths of the several portions of are may, with sufficient exactness, be considesed the same as the lengths of their chords, which chords are perpendicular to the radii which pass through D, E, de. If, then, the load on ten feet is $5,000 \mathrm{lbs}$., draw a b horizontally and equal, by any scale, to ths amount; then will $b g, b, f$, be, and $l d$, drawn parallel to the respective chords, give the amount of load on each division, at the successive points G, F, E, \&c. Upon scaling these amounts we will lay them off upon a vertital line, from 1 to 5 . In order to cause the equilibrimm poiygon to separate from the rill sufficiently to be easily seen in this small figure, we have taken the liberty of doubling the load on D, thus making it $4-6$, in place of $4-5$. The loads will therefore be, successively, about $5,400 \mathrm{lbs} ., 5,900 \mathrm{lbs} ., 7,000 \mathrm{lbs}$., and
$2 \times 10,000 \mathrm{lh}$ s., or $20,000 \mathrm{lbs}$., from G to D , and from 1 to 6 Since $H=\leq \frac{c-b}{2 k} W$, we have for its value

$$
\frac{1}{2} I I=\frac{3.5 \times 5.400+25 \times 5.900+15 \times 7.000+5 \times 20.000}{80}=6,760 \mathrm{lbs}
$$

If the given load were unsmmetrical with regard to a rertieal through ( , it would be necessary to calculate the two rertical components of the reactions at $\Lambda$ and B , or $\mathrm{P}_{1}$ and $\mathrm{P}_{2}$, the reaction at B being laid off from that end of the load line from which was measured the load nearest to $B$, and then to diaw a horizontal line from the point of division between $P_{1}$ and $P_{2}$. on which to lay off the value of H . But, if both sides of the roof are loaded alike, half a diagram and half an equilihriun prolygon will be sufficient. The load on the half-arch being 1-6, $6-1$ will be the vertical component of the reaction at B , and H will be laid off in the direction 1-0. Since we have calculated H for only one-half of the entire load. the above quantity must be doubled, and the total horizontal thrust will be 13.5 .538 1ha... $=1-10$. The reaction at B is therefore $6-0$.

Nothing remains but to draw, first a line from B to the vertical through D, parallel to $6-0$, then one, parallel to $t-0$, from the end of the last line to the vertical throngh E , and so on. the last line, parallel to $1-0$, passing through the hinge at C. an recpuired. The polygon on the side C A will be exactly similar. It is well to have the points of division quite numerons. The maximum ordinate between the rib and the equilibrium polygon, multiplied by If, gives the maximum bending moment.
26. Caution. - As this is the first example, it may be well to panse here, and renew the caution to the draughtsman to lay off the polygon of external forces in the order in which the forces are found in going round the arch or truss: otherwise he will fail to make his equilibrim polygon close on the desired point. Thus, begiming at (i, he should lave the weights at G, F, E. ©c.. or 1-2. 2-3. 3-4, \&c.. plotted, one after the other, down the rentical load line in the direction of their action, until the point

B is reached, for which he draws $6-0$, from 6 to 0 . Then the point 1 gives a similar line from 0 , slanting upwards toward the right; and the remaining loads on the left half of the arch come down a vertical line. and close on 1 , the starting-point. The decomposition of $6-0$ into $6-1$ and $1-0$ does not alter the case. If we had gone round the arch in the opposite direction. this stress diagram would have been reversed, or turned $180^{\circ}$.
27. Relation between Equilibrium Polygon and Curve. - The true equilibrim curve, for the load uniformly distributed along the rib, is a curve which will be tangent to the sides of the fanicular or equilibrium polygon just drawn. The closer together the points D, E, \&c., are taken, the nearer the two will come together. If the points at which the loads are concentrated divide the span into equal portions, that is. if the end distances are the same as the others, so that the portions of load near B and C are concentrated on those points, or, even with unequal spacing, when the load between each two assumed points is carried by those points as required by the principle of the lever, the trme equilibrim curse will pass through the vertices of the equilibrium polygon. Sucla a distribution of load is mate in roofs and bridge trusses, when a half panel weight is thrown on each abutment. Compare Part II., " lhidges." § 58.

The curve assumed by a rope or chain. of uniform weight per foot, when suspended between two points. is called a catenary. Since the equilibrium curve in Fig. 5. if we had not placed the extra weight on D, would have come quite near to the rib. it would have been a close approximation to a catenary. As we expect to make some use of this curve later, we will show how to draw one at that time.
28. The Parabola the Equilibrium Curve for a Load Uniform horizontally. - If the load on this arch were distrib)uted miformly horizontally, the curve of equilibrimm would be a parabola. In case the whole arch were a parabola, with the vertex at the crown, and the load extended over the entire span, the two curves, coinciding at the springing-points and crown,
would be identical throughout, and the rib itself would be in perfeet ernilibrimm. This same point was brought out in reference to the parabolic girder, l'art II., "Bridge," $\leqslant$ §3. That the parabola is the equilibrimen enve for a continuons hadd distributed miformly horizontally, may be shown as follows: -
Let A B, Fig. 6, be a portion of a cord, horizontal at 1 , which is in equilibrimu under such a uniform load, represented by $\mathrm{A} C$, suspended from the cord. The tension at A will be in the line of the tangent $A C$; the resultant of the load $\Lambda C$ will he vertical, and must pass through its middle point D. As the cond $A B$ is in equilitrium moler its load and the reactions or tensions of the other portions of the cord at $A$ and $B$, the tension along the tangent at B must, by the principle of the triangle of forces, also pass through D. As B C. drawn vertically, is parallel to the resultant of the loal, the sides of the triangle B C D will be proportional to the three external forces: and, if $\mathrm{A} \mathrm{C}=x, \mathrm{~B} \mathrm{C}=y, \mathrm{~W}=$ total load on $\mathrm{A} \mathrm{B},=w x$ (where $w=$ load per unit of length), and $H=$ tension at $A$, we have

$$
\mathrm{W}: \mathrm{H}=\mathrm{B} \mathrm{C}^{\prime}: \mathrm{D} \mathbf{C}=y: \frac{1}{2} x
$$

or

$$
y=\underset{2 \mathrm{H}}{\mathrm{~W}_{x}}=\frac{u}{2 \mathrm{H}} x^{x^{2}},
$$

the equation of a parabola with vertex at A .
Therefore an arched rib, of parabolic form, when loaded uniformly horizontally, has no tendency to change its shape, that is, experiences no bending moment, at any point.
29. Suspension Bridge. - A B , of Fig. if may represent a suspension bridge cable, A (' being the half--pan, and CB the height of the tower: hence, if $\lambda \mathrm{C}=r$ and ( $\mathrm{B}=k$, we have for the tension in the cable at the mid-span. 28 ,

$$
\mathrm{I}=\begin{gathered}
\pi r^{2} \\
2!
\end{gathered}=\begin{aligned}
& 4 \cdot r_{2}^{r_{2}^{2}} \\
& 2 k
\end{aligned} .
$$

The tension T at the tower will then be proportioned to H , as B D to D C, or as $\sqrt{k^{2}+\frac{1}{4} n^{2}}$ to $\frac{1}{2} c$ : therefore

$$
\mathrm{T}=\frac{w^{c} c}{2 k} \sqrt{4 k^{2}+c^{2}} .
$$

Each suspending rod must eary the greatest weight that can come at its foos. The pressure on the top of the tower from the half-iban will be the weight of the half-span, or 2 or to this minsi he added the rertical component of the temsion on the anclorage side of the tower. If the cable has the same inclination buth woys, at the top of the tower, the pressme is $2 w^{\prime} c$.
'The manner of stiffening a suspension bridge to resist the tembency to distortion under a partial load is treated in Chap. X.
30. Equilibrium Curve for Partial Load. - If the load extents over a portion only of the span of the arch, and is uniformly distributed horizontally, the curve for the loaded portion is parabolic, while that for an monded portion is a straight line: thus, if the load extends fiom one abutment to the middle, We shall have, on the moloaded half, a straight line from the abutment to the crown, and, on the loaded half, a parabola fiom the crown to the springing. As it was proved in Part II., "Bringes," ${ }^{S} 10$, that amy two sides of the fimicular polygon, when prolonged, meet on the vertical dawn thomgh the centre of gravity of so much of the weight as is included between these sides, the equilibrimu curves for any cases where the rib is hinged at three points can be drawn withont previously determining the value of II. Thus, in the ease just supposed, of a lonl over the half-span, from B to F in Fig. 4, the centre of gravity will be at $G$. Then, if G $C$ is the vertical drawn from (i, the side of the fmicular polygon, or, more properly, the tangent to the equilibrimm curve, at B , must pass through C , where C G meets A I), and the regnired parabola will be drawn fiom I) to B on I) C and B C as tangents. As one point of the curve we have the middle point of a line from $C$ to the middle of the chord D B. We can then find II by drawing 1-0 and 2-0, parallel to A C and C B. Henck"s "Field Book for Railroad Engineers" gives methods for constructing parabolas: two constructions are given in I'art II., " Bridges," şs 20 and 28, one of them applying when two tangents are given.
31. Suggested Examples. - We would suggest the following examples, Plate I • 1st, Given a semicircular rib, loaded
miformly horizontally over the whole span, amel pivoted at the erown and springings: find that the maximm bending moment oecurs at $30^{\circ}$ from the springing, and is egmal to one-sixtecenth of the total load multiplied by the radins of the arch. while 11 is equal to one-fourth of the total load. 2d, Given a lamabolic arch similarly pisoted, and in equilibrium moder a steaty load distributed as above ; add a simila travelling load from one abutment to the middle of the span: prove that the maximum bending moment is found at one-fourth of the span from either abutment, is of opposite signs at these two places, and is equal to one thirty-second of the travelling load then on the arch multiplied by the span, while $1 /$ for the travelling load equals the same product divided by one-fourth the rise of the arch, and for the steady load is twiee as much.
32. Extent of Load to produce Maximum Bending Moment. - It may be desired, when dexigning an arch of this: type, to find the extent of load which will produce the maximum bending moment at each point. and the value of that moment. Suppose the point N. Fig. 4 , to be examined : prolong B N until it meets A D at E: it is then manifest that any load in the rertical throngh E will canse mo bending moment at N ; that the equilibrimm polygon for any load on the right of E will patsis ontside of the arch at N. While the equilibrimen polygon for any load to the left of E will pass inside of N . Therefore the maximum bending moment at N of one kind will be found when all possible loads are put on the arch from $B$ to the vertical through E, and the maximm moment of the "ther kind oecurs when the load extends from A to E . As the arch tends to move away from the equilibrimm polygon, the kind of moment is easily distinguished. If can then be found. the equilibrime curve drawn, the ordinate sealed and multiplied by 11.
33. Braced Arch. - For the reason that the equinibrimm rurve is at once definitely located by intromang three hinges or pirots, no matter what form the arch may have, that type which used to be known as the braced arch, having a horizontal
upper and a curved lower member, the spandrel being filled with bracing, has usually been treated as free to turn at both crown and springings; in that case a diagram may be drawn by Clerk Maxwell's method, as set forth in Part I.. "Roofs." or the stresses may be found from the equilibrium curve. A braced arch, hinged at crown and springings, with an elliptical lower and a straight upper member, carries a track of the Pemnsylvania Railroad over'Thirtieth Street, Philadelphia. (See "Engineering, " July 22, 1870.) Fig. 4A illustrates an application of the three-hinged arch. Polygons are shown for one-half of the structure. The stress diagrams are seen at $\mathrm{C}, \mathrm{I}$, and K .

The t.eo half arehes at bottom of Plate I. show designs which have been used. The braced arell without hinges is treated in Chap. XII.
34. Shear; Temperature. - Since it is not practicable to draw a shear diagram until the form of the ril, is defined, we can only, at present, refer the reader to $\S 14$. After we have disenssed the parabolic and circular ribs, the reader can doubtless work up any special design of the present class for himself.
One adrantage possessed by this type of arch is that changes of temperature have no straining effect, for the crown rises and falls without affecting the two halves of the arch injurionsly. If the crown sinks a little. the value of II will be seen from Fig. 4 to be very slightly increased, while the equilibrium polygon will practically go with the arch.

## CHAPTER III.

## INTRODUCTORY TO PARABOLIC ARCHES.

35. Parabolic Arch. We propose to aply the facts which have leen developed thus far to the arch whose centre line is a parabola. This curve is chosen as one form ; becallse it is, as proved in $\$ 28$, in perfect equilibrium under a load distributed uniformly horizontally over the entire span. As in the case of a suspension bridge, so in some arches of iron, most of the steady load consists of a platform and such other parts as are distributed in accordance with this requirement (the arch itself and the vertical posts which carry the platform giving a somewhat greater intensity per horizontal foot as we approach the springings ), so that, for the former portion, as well as for the travelling load over the whole span, the arch will be subjected to no bending moments, and no shear; hence there will be no stress in the hateing. Then, again, the parabola for a given rise and span is easily plotted and designed : and, lastly, the determination of the equilibrium curves, for the cases taken up, will be simpler than for circular ares, and will naturally prepare the way by rendering the reader faniliar with the steps of the analysis. It may be well to add here that a cireular segmental rib, whose rise is not more than one-tenth of its span, is so nearly coincident with a parabolic arch of the same span and rise, that the investigations which follow will apply with sufficient aceurace to such flat segmental ribs.
36. Vertical Deflection of an Inclined Beam. - Let us
consider the two cases of a horizontal beam and of one inclined to the horizon at an angle $i$; it is known from the usual for mulie for deflection, Part II., " Bridges," Chap. VI., that, other things being equal, the deffection of a bean is directly proportional to the loard and the cube of the length. If, then, the inclined beam is of a length $l$, and the lomizontal one of a length $l$ cos $i$, as shown in Fig. 7, the deflection of sach, measured perpendiculdrly to the respective beams, will, as regards length only, be in the ratio of $l^{3}$ to $l^{3} \cos ^{3} i$. But, if each carries the same load W , the trenscerse component of W . which alone causes flexure of the inclined beam. the longitudinal component producing direct compression, will be W cos $i$; whence the deflection perpendicular to each bean will, for similar points, he proportioned as 1 to cos $^{2} i$. And, again, the vertical component of the deflection of the inclined bean will he to the perpendicular amomet as cos $i$ to 1 ; whence the vertical deflection of the inclined bean will be to that of the horizontal beam of the same cross-section as 1 to cos $i$. As the stiffness of a beam is directly proportioned to its hreadth, should the inclined bean be mate broader in its lorizontal dimension than is the horizontal bean, in the ratio of 1 to cos $i$, the repth being mehanged, the vertical ileflections of the two beams for the same load woukd be exactly the same.
37. Application to Arches. - Any very small portion of an arch, taken within such narrow limits as to be considered straight, behaves like the inclined bean, as regards its flexure moder a loal: and therefore it follows, that if an arch has the dimension perpendicular to its face increased, from the crown to the springing, in the ratio of the secant of the inclination to the horizon, it may he discussed as if it were a beam of uniform cross-section, of the same span, similarly supporter, and carrying the same load which produces flexure. In the arch some of the load does not produce flexure ; in the parabolic rib, for instance before citerl, a uniform horizontal load gives equilibrimm. We propose, in our analysis of the parabolic rib, to make this supposition, that the rib is broader at
the abutments than at the crown in the ratio just mentionerl. and thus to simplify the work of investigation. Iron arches whose flanges or chords are thicker, ats we approach the springing, in the above ratio, while the perpendicular depth between the two tlanges is constant, practically satisfy this case. In this class of ribs the intensity of the dinect thrust on the square inch for a complete uniform load will be the same at all cross-sections.

As we desire the reader to reproduce, on a much larger scale, the figures and problems for himself, we remind him that points on the curve of a parabolic rib are easily found by the coustruction of Fig. 8, Part II., "Bridges."

## PARABOLIC RIB, HINGED AT ENDS.

38. Equilibrium Polygon for Single Load. - Taking up the case of the parabolic rib, hinged at the ends only, let us place a single weight at the point I, Fig. 8. If the lines A C B fulfil the condition of $\S 7$, that the sum of the products of the ordinates I) E and E F for all points of the arch equals zero or

$$
\Sigma \mathrm{EF} \cdot \mathrm{D} \mathrm{E}=0 \text {, }
$$

A CB will be the required equilibrimm polygon. From the reasoning of $\$ 37$, it will be proper to divide the areas above the springing line A B by equidistant vertical lines, moderately near together, scale off the puantities corresponding to E F and D E, and find the proper position of A C B by one or two trials. It can thus be located with all desirable aceuracy, as a slight movement of the point $C$ vertically alters the quantities to be computer very materially. 'The reader who is not familiar with the higher mathematics can thas verify the results we are about to obtain.

Since C Gr may be considered the unknown quantity by which to locate A C and B C, its value may be deduced from the above equation. Let the half-span $\mathrm{AK},=\mathrm{KB},=r$ : the height or dise of the arch at the crown $=k$ : the distance $\mathbf{K} G$, from mid-span to the position of the single weight, $=b$;
and the required maximum ordinate $\mathrm{CG}=y_{0}$. Then will the value of $\mathrm{C} G$ be

$$
y_{0}=\frac{32}{5} k \frac{c^{2}}{5 c^{2}-b^{2}},
$$

which becomes, if $b=n c$, where $n=$ a fraction of the half-spans

$$
\begin{equation*}
y_{0}=\frac{32}{5\left(5-n^{2}\right)} k \tag{1.}
\end{equation*}
$$

a quantity independent of the span of the arch.
39. Proof of Formula. - Let A D, the distance from the abutment A to any ordinate D) E , between A and $\mathrm{G},=x . \quad \mathrm{A} \mathrm{G}=c+1 ; \mathrm{GI}=c-b$. since the ordinates to a parabola from the line A B are proportional to the product of the segments into which they divide the span, we have

$$
\text { 1) } \mathrm{E}: k=x(2 c-x): c^{2} \text {, or } \mathrm{D} \mathrm{E}=\frac{k}{c^{2}}\left(2 c x-x^{2}\right) \text {. }
$$

Also,

$$
\text { D } \mathrm{F}: y_{0}=x: r+b, \quad \text { or D } \mathrm{F}=\frac{y_{0}}{c+b} x
$$

The required condition is that

$$
\begin{gathered}
\Sigma \mathrm{EF} . \mathrm{D} \mathrm{E}=0 \text {, or } \Sigma(\mathrm{DE}-\mathrm{D} F) \mathrm{D} \mathrm{E}=0 ; \\
\mathrm{\Sigma I} \mathrm{E}^{2}=\mathrm{\Sigma} \mathrm{D} \mathrm{~F} . \mathrm{DE} .
\end{gathered}
$$

therefore,
(From the abore expressions we see, that, if the area included between the rib and $A B$ is considered positive, the area of the triangle A C B, superimposed unon it, will be deemed negative as before explained in Fig. 14.)

Substituting the values of the lines from ahove in (1.), multiplying by i $x$, and writing the sign of integration, we get for the left-hand member,

$$
\begin{gather*}
\int_{0}^{2 c} k^{c^{4}}\left(2 c x-x^{2}\right)^{2} d x=\frac{k^{2}}{c^{4}} \int_{0}^{2 c}\left(1 c^{2} x^{2}-4 c x^{3}+x^{4}\right) d x \\
\quad=\frac{k^{2}}{c^{4}}\left(\frac{1}{3} c^{2} x^{3}-c x^{4}+\frac{1}{5} x^{5}\right)_{0}^{2 c}=\frac{16}{15} k^{2} c . \quad(2 .) \tag{2.}
\end{gather*}
$$

For the right-hand member, between $A$ and $G$, we get

$$
\begin{aligned}
& \int_{0}^{c+b} \frac{y_{0}}{c+b} \cdot x \cdot \frac{1}{r^{2}}\left(2 c x-x^{2}\right) d x=\frac{k y_{0}}{c^{2}(c+b)} \cdot \int_{0}^{c+b}\left(2 c x^{2}-x^{8}\right) d x \\
& =\frac{l_{1}, y_{0}}{c^{2}(c+b)}\left(\frac{2}{3} c x^{3}-\frac{1}{1} x^{4}\right)_{0}^{c+b}={ }_{c^{2}}^{k}\left[\frac{2}{3} c(c+b)^{2}-\frac{1}{4}(c+b)^{3}\right] \cdot
\end{aligned}
$$

For the portion between $G$ and $B$, if we write $c-b$ for $c+b$, and reckon $\sim$ from B to the left, I) F will equal $\frac{y}{c-b} x$, while D E will be unchanged;
so that the integration for the right-hand member of (1.), between $G$ and $B$, and between the limits $x=0$ and $x=c-b$, will give, simply by writing $-b$ for $+b$,

$$
\begin{equation*}
\frac{k y_{0}}{c^{2}}\left[\frac{2}{3} c(c-b)^{2}-\frac{1}{4}(c-b)^{3}\right] . \tag{4.}
\end{equation*}
$$

These two portions (3.) and (4.), for the right-hand member of (1.), being added together, will produce, when the terms with the odd powers of $b$ are cancelled,

$$
\frac{k y_{0}}{c^{2}}\left(\frac{5}{6} c^{3}-\frac{1}{6} c b^{2}\right)
$$

Finally equate this value with (2.) to satisfy (1.), and

$$
\begin{equation*}
\frac{k y_{n}}{6 c}\left(5 c^{2}-b^{2}\right)=\frac{16}{15} k^{2} c ; \text { or } y_{0}=\frac{32}{5} k \frac{c^{2}}{5 c^{2}-k^{2}} \tag{5.}
\end{equation*}
$$

which is the desired value of CG in terms of the constant quantities, and the variable distance K G. This expression is plainly applicable to points on either side of K .
40. Formula for Horizontal Thrust. - For any position of the weight, plot the value of $y_{0}$, and draw the equilibrimu polygon. Then draw two lines from the extremities of the load line $W$, parallel to the sides of the polygon, and thus retermine $H$, and the two vertical components of the reactions, which vertical components will be the same as for a beam supported at its ends. But, from the simple relations of the similar triangles A G C and 031 , Fig. 8, as also BGC and 032 , we may write a general formula for H , if desired. Thus we have

$$
\begin{aligned}
& y_{0}: c-b=\mathrm{P}_{2}: \mathrm{H}, \quad \text { or } \mathrm{P}_{2}=\frac{y_{0}}{c-b} \mathrm{H} ; \\
& y_{0}: c+b=\mathrm{W}-\mathrm{P}_{2}: \mathrm{H}, \text { or } \mathrm{W}-\mathrm{P}_{2}=\frac{y_{0}}{c+b} \mathrm{H} .
\end{aligned}
$$

Eliminating $P_{2}$ in the second equation, by substituting its value from the first one, we get

$$
\begin{aligned}
& \mathrm{W}-\frac{y_{0}}{c-b} \mathrm{H}=\frac{y_{0}}{c+b} \mathrm{H}, \text { or }\left(c^{2}-b^{2}\right) \mathrm{W}=2 c y_{0} \mathrm{H} \\
& \mathrm{H}=\frac{c^{2}-b^{2}}{2 c y_{0}} \mathrm{~W}=\frac{1-n^{2}}{2} \cdot \frac{\overline{5}\left(5-n^{2}\right)}{32} \cdot \frac{c}{k} \mathrm{~W} .
\end{aligned}
$$

This value also will apply to a load on either side of the centre.

It will be observed that, to obtain this value of H , we have simply to divide $\frac{1}{2}\left(1-n^{2}\right)$ by the factor which multiplies k in (1.), $\leqslant 38$, to obtain the variable factor here.
41. Computation of $y_{0}$ and $\mathbf{H}$. - The numerieal values of these factors are worth obtaining, as, the computations once made, the results apply to every parabolic rib with pivoted ends. Let the span of the arch be divided into any convenient number of equal parts, and, for illustration, smppose that the number is ten, as shown in the figure ; let a weight W be placed successively over each point of division, being supported by the rib. The calculation may conveniently proceed in the following manner:-

Find the different values of $y_{0}$ for different positions of W , by equation (1.), §38. Then compute H by $\$ 40$. The ealculation and results are given below; the equilibrimm polygons and values of $H$ for one-half of the arch are represented in Fig. 8. As $n^{2}$ is positive, whether $n$ is + or - , the values of $y_{0}$ and II will be symmetrical on each side of the centre.


Multiply these factors by $\frac{c}{k} \mathrm{~W}$ to give II.
For any other desired division of the span, proceed in a similar way.
42. Remarks. - If every point of division were loaded with W at the same time. the value of the horizontal thenst would be equal to the sum of the $1 \mathrm{I}^{\circ} \mathrm{s}$ for each load, that is, the factor in column 0 phas twice each of the others, and the sum
 If a truss were miformly loaded horizontally, the bending moment at the middle would be one-eighth of the total load multiplied by the span, or, for a truss of ten panels, with $W=$ one panel load,

$$
M=\frac{10 \mathrm{~W} \cdot 2 c}{\mathrm{~S}}=2 \frac{1}{2} c \mathbb{W}
$$

and the tension in the lower chord, or the compression in the upper chord, would be found by dividing this quantity by the heiglit of the truss, $k$. If the span of the arch just treated had been divided into twenty equal parts, the value of Il. for loads at all the points of division, would have been $4.990 \frac{{ }^{\prime}}{k} \mathrm{~W}$. The truss, as before, would give $\frac{20 \mathrm{~W} \cdot 2 c}{8 k}=5 \frac{c}{k} \mathrm{~W}$.

We thus see that the equilibrim polygon, for a number of equal loads. equidistant horizontally, on a parabolic rib, gives a value of $H$ approximating closely to that for a miform load on a truss of height $k$, coming nearer as the loats increase in number, and agreeing when the load is continuons. Then the equilibrimm polygon becomes a curve, coinciding perfectly with the paraholic rib, and gives the horizontal thrust to which we are accustomed in the bowstring girder under a maximum load.
13. Computation of Bending Monents. - While the ordinates can be readily scaled from a diagram, one who wishes maty compute values of the bending moment $M$ for numerous points. when $\boldsymbol{W}$ is placed on any one 1oint. If $y$ denotes the ordinate from A B to the inclined line, and $z$ the ordinate of the parabola from any point 1 , the bending moment may be written, -

$$
\mathbf{M}=\mathrm{II}(y-z)
$$

## ARCHES.

If put in this form, it will be seen, that, in the neighborhood of $y_{0}$, , I will be positive, coinciding with the moments for a beam supported at its two ends. As this is the most familiar flexure of a beam or truss, we have chosen to consider it as positive: $\$ 13$. The ordinates !/ and $\approx$ can be readily calculated from the figure. Thus, if the weight is at 0.4 c from the middle of the span, we have found $y_{n}$ to be 1.3223 k . If the span is divided into ten parts, the number of divisions on one side of the weight being seven, $y$ will be successively $\frac{1}{7}, \frac{2}{5}, \frac{3}{5}$. \&c., of $y_{n}$ : on the other side $y$ will be $\frac{1}{3}$ and $\frac{2}{3}$ of $y_{n}$. The sum of the denominators always equals the number of divisions, and the fractions increase from both ends up to unity. After finding the first $y$ at each end, we get the others by simple addition, and the row is checked by obtaining $y$, at the proper point. As stated in $\$ 39$, the ordinate $z$ is proportional to the product of the segments into which it divides the span; $o r$, if it is at id distance $n c$ from the middle, we have,

$$
z=(1+n) c(1-n) c \frac{k}{c^{2}}=\left(1-n^{2}\right) k .
$$

The factors by which $l$ ' is to be multiplied can therefore be at once obtained by taking the decimals which are found in the second line of the table for $y_{0}$ \& $\$ 41$.

The computations may then be set down in the following shape, viz. : -

| $\begin{gathered} \text { Point } \\ \text { of } \\ \text { Division. } \end{gathered}$ | Values of M. |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | 1 | 2 | 3 | 4 | 5 | ${ }_{6}$ | $y_{0}$ | 8 | 9 |  |
| ${ }_{7}^{n} y_{0}=$ | . 1889 | .3778 | .5667 | . 7556 | . 9445 | 1.1334 | 1.3223 | . 8815 | . 4408 | $\frac{n}{3} y_{0}$. |
| z $=$ | . 36 | . 64 | . 84 | . 96 | 1.00 | . 96 | . 84 | . 64 | . 36 | $k$ |
| $y-z=$ | . 1711 | -.262.2 | -.2733 | -. 2044 | -. 0555 | +.1734 | +.4923 | +.2415 | +.0808 | $k$ |

$$
\text { Multiply by } \mathrm{II}=0.3176 \frac{\mathrm{r}}{\mathrm{k}} \mathrm{~W} \text {. }
$$

$\mathrm{M}=-.0543-.0833-.0868-.0649-.0176+.0551+.1532+.0767+.0257 \quad$ $\mathbf{W}$
With the explanation already given, this table will be understood. The letter $y_{0}$ is placed over 7 as a convenience, to show that the value $y_{0}$ occurs at this point of division. If the load is on the right of the centre, these numbers run from the left abutment; if the load is on the left of the centre, they must be reckoned from the right abutment.
41. Table of Bending Moments. - We have carried ont this computation for a load at each joint successively, the span being divided into ten ryual parts, and have prepared a table given on p. 53. A table for a span divided into twenty parts may be found in "Engineering News," Vol. IV. p. 108. As a load on either side of the middle gives the same set of values in the reverse order, it is necessary to calculate but one-half of the table.

As many decimals may be taken as will give sufficiently accurate results. By the nse of logarithms the labor of preparing another table for a different mmber of divisions is very little. Each column belongs to the point of division whose number stands at its top, the numbers commencing at the left abutment. Each horizontal line contains the factor for bending moment at each point of division for a load $\mathbb{W}$ on the point marked at the begimning of the line. The valus of 11 are placed for convenience in the last colum.

It is worthy of notice, that, while the value of $y_{0}$ is independent of the span of the arch, M is independent of the height of the arch. As it was proved, in $\$ 28$, that the parabola is the "duilibrimm cure for a load distributed uniformly horizontally, this arch ought to be very nearly in equilibrimm when we place at once on each one of the nine points a load W : by footing up the vertical columns of the table we shall find but a very small residnal moment at each joint.
45. Interpolation. - In the solution of a particular example, it may happen that the points at which the weight will be concentrated will not coincide with the points of division which we have taken. It will then be necessary to determine new values of $y_{0}$ and $H$, which may be done by the original formulie or by interpolation. A new table of M may then be calculated, values may be interpolated in the one given here, or, if preferred, from the value of $H$, and the vertical components of the reactions, we may draw an equilibrium curve for any combination of loads. The table here given, if not directly applicable in all cases, serves two purposes; one to show how a similar table can be made, and the other to indicate, by inspection, what armangement of loads on any arch will produce the maximmm bending moments.

If the successive values of any quantity increase at a tolerably miform rate, any intermediate value between two given ones may be fomm by simple proportion. Otherwise we may use the formula for interpolation, -

$$
\text { Desired quantity }=a+f\left[\mathrm{D}_{1}-\frac{1}{2}(1-f) \mathrm{D}_{2}\right],
$$

in which $a$ denotes the first given quantity, $f$ the fraction of a division from $a$ to the desired quantity, and $D_{1}$ and $D_{2}$ the first
and second differences. To illustrate, take the valnes of II in $\$ 41$. If we place these in a column as below, find the amount

| 〕. | II. | $1)_{1}$. | I). |
| :---: | :---: | :---: | :---: |
| 0 | . 3906 |  |  |
|  |  | -. 0186 |  |
| . $2 c$ | . 3720 |  | -.0358 |
|  |  | --.0544 |  |
| . $4 c$ | . 3176 |  | -. 0312 |
|  |  | -.0556 |  |
| . 60 | . 23.0 |  | $-.0235$ |
|  |  | -. 1094 |  |
| . 8 c | . 1226 |  |  |

of increase from quantity to quantity, and then subtract these differences from one another, marking each + if it is an increment, and vice versa, we obtain the columns of first and second differences as marked. Now suppose that we wish to determine a value of H at $b=.5 c ; a$ will be $.3176, f=\frac{1}{2}$, $\mathrm{D}_{1}=-.0856$, and $\mathrm{D}_{2}$ for an average value between .0312 and $.0238,=-.0275$. If we substitute in the formula, it then becomes

$$
\begin{gathered}
\text { II (for } .5 c)=.3176+\frac{1}{2}\left[-.0856-\frac{1}{2} \cdot \frac{1}{2}(-.0275)\right] \\
=.317\left(f+\frac{1}{2}(-.08 .56+.0069)=.2783 .\right.
\end{gathered}
$$

The factor for $y_{0}$, at one-third of the interval between $4 c$ and $.6 c$, will, in the same way, be

$$
1.3223+\frac{1}{3}\left[.0570-\frac{1}{2} \cdot \frac{2}{3}(.0283)\right]=1.3382 .
$$

Careful heed must be paid to the signs.
4ti. Examples. - It will help to fix the ideas, if we draw an equilibrium polygon for some combination of weights. We shall take but a few loads, in order to have the diagram clear; but the reader may vary the example by taking other amounts in other places. The values of the two vertical components of the abutment reactions will be the sums of the components for each load, and the amonnt of II for the whole load will be the sum of the separate $H$ s. Mnltiply each numerical factor which belongs to H by the mumber of mits of weight which are
placed on the point to which the factor refers, add "up the products, and plot the resulting value of H horizontally from the point of division on the load line between the two vertical components of the reations.

For example: Let us draw the equilibrium polygon for an arch of 100 feet span, 20 feet rise, whose weight is at present, for simplicity"s sake, negleeted, when it is loaded with weights of 3 tons, 2 tons, 4 tons, and 2 tons, at the end of the $3 \mathrm{~d}, 6$ th, 8th, and 9 th division from the left, of ten equal horizontal divisions, as shown in Fig. 9, where the numbers denote the weights and the points of division above mentioned. The supporting force on the left will be

$$
\begin{gathered}
P_{1}=2 \times 1+4 \times 2+2 \times 4+3 \times 7=3.9 \text { tons. } \\
10 \\
\therefore \quad P_{2}=7.1 \text { tons }
\end{gathered}
$$

From the table for H ,

$$
\begin{aligned}
\mathrm{I}=(0.3176 \times 3 & +0.37 .2 \times 2+0.232 \times 1+0.1226 \times 2) \frac{50}{20} \\
& =2.87 \times \frac{5}{2}=7.175 \text { tons. }
\end{aligned}
$$

These quantities are plotted in the stress diagram, as seen in the figure, and the equilibrimm polygon is then drawn. The reader who reproduces this figure, or draws another, can be assured of the atemacy of the construction by the closing of the equilibrium polygon on the point of support. The weight of the arch itself may he accomsted for by concentrating the proper amount at each point of division. Such amomuts will increase towards the springing in proportion to the sinate of the secant of inclination to the horizon ; for we recall the fact that the parabolic rib is to increase in hreadth from crown to springing, and the amount in length projected into a horizontal foot intreases in the same way. The weight of each division of the areh can be obtained with sufficient aceuracy from a moderately large figure.

Another good construction is the eurve for a miform load aver one-half of the span. The equilibrimu curve for such a load. on the left half of Fig. 8 , is represented in that figure; the
work may be carried out in detail by the reader, and compared with the same curve for the three-hinged rib.
47. Numerical Value of M. - It will be seen that the polygon and rib of Fig. 9 approach quite nearly at 3. We can find the distance between them vertically, if we wish, from the table of M. The benling moment will be, taking the column 3, $\mathrm{M}=50\left(+.153 \times 3-.073 \times 2-.075 \times 4-.043 \times{ }^{2}\right)=-3.650$ ft.tons.

$$
\mathrm{M}=\frac{-3.65}{7.2}=-0.5 \mathrm{ft} .=y-z
$$

A similar operation may be performed at any other point.
48. Shear Diagram. - This investigation of shear is intended to apply to ribs of an I-section or to those framed with (n)en-work or skeleton webs, and not to those of solid section, rectangular, circular, or otherwise, nor to stone arches: in these latter classes the shearing forces need seldom be taken into account.

Adhering still to the case of a single weight W , at a distance b) from the middle of the span, we found that the vertical component. $l_{2}$, of the reaction at the end nearest to the weight, would be ${ }^{\prime 2} \frac{+\prime}{2,}$ " $W$, and at the other end $\frac{c-b}{2 c} \mathrm{~W}$. As seen in Fig. 8, the diagram for shear on a beam will be, if we take the shear on the left of any section, $a d=\mathrm{P}_{1}=3-1$, on the left of the weight, and $l_{!}=-\mathrm{P}_{2},=3-2$, on the right of the weight, giving the two rectangles included between al and the broken line de.f. As the parabola is in equilibrimm under a load of miform intensity horizontally ( $\$ 28$ ), in which case there will he no bracing required, - no shear for any bracing to resist, - it is manifest that the diagram for that portion of the shear which is here carried, at each vertical section, by the flanges on chords, must be similar to the shear diagram for a uniform load on a beam supported at both ends; that is, to such a figure as a imm . lf, then, we can determine the value of "i, or of the equal ordinate $l n$, we can draw this portion of the figure.

It is a well-known property of the paraboli, that a tangent at
the springing of the arch will intersect the middle ordinate at a distance $k$ above the crown，equal to the rise of the areh．If， then，we draw a line $0-4$ in the stress diagram，parallel to the tangent A L，drawn as just deseribed，the distance 3－4，inter－ cepted on the vertical line，will be the amount of vertical force necessarily combined with H to give a thrust coineiding with the rib at the springing point．Lay off，therefore，3－4 at a ， and an equal amount at $l n$ ；then draw the straight line $i n$ ，cut－ ting $a l$ at its middle point $m$ ：the ordinates to this line from $a l$ ，

Parabolic Rib，Hinged at Enis．
§44． $\mathrm{M}=m c \mathrm{~W} . \quad$ Values of $m$ ．

|  | 1 | 2 | 3 | 4 | －） | ＊ | 7 | 8 | 9 | 11 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| W on 9 | －．024 | －． 039 | $-.04{ }^{\circ}$ | －．038 | －．023 | $+.002$ | ＋．037 | ＋．108： | ＋． 136 | ． $123 \frac{6}{6} \mathrm{~W}$ ． |
| ＂ 5 | －． 044 | －． 068 | $-.075$ | －． 063 | －． 032 | ＋．017 | ＋．105 5 | ＋．171 | ＋．076 | ．23：${ }^{\text {n }}$ |
| ＂ | －．054 | $-.08 .3$ | －．087 | －． 1165 | －．015 | ＋．103． | ＋．133 | ＋．110 | ＋．025 | ．314 |
| ＂ | －． 0.04 | －．0\％ 9 | $-.073$ | －．03i | ＋．0こ | ＋． 123 | ＋．047 | ＋．102 | －．014 | ．372 |
| ＂ | －． 041 | －．050 | －．028 | ＋．025 | ＋．103 | ＋． 1025 | －．029 | －．110\％ | －． 041 | ． 391 |
| ＂ 4 | －．014 | ＋．002 | ＋．047 | ＋．123 | ＋．がい | －．0137 | $-.083$ | －． 1178 | －0．54 | ．3：2 |
| ＂ | ＋．025 | ＋．076 | ＋．153 | ＋．035 | －．018 | －． 065 | －．．057 | －．08：； | －．054 | ．31\％ |
| ＂ 2 | ＋．076 | ＋．171 | ＋．085 | ＋．017 | －．03？ | －． 166 | $-.075$ | －． 068 | －．044 | 23： |
| ＂ | ＋． 136 | ＋． 082 | ＋． 037 | ＋．012 | －．1023 | －．038 | －．043 | －．039 | －． 024 | ．123 |

\＄n3．$V=n \mathbb{V}$ ．Values of $n$ ．

|  |  | 1 |  |  | 4 | 6 |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| W＇on 9 | －． 121 | $-.00^{2}$ | －． 023 | ＋．102 | ＋．105） | ＋．12 | ＋．173 | ＋ 223 | ＋．272 | －．6\％ |
| ＂ 8 | －． 218 | －． 125 | －．03： | ＋．061 | ＋．15：3 | ＋．247 | ＋．339 | ＋．432 | －．45 | －．382 |
| ＂ 7 | $-.252$ | $-.145$ | －．018 | $+.109$ | ＋．236 | ＋．364 | ＋． 491 | －．3＊2 | －．255 | －．124 |
| ＂ 6 | －． 270 | $-.121$ | ＋．028 | ＋．17 | ＋．305 | ＋．tin | －．37 | －．229 | －．079 | ＋．069 |
| ＂ 5 | －． 204 | －．047 | ＋． 109 | ＋265 | ＋．t22 | －．+122 | －．26．\％ | $-.109$ | $+.047$ | ＋224 |
| ＂＋ | －． 069 | ＋．079 | ＋．228 | ＋．37 | －．t5 | $-.325$ | $-.17$ | $-.028$ | ＋．121 | ＋． $\boldsymbol{F}_{11}$ |
| ＂ 3 | ＋．128 | ＋．25． | ＋．392 | －． 491 | －． 364 | －． 236 | －． 109 | ＋．018 | ＋．145 | ＋．272 |
| 2 | ＋．382 | ＋．475 | －．432 | $-.339$ | －．247 | －．153 | －．061 | ＋．032 | ＋．125 | ＋．218 |
| ＂ 1 | ＋．678 | －．272 | －．223 | $-.173$ | －． 125 | －．075 | －．026 | ＋．023 | ＋．072 | ＋．121 |

at all points，will represent the amount of vertical force to be combined with the horizontal thrust to put the rib）in equilib－ rium．The remaining ordinates are drawn at the middle of
each division ; and, where the amome subtracted is greater than the original shear, the remainder will be of the opposite sign. The signs are placed in the areas of this figure; and it will be apparent that the ordinates are reckoned from the inclined line i $n$, all above that line in our figure representing positive or upward shear on the left of a vertical plane of section, while those belore in will be neyative. See p. 31.
49. Shear on a Normal Section. - To obtain the shear on a right or normal section, as at $Q$, we must draw a line $q s$ parallel to the normal section at $\left(Q\right.$, and project $r \boldsymbol{q} /{ }^{\prime}$ finding $s q$ as the shear at $($ Q. A similar construction will determine the shear at any other point. The property of the parabola before alluded to makes it easy to find the direction of $q \times$, which will be perpendicular to a tangent at $Q$; a tangent at $Q$ will strike $K L$ at $S$, a distance above the crown equal to that of the extremity R of the horizontal line $\mathrm{Q} R$ below it. What has been done by the above steps may also be easily seen from the sketch above Fig. 8. At $A, \mathrm{I}_{1}$ will be ad or $3-1$, and the whole vertical force to be combined with H will be a $i$ or $3-4$, which when subtracted from ad leaves $i, d$ or $4-1$ as the negative shear on a vertical plane, and $\mathrm{F}, t d$, or $6-1$, as the shear on a right section at $\Lambda$.

In treating any arched rib, we shall desire to find the maximum shear at any section prodnced by a combination of weights at several points. It will he easier to find the sum of the several shears on a rertical section from single weights, and then find the normal component once for all, than to resolve each vertical shear separately; hence the shear diagram of Fig. 8 and of subsequent figures will simply show the shears on the several vertical sections before they are projected on the normal sections.
50. Formula for Vertical Shear. - A formula for this vertical shear may be deduced without difficulty. If Y is the ordinate to $i /$ from any point of al, and $Y_{1}$ its value at the springing, we have from the statement of the last section,

$$
\mathrm{Y}_{1}: \mathrm{H}=2 k: r \text { or } \mathrm{Y}_{1}=\frac{2 k}{c} \mathrm{H}
$$

The vertical shear $V$ in the web, at the abutment on the left, will then be,

$$
\mathrm{V}=\mathrm{P}_{1}-\mathrm{Y}_{1}=\frac{c}{2} \frac{b}{c} W-\frac{2 k}{c} \mathrm{II}
$$

For successive points, $P_{1}$ will remain the value of the original shear mintil we pass the weight, when it will become $P_{1}-W$ or $-P_{2}$. $Y$ will diminish at a constant rate; and, if we deduct at each point the ordinate from $a l$ to the inclined line, we shall get the desired results.
51. Computation of Shear. - As an example we will find the vertical shear miducay between the points of division of the arch of Fig. s with the load there shown.

$$
\mathrm{P}_{1}=0.3 \mathrm{~W} ; \mathrm{P}_{2}=0.7 \mathrm{~W} ; \mathrm{H}=.3176 \frac{c}{k} \mathrm{~W} ; \mathrm{Y}_{1}=.6352 \mathrm{~W}
$$

This value of $Y_{1}$ is applicable to any parabolic arch with hinged ends, since it involves neither $c$ nor $k$ : $Y$ at the middle of the first space $=\left(.635-\frac{.635}{10}\right) \mathrm{W}=.572 \mathrm{~W}$; for every succeeding ordinate it diminishes $\frac{.03 .5}{5} \mathrm{~W}$.

