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"COMPARATIVE NAVAL ARCHITECTURE OF MODERN FOREIGN SUBMARINES"

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COMPARATIVE NAVAL ARCHITECTURE OF MODERN FOREIGN SUBMARINES

by

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S.B. Mechanical Engineering, Massachusetts Institute of Technology, May 1980

SUBMITTED TO THE DEPARTMENT OF OCEAN ENGINEERING IN PARTIAL FULFILLMENT OF THE REQUIREMENTS FOR THE DEGREE OF

MASTER OF SCIENCE

at the

MASSACHUSETTS INSTITUTE OF TECHNOLOGY

May 1988

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COMPARATIVE NAVAL ARCHITECTURE OF MODERN FOREIGN SUBMARINES

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by

John K. Stenard, LT, USN

Submitted to the Department of Ocean Engineering on May 27, ¹⁹⁸⁸ In partial fulfillment of the requirements for the degree of Master of Science.

Abstract

A comparative design study of ten conventional and nuclear-powered fast attack submarines is performed. Data sources are limited to those available in the open literature. The analysis is confined to those submarines which are of the greatest interest and for which enough design information is available to conduct an adequate study. The data for each of the selected submarines is then parameterized, analyzed, and compared on the basis of design and military capabilities. The design philosophy and top level requirement of each submarine Is then Inferred from its naval architecture and military capabilities. It is concluded that automation of systems will allow a reduction of crew size, which then permits ^a larger battery and greater provision, fuel, and weapons loadouts. This will lead to greater combat effectiveness due to increased range, attack flexibility, speed, and weapons delivery potential.

Dedication

I dedicate this work to the hope that through the maintenance of a strong and effective defense by the United States, the world may avoid the waste and tragedy of armed conflict.

^I extend my sincere appreciation and thanks to Professor Paul E. Sullivan, for educating me on submarine design parameters, greatly assisting me In the extensive literature search, and helping me define the focus of this study, and whose patience during the preparation of this document allowed me the freedom to be most effective.

My sincere thanks and admiration go to Harry Jackson, P.E., CAPT USN (Ret.) who. although known as a world-class expert on submarine design, extended to me an opendoor policy to his home and personal library, and who provided mature engineering guidance to me on several occasions as ^I developed the computer models of each of the submarines.

^I wish to thank my parents, John D. and Ann E. Stenard, for always being loving and supportive of me, my brothers and sisters, and my family.

My special thanks go to my wife, Amy, for being the love of my life, and for always standing by me, as my partner for life. She also contributed immeasureably to the quality of this document by proofreading it. My special thanks also to our two sons, John G. and James, for being such good little guys.

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Number Submarine

#1 KILO

#2 WALRUS

#3 RUBIS

#4 BARBEL

#5 TYPE 2400

#6 TYPE 1700

#7 TYPE 2000

#8 SAURO

#9 VASTERGOTLAND

#10 MIDGET 100

Chapter ¹

INTRODUCTION

The Introduction of the submarine added a new dimension to the conduct of naval conflict; that of a potent undetected threat within striking distance. The ability of the submarine to travel from place to place and observe events undetected usually gives to the submarine the ability to attack first (or to decide not to attack) and has always been its greatest asset. The traditional weapon of the submarine has been the torpedo, which because of its underwater attack mode is particularly damaging to surface ships.

Today, the ability of the submarine to remain undetected is still its greatest asset. Technical advances in hydrodynamics, propulsion plant design, and acoustic silencing have made modem submarines more difficult to detect than ever. Similarly, the firepower of the submarine has increased greatly due to technical advances in submarine launched weapons systems.

Many nations include submarines as an important part of their fleet. Several navies consider their submarines to be their capital ships, and employ them for many peacetime uses. Some of the peacetime uses are oceanographic exploration and surveillance.

The primary wartime role of the submarine could be considered to be the same as it always has been, that of interdiction of sea traffic lanes, but the methods of accomplishing this task have been expanded, since most modern submarines are capable of loading mines and encapsulated cruise missiles as well as torpedoes.

The mining capability allows a nation to restrict or deny the use of a port or seaway choke-point to an adversary. This is a very important capability, and is possible for only a submarine in many cases, since a submarine can conduct mining operations under

conditions infeasible for aircraft or surface ships. In addition, the mining can be conducted In a covert manner, which Is essential In this day of cruise missile shore batteries.

The capability of a submarine to carry cruise missiles gives it the medium-range (50 nautical mile) stand-off attack mode against surface targets. This mode was previously the province of only surface ships and attack aircraft. Long-range strategic nuclear cruise missiles and rocket-propelled homing torpedoes have also been discussed and are in development for attack submarine loadout.

The sophistication of modern torpedoes has increased their range, speed, probability of hit, and overall lethality. While this thesis does not discuss weapons effects, it is generally accepted that a subsurface explosion is much more damaging to a surface ship than an equally-sized explosion in the superstructure. The weapon of choice for attack submarines is still considered to be some variation of the torpedo.

This thesis focuses primarily upon basic mission capabilities such as number and type of weapons carried, maximum speed, maximum mission length, submerged endurance range, and indiscretion rate of diesel-electric submarines. One small nuclear-powered craft is included for comparison. All of the submarines selected for analysis are "attack boats", as opposed to strategic nuclear ballistic missile submarines.

Design data for the craft studied in this thesis is analyzed in a comparative technique, which starts with a gross characteristics comparison. After gross differences are identified, a detailed study of several aspects of the designs is undertaken. Emphasis is placed upon identifying design differences, and on trying to establish the reason for these differences.

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Chapter 2

PURPOSE

The purpose of this thesis is twofold:

(1). To determine the capability of each of the selected submarines in terms of primary mission areas, which are generally of a military nature.

(2). To gain a greater understanding of naval architecture in general and submarine design in particular.

Chapter 3

SUMMARY OF SUBMARINES

The literature search having been conducted, the below listed submarines have been selected for Inclusion in the detailed analysis portion of this study. They are ilsted in order of decreasing displacement, followed by the builder's name, country of origin, and year the lead ship was launched.

1) KILO (Komsomolsk Shipyard, Union of Soviet Socialist Republics, 1980).

2) WALRUS (Rotterdamsche Droogdok Maatschappij B.V., The Netherlands, 1985)

3) SSN RUBIS (Cherbourg Naval Dockyard. France, 1979).

4) BARBEL (Portsmouth Naval Shipyard. United States, 1959).

5) TYPE 2400 "UPHOLDER" (Vickers Shipbuilding and Engineering Ltd.. Great Britain. 986).

6) TYPE 1700 (Thyssen Shipyard. Federal German Republic, 1982).

- 7) TYPE 2000 (Ingenierkontor-Lubeck, Federal German Republic, 1983).
- 8) SAURO (Fincantieri Shipyard. Italy. 1979).

9) VASTERGOTLAND (Kockums Shipyard. Sweden, 1986).

10) MIDGET 100 (Sub Sea Oil Services of Micoperi. Italy, 1984).

The BARBEL Class is included because it was the last diesel-electric submarine class to be constructed by the United States. The KILO Class is included because of its interest and widespread use among Communist Bloc and allied nations, and because it represents a state-of-the-art Soviet diesel-electric submarine. The RUBIS. a small nuclear-powered submarine, is included in the study to show the impact of its propulsion plant, compared to other designs.

The following pages summarize the gross attributes of the above selected submarine

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classes.
KILO Komsomolsk Shipyard Union of Soviet Socialist Republics 1980 Submerged Displacement: 3200 Lton Surface Displacement: 2500 Lton Standard Displacement: 1900 Lton (Estimate) Length: 229.6 ft Surfaced Draft: 23.0 ft Diameter: 29.5 ft Complement: 55 Men. Prime Mover Type: Diesel Engine/Storage Battery Diesel/Alternator Capacity: 44 80 KW Main Propulsion Motor Power: 4000 HP Maximum Submerged Speed: 18 Kts (Calculated) Maximum Surface Speed: 12 Kts (Estimate) Maximum Snorkel Speed: 10 Kts (Estimate) Diving Depth: 300 meters (Estimate) Overall Endurance Range at Six Kts: 5760 Nm Overall Endurance Range at Ten Kts: 9600 Nm Maximum Mission Duration: 45 Days Active Sonar Passive Sonar Array Sonar Navigation Radar Electronic Surveillance Gear. Number of Torpedo Tubes: 8 Number of Reloads Carried: 10 Cruise Missile Capable. May carry and launch a maximum of 18 SSN-21 Capable of Minelaying Maximum Possible Number of Mines Carried: 20 Not Capable of Delivering Swimmers

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WALRUS Rotterdamsche Droogdok Maatschappij B.V. The Netherlands 1985 Submerged Displacement: 2800 Lton Surface Displacement: 2450 Lton Standard Displacement: 1900 Lton Length: 223.1 ft Surfaced Draft: 21.6 ft Diameter 27 . 6 ft Complement : 50 Men. Prime Mover Type: Diesel Engine/Storage Battery Diesel/Alternator Capacity: 5170 KW Main Propulsion Motor Power: 5360 HP Maximum Submerged Speed: 20 Kts Maximum Surface Speed: 12 Kts Maximum Snorkel Speed: 12 Kts Diving Depth: In excess of 300 meters. Overall Endurance Range at Six Kts: 10080 Nm Overall Endurance Range at Ten Kts: 7178 Nm Maximum Mission Duration: 70 Days Active Sonar Passive Sonar. Array Sonar. Navigation Radar Electronic Surveillance Gear. Number of Torpedo Tubes: 4 Number of Reloads Carried: 20 Cruise Missile Capable. May carry and launch the SubBarpoon. Max number Carried: 26 Can carry and emplace 40 mines.