Values of V.

| space. | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 |  |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $1_{1}^{\prime}$ | .3 | .3 | .3 | .3 | .3 | .3 | .3 | -.7 | -.7 | -.7 | $-\mathrm{P}_{2}$ |
| Y | $.5 \div 2$ | .445 | .318 | .191 | +.064 | -.064 | -.191 | -.315 | -.445 | -.572 |  |
| $\mathrm{P}-\mathrm{Y}$ | -.2 .2 | -.145 | -.018 | +.109 | +.236 | +.364 | +.491 | -.382 | -.255 | -.128 | W. |

Three decimal places here will be as exact as four in the ralues of $\mathbf{M}$. It will be seen by the ordinates in the shear diagram of Fig. 8, how the sigus change.
52. Remarks on Shear. - We repeat that, as $\mathrm{P}_{1}$ was taken as positive, the signs of the shears apply to the left side of eath vertical or each normal section. In Fig. 10 the sketch marked $R$ is an instance of positive shear. which acts up or outward on the left of the imarimary section and inward on the right of the same section. From the way in which the two parts of the arch will tend to slide at the section, we see that at $R$ a tie will be required sloping down from the upper chord to the right (or a strut in the opposite direction), while negative shear, as represented in the sketeh marked $S$, ealls for a tie in the reverse direction.
53. Table of Shears. - A table has been computed by the precediug process, for shears at the middle points of ten elual spaces, into which the span is rivided. It is intended to supplement the previons table of bending moments, and will serve as a guide for the calculation of ally table with a greater or less number of spaces. It will be fomm on 1 . $\bar{\pi} ;$. $A$ shear at a joint cam be fomm, if desired. by taking the mean of two adjacent shears just obtained. It is easy to select from this table that combination of loads which will give on any parabolic arch. hinged at the ends muly, the maximm shear of rither kind in any one division, one arrangement being the eomplement of the other. These shears, as shouk be the calse, font up sery nearly to zero for an equal load on every joint. It is cmly necessary to calculate one-half of the table: the other half will contan the same mombers in the reverse order, with the ondmite signs. A table for all arch of twenty divisions was printed in "Engineering News," vol. iv., 1. 1こ2.
54. Extent of Load to Produce Maximum Bending Moments and Shears. - In single-span trusses the maximum bembing moments, and eonsequently the maximmon stresses in the elmols, oeroll when the britge is entirely covered with the live load: and the greatest shear at any section, or the greatest stress in any brace, exists when the bridge is covered with live load over one or the other, msually the longer, of the two segments into which the section divides the span. A simple inspertion of the tables for M and $V$. lately given, will show that such rules are not true for an areh. Why this is so, will he seen, if we eonsider the fact that the portion of the areh, Fig. 8 , between $B$ and the point where ('A crosses the rib, is moder a bending moment of the positioe kind, when there is a single weight at $I$, while from that point to A bending moments of the negative kind exist : and that an addition of another load near I will increase in amount most of the positive and negative moments, while one placed on the left half of the arch will hate an (1pposite rffect. The shearing fores for the braces. depending upon the change of stress in the flanges, will also be affected in the same way.

While an inspection of Fig .8 will show, as was pointerl ont with regard to liig. 4, in $\leqslant 32$, the extent of load to produce the maximum bending moment at any one point, and while the
foad to produce maximmm shear at the same point cant also be ascertained by inspection, os 15 , an attempt has been made to represent, by the horizontal lines in the diagrann, Fig. 11, those positions of the live load, or the extent of the loaded portion, which will give the maximm moments of both kinds at each of nineteen points of division represented in the figure, and also that arrangement of the live load which gives the maximum shear of either kind at the middle of each division. The full line denotes the loaded portion of the span when the maximum positive moment vecurs at that point whose number is plated at the end of the line, positive being molerstood to mean that kind of moment which would make a previonsly straight beam coneave on the upper side: and the remaining portion of the span must alone be covered with the live load to produce the maximum negative moment at the same point. Thms the maximum positive bending moment at 2 , and at $\because$ also, is found when the load is on all points from the left to $\bar{i}$ inclusive. A load from 8 to the right abutment gives the maximum - M. The maximum + M at 11 occurs when the arch is loaded from 9 to $1 t$ inclusive.

The live load required to produce the greatest positive shear through the web in any division is indieated by the broken line; and a load over the complementary blank portion will give the maximmm shear of the opposite kind.
5.). Resultant Maximum Stresses.-The steady or fixed laid. muless distributed miformly horizontally, gives definite bending moments and shears. If, at a given point, the bending moment from fixed weight is + , the addition of rolling loar which gives the maximmm $+M$ at that point will give an actual maximum + M. That rolling load whieh, in itself, gives a maximum - M. if large enongh to prevail against the +M , will produce an actual maximum - M ; but, if not, will only canse a minimum + M. Similar remarks might be made concerning shear. An alisolute maximmm II of either kind, for a miform luarl, will be fomm, if we sum up the quantities in the table, page $\quad$ di, to oecen at the middle of the half-span. The
loads to produce these values are seen in Fig. 11. The absolute maximum $\pm \mathrm{F}$ is found at the abutments, while another value, nearly equal in amount, occurs at the crown.

As the direet thrust must be combined with the stress from bending moment; as every additional weight increases the direct thrust, while added weights at some points will diminish the previously existing bending moments, and hence the tension and compression caused by them, the maximum resultant tension and compression in any portion of the flanges will be found for other arrangements of loads than those which give maximum lending moments. As the stress from bending moment depends upon the depth of the rib, and not on the rise of the arch, while that from H depends on $k$ and the inclination of the rib, it is not easy to determine a general expression for the load to produce maximum stress in either flange, but for ribs of trusswork, like Fig. 4B, the solution is more simple.

Suppose that this sketeh represents a portion of the rib of Fig. S, which figure shows only the centre line or axis of the rib. If one of the erfuilibrium polygons, as $O P$, passes between the two chords or flanges, the thrust along O P multiplied by the perpendieular dropped on it from a joint, as A or I (or H multiplied by the vertical ordinate), will be a moment which must be resisted by a compression in the opposite chord piece, E I opposite A, or A B opposite I. For that piece alone prevents rotation or collapse about the joint. Hence the quotient of the moment divided by the perpendicular A G dropped from the joint on the chord of the opposite panel length, or on the flange, will give the compression in that member.

If the polygon, as $\mathrm{T} V$, passes entirely without the rib, as at A, the thrust in T V multiplied by the perpendicular A P' (or II $A F^{\prime}$ ) and divided by the depth of the rib $A(G$ will be the tension in EI, while II • IF' divided by the perpendicular from I on the eloord A B will be the therust in A B.

Also, for the polygon S N R, the thrust in S N multiplied by ID, or in NR multiplied by IL, or II • IN, when divided by
the normal depth of the rib, gives the tension in A B, and II A E' divided by A G gives the thrust in EI.

Hence all equilibrium polygons passing, at any right section, between the two chords or flanges, will canse compression in both flanges. Equilibrinm polygons which pass above or helow the rib at any right section will cause compression in the nearer Hlange and tension in the farther flange. It is therefore comparatively easy to select, from a drawing like Fig. 8 , when the outline of the rib is drawn as in Fig. 4B, those arrangements of moving loads which will cause maximum compression and minimum compression or possible tension in any chord-piece or flange. See also § 106 .
56. Example of Flange Stresses.-Let the rib of Fig. 9, 100 feet span and 20 feet rise, be loaded with the four weights only. If the rib is made of a web and tro flanges $2 \frac{1}{2}$ feet from centre to centre, what will be the stress in each flange at 8 ? By the ordinate abore $\varepsilon$, or from the table,

$$
\mathrm{M}=(.082 \times 2+.1 .1 \times 4+.002 \times 2-.083 \times 3,50=30.15 \text { foot tons. }
$$

Dividing by the depth, $2 \frac{1}{2}$ feet, we get 12.06 tons compression in the upper flange and tension in the lower flange, from bending moment only. As the middle ordinate is 20 feet, the one at $\delta$ will be $20 \times \frac{1}{2} \frac{5}{5}=12.5$ feet, or 7.2 feet less than the crown ordinate; the tangent at this point will therefore strike the middle rertical at $\bar{i} .2$ feet abore the crown. Drawing 0-5 parallel to this tangent in the stress diagram, and dropping perpendiculars $3-6$ and $4-i$ on it from 3 and 4 , we find that the direct thrust just to the right of 8 is $0-6,8.6$ tons, and to the left of 8 is $0-\overline{7}, 6.9$ tons. Half of each force will be found in each flange. To the right of point S we therefore have $12.06+4.3=16.36$ tons compression in the upper flange, and $4.3-12.06=7.66$ tons tension in the lower flange; to the left of S we find $12.06+3.45=1.5 .51$ tons compression in upper, and $3.4 \check{5}-12.06=8.61$ tons tension in lower flange. On a right section close to, but on the right of $s$, there will be $4-\overline{7}, 2.1$ tons positive shear, and on the left of 8 will be found $3-6,1.5$ tons negative shear, to be resisted by the web.

## CHAPTER IV.

## PARABOLIC RII WITH FIXED ENDS.

57. Values of Ordinates. - Passing next to the parabolic arch, fixed at the ends, we recall, from $\$ 16$, that, to locate the equilibrime polygon for a single load at any point, we need threr ordinates, one at each end, and the third passing through the weight, and that the three conditions by which these must be obtained are, 1st, that the change of span is zero $; 2 d$, that the change of inclination at the abutments is zero ; and. Bo. that the abutment deflection is zero. As expressed in the notation used, the three equations of condition are

$$
\begin{array}{r}
\Sigma E F \cdot \mathrm{I} \mathrm{E}=0 \\
\Sigma \mathrm{E}=0, \\
\Sigma \mathrm{E} F \cdot \mathrm{I} \mathrm{~B}=0
\end{array}
$$

If. in Fig. 12, I N L represents the desired equilibrium polygon for a weight W, attached to the rib A Q B at a point distant T $\mathrm{G},=l$, horizontally from the middle of the span ; and if the shan $\Lambda B=2 c$, the rise of the areh $=k, A I=y_{1}, \quad$ i $N=$ $y_{0}$, and $\mathrm{BL}=y_{2}$, we will prove that

$$
\begin{align*}
& y_{0}=\frac{5}{5} k, \\
& y_{1}=\frac{2}{15} \cdot \frac{c+5 b}{c+b} k=\frac{2}{15} \frac{1+5 n}{1+n} k, \\
& y_{3}=\frac{2}{15} \cdot \frac{c-5 b}{c-b} k=\frac{1}{15} \frac{1-5 n}{1-n} k, \tag{3.}
\end{align*}
$$

when $b=n c$.
58. Value of First Equation. - As before, the first condition may be writtell,
 - - b. If $!/ 1$ or ! becmes negative it is to be laid off below A b, but otherwise above: the figure represents !/a as negative: ambl, in the majority of eases. $y_{1}$ and !! have onposite signs. If a lime be drawn horizontally from 1, 1) li, as long as it is on the left of $y_{0}$. will be divided into a constant part $y_{1}$ : and a remainder which varies 1 ith the distance from 1 . Hence we see that

$$
\mathrm{DF}=y_{1}+\frac{y_{0}-y_{1}}{1+1} x
$$

For the right-hand member of (1.). between $A$ and ( $\mathbf{r}$, we therefore get

$$
\begin{aligned}
& \int_{0}^{a+b}\left(y_{1}+\frac{1}{1}-y_{1}+y_{1}\right) l_{2}\left(2 c x-x^{2}\right) d x=
\end{aligned}
$$

$$
\begin{align*}
& \stackrel{k}{c^{2}} y_{1}\left[r(r+b)^{2}-\frac{1}{3}(r+b)^{3}\right]+\frac{l}{c^{2}}\left(y_{0}-y_{1}\right)\left[\begin{array}{l}
2 \\
3
\end{array} c(r+b)^{2}-\frac{1}{4}(c+b)^{3}\right] . \tag{ㄹ.}
\end{align*}
$$

For the portion between ( A and B , if we write $c-b$ for $c+b$, and reckon $x$ from ls to the left, we get

$$
\text { D) } \mathfrak{F}=y_{2}+\frac{y_{0}-y_{2}}{\sqrt{n}} x,
$$

the sign of $y_{s}$ being contained in the symbol. Then the integration for the right-hand member of (1.), hetween $B$ and $G$, or between the limits 0 and $c$ — $l$, will gire, when we substitute $!$, for $!$, and $c$ - $b$ for $c+b$.

$$
\frac{k}{c_{2}^{2}} \frac{1}{2}\left[c(c-b)^{2}-\frac{1}{3}(c-b)^{3}\right]+\frac{k}{c^{2}}\left(\frac{1}{4}-\frac{1}{2}\right)\left[\frac{2}{3} r(c-b)^{2}-\frac{1}{4}(c-b)^{3}\right] . \quad \text { (3.) }
$$

The left-hand member of (1.) was shown to he, in s39. (?.),

$$
\begin{equation*}
\int_{0}^{2 c} x^{2} c^{4}\left(2 c x-x^{2}\right)^{2} d x=\frac{15}{5} x^{2} c . \tag{4.}
\end{equation*}
$$

The two portions, (2.) and (3.), of the right-hand member, heing added tocether. when the coefficients of $y_{0}, y_{1}$, and $y_{2}$ are reduced, will be equated with (t.), the left-hand member of (1.), producing

$$
\begin{aligned}
& \frac{i}{6 c^{2}}\left\{y_{0}\left(5 c^{3}-c b^{2}\right)+\frac{1}{2} y_{1}(c+b)^{2}(3 c-b)\right. \\
& \left.\quad+\frac{1}{2} y_{2}(c-b)^{2}(3 c+b)\right\}=\frac{16}{15} k^{2} c
\end{aligned}
$$

or
$2 c\left(5 c^{2}-b^{2}\right) y_{0}+(c+b)^{2}(3 c-b) y_{1}+(c-b)^{2}(3 c+b) y_{2}=\frac{64}{5} k c^{3}$. (5.)
59. Values of Second and Third Equations. - It is not necessary to integrate in order to obtain equations from the other two conditions, although they may be derived quite simply in that way. The seeond condition may be written,

$$
\geq \mathrm{EF}=\mathrm{\Sigma}(\mathrm{DE}-\mathrm{DF})=0, \text { or } \mathrm{\Sigma} \mathrm{DE}=\mathrm{EDF} .
$$

The first member is the summation of all the ordinates to the areh, or the included area between the rib and the line A B. The area of a parabolic segment being equal to two-thirds of the rectangle of the same base and altitude, the area will be $\frac{2}{3} \cdot 2 c \cdot k$ or $\frac{4}{3} c k$. The second member will be the summation of all the ordinates to the two inclined lines, or the area of the two trapezoids, giving
$\frac{1}{2}\left(y_{0}+y_{1}\right)(c+b)+\frac{1}{2}\left(y_{0}+y_{2}\right)(c-b)$, or $c y_{0}+\frac{1}{2}(c+b) y_{1}+\frac{1}{2}(c-l) y_{2}$. Equating the two values, we obtain the seeond equation,

$$
2 c y_{0}+(c+b) y_{1}+(c-b) y_{2}=\frac{8}{3} c k
$$

The condition that $\leq E F . D B=0$, or that $\leq(D E-D F)$ D $\mathrm{B}=0$, gives

$$
\Sigma \mathrm{DE} \cdot \mathrm{D} \mathrm{~B}=\Sigma \mathrm{DF} \cdot \mathrm{D} \mathrm{~B},
$$

and this condition is satisfied by the equivalent step of multiplying each area, just obtained, by the horizontal distance of its centre of gravity from one abutment, the right one for example, and equating the products. The left-hand member will then plainly be $\frac{4}{3} c k$. $c$, or $\frac{4}{3} c^{2} k$. As the seeond expression ahove for the area of the trapezoids has three terms which correspond to the three triangles formed by drawing lines from N to A and B. we may multiply each triangle by the distance of its centre of gravity from B , obtaining

$$
\begin{gathered}
\text { or, } \quad c y_{0}\left(c-\frac{1}{3} b\right)+\frac{1}{2}(c+b) y_{1}\left[r-b+\frac{2}{3}(c+b)\right]+\frac{1}{2}(c-b) y_{2} \frac{1}{3}(c-b), \\
\frac{1}{3} \in y_{0}(3 c-b)+\frac{1}{6}(c+b) y_{1}(5 c-b)+\frac{1}{6} y_{2}(c-b)^{2} .
\end{gathered}
$$

Equating the two members, and clearing of fractions, we find that

$$
\begin{equation*}
2 c(3 c-b) y_{0}+(c+b)(5 c-b) y_{1}+(c-b)^{2} y_{2}=8 c^{2} k . \tag{2.}
\end{equation*}
$$

60. Solution of Equations. - Equations (5.), § 58, and (1.) and (2.), $\$ 59$, contain the three unknown quantities. The eliminations may be performed as follows: -

Multiply (1.) by $a-b$, obtaining

$$
2 c(c-b) y_{0}+(c+b)(c-b) y_{1}+(c-b)^{2} y_{2}=\left(c^{2}-b c\right) \frac{8}{3} k .
$$

Subtract from (2.)

$$
4 c^{2} y_{0}+4 c(c+b) y_{1}=\left(2 c^{2}+b c\right)_{3}^{8} k . \quad(a .)
$$

Multiply (2.) by $3 c+b$,

$$
\begin{gathered}
2 c\left(9 c^{2}-b^{2}\right) y_{0}+(c+b)\left(15 c^{2}+2 c b-b^{2}\right) y_{1}+(c-b)^{2}(3 c+b) y_{2}= \\
\left(3 c^{3}+b c^{2}\right) 5 k:
\end{gathered}
$$

Subtract (5.), and divide the remainder by $2 c$,

$$
\begin{equation*}
4 c^{2} y_{0}+6 c(c+b) y_{1}=\left(\frac{5}{5} c^{2}+b c\right) 4 k \tag{b.}
\end{equation*}
$$

Subtract (a.),

$$
2 c(c+b) y_{1}=\left(\frac{4}{15} c^{2}+\frac{4}{3} b c\right) k . \quad \text { or } y_{1}=\frac{2}{15} \cdot \frac{c+5 b}{c+b} k .
$$

Substituting this value in ( (1.) or (b.), we get

$$
y_{0}=\frac{6}{5} k,
$$

and by analogy, or by substitution,

$$
y_{2}=\frac{2}{15} \cdot \frac{c-5 b}{c-b} k
$$

61. Remarks. - The similarity between $y_{1}$ and $y_{2}$ is to be expected; for, when a load is mored from one side of the centre to an equal distance on the other. $y_{1}$ and $y_{2}$ change places. Therefore it must be remembered that $y_{2}$ is the value of the ordinate at that springing which is nearer to the weight. If
the load is in the middle, $b=0$, and $y_{1}=y_{2}$. It is worthy of notice that $y_{0}$ is a constant quantity for all positions of the weight. These ordinates can be easily computed for a weight at different points, and it will be seen that a value of $b$ greater than $\frac{1}{5} c$ will make $y_{2}$ negative, or to be plotted below the springing line. The original reasoning showed, and the above equations will prove, that the third condition may be taken about the other abutment, and will still give the same values: for the ordinates.
62. Computation of Ordinates $y_{1}$ and $y_{2}$ - If we propose to work out data for use with this type of arch also, we must first calculate the values of $y_{1}$ and $y_{2}$ for all points. Let a rib be divided into ten parts, equal horizontally as before; then, if $b=n c$, the results of the following table will be obtained. It

$$
\begin{aligned}
& \text { Values of } y_{1} \text { and } y_{2} \text {. } \\
& n=\frac{b}{c}=\begin{array}{llllll} 
& 0 & .2 & .4 & .6 & .8
\end{array} \\
& \begin{array}{clllll}
1+5 n \\
1+n
\end{array}=\begin{array}{llll}
1 & 2.0 & 3.0 & 4.0 \\
1 & 1.2 & \overline{1.7} & 1.6 \\
& 1 & 1.8
\end{array} \\
& { }_{1}^{2} \cdot \frac{1+5 n}{1+n}=\begin{array}{llllll} 
& 0.1333 & 0.2222 & 0.2857 & 0.3333 & 0.3704 \quad k=y_{1} .
\end{array} \\
& \begin{array}{rlccccc}
\frac{1-5 n}{1-n} & = & 1 & 0 & -1.0 & -2.0 & -30 \\
0.8 & \frac{-3}{0.6} & 0.4 & 0.2 \\
1_{15}^{2} \cdot \frac{1-5 n}{1-n} & = & 0.13333 & 0 & -0.222 .2 & -0.6667 & -2.0 \quad k=y_{2 .}
\end{array}
\end{aligned}
$$

is so similar to previous ones as to call for no explanation. Only remember that $y_{1}$ and $y_{2}$ change places for loads on the left of the crown. The equilibrium polygons for one half of the arch are shown in Fig. 12.
63. Formulæ for $H, P_{1}$ and $P_{2}$ - To obtain the value of H for a particular position of the load, we lay off $y_{1}, y_{0}$, and $y_{2}$ at A, G, and B, draw I N and N L, complete the stress diagram helow, and draw $0-3$ for H. The vertical components of the abutment reactions will be 2-3 and 3-1. If we draw the hori-
zontal dotted lines from I and $L$, we shall have similar triangles to those in the stress diagram, and may write

$$
\begin{gathered}
y_{0}-y_{1}: c+b=(2-3): \mathrm{II}, \text { or } \\
\mathrm{P}_{1}=(2-3)=\mathrm{II} \frac{y_{0}-y_{1}}{c+\frac{1}{1}}=\frac{8}{15} \cdot \frac{2+n}{(1+n)^{2}} \frac{k}{c} \cdot \mathrm{II}, \\
y_{0}+\left(-y_{z}\right): c-b=(3-1): \mathrm{II}, \text { or } \\
\mathrm{P}_{2}=\mathrm{W}-(2-3)=\mathrm{II} \frac{y_{0}-y_{2}}{c-b}=\frac{8}{15} \cdot \frac{2-n}{(1-n)^{2} c} \cdot \mathrm{H} .
\end{gathered}
$$

Substitute the value of (2-3) from the first equation, transpose, and obtain

$$
\mathrm{II}=\frac{\mathrm{W}}{\frac{y_{0}-y_{1}}{c+h}+\frac{y_{0}-\frac{y_{2}}{c-b}}{c-\frac{1}{3}}}=\frac{\left(c^{2}-b^{2}\right)^{2}}{c^{3} k} \cdot \mathrm{~W}=\frac{15}{3} \frac{n^{2}}{\left(1-n^{2}\right)^{2} \frac{c}{k} \mathrm{~W} .}
$$

64. Computation of Values. - The amount of II for a load at any one point will then be found in the several columns of the table below. The first three values will be seen to be

| Values of il, $\mathrm{P}_{1}$, ANid $\mathrm{P}^{\prime}$. |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $n=$ | 0 | .2 | . 4 | . 6 | . 8 |
| $1-n^{2}=$ | 1 | . 96 | . 81 | . 64 | . 36 |
| $\left(1-n^{2}\right)^{2}=$ | 1 | . 9216 | . 70.6 | . 4096 | . 1296 |
| H $=$ | . 1687 | .4320 | . 3308 | . 1920 | .0607 |


| $\left.\mathrm{H} \begin{array}{lllll}\frac{y_{0}-y_{1}}{(1+n) c}= & 0.5 & 0.352 & 0.216 & 0.104 \\ 0.02 s & W \\ \text { II } \frac{y_{0}-y_{2}}{(1-n) c}= & 0.5 & 0.645 & 0.78 t & 0.896 \\ 0.972 & W\end{array}\right\}=P$. |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | greater, and the last two to be smaller, than the corresponding $H$ 's in $\S 41$. It will next he necessary to find the vertical components of the reactions by multiplying II by the quantities noted in the last section: the results will be formel in the last two lines. The larger value of P occurs at the nearer abutment. It will be noted that these quantities differ in amount from the two supporting forces of a single--pan heam or truss.

If the II's for an equal load at each of the nime points of ${ }^{\circ}$ division are added together, we find that. for loads at all points.
$\mathrm{H}=2.4997 \frac{c}{h} \mathrm{~W}$, which agrees more closely with the amount for a truss or bowstring girder than did the ralue for a rib with hinged ends, $\S 42$. It is due to the fact that the equilibrium polygon for a single weight crosses the rib oftener in the present case than in that of a rib with hinged ends; so that, when several loads are combined, the polygon will deviate from the parabola (the form of the rib, and the true equilibrium curve for a uniform distributed load) very little.
6.). Computation of Bending Moments. - If, in place of scaling, we desire to compute the values of $M$ in this case also, we may use the former equation, S 43 .

$$
\mathrm{M}=\mathrm{II}(y-z)
$$

The ralues of the ordinates, $z$, to the parabola will be the same as before. If $x$ denotes the distance from A to the foot of the ordinate $y$, and $x^{\prime}=$ the distance from B to the foot of the same ordinate, in which case $x^{\prime}=2 c-x$, we shall have

$$
\begin{aligned}
& y=y_{1}+\frac{y_{0}-y_{1}}{c+b} x, \text { on the left of the weight, and } \\
& y=y_{2}+\frac{y_{0}-y_{2}}{c-b} x \text {, on the right of the weight, }
\end{aligned}
$$

the sign of $y_{2}$ being contained in the symbol.
Let us proceed to find the values of $M$, at both abutments and the nine other points, for a weight on the third point of division from the middle, towards the right. As above,

$$
\begin{gathered}
\mathrm{I}=0.192 \frac{c}{k} \mathrm{~W} ; \frac{y_{0}-y_{1}}{c+b}=0.5417 \frac{k}{c} ; \quad \frac{y_{0}-y_{2}}{c-b}=4.6667 \frac{k}{c} ; \\
z=.36 k, .64 k, .84 k, .96 k, k, .96 k ; \& \mathrm{c} ., \S 43 . \\
\text { VALUES of } \mathrm{M} .
\end{gathered}
$$



IV is placed over the number of the point to which it is allacherl, atul it double line is drawn on one side of W to denote the end of each series. running from the two ends of the table. The dividing line might just as well have been drawn on the left of $\mathbb{W}$, if preferred. More frequent valnes of any of the preceding quantities may be obtained by interpolation, as exphained before.
66. Table of Bending Moments. - A table of values of M has heen prepared for this ease of an arch with fixed ends, the span being divided into ten equal parts, mol is here presented. 1. 71. A table for twenty divisions may be found in "Engineering News," vol. iv., p. 178. At any one point, for at uniform load at all of the points of division, M reduces nearly to zero, as before. The greatest possible positive M, as well as the greatest possible negative M, for any combination of weights. occurs at each abutment; positive maximum when the span is loaded from the other abutment to and beyond the centre onn point; negative when the other portion only of the span is eovered. The load on the finst point from the midille prodnces no MI at the nearer abutment. 'There is another maximum at the third or seventh point, with loads nearly the reverse of the ones mentioned abore. An inspection of the table will show these facts.
67. Example. - As soon as H, P, $y_{1}$, and $y_{2}$ have heen obtained for all points, it is easy to draw an equilibrium polygon for any desired arrangement of load. Let us suppose that onfe must he constructed for weights of 2 tons, 6 tons, 3 tons, and 1 ton, on the $2 \mathrm{dl}, 4$ th, 5 th, and 8 th points respectivels, fiom the left abotment, of an arch of 100 feet span and 20 leet rise, Fig. 13, divided into ten equal parts along the span, as previonsly deseribed. We will proceed as follows: -

The vertical components of the reactions cannot he computerl for the load in the gross, as for a bean on two supports, but must be summed up from the values lately given. Referring to those data, we get
$\mathrm{P}_{1}$. H .

| 2d joint, |  | $\mathrm{P}_{1}$. |  |  | H. |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $0.896 \times 2=1.792$ tons. |  |  | $0.192 \times 2=0.384 \frac{c}{k}$ tons. |  |
| 4 th | " | 0.648 | $\times{ }^{4}=3885$ | " | $0.432 \times 6=2.592$ | ، |
| 5 th | " | 0.5 | $\times: 3=1.500$ | " | $0.449 \times 3=1.407$ | " |
| 8th | " | 0.104 | $\times 1=0.104$ | " | $0.192 \times 1=0.192$ | " |
|  |  |  | $\mathrm{P}_{1}=7.28 t$ | " | $\mathrm{II}=4.575$ | " |

$\mathrm{P}_{2}=12-7.284=4.716$ tons. $\quad \mathrm{II}=4.575 \times 2.5=11.44$ tons.
Since II $y_{1}=$ moment at the springing 1 , Fig. 13: since each of these loads has a separate $H$ and a definite $y_{1}$ : and since the IIs for the different loads all conspire to produce the total thrust, - we must ealculate the arm with which the latter acts at one or both springings, that is, the ordinate $y_{1}^{\prime}$ or $y_{2}^{\prime}$ of the point whence the equilibrium polygon must start. We satisfy the equation

$$
y_{1}^{\prime} \cdot \Sigma H=\Sigma H \cdot y_{1}, \text { or } y_{1}^{\prime}=\frac{\Sigma H \cdot y_{1}}{\Sigma H^{\prime}}
$$

which simply requires that the resultant moment shall be equal to the algebraic sum of the original moments. We therefore multiply each $H$ for a given weight by its $y_{1}$, and divide the sum of the products by the total H . The calculation having been made, as here set down, we find that $y_{1}^{\prime}$ is equal to - 0.2 feet, a comparatively insignificant amount. It is well to compute $y_{2}^{\prime}$ also. as a check on the accuracy of the subsequent drawing, and it will be found to be +3.34 feet.

$$
\begin{aligned}
y_{1} \cdot \mathrm{II.} & \mathrm{M} . \\
-.667 \times 0.384= & -0.256 c \text { tons. } \\
0 \times 2.592= & 0 \\
+.133 \times 1.407= & +0.188 \quad " \\
+.333 \times \frac{0.192}{4.575}= & +\frac{0.064}{-0.004} \quad " \\
& -0.0009 \mathrm{k} \\
& \frac{20}{y_{1}^{\prime}}= \\
& \frac{-0.018 \text { feet. }}{}
\end{aligned}
$$

While we may seem to have carried out this example in too much detail, we are aware that inattention to apmarently trivial points will sometimes callse trouble, and we have thorefore given most of the work at full length. Now lay off the weights in order on the load line. phot $P_{1}$ and $P_{2}$, lay off II on the proper side, draw the nsinal radiating lines to the extremity of 11 , start below $A$, a distance $-y_{1}^{\prime}$, and draw the efuilibriun polygon with sides parallel to the inclined lines of the stress diagram. checking the polygon by the fact that it strikes the extremity of the calculated ordinate $y_{2}^{\prime}$. Fig. 13 illustrates this example. The diagram for vertical shear is also shown below, and needs no explanation, as the construction is similar to previons cases. The dotted lines in the stress diagran determine the value of $Y_{1}$. It is 'unite noticeable in this figure, how the shear changes sign wherever the bending moment becomes a maximum.
68. Table of Shear. - To find the numerical value of the vertical shear, from which we may derive the normal components resisted by the braces of an arch with fixed ends, we proceed as we did in the case of an arch with hinged ends. The values of $P_{1}$. the vertical component of the abutment reaction at the left, have been found. We then need only calculate the value of $Y_{1}=2 \frac{k}{c} \mathrm{H}$, and form a table, as was done in $\S 51$. It is not necessary to repeat the operations here. A table of shears for an arch with fixed ends, and for ten divisions, has been prepared, and is appended. p. 70. The same remarks apply to it as to the previous similar table for the parabolic arch with hinged ends. For a table for twenty divisions. see Engineering News," vol. iv.. p. 193.

## 69. Extent of Load to produce Maximum M and F. -

 A diagram is also presented. Fig. 15, showing. ly the full lines, the loads recuired to produce the maximm + M. from live load, at the point whose number is attached to the line, and by the remaining hank portion the load required for maximm -M at the same point. The broken lines and the blank bortion in each spues represent the way of distributing the load for maximum +F and -F respectively. It is still more apparent from this figure than from Fig. 11, that any investigation which considers the rolling load as continuous from onePARABOIAC Lim, Fixed at ENDS.
sitic. $\quad \mathrm{l}=\mathrm{me}$ C $W$. Values of $m$.

|  | 0 | 1 | 2 | : | 4 | 5 | 6 | 7 | S | 9 | 10 | 1I |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| W on 9 | +.022 | $+.006$ | -.005 | -.01: | -.01:3 | -. 110 | -.0022 | +.011 | +. 1208 | +.051 | -.121 | . 001 | W. |
| " 8 | $+.064$ | $+.016$ | -. 017 | - 0:35 | -.0:37 | -. $0: 1$ | +.001 | +.0.48 | +.107 | -.01s | -.1:8 | . 13 (2) |  |
| " 7 | +.019.5 | --.019 | -. $0: 31$ | -.0.5) | -.050 | -.020 | +. $0: 30$ | +.11! | -.02s | -.0.36 | -.07: | . $: 3131$ | " |
| " 6 | +.6996 | +.011 | -.010 | -.050 | -.0:37 | +.011i | -. 101 | +.01:2i | -. 017 | -. 10.24 | ${ }^{1}$ | . 493 | " |
| " | +.06\% | -.000; | -.01:3 | -.0:31 | +.012 | -.0901 | -.01: | -. $0: 31$ | -.0:37 | -.001; | +.1193 | . 169 | ${ }^{6}$ |
| " 4 | 0 | -.020 | -. 017 | +.020 | +.104 | +.016 | -.0.37 | -. 050 | 一.010 | +.011 | +.0996 | .4is) | " |
| " 3 | -. 07.3 | -. $0: 311$ | +.028 | +.11! | +.11:36 | -.0:20 | $-.0 .0 .0$ | -.0.1) | -.0.:11 | +.01! | +.095 | . $3: 31$ | " |
| " | $-125$ | -.01s | $+.107$ | +.048 | +.004 | -. 021 | -.01:37 | -.0.3.5 | $-.017$ | -.014 | +.06) | .1912 | " |
| " 1 | $-.121$ | $+0.51$ | +.028 | $+.011$ | $-.019$ | -.011 | -..01: | -.012 | -.010.) | +.00\% | +.0\% | . 0161 | " |

$\equiv$
 $=$
 +i i i i i i i i $x$
 $++i 1 i+i+i$

1-
 $+1+i \mid i i+i$ $\because$
 L:
 $-$


abutment over a portion of the span will mot detemmine acthat maximum stresses. See $\widehat{5} t$.
70. Comparison of Ribs; Fixed and Hinged at Abutments. - 1 comphrison of Fig. 1is with Fig. 11 will ho instructive, as showing the different loating, when hinges are omitted, to produce maximum londing moments and farss. There are fom points near the emds of the riln with tixed ends, which require that luads should he on both ends of the span at once, tor podnce the maximmm +I at those joints: and five points at the middle which have the maximm - I Imader similar circumstances. In some structures such conditions can be realized. If we foot 1 l the plus and minns values of the columns in the tables for $M$ and $V$, we shall rearlily see that, with the exerption of the springing points, all the points in the arch with fixed ends have less maximum bemding monents of either kind, for a load $W$ at each loaded point, than in the ease of the areh with hinged ends, and, in most cases, the values are materially less. A similar comparison of maximum sheas will show that the areh with fixed ends has to earr more shear over its web or bracing for all the divisions of the first and last quarters of the span, amd less for the midelle half of the span, than an arch with hinged ends. These eonsiderations alone Would indicate the superiority of the areh with fixed ends over the other type as reguiring less material in the flanges on chords, and throwing the heavier bacing towards the abontments: the value of the dired throst. however, as imbleaterl by the previonsly computed amomats of Il, varies aceorting to the amomat of load, amel complites with the compmesion fom bending moment, so that the sertions of the two chords mast be designed for the maximmm compression and tension at all points: the effect of rise or fall of temperature will be shown to he greater on the rib) with fixed ends, requring a greater increase of section to powide for it.

## CHAPTER V.

## CHANGE OF TEMPERATURE.

71. Action of Change of Temperature. - If the arch, when either fixed or linged at the ends, is exposed to a change of temperature it will tend to change its shape. If the rib were perfectly free, its expansion or contraction would be uniform in all directions, so that the new areh would be the old arch on a slightly altered scale. In a bowstring girder, the tie expands and contracts with the bow, so that the horizontal projection of the change of length of the bow is the same as the elongation or contraction of the horizontal member. But as the abutments of the arch are considered as fixed, its span most remain machanged: and the alteration of the arch bey a change of temperature will be manifested by a rise or fall of the crown of the arch. which movement, in the case of a metal rib, may lee a marked puantity.

It is manifest, that, if we imagine the rib at its nomal temperature to be placed upon its springing points or skewbacks, it will have a horizontal thrust against the ahmoments due to its form and weight. If the temperature changes. the structure endeators to expand or contract in equal proportion in all directions: : and hence if possible the span would be lengthened just in proportion to the rise of temperature $t$, the coefficient of expansion $e$, and the span $2 c$, or the change of span would equal $2 t a c$. If $t$ expresses the number of degrees of fall in temperature it may be called mimus, and the quantity $2 t \rho \subset$
will denote the shortening of the span. But this attempted change of length, being resisted at the points of attachment, cannot take place, but must callise a horizontal force, either tension or compression, which keeps the span invariable. This +11 or - H mast exert a bending moment upon all parts of the rib, as well as a direct thrust, which moment is too inmportant to be neglected. It being reeollected that the condition ミEF.DE $=0$ denoted that the change of span equalled zero, it will be sufficient in this case to still make it zero, when we have added or subtracted a cquantity proportional to $2 t$ e $e$.
72. Change of Span influenced by Material and Crosssection of Arch. - The bending moment II at any point has
 the stress diagram multiplied by the vertical ordinate from that point to the equilibrium polygon. Then it was shown, ş 18 , that, if all these ordinates were summed up, that is, if we took $\leq \mathrm{E} F$ between two points, this sum would be proportional to the ehange of inclination between those two points: but it was not stated that this quantity was equal to the change of inclination, for neither the material nor the form of cross-section of the rib was taken into account. As the amount of flexure was stated, in P'art II., "Bridges." \$s 85 and 86, to vary inversely as the modulus of elastieity and the moment of inertia, we
 equal the change of inclination. The same thing is true of the expressions for deflection and ehange of span. When, however. the summation is made from one alsutment to the other, and then put equal to zero, if $\mathbf{E}$ aml $\mathbf{I}$ are constant, as well as H. it must be true that $\leq \mathrm{EF}=0$, as heretofore stated; and likewise of the other equations. Now $\mathbf{E}$ is constant, as the material of the rib is the same throughout: and since the parabolic rib, of cross-section rarying with the secant of the inclination of the rib to the horizon, hats been demonstrated. $\leqslant 36$. to deflect vertically like a straight beam of uniform section ectual to that of the ril) at the crown, $\mathbf{I}$ is likewise constant in these formule,
and represents the moment of inertia of the seetion at the crown. In short, where one quantity is directly proportional to another, if one is equal to zero, the other is also : consequently We can deal with areas, area moments, de., as if they were the changes of inclination, deflections, \&e., themselves.
7.). Formula for H from Change of Temperature. Bnt now we wish to introduce the distance $2 t b c$, the change of span which wonld ocem from change of temperature, were it unchecked. As this is an absolnte and not a proportional quantity, we must divide our original quantity for clange of span, § 7, by EI. We shall, therefore, have for the new eondition,

$$
\frac{\mathrm{H}_{t} \cdot \mathrm{\Sigma EF} \cdot \mathrm{DE}}{\mathrm{EI}} \pm 2 t e c=0
$$

where $\mathrm{H}_{t}$ is used to signify the horizontal force (thrust or tension) which is occasioned by the change of temperatme ; or, if we clear of fractions, we get the more convenient expression

$$
\mathrm{H}_{t} \cdot \Sigma \mathrm{EF} \cdot \mathrm{I} \mathrm{E} \pm 2 \mathbf{E} \mathbf{I} t e c=0 .
$$

A rise of temperature will make H a thrust or positive, while a fall of temperature will make II a tension or negative. The donble sign is not needed in the above equation if the sign is contained in the symbol $t$, that is, if $t$ is negative for a diminntion of temperature below the one at which the rib is constructed or laid out. The bending moments exerted on the ij) will be of the contrary kind when $\mathrm{II}_{t}$ is minus, while the ordinates are muchangerl.
74. Application to Parabolic Rib, Hinged at Ends. To take up first the case of the parabolic rib hinged at ends. The amoment of $\mathrm{I}_{t}$ is to be determined. As there can be no bending moment at either abutment, and $\mathrm{H}_{t}$ at each aboutment is the only applied force, the equilibriun polygon or line of thrust, Fig. 16, must be in the line joining the two springings. The bending moment at any point will, therefore, be equal to the ordinate to the rib at that point, multiplied by the desired value of $H_{i}$. The expression $亡 \mathrm{EF}$. D E therefore becomes for
this case $\leq \mathrm{D} \mathrm{E}^{2}$; and we have, transposing the second term of the equation of the previous section,

$$
\mathrm{H}_{t} \cdot \Sigma \mathrm{D} \mathrm{E}^{2}=2 \mathbf{E} \mathbf{I} \text { tec. }
$$

The value of $\Sigma \mathrm{D} \mathrm{E}^{2}$ was shown in $\$ 39$ (2.), to be $\frac{16}{15} k^{2} c$; therefore, substituting and transposing, we see that

$$
\mathrm{H}_{t}=\frac{15}{8} \cdot \frac{10 \mathbf{E} \mathbf{I}}{k^{2}}
$$

a value which is indepentlent of the span.
The maximum bending moment. which oceurs at the middle of the span, where the ordinate will be $k$, is

$$
\mathrm{M}(\max .)=\frac{15}{8} \cdot \frac{1 e \mathbf{E} \mathbf{I}}{h} .
$$

The ordinates at all the usual points of division will be the values of $z$, used repeatedly before : and, hy multiplying $\mathrm{I}_{t}$ by these several values of $z$, the hembing moments at all points wre obtaned for a given change of temperature $t$. An additional line ean be placed below the table of M to contan these prantities, so as to have them comvenient for use. All of these moments will be positive for a fall of temperature below. and negative for a rise above, that at which the rib was designed. The worst effect of either change must be provided for.
75. Formula for Change of Span deduced analytically. - If one likes to prove this value for change of span analytically. he may proceed as follows: Let any ordinate to the arch be denoted by $\%$. and the abscissa measured horizontally from one ahoutment by $r$. Then. if $r=$ the rertical deflection ordinate, that is. the deflection of any point from its original position, we may write the usual equations for currature, slope, and deflection of beams, recollecting that this arch acts like a bean of uniform section in deflecting vertically.

$$
\frac{d^{2} v}{d x^{2}}=\frac{\mathrm{M}}{\mathbf{E} \mathbf{I}} ; \frac{d c}{d x}=\int_{\mathbf{E} \mathbf{I}}^{\mathrm{M}} d x ; \text { and } x=\iint \frac{\mathrm{I}}{\mathbf{E} \mathbf{I}} d x^{s}
$$

Now $\mathrm{M}=\mathrm{H} y=\mathrm{II}{ }_{c^{2}}^{\text {lo }}\left(\underline{2} c x-x^{2}\right)$; therefore

$$
\frac{d v}{d x}=\frac{\mathrm{H}}{\mathbf{E I I}} \cdot \frac{k}{c^{2}} \int\left(2 c x-x^{2}\right) d x=\frac{\mathrm{II}}{\mathbf{E} \mathbf{I}} \cdot \frac{k}{c^{2}}\left(c x^{2}-\frac{x^{3}}{3}+\mathbf{C}\right) .
$$

$\frac{d v}{d x}=0$, for $x=c ;$ therefore $\mathbf{C}=-\frac{2}{3} c^{3} . \quad$ Then

$$
\frac{d v}{d x}=\frac{\mathrm{H}}{\mathrm{E} \mathbf{I}} \cdot \frac{k}{c^{3}}\left(c x^{2}-\frac{1}{3} x^{3}-\frac{2}{3} c^{3}\right)
$$

If $u=$ horizontal displacement of any point, the infinitesimal horizontal displacement $d u$, due to the movement of the portion of arc $d s$, will give, as may be seen to the right of Fig. 16,

$$
d u: d v=d y: d x
$$

Since $y=\frac{k}{c^{2}}\left(2 c x-x^{2}\right), d y=\frac{k}{c^{2}}(2 c-2 x) d x$, and we have

$$
d u=\frac{2 k}{c^{2}}(c-x) d v
$$

Substitute the value of $d v$ from (a.), and it becomes

$$
d u=\frac{\mathbf{H}}{\mathbf{E} \mathbf{I}} \cdot \frac{2 k^{2}}{c^{4}}\left(c^{2} x^{2}-\frac{4}{3} c x^{3}-\frac{2}{3} c^{4}+\frac{1}{3} x^{4}+\frac{2}{3} c^{3} x\right) d x .
$$

If this equation is integrated between the limits 0 and $2 c$, we obtain $u=-\frac{\mathrm{H}}{\mathbf{E} \mathbf{I}} \cdot \frac{16}{1} \frac{6}{5} h^{2} c$, which will be seen to correspond with the value of $2 t e c$ in the preceding section.
76. Application to Fixed Parabolic Rib. - If we turn next to the rib with fixed ends, it will be manifest, that, since there will be bending moments at the springings, the line which corresponds to the equilibrium polygon and limits the ordinates for bending moments cannot now pass through those points. As the resistance to expansion or contraction is the only cause of those moments, the two abutment moments will be equal. and the line will be horizontal. In order also to satisfy the condition that the change of inclination at the abutments shall equal zero, or, as expressed in $\S 18, \leq \mathrm{EF}=0$, the horizontal line must be so drawn as to make the areas within and without the arch equal to one another, which will occur when the line is drawn at a height of $\frac{2}{3} k$ above the springing, as seen in Fig. 17. To prove the equality of areas it is only necessary to recall the fact that the area of a parabolic segment equals twothirds of the enclusing rectangle. The area included within the
whole arch will therefore be $\frac{2}{3} k .2 \pi=\frac{4}{3} k c$. The rectangle of height $\frac{2}{3} k$ has the same area. Therefore the portions of the arch area and of the rectangle which do not conneide must be equal to one another. The third comblition, of S. 19, that $\leq E F . D B=0$, or the equality of area moments, is also satisfled by this construction; for the rectangle multiplied by the half span, which is the distance of its centre of gravity from one abotment, is equal to the area included by the whole arch multiplied by the same distance.