RUBIS Cherbourg Naval Dockyard **France** 1979 Submerged Displacement: 2670 Lton Surface Displacement: 2385 Lton Standard Displacement: 2250 Lton (Estimate) Length: 236.5 ft Surfaced Draft: 21.0 ft
Diameter: 24.9 ft Diameter: Complement: 9 Officers, 57 Enlisted Men Prime Mover Type: Nuclear Reactor, Liquid Metal Cooling Prime Mover Power: 48,000 KW Main Propulsion Motor Power: 10,000 HP Maximum Submerged Speed: 25 Kts Maximum Surface Speed: 20 Kts (Est.) Diving Depth: In excess of 300 meters. Overall Endurance Range at Six Kts: 8640 Nm Overall Endurance Range at Ten Kts: 14400 Nm Maximum Mission Duration : €0 Days Active Sonar Passive Sonar. Array Sonar. Navigation Radar Electronic Surveillance Gear Number of Torpedo Tubes: ⁴ Number of Reloads Carried: 10 Cruise Missile Capable. May carry a maximum of 14 SM-39 cruise missiles Can carry and place 20 mines

BARBEL Portsmouth Naval Shipyard United States 1959 Submerged Displacement: 2369 Lton Surfaca Displacamant: 2315 Lton Standard Displacamant: 2146 Lton Length: 219.1 ft Surfaced Draft: 28 ft Diamater: 29 ft Complement: ⁸ Officers, 69 Enlisted. Prima Mover Type: Diesel Engine/Storage Battery Diesel/Alternator Power: 3580 KW Main Propulsion Motor Power: 3150 HP Maximum Submerged Speed: 18 Kts (Calculated) Maximum Surface Speed: 15 Kts Maximum Snorkel Speed: 10 Kts Diving Depth: In excess of 120 maters. Overall Endurance Range at Six Kts: 8640 Nm Overall Endurance Range at Ten Kts: 9897 Nm Maximum Mission Duration: 60 Days Active Sonar Passive Sonar. Array Sonar. Navigation Radar Electronic Surveillance Gear. Number of Torpedo Tubes: 6 Number of Reloads Carried: 6 Cruise Missile Capable May carry and launch the 12 Sub-Harpoon. Can carry and emplace 12 mines Unknown if swimmer capable.

TYPE 2400 "UPHOLDER" Vickers Shipbuilding & Engineering Ltd. United Kingdom 1986 Submerged Displacement: 2400 Lton Surface Displacement: 2188 Lton Standard Displacement: 1850 Lton Length: 230.6 ft Surfaced Draft: 17.7ft Diameter 25 ft Complement: 7 Officers, 13 CPO, 24 Enlisted. (44 Total) Prime Mover Type: Diesel Engine/Storage Battery Prime Mover Maximum Power: 3620 HP Main Propulsion Motor Power: 5360 HP Maximum Submerged Speed: 20 Kts Maximum Surface Speed: 12 Kts Maximum Snorkel Speed: 10 Kts Diving Depth: In excess of 200 meters. Overall Endurance Range at Six Kts: 7056 Nm Overall Endurance Range at Ten Kts: 5221 Nm Maximum Mission Duration: 49 Days Active Sonar Passive Sonar Array Sonar. Navigation Radar Electronic Surveillance Gear. Number of Torpedo Tubes: 6 Number of Reloads Carried: 12 Cruise Missile Capable. May carry and launch 12 Sub-Harpoon missiles. Can carry and emplace 24 mines Equipped with airlock for five combat swimmers

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Type 1700 Thys sen Shipyard Federal German Republic 1984 Submerged Displacement: 2350 Lton Surface Displacement: 2140 Lton Standard Displacement: 1760 Lton Length: 216.5 ft Surfaced Draft: 21.3 ft Diameter: 23 . 9 ft Complement: 30-35 Men. Prime Mover Type: Diesel Engine/Storage Battery Diesel Generator Maximum Power: 4400 KW Main Propulsion Motor Power: 8844 HP Maximum Submerged Speed: 25 Kts Maximum Surface Speed: 15 Kts Maximum Snorkel Speed: 15 Kts Diving Depth: In excess of 300 meters. Overall Endurance Range at Six Kts: 10080 Nm Overall Endurance Range at Ten Kts: 10736 Nm Maximum Mission Duration: 70 days Active Sonar Passive Sonar. Array Sonar Navigation Radar Electronic Surveillance Gear. Number of Torpedo Tubes: 6 Number of Reloads Carried: 16 Not Cruise Missile Capable. Can carry and emplace 32 mines Not Capable of Delivering Swimmers

TYPE 2000 Ingenieurkontor-Lubeck Federal German Republic 1983 Submerged Displacement: 3106 Lton Surface Displacement: 2820 Lton Standard Displacement: 2200 Lton Length: 210.6 ft Surfaced Draft: 21 ft Diameter: 24.4 ft Complement: 33 Men. Prime Mover Type: Diesel Engine/Storage Battery Diesel Generator Maximum Power: 3600 KW Main Propulsion Motor Power: 7500 HP Maximum Submerged Speed: 25 Kts Maximum Surface Speed: 13 Kts Maximum Snorkel Speed: 15 Kts Diving Depth Overall Endurance Range at Six Kts: 12651 Nm Overall Endurance Range at Ten Kts: 9293 Nm Maximum Mission Duration: Days Number of Torpedo Tubes: 8 Number of Reloads Carried: 18 Not Cruise Missile Capable. Can carry and emplace 24 mines. Not Capable of Delivering Swimmers

SAURO Fincantieri Shipyard Italy 1979 Submerged Displacement: 1660 Lton Surface Displacement: 1480 Lton Standard Displacement: 1280 Lton Length: 191 ft Surfaced Draft: 17 ft Diameter 22 . 4 ft Complement: 35 Men. Prime Mover Type: Diesel Engine/Storage Battery Diesel Generator Maximum Power: 2160 KW Main Propulsion Motor Power: 3216 HP Continuous 4200 HP (Burst) Maximum Submerged Speed: 19.3 Kts Maximum Surface Speed: 11 Kts Maximum Snorkel Speed: 11 Kts Diving Depth: In excess of 300 meters. Overall Endurance Range at Six Kts: 6480 Nm Overall Endurance Range at Ten Kts: 6891 Nm Maximum Mission Duration: 45 Days Active Sonar Passive Sonar Navigation Radar Electronic Surveillance Gear. VLF Radio Receiver Number of Torpedo Tubes: 6 Number of Reloads Carried: 6 Not Cruise Missile Capable. Can carry and emplace 12 mines Not Capable of Delivering Swimmers

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VASTERGOTLAND CLASS Kockums Shipyard Sweden 1986 Submerged Displacement: 1150 Lton Surface Displacement: 1070 Lton Standard Displacement: 990 Lton Length: 159.1 ft Surfaced Draft: 17 ft Diameter: 20.3 ft Complement: 21 Men. Prime Mover Type: Diesel Engine/Storage Battery Diesel Generator Maximum Power: 2160 KW
Prime Mover Maximum Power: 2680 HP Prime Mover Maximum Power: Main Propulsion Motor Power: 2537 HP Maximum Submerged Speed: 20 Kts Maximum Surface Speed: 11 Kts Maximum Snorkel Speed: 10 Kts (Estimate) Diving Depth: In excess of 300 meters. Overall Endurance Range at Six Kts: 3231 Nm Overall Endurance Range at Ten Kts: 1956 Nm Maximum Mission Duration: 30 Days Passive Sonar Electronic Surveillance Gear. Number of Torpedo Tubes: 6 Heavyweight tubes 3 Lightweight tubes Number of Reloads Carried: 6 Heavyweight Not cruise missile capable. Can carry and emplace 12 mines May also carry mines in external belt. Not Capable of Delivering Swimmers

MIDGET 100 "LWT 27-4" Sub Sea Oil Services of Micoperi Italy 1984 Submerged Displacement: 136 Lton Surface Displacement: 120 Lton (Estimate) Standard Displacement: 100 Lton Length: 88.9 ft (27.1 meters) Diameter: 10 . 3 ft Complement: 12 (+ 4 combat swimmers) Prime Mover Type: Closed-Cycle Diesel Small battery installed for stealth. Main Propulsion Motor Power: 420 HP Diesel/Generator Total Power: 120 HP Maximum Sustained Submerged Speed: 16 Kts Does Not Need to Snorkel. Diving Depth: In excess of 200 meters. Overall Endurance Range at Six Kts: 1345 Nm Overall Endurance Range at Ten Kts: 819 Nm Maximum Mission Duration: 14 Days Active/Passive Sonar Array Sonar Navigation Radar Number of Torpedo Tubes: ⁴ (Lightweight) Number of Reloads Carried: None (Muzzle Loaded) Not Cruise Missile Capable. Twin 7.62mm Deck guns and Single 20mm Deck Gun. Capable of Minelaying. Maximum Possible Number of Mines Carried: 4 Variant carries two mine delivery vehicles with 10 x 600Kg mines.

Chapter 4

DATA GATHERING AND SOURCES OF ERROR

There were two methods of data acquisition for this study. The first type was a search of the open literature for articles, advertisements, and manufacturer's brochures of interest. The second data source was that gained by calculation or estimation of values directly or indirectly from the data which could be gleaned from the open literature. Sensitive, proprietary, or classified information, or information gained through such channels, must be excluded from the thesis. Therefore, some of the data in this study is "secondgeneration" data, calculated or estimated from available published data. This introduces the possibility of error.