Tor deduce in this case the value of $H_{t}$ : as before,

$$
\mathrm{H}_{t} . \Sigma \mathbf{E F} \cdot \mathrm{DE} \pm 2 \mathbf{E} \mathbf{I} t e c=0
$$

From what has just been stated,

$$
\begin{equation*}
\left.\left.\Sigma \mathrm{EF} . \mathrm{D}) \mathrm{E}=\Sigma\left(\mathrm{DE}-\frac{2}{3} k\right) \mathrm{DE}=\Sigma \mathrm{E}\right) \mathrm{E}^{2}-\frac{2}{3} k . \Sigma \mathrm{I}\right) \mathrm{E} . \tag{2.}
\end{equation*}
$$

The first term, as before, amomots to $\frac{16}{15} / k^{2} \cdot a$ since $\left.\Sigma \mathrm{D}\right) \mathrm{E}=$ area enclosed by the arch, $=\frac{4}{3} k r$, the second term is ${ }_{9}^{9} k k^{2} c$; therefore

$$
\mathrm{H}_{t} \cdot \frac{8}{45} \mathrm{l}^{2} c=2 \mathbf{E} \mathbf{I} t e c \text {, or } \mathrm{H}_{t}=\frac{4.5}{4} \frac{t e \mathbf{E I}}{\mathrm{l}^{2}} .
$$

The bending moment at the crown will therefore be

$$
\mathrm{M}=\mathrm{H}_{t} \cdot \frac{1}{3} k=\frac{1_{5}}{4} \cdot \frac{k \mathbf{E I}}{k}
$$

and at the springing,

$$
\mathbf{M}=\mathbf{H}_{t} \cdot \frac{2}{3} k=\frac{15}{2} \cdot \frac{t e \mathbf{E I}}{k},
$$

or double the former amount, but of the opposite kind. Whether the bending moment at either point is positive or negative, depends upon whether $H_{t}$ is tension on compression. These moments also can be conveniently added to the proper table for M, as explained for the first case.
77. Comparison of Arches under Change of Temperature. - The bending moments for temperature, in both the areh with hinged ends and that with fixed ends, will vary like those of a beam uniformly loaded, and cither simply supported or fixed at the ends. Part M., " Bridges," ss 95, 99.

It may be well to notice the comparative straining effect of the same change of temperature in the two classes of parabolic arches, for ribs of the same rise. II, is six times as great when the arch is fixed as when it is hinged at the ends, and the direct stress in the ribs will therefore vary in the same proportion. The maximum moment, at the springing, for the rib with fixed ends, is four times as great as at the crown of the rib with hinged ends. and of the opposite kind ; while the value of M at the two crowns is as two to one against the ribl with fixed ends.
78. Shear from Change of Temperature. - The shear on a right section can be shown by the accompanying Fig. 18. If a $l$ represents the amomat of H caused by a change of temperature, we may draw ad and $b c$ parallel to the upper and lower flange at any right section $S$ of the rib, when "a will be the value of the direct stress at the section, one-half in each flange, and $b e$ will be the shear.* The bending moment will have any magnitude, depending upon the length of the ordinate from the equilibrimn line to the point on the centre line of the arch where this section is taken. As $a e$ and $g b$ are parallel, the perpendicular distance $b e,=c d$, between them is constant, so that $f^{\prime} d$ may be taken, for our purpose, to represent the stress in one chord, and $y$ s that in the other due to bending moment, the resultant stresses being $a d$ and $c l$, while the shear on the right of a right section of the web will be $d r$. Since the resultant stress at any rection must be H, the directions of the forces. shown ly the arows, in this closed polygon, are at once fixed. As the inclination of the arch changes, the value of $c d$ will change, heing zero at the crown and a maximum at the springings. The arrows denote the case where II is a thrust. The bending moment will be negative, if the rib) is hinged at the ends, the bottom chord will be compressed, the top chord will have a force exerted upon it amounting to the difference between the direct thrust and the tension due to the moment, and consequently of will be the stress exerted by the top chord against the riglit side of the cross-section in the accompanying sketeh.
79. Diagram for Vertical Shear. - Let us suppose a fall of

[^0]temperature do take place: the rib will have a temblency to eome down at the reown. Wre reeall the fact that a milorm load hats a pamalona for its expilibriun cemve, and a load of the proper intensity on any paraholic: arch will protuce the value oil II whirh is now smpposed to exist. It is evident, then, as is alsa shown hy the sign of M, that the rib may le imagined to be loaded miformly horizontally with a weight sufticient to produce this deflection or these values of M . This imaginary weight will be just sufficient at all points to balance the com fonent of an opposite kind which is reguired in combination with the value of $\mathrm{H}_{t}$ (in this case a lomizontal temsion), in order to give a resultant stress in the direction of the tangent to the rib. Aurl, further, if this weight were not just suffirient to balance the alowe component, a remamaler, of one sign or the otler, would be found at the abutments, as a vertical component of the reaction there : but we know that no such rertical component exists. If a bent spring is placed with its two ends on a horizontal line, and compression or temsion is applied in that line, no rertical force is needed tor equilibrimm. As the miform weight was entirely imaginary, the vertical eomponents mast be supplied by the web and flanges, and hence we conclurle that the diagram for eretiod shear in the areh affected hy a rhange of temperatmre, will be that of an ordiany trus. supported or fixed at its two ends, and carrying a eomplete miform loarl. amd that the normal component will be caried by the weh). For a fall of temperature therefore the shear on a rertical section will be of the same kind as, and, for a rise of temperature, will be of the opposite kind to, that produced by a load on a truss with horizontal chords.

## CHAPTER VI.

## CIRCULAR RIB WITH HINGED ENDS.

80. Circular Rib to be of Uniform Section. - Passing next to the consideration of the arch whose curve is the are of a circle, we shall assume that the rib is of miform section, and not, as before, of increasing breadth from the crown to the springing. As the rib is of uniform section, it can no longer be compared to a horizontal beam, as regards its change of inclination and deflection under bending moments, and the length along the arcl, instead of its projection on a horizontal line, must he used in spacing off and in summing up the usial quantities: that is, the sum of the changes of inclination between any two points will be made up from the change of inclination at each successive point clony the rib. We must therefore use $d s$ for $d x$ in our integration, where $s$ denotes the length of an are; and polar co-ordinates will, in the more complex cases, be used in place of rectangular ones. In spacing off the rill, for egual divisions, or for summing the ordinates arithmetically, the measurements will be marle along the curve, and cach division will subtend the same angle at the centre of the circle.

We stated, it will be remembered, that a segmental arch of the circular type, if the rise did not exceed one-tenth of the span. might, without serions error, be treated as if it were parabolic. In discussing circular arches, there will be so many points similar to those we have already explained, that we shall
not go into muth detail on some points, but leave the reader to make the extended application as examples come up in his own practice.
81. Experimental Verification. - The values to be obtained for $y_{0}$ for a rib) of miform section, curved to the are of a cirele, and hinged or free to tum at the ends, can be readily verified or illnstrated experimentally as follows:-Take a piece of morlerately stiff iron wire, and bend it acemately into the desired shape, I (B, Fig. 19; suspend the wire from a horizontal bar E F by means of strings fastened at 1 and B . and then attach a weight at any point $C$. It will be convenient to streteh a thread fiom I to 1 . whieh, as the san is to be muchanged, will not interfere with the reactions. If the point E is now moved horizontally, the length of the string E I being at the sanc time changed, the line I $B$ ean be browght parallel with E, F, as can be readily aseertaned with a scale. Then EA and $F$ B prolonged will meet at $I)$ on ( ${ }^{\circ}$ ), and D (i will equal $y_{0}$. E I and F B will actually intersect on the vertital through the centre of gravity of the wire and weight combined: but if the weight of the wire is as small as is comsistent with stiffues. while the weight at ( C is large in eomparison, the centre of gravity will practically be in C D. If A B becomes slatek. it shows that E and F are not suffieiently far apart. By fastening two long threads independently to E and F , the lines E I and FF F an be easily prolonged to an intersection.

## 82. Semicircular Arch with Hinged Ends; Value of $y_{0}$.

- If the rih with hinged ends is first taken up for diseussion, the value of $y_{0}$ for a load at any point on a semicireular areh is easily obtained by a simple device. Recurring again to the nsual formula in its modified form. We must satisfy the condition

$$
\mathrm{\Sigma} 1) \mathrm{E}^{2}=\mathrm{E} \mathrm{D} \mathrm{E} . \mathrm{DF} .
$$

If we let I) E, Fig. $20,=z: \mathrm{I} \mathrm{F}=y: \perp \mathrm{D}=x$; and represent a small portion of are by $d s$, this equation becomes, for the entire semieircle,

$$
\int_{0}^{\pi} z^{2} d s=\int_{0}^{\pi} y z d s
$$

If we draw a radins fiom any point E of the rib to the centre O, and also draw the infinitesimal triangle whose sides are $d s$, $d x$, and $d z$, we shall have, from similarity of triangles,

$$
r: z=d s: d x \text {, or } z d s=r d x ;
$$

substituting this value in the above equation, we get

$$
r \int_{0}^{2 c} z d x=r \int_{11}^{2 c} y d x .
$$

The integral of $z d . x$ between the given limits is the area of the semicircle, while that of $y d x$ is the triangle A C B. Substitute the valuc ${ }_{2}^{1} \pi r^{2}$ for the former, and $r y_{0}$ for the latter, and we obtain

$$
\frac{1}{2} \pi r^{3}=r^{2} y_{0} ; \text { or } y_{0}=\frac{1}{2} \pi r=1.570 \mathrm{~s} r .
$$

The ordinate $y_{0}$, for a load at any point, on a semicirenlar rib with hinged ends, is therefore a constant quantity, equal to the length of the half rib. If we draw a horizontal line at this height alove the springing, it will contain the vertices of all the equilibrim polygons for single loads.
83. Segmental Arch; Value of $y_{0}$. - If the arch is segmental. that is, less than a semicirele, we shall use the following motation: Let the angle NOB, Fig. 21, subtented at the centre of the circle by the half arch, be denoted by $p$ : the angle N (ol, from the crown to the point where the weight is placed, be denoted hyr e: and the angle N O E from the crown to any point where the ordinates 1) E and E F are measured, be $\theta$. The radius of the areh $=r . \quad$ If, then, ACB is the desired curve of equilibrium, $\boldsymbol{C}: \boldsymbol{K}=y_{0}$. The value of this ordinate will be proved to be

$$
\left.y_{0}=r \frac{\left(\sin ^{2} \beta-\sin ^{2} a\right)\left(\beta 1+2 \cos ^{2} \beta\right.}{\sin \beta}-3 \cos \beta\right) .
$$

If the arch is a semicircle, $\beta=90^{\circ}=\underset{-2}{1} \pi$, and this value reduces to $y_{0}=\frac{1}{2} \pi \rho$, aseviously obtamed.

The work of computing $y_{0}$ for different values of 4 is not great; as, for a given arch, $\beta$ is constant, and the second factor
of the numerator is a constant quantity. Since a segmental arch may subtend any angle, it is not worth while to go into the eomputation here of values of $y_{0}$ for a given value of $\beta$; but, as examples of $y_{0}$, we will give

All that one needs for the calculation from this formula is an ordinary tallle of natural sines and cosines. The angles or ares $\beta^{\text {and }}$ a are to be expressed in lengths of are, which subtend the given number of degrees, to radius mity. The are for one degree being $\frac{\pi}{180}$, or $0.017+53$, any other are will be whtained by multiplying this quantity by the number of degrees which the are sulbtends, minutes being expressed as a decimat part of a degree.
st. Proof. - From Fig. -21 we have I $\mathrm{E}=r(\cos \theta-\cos ; 3)$.
DF: $\mathrm{C} K=\lambda \mathrm{I}): \AA K=r(\sin \beta+\sin \theta): r(\sin 3+\sin \alpha)$
on the left of K , or 1 ) $\mathrm{F}=\frac{\sin \beta+\sin \theta}{\sin \beta+\sin a} y_{0}$;
on the right of $\mathrm{K}, \quad \mathrm{D} F=\frac{\sin 3-\sin \theta}{\sin 3-\sin a} y_{0}$.
 we ohtain for the first member of the equation, remembering to use $d s=r d y$ in place of $d x$, and considering angles to the leit of $O N$ as negative.

$$
\begin{gathered}
r^{3} \int_{-\beta}^{+3}(\cos \theta-\cos 3)^{2} d \theta=r^{3} \int_{-3}^{-3}\left(\cos ^{2} \theta-2 \cos 3 \cos \theta+\cos 2,3\right) d \theta^{*} \\
=r^{3}\left(\beta+23 \cos ^{2} 3-3 \sin 3 \cos , 3\right) . \quad(11 .)
\end{gathered}
$$

For the integral of the second member between $a$ and - 3 we have

$$
\frac{r^{2} y_{0}}{\sin \beta+\sin a} \int_{-\beta}^{a}(\sin 3 \cos \theta+\sin \theta \cos \theta-\sin 3 \cos \beta-\cos \beta \sin \theta) d \theta \dagger
$$

$* \int \cos ^{2} \theta d \theta=\frac{1}{2}(\theta+\sin \theta \cos \theta) ; \cos -\beta=\cos \beta: \sin -\beta=-\sin \beta$.

$$
\dagger \int \sin \theta \cos \theta d \theta=-\frac{1}{2} \cos ^{2} \theta .
$$

$$
\begin{aligned}
& =\frac{r^{2} y_{0}}{\sin \beta+\sin a}\left(\sin \alpha \sin \beta-\frac{1}{2} \cos ^{2} \alpha-a \sin \beta \cos \beta\right. \\
& \left.+\cos a \cos \beta+\sin ^{2} \beta-\frac{1}{2} \cos ^{2} \beta-\beta \sin \beta \cos \beta\right)
\end{aligned}
$$

Likewise for the integral of the second member between $a$ and $+\beta$ we have

$$
\begin{gathered}
\frac{r^{2} y_{0}}{\sin \beta-\sin \alpha} \int_{a}^{3}(\sin \beta \cos \theta-\sin \theta \cos \theta-\sin \beta \cos \beta+\cos \beta \sin \theta) d \theta \\
=\frac{r^{2} y_{0}}{\sin \beta-\sin \alpha}\left(\sin ^{2} \beta-\frac{1}{2} \cos ^{2} \beta-\beta \sin \beta \cos \beta-\sin a \sin \beta\right. \\
\left.-\frac{1}{2} \cos ^{2} \alpha+\alpha \sin \beta \cos \beta+\cos a \cos \beta\right) .
\end{gathered}
$$

These two quantities are to be reduced to a common denominator, added together and equated with the first member (a.). Upon making simple cancellations, dividing through by $\sin \beta$, and factoring, we get the form of , $y_{0}$ given in the last section.
85. Formula for $\mathbf{H}$; Value of Ordinates. - When the value, of $y_{n}$ is computed, we can readily draw the stress diagram of Fig. 21, and scale the value of H : or the formma proved before, $\$ 40$, may be applied here, and is easily converted into the third form.

$$
\begin{equation*}
\mathrm{I}=\frac{\mathrm{W}}{y_{0}} \cdot \frac{c^{2}-b^{2}}{\underline{2} c}=\mathrm{W} \frac{\mathrm{AK} \cdot \mathrm{~K} \mathrm{~B}}{\mathrm{CK} \cdot \mathrm{AB}}=\frac{r\left(\sin ^{2} \beta-\sin ^{2} a\right)}{y_{0} \cdot 2 \sin \beta} \mathrm{~W} . \tag{1.}
\end{equation*}
$$

If calculations have already been made for $y_{0}$. the quantities resired for this formula are at hand.

Then the ordinate at each point of division, by which II is to be multiplied to girw MI for that point, will be, from § 84 , if $\theta$ is the angle between the two radii from the crown and the point E ,

$$
\begin{equation*}
\mathrm{EF}=\mathrm{DF}-\mathrm{D} \mathrm{E}=y_{0} \frac{\sin \beta \pm \sin \theta}{\sin \beta \pm \sin \alpha}-r(\cos \theta-\cos \beta) \tag{2.}
\end{equation*}
$$

The plus sign is to be used for points between the weight and the farther abutment, and the minus sign between the weight and the nearer abutment. We must remember, however, that, if $\theta$ is measured from the crown to the right as the positive direction, all angles $\theta$ on the left of the crown will be negative, and their sines will be minus. If EF is plus, it gives a positive bending moment, tending to make the arch less convex, and cice rersa.
86. Numerical Computation of $\mathbf{M}$. - In any practical case we should much prefer, as more easy and sufficiently accurate, to scale all of these quantities from a good-sized diagram; but it may be well to compute one set
of values of $M$ as an example, for fear the signs may give some readers trouble. Taking the case of Fig. 2.2 , let $\beta=45^{\circ}$ and $a=20^{\circ}$. Then the arc $\beta=.7554$ and $a=.3491$; sin $\beta=\cos \beta=.7071$; sin $a=.3420$, $\cos a=.9397$. These values, substituted in the equation of $\$ 5: 3$, give
$y_{0}=r \frac{(.5-.1170)\left(.75 .71 \frac{2}{.5071}-2.1218\right)}{.5-.1170+1.4142(.1194+.9397-.5554-.7071)}=\frac{.0384}{.0954} r=.403 r ;$ (1.), $\S 85$, will then become

$$
\mathrm{H}=\frac{(.5-.1170) r}{1.4142 \times .403 r} \mathrm{~W}=\frac{.383}{.570} \mathrm{~W}=.672 \mathrm{~W}
$$

$\operatorname{Sin} \beta+\sin a=1.0491 ; \sin \beta-\sin a=.3651 ;$
$\frac{y_{0}}{\sin \beta+\sin a}=\frac{.403 r}{1.0491}=.384 r ; \frac{y_{0}}{\sin \beta-\sin \alpha}=\frac{.403 r}{.3651}=1.104 r$.
Values of M.

87. Shear at any Right Section. - Suppose that the rib of Fig. 22 carries a single weight moler the point ( $C$, and that the curve of equilibrium is $\mathrm{A}(\mathrm{B}$. If 012 is the stress diagram, 2-3 will be the vertical component of the reaction at $A$, and $:-1$ that at $B$. To find the shear on a right section near $\Lambda$, as at E, lay off $2-3$, or $P_{1}$ in Fig. 23 , and draw II so that the arrows nay follow one another: then from 0 draw a line $0-4$ parallel to the tangent at E : the perpendicular distance $4-0$ will be the
shear in the wel. For we see by the direction of the arrows that these forces last drawn balance $P_{1}$ and $H$, and, as in Fig. 18, no matter how much the bending moment, and hence the flange stress, may be, the perpendienlar clistance $4-2$ is unchanged. The line $0-4$ will be the magnitude of the direct thrust. Both of these forces are given on the right of the section, and this shear is therefore negative. In the same way, for the point E near B , draw $1-3=-\mathrm{P}_{2}$ and $3-0=\mathrm{H}$; draw $0-8$ parallel to the tangent at $\mathrm{E} ; 8-1$, perpenticular to it, will be the shear on the right of the section, agmin negative, and 0-8 will be the direct thrust. It is noticeable that the nomal shear in the web near the left abutment is opposite in sign to $P_{1}$, while near the right abutment it agrees in sign with $\mathrm{P}^{\prime}$. For the lind of brace needed, see Fig. 10. It is evident that these figures may at once be drawn on the stress diagram, where $0-4$ and $4-2$ are already sketched. Such a way will answer well for a few points on a large figure, especially if we have applied such loads as give the maximum shear at any particular point. If, howerer, we desire to see the variation of the shear across the span, we may draw a different diagram.
88. Shear Diagram. - As the tangent is perpenticular to the ralins at the point of contact, we may at once see that the angles marked $\theta$ in Fig. 23 comespond with the angle $\theta$ made by the radins to the crown and that to the point E. Hence we get a value for the normal shear. P (ros $\theta$ - H sin A . Ss the point E is distant horizontally from the middle of the span an anoment $r$ sin $\theta$, the last term of this expression for shear varies directly th the distance from the centre; and if we draw $3-\overline{6}$. in the stress diagram of Fig. 22, parallel to the radins at $A$, cutting $0-6$ which is parallel to the tangent at $\lambda, 3-7$ will be $I I$ sin $\theta$ for A, and may be laid off at "t and $b r$ of Fig. 23. The rertical ordinate $r d$ will then represent $H$ sin $\theta$ at any point. $P_{1}$ is laid off at al, and $\mathrm{P}_{2}$ at $\mathrm{cm}_{\mathrm{m}}$ : with $a$ as centre, and these two distances as radii, draw the dotted ares seen in the figme ; laty off sereral angles $\theta$ at $c$, as, for instance. $l$ c! and $m$ in $h$ for the points E ; project $y$ and $n$ horizontally to $f$ muder the respective points E ;
df will be $\mathrm{P} \cos \theta$ and from several similarly located points the curves slt and $v f f^{\prime} r$ are fomd. Then at any point the certical distance $d f$-ed or of will be the normal shear in the wel, on the left of the section, positive if ahove the inclined line. negative if below it.

From the formula $\mathrm{P} \cos \theta$ - II $\sin \theta$, a table of sheas may be easily computed for any given arch. $\mathrm{I} \sin \theta+1 \mathrm{C} \cos \theta$ will give the direct thrust.
89. Distribution of Load to produce Equilibrium. - 1 series of lines drawn in the stress diagram from 0, parallel to the tangents at a mumber of equidistant points in a cirenlar rib, will cut off such portions of the load line as represent the loals necessary to make the successive sides of the enfuilitnimmpolygon parallel to these tangents, or, in slomt, coincident with the ril). But the lines radiating from 0) will successively intercept increasing lengths of load line. Hence the load which will keep a circular arch in equilibrimm must increase in intensity per horizontal foot from the crown to the springing, and must become infinite at the springing of a semicircular arch. Hence it follows that no amount and distribution of vertical load can make a semicircular arch a true equilibrium curve, that is, one which has no bending moment at any point. In fact, no curre which starts vertically from the abutment can be an equilibrium eurve under vertical loads. This may be seen in a more simple mamer if we consider that no arrangement of weights will cause a cord, attached at two points, to hang in a funicular polygon whose first side is vertical.
90. Effect of Change of Temperature. - The horizontal thrust or tension, due to a change of temperature, in a circular ribl hinged at the ends, is found by a similar method to that pursued for the parabolic rib. Referring, to avoid repetition. to what was said at that time, $\S(71-78$, the equation may be written, as given in § 74 ,

$$
\mathrm{H}_{t} \cdot \mathrm{\Sigma DE} \mathrm{E}^{2}= \pm 2 \mathrm{EI} \cdot t e c .
$$

Fig. 16 will answer for this case, if we imagine the are to be
 arch was $\frac{1}{2} \pi r^{3}$, a substitution in the above equation gives at once

$$
\mathrm{H}_{t}= \pm \frac{ \pm \mathbf{E} \mathbf{I} \cdot t e c}{\pi r^{-3}}= \pm 1.2\left(64 \frac{\mathbf{E} \mathbf{I} t e}{r^{2}}\right.
$$

for a semicircular rib. The bending moment at the crown, where it is a maximum, will be

$$
M(\max .)=\frac{4 \mathbf{E} \mathbf{I} / e}{\pi r}
$$

If the areh is less than a semicirele, (r.), s. 84 , gives

$$
\Sigma \mathrm{D} \mathrm{E}^{2}=r^{3}(\beta+2 \beta \cos \beta-: 3 \sin \beta \cos \beta),
$$

and $c=r$ sin $\beta$; therefore, substituting, we obtain

$$
\mathrm{II}_{t}= \pm \frac{\underline{2} \mathbf{E} t+\sin \beta}{r^{2}\left(\beta+\underline{3} \cos ^{2} \beta-3 \sin \beta \cos 3\right)},
$$

and the bending moment at the erown will he

$$
M(\text { max. })=\frac{2 \boldsymbol{E} \operatorname{tesin} \beta(1-\cos \beta)}{r\left(\beta+2 \cos ^{2} \beta-: \sin \beta \cos \beta\right)} .
$$

91. Shear from Change of Temperature. - If a load of the proper amomet and distribution were imposed on the rib to place it entirely in equilibrimm, and canse it to exert against the abouments the desired value of $H$ due to temperature, surf a load wonld smpply the amome of shear needed at each section. and, when the load is absent, the bracing must supply surh shear. The line wecer of the shear diagram of Fig. 23 will therefore limit the ordinates for shear at right sections of the web, under changes of temperature, when $0-3$ is the amount of $H_{t} . \quad \Lambda$ reference to $§ 78$ and $\$ 87$ will aid the reader in recalling these points.

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## CHAPTER VII.

## CIRCULAR RIB WITH FLXEI ENDS.

92. Values of Equations of Condition. - When the circular rib is fixed at the ends, we apply the three equations of condition which were developed in $\leqslant \leqslant 17-19$, summing up the ordinates, however, along the arch, as has just been tone in the preceding case, in place of the horizontal line. When the arch is a complete semicircle. or, as it is often callecl. a complete arch, as distinguished from a segmental one, the value of $y_{0}, y_{1}$. and $y_{2}$ may le obtained by a derice similar to the one employed in $\$ 82$. The equation to satisfy the first condition is easily derived, but the two others present more difficulty ; it is: therefore not expedient to take up the semicircle as a special case. but rather to work out the general equations, and make the necessary substitutions.

In the arch of Fig. 24, let A $N=y_{1}$. C K $=y_{0}$ and $\mathrm{B} \mathrm{R}=y_{2}$ : $\mathrm{MOB}=\mathrm{MOA}=\beta$. $\mathrm{MOI}=\mu$, and MOE , to any point E . $=A$. angles to the right of M being positive. The notation agrees with that just used. Then it may be proved that the three equations of condition will reduce to

$$
\begin{gathered}
\sin \beta y_{0}+\frac{1}{2}(\sin \beta+\sin a) y_{1}+\frac{1}{2}(\sin \beta-\sin a) y_{2}=(\beta-\sin \beta \cos \beta) r, \\
\quad-\sin \beta(\cos a-\cos \beta+a \sin a-\beta \sin \beta) y_{0} \\
+\frac{1}{2}(\sin \beta-\sin a)(\cos \alpha-\cos \beta+a \sin a+\beta \sin \alpha) y_{1} \\
+\frac{1}{2}(\sin \beta+\sin a)(\cos a-\cos \beta+a \sin a-\beta \sin \alpha) y_{2} \\
=\left(\sin ^{2} \beta-\beta \cos \beta\right)\left(\sin ^{2} \beta-\sin ^{2} a\right) r ;
\end{gathered}
$$

$[(\beta-\cos \beta \sin \beta) \sin a-(a+\sin a \cos a-2 \sin a \cos \beta) \sin \beta] y_{0}$
$+\frac{1}{2}(\sin \beta-\sin a)(a+\sin a \cos a+\beta-\sin \beta \cos \beta-2 \sin a \cos \beta) y_{1}$
$+\frac{1}{2}(\sin \beta+\sin a)(a+\sin a \cos a-\beta+\sin \beta \cos \beta-2 \sin a \cos \beta) y_{2}=0$. (3.)
It will be easier to solve the numerical equations alter the values of $\alpha$ and $\beta$, with their sines and cosines, are introduced, than to deduce independent values of $y_{1}$, \&c., at present. They may be written more briefly, for convenience in substitution, if

$$
\begin{gather*}
\sin \beta-\sin a=a ; \sin \beta+\sin a=b ; a+\sin a \cos a-2 \sin a \cos \beta=c ; \\
\beta-\sin \beta \cos \beta=d ; \cos a-\cos \beta+a \sin a=e \\
\sin \beta y_{0}+\frac{1}{2} b y_{1}+\frac{1}{2} a y_{2}=d r . \quad(4 .) \tag{t.}
\end{gather*}
$$

$$
\begin{gathered}
-(e-\beta \sin \beta) \sin \beta y_{0}+\frac{1}{2} a(e+\beta \sin a) y_{1}+\frac{1}{2} b(e-\beta \sin a) y_{2} \\
=a b(\sin \beta-\beta \cos \beta) r . \quad(5 .)
\end{gathered}
$$

$$
\begin{equation*}
(d \sin a-c \sin \beta) y_{0}+\frac{1}{2} a(c+d) y_{1}+\frac{1}{2} b(c-d) y_{2}=0 \tag{6.}
\end{equation*}
$$

93. Special Values for Semicircular Rib. - If the arch is a semicircle, $\beta^{\prime}=\frac{1}{2} \pi ; \sin \beta^{\prime}=1 ; \cos \beta^{\prime}=0$; and the three equations of the last section reduce to

$$
\begin{align*}
& y_{0}+\frac{1}{2}(1+\sin a) y_{1}+\frac{1}{2}(1-\sin a) y_{2}=\frac{1}{2} \pi r ; \quad \text { (1.) }  \tag{1.}\\
& \left(\frac{1}{2} \pi-\cos a-a \sin a\right) y_{0}+\frac{1}{2}(1-\sin a)\left(\cos a+a \sin a+\frac{1}{2} \pi \sin a\right) y_{1} \\
& +\frac{1}{2}(1+\sin a)\left(\cos a+a \sin a-\frac{1}{2} \pi \sin a\right) y_{2}=\left(1-\sin ^{2} a\right) r: \quad(2 .) \\
& \left(\frac{1}{2} \pi \sin a-a-\sin a \cos a\right) y_{0}+\frac{1}{2}(1-\sin a)\left(a+\sin a \cos a+\frac{1}{2} \pi\right) y_{1} \\
& \quad+\frac{1}{2}(1+\sin a)\left(a+\sin a \cos a-\frac{1}{2} \pi\right) y_{2}=0 . \quad(3 .)
\end{align*}
$$

If equation (1.) is multiplied by u, equation (3.) may be added to it, and then (2.) may be multiplied by sin c. and subtracted from their sum, when there will result

$$
\left(a+\frac{1}{4} \pi-\frac{1}{4} \pi \sin a\right) y_{1}+\left(a-\frac{1}{4} \pi-\frac{1}{4} \pi \sin a\right) y_{2}=\left(\frac{1}{2} \pi a-\sin a\right) r .
$$

If (1.) is multiplied by $\frac{1}{2} \pi-\cos$ u - usin $\kappa$. and equation (2.) is subtracted from it, we shall get, upon dividing by the common coefficient of $y_{1}$ and $y_{2}$.

$$
\frac{1}{2}\left(y_{1}+y_{2}\right)=\frac{\frac{1}{2} \pi\left(\frac{1}{2} \pi-\cos a-a \sin a\right)-\cos ^{2} a}{\frac{1}{2} \pi-2 \cos a-2 \sin a+\frac{1}{2} \pi \sin ^{2} a} r,
$$

which, if the quantity in the parentheses be represented by $g$, may be written,

$$
\frac{1}{2}\left(y_{1}+y_{2}\right)=\frac{\frac{1}{2} \pi!-\cos ^{2} a}{2 g-\frac{1}{2} \pi \cos ^{2} a} r
$$

Upon multiplying this equation by $2 \alpha-\frac{1}{2} \pi \sin \alpha$, and subtracting it from (4.), we obtain, by factoring the second member,

$$
\frac{1}{2}\left(y_{1}-y_{2}\right)=\left(\begin{array}{c}
\left.\frac{4}{\pi} \begin{array}{c}
\pi \\
\pi \\
2
\end{array}\right) \frac{\left(a \cos ^{2} a-y \sin a\right)}{2 y-\frac{1}{2} \pi \cos ^{2} a} r .  \tag{6.}\\
2 .
\end{array}\right.
$$

The sum of (5.) and (6.) will give $y_{1}$; their difference will give $y_{2}$; and these values, inserted in (1.), will readily give us $\% / 4$.
92. First Equation of Condition. - Many of the following expressions are similar to those of $\$ 54$, and a remembrance of the relation between $y_{1}$ and $y_{2}$ will, in a measure, prevent the ensuing work from seeming so involved as it otherwise may appear. Generally, coefficients of $y_{1}$ and $y_{2}$ will differ only in the signs of the terms which contain $a$ and sine $a$. The first condition is

$$
\Sigma \mathrm{DE}=\Sigma \mathrm{D} F . \mathrm{D} \mathrm{E}
$$

From \& st, we have

$$
\check{\mathrm{DE}} \mathrm{E}^{2}=r^{3}\left(\beta+2 \beta \cos ^{2} \beta-3 \sin \beta \cos \beta\right) .
$$

It will be seen, from Fig. 24, that $1 \mathrm{~F}=\mathrm{DL}+\mathrm{LF}=y_{1}\left(\right.$ or $\left.y_{2}\right)+\mathrm{LF}$, D L in the sketch being negative on the right of $\mathbb{K}$. and that, therefore, in place of the values of the section just referred to, we shall write

$$
\begin{aligned}
& \mathrm{DF}=y_{1}+\frac{\sin \beta+\sin \theta}{\sin \beta+\sin \alpha}\left(y_{0}-y_{1}\right) \text {, on the left of } \mathrm{K} ; \\
& \mathrm{DF}=y_{2}+\frac{\sin \beta-\sin \theta}{\sin \beta-\sin \alpha}\left(y_{0}-y_{2}\right) \text {, on the right of } \mathbf{K} .
\end{aligned}
$$

For the value of the second member of the above equation of condition between $\alpha$ and $-\beta$ we lave then, since $\mathrm{DE}=r(\cos \theta-\cos \beta)$,

$$
\begin{aligned}
& r^{2} \int_{-\beta}^{a}\left[y_{1}(\cos \theta-\cos \beta)+\frac{y_{0}-y_{1}}{\sin \beta+\sin \alpha}(\sin \beta \cos \theta+\sin \theta \cos \theta-\sin \beta \cos \beta\right. \\
& \quad-\cos \beta \sin \theta)]^{*} d \theta=r^{2}\left[y_{1}(\sin \alpha-a \cos \beta+\sin \beta-\beta \cos \beta)\right. \\
& \quad+\frac{y_{0}-y_{1}}{\sin ^{2} \beta+\sin a}\left(\sin a \sin \beta-\frac{1}{2} \cos ^{2} \alpha-\alpha \sin \beta \cos \beta+\cos \alpha \cos \beta\right. \\
& \left.\left.\quad+\sin ^{2} \beta-\frac{1}{2} \cos ^{2} \beta-\beta \sin \beta \cos \beta\right)\right] .
\end{aligned}
$$

Likewise, for the value of the second member between $a$ and $+\beta$

[^1]\[

$$
\begin{aligned}
& r^{2} \int_{a}^{3}\left[y_{2}(\cos \theta-\cos \beta)+\frac{y_{0}-1 / 2}{\sin \beta-\sin a}(\sin \beta \cos \theta-\sin \theta \cos \theta-\sin \beta \cos \beta\right. \\
& \quad+\cos \beta \sin \theta)]^{*} \lambda \theta=r^{2}\left[y_{2}(\sin \beta-\beta \cos \beta-\sin a+a \cos \beta)\right. \\
& \quad+\frac{y_{0}-y_{2}}{\sin \beta-\sin a}\left(\sin ^{2} \beta-\frac{1}{2} \cos ^{2} \beta-\beta \sin \beta \cos \beta-\sin a \sin \beta-\frac{1}{2} \cos ^{2} a\right. \\
& \quad+a \sin \beta \cos \beta+\cos a \cos \beta)] .
\end{aligned}
$$
\]

Equating the sum of these two quantities which make up the second member, with the first member. we obtain the first equation of condition, which, when cleared of fractions. becomes
$y_{0}\left(2 \sin ^{3} \beta-\sin \beta \cos ^{2} \beta-2 \beta \sin ^{2} \beta \cos \beta-\cos ^{2} a \sin \beta+2 \cos a \sin \beta \cos \beta\right.$ $\left.-2 \sin ^{2} a \sin \beta+2 a \sin a \sin \beta \cos \beta\right)+y_{1}\left(\frac{1}{2} \sin \beta \cos ^{2} \beta-\sin ^{3} a\right.$ $+a \sin ^{2} a \cos \beta+\beta \sin ^{2} a \cos \beta+\frac{1}{2} \cos ^{2} a \sin \beta-\cos a \sin \beta \cos \beta$ $-\frac{1}{2} \sin a \cos ^{2} a-a \sin a \sin \beta \cos \beta+\sin a \cos a \cos \beta+\sin a \sin ^{2}, 3$ $\left.-\frac{1}{2} \sin a \cos ^{2} \beta-\beta \sin a \sin \beta \cos \beta\right)+1 / 2\left(\frac{1}{2} \sin \beta \cos ^{2} \beta+-\sin ^{3} a\right.$ $-a \sin ^{2} a \cos \beta+\beta \sin ^{2} a \cos \beta+\frac{1}{2} \cos ^{2} a \sin \beta-\cos a \sin \beta \cos \beta$ $+\frac{1}{2} \sin a \cos ^{2} a-a \sin a \sin \beta \cos \beta-\sin a \cos a \cos \beta-\sin a \sin ^{2} \beta$ $\left.+\frac{1}{2} \sin a \cos ^{2} \beta+\beta \sin a \sin \beta \cos \beta\right)=r\left(\sin ^{2} \beta-\sin ^{2} a\right)\left(\beta+2 \beta \cos ^{2} \beta\right.$ $-3 \sin \beta \cos \beta$ ).
95. Second Equation of Condition. - The next condition to be satisfied is $\Sigma \mathrm{I}) \mathrm{E}=\mathrm{\Sigma}$ I) F , or, introlucing the values of these quantities from the preceding section,

$$
\begin{gathered}
r^{2} \int_{-\beta}^{+\beta}(\cos \theta-\cos \beta) d \theta=r \int_{-\beta}^{a}\left[y_{1}+\frac{y_{0}-y_{1}}{\sin \beta+\sin a}(\sin \beta+\sin \theta)\right] d \theta \\
+r \int_{a}^{\beta}\left[y_{2}+\frac{y_{0}-y_{2}}{\sin \beta-\sin a}(\sin \beta-\sin \theta)\right] d \theta .
\end{gathered}
$$

Performing the indicated integration, and clearing of fractions, we obtain

$$
\begin{aligned}
& y_{0}\left(2 \beta \sin ^{2} \beta-2 \cos a \sin \beta+2 \sin \beta \cos \beta-2 a \sin a \sin \beta\right)+y_{1}\left(-\beta \sin ^{2} a\right. \\
& \quad-a \sin ^{2} a+\cos a \sin \beta-\sin \beta \cos \beta+a \sin a \sin \beta+\beta \sin a \sin \beta \\
& \quad-\sin a \cos a+\sin a \cos \beta)+y_{2}\left(-\beta \sin ^{2} a+a \sin ^{2} a+\cos a \sin \beta\right. \\
& \quad-\sin \beta \cos \beta+a \sin a \sin \beta-\beta \sin a \sin \beta+\sin a \cos a-\sin a \cos \beta) \\
& \quad=2 r\left(\sin ^{2} \beta-\sin ^{2} a\right)(\sin \beta-\beta \cos \beta) .
\end{aligned}
$$

9f. Third Equation of Condition. - The thire comlition, in the monlifien
 this eonlition becomes. ly multiplying the previous condition ly D B.

$$
\begin{aligned}
& r^{3} \int_{3-}^{+i}(\sin \beta \cos \theta-\sin n \cos \theta-\sin \beta \cos \beta+\cos \beta \sin \theta) d \theta \\
& =r^{2} \int_{-3}^{a}\left[y_{1}(\sin \beta-\sin \theta)+\frac{y_{0}-y_{1}}{\sin \beta+\sin a}\left(\sin ^{2} \beta-\sin ^{2} \theta\right) d \theta\right. \\
+ & r^{2} \int_{a}^{3}\left[1_{2}(\sin \beta-\sin \theta)+\frac{y_{0}-y_{0}}{\sin \beta-\sin \alpha}\left(\sin ^{2} \beta-2 \sin \beta \sin \theta+\sin ^{2} \theta\right)\right] d \theta, *
\end{aligned}
$$

which, when integrated and cleared of fractions, gives

$$
\begin{aligned}
& y_{0}\left(2 \beta \sin ^{3} \beta-a \sin \beta-\sin a \cos a \sin \beta+2 \sin ^{2} \beta \cos \beta-2 a \sin a \sin ^{2} \beta\right. \\
& \left.\quad+\beta \sin a+\sin a \sin \beta \cos \beta-2 \cos a \sin ^{2} \beta\right)+y_{1}\left(-\frac{3}{2} \sin ^{2} \beta \cos \beta\right. \\
& \quad+\cos a \sin ^{2} \beta-\beta \sin ^{2} a \sin \beta-a \sin ^{2} a \sin \beta+\sin ^{2} a \cos \beta-\frac{1}{2} \sin ^{2} a \cos a \\
& \quad+\frac{1}{2} a \sin \beta-\frac{1}{2} \sin a \cos a \sin \beta+\frac{1}{2} \beta \sin \beta+\beta \sin ^{2} a \sin ^{2} \beta+a \sin ^{2} a \sin ^{2} \beta \\
& \left.\quad-\frac{1}{2} a \sin a-\frac{1}{2} 3 \sin a+\frac{1}{2} \sin a \sin \beta \cos 3\right)+y=\left(-\frac{1}{2} \sin ^{2} \beta \cos \beta\right. \\
& \quad+\cos a \sin ^{2} \beta-\beta \sin ^{2} a \sin \beta+a \sin ^{2} a \sin 3-\sin ^{2} a \cos \beta+\frac{1}{2} \sin ^{2} a \cos a \\
& \quad+\frac{1}{2} a \sin \beta+\frac{3}{2} \sin a \cos a \sin \beta-\frac{1}{2} \beta \sin \beta-\beta \sin a \sin ^{2} \beta+a \sin ^{2} a \sin ^{2} \beta \\
& \left.\quad+\frac{1}{2} a \sin a-\frac{1}{2} \beta \sin a-\frac{3}{2} \sin a \sin \beta \cos \beta\right)=2 r \sin \beta\left(\sin ^{2} \beta-\sin ^{2} a\right) \\
& \quad(\sin \beta-\beta \sin )
\end{aligned}
$$

97. Reduction of Equations. - If the second equation of condition is multiplied by $\cos \beta$, and added to the first, there results an equation in which, as soon as we write $1-\sin ^{2} a$ for $\cos ^{2} a$, and $1-\sin ^{2}, 3$ for $\cos ^{2} \beta$. there will be found a common factor ( $\sin ^{2} \beta-\sin ^{2} \alpha$ ). This being cancelled ont there results. (1.).s?2. The secom equation again may be divided hy and then factored. by simple inspection, into ( 2. ). s! ! . Finally, the secomi equation of condition may be multiplied by sim s, and subtracted from the third. when, mon factoring, we obtain (3.), s 92.

It will he seen that the solntion of (t.). (i).). and (ij.). s! ! - for any given arch, amb for several values of $a$, will mot involve much work. owing to the recurrence of the known factors denoted by a, l. c. $/$, anm . . As the arch inay subtemb any angle. it will not be experlient to gon into calculations here for any special values of 3. One case will be taken up later.
98. Values of $\mathbf{H}, \boldsymbol{\&} \mathbf{c}$. - When the desired ordinates for any arch are computed, we have the option of ohtaining the values

* $\int \sin ^{2} \theta d \theta=\frac{1}{2}(\theta-\sin \theta \cos \theta)$. See also note to $\S 84$.
of H , of the vertical components of the abutment reactions, and of the ordinates for bending moment, either by graphical construction, or by formulae similar to those applied to the parabolic rib. By noticing the expressions to be sulbstituted for $l$, $c$, and $k$ in the case of the circular arch with hinged ends, one can readily adapt the formule of $\$ 63$ and $\$ 65$ to the computations for this case. The ordinates to the circular areh will be the same as in $\$ 85$.

99. Table of $y_{0}, y_{1}$, and $y_{2}$ for Semicircle. - We may, however, obtain the ordinates $y_{0}, \& \in \cdot$, for a semicircle with comparative ease. and, as such a rib is sometimes nsed for large roofs, these values may be convenient. Semicircular masonry arches, having backing above the alntments, present a different case.

If $c$ is taken as $20^{\circ}$ or $.3491, \sin \alpha=.3420, \cos \alpha=.9397$, and $\frac{1}{2} \pi=1.5708$; lience, in $\leqslant 93, g=.5117$, and (5.) and (6.) become

$$
\begin{aligned}
& \frac{1}{2}\left(y_{1}+y_{2}\right)=\frac{-.0792}{-.3616} r=.2172 r ; \\
& \frac{1}{2}\left(y_{1}-y_{2}\right)=\frac{-.2977 \times .1333}{-.3616} r=.1088 r ;
\end{aligned}
$$

whence $y_{1}=.326 r$, and $y_{2}=.108 r$. By substitution in (1.), $\$ 93, y_{0}=(1.5708-.2187-.0357) r=1.316 r$.