In the literature search It was found that some performance figures, such as maximum speed and number of torpedo tubes, were almost always available, usually in Jane's Fighting Ships, (17). Beyond these data elements, the sources were incomplete or, in some cases, contradictory of one another. One reason for some of the contradiction in the literature is probably due to the inevitable unintentional misquote of some corporate or government spokesperson. Literature sources are usually quite close to one another, so that the error introduced was usually not of great significance. For example, a submarine designed to accomodate sixty-two men could doubtlessly sustain a crew of sixty-seven (albeit for a shorter mission duration). One other possible source of literature data discrepancy is that the authors of the articles may not all have the same initial data with which to conduct their analyses. Articles in the literature, as opposed to manufacturer's brochures, are authored by a certain group of naval architects and naval ship analysts, each of which doubtlessly has his own set of empirical relations, correlation coefficients, and rules of thumb with which to conduct his analyses. Even if

all of these naval architects were given the same Initial data on a given submarine, there is bound to be a certain range of calculated and estimated secondary data values resulting from each of them. Where conflicting values of data exist in the literature, a notation is made, and the author's judgement is used to select the preferred value.

As a result of the problems with the data mentioned above, the accuracy of much of this thesis Is probably not grater than ten-percent. This error comes from some things as simple as being unable to measure submarine dimensions with extreme accuracy from an isometric and only partially-exposed cutaway view in a magazine, to the fact that errors will compound when used in calculations.

Care has been taken to limit discussion to obvious design features and differences between ships. The magnitude of the error is, therefore, deemed acceptable for the purpose of this analysis.

4.1 Reference Convention

In the data tables and figures included in this study, the sources of the information are

referenced in the following manner:

- Information from the literature is denoted by a number, in parentheses, which corresponds to the reference from which it was taken.
- Values calculated in the course of this study are unreferenced.
- Values or conditions which are estimated by the author, in the author's best judgement, are referenced by an "(e)" next to the entry.
- Values or conditions which are inapplicable to a calculation are designated by "N/A".

Chapter 5

METHODOLOGY

The method by which this thesis was carried out is straightforward, and consists of the following:

(1). Acquisition of available data from open-literature sources.

(2). Calculation or estimation of neccessary data which is not readily available or which could not be found.

(3). Parameterization of each of the selected submarines according to reasonable mathematical indices of description.

(4). Comparison of each of the submarines according to its Indices of description.

Finally, an attempt is made to "reverse engineer" the design process of each submarine in order to determine the nature of the top-level requirement.

Chapter 6

VOLUME ANALYSIS

6.1 Volume Within the Pressure Hull

The pressure hull volume distribution Is of prime importance in the design of a submarine. The pressure hull volume is determined partly by the size of the payload, but it must also contain and protect the propulsion plant, electronics, weapons, and crew. The tradeoff in volume allocation between each of these areas determines, to an extent, the performance capabilities of the submarine. The overall volume of the pressure hull, and the allocation of that volume, give considerable Insight into the design philosophy of each submarine.

The pressure hull of most submarines is composed of sections of cylinders, cones, and spheres. The pressure hull of the MIDGET 100 is one exception, since its pressure hull has the same teardrop shape as its external envelope, rather than cylinders or cones. The pressure hull total volume is readily calculated from the formulas of Appendix A, provided a detailed reference picture of the vessel exists. The reference pictures of the submarines in this study were of detail sufficient to allow calculation of pressure hull volume to within five percent. Reference pictures were not available for KILO and TYPE 2000.

More difficult is the calculation of the volumes of the individual functional areas within the pressure hull. The assignment of pressure hull volumes to each functional area, for the purpose of this study, is defined below. Where two or more functional areas share the same space, a judgement is made of the volume occupied by each function.

(1). Mobility. Includes the spaces housing all propulsion machinery, non-distributed

electric plant equipment, bow thrusters, steering gear, batteries, and internal fuel tanks. Also includes trim and auxiliary ballast tanks, and HP air flasks.

(2). Weapons. Includes the volume of the torpedo tubes, handling gear, ejection and launching equipment within the pressure hull, and the volume of the torpedo room, excluding any volume used for berthing.

(3). Command, Control, Communication, and Information, (C3I). Includes radio, sonar, radar, electronic warfare, periscopes, computers, navigation center, and control rooms. Also includes (an arbitrary) forty percent of the air-conditioning plant.

(4). Ship Support. Includes berthing, messing, galley, sanitary, and passageway space. Also includes all auxiliary machinery except that alloted to C3I.

The calculated volumes of each functional group within the pressure hull are shown in Table 7-1

6,2 Volume External to Pressure Hull

The ballast tank volume is calculated from the difference in the values of the submerged and surfaced displacements, which in general can be found In the literature.

The free flood volume is assumed to be five-percent of the submerged volume. The reference pictures of each submarine tend to confirm that the free flood volume is concentrated primarily in the fairwater, around the bow sonar array and torpedo tubes, and at the stern in the vicinity of the shaft.

The envelope volume of each submarine is estimated by summing the submerged volume and the free flood volume.

The remaining volume external to the pressure hull is found by subtracting the ballast tank volume and the pressure hull volume from the envelope volume. The other volume and the second company of the second

reference pictures. (Sheet one of two).

Table 7-1: Functional group volumes calculated from measurement of reference pictures. (Sheet two of two).

may be made up of structure, fuel tanks, high-pressure air flasks, conformal or trailed sonar arrays, periscopes and masts, snorkel, fittings, and special-purpose equipment.

Table 7-1 shows the calculated values of each submarine's main ballast tank and free flood volume, other submerged volume, and the envelope volume.

6.3 Discussion

Figure 7-1 graphically depicts the actual measured and calculated volumes of each of the functional groups, plus main ballast tank volume and other volume external to the pressure hull, for each of the submarines. The volumes for KILO and TYPE 2000 are estimated, since reference pictures were not available.

The first item of interest in Figure 7-1 is the variance in scale between the ten submarines in this study. The largest boat, KILO, is over twenty-three times the size of the MIDGET 100, with the other submarines falling between those extremes. Since Figure 7-1 displays each of the actual functional area volumes, It is possible to compare the sizes of each submarines' weapons area, or electronics/command suites by inspection.

The C3I functional group volume is largest in the KILO of all the submarines. Though the installed electronic equipment aboard KILO is not thought to be any greater than that installed in the other submarines, Soviet electronics are probably more voluminous than similar Western electronics because of the extensive use of vacuum tubes rather than solid-state technology. The C3I volumes for the WALRUS, RUBIS, BARBEL, TYPE 1700, TYPE 2000, and VASTERGOTLAND are nearly the same, even though the vessels vary in submerged displacement by a factor of two from the smallest to the largest. This demonstrates that the volume required to enclose sensor electronics and a command center aboard an oceangoing submarine is not a strong function of the vessel displacement.

Volumes of Functional Groups

Figure 7-1 shows the actual volumes of each of the functional groups, plus main ballast and other external volume, for each submarine. Some values are immedlatly noticed in Figure 7-1, such as the large mobility volumes for the RUBIS, TYPE 1700, and TYPE 2000, each of which has a large propulsion plant. In fact they have the three largest installed shaft horsepower plants of the submarines studied, and together have more horsepower than the remaining seven combined. The TYPE 1700 and the TYPE 2000 have larger batteries than the others, and the RUBIS has a nuclear reactor contributing to the volume. Also noticable are the small mobility volumes for the VASTERGOTLAND and the MIDGET 100, each of which have less-powerful propulsion plants, and smaller batteries than the others.

The BARBEL and KILO each have large non-ballast volumes external to the pressure hull. The KILO has this volume because of Its double-hull, the BARBEL because of the placement of large banana-shaped high-pressure air tanks between the pressure hull and the hydrodynamic envelope.

The ship support volume of each submarine would be considered a function of the complement, but each designer/builder has a different opinion of the habitability standards required by a submarine crew. Appendix K discusses some factors affecting crew endurance, not the least of which is volume-per-man within the pressure hull. The large differences in ship support volume among the submarines does not correlate to the variances in their complements. Chapter 10 discusses this in greater detail.

6.4 Volume Allocation

The allocation of volume in a submarine can indicate the functional groups which were most important to its designer. Figure 7-2 shows the volume distribution of each submarine. Note the high fraction of the volume dedicated to mobility in RUBIS, TYPE 1700, and TYPE 2000.

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KILO and BARBEL have large non-ballast volumes external to the pressure hull, because of their double-hull conmstruction. This volume is proportionately large in MIDGET 100 also, but it is due to the disproportionately large fairwater which cannot be made smaller or it would be unusable.

WALRUS and MIDGET 100 have very high ship support fractions. This was probably planned in the case of WALRUS, because of Its long mission duration. For MIDGET 100, it is unavoidable due to the scale effect of having the diameter of the vessel comparable to the human body height.

Volume Allocation Comparison

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Chapter 7

DISPLACEMENT AND WEIGHT ANALYSIS

7.1 Displacements

Each submarine may be described by three displacements:

1. Standard displacement is the displacement of the submarine on the surface when unloaded with fuel, ammunition, provisions, and crew.

2. Surface displacement Is the displacement of the submarine on the surface when loaded with fuel, ammunition, provisions, and crew. It Is equal to the standard displacement plus variable loads.

3. Submerged displacement is the displacement of the submarine when loaded, operating submerged. It is equal to the surface displacement plus main ballast tanks.

The literature has many of the values of these displacements, but in many cases the values differ slightly from one reference to the next. The literature usually provides no more than two of the three displacements, but knowledge of two can yield a reasonable estmate of the third. The variable loads and the ballast tank weight can be calculated from these known displacements, or if known, can be used to calculate the displacements.

Table 8-1 lists the displacement values for each of the submarines. Also shown are the weights of the variable loads and main ballast tanks.

It should be noted that diesel electric submarines must have large-capacity auxiliary ballast tanks to compensate for the lost weight of the bunker fuel. Alternatively, and preferably, is the installation of a fuel compensating system, and using the fuel tanks as

Table 8-1: Displacements and functional group weights.

(Sheet one of two).

SUBMARINE NAME

(Sheet two of two).

auxiliary ballast tanks. This necessitates the installation of a reliable and effective fuel oli filter and coalescer system as well. The literature was inconclusive about the presence of fuel-compensating systems, except for the TYPE 2400, which does, Reference (32).

The ballast tank weight is a big selling point and is also a matter of contention among submarine builders and designers. In the event of hull damage severe enough to cause flooding of the submarine, the bouyancy lost to the flooding water may be recovered, at least temporarily, by blowing down the main ballast tanks, hence their alias as "reserve bouyancy". Creating volume on a submarine is expensive, and even the extra ballast tank volume to accomodate a little extra reserve bouyancy will cost, in terms of speed, range, payload, crew habltabiliy, electronics, or construction cost. At the same time, it is acknowledged that each manufacturer wishes to present his product in the best light possible, and it is desirable to have large main ballast tanks, hence the source of the tradeoff.

7.2 Functional Group Weights

The weights of specific machinery and other equipment aboard the submarines could not be found in the literature. To estimate the functional group weights, empirical formulas were developed which related data parameters which are found in the literature to the elusive weight groups. Reference (16) was crucial in this regard. The details of this process are described in Appendix I. The result of the Appendix ^I calculations for functional group weights are listed in Table 8-1.

A rigorous analysis of the functional group weights given in Table 8-1 would be tonguein-cheek at best, since nearly all the weights are calculated from the same empirical formulas. Instead, a qualitative approach will be taken in relating the weight groups of

each submarine to the other submarines, to attempt to understand how the overall performance of each submarine is affected by the weights of its functional groups.

Using this approach, and with the aid of Figures 8-1 and 8-2, one may see that the boats with the higher top speeds and longer endurance ranges, such as RUBIS, TYPE 1700, and TYPE 2000, have the higher weights in the mobility functional area. Those with lower top speeds, such as BARBEL, KILO, and MIDGET 100 have proportionately smaller mobility weights.

The weights of the ship support, C3I, and weapons functional groups are small compared with the displacement of the corresponding submarine. This reflects the nature of the materials from which these groups are constructed. It also reflects the weight density of the spaces associated with those functional groups. It is reasonable to expect that dlesel engines, alternators, and lead-acid batteries make up a much larger proportion of the displacement of the submarine than habitability or electronics spaces.

The vessels rated at a shallower immersion depth, TYPE 2400 and MIDGET 100, have a smaller proportion of their displacement attributed to structural weight. An exception is BARBEL, rated at 120 meters immersion, whose structural weight is proportionatly as great as submarines rated at 300 meters immersion. One reason for its higher structural weight is that it, and KILO as well, is a double-hull design. One could conclude that the empirical formulas of Appendix ^I are inaccurate by a factor of three, that the formulas may be accurate but BARBEL is fabricated of a weaker material than the more modern submarines, or that the formulas are accurate, but BARBEL is underrated at only 120 meters immersion.

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Weights of Functional Groups

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Weight Allocation Comparison

Chapter 8

MILITARY PERFORMANCE

8.1 Propulsion and Mobility

The speed, range, and depth capabilities of a submarine are three of its prime attributes, and high values for each of these parameters is desirable, as they allow the submarine to act with greater flexibility, and hence, greater effectiveness. Specific values of these parameters are not available in the literature for the range of speeds of which these submarines are capable. This section focuses upon developing such a comprehensive database. Public-domain data germane to the mobility functional area is summarized in Table 9-1.

8.1.1 Required Shaft Horsepower

The study commences with calculations of the maximum sustained speed of each submarine. Extensive analysis of each submarine's propulsion characteristics are performed in this study. Computer models of the hydrodynamic envelope are established in Appendix B, and used to calculate the shaft horsepower required at various speeds at deep and snorkel depths in Appendices C and D. The resulting values of required shaft horsepower at deeply submerged depths are shown in Figure 9-1. while the ratio of the larger shaft horsepower required when operating at snorkel depth is depicted in Figure 9-2.

Figure 9-1 shows the characteristic cubic dependency of the power upon speed, for a body moving in a viscous medium without the generation of gravity waves.

Figure 9-2 indicates humps and other irregularities in the speed/power curve for operation at a depth where gravity waves are generated. The irregularities are caused

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Table 9-1: Propulsion plant and other mobility group parameters. (Sheet one of two).

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Table 9-1: Propulsion plant and other mobility group parameters.
(Sheet two of two).

Shoft Horsepower Required As a Funation of Speed 10 $\hat{\mathbf{B}}$ \bullet $\overline{\mathcal{F}}$ Ò $\overline{\mathbf{a}}$

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Shaft Horsepower Required 400 350 300 Shaft Horsepower, HP 250 200 150 100 30 \mathbf{a} ÿ. 5 Я SPEED, KH **T/FE 2000 YASTERGOTLAND** $\frac{1}{2}$ \vec{a} Figure 9-1: Required Shaft Horsepower at various speeds. (Sheet 2 of 2)

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Ratio of SHP Required Conceivaling Depth) (Deeply Submerged)

 $-51-$

as the generated waveform alternately hinders to a greater extent, then to a lesser extent, then to a still greater extent, the progress of the submarine through the water. This wave drag may be predicted as a function of Froude Number, and submergence ratio, according to the method of Appendix D.

8.1 .2 Fuel Endurance Range

The endurance range based upon bunker fuel load is calculated in Appendix E, and the results are displayed in Figure 9-3.

This is in general quite a lengthy range, since the submarines are loaded with enough fuel to travel goodly distances at higher speeds, and the speed for maximum fuel endurance Is usually In the vicinity of four or five knots. The fuel endurance range Is not the final word on endurance range for the submarine, considering all factors, but it is an excellent way to compare designs In the area of hull efficiency and amount of bunker fuel loaded. The fuel endurance range is calculated conservatively, using the value of SHP at snorkel depth, since much of the transit would be accomplished under this operating condition.

There Is an economy of scale concerning range. Since the SHP required is a function of the wetted surface area of the submarine, and since the amount of diesel fuel (or the number of battery cells, or the number of days of provisions), which can be carried is a function of the internal volume of the submarine, then the endurance range of a submarine will increase for increasing displacement, all else being equal.

To compensate for its low endurance range, the MIDGET 100 is equipped with a bowmounted towing cable, which would allow it to be deployed from a mothership when within a manageable range of the operating area.

Appendix E gives a relation for calculating the optimum speed for maximizing fuel endurance range.

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Snorkeling/Fuel Endurance Range

ENELEANCE PANGE NTI

Figure 9-3: Endurance range based upon fuel load, at snorkel depth.

8.1.3 Battery Endurance Range

The battery endurance range is calculated In much the same way that the fuel endurance range is calculated, except that the source of power is the storage battery Instead of diesel fuel. Battery endurance range is of great importance militarily, since the submarine may travel much more quietly on electric motor than on snorkeling diesel, and is also much less susceptible to radar and infrared detection than when snorkellng. The problem with calculating battery endurance range Is that the available energy to propel the submarine decreases as the rate of demand for it (the power level) is increased. This is due to the fact that at high power rates, as much as forty-percent of the stored chemical energy is dissipated as heat, and Is unavailable for useful work. Further discussion of this Is contained In Appendix F.

The calculation of the battery endurance range is conducted in Appendix G, and the resulting plot Is shown In Figure 9-4. An inspection of Figure 9-4 reveals the advantage of outfitting a submarine with a large battery, when submerged endurance counts. The tremendous battery range of the TYPE 1700 is due primarily to Its very large battery, and also to Its moderate hotel load and required shaft horsepower.

RUBIS has a low battery endurance range because it is equipped with a small battery. MIDGET 100 has a very small battery, and a relatively high hotel load as well. Both of these subs have primary propulsive power which is independent, to a degree, of the atmosphere, and so the need to avoid snorkellng Is not present. RUBIS and MIDGET 100 presumably have a battery just large enough allow them to operate stealthily for a short mission, perhaps just enough power to operate hotel services while remaining as a silent sentry or picket at bare steerageway.

The RUBIS has a nuclear reactor to generate steam for the turbo-alternators, which produce electric power for the main propulsion motor and hotel electricity.

-54-

Bottery Endurance Range

Battery Budurunos Range, Nm

Battery Endurance Range, Nm

Bottery Endurance Range

Bettery Endurance Range, Nm

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The MIDGET 100 has a closed-cycle diesel main propulsion engine clutched to the main shaft. There are also two smaller closed-cycle diesel alternator sets, which supply hotel electric, charge the battery, and can supply the emergency electric propulsion motor.

8.1 .4 Indiscretion Rate and Interval

Indiscretion rate, evaluated at a particular speed, is the fraction of time which a submarine must spend snorkellng, In order to charge Its battery. Indiscretion Interval, evaluated at a particular speed, Is the duration of time which elapses between Indiscretion periods.

The indiscretion rates of each of the submarines Is calculated in Appendix H, and is displayed for the range of snorkel-capable speeds in Figure 9-5. As expected, the submarines with large batteries and large alternator capacities have the lowest indiscretion rates for a given speed. As discussed in Appendix H, the alternator capacity is very important in keeping indiscretion rate low, because the recharging time is less. However, there is a limit to the recharging rate, since the same type of inefficiency exists in recharging the battery as in drawing power from it.