If similar computations are carried out for other values of ", we shall complete the following table for a semicircular rib) with fixed ends:

| $a$ | $y_{1 \cdot}$ | $y_{0}$. | $y_{2 .}$ |
| :---: | :---: | :---: | :---: |
| $0^{\circ}$ | $.241 r$ | $1.330 r$ | $.241 r$ |
| 10 | .288 | 1.326 | .183 |
| 20 | .326 | 1.316 | .108 |
| 30 | .360 | 1.298 | -.111 |
| 40 | .387 | 1.27 .5 | -.125 |
| 50 | .413 | 1.24 .5 | -.330 |
| 60 | .434 | 1.210 | -.665 |
| 70 | .455 | 1.170 | -1.333 |
| 80 | .475 | 1.125 | -3.319 |

Other intermediate values can be obtained, if desired, by the
formula for interpolation, $£ 45$. The number of decimals it is droimble to use in any particular ease will depend upon the value of $r$. The equilibrimm polygons for these ordinates have been drawn in Fig. 25, and from them we get the different values of $H$, for a weight $W$ at the several divisions, as shown in the aceompanying stress diagram.
100. Example. - As an application of these results, let us draw the equilibrium corve for a semicircular arch of uniform section carrying only its own weight. As this weight is symmetrically disposed. $y_{1}^{\prime}=y_{2}^{\prime}$. By drawing the stres diagram of Fig. 2. to a sufficiently large scale, we shall find by measmement, that II, for a weight at the erown, $10^{\circ}, 20^{\circ}$, \&e., from the crown. will he $.46, .44, .39, .31, .28, .14, .07, .02$, and .01 W respectively. If we double all of these values except the one for a weight at the crown, and take the sum of the whole, we shall obtain for the horizontal thrust, $\mathrm{I}^{\prime}=3.6$ ? $\mathrm{W}^{\prime}$ for 17 loads, each equal to $W$, at the 17 points of division in the whole areh.

To find $y_{1}^{\prime}$, multiply each $y_{1}$ by its $H$, remembering, that, when the weights are on the left of the crown, the values of $y_{2}$ in the table of $\$ 99$ become $\%$, and that we may, therefores before multiplying by H, add together $y_{1}$ and $y_{2}$ for each point except the crown, and then divide the sim of these products by II', just obtained. (Compare si 6T.) For example, for a load W on each of the two pronts distant $30^{\circ}$ from the crown, $\mathrm{H} y_{1}+\mathrm{Il} y_{2}=.31 W(.360+.011) \cdot i=.11 .5 \cdot \mathrm{~W}$, the value of M at the abutments. Performing the operations, and taking the



To comstruct the equilibitum turve we divile the semicircle
 and draw rerticals through the points of division. Assume the weight of the arch to be represented hy a rertical line of any convenient leugth. Since the loads are supposed to be concentrated at the points of division, one-eighteenth of the gross
weight of the arch will be found at cach of these points, and one-thirty-sixth at A and B; for A and B will cach carry directly onehalf of the adjacent division. Therefore, beginning and closing with one-thirty-sixth, space off the load-line into eighteenths; from the middle of the load-line lay off $\mathrm{II}^{\prime}=3.68 \mathrm{~W}=3-0$, where $\mathrm{W}=$ weight of one division, or $\mathrm{H}^{\prime}=\frac{3.68}{18}=204$ of the whole weight of the rib. One-half of this load-line is 1-3. Lay off $y_{1}^{\prime}$ and $y_{2}^{\prime}=.17 r$, at $A$ and $B$, and draw the sides of the equilibrimu polygon parallel to the lines which ruliate from the extremity of $\mathrm{H}^{\prime}$ to the points of division of the lomiline, thus obtaining the curve E G D. The second half of the eurve was obtained by spacing off $0^{\prime}-3$ to the left.
101. Practical Application. - Having at hand a wooden model of an arch-ring, representing the vonssoirs, or stones, of a semicircular arch, we tried some experiments as tests of the accuracy of this method of analysis and of the correctness of these results. The arch is represented by Fig. 26 , and consisted of forty-two independent voussoirs. The span, $A B$, of the middle line of the ring, 18 inches, was 13.09 times the thickness of the ring, and the structure would apparently just stand alone when left to itself: a slight additional weight at the crown would cause that part to sink, the haunches to move outwards. and the ring to fall in pieces. Considering that this arch, so long as it rested squarely on the faces at $A$ and $B$, was fixed in direction, or not free to turn at the ends, we laid off at $A \mathrm{E}$ and BD the value of $y_{1}$ obtained in the last section, and drew the equilibrium polygon, as just described, on the centre line of the ring, begimning at D with a line parallel to $0-4$. It will be noted that no line is used from 0 to 1 ; for the weight represented by $1-4$ is directly supported at $B$; while the amount $4-5$ is the weight concentrated on the first vertical just above I).

As the arch is a continuous ring, the weights may properly be concentrated at a greater number of points; so that finally the true equilibritun curve will pass through the vertices of the poly-
gon we have just constrinted: the difference between the two is mimportant, however, and is only appreciable near the crown. The bending moment at any point has heen proved to be equal to 11 multiplied by the vertical ordinate between the centre line and the equilibrimu curve, or, by $\$ \mathbf{S} 10$, also equal to T, the thrust along the tangent to the equilibriun curve, muttiplied by the perpendicular from a point on the centre line io this tangent: therefore if we draw E F as this tangent, the bending moment at $A$ will equal either H . E A, or the thrust along E F multiplied by the perpendicular from A. The direction of the thrust E F, if prolonged, cuts the springing joint very close to the outside edge: it will also be noticed that the equilibrium curve approaches quite near to the elge of the coussoirs at the crown ( t . Now, as we reminded the reader in $\$ 11$ that the force T , or $0^{\prime}-1$, at the distance F A from the centre line of the rib, is equal to the same force at the centre line and the couple which produces hending moment, conversely, the resultant of the pressure of this rib at the end $A$ must cut the base in the prolongation of the line E F: in short, the tangent to the equilibrium curve at each point gives the direction and point of application of the resultant thrust at that right section of the ril) to which it belongs, as astertained by erecting a vertical from the middle point of the section.
102. Limiting Position of Equilibrium Curve. - If, as is usually the case, the intensity of the resisting force of the abutment at A is assumed to vary uniformly from one edge to the other, then, in case the resistance is zero at the inside edge and a maximum at the outside edge, the intensity at all points ean be represented, as shown in the small sketch marked $I^{\prime}$, by the ordinates of a triangle whose base is the breadth of a voussoir. and whose longest ordinate is the intensity of the presiure at the edge near F . The total pressure will be equal to the area of the triangle, and the resultant will pass through the centre of gravity of the triangle, cutting the base at one-third of its length from the onter edge. If there existed any tension near the imner edge, we should have two triangles, as shown in the
other sketch, the inclined line cutting the base at the point where the stress changed from tension to compression; and the resultant of the two stresses must, since they are of oprosite kinds, lie outside of their separate resultants, and on the side of the greater one. This fact as to the position of the resultant of two opposite parallel forces was indicated in $\stackrel{\Im}{3} 11$, Fig. 2 , and is one of the well-known properties of the lever, as proved in Mechanics.
Since, then, the resultant force, or the thrust on a section of the rib) of Fing. 26, at A, B, and ('. passes near the edge of the section, or, as it is often stated, outside of the middle third of the cross-section, we shonld expeet to find tension at the inside edge of the joint at these proints. As this model consist. simply of wooden blocks placed in juxtaposition, a vousioir cannot exert tension on its neighbor at any point of contact. and movement must immediately take place when the weight of the rib is allowed to act freely, rotation being set up about the outside edges at F, ( r , and Q . The crown will sink, the haunches will move outwards, and the arch may be expected to fall. The reader will remember that it was explained, in $\S 12$, that an arch tends to move away from the equilibrium curve.

Since any material is compressible, it is probable that the assumption of a uniform variation of intensity of stress at any section will not be strictly true; that the stress may not be exerted over the entire surface of the ariginally phane joint: and that therefore the equilibrium curve may pass somewhat outside of the middle third of the joint without cansing the arch to fall, although the joint will then open slightly at the edge where no pressure is exerted, by reason of the compression causing the joint to be no longer plane. But such an assump,tion gives an additional element of safety to a design, when the engineer so proportions his rib of rectangular section that the equilibrium eure of the load at any time shall never leave the limits of the middle third, and the tensile strength of the eement will not then be relied upon to assure stability.

103．Model as hinged at Three Points．－Tha arch of Fig．26 stonel when the string which at first passed aromud the exterior wats removed，although a slight change of shapee wan whercable．A close inspection，howerer．showed that the rous－ soins at the crown and the two springings were then in contact only at the outer edges．The rotation at thene joints．indicatem in the last section as probable，had commeneed：but，als somen ats the rib，became thus hinged at three points，it was in equili－ hrium．It is desirable then，as a further test，to drath the equilibrium curve for this rib hinged at the crown and spring－ ings．As the change of shape and curvature was very little， the supposition that the weight of the vonswoins is concentrated along the are $\mathrm{K}(\mathrm{Q}$ will be sufficiently near the truth for our purpose．

The half－weight being represented by 1－3，the first step in to find the valne of H for this case，when the load is concentratect at intervals of ten degrees along the onter semicircle．We can arail ourselves of the formula of $\leqslant 23$ ．tinding the different values of $b$ by measurement．or from tables of sines，sinc⿻日． $l=r \sin \theta$ ，and summing up the several amomits of $H$ for the whole semicircle；or，as is done in this figure．we may use the principle explained in $\$ 30$ ，that any two sides of the funicular polygon，or two tangents to the equilibrium curre will meet， when prolonged，on the vertical through the centre of gravity of the included weight．Since the arch is symmetrically loaded． the thrnst at the crown will be horizontal，and therefore lie in the line K L：the centre of gravity of the guadrant are $\mathrm{Ki} Q$ will be on the rertical line P L．drawn at such a distance．Ki L ， from the crown as to satisfy the value for the ordinate from the centre of a circle to the centre of gravity of a circular arc，riz．． $\frac{\text { radins } X \text { ehord }}{\text { length of arc }}$ ；and therefore the thrust at the pringing will lie in the line $Q L$ ．drawn from（ 2 to the intersection of the other two forces．As $1-3$ represents the weight of one－half the arch，and the thrust at the crown is parallel to 3－0．a line from 1，parallel to Q L，will complete the triangle of forces，and，
cutting the horizontal line at 9 , will determine $3-9$ to be the desired value of H . The equilibrimm polygon can now be drawn from $\mathbf{Q}$ to K , its sides being successively parallel to lines radiating from 9, the first line being 9-4 and the last one 9-1). These lines are not drawn in the stress diagram. The other half of the polygon may be added, if desired.

It will now be seen, that, excepting the hinged points, the nearest approach of the equilibrimm curve to the edge of a roussoir is at $P$, where it is still well within the rib, and consequently no further movement of the rib is to be expected. Another model, somewhat thimer than the one here illustrated, was experimented with, and would not stand. If the arch of Fig. 26 is slightly weighted at K , the joint at P begins to open on the outside, confirming the result, that the equilibrium curve here passes nearest to the inner edge. If it be objected that the change of outline previously referred to carries the portion of the rib near P farther from the centre, so that the equilibrium curve may rin nearer the edge than we have plotted it, we rejoin, that such a movement, carrying the centre of gravity, and hence the line PL, in the same direction, will cause QL to make a slightly less angle with the vertical, diminishing the value of H , and moving the equilibrium curve also a little away from P.
104. Model as hinged at Abutments. - For the purpose of making an additional test of our results, we finally placed it small wire at A and B , thus hinging the rib on its centre line at these points. The equilibrimm curve for one-half of the arch is A NK. The amount of $H$ is determined by computation from the formula of $\S 85$, which becomes, for a semicircular rib, $\mathrm{H}=\frac{\cos ^{2} \varepsilon}{\pi} \mathrm{~W}$ : and the summation for the whole arch, carrying W at intervals of ten degrees along the centre line, is $\mathrm{H}=2.86 \mathrm{~W}$. laid off at $3-8$. Radiating lines between 8-4 and 8-6 will enable one to draw A NK. The arch, when released, fell in ruins, and the first joint to open, on the outside at the haunch, was near N , lower than P ' in the former case.

We have dwelt on these curves at some length, as they give so good a confirmation of previous deductions and results, and as they will aid the reader in assuring himself that he understands the method of treatment. Such diagrams must, for aecuracy, be drawn to quite a large scale, and the results will then be very satisfactory.
105. Effect of Change of Temperature. - It remains to find the effect of ehange of temperature on the circular rib with fixed ends. As was previously indieated in $\$ 76$, we must find the height $\mathrm{A} \mathrm{C}=\mathrm{BI}=y_{1}$, at which the equilibrinum line shall be drawn in Fig. 27, by the condition that the change of inclination at the abutments, or $\triangle \mathrm{EF}=0$. If the notation of the angles subtended by portions of the arch is as before, and as marked in the figure, we have $\mathrm{EF}=\mathrm{D} \mathrm{E}-y_{1}$, and

$$
\pm \mathrm{E} \mathrm{~F}=\int_{-\beta}^{+\beta} r\left(r \cos \theta \quad r \cos \beta-y_{1}\right) d \xi=2 r\left(r \sin \beta-r \beta \cos \beta-y_{1} \beta\right)=0,
$$

${ }^{\circ}$

$$
y_{1}=r\left(\frac{\sin \beta}{\beta}-\cos \beta\right),
$$

which becomes, for a semicircle.

$$
y_{1}=\stackrel{\ddot{2}}{\pi} r=0 . f i n \cdot 2 r .
$$

The first term of (1.). From $\leqq 84, ~ \leq D E^{2}=r^{3}\left(\beta+2 \beta^{\prime} \cos ^{2} \beta-3 \sin \beta \cos \beta\right)$, while $y_{1} \cdot \underline{\mathrm{D}} \mathrm{E}$ gives, as above, $r^{3}\left(\frac{\sin \beta}{\beta^{3}}-\cos \beta^{\prime}\right)\left(2 \sin \beta-2 \beta \cos \beta^{\prime}\right)$; so that the first term reduees to $r^{33}\left(\beta^{\prime}+\sin \beta \cos \beta-\frac{2 \sin ^{2} \beta}{\beta}\right)$, and (1.) $\S 76$, takes the form of

$$
\begin{aligned}
\mathrm{H}_{t} \cdot r^{3}\left(\beta+\sin \beta \cos \beta-\frac{2 \sin ^{-} \beta}{\beta}\right) & = \pm 2 \mathbf{E} \mathbf{I} \text { ter } \sin \beta \\
\mathrm{H}_{t} & = \pm \frac{2 \mathbf{E I} t e}{r^{2}\left(\frac{\beta}{\sin \beta}+\cos \beta-2 \frac{\sin \beta}{\beta}\right)} .
\end{aligned}
$$

For a semicircle, the formula for horizontal thrust simplifies into

$$
\mathrm{H}_{t}= \pm \frac{2 \mathbf{E} \mathbf{I} t e}{r^{2}\left(\frac{\pi}{2}-2 \frac{2}{\pi}\right)}= \pm 6.75 \frac{\mathbf{E} \mathbf{I} t e}{r^{2}} .
$$

The bending moments at the crown and springing can now be written, and compared with $\$ 90$. II is about five times as great, M at springing about 3.2 times and M at crown abont 1.9 times as erreat as when the conds are hinged. The remarks of $\$ 91$ in regard to shear will apply equally well here.

For the Elliptic Rib, see $\S 153$.
106. Maximum Stress determined by Length of Ordinate; Rib of Rectangular Section. - It may sometimes be convenient to have the means of determining from a simple inspection of a diagram, by noting the position of the equilibrium polygon, how much the maximum intensity of stress at any section exceeds the mean intensity. As the mean intensity $f=\mathrm{T} \div \mathrm{S}$ where T is the direct thrust and S is the area of cross-section, and is obtained at any point from the known value of the thrust in the side of the equilibrium polygon, the maximum intensity of stress will be readily fomm by multiping by the proper ratio. The stress arising from bending moment in a solid section is always taken as uniformly varying (see Fig. 2). The combination of direct stress with that from bending moment will also give a uniformly varying stress.

Considering, first, the rib of rectangular cross-section, Fig. 28. we see, that if we call the intensity, A C, of direct stress unity, a bending moment which will produce a compression, D E, of unity at the upper extreme fibre, and a tension, C A, of unity at the lower extreme fibre, will bring the resultant stress at all points to the amounts indicated in the left-hand sketch, twice the mean intensity at one edge, and zero at the other. If the (ross-section is treated by the method of Part I., "Roofs," p. 57. Fig. 24, in order to make an equivalent area of miform stress equal to the maximum, we get the shaded area of the section on the left, which is evidently one-hall of the whole
section. 'The centre of gravity of this area, lying at one-thind the height from the upper erge, will he the point of application of the resultant force ont the ross-section. If the bending moment is reversed, the sketch will be inserted: hence, when the line of thrist, or the side of the equilibrimn polygon, pasies at one-sixth of the depth above or below the axis of the rib, the intensity of stress at that edge of the rib which is nearer the line of thrust will be twice the mean intensity.

If, again, the maximum intensity is to be thrice the mean, the line $F \mathrm{G}$, starting at a distance $\mathrm{B} F=3 \mathrm{~F}=3 \mathrm{D}$, still cuts $(1)$ at its middle point in order to make the total tension from bending moment equal to the total compression from the same cause. Noting where F (x euts $\lambda$ B, we have the point of no stress at ${ }_{4}^{3} h$ from the upper edge of the section: hence the shaded areas are drawn as given in the section on the right, the upper one for compression, the lower one for tension. 'The area of the upper one is $\frac{1}{2} h \cdot{ }_{4}^{3} h=\frac{3}{3} b h:$ the lower one, being similar, but of one-third the altitule, has one-ninth the area of the other, or $\frac{1}{2} \frac{1}{4} b h$. The difference is $\frac{1}{3} b h$, or one-third the area of the cross-section, as regnised if the maximmm intemsity is to be three times the mean. Letting these areas represent the forces, and taking moments about the rpper edge, each force being applied at the eentre of glavity of its triangle, we have for the position of the resultant, measured from the upper edge,

$$
\frac{\frac{3}{8} b h \cdot \frac{1}{4} h-\frac{1}{2+} h h \cdot \frac{1}{1} h}{\frac{1}{3} h h}=\frac{1}{6} h .
$$

If, therefore, the line of thrust passes at $\frac{1}{6} \mathrm{~h}$ from the efge, or one-third the depth from the axis, the intensity of compression on the outside fibre nearer the line will be three times the mean compression, and at the other edge there will be a tension erfual in magnitude to the mean stress.

In the same way it may be shown, that, when the line of thrust ents the edge, the compression there will be BI, four times the mean, and the tension at the other edge will be $A \mathrm{~K}$, twice the magnitude of the mean stress. 'Thus it will be seen,
that, for every one-sixth $h$ that the line of thrust is distant from the axis, the compression on the square inch will be increased by unity on the side to which the line deviates, and diminished by unity on the other side, the mean compression being denoted ly unity. This is indicated by the numerals marked on the sketches of Fig. 29.
107. Rib of Two Flanges. - If the rib is composed of two flanges and an open-work web, the stress in either flange is easily determined. If the line of thrust is in the axis, each flange will carry one-half of the direct stress. If the line of thrist passes throngh one flange, Fig. 30, that flange may be considered to carry all of the compression uniformly distributed, and the other flange to be under no stress; for the depth of the flange is so small, compared with the whole depth of the rib, that no error of importance is involved in considering the stress as uniformly distributed over the section of one flange. If the line of thrust passes without the rib a distance equal to its depth, we get, by taking moments at A, Fig. 30,

> Thrust at $\mathrm{C} \times 2 \mathrm{AB}=$ Compression at $\mathrm{B} \times \mathrm{AB} ;$ or, Compression at $\mathrm{B}=2 \times$ direct stress.

If moments are taken at $B$, we find,

$$
\text { Tension at } \mathrm{A}=\text { direct stress. }
$$

In the same way, if $\mathrm{B}^{\prime} \mathrm{C}^{\prime}=2 \mathrm{~A}^{\prime} \mathrm{B}^{\prime}$,
Compression at $\mathrm{B}^{\prime}=3 \times$ direct stress; Tension at $\mathrm{A}^{\prime}=2 \times$ direct stress.
Hence we may draw a sketch for this rib similar to the one for the rectangular rib. The numerals here denote that one flange carries once, twice, \&c., the entire direct stress. If the rib has a plate web, or is an I beam, the above method will give a good approximation to the true stresses. If the web is heary, the method of the next section may be applied.
108. Rib of Circular Section; General Construction. When the rib is of less simple section, we must return to the
graphical construction first referred to. As an instance, suppose the cross-section of the rib to be a circle. The variation of stress on a diameter, in the direction of deviation, is indicated by the left-land sketeh of Fig. 31, when the intensity of stress is twice the mean at one edge, and zero at the other. By constructing, according to the principles already laid down, Part I., "Roofs." the equivalent area of maximum intensity, we obtain the shaded area of the figure, and then we determine its centre of gravity by cutting out the area, and balancing it over a knifeedge. The deviation of the line of thrust from the centre of the circle, to make the maximum intensity twice the mean, and the minimum zero, is thus found, and proves to be onefourth the radius.

By the construction of the other sketch, taking moments an in $\S 106$, or reasoning by analogy, we find that the deviation, in order that the maximum shall be thrice the mean intensity of compression, and the tension at the other end of the diameter shall equal the mean stress, must be one-half the radius from the centre: hence, when the line of tlurust cuts the edge, the maximum compression equals five times the mean, and the tension at the other extreme of the diameter is three times the mean compression. Thus we get the numerals and their positions, as given in the figure.

In a thin tube of circular, elliptical, or oval section, the maximum compression is nearly three times the mean intensity of direct stress where the equilibrim polygon cuts the surface of the tube: and a tensile stress equal in magnitude to the mean will then be found at the other end of the extremity of the diameter: hence proportionate distances of the side of the equilibrium polygon from the axis of the rib will give twice, four times, \&e., the mean stress.

## CHAPTER VIII.

ARCHED RIBS UNDER WIND PRESSURE: HORIZONTAL FORCES.
109. Wind Pressure on an Inclined Surface. - When archerl ribs are used, as is often the case, for the support of a roof, the pressure of the wind, being normal to the surface, will have a different effect upon the arch from that cansed by a simple weight or vertical force. While referring to Part I., "Roofs," p. 31, for some remarks about the action of wind on a roof, we will repeat here, that, if P equals the horizontal force of the wind on a syuare foot of a vertical plane, the perpendicular pressure on a scquare foot of a surface inclined at an angle $i$ to the horizon may be expressed by the empirical formula, -

$$
P \sin i^{1.84 \operatorname{cosi} i-1} \text {. }
$$

If, then, the maximm force of the wind be taken as forty pounds per square foot, which is an amount sufficiently great for the purposes of a design, the perpendicular or normal pressure per square foot, on surfaces inclined at different angles to the horizon, will be : -

| Angle of <br> Roof. | Normal <br> Pressure. | Angle of <br> Roof. | Normal <br> Pressure. |
| :---: | :---: | :---: | :--- |
| $5^{\circ}$ | 5.2 lbs | $35^{\circ}$ | 30.1 lbs. |
| 10 | 9.6 | 40 | 33.4 |
| 15 | 14.0 | 45 | 36.1 |
| 20 | 18.3 | 50 | 35.1 |
| 25 | 22.5 | 55 | 39.6 |
| 30 | 26.5 | 60 | 40.0 |

For steeper pitehes, the pressure may be taken as forty pounds.

The resultant pressure at each of the joints in the rafter which is on the side of the wind is then ascertained as in the case of any roof. If the roof surface is curved, any short pordion between two points where braces abut, or purlins rest. maty be considered as straight, and the wind force will then be perpendieular to such portion ; this pressure being the only force exerted by the wind. If the resultant pressure at each juint is then fomd, either graphically or otherwise, and is resolved into vertical and horizontal components, we may include the rertical component in the analysis already carried out in cletail. The effect of the horizontal component remains to be considererl.
110. Form of the Equilibrium Polygon; Vertical Component of Reaction. - The tendency of such a force to distort the arch being resisted by the stiffness of the rib, the equilibrium polygon for a single horizontal foree H , applied at any point I on the rib, Fig. 32. must. if the arch is hinged at the ends, be two straight lines. which start from the two springing points, and meet on the prolungation of the line of action of H ; for the rib must be in equilibrium under H and the two forces at the abutments. In the case of the arch A C B of Fig. 32, the reactions at $A$ and $B$ must lie in the lines $A G$ and $B G$, the point $G$ being found on the horizontal line I G , but its location on that line being at present unknown. It will be evident, when we eoneeive H to be applied to the equilibrium polygon at $G$, that the side $A$ G will be in tension, while $(G B$ is compressed : therefore the reaction at $B$ will be a thrust, as usinal, but that at A will be a temsion: and. if If were the only applied force, the arch would tend to rise from the abutment $A$. and would require fastening down.

As $H$ acts at a vertical distance $I L$ above the springing line, the moment which tends to werturn the frame is H.IL. If we take either abutment as the axis of moments. the condition of equilibrium that the moments of exterior forces must balance
gives $\mathrm{H} . \mathrm{IL}=\mathrm{P} . \mathrm{AB}$; and consequently the vertical component of the reaction at either abutment is, -

$$
\mathrm{P}=\mathrm{H} \frac{\mathrm{IL}}{\mathrm{AB}},
$$

being tension at the side nearer to $I$, and compression on the other side. H will be partially resisted at each abutment. The stress diagram will be a figure like 123 , in which $3-4$ and $4-1$ are $-P$ and $H_{1}$ for $A$, while 2-4 and $4-3$ are $H_{2}$ and $+P$ for $B$, 1-2 being equal to H .
111. Rib hinged at Three Points. - As was the case with arches under vertical forces only, so also with ribs under a wind pressure: the hinging of the rib at three points makes the analysis at once very simple. If the arch of Fig. 32 is pivoted or jointed at $\mathrm{A}, \mathrm{C}$, and $\mathrm{B}, \mathrm{C}$ being usually taken at the crown of the rib, and the external horizontal force $H$ is applied at $I$, the line of thrust for the right-hand portion of the arch must be B C. This will be plainly seen, if we consider that the part B E C of the rib is supported by a reaction at B and the thrust of the other half of the arch at C , while there is no other force exerted upon it: for equilibrium, therefore, these two forces must lie in one straight line, which can be no other than B C, drawn through the two points of application. Then, as proved before, the reaction at A must lie in A G, drawn to the intersection of $H$ with $B C$. It may be noted that $1-4$, or $H_{1}$, is ahways greater that one-half of H.
112. Value of Bending Moments. - If we make a section at any point E on the right of C , the only force acting on the right of the section is the inclined reaction at the abutment $B$. The bending moment at E is, therefore, equal to (3-2) E N, or to either of the cqual products $\mathrm{H}_{2}$. E F and P.EK. The bending moment at any point between C and I, fur the same reason, will still be expressed by $\mathrm{H}_{2}$. E F or I'. E K, but will be of the opposite kind, since we passed a point of no bending moment at C. and E F or E K is drawn in a reverse direction. For sections between I and $A$ it will be easier to take the force on the left
of the plane of spction, which will be the tension of the left abutment, as this is the only force on that side: the bending moment will therefore be $\mathrm{H}_{1}$. E F or P . E K. It will be perceived, on a little reflection, that these moments will agree in kind with those between C and I; the reversal of the ordinate EF from the outside to the inside of the rib offectting the change from $\mathrm{H}_{2}$, compression, to $\mathrm{H}_{1}$, tension. The appliation of ${ }^{\circ}$ H at I to a moderately flexible wire of the shape $A$ ( $B$ womld flatten the left portion, and make the right portion more convex.

We may very simply consider the bending moment at any point of the rib to be represented by the prochuct P . E K, where EK is the horizontal distance or abscissat from E to the equilibrium polygon. We thus have an evident analogy between the erfuilibrium polygons for horizontal and for vertical forces, if the ordinate for bending moment is taken parallel to the applied force, and is then multiplied by a constant, P in this case, HI in the other. The point of contraflexure is where the polygon meets the ril, and one point of. maximum flexure is at $I$, the point of application of the external force.

The insertion of pivots at three points of the rib enables one to traw the equilibrimm polygon at once for one or all of the forces to which the roof may be at one time subjected, and we will therefore proceed, without further delay, to consider the case of the parabolic rib hinged at the abutments only.
113. Parabolic Rib hinged at Abutments; Formula for $x_{0}$ - If Fig. 33 represents a parabolic ribl linged at $A$ and $B$, with a horizontal force $H$ applied at I, the point of intersection of A G and B N must be determined. Since it will lie upon the horizontal line drawn through $I$, the distance of (i horizontally from the middle of the span will be denoted by $x_{0}$, positive when measured from the middle away from I. The well-known condition that change of span shall be zero mar be put either

[^2]or
\[

$$
\begin{equation*}
\text { P. } \Sigma \mathrm{EK} \cdot \mathrm{DE}=0, \tag{1.}
\end{equation*}
$$

\]

If $l$ denotes the horizontal distance of I from the middle of the span, and $c$ the half-span,

$$
\begin{aligned}
& x_{0}=\frac{b^{3}}{4 c^{4}}\left(5 c^{2}-b^{2}\right)=\frac{1}{3} n^{3}\left(5-n^{2}\right) c, \\
& \begin{array}{lllll}
n= & 0 & 0.2 & 0.4 & 0.6
\end{array} \\
& \begin{array}{lllll}
x_{0}= & 0.01 & 0.0 \pi & 0.25 & 0.56 c,
\end{array}
\end{aligned}
$$

When $b=n c$. We shall see that $x_{0}$ is always laid off on the oposite side of the eentre from $b$, and henee that $\mathrm{H}_{1}$, the horizontal tension, is always greater than one-half of II. The value of $x_{0}$ is independent of $k$.
114. Proof of Formula. - Retaining the usual notation, we have $\mathrm{A} \mathrm{L}=c-b ; \mathrm{L} \mathrm{B}=c+b$; and $\mathrm{G} \mathbf{Q}=\mathrm{I} \mathrm{L}=\frac{k}{c^{2}}\left(c^{2}-b^{2}\right)$. If $x$ denotes the horizontal distance, B D, to the abutment, from any ordinate, D E, on the right of I we have
$\mathrm{DE}=\frac{k}{c^{2}}\left(2 c x-x^{2}\right)$, and $\mathrm{DF}: \mathrm{DB}=\mathrm{GQ}: \mathrm{QB}$, or $\mathrm{D} \mp=\frac{k}{c^{2}}\left(c^{2}-b^{2}\right) \frac{x}{c-x_{0}}$.
As $\mathrm{E} \mathrm{K}: \mathrm{EF}=\mathrm{Q} P: \mathrm{G}$ Q , and $\mathrm{EF}=\mathrm{DE}-\mathrm{DF}$, we have

$$
E K=(D E-D F) \frac{Q B}{G Q}, \text { and } E K \cdot D E=\left(D E^{2}-D E \cdot D F\right) \frac{Q B}{G Q}
$$

Sulstituting the values of these quantities, we get

$$
\text { ェEK.DE }=\int \frac{k}{c^{2}}\left[\left(2 c x-x^{2}\right)^{2}-\left(2 c x-x^{2}\right) x \frac{c^{2}-b^{2}}{c-r_{0}}\right] \frac{c-x_{0}}{c^{2}-b^{2}} d x
$$

as the expression which is applicable from B to I. From A to I the abscissa Elf will be limited by the line A $G$, which differs in inclination from B C. If $r$, however, is now reckoned from $A$ to the right, and $A(Q$, denoted by $r+x_{0}$, is used in place of $Q B$, we have an expression for the space from A to I. This expedient was used in previous sections. As A G is in tension while B (' is compresserl, these two portions of (1.), $\S 113$, will have olprosite signs, and. when integrated, must be equal : we may, therefore, in equating, strike out the common constant quantities, obtaining
$\left(c-x_{0}\right) \int_{0}^{c+b}\left(4 c^{2} x^{2}-4 c x^{3}+x^{4}\right) d x-\left(c^{2}-b^{2}\right) \int_{0}^{c+b}\left(2 c x^{2}-x^{3}\right) d x$
$=\left(c+x_{0}\right) \int_{n}^{c-b}\left(4 c^{2} x^{2}-4 c x^{3}+x^{4}\right) d x-\left(c^{2}-b^{2}\right) \int_{0}^{c-b}\left(2 c x^{2}-x^{3}\right) d x$.

Performing the indicated integration, we get
$\left(c-x_{0}\right)\left[\frac{1}{3} c^{2}(c+b)^{3}-c(c+b)^{4}+\frac{1}{5}(c+b)^{5}\right]-\left(c^{2}-b^{2}\right)\left[\frac{2}{3} c(c+b)^{3}-\frac{1}{4}(c+b)^{4}\right]$
$=\left(c+x_{0}\right)\left[{ }_{3}^{4} c^{2}(c-b)^{3}-c(c-b)^{4}+\frac{1}{5}(c-b)^{5}\right]-\left(c^{2}-b^{2}\right)\left[\frac{2}{3} c(c-b)^{3}-\frac{1}{4}(c-b)^{4}\right]$,
which at once reduces to

$$
\frac{166}{15} c^{5} x_{0}=\frac{4}{3} c^{3} b^{3}-\frac{4}{15} c b^{5},
$$

or

$$
x_{0}=\frac{l^{3}}{4 \frac{3}{r^{4}}}\left(5 c^{2}-b^{2}\right) .
$$

115. Another Proof. - We may, if we please, find the desired distance $x_{0}$ by another method. Imagine the roof of Fig. 34 to have two equal but opposite forces. II, applied at the two points. C and (i in the same horizontal line. These forces, if acting alone, will tend to diminish the pan of the roof: there will be no rertical forces; and as the bending moments cansed by then. in vase the rib did not rest upon abutments. would be directly proportional to E F , the change of span woukl be proportional to $\perp$ EF. I) E from (' to (i. When the rilh is retained by abutments, one $I f$ will give rise to $H_{1}$ at A , and $\mathrm{H}_{2}$. at $B$ : the other $H$ will catuse $H_{2}$ at A . and $I_{1}$ at B . As $H_{1}$ is alwars "pposite in sign to $H_{2}$, the resultant foree at each abntment will be $\mathrm{H}_{1}-\mathrm{H}_{2}$, and is manifestly a tension exerted by the abutment on the rib. The change of span due to $H_{1}-H_{2}$ will be proportional to $\Sigma \mathrm{D} \mathrm{E}^{2}$ from A to $B$ (compare $\underset{\text { S }}{ } \mathrm{T} t$ ), and this change of span must offset the one from H.

If D is at a distance $x$ from the middle of the span, and (: is distant $b$ from the same point. we have $D \mathrm{E}=\frac{k}{t^{2}}\left(r^{2}-r^{2}\right)$, and $\mathrm{E} \mathbf{F}={ }_{r^{2}}^{k}\left(l^{2}-x^{2}\right)$. Since the rib is acted upon symmetrically, We need only integrate from the middle to one side: and we therefore have, when we drop the common factor $\frac{k}{c^{2}}$

$$
\left(\mathrm{H}_{1}-\mathrm{H}_{2}\right) \int_{0}^{c}\left(c^{2}-x^{2}\right)^{2} l x=\mathrm{H} \int_{0}^{b}\left(l^{2}-x^{2}\right)\left(c^{2}-x^{2}\right) d x
$$

or

$$
\left(\mathrm{H}_{1}-\mathrm{H}_{2}\right) \frac{8}{15} c^{5}=\mathrm{H}\left(\frac{2}{3} b^{3} c^{2}-\frac{2}{15} c^{5}\right)
$$

From the stress cliagram of Fig. 33 we see that

$$
\mathrm{H}_{1}: \mathrm{H}_{2}: \mathrm{H}=c+x_{0}: c-x_{0}: 2 c ;
$$

whence

$$
\mathrm{H}_{1}-\mathrm{H}_{2}=\mathrm{H} \frac{c+x_{0}-c+x_{0}}{2 c}=\mathrm{H} \frac{x_{0}}{c} .
$$

Substituting this value in (a.) we get, as before, § 114 ,

$$
x_{0}=\frac{b^{3}}{4 c^{4}}\left(5 c^{2}-b^{2}\right)
$$

116. Formulæ for $\mathbf{H}_{1}$ and $\mathbf{P}$. - The value of $H_{1}$ is seen to be. from the above proportion,

$$
\mathrm{H}_{1}=\mathrm{H} \frac{c+x_{0}}{2 c}=\mathrm{H}\left(\frac{1}{2}+\frac{x_{0}}{2 c}\right)=\mathrm{H}\left[\frac{1}{2}+\frac{b^{3}}{8 c^{5}}\left(5 c^{2}-b^{2}\right)\right] .
$$

We also have. from Fig. 33,

$$
\mathrm{P}: \mathrm{H}=\mathrm{G} \mathrm{Q}: \mathrm{AB}=\frac{k}{c^{2}}\left(c^{2}-b^{2}\right): 2 c
$$

or

$$
\mathrm{P}=\mathrm{I} \frac{k}{2 c^{3}}\left(c^{2}-b^{2}\right)=\mathrm{I} \frac{k}{2 c}\left(1-n^{2}\right)
$$

The reader may now calculate, if desirable, numerical values of $x_{0}, H_{1}$, and P , for different values of $b$, as was previously done for vertical forces. The several values of $x_{0}$ for four different positions of H are plotted in Fig. 83.
117. Shear and Direct Stress. - The shear will undergo some modification when the force applied to the arch acts horizontally, instead of vertically. The stress diagram is, as we have seen, a triangle, whose base is $I I$, and whose altitude is $P$, represented by 012 of Fig. 36. At $A$ of the parabolic rib the thrust is $1-0$ : if $1-4$ is drawn parallel to the tangent at $A$, and $0-8$ perpendicular to it, $1-8$ will be the direct thrust, and $8-0$ the negative shear, on a right section at $A$. This shear will
diminish at suceessive sections until we reach a point where the tangent to the rib is parallel to $A(\mathbf{i}$, when the shear will be zero, and the direct thrmst 1-0. Beyond this point the shear will be positive until we pass $I$. At the abutment $B$, there is a tension $2-0$ : if $2-7$ is drawn parallel to the tangent at $B, 2-9$ will be the direct tension, and $9-0$ the shear, again negative, on a right seetion at $B$. In the same way the shear just to the left of $I$ will be $10-0$, positive, and to the right of $I, 11-0$, negative. It will be remembered that positive shear acts upward on the left of any section.
118. Shear Diagram. - A shear diagram may be drawn for a rib under a horizontal force by a similar method to the one previously explained, showing the vertical shear which will be projected on each right section. Lay off at a the quantity $\mathrm{P}=3-0=a f$, which is the vertical component of the reaction at A , and as P is constant across the entire span, being, in fate, the only external vertical force, complete the rectangle "f $d b$. The vertical component which is required at $A$ to produce $1-4$ is $3-4$, laid off at,$e$; and at $B$ is $3-7$. laid off above the line at $b l$, because $0-2$ is a tension. A load of uniform intensity horizontally being required to put a parabolic rib in equilibrium, and $\mathrm{H}_{1}$ being constant as far as I , draw $e c g$ through $c$, the middle point of $a b$, and draw $l n$ so as to pass through $c$, if prolonged. Then will the vertical ordinates between the inclined lines and $f d$ represent the shear on a vertical section, and the projection of these ordinates on the respective normal sections will be the shear in the web. Thus ef is $4-0$, which gives by projection $8-0, i y$ is $0-5$. and $i n$ is $0-6$. As in previous diagrams, the ordinates will be measured from the inclined lines, positive above and negative below, as marked. The shear will change sign at the point of maximum bending moment, and it will plainly be equal to P at the crown of the arch.

If it is remembered that the abutment reaction at $B$ is of the opposite kind to that at $A$, or to the usual reaction for a weight $W$, the rotation of the diagram on the right of $i$, from the customary position below the line to its present place above
ab, will be accounted for. The force Il has been assumed on the right in Fig. 36, in order that this shear diagram may be compared with that of Fig. 8 . The vertical shear from a normal force may be found from an addition of these two figures. Moment diagrams cannot be added together in the same way, as the values of H and $\mathrm{H}_{1}$ or $\mathrm{H}_{2}$ will not be the same in the two cases.
119. Circular Rib hinged at Ends. - The method of tinding $x_{0}$. introduced in $\$ 115$, is easily applied to the circular rib hinged at the ends: while the process of $\$ \mathbf{S} 114$ is considerably more involved. Let the angle subtended, in Fig. 35, by the half arcla of radins $r$ be denoted by $\beta$; the angle from the crown to the point of application of the external horizontal force, H , be $\varepsilon$; and the variable angle from the crown to any point be $\theta$. Let $H$ be applied at two opposite points at the same level, as shown by the arows in the figure, and let the abntment reactions be $\mathrm{H}_{1}-\mathrm{H}_{2}$. Then, by parallel reasoning to that of § 115, we have, if $y$ denotes any ordinate, and $a$ the ordinate to the point of application of H ,

$$
\begin{gathered}
\left(\mathrm{H}_{1}-\mathrm{H}_{2}\right) \int_{0}^{3} y^{2} d s=\mathrm{H} \int_{0}^{a}(y-a) y d s . \\
y=r(\cos \theta-\cos \beta) ; a=r(\cos a-\cos \beta) ; \therefore \\
\left(\mathrm{H}_{1}-\mathrm{H}_{2}\right) r^{3} \int_{1}^{3}\left(\cos ^{2} \theta-2 \cos \theta \cos \beta+\cos ^{2} \beta\right) d \theta \\
=\mathrm{H} r^{3} \int_{1}^{a}\left(\cos ^{2} \theta-\cos \theta \cos \beta-\cos \theta \cos a+\cos a \cos \beta\right) d \theta .
\end{gathered}
$$

Performing the integration, we get

$$
\begin{gathered}
\quad\left(\mathrm{H}_{1}-\mathrm{H}_{2}\right)\left(\frac{1}{2} \beta-\frac{3}{2} \sin \beta \cos \beta+\beta \cos ^{2} \beta\right) \\
=\mathrm{H}\left(\frac{1}{2} a-\frac{1}{2} \sin a \cos a-\sin a \cos \beta+a \cos a \cos 3\right) .
\end{gathered}
$$

- As in $\S 115, x_{0}=\frac{\mathrm{H}_{1}-\mathrm{H}_{2}}{H} c=\frac{\mathrm{H}_{1}-\mathrm{H}_{2}}{H} r \sin \beta$ : whence

$$
\begin{equation*}
x_{0}=r \sin \beta \frac{a-\sin a \cos a-2 \cos \beta(\sin a-a \cos a)}{\beta-3 \sin \beta \cos \beta+2 \beta \cos ^{2} \beta} . \tag{1.}
\end{equation*}
$$

If the rib is a semicircle, $\beta^{\prime}=\frac{1}{2} \pi ; \cos ^{\beta}=0 ; \sin \beta=1$; and (1.) becomes,

$$
\begin{equation*}
x_{0}=\frac{2 r}{\pi}(u-\sin a \cos a) . \tag{2.}
\end{equation*}
$$

120. Formula for $\mathbf{H}_{1}$ and $\mathbf{P}$. - The value of $\Pi_{1}$ will be, as in § 116 ,

$$
\begin{gathered}
\mathrm{H}_{1}=\mathrm{H} \frac{c+x_{0}}{2}=\mathrm{H}\left(\frac{1}{2}+\frac{x_{0}}{2 r \sin \beta}\right) \\
=\frac{1}{2} \mathrm{H}\left(1+\frac{a-\sin a \cos a-2 \cos \beta(\sin a-a \cos a)}{3-3 \sin \beta \cos \beta+2 \beta \cos ^{2} \beta}\right),
\end{gathered}
$$

and

$$
\mathrm{P}=\frac{\mathrm{H} a}{2 c}=\frac{\cos a-\cos \beta}{2 \sin 3} \mathrm{II} ;
$$

or, for a complete semicircle.

$$
\mathrm{H}_{1}=\frac{\frac{1}{2} \pi+a-\sin a \cos a}{\pi} \mathrm{H} ; \quad \mathrm{P}=\frac{1}{2} \cos a \mathrm{H} .
$$

121. Experimental Verification. - The valuen of $x_{v}$, obtained above, can be readily shown to be true by turning the model previonsly referred to through an angle of ninety degrees. A moderately stiff wire carefully bent to a curve A (i B. Fig. 37, symme rical with regard to the point (r (an are of a direle being probably the easiest one to fashion). is suspended from prints C and D ley strings from A to C and from B to D. If the string B D is cloubled sor as to pass on both sides of the wire above G, A GB will be presented from swinging round. A thread from A to B will hinder the span from enlarging, and will indicate ly its slackening when the opan is narrowerl. If, then, a weight is attached at E. and the string at (' remaining stationary, that at D is mored until B is vertically helow A , as proved hy plumbing the threat AB, C A. when prolonged. will be found to intersect BD at Fin the vertical line E F, giving the desired value of $x_{n}$. The puint of intersection will be slightly changed hy the weight of the wire as before suggested in $\$ 81$. It is wortly of unte that. II now leing an external pull on the rilh, in phace of the usisal thrust. $x_{0}$ will, in Fig. 37, be found on the simme side of the centre with H .
122. Parabolic Rib fixed at Ends; Formulæ for $x_{0}, x_{1}$, and $x_{2}$. - Referring to Fig. 38, we will suppose that the external force $H$ is applied at $I$, on the left of this parabolic rib with fixed ends; that the desired equilibrium polygon is given by the lines $\mathrm{L}, \mathrm{G}$ and NGG ; and that the abscissæ, at present unknown, are, $\mathrm{A} \mathrm{L}=x_{1}, \mathrm{~B} \mathrm{~N}=x_{2}$, and $\mathrm{O} \mathrm{Q}=x_{0}$, the latter being measured from the middle of the span, and all being eonsidered as positive when laid off as shown in this figure. The rest of the notation agrees with that used before. It may be proved that the abscissæ have the following easily computed values:

$$
x_{1}=\frac{1}{3}\left(c+\frac{4 b^{2}}{c-l}\right) ; x_{2}=\frac{1}{3}\left(c+\frac{4 b^{2}}{c+b}\right) ; x_{0}=2 \frac{b^{3}}{c^{2}},
$$

or

$$
x_{1}=\frac{1}{3} c\left(1+\frac{4 n^{2}}{1-n}\right) ; x_{2}=\frac{1}{3} c\left(1+\frac{4 n^{2}}{1+n}\right) ; x_{0}=2 n^{8} c .
$$

Several of these values, for different positions of II, are plotted in Fig. 38.