The indiscretion interval is also discussed and computed in Appendix H, and the results are shown in Figure 9-6. For very low speeds, the indiscretion interval becomes much greater, then tapers off to a maximum. The batteries of all of the submarines benefit from being operated at a lower power level, which frees up more available energy, and accentuates the already increasing indiscretion Interval.

8.1.5 Overall Endurance Range

For the purposes of this study, overall endurance range shall be defined as the range the submarine can achieve at constant speed, all factors considered. In other words, when the submarine exhausts one set of supplies, be it fuel, water, provisions, or

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Indiscretion Rate

Indiavation Rate

Indiavation Rate

Indiscretion Interval at 80% DoD

Indiscretion Interval at 80% DoD

Interval, Hrs

battery, It has completed its journey, and its range at that speed is defined as the length of that journey. However, battery range does not figure Into overall range, since the battery may be recharged as long as there is fuel remaining. So overall range will depend upon whether provisions or fuel are exhausted first at a given speed.

Figure 9-7 shows plots of provision and fuel range for speeds between two and ten knots. Provision range is directly proportional to the vessel speed, since the time rate of provision consumption is assumed constant. If a submarine were to be designed solely to maximize endurance range at a constant speed, then Ideally provisions and fuel would be exhausted simultaneously, at the speed of best fuel endurance range. Real diesel-electric submarines usually need extra fuel since they may need to conduct highspeed actions which require more fuel per mile. As such, Figure 9-7 reveals that nearly all of the conventionally-powered boats have provision ranges less than their fuel range at the optimum fuel range speed. This Indicates a deliberate loading of additional fuel to allow the overall range to be increased, and for it to occur at a greater speed.

For the nuclear-powered RUBIS, the overall range is normally taken as the provision range. The fuel range on emergency diesel, with 100% expenditure of bunker fuel, is shown in Figure 9-7 for comparison.

8.2 Weapons Systems

8.2.1 Weapons Launching Systems

The number, length, diameter, and launching method of a submarine's torpedo tubes are important military parameters. They determine the size and type of weapon which may be employed by the submarine. The number of torpedo tubes is related to the number of weapons which may be fired in a salvo, and perhaps also to the fire rate. Whether a submarine has the ability to track multiple targets and direct multiple weapons to those targets was not available in the open literature.

Provision vs Fuel Endurance: WALRUS

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Provision vs Fuel Endurance: BARBEL

Figure 9-7: Provision Endurance plotted with Fuel Endurance. (Sheet 2 of 5)

Provision vs Fuel Endurance: TYPE ¹ 700

Figure 9-7: Provision Endurance plotted with Fuel Endurance. (Sheet 3 of 5)

Provision vs Fuel Endurance: TYPE 2000

Provision vs Fuel Endurance: SAURO

 $-64-$

Provision vs Fuel Endurance: MIDGET 100

Figure 9-7: Provision Endurance plotted with Fuel Endurance. (Sheet 5 of 5)

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The launching system Is Important because It determines whether cruise missiles may be fired from the torpedo tubes. At this time, nothing was found In the open literature to state that swim-out encapsulated cruise missiles have been developed, so any submarine not using some type of positive ejection system to launch weapons cannot employ cruise missiles. It is the author's opinion however, that self-launching cruise missiles may be In development.

The standard tube diameter In the West Is 21 Inches (533mm) which will accomodate the heavyweight torpedoes and encapsulated cruise missiles made In the West. Lightweight torpedoes are 15 inches In diameter, and are carried on surface ships, aircraft and some smaller submarines, such as the MIDGET 100.

Table 9-2 lists weapons systems parameters. The term "Water Slug" is used to denote a positive ejection launch mechanism, although the details of the exact type were not found In the literature. Note that KILO, WALRUS, RUBIS, BARBEL, and TYPE 2400 employ positive ejection methods while the remaining five do not.

Evaluating the combat systems effectiveness of a submarine based upon the number of tubes and reload torpedoes possessed Is tricky. On one hand, the assumption could be made that all torpedo tubes have equal fire and reload rates, and that each submarine can compute and maintain fire control solutions for as many targets and torpedoes as it is equipped with torpedo tubes. In this scenario, advantage clearly belongs to the submarine with the most tubes. On the other hand, it could be assumed that a designer equips a given submarine with an abundance of tubes because of anticipated poor tube reliability, or poor weapon kill probability. In actuality, there is not enough data in the open literature to make a detailed evaluation of the combat systems effectiveness. For the purposes of this study, it shall be assumed that all torpedo tubes have equal fire and reload rates, and that each submarine can compute and maintain fire control solutions for half as many targets and torpedoes as it is equipped with torpedo tubes.

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Table 9-2: Weapons Systems parameters. (Sheet one of two).

Table 9-2: Weapons Systems parameters. (Sheet two of two).

8.2.2 Torpedoes

Torpedoes have been the weapon of choice for submarine use. Although much slower than guns or missiles, the torpedo Is employed very effectively by submarines because of the submarine's stealth. Because the torpedo warhead explodes beneath the surface of the water, It Is more damaging to the hull structure of a surface vessel than an equally-sized missile warhead.

There are several heavyweight torpedoes manufactured in the West, all of which are compatible with the free-world submarines of this study. All are effective weapons within their firing envelopes. The size of the envelope Is the important criteria, and is governed by the speed, range, and depth capabilities, by the onboard sensing and logic systems, and by the presence or absence of a datallnk to the parent submarine. Superior speed is needed to overtake the target, the rule of thumb being twice the anticipated target speed, Reference (14). Sufficient range and depth capabilities are also necessary to complete the pursuit, and the sonar and logic circuits aboard the torpedo are important for terminal guidance. The datalink (such as wire-guidance) with the mother sub is Important for mid-course guidance.

Perhaps the most capable heavyweight torpedo in the West is the MK-48 ADCAP, Reference (14), primarily because of its speed, range, and depth capability, the exact values of which are classified. However, the torpedo parameters listed in Table 9-2 are suitable for comparison.

8.2.3 Cruise Missiles

The capability of a submarine to carry cruise missiles gives the it the medium-range (50) nautical mile) stand-off attack mode against surface targets which was previously the province of only surface ships and attack aircraft. Only those submarines equipped with

a positive ejection system (and the requisite fire control electronics) may currently employ cruise missiles. Fire control solutions would most likely be gained by passive array sonar, which may have ranges up to 45 kilometers or more, the exact values being classified.

The ability of a submarine to carry cruise missiles is clearly an advantage. For those ships able to employ them, encapsulated cruise missiles may be loaded in lieu of heavyweight torpedoes on a one-for-one basis.

8.2.4 Mine Laying

A submarine, particularly a diesel-electrlc submarine, is Ideally equipped because of its stealth to conduct covert mining operations. Many offensive mining scenarios call for covert placement of the mines. In general, two small mines may be loaded in lieu of one heavyweight torpedo. Table 9-2 lists mine laying parameters. Submarines not equipped with positive ejection tubes must employ self-propelled mines. Kockums Shipyard, manufacturer of the VASTERGOTLAND, has developed an external mine-belt conveyance system, the advantage of which Is that a full load of mines may be carried without affecting the torpedo load.

8.2.5 Other Weapons Systems

The use of combat swimmers for reconnaisance and other activities is believed to be a primary mission area of some diesel-electrlc submarines. It is known that the TYPE 2400, TYPE 1700, and TYPE 2000 submarines are equipped with swimmer lockout chambers, detailed in Table 9-2. KILO is judged by the author to have this capability as well. A variant of the MIDGET 100 is constructed with a four-person swimmer lockout chamber instead of the four lightweight torpedo tubes. The MIDGET 100 may also tow swimmer delivery vehicles to the operating area, but this must reduce its endurance range.
KILO may be equipped with anti-air missiles mounted in the fairwater. These may have been installed in response to the high state of aircraft-based anti-submarine warfare (ASW) capabilities among NATO forces.

MIDGET ¹⁰⁰ Is equipped with ^a 20mm and ^a ⁴⁰ mm deck gun. This further Indicates that the primary mission area of this vessel is special operations.

8.3 Command, Control, Communication, and Information

8.3.1 Sonar

The primary sensor of the modern submarine Is passive sonar. The structure of the sonar may be conformal hull-mounted array, trailed linear array, or spherical, cylindrical bow-mounted array, or a composite sensing system made up of several of these arrays. The advantage of passive sonar is that the submarine can remain undetected while observing its environment. A submarine with a passive sonar is able to determine the bearing of a sound source. When equipped with a sensitive conformal or trailed linear array, the submarine can get range information as well from the time delay in reception of the incident sound waves. With range and bearing information, the computation of fire control solutions is possible.

Active sonar is usually used tactically to confirm the computed target range by a single active "ping" immediatly prior to weapon launch. It is typically at this point that opposing sonar-equipped vessels, both surface ships and other submarines, first become aware of the sub's presence. For reasons of stealth, active sonar is not often used by a submarine on patrol.

The quintessential parameter of sonar performance is sensitivity. Sensitivity and discrimination, to be able to detect a potential target and separate its sound from the ambient noise in order to verify its existence and possibly its identity. The detection

range is dependent upon the sensitivity of the sonar, and detectability is the "name of the game" when stealth and first-strike capability are of paramount Importance. Unfortunately, because of its importance, detection capabilities of sonar equipment is not available In the literature. The literature does have Information on the manufacturers, and in some cases the particular model, of the various sonars installed in the subject submarines, as may be seen in Table 9-3. All of the submarines in this study are equipped with both active and passive sonar, and most have a towed linear or flank conformal array sonar as well.

8.3.2 Periscopes

The traditional submarine sensor is the periscope. Modern periscopes are equipped with telescopes, rangefinders. infrared adapters, and electro-optical and photographic adapters. All of the submarines in this study are equipped with two periscopes, search and attack. It is the author's opinion that every submarine is fitted with the above mentioned periscope augmentation gear, although the literature did not confirm this. The names of the periscope manufacturers are listed with their host submarines in Table 9-3.

8.3.3 Radar

Radar is used by submannes primarily for navigation during sea detail and other navigational situations, but could also be used in a combat role. Of particular interest is the Decca radar mounted on WALRUS. The Decca is popular on a number of commercial vessels, so the employment of it by WALRUS in a crowded shipping lane (and under limited visibility conditions) would not raise alarm. Available radar information is listed in Table 9-3.

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(Sheet one of two).

(Sheet two of two).

8.3.4 Electronic Surveillance Measures

Electronic surveillance (ESM) is a more valuable combat tool than radar because the submarine does not reveal itself when using ESM. ESM is the passive sonar of the electronic information realm, and may be used to assist in the identification of a contact. The manufacturers of the submarine ESM gear are listed in Table 9-3.

8.3.5 External Communications

Table 9-3 lists the available information on communications systems.

The necessity of a submarine to be able to communicate with friendly operating forces is essential. Because of data links with aircraft and other surface units, surface ships generally have knowledge of a much greater area than submarines. The submarine must communicate with friendly forces in order to cooperate most effectively with friendly forces. The methods of communication available are various radio frequency bands, and underwater telephone. High frequency (HF) radio is generally used to communicate with shore stations by teletypewriter. Very high frequency (VHF) radio is used to communicate at distances just beyond the horizon. Ultra high frequency (UHF) is useful for line-of-sight communication, and as such has a shorter range but will allow the submarine to remain undetected to surface units beyond the horizon. Very low frequency (VLF) radio receivers were designed for use aboard strategic ballistic-missile submarines, but have been installed on some patrol submarines as well.

Underwater telephone may be used for two-way communication while the submarine is submerged, which is not possible with radio. Underwater telephone uses encoded sound pulses sent through the main active sonar array or through a separate dedicated transducer. It has limited range.

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8.3.6 Automated Controls

Automated control systems have revolutionized the design of the submarine. By automating the propulsion and auxiliary plants, and integrating and computerizing the sensors and command centers, the required complement has been halved. A smaller complement frees up space and weight for other areas such as provisions, fuel, battery. or weapons reloads. The volume and weight cost of automating is less than the volume and weight saved due to the crew reduction It allows. The other costs of automating are a sharp increase in system complexity, with a multiplication of the probability of system failure, a decrease in systems availability, and an increase in preventative and corrective malntainance actions. Additionally, during casualty situations, when manual backup may become necessary, it is an advantage to have a high man-to-equipment ratio.

All of the submarines except BARBEL and possibly KILO use advanced automation technology and hardware. TYPE 1700, TYPE 2000, VASTERGOTLAND. and MIDGET 100 use it the most extensively, and with good results. Manufacturers of automation and control hardware are listed in Table 9-3.

8.4 Ship Support

The ship support functional group is concerned with the amount of space and weight needed to support the mobility, weapons, and C3I groups. It is made up of the habitability spaces, passageways, and provisions, and is directly proportional to the number of crewmembers. Depending upon one's viewpoint, the crew may or may not be included in the ship support functional group, but for this study, the crew itself is considered to be an integral and operational part of the other three functional groups. So ship support systems do not contribute directly to the performance of the submarine's mission, but are nonetheless essential to the proper functioning of the submarine as a whole.

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Appendix K lists the most important physical and psychological factors affecting crew endurance. Submarine crews are an elite and dedicated group who are accustomed to the close quarters of life aboard a submarine, but each man has his own tolerance level for spartan conditions. Table 9-4

lists ship support and habitability parameters, many of which are only estimated from the reference pictures of the submarines. The most Important parameter is the volume-perman, given in cubic feet per man. WALRUS is by far the most voluminously-appointed vessel with 555 cubic feet/man, and MIDGET 100 is by far the least with one fifth of that value. The other boats are furnished with between approximately one-half to two-thirds the volume per man as WALRUS. An examination of the days of provision loadout for each vessel hints at the reasons for the great disparity in specific volumes - the mission duration of WALRUS is about five times that of MIDGET 100. Another explanation is that different cultures have different levels of personal privacy needs, and the respective shipbuilders have reflected that in thier designs.

Of particular note is that on BARBEL and SAURO, and MIDGET 100 as well, when loaded with combat swimmers, some berthing Is located on the torpedo racks, whereas the other submarines have all berthing located in designated berthing compartments. Also, MIDGET 100 does not have a mess room, although there is a space designated as the galley/scullery.

8.5 Acoustic Countermeasures

Stealth and undetectability are essential for effective combat actions particularly for diesel-electric submarines, which have a limited submerged range, and must be indiscrete while recharging batteries. Diesel-electric boats are considered quieter than nuclear boats when operating on battery, and there has been an intense effort by all

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Table 9-4: Ship support systems parameters.

submarine manufacturers to reduce the sound emanation level as low as possible, with the desired goal of being only as noisy as the ambient ocean. This is actually a variable goal, since high sea-states are much noisier than low sea-states, and the silencing goal has certainly been met on a number of submarines for higher sea-states.

Some of the more prevalent and unclassified methods of submarine silencing are shown in Table 9-5. The use of propeller silencing, consisting of refinements in the hydrodynamic shape of the propeller, resilient mounts for machinery, and a low speed main shaft is common to all the boats. Only KILO and TYPE 2400 employ anechoic hull covering, and all boats except BARBEL have gearless main shaft drives. These parameters still only give qualitative indications of the silence of each submarine in operation, since the effectiveness of the silencing methods is likely to vary among the ships.

8.6 Survivability and Damage Control

A submarine is inherently a warship. Because of its limited volume and relatively high cost per ton, there are few commercial ventures which would choose a submarine over a surface displacement vessel. Being a warship, it must be expected that it shall be required to venture into harm's way. The importance of stealth, silencing, first detection, and first-strike capabilities have been discussed. The survivability shall now be discussed.

In the event that a submarine is hit. the strength and toughness of its hull, and its reserve bouyancy (ballast tanks) are the material-world determinants of its future. Information on hull strengths and geometry of construction are not available in the literature, but an estimate may be gleaned from knowledge of the immersion depth. Appendix L lists several factors impotant to submarine vulnerability and survivability.

Table 9-5: Countermeasure outfit.

Table 9-6: Compartment measurements germane to damaged survivability.

Table 9-6 details calculations of reserve bouyancy limits in a scenario involving flooding to the single largest compartment of the submarine. In the event of flooding, the main ballast tanks could be blown down, enabling the submarine to aviod sinking. The "MBT/COMPT" ratio is the fraction of the largest compartment volume which could be flooded before 90% of the reserve bouyancy would be expended in attempting to keep the submarine afloat. The favorite is KILO, which because of its greater degree of compartmentation and large ballast tanks, would be able to avoid sinking If damaged in only one compartment. This is not to say that KILO would remain mission-capable, or that subsequent shots would cause more extensive and irrecoverable damage, but all the other submarines are clearly one-shot platforms.

The KILO is the most capable submarine of those in this study to withstand severe damage. Its double-hull construction can withstand explosive warheads better than a single-hulled submarine of equal test depth rating, because the outer hull prevents the warhead from detonating as close to the pressure hull as it would have with a single-hull design. Reference (27) provides some insight to additional possible reasons for the use of double-hull designs by the U.S.S.R.:

The pitiful peacetime safety record of Soviet submarines suggests serious design flaws, inattention to safety, lack of crew/shipyard maintainance of onboard equipment, and poor seamanship. Given the propensity of Soviet submarines to collide with submerged and surface objects, it was probably a wise decision to continue building more survivable doublehull submarines.

The bottom line of all this is that the capacity to withstand severe damage is certainly an asset, but the ability to avoid any damage at all, due to superior stealth, sensor range. weapon effectiveness, speed, and crew training state is a much greater asset.

8.7 Escape and Rescue

Table 9-7 lists the submarine escape and rescue facilities. Note that all of the submarines have been provided with at least one escape scuttle. The TYPE 1700 and TYPE 2000 are also equipped with escape pods of large enough capacity to hold the entire crew and provide them with four days sustainance.

Table 9-7: Escape and Rescue capabilities.

Chapter 9

COMPARATIVE NAVAL ARCHITECHTURE

9.1 Specific Volumes

Table 10-1 lists the values of the weights of the functional groups divided by the volumes of the functional groups.

9.1.1 Mobility

For the mobility functional group, the weight/volume ratio seems to be related to the endurance range, with higher ratios occuring in submarines with shorter ranges. This is because bunker fuel is less dense than the propulsion machinery and battery, and the large fuel loads required for long ranges drop the average weight of the entire functional group.

9.1.2 Weapons Systems

The weight/volume ratios for the weapons systems are a function of how densely the space is packed with reload torpedoes. For MIDGET 100. the exceptionally high value is due to the exceptionally small portion of the pressure hull devoted to weapons, since the torpedoes are muzzle-loaded and there is no positive launching gear to take up space either. The ratios for BARBEL and SAURO are comparable to the ratios of the other subs because the volume over the torpedo racks used for berthing was charged to the ship support group.