If $b$ is given successive values from $0.1 c$ to $0.9 c$, these quantities will be found to be

| $\ell$. | $x_{1}$. | $x_{0}$. | $x_{2}$. |
| :---: | :---: | :--- | :---: |
| $0.1 c$ | $0.35 c$ | $0.002 c$ | $0.35 c$ |
| .2 | 0.40 | 0.016 | 0.38 |
| .3 | 0.50 | 0.054 | 0.43 |
| .4 | 0.69 | 0.128 | 0.49 |
| .5 | 1.00 | 0.250 | 0.56 |
| .6 | 1.53 | 0.432 | 0.63 |
| -.7 | 2.51 | 0.688 | 0.72 |
| .8 | 4.60 | 1.024 | 0.81 |
| .9 | 11.17 | 1.442 | 0.90 |

If $l$ exceeds $0.7 c$, the point of intersection falls without the rib.
123. First Equation of Condition. - If we remark that Q G, Fig. 38, the ordinate to the line of action of $H$, will be equal to $I S$, or to $\frac{k}{c^{2}}\left(c^{2}-b^{2}\right)$, and that $\mathrm{RK}=\mathrm{DE}$, we may find the value of EK as follows:
$\mathrm{EK}=\mathrm{RN}-\mathrm{DN} ; \mathrm{RN}: \mathrm{RK}=Q N: Q G$, or $\mathrm{RN}=\frac{\mathrm{RK} \cdot Q N}{Q G} ;$
therefore

$$
\left.\mathrm{EK}=\frac{\mathrm{DE} \mathrm{E} \cdot(\mathrm{QN}}{(\mathrm{Q} G}-\mathrm{D}\right) \mathrm{N} .
$$

These quantities, in the notation employed, may be written, if $x$ is measured from the right abutment.

1) $\left.\mathrm{E}=\frac{k}{c^{2}}\left(2 c x-x^{2}\right) ; \mathrm{QN}=c+x_{2}-x_{0} ; \mathrm{D}\right) \mathrm{N}=x_{2}+x ; \mathrm{Q} \mathrm{G}=\frac{k}{c^{2}}\left(c^{2}-l^{2}\right)$.

As $\frac{k}{c^{2}}$ will be a common factor in the equations which follow, we shall omit it. Substituting these values, we shall get, as the expression to be summed from $B$ to I, for the first condition.

$$
\text { こEK. NE }=\int_{11}^{c+b}\left[\frac{c+x_{2}-r_{0}}{c^{2}-1,2^{2}}\left(4 c^{2} x^{2}-4 c x^{3}+x^{4}\right)-\left(x_{2}+x\right)\left(2 c x-x^{2}\right)\right] d x
$$

If $x$ is measured from the left abutment. L Q substituted for Q N, and $x_{1}$ written for $x_{3}$, we get an expression which is applicable from A to I, or

$$
\text { こEK. I) } \mathrm{E}=\int_{0}^{4-b}\left[\frac{c+x_{1}+x_{0}}{c^{2}-b^{2}}\left(4 c^{2} x^{2}-4 c^{x^{3}}+x^{4}\right)-\left(x_{1}+x\right)\left(2 c x-x^{2}\right)\right] d x
$$

As in § 114 , these two expressions will be equated to make the change of span zero, and upon performing the indicated integrations, and multiplying through by $c^{2}-l^{2}$, we obtain

$$
\begin{aligned}
& \left(c+x_{2}-x_{0}\right)\left[\frac{4}{3} c^{2}(c+l)^{3}-c(c+l)^{4}+\frac{1}{5}(c+l)^{5}\right]-\left(c^{2}-l^{2}\right)\left[c^{2} c_{2}(c+l)^{2}\right. \\
& \quad-\frac{1}{3} x_{2}(c+l)^{3}+\frac{2}{3} c\left(c+l^{3}-\frac{1}{4}(c+l)^{4}\right]=\left(c+l_{1}+x_{0}\right)\left[\frac{1}{3} c^{2}(c-l)^{3}\right. \\
& \left.\quad-c(c-l)^{4}+\frac{1}{5}(c-l)^{5}\right]-\left(c^{2}-l^{2}\right)\left[c x_{1}(c-l)^{2}-\frac{1}{3} x_{1}(c-l)^{3}\right. \\
& \left.\quad+\frac{2}{3} c(c-l)^{3}-\frac{1}{4}(c-l)^{4}\right] .
\end{aligned}
$$

This equation, by reduction and factoring, may be written.

$$
\begin{gathered}
8 c^{5} x_{0}-\left(c^{5}-5 c^{3} b^{2}+5 c^{2} l^{3}-l^{5}\right) x_{1}+\left(c^{5}-5 c^{3} l^{2}-5 c^{2} l^{3}+b^{5}\right) x_{2} \\
=10 c^{3} b^{3}-2 c l^{5} . \quad(1 .)
\end{gathered}
$$

124. Second and Third Equations of Condition. - The seconl condition, that the change of inchination at the abutments shall equal zero, is $\Sigma \mathrm{EK}=0$, and the portion of this expression from l to I will be,

$$
\mathrm{\Sigma} \mathrm{E} \mathrm{~K}=\int_{0}^{c+t}\left[\frac{c+x_{2}-x_{0}}{c^{2}-b^{2}}\left(2 c x-x^{2}\right)-\left(x_{2}+x\right)\right] d x
$$

while from $\Lambda$ to I we may write, as just explained,

$$
\Sigma \mathrm{E} \mathrm{~K}=\int_{0}^{c-b}\left[\frac{c+x_{1}+x_{0}}{c^{2}-b^{2}}\left(2 c x-x^{2}\right)-\left(x_{1}+x\right)\right] d x .
$$

Equating, integrating, and reducing, we get

$$
\begin{gathered}
\left(c+x_{2}-x_{0}\right)\left[c(c+b)^{2}-\frac{1}{3}(c+b)^{3}\right]-\left(c^{2}-b^{2}\right)\left[x_{2}(c+b)+\frac{1}{2}(c+b)^{2}\right] \\
=\left(c+x_{1}+x_{0}\right)\left[c(c-b)^{2}-\frac{1}{3}(c-b)^{3}\right] \\
\quad-\left(c^{2}-b^{2}\right)\left[r_{1}(c-b)+\frac{1}{2}(c-b)^{2}\right] ;
\end{gathered}
$$

or

$$
\begin{equation*}
4 c^{3} x_{0}-\left(c^{3}-3 c b^{2}+2 b^{3}\right) x_{1}+\left(c^{3}-3 c l^{2}-2 b^{3}\right) x_{2}=4 c l^{3} . \tag{1.}
\end{equation*}
$$

In writing the third condition, that the abutment deflection shall equal zero, or $\leq E K . D B=0$, we must, if we use the valnes of $E K$ already adopted, make $D$ B equal to $x$ on the right of I , and equal to $2 c-x$ on the left of I. We then have, from B to I,

$$
\int_{n}^{c+b}\left[\frac{c+x_{2}-x_{0}}{c^{2}-b^{2}}\left(2 c x^{2}-x^{3}\right)-\left(x_{2}+x\right) x\right] d x
$$

and from A to I,

$$
\int_{0}^{c-b}\left[\frac{c+x_{1}+x_{0}}{c^{2}-l^{2}}\left(4 c^{2} x-4 c x^{2}+x^{3}\right)-\left(x_{1}+x\right)(2 c-x)\right] d x
$$

Equating these two expressions and integrating, we find that

$$
\begin{gathered}
\left(c+x_{2}-x_{0}\right)\left[\frac{2}{3} c(c+b)^{3}-\frac{1}{4}(c+b)^{4}\right]-\left(c^{2}-b^{2}\right)\left[\frac{1}{2} x_{2}(c+b)^{2}+\frac{1}{3}(c+b)^{8}\right] \\
\quad=\left(c+x_{1}+x_{0}\right)\left[2 c^{2}(c-b)^{2}-\frac{4}{3} c(c-b)^{3}+\frac{1}{4}(c-b)^{4}\right] \\
\quad-\left(c^{2}-b^{2}\right)\left[2 c x_{1}(c-b)+\frac{1}{2}\left(2 c-x_{1}\right)(c-b)^{2}-\frac{1}{3}(c-b)^{3}\right] .
\end{gathered}
$$

which reduces to

$$
\begin{gathered}
16 c^{4} x_{0}-\left(7 r^{4}-18 c^{2} b^{2}+8 c l^{3}+3 b^{4}\right) x_{1}+\left(c^{4}-6 c^{2} l^{3}-8 c b^{3}-3 l^{4}\right) x_{3} \\
=-2 c^{5}-4 c^{3} b^{2}+16 c^{2} l^{3}+6 c b^{4} .
\end{gathered}
$$

From (1.). §123, and (1.) and (2.) of the present section, we may readily eliminate $x_{0}$, obtaining

$$
\left(c^{3}-b^{3}\right) x_{1}-\left(r^{3}+b^{3}\right) x_{2}=2 c b^{3} .
$$

and

$$
\left(c^{2}-b^{2}\right) x_{1}+\left(r^{2}-l^{2}\right) x_{2}=\frac{2}{3} c^{3}+2 c b^{2}
$$

whence may be deduced the formulix of $\S 122$.
125. Formulæ for $\mathbf{H}_{1}$ and $\mathbf{P}$. - The values of $\mathrm{H}_{1}, \mathrm{H}_{2}$, and P, can now be scaled from the stress diagram, which will also give, if preferred, the proportion

$$
\mathrm{H}_{1}: \mathrm{H}_{2}: \mathrm{H}=c+x_{1}+x_{0}: c+x_{2}-x_{0}: 2 c+x_{1}+x_{3}
$$

or
$\mathrm{H}_{1}=\mathrm{H} \frac{c+x_{1}+x_{0}}{2 c+x_{1}+x_{2}}=\mathrm{H}\left[\frac{1}{2}+\left(5 c^{2}-3 b^{2}\right) \frac{b^{3}}{4 c^{5}}\right]=\frac{1}{2} \mathrm{H}\left[1+\frac{1}{2} n^{3}\left(5-3 n^{2}\right)\right]$.
$\mathrm{H}_{1}$ will therefore always be greater than ${ }_{2}^{1} \mathrm{H}$.
Likewise we have, for the vertical component of the abutment reactions.

$$
\mathrm{P}: \mathrm{H}=\frac{k}{c^{2}}\left(c^{2}-b^{2}\right): 2 c+x_{1}+x_{2} .
$$

or

$$
\mathrm{P}=\mathrm{H} \cdot \frac{3}{8} k \frac{\left(c^{2}-l^{2}\right)^{2}}{c^{5}}=\frac{3}{8} \mathrm{H} \frac{k}{c}\left(1-n^{2}\right)^{2} .
$$

The shear diagram for this case will follow the explanation given in § 118.
126. Circular Arch fixed at Ends. - There remains to be considered the circular rib, fixed at the ends, under the action of an external horizontal force. The notation of the angles is the same as that previously used for the circular arch. As II is here applied at a point on the right side. $x_{0}$, measured from the middle of the span. will now lie on the left of the centre 0 . Then we will prove that

$$
\begin{align*}
& x_{1}=\left[\frac{f}{a}-\frac{a b-d e}{a c}\left(\frac{f}{a}+\sin \beta\right)\right] r ;  \tag{1.}\\
& x_{2}=\left[\frac{f}{a}+\frac{a b-d e}{a c}\left(\frac{f}{a}+\sin \beta\right)\right] r ; \tag{2.}
\end{align*}
$$

in which equations

$$
\begin{array}{ll}
a=\cos a-\cos \beta, & d=\beta \sin a-a \sin \beta, \\
b=a \beta-\sin a \sin \beta, & e=1-\cos a \cos \beta, \\
c=\beta^{2}-2 \sin ^{2} \beta+\beta \sin \beta \cos \beta, & f=\beta-\cos a \sin \beta .
\end{array}
$$

It will be noticed that $c$ is constant for a given arch. The value of $x_{0}$ can then be obtained from the equation

$$
\begin{gathered}
2(\sin \beta-\beta \cos \beta) x_{0}-[\sin \beta+\sin a-(\beta+a) \cos a] x_{1}+[\sin \beta-\sin a \\
-(\beta-a) \cos a] x_{2}=2 r \sin \beta(\sin a-a \cos a)
\end{gathered}
$$

The distance $x_{1}$ and $x_{2}$ will, in every case, be laid off outwards from the abutments, and $x_{0}$ will be plotted away from the side where the force is applied. In these formula, $x_{1}$ is on the oppusite side of the arch from the applied force, as is also $H_{1}$. In any case it is easy to distinguish between numerical values of $x_{1}$ and $x_{2}$, or $\mathrm{H}_{1}$ and $\mathrm{H}_{2}$, if we notice that the larger value belong: to the abntment which is nearer to the point of application of the external force.

Sereral of the eqnilibrium polygons have been drawn in Fig. 39 for a horizontal force applied at different distances from the erown. The angle $\beta$ of this rib) is $60^{\circ}$ : and the computed values of the absicissie, for II at points distant $10^{\circ}$ successively from one amother, are

| $a$. | $x_{1}$. | $x_{0}$. | $x_{2}$. |
| :--- | :--- | :--- | :--- |
| $10^{\circ}$ | $.3704 r$ | .0186, | $.4212 r$ |
| 20 | .475 .5 | .0762 | .5860 |
| 30 | .5892 | .2517 | 1.0345 |
| 40 | .7291 | .59 .50 | 2.1559 |
| 50 | .8649 | 1.1339 | 5.99 .3 |

127. First Equation of Condition. - The processes to be followed are akin to those already given: althongh the work is somewhat more tedions, it presents no difficulty. As in $\S 123$, we shall find that, Fig. 39, $\mathrm{EK}=\mathrm{RN}-\mathrm{DN}=\frac{\mathrm{DE} \cdot \mathrm{QN}-\mathrm{DN} .}{\mathrm{QG}-\text { In the usual notation }}$

$$
\begin{array}{ll}
\text { 1) } \mathrm{E}=r(\cos \theta-\cos \beta), & \mathrm{QN}=r \sin \beta+x_{2}+x_{0}, \\
(\mathrm{Q}(\mathrm{~B}=r(\cos \alpha-\cos \beta), & \mathrm{DN}=r \sin \beta+x_{2}-r \sin \theta .
\end{array}
$$

We therefore have

$$
\mathbf{E K}=\frac{r \sin \beta+x_{2}+x_{0}}{\cos \alpha-\cos \beta}(\cos \theta-\cos \beta)-\left(r \sin \beta+x_{2}-r \sin \theta\right)
$$

on the right of I . [pon the left of I. since E ' K now equal. () L - I L L , this expression will change in sign; aml, since we measure from $L$, we must substitute $x_{1}$ in place of $x_{2}$, mmst subtract $i_{0}$ in place of adding it, and must change the sign of $r$ sin $n$ : hence. on the left of $l$,

$$
\mathrm{E} \mathbf{K}=-\underset{r \sin \beta+\cdots-r_{0}(\cos \theta-\cos \beta)+\left(r \sin \beta+x_{1}+r \sin \theta\right) .}{r} \text {. }
$$

The first condition, invariability of span, will now give,

$$
\Sigma_{a}^{\prime 3} \mathrm{E} \text { K. DE }+\mathrm{E}_{-\beta}^{a} \mathrm{E} \mathrm{~K} . \mathrm{DE}=0,
$$

or, multiplying by $\cos a-\cos \beta$,

$$
\begin{aligned}
& r \int_{a}^{\beta}\left[\left(r \sin \beta+x_{2}+x_{0}\right)\left(\cos ^{2} \theta-2 \cos \theta \cos \beta+\cos ^{2} \beta\right)\right. \\
& \left.\quad-(\cos a-\cos \beta)\left(r \sin \beta+x_{2}-r \sin \theta\right)(\cos \theta-\cos \beta)\right] d \theta \\
& \quad+r \int_{-\beta}^{a}\left[\left(r \sin \beta+x_{1}-x_{0}\right)\left(-\cos ^{2} \theta+2 \cos \theta \cos \beta-\cos ^{2} \beta\right)\right. \\
& \left.\quad+(\cos a-\cos \beta)\left(r \sin \beta+x_{1}+r \sin \theta\right)(\cos \theta-\cos 3)\right] d \theta=0 .
\end{aligned}
$$

The integration is similar to that already given for the circular rik in the earlier sections. There results, upon bringing together common factors.

```
\(\left(3-3 \sin 3 \cos \beta+2 \beta \cos ^{2} \beta\right) x_{0}-\left(\frac{1}{2} \beta+\frac{1}{2} a-\frac{1}{2} \sin \beta \cos 3-\frac{1}{2} \sin a \cos \alpha\right.\)
    \(-\sin a \cos \beta-\cos a \sin \beta+\beta \cos a \cos \beta+a \cos a \cos \beta) x_{1}\)
    \(+\left(\frac{1}{2} \beta-\frac{1}{2} a-\frac{1}{2} \sin \beta \cos \beta+\frac{1}{2} \sin a \cos a+\sin a \cos \beta-\cos a \sin \beta\right.\)
    \(+3 \cos a \cos 3-a \cos a \cos 3) x_{2}=r \sin \beta(a-\sin a \cos a-2 \sin a \cos 3\)
    \(+2 a \cos a \cos 3)\).
129. Second and Third Equations of Condition. - The second condition, that \({\underset{a}{a}}_{3}^{\mathrm{E}} \mathrm{K}+\mathrm{s}_{-3}^{a} \mathrm{EK}=0\). similarly gives.
\[
\begin{aligned}
\int_{a}^{3}[(r \sin \beta & \left.\left.+x_{2}+x_{0}\right)(\cos \theta-\cos \beta)-(\cos a-\cos \beta)\left(r \sin \beta+x_{2}-r \sin \theta\right)\right] d \theta \\
& +\int_{-\beta}^{a}\left[\left(r \sin \beta+x_{1}-x_{0}\right)(-\cos \theta+\cos \beta)\right. \\
& \left.+(\cos a-\cos \beta)\left(r \sin \beta+r_{1}+r \sin \theta\right)\right] d \theta=0 .
\end{aligned}
\]

From this equation we obtain. by integrating and factoring.
\[
\begin{align*}
& (2 \sin \beta-2 \beta \cos \beta) x_{0}-(\sin \beta+\sin a-\beta \cos \alpha-a \cos a) x_{1} \\
+ & (\sin \beta-\sin a-\beta \cos a+a \cos \alpha) x_{2}=r \sin \beta(2 \sin a-2 a \cos a) . \tag{1.}
\end{align*}
\]

The third condition, that \(\Sigma_{a}^{\beta} \mathrm{EK}\) K.D B \(+\Sigma_{-\beta}^{a} \mathrm{EK} . \mathrm{DB}=0\), will give, when we introduce the value of \(\mathrm{D} \mathrm{B}=r(\sin \beta-\sin \theta)\),
\[
\begin{aligned}
& r \int_{a}^{\beta}\left[\left(r \sin \beta+x_{2}+x_{0}\right)(\cos \theta-\cos \beta)(\sin \beta-\sin \theta)\right. \\
& \left.\quad-(\cos a-\cos \beta)\left(r \sin \beta+x_{2}-r \sin \theta\right)(\sin \beta-\sin \theta)\right] d \theta \\
& \quad+r \int_{-\beta}^{a}\left[\left(r \sin \beta+x_{1}-x_{0}\right)(-\cos \theta+\cos \beta)(\sin \beta-\sin \theta)\right. \\
& \left.\quad+(\cos a-\cos \beta)\left(r \sin \beta+x_{1}+r \sin \theta\right)(\sin \beta-\sin \theta)\right] d \theta=0
\end{aligned}
\]

Operating upon this equation also, we find that
\[
\begin{aligned}
& \left(2 \sin ^{2} \beta-23 \sin \beta \cos \beta\right) x_{0}-\left(\sin ^{2} \beta+\sin a \sin \beta-\frac{1}{2} \cos ^{2} \beta-\frac{1}{2} \cos ^{2} a\right. \\
& \quad+\cos a \cos \beta-\beta \cos a \sin \beta-a \cos a \sin \beta) x_{1}+\left(\sin ^{2} \beta-\sin a \sin \beta\right. \\
& \left.\quad+\frac{1}{2} \cos ^{2} \beta+\frac{1}{2} \cos ^{2} a-\cos a \cos \beta-\beta \cos a \sin \beta+a \cos a \sin 3\right) x_{2} \\
& \quad=r \sin \beta\left(2 \sin a \sin \beta-\cos ^{2} a+\cos a \cos \beta-2 a \cos a \sin \beta\right) \\
& \quad+r \beta(\cos a-\cos \beta) . \quad \text { (2.) }
\end{aligned}
\]
129. Reduction. - From (1.), \(\S 127\), and (1.) and (2.), \(\S 128\), we can determine the desired quantities \(x_{0}, x_{1}\), and \(x_{3}\), by any of the usual steps for elimination. If the second equation of condition is multiplied by sin \(\beta\), and then subtracted from the third, there will result
\[
\begin{aligned}
& \left(\frac{1}{2} \cos ^{2} 3-\cos a \cos \beta+\frac{1}{2} \cos ^{2} a\right)\left(x_{1}+x_{2}\right) \\
= & \cdot \sin 3\left(\cos a \cos \beta-\cos ^{2} a\right)+r \beta(\cos a-\cos \beta),
\end{aligned}
\]
which, upon being divided by \(\frac{1}{2}(\cos a-\cos \beta)\), becomes
\[
(\cos a-\cos \beta)\left(x_{1}+r_{2}\right)=2 r(\beta-\cos a \sin \beta) . \quad(a .)
\]

Again: the second equation may be multiplied by \(\cos \beta\), and added to the first, after which the values of \(x_{0}\) from the new equation and from the second equation of condition may he equated. If we then clear of fractions, and factor the resulting equation, it may be written
\[
[a(b-c)-l e] x_{2}+[a(b+c)-d e] x_{1}=-2 r \sin \beta(a b-d e), \quad(b .)
\]
while equation (a.) will be
\[
a\left(x_{1}+x_{2}\right)=2 f r ; \quad(c .)
\]
in which equations the literal coefficients stand for the quantities already given in § 126.

From (b.) and (c.) it is easy for one to obtain the half sum and the half difference of the two unknown quantities, and thence equations (1.) and


1:30. Formulæ for \(\mathbf{H}_{1}, \& c\). ; Semicircular Arch. - To fint the values of \(\mathrm{H}_{1}, \mathrm{H}_{2}\), and P by formula, we make use of similar expressions to those of \(512 \%\). The figure gives us
\[
1 \mathrm{I}_{1}: \mathrm{H}_{2}: \mathrm{H}=r \sin 3+x_{1}-r_{0}: r \sin 3+x_{2}+x_{0}: \underline{2} \cdot \sin \beta+x_{1}+x_{2} ;
\]
or
\[
\begin{gathered}
\mathrm{H}_{1}=\mathrm{II} \frac{r \sin 3+\frac{x_{1}-x_{0}}{2 r \sin \beta+x_{1}+x_{2}}=\frac{a}{2 r} \cdot \frac{r \sin \beta+x_{1}-x_{0}}{\beta-\sin \beta \cos \beta} \mathrm{H} .}{\mathrm{P}: \mathrm{H}=r(\cos a-\cos \beta): 2 r \sin \beta+x_{2}+x_{2}=a r: 2 r \sin \beta+\frac{2 f r}{a} ;}
\end{gathered}
\]
or
\[
\mathrm{P}=\frac{1}{2} \frac{u^{2}}{a \sin \beta+f} \mathrm{H}=\frac{1}{2} \mathrm{H} \frac{(\cos a-\cos \beta)^{2}}{\beta-\sin \beta \cos \beta} .
\]

If the arch subtemts a semicircle, \(\beta^{\prime}=\underset{2}{1} \pi, \sin \beta^{\prime}=1, \cos \beta=0\), and the preceding values are much simplified. Without writing them in detail, it will be sufficient to indicate that then
\[
\begin{array}{lll}
a=\cos a, & c=\frac{1}{4} \pi^{2}-2, & \epsilon=1, \\
b=\frac{1}{2} \pi a-\sin a, & d=\frac{1}{2} \pi \sin a-a . & f=\frac{1}{2} \pi-\cos a .
\end{array}
\]
131. Sign of Bending Moment. - In determining the sign of the bending moment at any point when the arch is acted upon by a horizontal force, it will be well for the realer to recollect, that, when there is a thrust along any portion of the equilibrium polygon, the arched rib tends to move away from the polygon, but, when there is tension in any portion, tle arch moves towards the polygon. This tendency to move in one direction or the other is easily fixed in the mind. if one thinks of the alteration of curvature of a bent wire when a force is applied at each end in the line joining the two ends. The same thing was noticed in the suspended arch of Fig. 1 and in those under vertical forces. Therefore, in Fig. 32 and the following
ribs, the arch tends to approach the tension side of the equilsbrium polygon, and to recede from the compression side. If then, as before, that moment which makes any portion of the rib less eurved, or which, if exerted on a beam supported at both ends, would make it concare on the upper side. be called positive, the areas of - M will ocem between l' and C in Figs. 32 and 33 , and those of \(+M\) will be found between \(C\) and \(A\). Ribs fixed at the ends will be strained similarly. In Fig. 38, for example, the area to the right of B will give +M ; from the point where \(\mathrm{N}_{\mathrm{G}}\) a crosses the rib to C there will be - M, which then clanges to +M on the left of C , and to - M. when the polygon crosses the rib abose \(A\).
132. Example of Normal Forces. - As we have now ascertained the values of the abutment reactions when a ril) is acted upon by a horizontal force, we will show, by au example that the various horizontal and vertical forces which are exerted at one time at different points of the rib may be provided for in one polygon, without the necessity for separate treatment of the horizontal and vertical components into which the normal or oblique external forces can be decomposed. We will suppose that a parabolic rib of 100 feet span and 50 feet rise is to be used as a principal to carry a roof, and that it is desired to ascertain the bending moments arising from the action of the wind upon one side. We will take the ease where the rib is fixed at the ends as heing less simple. After this discussion, the reader will have no difficulty in applying a similar treatment to other ribs.

Let the rib be represented by A C B. Fig. 40, and let us suppose that the normal wind pressure is directly resisted hy the flanges and bracing of the rib at points D, E, F. and G, at which purlins rest, and which are distant 40 feet, 30 feet, 20 feet, and 10 feet horizontally from the middle of the span. The amount of the pressure \(\mathrm{N}_{2}\) at E will be the total or resultant of the distributed pressure on \(m n\), the points \(m\) and \(n\) being taken midway of the spaces on each side of E . There will be no error of consequence in assuming that the wind pressure on \(m n\) is
perpendicular to the straight line \(m n\), or to the tangent of the parabola at E.* To find this tangent, draw \(\mathrm{E} \mathrm{E}^{\prime}\) horizontally, make \(\mathrm{C} E \mathrm{E}^{\prime \prime}=\left({ }^{\prime} \mathrm{E}\right.\) ', and E E'' will be the desired tangent. The tangents at the other points are found in the same waty. The angle E'E E:' \({ }^{\prime}\) is rery nealy \(50^{\circ}\) : the intensity of wind pressure, by the table of \(\$\) S 109, is 38 pounds on the square font of roof ; and if the principals are 10 feet apart, and \(m " 1\) is \(15 \frac{1}{2}\) feet, the total normal force \(\mathrm{N}_{2}\) at this point will be \(38 \times 10 \times 15 \frac{1}{2}=\) 5,890 pounds. For the four points we therefore find in detail


These normal forces are photed on the figme and then decomposed graphically into their vertical and horizontal components, which, scaled to the nearest ome handred pominds, are found above in the columns headed \(\mathrm{V}^{*}\) and H . The figure and diagrams are drawn to seakes of forty feet and ten thousand pounds equal one inch.
133. Finding the Reactions. - The next step will be to find the values of \(\mathrm{H}_{1}, \mathrm{H}_{2}, \mathrm{P}_{1}\), and \(\mathrm{P}_{2}\), for the above forces. First, upon referring to \(\leqslant 64\), we see that a vertical force at E , Fig. \(40,0.6\) e from the middle of the span, will canse a vertical reaction of 0.896 V at A , one of \(0.104 \mathrm{~V}^{\circ}\) at B , and will give rise to H, at each abutment, of the amome \(0.192{ }_{k}^{c} \mathrm{~V}=0.192 \mathrm{~V}\). We also see, by the table of \(\leqslant 62\), that the ordinate at \(\Lambda\) will be -0.667 k , and at \(\mathrm{I}+0.338 \mathrm{k}\). for the same force at E : and we can then obtain the ralues of MI at the abntments arising from \(V\) ly multiplying these ordinates by \(\mathrm{H}=0.192 \mathrm{~V}\). just ascertained. The computations for the form loaderl points may be grouped together as follows:

\footnotetext{
* If preforred, analyze the wind pressures as in Part I., Roofs, p. 44.
}

\begin{tabular}{|c|c|c|c|c|c|}
\hline 1 & \[
\begin{gathered}
11 \\
243
\end{gathered}
\] & \[
\begin{array}{r}
y_{1} \cdot \\
-2.000
\end{array}
\] & \[
\begin{gathered}
\mathrm{M}_{1} \\
-24,300
\end{gathered}
\] & \[
\begin{gathered}
y_{2} \cdot \\
0.370 \mathrm{k}
\end{gathered}
\] & \[
\begin{aligned}
& \mathrm{M}_{2} \\
&+ 4,495
\end{aligned}
\] \\
\hline 2 & 730 & \(-0.667\) & \(-24,333\) & 0.33:3 & 12,167 \\
\hline 3 & 1,05!) & -0.20.2 & \(-11,767\) & 0.286 & 15,144 \\
\hline 4 & 864 & 0.000 & 000 & 0.2 .29 & 9,600 \\
\hline
\end{tabular}

Totals . . \(\mathrm{M}_{1}{ }^{\prime}=-150,400 \mathrm{ft}\). lbs. \(\quad \mathrm{M}_{2}{ }^{\prime}=+41,406 \mathrm{ft}\). Ibs.
It is to be molerstood that \(y_{1}, \mathrm{P}_{1}\), and \(\mathrm{M}_{1}\) refer to the left abutment. the others, to the right abutment.

From \(\$ 122\) mol \(\$ 125\) we now compute the reactions from the horizontal forces at the fom loaded points. and the accompanying bending moments:

11.100

Totals, \(\mathrm{P}^{\prime}\) from II's \(= \pm 2,001 \mathrm{lbs}\).
\begin{tabular}{|c|c|c|c|c|c|}
\hline & P . & \(x_{1}\). & \(\mathrm{M}_{1}\). & \multicolumn{2}{|l|}{\(x_{2}\).} \\
\hline 1 & :311 \(\times\) & \(4.600 c\) & - \(71,530 \mathrm{ft}\). l bs. & 0.807 & 12,549 \\
\hline 2 & 691 & 1.5.3) & -52,976 & \(0.63 \cdot 3\) & 21,870 \\
\hline 3 & 685 & 0.689 & - 2.3 .702 & 0.486 & 16,718 \\
\hline 4 & 311 & 0.400 & - 6.220 & 0.378 & 5.878 \\
\hline & Totals & \(\mathrm{M}_{1}^{\prime}\) & -154.428 ft . lbs . & \(\mathrm{Ma}_{2}{ }^{\prime}\) & 57,015 \\
\hline
\end{tabular}

The final abutment moments will be
\[
\begin{aligned}
& \mathrm{M}_{1}^{\prime}=-60,400-151,428=-214,828 \mathrm{ft} . \mathrm{lbs} . \\
& \mathrm{M}_{2}^{\prime}=41,406+57,015=+98,421 \mathrm{ft} .1 \mathrm{bs} .
\end{aligned}
\]

The components of the reaction at A are, if thrusts are considered positive.
\[
\begin{aligned}
& \mathrm{P}_{1}^{\prime}=\mathrm{P}_{1}-\mathrm{I}^{\prime}=11,098-2,001=+9,097 \mathrm{lbs} . \\
& \mathrm{H}_{1}^{\prime}=\mathrm{II}+\mathrm{H}_{1}=2,596-10,572=-7,976 \mathrm{lbs} .
\end{aligned}
\]

The components at B will be
\[
\begin{aligned}
& \mathrm{P}_{2}^{\prime}=\mathrm{P}_{2}^{\prime}+\mathrm{P}=1,9012+2,001=+3,903 \mathrm{lbs} . \\
& \mathrm{H}_{2}^{\prime}=\mathrm{II}+\mathrm{H}_{2}=2,596+3,528=+6,424 \mathrm{lbs} .
\end{aligned}
\]

The arrows at A and B show these reactions. If the rib consists of chords and bracing, the stresses on the pieces can be found by a diagram like Fig. 21, l'art I., "Roofs," care being taken to have the stresses in the two flanges at the aboument give the proper reaction (see \(\$ \mathbf{1 9 5}\) ). If the equilibrimen polygon is to be drawn, from whith to tind bending moments and chord stresses, we need the point of beginning for the polygon.

The abscissa, or ordinate to the equilibrimm polygon at \(\lambda\), will be found by dividing the total M at that point \(\mathrm{l}_{\mathrm{y}} \mathrm{P}_{1}^{\prime}\) or \(\mathrm{H}_{1}^{\prime}\); and similarly for the abutment B; thus,
\[
\begin{array}{ll}
x_{1}^{\prime}=\frac{-214,820}{+9,097}=-23.6 \mathrm{ft} . & x_{2}^{\prime}=\frac{+98,421}{+3.903}=+25.2 \mathrm{ft} . \\
y_{1}^{\prime}=\frac{-214,828}{-7,975}=+27.0 \mathrm{ft} . & y_{2}^{\prime}=\frac{+90,421}{+6,424}=+15.3 \mathrm{ft} .
\end{array}
\]

As in previons examples, the ordinate at one abutment alone is needed; but the others are useful as a check on the accuracy of the drawing.
194. Equilibrium Polygon; Bending Moments. - We may now proceed to draw the stress diagram. Lay off 1-2, 2-3, 3-4 and \(4-\overline{5}\), parallel successively to the external forees at \(G, F, E\),
and D, and equal to the calculated amounts by any desirable scale; make \(5-6=\mathrm{H}_{1}{ }^{\prime}\), and \(6-0=\mathrm{P}_{1}^{\prime}\), so that \(5-0\) shall represent the reaction at A in the proper direction as expressed by the signs obtained above, \(\mathrm{P}_{1}^{\prime}\) being a compression, and \(\mathrm{H}_{1}^{\prime}\) a tension ; finally, lay off \(0-7=\mathrm{P}_{2}^{\prime}\), and \(7-1=\mathrm{H}_{2}^{\prime}\), giving \(0-1\) for the reaction at \(B\). The closing of \(0-1\) on the point 1 proves that the diagram has been drawn with tare. Having drawn \(\mathrm{B} \mathrm{Q}=+y_{2}^{\prime}\), or \(\mathrm{B} \mathrm{R}=+x_{2}^{\prime}\), draw through Q or R a line parallel to 0-1, as far as O, where it meets the normal force at G. Then draw O L parallel to \(0-2\), to cut the force \(\mathrm{N}_{3}\) at L . Follow with L K and K I, parallel to \(0-3\) and \(0-4\), closing with a line through I, parallel to \(0-5\), which, if the polygon has been accurately drawn, will make \(\mathrm{A} W=y_{1}^{\prime}\), as recently computed, or \(\Lambda \mathrm{U}=-r_{1}\).

As neither II nor P is constant for oblique forces on an arch, the bending moment at any point will equal the product of the force acting along a side of the polygon just drawn multiplied by the perpendicular from the point to the sile: thas the heading moment at E is \(\mathrm{ES} \times(0-3)\), or \(\mathrm{E} \mathrm{T} \times(0-4)\). If the external forces had been considered as applied at a greater number of points, or as distributed along the principal rafter itself, we should have obtained a polygon which approached nearer is a regular curve, and such a curve has been sketched through the rertices of the polygon just drawn.
135. Equilibrium Polygons for the Vertical and Horizontal Components. - Since most of the needful data have already been obtained, we have thought it expedient to draw the equilihrim polygons for the vertical and horizontal components separately, so that they may be compared with each other and with the polygon for normal forces. If a horizontal and a vertical line are drawn from 1 and 5 , the components H and V can be at once projected upon them. Upon laying off \(\mathrm{H}_{1}\), and plotting \(\mathrm{P}^{\prime}\). we shall locate the pole \(0^{\prime \prime}\); and \(0^{\prime \prime}-2^{\prime \prime}, 0^{\prime \prime}-3^{\prime \prime}\), dec., will be parallel to the lines of the polygon for horizontal forces. In the same way, \(P_{1}\) and \(H\) for vertical forces will determine \(0^{\prime}\). The value of \(y_{2}\) will be found, upon dividing the \(\mathrm{M}_{2}\) which
comes from V by H, to be 14.3 feet. giving the starting-pmint just helow (Q. Upon drawing the polvgon so that the anglen are made at the verticals through the loaded points, we obtain the broken line which fimally rums below \(A\). This ordinate \(y_{1}\) may be verified. If \(\mathrm{M}_{2}\), from the H's is divided by P ', we have \(x_{2}=28.5\) feet, an ordinate a little longer than 13 R . The polygon, if now drawn, will be the broken line which passes near \(\mathrm{E}^{\prime}\), and extends to it considerable distance, 77.2 feet, to the left of A. All the sides of this polygon except the first are in tension.
136. Shear and Direct Stress. - To complete this example, the normal shear at the middle of each division is fomm and at the same time the direct stress. The small letters \(l, m, m\), de., mark the middle of each division. Dialw \(0-1\) in the stress diagram, parallel to the tamgent at \(l\) in the rib, and \(5-l\) perpendicular to it: then will \(5-l\) be the normal shear at \(l\), and \(l-0\) the direct thrust. To satisfy ourselves in regard to the sign of this shear, we mote that 5-0 is the thrmst in the side U' I of the equilibrium polygon, and will therefore be the resultant foree on the left of any section between \(A\) and \(D\) : the forces. \(5-1\) and \(l-0\). in the directions named, will be its components, also on the left of the section \(l\) : hence we have positive shear and a direct thrust. In the same way at \(m\), since \(t-0\) is the thrust in I K. \(4-m\) will he the positive shear, and \(m-0\) the direct thrust. Between \(m\) and \(n\) the shear changes sign ; for at \(n\) we find \(: 3-n\) and \(n-0\). the former being drawn down instead of up. Paswing on, we see that the shear again changes between \(r\) and \(\stackrel{\infty}{ }\), becanse \(1-r\) and 1 -s run in opposite directions. As noted before this change of sign oceurs at points of maximum bending moment.
137. Vertical Shear Diagram. - We may draw a vertical shear diagram, if desired, and from that obtain the nomal components: but it is not so conveniently constructed in the case of several forces which are always applied together as for a case of a single load. If ab represents the span. \(\mathrm{P}_{1}^{\prime}\) or \(\left.{ }_{(i-0}\right)\) is laid off at \(a u\), upwards as usual : then the subtraction of \(V_{1}\), at D ), or \(4-5\), brings us to the line \(d\); thence a step is made to \(\epsilon\), to \(f\), and finally to !, closing at \(b\) with \(0-7\), the reaction at 13 . The horizontal line below \(a b\) cuts off P , or \(0^{\prime \prime}-3^{\prime \prime}\), so that the vertical components shown in the line \(5-1^{\prime}\)
might be considered as laid off from this lower line, and the constant quantity P, due to the horizontal components, then subtracted. As the thrust at \(B\) is \(0-1\), a line drawn through 0 , parallel to the tangent at \(B\), will cut off from a vertical line drawn from 1 as much vertical force as is required, in addition to \(0-7\), to give a resultant in the direction of the rib at \(B\). The amount so determined is laid off at \(\eta^{\prime} r^{\prime}\). Since it has been shown that all inclined lines are drawn towards the middle of the span \(c\), aurl are mintermpted until an external force is encomitered, we draw through \(c\) the line \(r^{\prime} c s\).

In a similar way, a line \(0-10\) from 0 , parallel to the tangent at \(\Lambda\), will cut the vertical through 5 at a distance \(5-10\), equal to \(w u\); a line from 0 , parallel to the tangent at \(D\), will cut off the distance from a vertical through 4 , which is plotted from \(d\) to \(k\); one parallel to the tangent at E will cut off 3-8, which is plotted at eo: and the tangent at F gives \(0-9\), so that \(2-9\) is laid off at \(f p\). If inclined lines are drawn throngh the points thas found, running towards the point \(c\), the diagram will be completed. Normal components of the ordinates between the two sets of lines just constructed, measured above \(l, m, n\), \&c., will agree with the values of the last section, - positive when above the inclined lines, negative when below.




\section*{CIIAPTER IX.}

\section*{STONE ARCHES.}
138. Location of Equilibrium Curve determines Thickness of Voussoirs. - Stone arches may be treated as belonging to the elass of ribs with fixed ends, as the voussoirs have sufficient breadth at the skew-backs to make a firm bearing. We can. then, for a given rise, span, and distribution of steady and travelling load, draw the equilibrimm enrve, and thence determine the required thickness of the arch-ring. To repeat what was mentioned incidentally earlier: if no reliance is placed upon the tenacity of the cement, and if the intensity of pressure at a joint between any two vonssoirs or areh-stones is considered to vary miformly from the outside to the inside edge, the extreme case of cleviation of the resultant pressure from the middle of the joint consistent with safety will ocelur when the pressure is zero at one edge. As the varying intensity of pressure will be represented by the ordinates to an inclined line which passes through the point where the pressure is zero, the total pressure will be equal to the area of a triangle, and the resultant will \(p^{\text {anss }}\) through the centre of gravity of the triangle. or at a distance of one-third the breadth of the ring from that edge where the pressure is most intense. Since the equilibrimume cis the locus of the resultant force at cach joint, the condition that the presisure shall never be less than zero at amp pint, or that there shall be no tension, is equivalent to requiring that the equilibrium curve shall never pass beyond the middle third of the
arch-ring, however the distribution of the load may be varied: hence, when the equilibrium cmres are drawn, the thickness of the voussoirs is readily determined. The tensile strength of the cement after it has become firm, and any deviation from the assumption that the force between two stones must be distributed over the whole joint, increase the safety of the structure, and thus give what is akin to the factor of safety in other cases.
139. Intensity of Pressure. - When the stability of the arch-ring is thus assured, it is an easy matter to find the greatest intensity of pressure, and hence to see whether the material proposed for the arch will have strength enough. When the equilibrium curve passes through the centre of the joint, the pressure on the square inch will be fomed by dividing the thrust at that joint by the area of the bearing surface. If the curve touches the extreme limit, the edge of the middle third, the most intense pressure, at the edge of the joint nearest to the curve, will be twice the mean pressure; for the height of the triangle whose ordinates represent the varying intensities is twice its mean ordinate. In some rare cases, where the span is large, and the stone is of a weak quality. We may have to increase the depth of the areh-ring in order to provide sufficient strength.
140. Circular Arch; Load for Equilibrium. - Althongh the curve of the arch-ring may be any one of a number of forms, the circular arch is the more common type. and we have therefore thonght it best to take such an arch as an example of this method: the steps will apply to any form. The Gothic arch will be classed with the example of \(\$ 194\). If the load is entirely, or almost entirely, steady, as in the aqueduct or canal bridge, it will be advisable, on the seore of economy, to find that distribution of the load which shall catuse the equilibrimm curse to coincide with the centre line of the arch-ring. Then, by arranging the filling and the empty spaces above the areliring so as to conform to that distribution, the voussoirs can be made of moderate depth.

Thus, if B C, Fig. 45, be one-half of an arch which it is desired to load in this way, divide it, by vertical lines. into) (quite a large number of parts, equal horizontally. If the divisions are small, the areas of these portions between the soffit of the arch and the upper line may be considered trapezoids, and the midtle ordinate of each division will be proportional to its volme for mity of thickness, and to its weight, if homogeneous. It is then evident, that, if there is to be mo bending moment at any point, the equilibrium curve must coincite, either with the tangents to the centre line of the ring at these loaded points, or with the chords drawn between these points, according as the first loaded point i.s taken at half a division's distance from the abutment, or at the abutment itself. Sce l'art II., " Bridges." \(\$ 58\). Let this weight he concentrated, in imagination, on each middle ordinate.

Upon drawing. from any point 0. radiating lines parallel to the tangent.. or perpendicular to the radii, at the successive points of division, and cutting them all by a sertical line 1-12 at any convenient distance, loads in each division, supposed to be concentrated at the intersection of the above tangents.* and proportional to the several portions of the vertical line intercepted by the inclined lines, will lee the ones required for equilibrium ; and the distributed londs spread over all of each division. or. in other worts. it continuous load over the whole arel, will thus be fumd. If \(1-2\) is placed at such a distance from 0 that it will represent. by a convenient seale, the mean depth, as well as the weight of the load. in the first division on the right of C, 2-3, 3-t. ©́c.. will represent the required depth of loading in the succeeding divisions. As the angle made by "-2 with the horizontal line is the same as that subtended at the centre by the tirst division near C . there is no difficulty in finding. ly calculation, the exact length of \(0-1\), when \(1-2\) is given, in ease the angle at 0 is too acute to give any accurate result graphically. In our figure the depth of the load at the

\footnotetext{
* The tangents will not intersect exactly in the middle of each division.
}
crown was assumed to be five feet, and the intercepted portions of the rertical line were then plotted from the points where verticals at the midale of each division would cht the centre line of the arch. The curved line drawn through the upper ends of these ordinates will then show the desired amonnt of homogeneons load to be spreal over the arch to produce equilibrium.
141. Limiting Angle for Arch-Ring without Backing. It is now worthy of notice that, while the required depth of loading increases but slowly for some distance after we leave the erown. when we reach the hamehes, the ordinates rapidly. lengthen. and the curve through their upper ends will finally become vertical, if the arch springs vertically from the abutment. This point was also referred to in \(\leqslant 89\). It is apparent, therefore, that it is not practicalle to so load with rertical forces a circular arch, beyond a certain distance from the erown, that the line of thrust shall coincile with the centre line of the arch-ring. As the rombay must not deriate greatly from a horizontal line. we see, that, for an arch extending \(60^{\circ}\) each way from the crown, the amount of material as heary as masonry required over the springing will fill all of the available space, and, when the spandrel filling is lighter, the limiting angle will probably be in the neighorhood of \(45^{\circ}\). In ordinary cases of loading, the equilibrium chrve will deviate so much from the centre line in this portion of the rib as to require very deep volssoirs to retain the eurve within the middle third when the attempt is made to extend the massisted arch-ring much farther. It is customary, therefore to carry the masonry backing, in horizontal courses, up, to the neighborhood of the point where the areh-ring is inclined at an angle of \(45^{\circ}\) : below this point any attempt of the arch-ring to move ontwards under the thrust of the upper portion is immediately resisted by the backing, and the arch will be designed as if the springing points were at the joints level with the top of this masonry lacking. The portion below really forms a part of the abutment.
142. Example; Data. - In accordance with the above statements, and as: in example of the application of preceding principles, we propme to design a circular segmental arch of stome. for a railroad bridge, which shatl subtend \(100^{\circ}\), with a ranlins. for the centre line of the vonssoins, of \(\mathbf{1 0 0}\) feet, making the span, from centre to centre of skew-backs, about 153 fect, and the rise about 36 feet. The rolling load will be 3,000 pounts per romming foot of track, and the width of the bridge over which this load is distributed will be ten feet. The backing will be (arried "11) to the point where the rib) is inclined at \(45^{\circ}\), and the remainder of the spandrel will be filled with such material, or will hate such an amome and distribution of empty spaces. that it shall weigh, on the average, one-half as much per culbie foot as cloes the masomy of the arch-ring. The erfuilibrimm curve for steady load will now first be fomml ; then such pessible combinations of rolling load will be disenssed as will increase the deviation of the steady load curve at those points where it already deviates most from the centre line of the archring; and, finally, the necessary depth of the ronsisirs will be determined by the rule suggested in \(\S 138\). The depth of the vonssoirs at the crown is assumed, in our present ignmance of the final dimensions, at five feet; two feet of filling, earth or some other material, is added at that point, and the lomizontal line drawn seven feet above the soffit at the crown will be the upper bomdary of the spandrel filling. If, then, the arcll-ring is taken at a miform thickness of five feet, as shown at \(A\left({ }^{\prime}\right.\). on the left half of Fig. 45, the dep,th of a homogeneous load equal to stone will be found by shortening each ordinate above the areh ring one-half. Thus was obtained the curve D E. By dividing the area between this curve and the soffit into small portions by vertieal lines, we may find the weight to be concentrated on the several assumed loaded points of the arch-ring.
143. Calculations for Steady Load. - From the equations of \(\S 92\), after making \(\beta^{\prime}=45^{\circ}\), and giving to o the successive values, \(5^{\circ}, 10^{\circ}, 15^{\circ} \ldots 40^{\circ}\), we have worked ont the quantities \(y_{1}, y_{0}\), and \(y_{2}\), for a weight at such distances from the crown, and
these quantities are given in the first portion of the following talbe, it being understood that the weights are here placed on the left of the crown to correspond with our figure: -
\begin{tabular}{|c|c|c|c|c|c|c|}
\hline a. & \(y_{1}\). & \(\%\). & ys. & 1. & \(\mathrm{P}_{1}\). & \(\mathrm{P}_{2}\). \\
\hline \(0^{\circ}\) & . 0449 r & .3557 \(r\) & . \(0149 r\) & 1.126 W & . 5 W & . 5 W \\
\hline 5 & .0252 & .39.5.) & .0607 & 1.09.) & . 5995 & . 411 \\
\hline 10 & . 0001 & . 3.35 & .073. 5 & 1.007 & .6.3) & .30.5 \\
\hline 15 & -. 0311 & . 3569 & .0842 & 0.866 & . 760 & .244 \\
\hline 20 & -. 0517 & . 9.950 .5 & . 0930 & 0.690 & .830 & .172 \\
\hline 2.5 & \(-.1536\) & . 3.3 .37 & . 1012 & 0.495 & . 890 & . 111 \\
\hline 30 & -. 2730 & . 3.515 & . 1078 & 0.311 & .93: 9 & .06:3 \\
\hline 3.) & -.5137 & . 3487 & .1142 & 0.150 & . 97.2 & . 0.27 \\
\hline 40 & \(-1.2407\) & . 3470 & .118:3 & 0.010 & .993 & . 007 \\
\hline
\end{tabular}

These values of \(y_{1}, y_{0}\), and \(y_{2}\), have been plotted on the arch of Fig. 44, and the several stress diagrams have been drawn on a vertical line which represents \(W\). From this figure the amounts of \(H\) and of the vertical components of the abutment reactions for a load \(W\) at successire points can be scaled off, and thus we obtain the last three columns of the above table. Similar values for a circular arch subtending \(140^{\circ}\) are given on page 191.

Having divided the centre line \(C\) A of the arch-ring of Fig. 45 at points \(\mathrm{C}, \mathrm{F}, \mathrm{G}, \& \mathrm{C}\). , distant five degrees from one another, the weight to be concentrated at each of these loaded points is next computed, for an arch one foot thick, perpendicular to the plane of the paper, by scaling the area between the dotted ordinates, marked on the horizontal line, and placed midway between the points of division, and multiplying this area by the weight of a cubic foot of masonry, here assumed at 150 pounds. The weights at the several points, to the nearest hundred pomuds, will then be
\[
\begin{gathered}
\mathrm{C}=7.500 . \quad \mathrm{F}=7.600 . \quad \mathrm{G}=8,400 . \quad \mathrm{I}=9.600, \quad \mathrm{~K}=11.100 \\
\mathrm{~L}=12.500 . \quad \mathrm{N}=14.600, \quad O=16.600 . \quad \mathrm{P}=19.300 \mathrm{lbs} .
\end{gathered}
\]
making the weight of the half-arch (when we take one-half of the load at C , and add 9,800 pomds for the load at A ),\(=113\),450 pounds.

Calculate II for steady load loy multiplying each corefficient of H in the table above by its \(W\) in pounds just ascertained, and adding all the results for both halves of the arch. The work in detail is below. As the two halves of the arch are alike. we add up the column for H , add in again all but the amome for the load at the crown, and have \(\mathrm{H}^{\prime}\) for the entire arch. Each vertical reaction will equal the weight of the half areh.

To find the ordinate \(y_{1}^{\prime}=y_{2}^{\prime}\). for the combined weights, multiply each H by its \(y_{1}\), add the products, and divide by \(\mathrm{H}^{\prime}\). As, for each weight on one half of the arch, there will be a corresponding and equal weight on the other half, it will shorten the process to add \(y_{1}\) and \(y_{2}\) together for each point on one-half of the rib, coxcept the centre one at C .
\begin{tabular}{|c|c|c|c|c|c|c|}
\hline & \multicolumn{2}{|r|}{IV.} & H. & \multicolumn{3}{|l|}{\multirow[t]{2}{*}{\[
\begin{aligned}
& y_{1}+y_{2} . \quad \mathrm{M}_{1} . \\
& .045 r+3 \mathrm{~s} 0.0 r \text { lbs. }
\end{aligned}
\]}} \\
\hline C. \(\theta^{\circ}\) & 1.126 & 7.500 & \(=8.44 .5 \mathrm{lbs}\). & & & \\
\hline F. . & 1.09.5 & 7.1500 & 8.32: & . 0.54 & 71.5 & \\
\hline (8. 10 & 1.007 & 8.100 & 8,4.59 & . 074 & 629.0 & \\
\hline I. 1.5 & 0.s.ifi & 9.600 & 8.314 & .0.50 & 41.5 & \\
\hline K. \(\because 0\) & 0.6 .91 & 11.100 & \(7.8 .5)\) & . 011 & S4. & \\
\hline L. \(0^{-5}\) & 0.493 & 12.800 & 6.374 & -.0.93 & & -337.8 r l lbs \\
\hline N. 30 & \(0: 311\) & 1-1,400 & 4.511 & \(-.165\) & & 749.2 \\
\hline (1). 3.5 & 0.1 .50 & 16,1500 & -2,490 & \(-.400\) & & 995.0 \\
\hline \multirow[t]{4}{*}{1. 40} & \multirow[t]{4}{*}{0.010} & \multirow[t]{3}{*}{19,300} & 772 & \multicolumn{2}{|l|}{\(-1.123\)} & 867.0 \\
\hline & & & 5. 5.37 T 5 lbs . & & 2.21 .6 & \(-2,950.0\) \\
\hline & & & 46.93:1 & & & \\
\hline & & \(\mathrm{II}^{\prime}=\) & 102.307 lh . & ) & \(\underline{2} 8.4\) & \(100(-.712 \mathrm{ft} .=\) \\
\hline
\end{tabular}
144. Equilibrium Curve for Steady Load. - Plot the weighto of the above table on a vertical line from \(1^{\prime}\) to \(10^{\prime}\), lay
off \(\mathrm{H}^{\prime}\) from the middle of \(1^{\prime}-\vartheta^{\prime}\) to \(0^{\prime}\). and. starting at 0.71 feet below \(A\), draw an equilibrium polygon with its sides successively parallel to the lines which wonld radiate from 0 '. 'This polygon will run quite close to the centre line. crossing it twice between \(A\) and \(C\), and passing \(0 . t\) feet below it at the crown. In any actual example the whole polygon shonld be drawn. ds its aceuracy will be proved by its striking the ordinate from \(B\) at the proper distance. If this arch were never to be subjected to any other than a stearly load, or shond the travelling load always be light, roussoirs of moderate depth would contain this polygon within their middle third. The true equilibrimm curve will pass through the angles of the polygon just drawn.
145. Calculations for Rolling Load. - But, as we stated that a line of lailroad was to be carried over this arch, let us suppose that the rolling load of one ton and a half per foot of track, or 3.000 pounds, is distrilmed over the ten feet of width of the arels; the moving load will then amount to 300 pounds per font of span on the rib of om tigure. The sleepers, the filling over the rib, and the bond of the arch-stones, will distribute any concentrated load over a considerable area.

At the crown of the arch the enrve already drawn falls somewhat below the centre line. Cpon inspecting Fig. 44 we see that six of the polygons there drawn pass below the crown of the rib. If, therefore, we place upon the stone areh a rolling load which covers six points of division from each abutment, that is. from ( \(Q\) to \(R\) on one side. and a corresponding distance on the other half arch, this distribution of load, if a practicable one moler the nsual method of rmming trains, will canse the greatest deviation of the equilibrium curve at the crown \(C\).

To draw the polygon for this rolling load alone: first multiply each horizontal distance belonging to I, K, L. \&e.. by 300 pounds, to obtain the concentrated load on each point: then multiply by the proper co-efficients of II already obtained: sum the products, and double the results for both halves of the arch; multiply each \(H\) by its \(y_{1}\) and \(y_{2}\); divide the algebraic
sums of these products by \(\mathrm{H}^{\prime \prime}\). The operations are carich ont below.


Lay off the loads for one-half of the rils on a vertical line from \(t^{\prime \prime}\) to \(10^{\prime \prime}\); make \(t^{\prime \prime}-0^{\prime \prime}=\mathrm{H}^{\prime \prime}\) : imml. laying off \(y_{1}^{\prime \prime}=-2.2\) feet, at A. draw the polygon which passes horizontally below C at a distance, by scale, of 2.3 feet.
146. Increase of Bending Moment at Crown; Required Depth of Keystone. - We can now find how much this added loal increases the negative bending moment at the crown of the rib, or how much it callses the equilibrium curve to move inwards. If we multiply \(\mathrm{H}^{\prime}\) and I1" by the ordinates to their respective curves at the crown. which ordinates are 0.4 feet and
```