-84-

Samparison of performance.

Table 10-1: Comparison of performance and design parameters. (Sheet two of two).

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9.1.3 Command, Control, Communications, and Information

All the ratios are about the same except for KILO, which is presumably lower due to the extra volume taken up by the vacuum-tubes and the additional HVAC ducting needed to maintain the temperature.

9.1.4 Ship Support

All of the values are about the same except for WALRUS, which has an exceptionally large ship support volume. WALRUS apparently has been designed with extra volume to allow greater crew comfort during the exceptionally long (70+ days) missions of which this vessel Is capable. The empirical formula for ship support weight developed In Appendix ^I is a stronger function of complement than of vessel size.

9.2 Mobility Weight/Installed Power

An economy of scale is evident In the ratio of mobility weight to installed shaft horsepower, with lower ratios occuring for higher SHPI, on vessels such as RUBIS, TYPE 1700, and TYPE 2000. The low value for SAURO is due to its densely-packed engine room, a design trait for which Fincantieri is well known.

9.3 Overall Endurance Ranges at Six and Ten Knots

Ten knots is a reasonable speed for a diesel-electrlc submarine In transit to or from the operating area. Six knots is a reasonable speed for patrolling a choke-point operating area. Overall endurance range at ten knots is the fuel endurance range for the subject submarines, and the overall endurance range at six knots is in general determined by the provision endurance. Figure 10-1 shows a comparison of these ranges. The long range of the nuclear-powered RUBIS at ten knots is of note, as is the short range of

MIDGET 100 at both speeds. MIDGET 100 may be towed to the operating area, but is probably best at missions of shorter duration, as Is apparently also the case for TYPE 2400 and VASTERGOTLAND. VASTERGOTLAND was most likely designed to patrol the coast of Sweden looking for KILO and other Soviet submarines, which have an affinity for the fiords. The other boats were probably designed for, and are capable of, long range solo transits to and from a remote location, and with enough fuel and provisions remaining to spend a healthy amount of time at the operating area.

9.4 Battery Endurance Range

Figure 10-2 shows a comparison of the battery ranges of each submarine at speeds of six, ten, eighteen, and twenty-five knots. TYPE 1700 and TYPE 2000 are the best in this area. They are also the only submarines to have appreciable battery range at twentyfive knots, although RUBIS would be expected to go nuclear Its limited battery energy was exhausted and It still needed to make that speed. Battery endurance range Is of great combat significance, because the submarine is much less detectable when on battery than when snorkeling.

9.5 Indiscretion Rate and Interval

Once at the opearating area, the diesel-electric submarine will need to recharge its batteries from time to time. If operating in a war zone, low indiscretion rate and long indiscretion interval may be crucial to the submarine's combat effectiveness. With a long indiscretion interval, the submarine skipper has the flexibility to choose the best time to recharge batteries. Notification of that time could even come from a shore station or other friendly units, based upon satellite information or other sensor data.

Figures 10-3 and 10-4 show a comparison of the indiscretion rates and intervals at

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Figure 10-1: Comparison of Overall Endurance Ranges at Six and Ten Knots.

Comparison of Battery Endurance Range at Six Figure $10-2$: and Ten Knots.

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speeds of six and ten knots. Of immediate note is the fact that RUBIS and MIDGET 100 have zero indiscretion rates and indefinitly long Indiscretion intervals, since neither of them needs to snorkel in order to recharge batteries. TYPE ¹ 700 and TYPE 2000 have the next best ratings, due to their very large batteries and large dlesel alternator sets.

9.6 Escape Capability

Figure 10-5

shows each submarine's rating on an arbitrary parameter designed to evaluate escape capability once detected. The parameter places a premium upon top speed due to its importance in outrunning a torpedo. The parameter is the product of the immersion depth and the square of the top submerged speed. Immersion depth is important because internal combustion propelled torpedoes have decreased range with increased depth. The high scorers are RUBIS, TYPE 1700, and TYPE 2000. The low ratings for BARBEL, TYPE 2400, and MIDGET 100 are a result of their poor immersion depth and lower top speed.

9.7 Weapons Delivery Capabilities and Platform Efficiencies

9.7.1 Torpedoes

The ability to deliver ordnance on target is essential for combat effectiveness. For a measure of overall torpedo delivery effectiveness, an arbitrary parameter is the product of battery endurance range at ten knots, number of torpedo tubes, and number of torpedoes carried. This product is then divided by 1000 for scaling. An arbitrary parameter for the platform efficiency of torpedo delivery would be the same product divided by submerged displacement. Figure 10-6 compares the calculated values of these parameters.

Figure 10-3: Comparison of Indiscretion Rates at Six and Ten Knots.

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sunoH *|DAJetu| Figure 10-4: Comparison of Indiscretion Intervals at Six and Ten Knots.

Submarine

Figure 10-5: Comparison of Escape Capability Parameter.

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Torpedoes and Figure 10-6: Weapons Delivery Parameter: Battery Range at Ten Knots.

 $-95-$

The high values for TYPE 1700 and TYPE 2000 are again a result of the outstanding battery endurance range of those vessels, combined with their large torpedo loadout. The low score for RUBIS Is not truly repesentatlve of that submarine's combat effectiveness, since reactor range at ten knots is much greater, but is included for comparison. The platform efficiencies of KILO, WALRUS, BARBEL, TYPE 2400, SAURO, and VASTERGOTLAND are all inthe same range, and the platform efficiency of MIDGET 100 Is not much below that.

9.7.2 Cruise Missiles

For a measure of overall cruise missile delivery effectiveness, an arbitrarty parameter is the product of overall endurance range at six knots and the number of cruise missiles carried. This product is then divided by 1000 for scaling. An arbitrary parameter for the platform efficiency of cruise missile delivery would be the same product divided by submerged displacement. Figure 10-7 compares the calculated values of these parameters. The overall endurance range at six knots Is used because the stand-off launch mode of the cruise missile does not require lengthy periods on battery as torpedo attacks do, and instead, the emphasis should be placed upon endurance on station.

WALRUS stands out as the leader in this area because of its high weapons loadout capacity and excellent slow-speed endurance range due to its high provision loadout. TYPE 1700, TYPE 2000, SAURO, VASTERGOTLAND, and MIDGET 100 fail to score in this area due to the inability of their torpedo tubes to launch cruise missiles. KILO, RUBIS, BARBEL, and TYPE 2400 are all approximately equal in both overall capability and platform efficiency.

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Weapons Delivery Parameter: Cruise Missiles Figure 10-7: and Fuel Endurance Range at Six Knots.

9.7.3 Mines

For a measure of overall mine delivery effectiveness, an arbltrarty parameter Is the product of overall endurance range at six knots and the number of mines carried. This product is then divided by 1000 for scaling. An arbitrary parameter for the platform efficiency of mine delivery would be the same product divided by submerged displacement. Figure 10-8 compares the calculated values of these parameters. The overall endurance range at six knots is used to represent the degree of flexibility the submarine would have in remaining on station to pick the best time to place the mines. Actual placement of the mines would usually be conducted on battery.

WALRUS again leads the pack in this area due to its high loadout of provisions for long endurance and mines for combat effectiveness. TYPE 1700 and TYPE 2000 achieve good scores as well, and have about the same platform efficiency as WALRUS. MIDGET 100 is next for platform efficiency, and might be even higher if the analysis were to Include the contributions of the auxiliary mine pods which may be loaded externally. The remaining boats are reasonably effective minelaying vehicles.

9.8 Conclusions

This study places a great deal of importance on speed and endurance range. The author states that these are essential attributes for combat effectiveness, and are suitable indices of comparison in lieu of more subtle, or unavailable, parameters such as sensor ranges, sound emanation profiles, equipment failure rates, or casualty control needs.

The main conclusion to be drawn is that automation of systems will allow a reduction of crew size, which then permits a larger battery and greater provision, fuel, and weapons loadouts. This will lead to greater combat effectiveness due to increased range, attack flexibility, speed, and weapons delivery potential.

Weapons Delivery Parameter:
Fuel Endurance at Six Knots. Figure $10-8$: Mines and

-100-

Chapter 10

AREAS FOR FURTHER STUDY

The following areas would increase the depth of this study and enable a more comprehensive comparison between the subject submarines.

10.1 Maneuvering Characteristics

The maneuvering capabilities of each submarine could be modeled and the results used to develop tactical engagement and attack parameters and techniques.

10.2 Weight Distribution

The determination of weights and weight distributions of each system and functional group needs to be accomplished. The weights calculated in this study are all based upon the same empirical formulas, so the notion of comparing one submarine's weight groupings to another's is impossible. It may be impossible to publish a study of this detail, since the actual values of submarine weight groups are often proprietary data, if not classified.

10.3 Specific Fuel Consumption Increase at Snorkel

Many state-of-the-art diesel engines have SFC's in the low 0.30's (Ib/HP-Hr) when run on the test bed. The additional fuel consumption increase due to the flow restrictions in the intake and exhaust ducts could be accomplished. The result would improve the accuracy of the calculated endurance range. Additionally, a relation should be possible between the increase in SFC due to the length of the intake and exhaust trunks, and the

additional horsepower needed to operate close to the surface, so that an optimum sail height for anticipated snorkeling speed could be found.

10.4 Hull Strength Estimation

A model of each submarine's pressure hull could be developed, and analyzed for actual crush-depth estimate. Several of the submarines have the same published minimum (normal) operating depth, how much safety factor (or military discretion) has been employed In each? This model also probably could not be accomplished with much accuracy with only open-literature sources.