102.307 < 0.t = 40.922.5 ft. lbs.
12,318
Orlinate at }\textrm{C}=\quad0.60\textrm{ft}

```
2.3 feet, as lately stated, and add the products, we shall obtain the existing moment at the crown, and, upon dividing by \(\mathrm{I}^{\prime}+\mathrm{H}^{\prime \prime}\), we get the ordinate from the centre line at \(C\) to the curve for the combined loads. It is worthy of note how little effect the rolling load produces, owing to the great thrust of the masomry itself.

In order that this deviation of 0.6 feet from the middle of the joint shall not bring the equilibrium curve outside of the
middle third, the keystone and adjoining vonssoirs must not be less than \(0.6 \times 6=3.6\) feet deep. The greatest intensity of pressure, fomid at the inner edge, will then be twice the mean intensity of pressure, or \(2[114,625 \div(3.6 \times 144)]=442\) pomuls per square inch, giving a factor of safety against crushing of about ten, for good limestone or sandstone. If the depth of the joint be increased to four feet, the greatest intensity of pressure at the imer edge will be reduced to \(\frac{4+3.6}{4} \cdot \frac{114,625}{4 \times 144}=378 \mathrm{lbs}\). per square inch.
147. Increase of Bending Moment at Haunch. - The steady load eurve deviates outwardly from the centre line the greatest distance, 0.5 feet, at L. Fig. 44 again shows that a rolling load from Q to R of Fig. 45 will inerease this deviation to the greatest extent. The value of the horizontal thrust, \(H^{\prime \prime \prime}\), for this load, will be seen, from the table of \(\$ 145\), to be 6,159 pounds. Multiplying the same values of IH by the then existing values of \(y_{1}\), and proceeding as usual, we shall obtain \(y_{1}^{\prime \prime \prime}\). If the total \(\mathrm{M}_{1}\) of this table is subtracted from

that of the tahle in \(\leqslant 145\), we shall obtain the moment at B, and thence find \(y_{2}^{\prime \prime \prime}\). To obtain the vertical eomponent of one reaction, multiply each load \(\mathrm{l}_{\mathrm{y}}\) the proper eo-efficient of \(\mathrm{P}_{1}\) or \(\mathrm{P}_{2}\). given in \(\$ 1+3\). Since \(\mathrm{P}_{2}{ }_{2}^{\prime \prime \prime}\) is \(1,514.4\) pounds, lay this amount off from \(4^{\prime \prime}\), draw \(\mathrm{H}^{\prime \prime \prime}\) to \(0^{\prime \prime \prime}\), and plotting \(-y_{1}^{\prime \prime \prime}\) at \(A\), and
- \(y_{2}^{\prime \prime \prime}\) at B , draw that equilibrimu prlygon which panses \(i .1\) feet above 1 .

By the same process as before, we find that the equilibrium curve for the steady load, combined with these six loads on the left side of the arch, will be displaced from the centre line vertically at L 0.875 feet. The depth of the arch-ring at this point should, therefore, not be less, verticall!, than 5.25. or,
\[
\begin{aligned}
& 102,307 \times(0.5)=51.153,5 \mathrm{ft} .1 \mathrm{ln} . \\
& \frac{6,15!}{105,465} \times 71=\frac{43.5 \cdot 5!9}{)!4.50 .2} \\
& \text { Ordinate at } \mathrm{L}=0.57 \mathrm{ft} \text {. }
\end{aligned}
\] 4.76 feet.
148. Influence of an Additional Load. - When it is noticed that an additional load on the point (r will callse the greatest positive moment at \(K\). it may be suspected that these seven loads will cause a greater deviation at K than the one just found at L. To ascertain the fact, we may dispense with any new polygon by proceeding as follows: The new load of will he \(8.6 \times 300=2.580\) pouncts. H for this point, being 1.007 W . will equal \(2.580 \times 1.007=2.598\) pounds. By sale. in Fig. 4t. the ordinate from the proper polygon to the arch at the point K is. \(017 r=1.7\) feet. The ordinates to the curres alrearly drawn in Fig. 45 being scalest at K . the annexed computation is readily made, and the quotient is seen to be less
\[
\begin{aligned}
102,307 \times 0.35 & =35,507.4 \mathrm{ft} . \mathrm{lbs} . \\
6,159 \times 5.10 & =49,557.9 \\
\frac{2.595}{111.064} \times 1.70 & =\frac{4,416.6}{90,111.9}
\end{aligned}
\] than the amount at L. Kindred steps might le taken for any point.
14.) Increase of Bending Moment at Springing; Maximum \(\mathbf{H}\).- The remaining point of maximum deriation of the curve for steady load is at the springing \(A\), where we have foum it to be .71 feet. As the same six loads from \(Q\) to \(R\) will be seen. from Fig. 44. to produce the maximum effect at \(A\), the polygons are already drawn to our hand, and the moments at the springing loint are seen in the respective tables. There-
fore the ordinate at \(\Lambda\) is 1.45 feet, and the normal displacement is \(1.45 \times \cos 45^{\circ}=1.45 \times .707=1.03\) feet. The necessary
\begin{tabular}{rl}
\(102.307 \times .71\) & \(=72,840\) \\
\(\frac{6.159}{108,466} \times 13.8\) & \(=\frac{84,970}{\boxed{157,810}}\) \\
Ordinate at \(A\) & \(=\frac{1.45}{\mathrm{ft}}\).
\end{tabular} depth for this joint will be 6.2 feet. If the amomet of \(P_{1}\) from rolling load, 12,226 pounds, is laid off below \(10^{\prime}\), and \(\mathrm{H}^{\prime \prime \prime}\), 6,159 pounds, is plotted to the right of \(0^{\prime}\), the line connecting the two points thas found will be the thrust at \(A\), and, from its projection on a line inclined at \(45^{\circ}\), we get 158.000 pounds for the direct thrust at A. The maximum intensity of compression on this joint will be at the inner edge, and will be \(\simeq[158,000\) \(\div(6.2 \times 144)]=354\) pounds per square inch.

The maximm value of H will occur when the rolling load covers the whole bridge. If the amounts of II for the points which have not yet been loaded are computed, the horizontal thrust for a complete travelling load will lie found to be 26.206 pounds. The equilibrimu curve for such a load will be a parabola; the ordinates \(y_{1}\) and \(y_{2}\) will be 1.19 feet, and the curve will pass the crown at a distance of +0.5 feet vertically. As this parabola, when drawn if desired, will be found to lie at most points on the opposite side of the centre line from the curve for steady load, the effect of a complete rolling load will be to bring the arch quite near to actual equilibrimm. The deviation at the crown will be reduced to - 0.2 feet, and, as the total thrust will then be 128,513 pomils, the greatest intensity of compression at that section, for at four-foot voussoir, will be \(\frac{4+1.2}{t} \cdot \frac{128,513}{4 \times 144}=290 \mathrm{lbs}\). on the square inch. We have now examinerl in detail all of the critical points of this arch.
150. Final Dimensions of Arch. - The arch-ring was assumed, at the start, to lee five feet deep. It is apparent, from our investigation and the conditions imposed, that this depth is greater than is necessary for the larger part of the arch, but is less than is required near the springings. For a traveling load of somewhat less intensity, a ring having a uniform depth of
five feet will be entirely satisfactory. Guided by these results, we may redistribute the steadly load in the paturdels so as to hring the equilibrium curve for that load nearer the centre line at the apringings. Another trial will probably aceomplish the desired end, and the above curves for rolling lond can be used anew. Otherwise the arch-ring may he made four feet deep at the erown, and six feet amd a half deep at the apparent springings, as shown on the right half of Fig. t5, and in that case the curves which have been discussed will lie within the middle third of the rib. Although the formula for the circular arch were derived upon the assumption that the rib was of constant thickness, the deviation which we suggest will hardly be of serious consequence. The tenacity of the cement. and the greater or less resisting power of the material immediately in contact with the ring, will sufficiently provide for all contingencies. We have therefore drawn this form as the final determined shape of the arch-ring, the centre line being undisturbed, and the radii of the intrados and extrados being about 95 feet and 104 feet respectively. One must remember, that, as the ring has been altered from a miform depth of five feet, care must be taken to put a little more filling at the crown, and less at the springing, in order that the distribution of the steady load may be muchanged.
151. General Remarks. - If the exterior spandrel wall is massive, a separate equilibrimu curve may be reguired for that portion of the ring which carries the wall: such portion will he subjecterl to a steady load equal to the weight of the wall, but need not he considered as carrying any travelling load. It was not our purpose to enter into the subject of the construction of stone arches, but to show the method of finding the forcen which act on a given or assumed rill. Two or three matters. however, will be briefly referved to. If. at any point, the direction of the resultant pressure makes a considerable angle with the tangent to the centre line of the ring, the two bousoir: having a joint at that place might slip on one another if the joint were radial. No joint should deviate very firl from it
plane perpendicular to the pressure. Generally this angle of deviation is too small to be of importance, and the joints are made normal to the intrados. If several arches are built in a series, it is well to so proportion the spans and rises that the H's from steady load may nearly balance, to avoid a disturbance of one arch by the other, and carry the arches on reasonably slender piers. If one arch has more thrust than the other, and the pier between the two yields, we have a change of span, like that due to temperature.

Knowing the direction, amount, and point of application of the thrust at the springing, we can construct the line of thrust, or equilibrium curve for the abutment, by combining the weight of the abutment and of the masonry immediately above it with this thrust at the springing, the weight of the masonry just above this point being first compounded, and then the weights of successive portions of the abutment. Hence the required thickness of the abutment is ascertained.

Since some of the equilibrium curves may run quite close to the centre line, especially the one for steady load, it may improve the accuracy of measurement of the ordinates or displacements to exaggerate the rertical scale of the drawing. In this case, since all vertical lines will be increased in length, the load lines of the stress diagrams must be laid off with the same proportion to those which represent H .

\section*{152. Location of Equilibrium Polygon by Trial.-} It is comparatively an easy matter to locate the equilibrium polygon tentatively to satisfy one condition only, as in the case of a rib linged at the ends. See \(\$ 38\). The location of the equilibrimm polygon can be made to satisfy two conditions with a little more labor. To fulfil three requirements at once, as is necessary in a rib without hinges, is a very much larder matter. By a method of attack now to be suggested, the labor of experimentally drawing the approximately correct polygon for ribs not parabolas nor circular arcs, under any giveu loading, is much reduced. Familiarity with the arches already analyzed in
this book will aid the judgment in assuming reasonable values for \(\mathrm{H}, y_{1}\), and \(\mathrm{P}_{1}\).
153. Rib Hinged at Abutments.-The equilibrium polygon for any known loading, on an arch of any outline or form, hinged at abutments can be drawn in a short time, by trial, with sufficient accuraey for practical purposes. Lay off the loads in sequence on a load line, compute \(\mathrm{P}_{1}\), assume a value for H , complete the stress diagram, and draw the equilibrium polygon from hinge to linge, as in Fig. 9. Divide the axis or centre line of the rib into a reasonable number of equal parts, draw and scale the ordinates E F and D E, as in Fig. S, at these points of division, and try whether \(\Sigma \mathrm{EF} \cdot \mathrm{DE}=0\). If not, as will probably be the case, increase or diminish H , as appears necessary, draw a new polygon, and apply the condition again. Two or three trials will usually locate the polygon as closely as the assumptions as to load and distribution, as well as the possibilities of designing, call for.
154. Rib Fixed at End; Symmetrical Load.-If a rib with ends fixed is symmetrically loaded on its two halves, of the three conditions to be satisfied the deflection condition \(\Sigma \mathrm{EF} \cdot \mathrm{DB}=0\) need not be tried, as when the area condition \(\Sigma \mathrm{EF}=0\) is satisfied the former will be, since the positive and negative areas have common centres of gravity on the middle ordinate of the areh. Hence

For symmetrical loads, Fig. 43 , plot the loads on the halfarch CB on the load line \(2-1\). Assume \(\mathrm{II}=0^{\prime}-2\), according to judgment. Begin at K, any assumed point, above or below the crown C, and draw the polygon KI, witlı sides parallel to \(0^{\prime}-3,0^{\prime}-4\), etc., as usual. Draw a convenient number of ordinates EF, equidistant along the rib, and by the eye, or by actual summation, see whether \(\sum \mathrm{EF}=0\). In the sketch the dotted equilibrium pulygon evidently makes \(\Sigma \mathrm{EF}\) above the rib exceed \(\Sigma \mathrm{EF}\) below it. It is also apparent that \(+\Sigma \mathrm{EF} \cdot \mathrm{DE}\) is greater than \(-\Sigma \mathrm{EF} \cdot \mathrm{DE}\). It will not be enough to move the point K down and draw a polygon parallel to the first, but

Il must he reduced to \(0-2\), and then the polygon shown by a full line results.
155. Rib Fixed at Ends; Unsymmetrical Load.If the given loads on the rib are unsymmetrically placed, Fig. 43 A , make the loading symmetrical by putting similar loads on the other half of the arch (or select the unsymmetrical portion and add its complement), and then proceed as in § 154, Fig. 43. When H has been thas found, \(\frac{1}{2} \mathrm{H}\) will be the horizontal thrust for the first given loads (or for the unsymmetrical portion); for equal loads on the two halves of the arch will canse twice the thrust due to the loads on one-half the rib.

Then assume, in Fig. 43A, a point 6 on the load line \(2-1\) for the given unsymmetrical loading, lay off \(6-0=\frac{1}{2} H\) just found, assume \(y_{2}\) or \(y_{2}\) at one abutment, making \(y_{1}\) positive, as shown, at the abutment farther from the load. Then draw the equilibrium polygon, closing with a \(y_{2}\) or \(y_{1}\), at the other abutment. Regard being paid to their signs, \(\frac{1}{2}\left(y_{1}+y_{2}\right)\) should equal \(y_{2}\) of Fig. 43. As it probably will not at the first trial, change the ordinate \(y_{3}\) or \(y_{2}\) and draw a new polygon. Then \(\operatorname{try} \sum \mathrm{EF} \cdot \mathrm{DB}=0\) and shift the point 6 until both conditions are satisfied.
156. Catenary. - There is one special case which it may be well to take up. It not seldom occurs in construction that an opening in a wall is to be spanned by an arch, and the masonry at top is limited by a horizontal line, while the load is permanent. If we can make the arch of the form of the equilibrium curve for such a load, we may get a rib of good stability with a very moderate depth. A method of constructing such a curve will now be shown. We stated, in the early part of the book, that the curve assumed by a cord or chain hanging between two points of suspension, and under the action of its own weight only, was called a catenary. The load is distributed uniformly along the curve ; that is, the intensity per foot of the curve is constant. To draw a catenary, proceed as follows: Lay off on a vertical line, 1-11, Fig. 41, a convenient number
of equal spaces, 1-2, 2-3, \&(c., the more the better, and let each of these pates represent the weight of a certain short length of chain, as, for instance, in our figure, 6.4 feet. They may he of the same length as the pieces of chain, if desired. As we do not know the value of H at present, assume it, and draw 1-0 horizontally, equal to H ; draw 11-0 ; consider the weight of the first piece of chain to be concentrated at its middle, and make A B equal to one-half piece of chain, say 3.2 feet : then draw B C parallel to \(10-0\), C D parallel to \(9-0\), and so on, B C, C I), \&c., being successively laid off equal to one piece of 'chain, here 6.4 feet. We shall close with NO parallel to 1-0, and equal in length to A B. A curve from A to O, tangent to this broken line, will be a catenary. If \(1-11\) represents the weight of the chain \(\backslash O, 1-0\) will represent the tenision at (), and hence the weight of a piece of chain. which. hanging over a smooth peg at O, will keep the emere in equilibrimm. Lat OP represent the lengtl of the piece which weighs Il , or \(0-1\). Then a horizontal line l' Q . drawn through P . is known as the directrix of the catenary. This curve has some pecmliar attributes, which may be deduced by mathematical analysis, and may be rerified, in any particular case, from the drawing. Any rertical ordinate to the curve will represent the tension along the curve at the point to which it is drawn. Further, this come will also be in equilibrium under a load which shall fill the entire area inchded hetween \(\mathrm{P}^{(Q}\) and ()\(A\) with a mitionn loarl per square foot of the area. Since, howerer. when () I' is given. the entire curve is fixed, it is posible to make at catenary come of but one span and rise. if the depth of load at the crown is fixed ; and hence the catenary itself is not applicable to the form of an areh where the three quantities just mentioned are given. This arises from the fact that all catenaries are similar figures: therefore, two of the above quantities being given. as for instance. span and rise, the third, the depth at crown, is definitely determined from them.
157. Transformed Catenary ; Example. - It is possible, however, to find a curve which shall be in equilibrium under
such a load, when the span, rise, and depth are all given. In the same way that an ellipse is derived by projection from a circle, a curve, called a transformed catenary, can be projected from a catenary, and will be in perfect equilibrium under the desired or prescribed wall. While some of the quantities used are derived by mathematical analysis, which we will not insert here the accuracy of these quantities can be verified from the diagram.

Let it be desired to find the form of the arch, of half span \(P(\). which shall be in equilibrium under masonry whose depth at the crown shall be SP, and at the springing \(R Q\). It is understood that the arch will be inverted from this figure, and it will be seen that this type of arch may be applied to any span and rise. Let \(\mathrm{PQ}=c, \mathrm{PS}=h_{0}, \mathrm{Q} \mathrm{R}=h_{1}, \mathrm{P} O=m\), and \(Q A=y_{1}\). The first step will lee to find the value of P O, and thus determine the original catenary. This will be done by solving the equation
\[
m=\frac{c}{2.30158 \times \log \left(\frac{h_{1}}{h_{0}}+\sqrt{\frac{h_{1}{ }^{2}}{h_{0}{ }^{2}}-1}\right)}
\]
where log. denotes the common logarithm of the quantity in the parenthesis. Let the half-span be 30 feet, the rise 8 feet. and the depth of load at the crown 2 feet; then is \(h_{1} 10\) feet, and the above expression becomes
\[
m=\frac{30}{2.301 .58 \times \log (5+\sqrt{24})}=\frac{30}{2.29242}=13.09 \mathrm{ft} .
\]

Then by proportion
\[
h_{0}: m=h_{1}: y_{1}, \quad \text { or } y_{1}=\frac{m h_{1}}{h_{0}}=13.09 \times 5=65.45 \mathrm{ft} .
\]

We next obtain from the following formula, the length of the catenary,
\[
s=V\left(y_{1}^{2}-m^{2}\right)=V\left(65.45^{2}-13.09^{2}\right)=64.1 \mathrm{ft} .,
\]
and
\[
\frac{\mathrm{P}_{1}}{\mathrm{H}}=\frac{s}{m}=\frac{64.1}{13.09}=4.9 .
\]

We may now proceed to draw the eatenary breween the points \(\Lambda\) and \(O\). Any length of load line may be laid off. amel Il then drawn of the proper propertiomate amount just foumd.
 eatenary, which will he the area betwern the eurve and the slirectrix multiplied be the weight of a cubic foot of masomer. The area cam be pored equal to \(m\) s, or the product of PO by the length of the curve just found. Divide the load line into a certain number of egral parts, and divide s by the same number. Then proceed with the construction of \(\$ 156\).
158. Construction. - The transformed watenary must be it projection of the catenary so drawn, amd the load and load line will be reduced in the same proportion. 'To save the trouble of redividing the load line multiply \(1-0\) by the matio \(m \div h_{0}\); that is, enlarge the scale of the stress diagram, and lay off that distance from 1 to \(0^{\prime}\). Radiating lines firom \(0^{\prime}\) to the old points of division will be parallel to those which might be drawn from 0 to new points of division: therefore, starting from R, draw the curve RS by making its sides parallel to lines radliating from \(0^{\prime}\), and bringing the points \(\mathrm{I}^{\prime},\left({ }^{\prime}, \mathrm{D}^{\prime}\right.\), d゙c.. vertically below B, C, D. de. But it must be remembered that II in the new curve is the same in amount as \(H\) in the old one, while \(P_{1}\), the vertical component of the reaction, is reduced in the ratio just referred to. 'The rib) need only be deep enough to have strength to resist the thrmst. Fig. 40 shows the arch in an erect position.
159. Many-centred Arch. - If it is wished to lay out an approximation to the transformed catenary, composed of ares of circles, draw normals at the middle points of the successive sides of our construction, and, to get them atecurately, make them perpendicular to the radiating lines of the stress diagram. Prolong them until they intersect one amother, and, on or near the curve which can be sketched through those intersections, select as many centres as may le desired for the circular ares. Thus arches of three, five or seven centres may le drawn, which will he good approximations to the transformed catenary.

\section*{CHAPTER X.}

\section*{STIFFENED SUSPENSION-TRIDGES.}
160. Necessity for Stiffening. - That the curve of equilibrium for the cable of a suspension-bridge, when the load is supposed uniform per horizontal foot, and corers the entire span, is a parabola, was proved in \(\$ 28\). Fig. 6. The steady load will always be carried by the cable. When, however, a moving load is upon the structure, the cable will tend to become flatter in curvature over the lightly-loaded portion, and more curved over the heavily-loaded portion, thms throwing the roadway from its proper line. Some means of stiffening the roadway or chain against distortion is therefore needed. Bridges subjected to travelling loads of but morlerate amome may be stiffened by the longitulinal beans of the rodhay: but heary hads necessitate the employment of trusses or girders in some form.
161. Inverted Arch. - If the calle is divided into two parallel members, braced together as shown in Fig. the. it becomes an inserted arch, and follows the treatment adready given in either ('hap). II., III., or IV., depending upon whether hinges are or ate not introdnced at the piers and the middle. From the fact that the calles are carried over the towers to anchomages, and that movement over the top of the tower will take place both from change of loal and change of temperatime the span camot he assumed invariable: hence there is greater liability to alteration of stress in the several members: from matroidable canses : and a larger factor of safety than is commonly employed
in structures will be appropriate. The introduction of three hinges will do away with these sources of error. This type of stiffening truss will be discussed further in connection with the one which follows.
16.2. Horizontal Girder. - It is. much more common to employ a horizontal truss or girder, as shown in Fig. 47, to stiffen the suspension-bridge. If we note that the office of the arch or inverted arch is twofold, - first to resist the direct stress, and, second, to resist the bending moments at successive sections, - we see that the horizontal girder of this figure will be subject to the same bending moments at similar sections as the inverted arch or braced rib of Fig. 46 , while the chain will here carry the direct stress, which in the former case was also resisted by the rill.

If the truss is hinged at the middle as well ats at the abutments, it comes under the class of (hap. II.; and the effect of one or more loads is easily determined. We may draw Fig. 48, if desired, and find by inspection the extent of rolling load required to produce the maximum bending moment of either kind at any point. See \(\S 32\). Thus, at one-fourth the span from one abutment, the maximum bending moment of one kind oceur's when the rolling loar covers four-tenthes of the span on the same side; and the maximum bending moment of the opposite kind, when the rolling load covers the other six-tenths of the span. The maximum moment at a point near the abutment is found when the head of the load is at one-third the span from that abutment. These ralues are easily deduced by finding the horizontal distance of the point of intersection D, in Fig. 48, on A F, of that line. which, starting from B. passes through E, the extremity of a certain ordinate. Those authors who make maximum bending moments at all points oceur, for a stiffening girler hinged at ends and middle, when the halfspan is covered, are in error. The shear diagrams are constructed as explained in the earlier chapters. The construction for normal shear will be applicable to Fig. 46, and the rertical shear cliagram to the stiffening truss of Fig. 47.
163. Distribution of Rolling Load between Cable and

Truss. - It may be well to call more particular attention to the distribution of the rolling load between the triss and cable of Fig. 47 , and the way in which bentling moments are cansed in the unloaded portion of the horizontal girder. If the bridge is unequally lowled, and no stiffening apliances are used, a distortion is produced, as explained in the first section of this chapter. When a weight \(W\) is applied on a strepension-bridge of half-span \(c\), at any point distant 7 fiom the middle hinge, we know, in the first place, that the total reaction at A, Fig. 47, the end farthest from the weight. is \(\mathbb{W} \frac{-b}{-r}\), and at \(B\) is \(W \frac{c+b}{2 c}\); and, in the second place, as there can be no shear in the cable, we see, from the equilibrimm polygon of Fig. 48, and the lines \(0-4\) and \(0-3\), drawn in the stress diagram parallel to the tangents to the cable at the tops of the towers, that \(\dot{5}-4: 11=2 k: c\),
 of vertical force combined with II of the cable is \(W \frac{c-b}{c}\). Hence at \(\Lambda\) and at \(B\) the cable itself produces a reaction of \(a-b\), the balance of the reaction comes from the truss ; the reaction of the truss at \(\Lambda\) will therefore be \(-W \frac{c-b}{2 c}\), and at B will be \(\mathrm{W}\binom{c+b}{\frac{c}{2}-\frac{c-l}{e}}=\mathrm{W} \frac{3 b-c}{2 c}\). This reaction also will be negative when \(b\) is less than \(\frac{1}{3} c\). Such is the ease in Fig. 48, for the polygon A D B: and we have a corroboration in the negative bending moments near each end.

As the rertical force at \(A\) or 13 from the cable is the load on the half-span of the cable, and this load must be uniformly distributed horizontally to keep the cable in its eurve, the intensity of vertical pull exerted between the cable and the rods per horizontal foot is found by dividing the above force by the halfspan: hence it is \(W \frac{c-b}{e^{2}}\). This will be the upuard pull on
the girder per horizontal foot at all points and the eause of the bending moments. Of course at the point of application of Wr the resultant force acts downward. 'The action of a continnous load over a greater or less portion of the girder will follow the same law; and we shall have downward forces on the loaded portion of the girder equal to the difference between the imposed load and the pull of the vertical rods, and upward forces on the unloaded portion.

It is convenient to notice that the amount of \(W\) carried by either half of the cable is that portion which would be carried by the middle hinge if the half-girder alone supported W. As the girder reaction at the farther abutment is one-half of this amount, and the half-girder on the unloaded side is subjected to a uniform upward force, the shear on the middle linge will also be one-half of this amount. or \(\mathrm{W}^{c-b} \underset{\sim}{c}-\quad\). The shear diagram is given in Fig. 48. For any extent of load it will now be easy to find the amount carried by the cable: for we have only to calculate the portion which would come upon the middle hinge, were that a point of smpport of a simple truss of span \(c\), and this portion will be the load on the half-cable.
164. Comparison of Inverted Arch and Horizontal Girder. - All statements in regard to the horizontal stiffening girder are equally true of the two parallel chains with bracing. While, in the bridge formed of cable and horizontal girder, the girder resists bending moments, and the chain takes up the direct stress, in the latter case the cables have to resist both moment and direct stress. But the maximum direct stress at any section, half of whicla is borne hy each cable, oceurs when the bridge is fully loaded: the maximum bending moment is found with a partial load, at which time the direct stress is less. Hence less material is theoretically required for the cables and truss of the trpe of Fig. 46 than for one like Fig. 47 , - perhaps three-fourthes as much. The introduction of the middle hinge in the axin oí the rib of Fig .46 , with connections of sufficiont strength to transmit the cable stresses, is attended with a little diffenlty, which does not exist in the othere catse.

The three-hinged girder or rib may have the third hinge removed from the middle towards one end, as shown in Fig. 50, where one portion of the girder takes the form of a short link, extending to the first suspending roll.

\section*{165. Horizontal Stiffening Girder hinged at Ends} only.-In case the middle linge is omitted the girder will be exposed to bending moments, as explained in Chap. III. Here, again, an inspection of Fig. 8 will show the extent of load required to prodnce maximum M of either kind; and an examination of the table of bending moments will show that an absolute maximum MI occurs at one-fourth of the span from either abutment for a continnons load extending from one end to a point distant 0.43 of the span from the end nearer to the point of maximum M. Its amount is about . 133 of the maximum moment at the middle of an unassisted girder of the entire span. The stretching of the eables on both sides of the towers impairs the aceuracy of these deductions. For twenty divisions in the span \(X[\) is maximum at 5 for a load from 1 to 8 inclusise, giving \(\mathrm{M}=+0.665 \mathrm{t} \cdot \mathrm{W}\). Loads from 9 to \(1: 9\) give an equal negative moment at 5 . The point of contratlexure in the first case is between points 9 and 10 , and not exactly at liead of load.
\[
\frac{1}{8} W l=\frac{1}{8}(20 w \cdot 2 c)=5 w c . \quad 0.6654 \div 5=0.133 .
\]

From the value of \(\mathrm{Y}_{1}, \S 50\), it is evident that
\[
\frac{5}{32}\left(1-n^{2}\right)\left(5-n^{2}\right) \mathrm{W}
\]
is carried by either half-elain, and this quantity dirided by \(c\) will give the intensity of upward pull on the triss from a load \(W\) at one point. The above amomet is again that which would be carried to the point of contratiexure of the truss, if that were the point of support of the unassisted truss, and the truss were discontimous over the support. (Compare Rankine's "Applied Mechanies," page 375, note.)
166. Stiffening Girder of Varying Depth.-Returning anew to the case of the stiffening girder with three linges, it is
evident, that if the girder has a variable depth, greatest at the points of maximm bending moment, the stresses in the flanges or chords will be diminished proportionally, with an economy of material. If, at the same time, the girder is itself the suspension cable, we can so adjust the depth, that the flange stresses for a partial load shall never exceed those arising from an entire load. Modifications having this end more or less in view have been suggested and carried out. Let us first draw, in Fig. 49, the equilibrime curve for a rolling load alone over half the span. While this curve will not give maximum bending moments, it will not differ greatly from the curves of maximum M , and it offers a very convenient and sufficiently accurate basis of comparison. Its form will be a straight line over the monoaded half of the span, and a parabola tangent to that line for the remaining portion. As the tangent at the abutment end of this parabola meets the tangent from the other end in the vertical through the centre of gravity of the load, the tangent A D is at once drawn. Draw the chord A C. The parabola cuts the middle vertical ordinate E D from the chord A C at its middle point \(F\). If the height of the original parabola of the cable is \(k\), the ordinate at one-fourth the span is \(\frac{3}{4} k\). G I) \(=\frac{3}{2} k ;\left(\mathrm{G} \mathrm{E}=\frac{1}{2} k:\right.\) therefore \(\mathrm{E} \mathrm{D}=k ; \mathrm{E} \mathrm{F}=\frac{1}{2} k\); and \(\mathrm{F}(\mathrm{a}\) \(=k\). Hence the remaining ordinate for bending moment at one-fourth the span is \(\frac{1}{4} k\) on citlrer side, anel of opposite signs.
167. Ead's Arch, or Lenticular Stiffening Girder. - If the two half-ribs of the arch of Fig. 51, or of the stiffened suspen-sion-bridge, are each made of two equal parabolas, the outer ones being the continuous equilibrim curve for a complete load, the rertical depth of the semi-girders at their middle sections E and F will be one-half the rise or height, \(k\). Let \(u\) s denote the horizontal thrust or tension from steady load w by H ; that from a full rolling load \(w^{\prime}\), by \(\mathrm{H}^{\prime}\). The horizontal stress due to a rolling load extending from one abutment over half the span will be \({ }_{2}^{1} \mathrm{H}^{\prime}\); for a similar load over the other half-span must give an equal stress, and both combined must equal \(\mathrm{H}^{\prime}\). When the above bridge is fully covered with mov-
ing load. the equilibrium curve will coincide with the continuous curve, and the stress at each section of the main cable will be that due to \(\mathrm{II}+\mathrm{II}^{\prime}\). The anxiliary ribs and bracing will experience no stress. When the bridge is half loaded, say from (: to B, the equilibrimm polygon for rolling load will be the one sketched in our figure: it passes at \(\mathrm{I}, \frac{7}{\ddagger} k\) below the main calle at D , and through the middle or axis of the truss A C . The horizontal component of the stress at D , due to \(\frac{1}{2} \mathrm{II}^{\prime}\) at I , is, from the equation of moments about \(\mathrm{E}, \frac{3}{} \mathrm{H}^{\prime}\); that is, \(\frac{1}{2} \mathrm{H}^{\prime} . \frac{3}{4} k=\) hor. comp. at \(\mathrm{D} \times \frac{1}{2} k\). Taking moments about D , \(\frac{1}{2} \mathrm{II}^{\prime} . \frac{1}{4} k=\) - hor. comp. at \(\mathrm{E} \times \frac{1}{2} k\); or horizontal component at E is \(-\frac{1}{4} \mathrm{II}^{\prime}\). At F and G the horizontal component is, in each member, \(\frac{1}{4} \mathrm{HI}^{\prime}\). The minus-sign denotes opposite stress, here compression, in the arch, tension. We may therefore write the following table of cases:


Since F and (r change places with E and D for a load on the other half-s man, we see that the lower member, or main cable, experiences a horizontal component which fluctuates from II to \(\mathrm{H}+\mathrm{H}^{\prime}\), always tension; while the auxiliary rilb has a stress whose horizontal component ranges between \(\frac{子}{子} \mathrm{HI}^{\prime}\), tension, and \(\frac{1}{4} \mathrm{H}^{\prime}\), compression. The bracing will undergo no stress from a full load. The stress in the bracing for partial loads may be worked out by the method of the previous chapters for finding the amome of shear remaining after subtracting the vertical components for the two cables at a section, by the method of Part II., "Bridges." Chap. V., or by drawing stress diagrams as given in lart I., "Roofs."

As the parabola through I is a projection of that through D. the above deductions for the points D and E are true for the other points of the girder. Although, as pointed out in \(\S 162\),
the bending moments are a little greater for loads which cover not quite half the span, it is evident that the horizontal compor nent of the stress in the main cable can never exceed II \(+I I^{\prime}\), and in the comuter-rib will but slightly exceed \(\pm \frac{1}{4} \mathrm{II}^{\prime}\). This form of arch was designed and patented hy James B. Eads: a praper upon it by him may be fomed in the "Transactions of the American Society of Civil Engineers," vol. iii.. No. bi, October, 1874 .
168. Bowstring Stiffening Girder. - If the auxiliary memhers connecting the hinges A, C, and B, Fig. 52, are straight, we have a rariation in the method of stiffening and a change in the stresses. The equilibrium curve A F C I B. for a rolling load over one-half the span, is also drawn here, coinciding with A C, and passing through I, \(\frac{1}{4} k\) helow D. The steady load will he entirely carried by the main cable as before as will also a complete rolling load. The half rolling load, being entirely supported on the left by A F (' will ranse in that member a tension whose horizontal compouent is \(\frac{1}{2} \mathrm{H}^{\prime}\) : a horizoutal tension in D, of \(\mathrm{H}^{\prime}\), and a horizontal compression in E, of \(\frac{1}{2} \mathrm{H}^{\prime}\). as is fomd ly similar conations of moments to those in the last section. There results, then, for this type the following cases: -


The stress on the main cables will be very slightly increased for some partial loads, as shown before. The increase will, however, be small, for the direct stress is lecreased at the time the bending moment is increased; so that the absolute maximum may be called \(\mathrm{H}+\mathrm{H}^{\prime}\) withont any error of importance. The stress in the straight stiffening rib ranges from a tension of \(\frac{1}{2} \mathrm{H}^{\prime}\) to a compression of \(\frac{1}{2} \mathrm{H}^{\prime}\). While the member A C or C B has to resist double the force of the preceding case, and that
force also completely reversed for a moving load over one-half of the bridge, the unbraced lengths are shorter than in Fig. 51, the construction of a straight member is simpler, and the wel, members are only one-half as long: the cost may therefore be sufficiently influenced to cause this design to commend itself more to the practical builder than does the former. A notable example of this type is the Point Bridge at Pittsburgh, Pemn., eight hundred feet span, built by the American Bridge Company of Chicago, in 1876.
169. Fidler's Stiffened Suspension-Bridge. - Again, let us conceive of two calles, A F C I) B and B E C G A, Fig. 53, each separately subject to, and in equilibrium under, a rolling load over one-half the span. and then let their places be taken by the two girders shown. A C and C B will be straight, as in the last figure; A (i C and C D B will be parabolas, each tangent at \(C\) to the chord of the other: and the equilibrium curve for a complete load will pass through the middle of each truss, as shown by the dotted line. These trusses are, therefore, of the form of Fig. 52; but they have a depth equal to that of the trusses of Fig. 51. The horizontal component H. of steady load, and \(\mathrm{H}^{\prime}\), of complete rolling load, will be carried equally by both members of each truss. \(\frac{1}{2} \mathrm{H}\) and \(\frac{1}{2} \mathrm{H}^{\prime}\) on each. A rolling load on the right half of the span will canse a horizontal tension of \(\frac{1}{2} \mathrm{H}^{\prime}\) at D and at F . We may, then, write, for this type,
\begin{tabular}{rrrrr} 
Horizontal component of stress at & E & D & F & G. \\
With steally load only & . . \(\frac{1}{2} \mathrm{H}\) & \(\frac{1}{2} \mathrm{H}\) & \(\frac{1}{2} \mathrm{H}\) & \(\frac{1}{2} \mathrm{H}\),
\end{tabular}
" .. " and one-half roll-
\[
\text { ing load . . . . . . } 12 \mathrm{H} \quad \frac{1}{2} \mathrm{H}+\frac{1}{2} \mathrm{H}^{\prime} \frac{1}{2} \mathrm{H}+\frac{1}{2} \mathrm{H}^{\prime} \frac{1}{2} \mathrm{H} \text {, }
\]
with steally load and complete roll-
\[
\text { ing load . . . . . } \cdot \frac{1}{2} \mathrm{H}+\frac{1}{2} \mathrm{H}^{\prime} \text { " " } \frac{1}{2} \mathrm{H}_{2}+\frac{1}{2} \mathrm{H}^{\prime} \text {. }
\]

The stresses will, therefore, always be tension, and the horizontal component will vary in each member from \(\frac{1}{2} \mathrm{H}\) to \(\frac{1}{2}\) \(\left(H+H^{\prime}\right)\), a most favorable showing for the structure. The
renark of \(\S 162\) in regard to maximum bending moments. applies here also. The maximum stresses in the bracing can be worked up in the way thought most convenient. This type may also be analyzed ats two inverted howstring girders, it weight on one causing simply a tension in the tie of the other and an inclined reaction in its line at the middle hinge. Il conce the investigation of the bowstring girder in lart II. may be applied here. A very interesting analytical disenssion of the types of hritges and arches of this chapter may be fomed in "Engineering," rol. xx. for 1875, from the pen of Mr. T. Claxton Fidler, the inventor and patentee of the type discussed in this section.
170. Ordish's Suspension-Bridge. - Another stiffened sini-rension-hridge, in which the problem of resisting distortion from a partial load is solved in quite a different way, is what is known as Ordish's, shown in Fig. 55. The Alhort bridge over the Thames, at Chelsea, Eng.. is of this type; and one of morlerate span has been erected orer the Pemsylyania Railroad. at 40th Street, Philadelphia. Here a certain initial stiffness is. given to the platform itself. and it is then directly supported at several points from the tops of the towers. It is intencled that the weight shall be entirely carried by the inclined ties. As these ties, from their length, would sag considerably muder their own weight, a fassing load would callse the radway to move reatically: for an increased pull on a tie would tend to straighten it. They are, therefore. suspended, at the joints in the several lars which make mp the tien, from a light cable, which is lesigned simply to carry the weight of the ties: and the suspending rods are so adjusted, that the ties shall be straight. No movement of the roadway of any importance (an then take place. The analysis is very simple.
171. Brect and Inverted Arch combined. - The Inidge wer the Elbe, at Hamburg, one span of which is shown in Fig. ist, is a combination of the erect and inverted arch. This con--truction dispenses with abutments to withstand a thrust, as the thrust of the upper rib will at all times be balanced by the
tension of the lower rib. If the rils are of equal stiffuess, any load may be considered as diviled equally between the two systems: if the ribs, while having the same curvature, are not alike in eross-section, the load will probably be distributed in the ratio of their moments of inertia. As the erect arch always tends to move away from its equilibrimm curve. and the inverted arch to approach the equilibrimm curve the tangents at the abutment ends will move in the same direction, and therefore the structure should be treated as hinged at the ends, unless each flange is firmly bolted to the skew-back. If the structure is carried on eolumns or a pier, it appears to us that the ends cannot be rigid, and we judge that the two ribs will begin to turn aloout the middle of the depth withont the introdurtion of a pirot or hinge.

The effect of temperature is :mnullerl. Also the shortening of the erect arch under the direct compression being opposite to the extension of the inverted arch under the direct tension, the span will tend to remain maltered; but the ribs themselves will be changed in form, one rib flattening as the other becomes more convex. If, in making such a design the section of the arch is found to differ much from the section of the inverted ril), it will be well to calculate the relative deflections of the two ribs at the middle. The amount of load each will rary varies inversely as the deflertion muder equal loarls. since they must deflect equally: and hence. if the arch is first designed of such shape, for the purpose of resisting compression, that it is stiffer or has less deflection than the chain. when each has one-half the load, the cros-section of the arch must be in(reased, and that of the chain may be diminished. This type of structure must not be confounderl with a lenticular girder: the absence of bacing between the ribs makes them independent.

\section*{CHAPTER XI.}
bending moments from change of form. \({ }^{1}\)
172. Displacement from Bending Moments. - It follows, from the fact that the arched rib moves away from the equilibrimm polygon or eurve, that the bending moments and chord stresses will have a slight tendency to increase. When the rilb changes in shape, however, the equilibrimm polygon must also move enough to still satisfy for the new form the equations of condition ly which it was first established, and this movement will in some measure counteract the former. Besides, the equilibrim curve for steady load generally runs so close to the axis of the rib, that the change of shape from bending moment. is very slight: and, even when the influence of rolling load is added, the increments of the bending moment ordinates are too small to be of material consequence.

The rertical displacement at any point E, Fig. 56, produced by any load, will be found, for the parabolic rib. by taking area moments, as explained in Part II.. "Bridges." Chap. VI.. or for the circular rib by summing the ordinates as usual along the rib. As was done in the treatment of beams, it will here he necessary to find the point I) where the tangent to the rib in its new form is horizontal, which point will not be at the crown.

\footnotetext{
1 Many of the deductions in this chapter are only intended as guides in pratical construction, to indicate where, and to show approximately how much, allitional stress may be anticipated from change of form. Exact results are not attempted.
}
exept for symmetrical loads. I) is then to he assumed momentarily as a fixed point, and the deflection or area moment of A and E obtained with reference to it : the subtraction of the latter from the former gives the displacement of E relatively to the abutment A ; that is, from the area moment between I) and A subtract the area moment between D and E ; and the remainder, when multiplied by \(I f \div \mathbf{E I}\), will be the vertical displacement of \(\mathbf{E}\). As just stated, these displacements may be neglected.
173. Displacement and Bending Moments from Compression. -The thinst which exists at each section of the rib must, by its compression of the particles, canse a shortening of the rilh, and, as the shorter rib must fit the same abutments, it is necessarily lowered at the crown. The resulting bending monents may be of consequence. So far as the ribl retains sensibly its old form, parabolic or the segment of a circle, the equilibrim polygon is lowered proportionally to the sinking of the rib, as indicated in Fig. 57, in order to still satisfy the equations of condition; but, as the deflection \(v\) at the crown is very small comprared with \(k\), the alteration of the bending moment ordinates is rery trifling. On the other hand, this lowering of the apex of the equilibrium polygon at once increases the value of II, offsetting the change first pointed out. This will be seen, also, from the values of \(M, \S 44\), into which 6 does not enter. The hending moments from the external load are therefore practically unaltered by the change of form.

To produce this change of form, however, or to loring the arch down to its new position, requires a change of inclination, and consequently a bending moment, at most points of the rib. The strains thus induced should be examined. Strictly acenrate theoretical investigations for the different ribs camot easily be made; but formula may be deduced which will serve all practical purposes.
174. Parabolic Rib hinged at Ends. - Thee parabolic rib which we have treated varies in cross-section, from the crown
(t) the springing, aecording to the secant of the inclination to the horizon, 537 ; and, as the magnitude of the direet thrmst lon a complete miform load varies in the same way, the intensity of direct compression per unit of eross-section arising from II will be eomstant, and every unit of length of are will be shortened hy that thrust the same amount, so that the arch will he altered as if exposed to a change of temperature. We will assume that the new fomm of the rib is still a pambola with a rise \(k^{\prime}\) in pace of \(k\), but with the original span \(2 c\).

By definition, Part II. " Bridges," \& 85 , the morlulas of elasticity \(\mathbf{E}\) equals the intensity of stress divided by the shortening of a unit's length. İet the constant intensity of thrust equal the thrust at the crown II, divided by the cross-section at the crown A ; let the eompression of a mit's length egnal the difference, \(s-s^{\prime}\), between the lengths of are before and after commession divided by the original length \(\underset{\text {. Then }}{ }\)
\[
s-s^{\prime}=\frac{I I}{\Lambda} \mathbf{E}^{\prime}
\]

An aproximate formula for the length of a parabolic are is, in om usnal notation, \(s=\ddot{\sim} c+\frac{4}{3} \frac{k^{2}}{6}\). The value of \(s^{\prime}\) will be whtained by writing \(k^{\prime}\) for \(k\); then
\[
s-s^{\prime}=\frac{4}{3 c}\left(k^{2}-k^{\prime 2}\right)=\frac{\mathrm{H} s}{\mathrm{AE}}=\frac{2 \mathrm{II}}{\mathrm{~A} \mathrm{E}} \cdot \frac{3 c^{2}+2 k^{2}}{c}
\]

As \(r\), the deflection at the erown and the elifference between \(k\) ant \(k^{\prime}\), is very mall, wr may write withont sensible error, \(k-k^{\prime}=v\), and \(k+k^{\prime}=2 k:\) whence \(k^{2}-k^{\prime 2}=2 k r\), and we have
\[
{ }_{3}^{8} c^{k v}=\frac{-2 \mathrm{H}}{3 \mathrm{~A} \mathbf{E}} \cdot \frac{3 c^{2}+2 k^{2}}{c}, \text { or } r=\frac{11}{4 \mathrm{~A} \mathbf{E}} \cdot \frac{3 c^{2}+2 k^{2}}{k} .
\]

It was poved, in horizontal beam of miform section: hence to bring the arch down to its new position will ereate bending moments at all points such as would aceompany the same deflection in a
straight beam, supported at the ends, uniformly loaded, and of a cross-section equal to that of the rib at the crown. In Part II., " Bridges," \(\S 95\), we found, for a beam supported and loaded as above with "f per foot,
\[
v=\frac{5}{384}, \frac{w l^{4}}{\mathbf{E} \mathrm{I}}=\frac{5 \psi c^{4}}{24 \mathbf{E} \mathrm{I}}=\frac{5 M \mathrm{M}_{0} c^{2}}{12 \mathbf{E} \overline{\mathrm{I}}}
\]
if \(\mathrm{M}_{0}\) is the bending moment at the middle. Equating these two values of \(v\), we obtain
\[
\frac{5 \lambda_{0} c^{2}}{12 E I}=\frac{H}{4 A E} \cdot \frac{3 c^{2}+2 k^{2}}{k}
\]
or
\[
\mathrm{M}_{0}=\frac{3 \mathrm{IH}\left(3 c^{2}+2 k^{2}\right)}{5 A c^{2} k}
\]
the additional positive bending moment at the crown of the arch, cansed by its compression under the thrust H.

The bending moments at other points may then be taken to compare with those of the beam, that is, as the ordinates to the parabola, being \(\frac{3}{4} \mathrm{M}_{0}\) at the quarter-span.
175. Remarks; Example.- It will be noticed that \(\mathbf{E}\) has disappeared from the expression for \(\mathrm{M}_{0}\) : hence the bending moment will be the same, whether the material be iron, steel, or wood. As \(\mathbf{I}\) contains \(A\), and may be witten \(n \lambda h^{2}\), Part II., - Bridges," \(\$ 86, n\) being a mmerical factor, it is seen that the bending moment from deflection of the rib) due to compression increases with the square of the depth of the rib, and, as \(\mathrm{M} \div h_{6}\) equals the flange stress, this stress will increase directly as the deptl. To diminish the effect of change of form alone, employ a shallow rib.

If \(\|=20\) tons, \(c=100\) feet or \(l=200\) feet, \(k=20\) feet. and \(h=2 \frac{1}{2}\) feet, for a rib with two plate flanges and thin or open web, \(\mathbf{I}=2\left\{\frac{1}{2} \mathrm{~A} \cdot\left(\frac{1}{2} h\right)^{2}\right\}=\frac{1}{4} \mathrm{~A} h \ell^{2}\), and
\[
\mathrm{M}_{0}=\frac{3 \times 25 \times 20 \times 30.800}{5 \times 16 \times 10,000 \times 20}=2.9 \mathrm{ft} . \text { tons at crown, }
\]
giving 1.16 tons compression on upper flange, and an equal tension on lower flange.
176. Displacement from Change of Temperature. - The deflection produced by a fall of temperature in the parabolic rib hinged at the ends will be found by taking the area moment of the half parabolic segment, Fig. 16, from the crown to the springing about one abotment, and multiplying by \(\mathrm{H} \div \mathbf{E} \mathbf{I}\). Hence, as in Part II., "Brilges," § 95,
\[
c_{t}=\frac{\mathrm{HI}_{t}}{\mathrm{EI}} \cdot \frac{2}{3} c k \cdot \frac{5}{8} c=\frac{5}{12} \cdot \frac{\mathrm{H}_{t}}{\mathrm{EI}} \cdot c^{2} k,
\]
the deflection at the crown when the temperature falls, and the rise of the crown when the temperature rises. One may prefer to consider the rib in its new position as the proper curve fiom which to obtain the area moment. If it is assumed to still be a parabola with the rise \(k^{\prime}\), we have
\[
v=\frac{5}{\frac{5}{12}} \mathbf{I I} \mathbf{E I}^{c^{2} k^{\prime}, \text { and }} k^{\prime}=k \pm v .
\]

Substitute this value of \(k^{\prime}\). and \(v\) becomes
\[
v=\frac{5 \mathrm{H} c^{2} k}{12 \mathrm{EI} \mp \overline{5} \mathrm{H} \mathrm{c}^{2}}
\]

This deflection is the result of the bending moments arising from \(I I_{i}\), and is not to be regarded in the light of the preceding section. The moments were eomputed in \(\S 74\). These moments will be slightly altered ley the movement, as it shortens or lengthens the ordinates; but \(\mathrm{H}_{l}\) will be changed in the opposite direction, reducing the actual modification of the moments. Since
\[
\mathrm{H}_{t}=\frac{15}{8} \cdot \frac{1 e \mathbf{E ~ I}}{l^{2}}, \quad c_{t}=\frac{25}{32} \cdot \frac{t e c^{2}}{k},
\]
a quantity independent of the cross-section of the rib, and, so far as the material is concerned, affected by the co-efficient of expansion only.

The bending moments due to the direct thrust, whether arising from a load or change of temperature, have been considered, as well as the resulting deflection. When the temperature rises, \(\mathrm{H}_{t}\) is thrust, and in itself tends
to shorten the rib, and thes reduce the above amount of rise due to ex ansion. The ratio of the two deflections will be
\[
\frac{v}{c_{t}}=\frac{\mathrm{H}_{t}}{4 \mathrm{~A} \mathrm{E}} \cdot \frac{3 c^{2}+2 k^{2}}{h} \div \frac{5}{12} \mathrm{H}_{t} c^{2} k=\frac{3}{5} n h^{2}\left(\frac{3}{k^{2}}+\frac{2}{c^{2}}\right)
\]

In the example previously cited this ratio becomes
\[
\frac{v}{v_{t}}=0.6 \times \frac{2.5}{16}\left(\frac{3}{400}+\frac{2}{10,0011}\right)=.007 .2
\]
a reduction of three-fourths of one per cent. When the temperature falls, \(\mathrm{II}_{t}\) is a tension, and, in lengthening the rib, slightly reduces the deflection.

The deflection for a co-efficient of expansion of .000007 and a range of temperature of \(30^{\circ}\) will be, in our example of ş 175 ,
\[
c_{t}=\frac{25 \times 30 \times .000007 \times 10.000}{32 \times 20}=.082 \mathrm{ft} .=1 \text { inch. }
\]
[The expansion or contraction of a straight bar may be conveniently stated as \(\frac{1}{4}\) inch in one hundred feet for \(30^{\circ} \mathrm{F}\).] The theoretical movement of the rib at the crown for a range of \(30^{\circ}\) above and below the temperature at which it was constructed will therefore be two inches. The actual movement is generally less than theory would indicate, owing to gradual transition from one extreme to another, protection of some portions of the structure from extremes of temperature, as by shielding from the direct rays of the sun, \&c., and, finally, imperfect freedom of motion.
177. Initial Camber for Arch. - It may he experlient to make the rib a little longer than the distance between the springings to compensate for the amount of compression which will arise from the steady load, or else to wedge up the springing points until the crown of the rib, when not under strain, shall be a distance \(v\) above its normal position: the rilb will then, when in place and under its steady load, come down to the curve for which it is designed, and will be free from that portion of initial bending moment due to change of form from steady load. This will be true, because, in forcing the rib up,
we lave introduced bending moments of the opposite kind to an equal amomet. An additional allowance may be made for an ordinary travelling load. If the rib is to be made longer to offiset the compression, find re, S. 17 t, or H from steady load, and make the parabolic rib) of a simn \(2 e+\pi\) and ative \(k\), so that,
 \(k+z\), if it were not compressed at the same time.

Noticing, from § 17 , that this compression acts like a fall of temperature in shortening the rib, we have, from \(\$ 74\),
\[
\mathrm{II}_{t}=\frac{15}{S} \cdot \frac{\mathrm{EI}}{c k^{2}} \cdot t e c=\frac{15}{5} \cdot \underset{ }{\mathrm{E}} \mathrm{I}^{2} \cdot \frac{u}{2},
\]
since \(u\) must equal \(2 t\) ec. But \(\mathrm{I}_{t}=1_{5} 2 \frac{\mathbf{E I}}{c^{2} k} v\), by \(\S 176\), and, equating these two values, we get
\[
\frac{15}{16} \cdot \frac{\mathbf{I}}{c k^{2}} \cdot u=\frac{12}{5} \mathbf{E I}_{c^{2} k}
\]
or
\[
u=\frac{64}{25} \cdot \frac{k}{c} \cdot v=\frac{10}{25} \text { II } \cdot \frac{3 c^{2}+2 k^{2}}{c} .
\]

If, in our preceding eximple, \(A\) is eight square inches, and \(\mathbf{E}\) is \(\geq 4,000,000, u\) becomes lialf an inch.
178. Parabolic Rib with Fixed Ends. -In this case the deffection will naturally correspond with that of a beam of uniform section, uniformly loaded, and fixed at the ends, ats will be seen loy comparing the equilinimm curve of Fig. 17, where If from temperature alone acts, with that of such a beam. In Part II., "Bridges," \(\$ 99\), and Fig. 47, we found that
\[
v=\frac{w l^{4}}{38+\mathbf{E I}}=\frac{w c^{4}}{2+E I}=\frac{\mathrm{I}_{0} c^{2}}{4 \mathbf{E} \mathrm{I}},
\]
if \(\mathrm{M}_{0}\) is the bending moment at the middle. Equating this value of \(v\) with the one found in \(\S 174\), we olstain
\[
\mathrm{M}_{0}=\frac{\mathrm{III}\left(3 c^{2}+2 k^{2}\right)}{\mathrm{A} c^{2} k}
\]

The bending moment at the springings will be double this amount, and of the opposite sign.

The deflection produced by a change of temperature will be fomd by taking the area moment of the semi-segment of the parabola already obtained in \(\S 176\), and subtracting the area moment of the rectangle whose height is \(\frac{2}{3} k\) and base \(c\).
\[
v_{t}=\frac{\mathrm{II}_{t}}{\mathbf{E} \mathbf{I}}\left(\frac{5}{12} c^{2} k-\frac{2}{3} c k \cdot \frac{1}{2} c\right)=\frac{1}{12} \mathbf{I I}_{\mathbf{E}} \mathbf{I}_{t} c^{2} k .
\]

Applying the data of the previous example of \(\S 175\), we have
\[
\mathrm{M}_{0}=\frac{25 \times 20 \times 30,800}{16 \times 10,000 \times 20}=4.8 \mathrm{ft} . \text { tons at crown, }
\]
giving 1.92 tons, compression on upper flange and an equal tension on lower flange at crown, and 3.85 tons, tension on upper flange with an equal compression on lower flange, at either springing.

To find such additional length of span for the parabolic rib fixed at the ends, that, when compressed muder steady load, it may have no bending moments due to change of form, we pursue again the method of \(\$ 177\). From \(\$ 76\),
\[
\mathrm{H}_{t}=\frac{45}{4} \cdot \frac{\mathrm{EI}}{c k^{2}} \cdot t e c=\frac{45}{4} \cdot \frac{\mathrm{EI}}{c k^{*}} \cdot \frac{u}{2} .
\]

As above,
\[
\mathrm{H}_{t}=\frac{12 \mathbf{E I}}{c^{2} k} v ;
\]
therefore
\[
u=\frac{32}{15} \cdot \frac{k}{c} \cdot v=\frac{8}{15} \cdot \frac{11}{\mathrm{AE}} \cdot \frac{3 c^{2}+2 k^{2}}{c},
\]
a quantity five-sixths of that for the ril) with hinged ends.
179. Circular Rib hinged at Ends. - It is more difficult to obtain the amount of detlection from change of form produced by the eompression at cach section of a circular rib, even approximately. As the equilibrium polygon for steady load will not deviate much from the axis of the rib, the thrust \(T\) may be assmmed to vary as secant \(\theta\), the inclination of the rib
at successive points the therizon: hence the shortening of a small portion, \(/ \mathrm{s}\), of are moter the thrust will be
as the section is constant,
\[
\begin{equation*}
s-s^{\prime}=\frac{\mathrm{II} r}{\mathrm{AE}} \int_{-\beta \cos H}^{+\beta} \frac{\lambda \theta}{\cos }=\log \frac{1+\sin \beta}{1-\sin \beta} \cdot \frac{\mathrm{II} r}{\mathrm{AE}} \tag{1.}
\end{equation*}
\]
(The symbol log denotes the hyperbolic logarithm; to obtain it, multiply the common logarithm by 2.30158 .)

As, with a small deflection, the rib will vary but slightly from its original form, let it be assumed to be an are of a circle after compression. We have then \(s-s^{\prime}=\varrho r^{\prime}-2 r^{\prime} \beta^{\prime}\), where \(r^{\prime}\) is the new radius, and \(p^{\prime}\) the new angle subtended by the halfarch. Now
\[
r=\frac{c^{2}+k^{2}}{2 k}, r^{\prime}=\frac{c^{2}+(k-r)^{2}}{2(k-c)} \text {, and } \sin \beta^{\prime}=\frac{c}{r^{\prime}} \text {. }
\]

By assuming a value for \(v, r^{\prime}\) and \(\beta^{\prime}\) can be obtained, and the value of \(2\left(r p^{\prime}-r^{\prime} \beta^{\prime}\right)\) calculated: if it agrees with the value \(s-s^{\prime}\) of equation (1.), the assumed \(v\) is sufficiently near the truth ; if not, the process of approximation may be repeated. We may iddopt, as a value which will answer very well in many cases, \(v=\frac{s-s^{\prime}}{p^{\prime}}\). Then
\[
v=\frac{\mathrm{H} r}{\Lambda \mathbf{E} \beta} \log \frac{1+\sin \beta}{1-\sin \beta} .
\]

This logarithmic expression may be written as a series,
\[
v=\frac{\mathrm{I} r}{\mathrm{~A} \mathbf{E}_{\beta} \beta}\left(\sin \beta+\frac{1}{3} \sin ^{3} \beta+\frac{1}{5} \sin ^{5} \beta, \& c .\right) .
\]

It was shown in \(\$ 30\) that the vertical deflections of two beams of the same cross-section and carrying the same gross load uniformly distributed. - one inclined at an augle \(i\), and the other the horizontal projection of the former. - were in the pro-
portion of \(1:\) cos \(i\). If, then, the load on the horizontal beam is increased in intensity in the ratio see \(i: 1\), the vertical deflections of the two beams will be the same. We desire to find the amome and distribution of load on a straight beam of the same span as the circular arch. Fig. 58, and the same cross-section, which shalt produce the same deflection at the middle. By what has just been stated, the load on any horizontal foot of a straight beam must be to the intensity on an inclined beam as \(\pi\) sec \(\theta\) to \(u\). A small portion of the areh \(d s=\sec \theta d x\) : hence it follows, that, if the areh is carrying \(w\) per horizontal foot over the whole span, a horizontal beam, as above, loaded with the varying intensity \(u \sec \theta=w \frac{d s}{d x}\) per foot, will have the same deflection. This load will be the projection of a load of uniform intensity measured along the rib, or the load on the beam is \(u^{\prime} s\), or \(2 w r r^{\prime}\), in our usual notation.

In any particular case we may easily solve the problem graphically. Lay off 1-2, Fig. 58, equal w. A B : divide A B into a number of equal parts, and 1-2 into the same number. with half-loads at 1 ant 2 as usual. Make \(2-0\) equal to \(H\) for this load, and, with 0 as a pole, draw the equilibrium polygon \(\mathrm{A}^{\prime} \mathrm{B}^{\prime}\), which, for an arch of moderate rise, will be a close approximation to a catenary. \(\mathrm{C}^{\prime} \mathrm{B}^{\prime} \cdot(0-2)\) will be the desired bending moment \(\Lambda_{n}\), for a deflection found by taking the area moment of \(\Lambda^{\prime} \mathrm{B}^{\prime} \mathrm{C}^{\prime}\) about \(\Lambda^{\prime}\), multiplying by \(0-2\), and dividing by EI. Use these values as we did those of \(\leqslant 17 t\). In constructing, increase the length of the rib by (1.) if thought desirable. The valnes of the following section may be taken if preferred.
180. Analytical Discussion. - The exact values may be deduced by the usual process for finding the deflection of a beam. If \(x\) is the, distance of any point of the beam from one abutment (Fig. 59), \(\beta\), the angle subtended at the centre by the half-arch, \(\theta\), the angle from the crown to any point whose projection is \(x\), and \(w\), the load per foot on the arch. and also at the middle of the beam, then \(x=r(\sin 3-\sin \theta), d x=-r \cos \theta d \theta\), the load at any point \(=w \sec \theta\) per foot, and load on \(d x=w \sec \theta d x\)
\(=-w r \sec \theta \cos \theta d \theta=-w r d \theta\). The load on one-half of the span is shown in the figure.
\[
\text { Loart on half-span }=\int_{0}^{c} w \sec \theta d x=w r \int_{0}^{\beta} d \theta=w r \beta \text {. }
\]

This expression is the reaction \(P_{1}\) at the abutment. If \(x^{\prime}\) is the distance from the abutment to any section at which we desire the bending moment. and the corresponding angle is \(u^{\prime}\), we have the bending moment
\[
\begin{gathered}
\mathrm{M}=\mathrm{P}_{1} x^{\prime}-\int_{0}^{x^{\prime}}\left(x^{\prime}-x\right) \mu^{\prime} \sec \theta d x \\
=w r^{2} \beta\left(\sin \beta-\sin \theta^{\prime}\right)-w r^{2} \int_{\beta}^{\theta^{\prime}}\left(\sin \theta^{\prime}-\sin \theta\right) d \theta \\
=w \cdot r^{2}\left(\beta \sin \beta+\cos \beta-\theta^{\prime} \sin \theta^{\prime}-\cos \theta^{\prime}\right),
\end{gathered}
\]
which becomes at the middle
\[
M(\max )=\pi r^{2}(\beta \sin \beta+\cos \beta-1)=\pi r(c \beta-k) .
\]

Writing the usual expressions for inclination and deflection, and dropping the accents, we have
\[
\begin{aligned}
{ }_{x} & =\int_{x}^{\mathrm{e}} \mathbf{\mathrm { E } I} d x=-\frac{w r^{3}}{\mathbf{E} I} \int_{0}^{\theta}(\beta \sin \beta+\cos \beta-\theta \sin \theta-\cos \theta) \cos \theta d \theta \\
& =-\frac{\mu r^{3}}{\mathbf{E} \mathbf{I}}\left(\beta \sin \beta \sin \theta+\cos \beta \sin \theta-\frac{3}{4} \sin \theta \cos \theta-{ }_{4}^{3} \theta+\frac{1}{2} \theta \cos ^{2} \theta\right) .^{*}
\end{aligned}
\]

The slope at the abutment, when \(\theta=3\), is \(-\frac{\psi_{4} r^{3}}{4 \boldsymbol{I}}\left(\beta \sin ^{2} \beta-\beta \cos ^{2} \beta+\sin \beta \cos \beta\right)\), which, if we remove \(\frac{\text { II }}{\mathbf{E I}}\), is the area of the half equilibrinm polygon \(\lambda^{\prime} B^{\prime} C^{\prime \prime}\) of Fig. 58. The deflection of the centre is
\[
\begin{aligned}
v=\int_{0}^{c} i d x & =\frac{w r^{4}}{\mathbf{E} \mathbf{I}^{\prime}} \int_{0}^{\beta}\left(\beta \sin \beta \sin \theta+\cos \beta \sin \theta-\frac{3}{4} \sin \theta \cos \theta-\frac{3}{4} \theta+\frac{1}{2} \theta \cos ^{2} \theta\right) \cos \theta d \theta \\
& =\frac{w r^{4}}{\mathbf{E I}}\left(\frac{1}{3} \beta \sin ^{3} \beta+\frac{7}{36} \sin ^{2} \beta \cos \beta-\frac{1}{4} \beta \sin \beta-\frac{1}{9} \cos \beta+\frac{1}{9}\right) .^{*}
\end{aligned}
\]
* These expressions are reduced. To aid any who desire to prove them, we give the following integrals: \(\int \theta \cos \theta d \theta=\theta \sin \theta+\cos \theta: \int \theta \sin \theta \cos \theta d \theta\) \(=-\frac{1}{2} \theta \cos ^{2} \theta+\frac{1}{4} \cos \theta \sin \theta+\frac{1}{4} \theta ; \int \cos ^{2} \theta d \theta=\frac{1}{2} \sin \theta \cos \theta+\frac{1}{2} \theta\); \(\int \theta \cos ^{3} \theta d \theta=\theta \cos ^{2} \theta \sin \theta+\frac{7}{9} \cos ^{3} \theta+\frac{2}{3} \theta \sin ^{3} \theta+\frac{2}{3} \sin ^{2} \theta \cos \theta\).

From this expression, by removing \(\frac{I}{E I}\), we obtain the area moment of \(\mathrm{A}^{\prime} \mathrm{B}^{\prime} \mathrm{C}^{\prime}\).

The quantities representing \(v\) and \(M\) will now be introduced in the equation of \(\S 179\) : hence we get
\(\frac{\text { II }}{A \beta} \log \frac{1+\sin \beta}{1-\sin \beta^{3}}=\frac{v \cdot r^{3}}{18 \mathrm{I}}\left(12 \beta \sin ^{3} \beta+7 \sin ^{2} \beta \cos \beta-93 \sin \beta-4 \cos \beta+4\right)\).
Find the value of \(M\) for the special arch, and value of \(\beta\), and also the value of \(r\). Let \(r \div \mathrm{M}=\mathrm{B} r^{2}\); then
\[
\mathrm{M}=\frac{\mathrm{I} r}{2 \mathrm{Br} r^{2} \Lambda \mathbf{E} \beta} \log \frac{1+\sin \beta}{1-\sin \beta} .
\]

If the arch is a semicircle,
\[
\mathrm{M}(\max )=\frac{1}{2} w r^{2}(\pi-2) ; i=-\frac{w r^{3}}{4 \mathbf{E I}} \cdot \frac{\pi}{2} ; v=\frac{w r^{4}}{3 \mathfrak{E} \mathrm{I}}\left(\frac{3}{2} \pi+4\right)
\]
181. Circular Rib Fixed at Ends. - From the method of treating the parabolic rib with fixed ends, as compared with the parabolic rib with hinged ends, we would suggest that the deflection and the bending moments at crown and springing of the circular arch with fixed ends, due to the compression of the rib from II, may be obtained from a drawing like Fig. 58, when -0 is made equal to the \(H\) of this case, by plotting the closing line of Fig. 27 on the arch of Fig. 58, at the height ahore \(A\) of \(r\left(\frac{\sin \beta^{\prime}}{\beta^{\prime}}-\cos \beta\right)\) (see ş 105), projecting the points of contraflexure vertically on \(\mathrm{A}^{\prime} \mathrm{B}^{\prime}\), drawing the horizontal closing line of this equilibrimm polygon, and then finding M and \(v\) for the beam fixed at the ends.

For circular arehes of morlerate rise, the treatment for parabolic arches will probably suffice.



\section*{CHAPTER XII.}

BRACED ARCH WITH HORIZONTAL MEMBER; OTHER SPECLAL FORMS ; CONCLUSION.
182. The Usual Analysis not Applicable. - The difficulty in the way of a successful application of the usual formula \(\leq \mathrm{EF} . \mathrm{DE}=0\) for the change of span of the braced arch with horizontal member, of, Fig. 60, or, as it is sometimes called, the rib with spandrel bracing, arises from the fact that the moment of inertia of successive cross-sections camot be left out of the equation as a constant. In fact, it varies rapidly; and its amount at any section is unknown until the sizes of the respective pieces are detcrmined. It was shown, in \(\$ 72\), that I must be placed in the denominator of the above formula: and, if not constant, it must come within the sign of summation.

This arch is pivoted at the springings, but continuous at the crown. If it were hinged at the crown by the omission of a piece in either the lower or the upper chord, the thrusts at the abutments could at once be determined by the principles of Chap. II. ; and a diagram by the method of Part I., "Roofs," would at once give the stresses in all the pieces for any given load. For the treatment of the ease represented in Fig. 60, the following practicable method is offered. It was published in "The Engineer," Feb. 10, 1873, and will also be found in the ninth edition of "The Cyclopredial Britamica," art. "Bridges," where it is attributed to Professor Clerk-Maxwell.
183. Change of Span from Stress in a Piece. - From previous statements, we know that the modulus of elasticity \(\boldsymbol{E}\) is the measire of the extensibility or compressibility of the kind of material to which it refers, so long as the stress does not surpass the elastic limit, and is equal to the quotient of the intensity of the stress on a cross-section divided by the extension or compression of a unit's length of the piece in which the stress is excrted. Thus, if \(l\) is the length of a piece in inches, \(A\) its cross-section in square inches, \(T\) the thrust or tension in pounds to which it is exposed, and \(\Delta l\) the change of leingth produced,
\[
\mathbf{E}=\frac{\mathrm{T}}{\mathrm{~A}} \cdot \frac{l}{\Delta l} ; \quad \text { or } \quad \Delta l=\frac{\mathrm{T}}{\mathbf{E} \mathrm{~A}} l
\]

If the piece A of the frame of Fig. 61 is changed in length, and every other piece is unchanged, while the portion of the frame to the right is held firmly in place, the span L of the frame will undergo an alteration \(\lrcorner \mathrm{L}\). In this case the motion takes place about the joint opposite to A , and we may write
\[
\Delta \mathrm{L}: \Delta l=a c: a b
\]
or the distance described by the point \(b\) for a small displacement around the axis " will be to the horizontal movement of \(d\) as the arm \(a b\) to the arm of \(d\), or \(a c\). A similar proportion will be true, if one of the lower chord pieces is supposed to alter in length. In case any diagonal is changed in length, as, for instance, \(f!\), the four-sided figure ef \(i g\) must alter to e \(f^{\prime \prime} z^{\prime} g^{\prime}\) of the sketch below, the point \(i\) turning about \(f\) as a centre, and the point \(g\) about \(e\) : hence, for a small displacement, the centre of motion is at the point of mecting, \(o\), of if and \(g\) e prolonged, which, for this arch, will lie in the upper chord; and the perpendicular \(p\), dropped on the line of the piece, will take the place of \(a b\) above.

18t. Stress in a Piece from \(\mathbf{H}\) and \(\mathbf{P}\). - Let \(t\) be the stress prorluced in a member by a horizontal foree H acting between the springing points. Then the principle of equality of moments as necessary for equilibrium about the point around

Which motion would otherwise begin, and which is no wher than the point noticed at the elose of the last section, will detemine the relation of the forces. A general rule for finding the axis about which rotation will begin is, Make a section which shall cut three pieces only; prolong the lines of two of the pieces until they meet: the moment of the stress in the third piece about that point of meeting will equal the moment of H about the same point. Hence we have, for the piece A
\[
\therefore \quad t \cdot a b=1 \mathrm{H} \cdot a c, \text { or } t={ }_{a b}^{a r} \mathrm{H} .
\]

Similarly, let \(t^{\prime}\) he the stress produced in A by a vertical force l'applied at one springing, while the other end of the frame is held rigidly so that it camont turn. As the arm of l ' will he \(d c\), we may write
\[
t^{\prime} \cdot a b=\mathrm{P} . a c, \text { or } t^{\prime}=\frac{d c}{a b} \mathrm{P} \text {. }
\]

The distances \(d c\) and \(a c\), being respectively horizontal and vertical, may be denoted in genemal for ans piece by \(x\) and \(y\). In order to make the symbol ib of the last section and of this one general, so as to apply to a diagonal is well as a chord piece. let as write for ab the perpendicular \(f\), drawn from the axis of rotation upon the action-line of the piece.

Any thrust at the springing having horizontal and vertical components H and P will produce a stress T in the piece, equal to \(t+t^{\prime}\), or
\[
\begin{equation*}
\mathrm{T}=\frac{a c \cdot \mathrm{H}+\| r \cdot \mathrm{P}}{a b}=\frac{\mathrm{II}_{y}+\mathrm{P} r}{\mu} . \tag{1.}
\end{equation*}
\]

It is erident that heed must he paid to the kind of stress produced \(\mathrm{l}, \mathrm{H}\) and P : thus, in any piece of the top member. 11 will produce tension and elongation, while P' will produce compression and shortening: the reverse will be trte of the lower member; how the diagonals are affected will be seen when we come to our application. Appropriate signs, therefore. must he given to the arithmetical values of the stress and alter-
ation of length; thus compression and shortening may be called positive, tension and lengthening, negative.
185. Formula for H. - From equations (1.) and (2.), § 183, upon writing \(y\) and \(p\), as indicated above, for \(a c\) and \(a b\), we get the change of span for any stress, T, in a particular piece,
\[
\Delta \mathrm{L}=\Delta l \frac{y}{p}=\frac{\mathrm{T} y}{p} \cdot \frac{l}{\mathbf{E} \mathrm{~A}^{\prime}},
\]
or, upon inserting the value of T from equation (1.). last section,
\[
\Delta \mathrm{L}=\frac{\mathrm{H} y^{2}+\mathrm{P} x y}{p^{2}} \cdot \frac{l}{\mathrm{EA}} .
\]

This same quantity can be calculated for the extensibility due to each member of the frame; and the result will not be altered by the slight yielding of all the others, muless this rielding produces sensible deformation, making appreciable changes in \(\frac{x}{p}\) and \(\frac{y}{p}\) : hence the sum of all the changes of span, or the total change of span, will be
\[
\mathrm{H} \Sigma \frac{y^{2}}{p^{2}} \cdot \frac{l}{\mathrm{EA}}+\Sigma \mathrm{P} \frac{x y}{p^{2}} \cdot \frac{l}{\mathrm{EA}} .
\]

If the abutments do not yield, this expression is zero. If the span changes, by a yielding of the abutments, so that \(e\) is the elongation of span for one ton of \(I I\), then the above expression for change of span equals \(e \mathrm{H} . \mathrm{P}\) is the vertical component of the reaction at one abutment, found as for any frame loaded as this arclr may be: hence \(H\) may be found. If the abutments do not yield, we then obtain
\[
\begin{equation*}
\mathrm{H}=\frac{\mathrm{\Sigma P} \frac{x y}{p^{2}} \cdot \frac{l}{\mathrm{EA}}}{\mathrm{\Sigma} \frac{y^{2}}{p^{2}} \cdot \frac{l}{\mathrm{EA}}} . \tag{1.}
\end{equation*}
\]
186. Application of Method. - Let a single weight, W. be applied at any one of the top joints of the braced arch, Fig. 60.

Inelined reactions will be produced at each abutment, whose components will be H and \(\mathrm{P}_{1}\) at the left, H and \(\mathrm{P}_{2}\) at the right. The calculations for the resulting stresses in the pieces are then best made as follows: Construct tables of the values \(x \div 1\), and \(y \div p\) for each member of the frame; the method of sections through the opposite joints, or of moments, will inswer best for the top and bottom members, and a diagram stuch as has been drawn for a roof, for the diagonals; assume a cross-section for each member for an assumed probable value of the abutment thrust; make tables of \(\frac{x y}{p^{2}} \cdot \frac{l}{\boldsymbol{A}}\) and \(\frac{y^{2}}{p^{2}} \cdot \frac{l}{\boldsymbol{E} \Lambda}\), or, what is equivalent when all the frame is of one material, so that \(\mathbf{E}\) is constant, make tahles of \(\frac{x y l}{p^{2} \Lambda}\) and \(\frac{y^{2} l}{p^{2} \Lambda}\). The summations indicated in (1.). \(\$ 18.5\), can then be made. In summing \(\mathrm{P} \cdot \frac{x^{\prime} y}{p^{2} \Lambda^{\prime}}\), the value \(P_{1}\) must be used for all pieces to the left of the loaded joint, and \(P_{2}\) for all pieces to the right of the load. Equation (1.), abore, will now give the value of If for this single load.

The process of finding the numerator of' (1.) must be repeated for each joint which is loaded. The abutment reactions having thas been found, the stress in each piece will be computed by (1.) § 184 , or will be sealed from a diagram drawn as in Part I., "Roofs." If, upon finding the maximun stresses in the pieces, resulting from the steady load and such rolling loads as will have the worst effect, the assumed seetions are not strong enough for these stresses, fresh cross-sections must be assumed, and the whole calculation repeated. The change in cross-sections will eause some change in the values of H ; but this tentative proeess need seldom he repeated but once.
187. Example; Stresses from H and P.-These processes will probably be rendered more clear by ann example. Let the arched frame of Fig. 60 be 120 feet in span, 12 feet rise to the curved member, and 17 feet rise to the straight member, making the depth at mid-span 5 feet. Let the upper member be divided into panels of 10 feet each. and the parabolic or circu-
lar are into portions of \(10.26: 3\) feet each. \({ }^{1}\) The radius of the emved member will be 156 feet. Let it be desired to design this arched structure to bear a steady load of ten tons per joint of the top member and a travelling load of the same intensity.

If a lorizontal line \(L \mathrm{O}\) is drawn to represent a certain value of H , we maty coustruct Fig. 62 by the method used in Part I., "Roofs," and by scale determine the magnitude of the stress in each piece due to this \(H\), as the only forme, applied as a thrust at each abutment; all of the stresses being measured as firurtions of H, and the kind of stress noted. One-half of the diagram is sufficient, as it will he symmetrical. The magnitude of any stress in a top or bottom piece can be readily proved by the method of moments. We may now fill the columns of a tahle with these ratios which represent ! \(\div p\), being not only the ratios of the stresses to \(H\), but or ane change of span to change of length. Bow s notation is used, and the stresses in one half of the frame will correspond with those in the other half. The sign + lenotes compression, the sign - denotes tension.
\[
\text { Values of } \frac{!}{p}
\]
\begin{tabular}{|c|c|c|c|}
\hline \(130-0.272\) & A \(\mathrm{L}+1.20: 3\) & \(0.1-0.444\) & A \(\mathrm{B}+0.150\) \\
\hline 1) \(0-0.669\) & \(\mathrm{CL}+1.520\) & B \(\mathrm{C}-0.47 \mathrm{O}\) & C 1) +0.450 \\
\hline F O - 1.117 & E L + 1.927 & I) \(\mathrm{E}-0.500\) & \(\mathrm{E} \mathrm{F}+0.502\) \\
\hline I \(0-1.67\) \% & G \(\mathrm{L}+2.427\) & F G-0.484 & (i I +0.488 \\
\hline KO-2.18. & .J \(L+2.942\) & I .J-0.384 & .J K +0.386 \\
\hline N O-2.400 & M1 \(1+3.293\) & K M - 0.153 & \(\mathrm{MN}+0.151\) \\
\hline
\end{tabular}

In the same way a diagram constructed upon a vertieal line which represents \(\mathrm{P}_{1}\), Fig. 63 , will give the stresses in the several pieres cansed by this vertical force only, applied in an upward direction at the left abutment, while the right end is held rigidly ia phace ly fixing the end brace in position. 'This figure will not be symmetrical, and therefore all the pieces must be entered in the table. \(P_{2}\) at the right abutment, in place of \(P_{1}\) at the left, will reverse the table, \(\mathrm{B}^{\prime} \mathrm{O}\) taking the place of BO O , se. 'The ratio of these stresses to I ' will give \(x \div p\).

\footnotetext{
\({ }^{1}\) If the arc is parabolic, the length of a piece will be 10.2 is feet. The difference is not material for our example.
}

Values of \(\frac{x}{p}\).
\begin{tabular}{|c|c|c|c|}
\hline \(\mathrm{B} 0+0.715\) & A \(\mathrm{L}-0.35 \cdot 1\) & \(0.1+1.155\) & A B - -1.1 \\
\hline 1) \(0+1.572\) & C L - 1.311 & 13 \(\mathrm{C}+1.005\) & ( 1 1) -1.750 \\
\hline \(\mathrm{F} 0+3.60{ }^{2}\) & 1: L - -.s:3: & 1) \(\mathrm{E}+1.5 \%\) & 1: \(\mathrm{F}-1.50\) ? \\
\hline I \(0+15.224 ;\) & ( L - 4.996 & F G + +2.214 & (: 1 -2.23) \\
\hline \(\mathrm{K} 0+0.310\) & . L - 7.757 & \(1 \mathrm{~J}+2.311\) & .J K-2.35: \\
\hline \(N 0+12.000\) & M L - 10.65 .5\()\) & KM + 1.90\% & M \({ }^{+}\)-1.920 \\
\hline \(\mathrm{K}^{\prime}()+13.10 \cdot 3\) & M' \(1 .-12.592\) & \(\mathrm{N}^{+} \mathrm{IJ}^{\prime}+0.833\) & M'K'-0. \\
\hline I' \(0+10.675\) & J \({ }^{\prime}\) L -12.975 & \(\mathrm{K}^{\prime} \mathrm{J}^{\prime}-0.371\) & . J \({ }^{\prime} \mathrm{I}^{\prime}+0.309\) \\
\hline \(\mathrm{F}^{\prime} 0+11.28: 3\) & \(\mathrm{G}^{\prime} \mathrm{L},-12.13{ }^{\text {c }}\) & \(\mathrm{l}^{\prime} \mathrm{G}^{\prime}-1.212\) & \(\mathrm{B}^{\prime} \mathrm{F}^{\prime}+1.2\) \\
\hline 1) \(0+9.698\) & E'L - 10.767 & \(\mathrm{F}^{\prime} \mathrm{E}^{\prime}-1 . \mathrm{f} .57\) & \(\mathrm{E}^{\prime} \mathrm{I}^{\prime}+1.664\) \\
\hline \(13^{\prime} 0+8.260\) & \(\mathrm{C}^{\prime} \mathrm{L}-9.387\) & 1)' \(\mathrm{C}^{\prime \prime}-1.87\) (; & ( \({ }^{\prime \prime} 1 \mathrm{~B}^{\prime}+1.850\) \\
\hline & \(\mathrm{A}^{\prime} \mathrm{L}\), - 8.13:9 & \(\mathrm{B}^{\prime} \mathrm{A}^{\prime}-1.367\) & \(\mathrm{A}^{\prime} \mathrm{O}\) \\
\hline
\end{tabular}
188. Computation of Tables. - We may now write a talle for \(\frac{y^{2} l}{p^{2}}\), and another for \(\frac{x y l}{p^{2}}\), for each piece of the frime. The first table, involving squares, will be positive throughout. The lengths of the horizontal and rib pieces will be multiplied by the footing of their respective columns to save labor; but the lengths of the diagonals are carried in as indicated.
\[
\text { Valces of } \frac{y^{2} l}{p^{2}} \text {. }
\]
\begin{tabular}{|c|c|c|c|c|c|c|c|c|c|c|}
\hline B O 0.07t & & I. \(1.44 \%\) & 0.1 & \(0.197 \times\) & 17.72 & \(=3.491\) & A I & B \(0.202 \times\) & 14.05 & 2.844 \\
\hline D O 0.409 & C & . 2.310 & 13 \({ }^{\prime}\) & U.204 \(\times\) & 14.33 & 3.267 & C & I) \(0.230 \times\) & 11.17 & 2.569 \\
\hline F O 1.24 & E & 3.718 & I) E & \(0.250 \times\) & 11.5 s & 2.595 & & F \(0.250 \times\) & 9.15 & 2.306 \\
\hline 1 () 2.816 & (i) & L. 5.540 & F 1 i & \(0.234 \times\) & 9.67 & 2.263 & G & I \(0.235 \times\) & 7.75 & 1.544 \\
\hline K 0 4.7.4 & . J & 8.65 .5 & I J J & 0.14i \(X\) & 8.25 & 1.213 & . & \(0.149 \times\) & 7.17 & 1.065 \\
\hline N05.0bl & M & , 10.844 & に】 & \(0.023 \times\) & 7.50 & 0.172 & M & F \(0.024 \times\) & 7.07 & 0.170 \\
\hline \multicolumn{2}{|l|}{\(15.150 \times 10\)} & 32.859 & & & & 13.301 & & & & 10.801 \\
\hline \multicolumn{2}{|l|}{\(9.320 \times 10\)} & 2 & & & & 2 & & & & 2 \\
\hline \multicolumn{2}{|l|}{244.000} & 65.115 & 10.26 & \(3=6 i 4.46\) & & 26.602 & & & & 21.602 \\
\hline
\end{tabular}

Summing these columns, and doulling for the whole arch, we obtain \(244.00+674.46+26.60+21.60=966.66=\leq . \frac{y^{2} l}{p^{2}} . \quad\) If, in the first trial, all the sections are supposed equal. A mar he omitted from (1.). S. 185, and 966.66 beeomes the denominator of that expression.

We next compute the following table, and multiply by the lengtl of each piece as we advance. It will be convenient to add other columns, marked \(\leq\). containing successive summations of the factors for each set of pieces, as these numbers will be used in turn. The summations are all negative, as will be readily seen, and hence the sign - is omitted.
\[
\text { Values of } \frac{x y l}{p^{2}} .
\]
\begin{tabular}{|c|c|c|c|c|c|c|c|}
\hline \multicolumn{3}{|c|}{\(\Sigma\)} & \multicolumn{2}{|l|}{\(\Sigma\)} & \multicolumn{2}{|l|}{\(\Sigma\)} & \(\Sigma\) \\
\hline \(13 O-1.95\) & 1.95 & A L - 4.37 & 4.37 & \(0 \mathrm{~A}-9.2 \overline{7}\) & 9.27 & A B -7.53 & 7.53 \\
\hline 1) \(0-11.96\) & 13.91 & C L - 20.92 & 25.29 & B C -10.30 & 19.57 & C D -9.54 & 17.07 \\
\hline F 0 O -40.90 & 54.51 & E L - 56.05 & 81.34 & D E - 10.84 & 30.41 & \(\mathrm{EF}-\mathrm{s} .63\) & 25.50 \\
\hline \(1 \bigcirc-104.4{ }^{\text {. }}\) & 159.25 & G L -124.44 & 205.5 & F G - 10.37 & 40.78 & G I - 8.44 & 34.14 \\
\hline K () - 203.6\% & 362.90 & .) L - 235.11 & 440.89 & I J - \(\mathbf{7 . 4 2}\) & 48.20 & J \(\mathrm{K}-6.51\) & 40.65 \\
\hline さ 11 - 2 SS.00 & 650.90 & M L -360.10 & 800.99 & KM - 2.19 & 50.39 & M N - 2.09 & 42.74 \\
\hline \(\mathrm{K}^{\prime}()-25 \% .61\) & \(93 \bigcirc .51\) & M \({ }^{\prime} \mathrm{L}-425.55\) & 1226.54 & \(\mathrm{N} \mathrm{M}^{\prime}+0.95\) & 49.44 & \(\mathrm{M}^{\prime} \mathrm{K}^{\prime}+0.10\) & 42.64 \\
\hline \(\mathrm{I}^{\prime} 10-219.69\) & 1151.20 & .J' L - 391.85 & 1618.39 & \(\mathrm{K}^{\prime} \mathrm{J}^{\prime}-1.1\) r & 50.61 & d' \(\mathrm{I}^{\prime}-0.85\) & 43.49 \\
\hline \(\mathrm{F}^{\prime} \mathrm{O}-126.013\) & 1272.23 & \(\mathrm{G}^{\prime}\) L - 302.23 & 1920.62 & \(1^{\prime} \mathrm{G}^{\prime}-5.68\) & 56. 29 & \(\mathrm{G}^{\prime} \mathrm{F}^{\prime}-4.55\) & 48.24 \\
\hline \(\mathrm{n}^{\prime} \mathrm{O}\) - 61.97 & 1339.20 & E. L - 212.94 & 2133.56 & \(\mathrm{F}^{\prime} \mathbf{E}^{\prime}-9.59\) & 65.85 & \(\mathrm{E}^{\prime} \mathrm{D}^{\prime}-\) \%.64 & 55.68 \\
\hline \(\mathrm{B}^{\prime} \mathrm{O}-22.47\) & 1361.67 & (1) L - 146.43 & 2280.00 & \(\mathrm{D}^{\prime} \mathrm{C}^{\prime}-12.85\) & 78.73 & (" \(\mathrm{B}^{\prime}-10.0\) \% & 65.15 \\
\hline & & A' L - 100.48 & 2380.48 & \(\mathrm{B}^{\prime} \mathrm{A}^{\prime}-10.76\) & 89.49 & \(A^{\prime} \mathrm{O}\) fixed. & \\
\hline
\end{tabular}
189. Values of \(\mathbf{H}\). - The calculations for H can now be proceeded with, and they are given below. An explanation of one computation will suffice for all. If a weight W is placed on the third upper joint from the left, the vertical component of the left abutment reaction, \(\mathrm{P}_{1}\), is \({\underset{2}{2} 9}_{19} \mathrm{~W}\). Then, for the two pieces of the upper chord to the left we have \(\leq \mathrm{P}_{1} \frac{x y}{p^{2}} l=13.91 \mathrm{P}_{1}\); for the two pieces of the rib to the left, we get \(25.29 \mathrm{P}_{1}\), and, for the five web-members to the left, \(30.41+17.07=47.48 \mathrm{P}_{1}\). On the right of the weight, the nine remaining pieces of the upper chord give \(\Sigma \mathrm{P}_{2} \frac{x y}{p^{2}} l=1277.23 \mathrm{P}_{2}\), which will he found opposite \(\mathrm{F}^{\prime} \mathrm{O}\), as the vertical force is now applied at the right end ; for the ten pieces of the rib we find \(2133.56 \mathrm{I}_{2}\), and for the rest of the web to \(\mathrm{E} F\) we find opposite \(\mathrm{E}^{\prime} \mathrm{F}^{\prime}\) and \(\mathrm{F}^{\prime} \mathrm{G}^{\prime}\), for the reason
just stated, \(65.88+48.04-113.92 \mathrm{P}\). As the piece E L, below the weight, is atted upon by \(\mathrm{P}_{1}\) on one side, and \(\mathrm{P}_{2}\) on the other, it makes no difference whether it is considered to lie to the left or the right of the loaded point. Adding up the respective numbers, multiplying one by \(\frac{19}{2 i}\), and the other by \({ }_{2}^{3}\), adding, and dividing by \(\sum \frac{y^{2}}{p^{2}} l=966.66\), we get \(\mathrm{H}=0.831 \mathrm{~W}\) for a load on the third joint only. The divisor \(966.66 \times 24=23,200\), is used.


Having completed the computations for six joints, we add the H's, and multiply by two, obtaining 10.764 W as the value of H for an entire loal of W on each upper joint.
190. Diagrams and Table of Stresses for Equal Crosssections. - We may now draw a diagram for a single load W on any one joint, plotting the reactions, just obtained, and proceeding by the method of Part I., "Roofs," Fig. 21. Six diagrams, four of which are drawn, the scale being too small to make the other two clear. Fig. 64, will give all the stresses, as, by symmetry, loads on the right will canse stresses in pieces marked with unaccented letters equal to those now found in pieces marked with accents. The stresses are scaled in tons, tabulated, and marked with their proper signs, in the following table. They might be calculated by (1.), § 184, if preferred, and their sum might be checked by a diagram for complete load. The sums of the respective compressions and tensions are written below, and in the next line are found the differenees of these quantities, or the stresses from steady load, marked S. L. \(\mathrm{U}_{\mathrm{p}}\) on adding to these latter the tensions or compressions first referred to, we obtain the maximum stresses in the pieces for a moving load of the same intemsity.

It will he seen that the horizontal member is always compressed ; the curved rib may have at times a little tension in its middle portion, lont the larger part of it is always compressed; the webl members are struts and ties alternately, mitil we reach JK; the pieces from there to the middle may be exposed to a reversal of stress.
191. Sections proportioned to Stresses. - (inided by these stresses, we will now assume sections of the different pieces, which shall vary approximately as do the stresses just found. Of the wel) members, those under compression are intended to be proportionately heavier than those in tension, as they will not safely resist so large a unit stress. The assumed ratio of the sections is marked on the figure. The quantities \(\frac{y^{2}}{p^{2}} \cdot \frac{l}{\Lambda}\) and \(\frac{x y}{p^{2}} \cdot \frac{l}{A}\) are now found anew ly simply dividing the previous similar quantities by the section ratios just referred to. The results follow on p. 184. \(=\frac{y^{2}}{p^{2}} \cdot \frac{l}{A}\) is now 161.18.

\section*{ARCHES．}

183

\section*{Stressfs in Pifces，All Cross－Section：Equal．}

\begin{tabular}{|c|c|c|c|c|c|}
\hline \[
\begin{gathered}
\text { AL. } \\
-0.13
\end{gathered}
\] & \[
\begin{gathered}
\text { CL. } \\
-0.13
\end{gathered}
\] & E L．
-0.10 & G 1．
-0.07 & J L．
-0.01 & I． \\
\hline \(+0.30\) & 9． 69 & 5 & －0．25 & －0．05 & ＋0．14 \\
\hline ＋0．70 & ＋10．17 & －0．69 & － & －0．31 & ＋ \\
\hline \(+1.0 \%\) & ＋0．72 & 1 & － & － & －0．04 \\
\hline & \(+\) & \(+\) & \(+\) & \(-1.06\) & － 0.44 \\
\hline ＋1．53 & \(+\) & \(+1.20\) & ＋0．73 & \(-0.04\) & \(-1.16\) \\
\hline & \(+\) & & & ＋0．56 & －0．26 \\
\hline ＋1．45 & ＋ & ＋1．45 & ＋1．30） & ＋0．93 & 32 \\
\hline & \(+1.27\) & ＋1．29 & \(+1.21\) & ， & 52 \\
\hline ＋0．90 & \(+\) & \(+0.97\) & ＋0．93 & ， & 46 \\
\hline ＋0．55 & & & ＋0．62 & 3 & ． 37 \\
\hline ＋0．19 & \(+1.20\) & 1．2 & 1 & ＋0．19 & 15 \\
\hline 10．S2 & 9.4 & 7.9 & 6.14 & 3.97 & 2.09 \\
\hline 0.13 & 0.52 & 1.14 & 1.78 & 2.10 & 1.90 \\
\hline \(+10.6\) & \(+5.96\) & \(+6.85\) & \(+4.36\) & \(+1.57\) & ＋0．19 \\
\hline \multicolumn{6}{|l|}{\multirow[t]{2}{*}{\[
\begin{array}{r}
+21.51+15.44+14.84+10.50+5.84+2.24 \\
-0.23-1.71
\end{array}
\]}} \\
\hline & & & & & \\
\hline
\end{tabular}
\begin{tabular}{|c|c|c|c|c|c|c|c|c|c|c|c|}
\hline Load Oll 1st． & \[
\begin{array}{r}
A B . \\
+0.01
\end{array}
\] & &  &  &  & \[
-0.04
\] &  & \[
-0.0 \mathrm{i}
\] & \[
-0.116
\] & ¢ & M N \\
\hline 2d．+0.80 & －0．82 & & & －0．07 & & & & \(-0.16\) & & ¢ & \\
\hline ＋ & －0．36 & & －0． & & & & & & & & \\
\hline 4th．+0.33 & － & & & & －0．79 & & & & & 2 & \\
\hline \(5 \mathrm{th} .+0.16\) & － & & & & & & & & & & \\
\hline fith． 0. & & & & & & & & & －0．74 & & ＋0．0 \\
\hline 7th． & & & & & & & & & & & \\
\hline －0．1 & & & & & － 0.04 & & －0．19 & \(+0.37\) & 0 & 1 & －1）．5\％ \\
\hline ． & & & & & & & －1）．12 & ＋18．25 & －13．20 & ＋0．85 & －11 \\
\hline 10th．-0.10 & & － & \(+0.07\) & － & ＋10．10 & ＋0．07 & \(-0.07\) & & －0．14 & 7 & －0．2 \\
\hline 11th．-0.07 & \(+\) & －0．1 & ． 05 & \({ }^{11} .0\) & ．n． & ＋10．03 & －10．02 & ＋0．09 & －0．09 & ＋0．15 & －0． \\
\hline 12th．-0.02 & ＋0．02 & －0．11 & ＋0．01 & 11.00 & 0.00 & ＋10．01 & －0．01 & \(+0.02\) & －0．02 & ＋10．05 & －1 \\
\hline
\end{tabular}
\begin{tabular}{|c|c|c|c|c|c|c|c|c|c|c|c|}
\hline ご＋2．92 & 0.54 & 2.90 & 0.27 & 2.8 & 0.24 & 3.03 & 0.45 & 3.09 & 1.19 & 2.3 .5 & 2.09 \\
\hline こ－ 0.51 & 1.90 & 0.25 & 1.87 & 0.17 & 1.85 & 0.27 & 2.05 & 0.79 & 2.20 & 1.47 & 2.60 \\
\hline s．L．+2.41 & \(-1.36\) & ＋2．65 & \(-1.60\) & \(+2.70\) & －1．61 & ＋2．66 & \(-1.60\) & \(+2.30\) & \(-1.03\) & \(+1.38\) & 9．00 \\
\hline ¿i \(1+5.33\) & & \(+5.55\) & & \(+5.57\) & & \(+5.99\) & & \(+5.39\) & ＋10．16 & ＋ 4.23 & 2．09 \\
\hline \(\geq 1\) & \(-3.26\) & & \(-3.45\) & & \(-3.46\) & & \(-3.65\) & & \(-3.25\) & －10．09 & \(2.00^{\circ}\) \\
\hline
\end{tabular}

\begin{tabular}{|c|c|c|c|c|c|c|c|}
\hline & \(\Sigma\) & & \(\Sigma\) & & \(\Sigma\) & & \(\Sigma\) \\
\hline \(130-0.78\) & 0.78 & A \(\mathrm{L}-0.21\) & 0.21 & \(0 \mathrm{~A}-1.54\) & 1.54 & A \(13-2.51\) & 2.51 \\
\hline D O - 1.99 & 2.77 & C L - 1.16 & 1.37 & 13 \(\mathrm{C}-1.12\) & 3.26 & C I) -2.38 & 4.84 \\
\hline H O - 4.09 & 6.86 & E L - 3.74 & 5.11 & 1) \(\mathrm{E}-1.81\) & 5.07 & E F -2.16 & 7.05 \\
\hline \(10-6.96\) & 13.52 & G L - 11.31 & 16.42 & F G - 1.83 & 6.80 & ( 1 I -2.11 & 9.16 \\
\hline Ki) -10.72 & 24.54 & \({ }^{\text {J }}\) L \(\mathrm{L}^{\text {- }} 33.59\) & 50.01 & I. J -1.48 & s.28 & . 1 K -2.17 & 11.33 \\
\hline N () -13.71 & 38.25 & M L - 144.04 & 194.05 & K M - 0.55 & 8.83 & M N - 1.05 & 12.38 \\
\hline \(\mathrm{K}^{\prime} \mathrm{O}-15.14\) & 53.39 & M' L, -170.22 & 364.27 & \(\mathrm{N} \mathrm{M}^{\prime}+0.48\) & 8.35 & \(\mathrm{M}^{\prime} \mathrm{K}^{\prime}+0.02\) & 12.2.6 \\
\hline I' O -14.18 & 67.57 & \(\mathrm{J}^{\prime}\) L -55.99 & 420.2 .5 & \(\mathbf{K}^{\prime} \mathbf{J}^{\prime}-0.39\) & 8.74 & \(J^{\prime} \mathrm{I}^{\prime}-0.17\) & 12.53 \\
\hline \(\mathrm{F}^{\prime} \mathrm{O}-12.60\) & 80.17 & ( \(\mathrm{i}^{\prime} \mathrm{L}\) - -27.49 & 47.73 & \(I^{\prime} \mathrm{C}^{\prime}-1.4{ }^{\text {a }}\) & 10.16 & \(\mathrm{G}^{\prime} \mathrm{F}^{\prime}-0.71 \%\) & 13.20 \\
\hline \(\mathrm{D}^{\prime}()-10.33\) & 90.50 & \(\mathbf{E}^{\prime} \mathrm{L}-14.20\) & 461.93 & \(\mathrm{F}^{\prime} \mathrm{E}^{\prime}-2.40\) & 12.56 & \(\mathbf{E}^{\prime} \mathrm{D}^{\prime}-1.2 \%\) & 14.56 \\
\hline \(\mathrm{B}^{\prime} \mathrm{O}-8.99\) & 99.4.7 & \(\mathrm{C}^{\prime} \mathrm{L}-8.13\) & 470.06 & \(\mathrm{I}^{\prime} \mathbf{( ' \prime \prime}^{\prime \prime}-3.21\) & 15.\% & \(\mathrm{C}^{\prime} \mathrm{B}^{\prime}-1.68\) & 16.24 \\
\hline & & \(\mathrm{A}^{\prime} \mathrm{L}-4.78\) & 474.84 & \(\mathrm{B}^{\prime} \mathrm{A}^{\prime}-3.55\) & 19.3. & \(\mathrm{A}^{\prime} \mathrm{O}^{\prime}\) fixed. & \\
\hline
\end{tabular}

The above summations are negative.
Next follow, as before, the computations of H (p. 185).

It will be seen that the change in the sections of the pieces has made but little change in the values of \(H\) : the thrust 10 w being 10.820 W for a steady load of W on each joint. We may therefore proceed to draw anew the diagrams for a single load W on any one joint, or we may, by the use of lines of another color, alter the figures already drawn. As \(H\) has been changed so little, the new stresses will determine the final
\begin{tabular}{|c|c|c|c|c|c|}
\hline \multicolumn{2}{|r|}{\multirow[b]{2}{*}{W on 1st Joint.}} & \multicolumn{4}{|c|}{Villes of H .} \\
\hline & & \multicolumn{2}{|r|}{W\% on ed Joint.} & \multicolumn{2}{|r|}{W on 3d Joint.} \\
\hline 0.00 & 99.49 & \(0 . \pi\) & 10.50 & \(2 . \% 5\) & 80.17 \\
\hline 0.00 & 4.4 .84 & 0.21 & \(4 \% 0.06\) & 1.37 & 461.93 \\
\hline 1.54 & 19.3.) & 3.26 & 15.\% & 5.07 & 12.56 \\
\hline 1.54 & 16.24 & 2.51 & 14.50 & 4.89 & 1:3.29 \\
\hline 23 & 609.92 & \({ }^{6.76}\) & 5!0. 5 ) & 14.10 & 507.95 \\
\hline \(\overline{35.42}\) & & 21 & 3 & 19 & 5 \\
\hline 609.92 & & 141.96 & 1720.67 & 267.90 & 2839.15 \\
\hline \multicolumn{2}{|l|}{\(\overline{645.34} \div 3868=.167 \mathrm{~W}\).} & \(17 \% 2.6 \pi\) & & \(2839 . \% 5\) & \\
\hline
\end{tabular}
\begin{tabular}{|c|c|c|c|c|c|}
\hline \multicolumn{2}{|r|}{W on 4th Joint,} & \multicolumn{2}{|r|}{W on 5th Joint.} & \multicolumn{2}{|r|}{W ou 6th Joint.} \\
\hline 6.86 & \(6 \pi .9 \%\) & 13.82 & 53.39 & 24.54 & 38.2. \\
\hline 5.11 & 447.58 & 16.42 & 420.25 & 50.01 & 364.27 \\
\hline 6.50 & 10.16 & 8.28 & 8.74 & 8.8:3 & 8.35 \\
\hline 7.0.) & 12.53 & 9.16 & 12.36 & 11.3:3 & 12.38 \\
\hline 2.58 & 537.99 & 47.68 & 494.54 & 94.11 & 423.2. \\
\hline \(1 i\) & \% & 1.) & 9 & 13 & 11 \\
\hline 438.94 & 3765.93 & 715.20 & 445..66 & 12:312:3 & 46.5.0.\%) \\
\hline 876\%.93 & & 44.5.f6 & & \(46.5 . \pi\) & \\
\hline
\end{tabular}
dimensions of the pieces. A sample of the stresses obtained in the upper chord is given below for comparison.
\begin{tabular}{rrrrrr} 
B O. & D O. & F O. & I O. & K O. & N O. \\
\(\mathrm{s}+1.45\) & 3.18 & 5.10 & 7.05 & 9.23 & 10.20 \\
\(\mathrm{~s}-0.42\) & 0.63 & 0.51 & 0.07 & 0.00 & 0.00 \\
S. L. 1.03 & 2.55 & 4.59 & 7.01 & 9.23 & 10.20 \\
Max. +2.48 & +5.73 & +9.69 & +14.09 & +18.46 & +20.40
\end{tabular}

A certain allowance in section may he made for the stresses from change of temperature, or the effect of the change of length in each piece may be worked out separately.
192. Bracing with Vertical Struts. - The bracing of the arch just described is of the Warren or triangular type. The design of Fig. 65 has been used with success, is probably more economical of material, and is, in our judgment, more pleasing to the eye. The inelined braces are ties, and the introduetion of the connters at the crown obviates the reversal of stress in the braces. When the upper member approaches the curved nember elosely at the crown, the web may be made of a plate for a distance of two panels: sometimes the two members are bronght into contact at the crown.
193. Cast-Iron Arch as a Breast-Summer.- Builders sometimes employ a east-iron member, shaped like Fig. 66, for spanning openings of considerable size, and carrying the weight of a brick wall. Aside from the fact that east-iron in large masses is of very uncertain strength, by reason of internal stresses prodnced by contraction in cooling, an additional element of monertainty is introduced by the method of construeting these ribs. The thrust of the arch is resisted by a wrought-iron rod, represented by a straight line in the figure, which, in place of being fastened by bolts or muts, is fitted into recesses in the casting at its ends. In order to have the rod tight, it is made shorter than the distance between bearings, then heated, and shrunk into place. The rod is therefore under an initial tension, and the rib under initial compression, both of which are likely to be of uncertain anount, and detrimental; for, when the arch is loaded, its horizontal thrust will be added to the tension in the bar, and the eompression of the rib will be inereased. As, however, the bar elongates under the pull, it would be well, were it possible, to have the bar so much shorter than the normal span of the areh, that the value of H proper to the areh under the proposed load should elongate the rod to that normal span; then the initial bending moments produced in the rib by shrinking on the rod will be removed. It wonld seem possible, by a eareful measurement of the extension of the rod between two marks some ten or twenty feet apart, especially if the stretch has been previously tested, to determine the initial tension.

If the arch is well built into the masonry at the ends, and if the bearings are long, the rib may be considered as fixed at the ends. If not so built, and in preliminary testing on two supports under an applied weight, the rib must be considered as pivoted at the ends. From the small rise, such arches may be assumed, in either ease, to be parabolie. In testing, therefore, under a single weight \(W\) applied at the middle, by \(\S 40\) \(\mathrm{H}=\frac{25}{6} \frac{9}{6} \mathrm{~W}\). At that time temporary bearings ought to be placed at A to prevent the areh from bearing at C when loaded. Under the load of the wall, unless the latter is cut by large openings, so that a pier concentrates the weight on a small portion of the rib, there will be no bending moments, as the load is uniformly distributed.
194. Gothic Rib for Roofs. - The rib which supports the roof of the Grand Central Depot in New-York City is probably circular, and will be analyzed readily by the principles already laid down; but the Gothic rib requires some special treatment. Fig. 67 is a sketch of the rib which sustams the roof over the train-honse of the Boston and Providence Railroad Depot in Boston, Mass. The span is 125 feet between walls, and the height is 55 feet to the axis of the rib. As height impresses one more than horizontal distance, it is evident that this roof appears lofty when viewed from the inside. In order to give height quickly near the walls, the half-rib is struck with two radii, as indieated in the figure. The lower portion is built with a solid web; while most of the upper portion has a uniform depth of three feet. If the junction at the crown or apex of the roof allows any movement, if the ribs can rock or turn on castings at their bases, and if they are independent of the side walls, they may he treated as linged at three points, and discussed like any three-hinged arch. If there is no opportunity for movement at the bases, and especially if the ribs about closely against the side walls and buttresses, while still a joint is provided at the crown, the condition of invariability of span must be applied, and also the condition that the deflection of
the erown when measured by area moments from the tangent at one abutment shall equal the deflection of the crown fiom the tangent at the other abutment. The integration will then be between limits which will appear from the discussion of the third supposition.

The rib may be fixed at the ends and crown, and will then offer a tronblesome case for treatment by reason of the great dep,th at the haunehes, unless we assume that it is well buttressed by the wall. In this case, the portion below the top of the wall and the wall itself will act as an abutment ; and, as it will only require a moderate tension in the inside flange at the springing to resist the overturning moment, such an assumption seems entirely warrantable. Above the wall, then, some 25 feet high, where the horizontal mark is made on the left-hand side, we assume the springing line of the arch, and consider the remainder as a rib fixed at the ends. and continuous at the (rown. In applying the conditions for a ril) with fixed ends to this case, we must change the derived equations, as the curve is not continuous at the crown. A parabola drawn through the middle of the depth of the rib at erown, springing, and a third point near the upper end of the straight portion of the rafter, will agree very elosely with the axis of the rib throughont. We must first determine \(k\) and \(c\) for this parabola. In Fig. 68 let \(h\) be the height or rise of the arch at the apex, a the horizontal distance from \(h\) to the point where the parabola would become horizontal ; then
\[
h=\frac{k}{c^{2}}\left(c^{2}-a^{2}\right) ; \text { or } k=h \frac{c^{2}}{c^{2}-a^{2}} .
\]

For another ordinate \(h^{\prime}\), distant \(c-a^{\prime}\) from the springing, we write
\[
k=h^{\prime} \frac{c^{2}}{c^{2}-a^{\prime 2}} .
\]

In this case \(c-a=55.75\) feet, \(h=30.3\) feet, \(c-a^{\prime}=22.5\) feet . and \(h^{\prime}=17\) feet: hence we find that \(k=31.68\) feet, \(c=70.48\) feet, and \(a=14.7\) :3 feet.

In place of performing the integrations of \(5.58-59\) between the limits there given. We must omit or subtract from the ergations the integral. between the limits for and -u, as this portion is cut out of the parabola. Thus the equation (1.) of sis will be written
\(\int_{0}^{2 c}\) D \(\mathrm{E}^{2}-\int_{c-a}^{+a}\) D \(\mathrm{E}^{-}=\int_{0}^{c+b} \mathrm{DF}\). DE \(-\int_{c-a}^{c+a} \mathrm{DF}\). D \(\mathrm{E}+\int_{0}^{c-b}\) D F . D) E .
As limits \(c+a\) and \(c-a\) will yield terms similar to limits \(c+b\), and \(c-b\), the subtractive quantities above can be witten from inspection of (2.), 5.58, and (2.), §39. A similar treatment of the other equations of condition will be required. The solution will then proceed as usual.

If the weight at the apex of the roof, arising from the rentilator, Éc., is sufficiently great, it will take the place of the omitted portion of breadth \(2 a\). so that the rib will be very nearly in equilibrium under steady load.
19.5. Remarks on Designing. - The examples which have been given in the preceding pages will indicate the steps to be pursued in working out a specific design. The type of structure having been determined upon, the moving load must be taken of an intensity in hamony with the position of the bridge. or we must deride upon the weight of show and pressure of wind to which the roof will be liable. The dead weight of the structure must then be assumed, of such an anoment as our judgment and experience dictate. to be atterwards verified and corrected from the actual sections. The abutment reactions and bending moments from the applied forces will then be fomul, after which, stress diagrams may be constructed, or equilibrium polyons drawn: from the first we obtain stresses direct ly, as in Part I.; from the second. benting moments, with shear:and direct thrusts. from which the stresses in the several piecon will be found, as in Part II. The first method is probably thee shorter for roofs, unless the rib is solid, or has a plate web, as all of the load of one kind may be included at one operation: the second method will be preferred where a moving load has to he
considered. The stresses will then be tabulated, and the maximum compression and tension on each piece found.
A point which may call for a little explanation is illustrated by Fig. 69. We desire to draw a stress diagram for an arehed rib, which is fixed at the end \(A B\), the equilibrium curve beginning with the line G D, and the bending moment at A B being \(\mathrm{T} \cdot p\), or its equivalent. The flanges at A and B will transmit direct force only: therefore decompose T into C , the compression parallel to the flanges, at the springing, and F , the shear at right angles. Then, by moments about A, Thrust at \(B . A B=C . A G\). or Thrust at \(B=\frac{C \cdot A G}{A B}\); by moments about B , Tension at \(\mathrm{A}=\frac{\mathrm{C} \cdot \mathrm{BG}}{\mathrm{AB}}\). The shear F will be resisted either at A or B, depending upon which of the braces is designed to carry it: if the braces are ties, it must pass through the one at A. Thus we obtain the forces with which to begin the stress diagram. In case of a hinge at the abutment. the point \(G\) is found midway between \(A\) and \(B\), and there will be \(\frac{1}{2} C\), compression, at each flange. \(F\) will be found in the proper brace as above.

The arched rib must be thoroughly stayed laterally ; for so much of either flange as is compressed is in unstable equilib. rium ; between lateral stays, the breadth of a compressed flange must be determined from the formule for columns. For formulee and directions for detailing, see the author"s "Structural Mechanics."

\section*{APPENDIX.}

\section*{196. Circular Arch with Springings 70 Degrees from the Crown.}
\begin{tabular}{|c|c|c|c|c|}
\hline \(a\). & \(!1 / 1\). & \(1 / 0\) & \(y_{2}\). & II. \\
\hline \(0{ }^{3}\) & \(+.1211 r\) & . \(8369 r\) & . \(1211 r\) & .6.58W \\
\hline i) & + .0980 & .832\% & .14\%.) & (6.5) \\
\hline 1) & \(+.0690\) & .830) & . 1690 & .6831 \\
\hline 15 & \(+.0316\) & .898\% & . 1862 & . 58 \\
\hline 20 & - . 0162 & . 8281 & . 2000 & . 505 \\
\hline 2.\()\) & - . 0688 & .8214 & .2145 & . 460 \\
\hline 30 & - .1352 & . 8141 & - \(22 \times 1\) & . 390 \\
\hline 3.5 & - .2935 & . 8086 & .2396 & . 315 \\
\hline 40 & - . 2145 & . 8030 & .2487 & . 237 \\
\hline 4. & - . 5096 & . 2946 & .2591 & . 171 \\
\hline 50 & - . \(76 \% 6\) & .7870 & .2670 & . 108 \\
\hline 5.) & - 1.2000 & . 7380 & .2790 & . 066 \\
\hline 81 & -2.7570 & \(.74 \% 0\) & . 2960 & . 016 \\
\hline 6.) & \(-4.5410\) & . T®き0 & . 2980 & . 006 \\
\hline
\end{tabular}
197. Parabolic Rib, Hinged at Crown Only.-As the rib hinged at crown and springings, the rib hinged at springings only, and the rib with no linges have been treated, the analysis of a rib with a linge at the crown will make the discussion complete. The case is shown in Fig. 43B, Plate V.

There are two unknown quantities, \(y_{1}\) and \(y_{2}\), the ordinates at the two abutments. Hence we must apply two conditions. Under a given load the horizontal and the vertical displacement of the hinge \(Q\) at the crown, when computed for the left half of the rib, must be equal to the corresponding displacements of
the hinge Q for the right lialf of the rib. The signs of these displacements in the necessary equations will be the same as the signs of the bending moments. In the left half of a rib a positive bending moment will tend to canse a horizontal movement to the left, and a negative moment will tend to canse the same kind of a horizontal movement in the right half. The eoordinates of the hinge Q from any point E where a bending moment is felt are EK and K Q. Hence the erquation for the horizontal displacement of Q may be written
\[
\Sigma E F \cdot E K \text { from } A \text { to } Q+\Sigma E F \cdot E K \text { from } B \text { to } Q=0
\]

An upward deflection in both halves of the rib will be caused by positive moments and a downward deflection by negative moments. Hence for vertical displacement of the point Q,
\[
\Sigma E F \cdot K Q \text { from } A \text { to } Q=\Sigma E F \cdot K Q \text { from } B \text { to } Q \text {, (2.) }
\]

Apply these equations to the parabolic rib, using the notation employed in elhapter IV., but measuringe \(x\) from the middle ordinate through Q .
\[
\begin{aligned}
& \mathrm{D} \mathrm{E}=\frac{k}{c^{2}}\left(c^{2}-x^{2}\right) ; \quad \mathrm{E} \mathrm{~K}=k-\mathrm{D} \mathrm{E}=\frac{k}{c^{2}} x^{2} . \\
& \mathrm{E} \mathrm{~F}=\mathrm{D} \mathrm{E}-\mathrm{D} \mathrm{~F} . \\
& \text { From A to } \mathrm{Q}, \quad \mathrm{DF}=y_{1}+\frac{y_{11}-y_{1}}{c+b}(c-x) \\
& \text { " } \mathrm{Q} \text { to } \mathrm{N}, \quad \mathrm{DF}=y_{1}+\frac{y_{0}-y_{1}}{c+b}(c+x) . \\
& \text { " } \mathrm{N} \text { to } \mathrm{B}, \quad \mathrm{DF}=y_{2}+\frac{y_{0}-y_{2}}{c-b}(c-x) .
\end{aligned}
\]

Hence for the first condition
\[
\begin{aligned}
& \frac{k}{c^{2}} \int_{0}^{c}\left[y_{1}+\frac{y_{0}-y_{1}}{c+b}(c-x)-\frac{k}{c^{2}}\left(c^{2}-x^{2}\right)\right] x^{2} d x+\frac{k^{2}}{c^{2}} \int_{v}\left[y_{1}+\frac{y_{0}-y_{1}}{(c+v}(c+x)\right. \\
& \left.\quad-\frac{k}{c^{2}}\left(c^{2}-x^{2}\right)\right] x^{2} d x+\frac{k}{c^{2}} \int_{b}^{c}\left[y_{2}+y_{0}-y_{2}(c-b-x)-\frac{k}{c^{2}}\left(c^{2}-x^{2}\right)\right] x^{2} d x=0
\end{aligned}
\]

Integrating and reducing we have
\[
\begin{align*}
& y_{0}\left(2 c^{5}-2 c b^{4}\right)+y_{1}\left(3 c^{5}+c^{4} b-4 c^{3} b^{2}+c b^{4}-b^{6}\right) \perp \\
& y_{2}\left(3 c^{5}-c^{4} b-4 c^{3} b^{2}+c b^{4}+b^{5}\right)={ }_{5}^{5_{5}^{6}} k c^{3}\left(c^{2}-b^{2}\right), \tag{3.}
\end{align*}
\]

The second condition, equation (2.), is erpuivalent to equating the moments of the areas between the rib and the polyem about the middle ordinate through \(Q\), or
\[
\begin{gathered}
\frac{1}{b} c^{2}\left(k-y_{1}\right)+\frac{1}{2} c^{2} y_{1}=\frac{1}{2} l^{2} k+\frac{1}{b} b^{2}\left(y_{0}-k\right)+\frac{1}{6}(c+2 b)(c-b)\left(y_{0}-y_{2}\right) \\
+\frac{1}{2} y_{2}\left(c^{2}-b^{2}\right), \quad(4 .)
\end{gathered}
\]

Since one side of the polygon always passes through the hinge at the crown,
\[
\begin{equation*}
y_{1}=\frac{(r+b) k-c y_{0}}{b}=\stackrel{(1+n) k-y_{0}}{n}, \tag{5.}
\end{equation*}
\]

Combining (3.). (t.), and (5.), and substitnting \(n c\) for \(b\), we get
\[
y_{0}=\frac{k}{5} \cdot \frac{11 n^{2}+22 n+15}{n^{2}+2 n+3} ; \quad y_{2}=\frac{k}{5} \cdot 5 n^{3}+11 n^{2}+11 n-3 .
\]

COMPCTED VATIVES.
\begin{tabular}{|c|c|c|c|c|c|}
\hline \(n=\) & 0 & . \({ }^{2}\) & . 4 & . 6 & . 8 \\
\hline \(y_{0}=\) & 1.0 & 1.15 & 1.29 & 1.41 & \(1.51 k\) \\
\hline \(y_{1}=\) & 0.2 & 0.23 & \(0.2 \%\) & 0.32 & \(0.36 k\) \\
\hline \(y_{2}=\) & 0.2 & 0.02 & - -0.29 & -0.9\% & \(-2.94 k\) \\
\hline \(\mathrm{H}=\) & 0.62 .5 & 0.459 & 0.300 & 0.152 & \(0.044 \frac{c}{k} \mathrm{~V}\) \\
\hline \(\mathrm{P}_{1}=\) & 0.5 & \(0.35 \%\) & 0.208 & 0.104 & 0.028 W \\
\hline \(P_{2}=\) & 0.5 & 0.648 & 0.792 & 0.896 & 0.97 .2 W \\
\hline
\end{tabular}

Temperature Stresses. - As hoth halves of the rib expand or contract equally, the hinge Q will always remain at the middle of the span, so far as change of temperature is concerned. Therefore the horizontal displacement of the point \(Q\) from the moments in \(\mathrm{A} Q\) must be equal and opposite to the change in
span of the half-areh, if free, due to the change of temperature. The line which corresponds to the equilibrium polygon must be horizontal and must pass through the hinge. Hence (right. hand sketch)
\[
\begin{aligned}
& \mathrm{H}_{t} \cdot \mathbf{\Sigma} \mathrm{E} \mathrm{~F}^{2} \text { from } \mathrm{A} \text { to } \mathrm{Q} \text { or } \mathrm{B} \text { to } \mathrm{Q}=\mathbf{E} \mathbf{I} \cdot t e c . \\
& \mathbf{E F}=\frac{k}{c^{2}} x^{2} . \quad \mathrm{H}_{t} \int_{0}^{c} \frac{k^{2}}{c^{4}} x^{4} d x=\mathbf{E} \mathbf{I} \cdot t \in c . \\
& \quad \mathrm{H}_{t}=\frac{5 t e \mathbf{E} \mathbf{I}}{k^{2}} \\
& \mathbf{M} \text { max. }=5 \frac{t c \mathbf{E} \mathbf{I}}{k} \text { at each abutment. }
\end{aligned}
\]

Anches.

rig. 3.

Fig. 1 .





F'ig. 2.





Archer










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[^0]:    *In Fig 18, the point $f$ should bisect e $\boldsymbol{\prime}$.

[^1]:    * Compare § 84.

[^2]:    