10.5 Weapons and Sensors Capabilities

The focus of this study was comparative design of the marine engineering aspects of the submarines. The weapons systems capabilities and the sensor ranges of each submarine could be researched or estimated, and a more thorough evaluation of the combat effectiveness of each submarine could be accomplished.

Chapter 11

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APPENDIX A: GEOMETRY CALCULATIONS FOR ESTIMATING SUBMARINE COMPARTMENT VOLUMES

In order to determine the internal volumes of the submarines to the greatest degree possible, generic submarine hull components were modeled on a computer spreadsheet.

Since the pressure hull is composed primarily of cylynder sections, truncated cones, and hemispheres, the computer modeling is as easy as summing the volumes of the component sections. The formulas used to model these compont sections are as shown below. The formulas were input to a computer spreadsheet, Reference (21), to facilitate computations. This worksheet, "SECTOR3.WK1" is included on the following page.

CYLYNDER: Vol = $2*PI*R*L$ EQN A-1 $L \star R^2$ (R-z) $L \star R \star (R - z)$ CYLYNDER SECTION: $Vol = --- - - + (ACOS ---) - (---- - -)$, EQN A-2 2 R 2

TRUNCATED CONE: Vol = $(PI/3) * L * [R^2 + R^*r + r^2],$ EON A-3 TRUNCATED CONE SECTION:

Vol
$$
\sim
$$
 = $\begin{array}{ccc} n & iL * R^2 & (R - i z) & L * R * (R - i z) \\ -- -- -- - * (ACOS -- -- --) - (-- -- -- -- --) , & EQN A-4 \\ i = 1 & 2 & R & 2 \end{array}$

 $A-1$

"SECT0R3.WK1" (MAY BE USED FOR SUBMARINE VOLUME CALCULATIONS) (ASSUMES THREE-DECK ARRANGEMENT SCHEME) CALCULATES COMPARTMENT VOLUMES. SECTOR AREA ⁼ R~2CAC0S((R-H) /R)- (R-H) (SORT (R'^2-(R-H) ^A2) 13.8 (input) CIRCLE RADIUS: (R) CIRCLE RADIUS: (R) 13.8 (input) TOPSECTOR HEIGHT: 20.48 (input) BOTMSECTOR HEIGHT: 2.936 (input) -6.68 (R-H): 10.864 (R-H): TOPSECTOR AREA: 476.0334 BOTMSECTOR AREA: 34.09241 MID SECTOR AREA: 88.159004426 TOPCOMPT LENGTH: 23.487 (input) BOTMCOMPT LENGTH: 21.31 (ⁱ nput TOPCOMPT VOLUME: 111180.50 BOTMCOMPT VOLUME: 726.5093 18.47409 MIDCOMPT LENGTH: 46 (input) CYLYNDER LENGTH: 44 MIDCOMPT VOLUME: 4055.314 CYLYNDER VOLUME: 26324.53 <mark>==================</mark>CYLYNDER DISPL: 752.1295 === VOLUME OF A TRUNCATED RIGHT-ANGLED CONE: DIMENSIONS: SMALL END RADIUS: ^r = 10.996 LARGE END RADIUS: R = 13.772 LENGTH: L = 7.79 $V = 3768.981$ (PI*L/3)*(R^2+r^2+rR) ------------------------------VOLUME OF A CONE: (LARGE END AND LENGTH ONLY) VOLUME: V = 1547.248

APPENDIX B: NUMERICAL APPROXIMATION OF SUBMARINE WETTED SURFACE AREA

The calculation of the wetted surface area of the bare hull, sail, and appendages is crucial to being able to solve for the effective horsepower required at various speeds. An accurate calculation of the wetted surface is difficult because of the generally non-isometric drawings of the submarines in the literature. Accuracy may be improved by using numerical modeling methods. Such a method is used in the interpolating program "SPLIN500 .WKl" on the following pages which uses cubic spline matrices to approximate the surface of a body of revolution, then numerically evaluates the surface area from the interpolating polynomials. The cubic spline technique is from Reference (28), and the program is implemented on Lotus 1-2-3 (Rel 2), of Reference (21). The inputs are the measured radii at evenly-spaced stations, and the overall length of the submarine. the outputs are a set of station slopes, interpolating third-order polynomials, and the surface area of each section and the entire submarine as well. Table B-l method should be used with caution, because although the cubic-spline method guarantees that the cubic interpolating polynomials, and their first and second derivatives, will be continuous over the body, it cannot guarantee that the surface so generated will accurately represent the surface from which the radii were extracted.

Another numerical modeling scheme is presented as "HERMFAST.WKl" This model uses Hermite polynomials, also from Reference (28), to determine the cubic interpolating functions. The inputs to this model are both the radii and slopes at sequential stations, and with the added information of the slopes, this model is able to calculate the surface area to as accuarate as the input data from the reference drawing will allow. Stations need not be evenly spaced either. As few as two station data inputs are possible for the model to work, but accuracy is improved with more stations, with a maximum of eleven input stations currently programmed. The outputs are interpolating polynomials, the section areas, and the wetted surface of the entire bare hull. An run of HERMFAST is presented here, with the input values for TYPE 2400 input as an example

The wetted surface areas calculated by these models are accuate for bodies of revolution, so corrections must be made to account for the deck, skeg, tumblehome, and any other protrusions.

The surface areas of the sail and the appendages are measured from the available photographs or diagrams in the literature as best as possible

"SPLIN500. WK1" USED TO CALCULATE SURFACE AREA OF A BODY OF REVOLUTIO

The inputs are the station radii. The outputs are cubic interpolati polynomials which form a curve which has continuous first and second derivatives. CAUTION: The resultant body may not adequately model ^t geometry of the actual body, since station slopes are not specified!

(USES CUBIC SPLINE MATRICES, FROM STRANG, pp 177-180.)

PROCEDURE:

1) INPUT ACTUAL LENGTH AND RADII AT THE ELEVEN (EQUALLY-SPACED) STA 2) HIT CF9] TO CALCULATE.

3) MULTIPLY THE MATRIX AT K32 BY THE MATRIX AT W32, RESULT TO Y32. (4) HIT CF9I AGAIN TO GET SLOPES, INTERPOLATING COEFFICIENTS, AND AR

 $- B2 -$

 $-B3-$ OUTPUT: COEFF. OF (ACTUAL) INTERPOLATING POLYNOMIALS, U(X). A B C D
1F-1R 0 -4 1F-1P $0 - 4.1E - 18$ $4.1E-18$ 0 to 1 : LAMDA: - O -1 to 2: -9.4E-17 3.7E-16 -4.5E-16 1.7E-16 $\overline{1}$ 2 to 3: 1.6E-15 1.6E-15 -3.9E-15 3.1E-15 -9 27 3 to 4 : -27 4 to 5 : -3 39 -165 229 - 3 - -51 285 $5 to 6:$ -521 -1 21 343 6 to $7:$ -147 7 to 8: 1.8E-16 -4.0E-15 3.0E-14 -7.6E-14 8 to 9: 4.2E-17 -1.1E-15 9.8E-15 -2.9E-14 $9 - 10: -9.3E - 18$ 2.8E-16 -2.8E-15 9.2E-15 $(ACTUAL = - > "XY")$ **Example XA2** X and X $\mathbf{1}$ **RELATIONS:** LET lamda = 1 THEN: $X = 1 * x$ AND: $U(X) = 1*U(x)$ THEREFORE $A = a/1^2$ $B = b/l$ $C = c$ $D = d*1$ $\langle \hat{\mathbf{Q}} \rangle$ \bullet O \overline{Q} \mathbb{Z} \sim 0 \sim \bullet O \bullet O \bullet \circ \bigcirc \sim 1 \sim 0 \sim $\frac{1}{4}$ \bullet 0 -4 -0 \overline{O} -0 \circ O \bullet O \bullet \bullet O \circ $\mathbf{1}$ \circ $\vert 1 \vert$ $\frac{1}{2}$ \bullet O \bullet $-$ O \bullet O \bullet O \circ \circ \ddot{O} $\mathbf{1}$ $\begin{pmatrix} 1 \\ 0 \end{pmatrix}$ \circ O \circ \circ \circ $\ddot{\text{o}}$ -0 $-4 \bullet$ 0 \bullet 0 \bullet \bullet O $\begin{bmatrix} 1 \\ 0 \end{bmatrix}$ -4 $\overline{1}$ \bullet O \bullet \bullet O \bullet \bullet O $-$ O \circ \circ \ddot{O} $\begin{array}{c}\n1 \\
1 \\
0 \\
0\n\end{array}$ \overline{O} \overline{Q} -4 \overline{O} $\ddot{\text{o}}$ \ddot{O} Ō \ddot{O} $\mathbf{1}$ $\begin{array}{c} 1 \\ 0 \end{array}$ $\begin{array}{c} 0 \\ 0 \end{array}$ \sim 0 $^{-1}$ -4 $\overline{1}$ \ddot{Q} -0 \bullet O \bullet \bullet O \circ \ddot{O} \ddot{o} -0 $\overline{1}$ $\mathbf{1}$ \overline{O} \circ -4 $\frac{1}{\dot{Q}}$ \circ -0 \bullet O \bullet \bullet O \bullet \bullet O \bullet \bullet O \bullet -0 -4 $\overline{1}$ \circ -0 $-$ O \circ \circ O \circ -0 $-$ Q -0 $\mathbf{1}$ -4 $\mathbf{1}$ $\mathbf{Q} = \mathbf{Q}$ $\boxed{1}$ \ddot{O} \bullet 0 $\overline{2}$ \ddot{O} \bullet \circ O \circ \circ \circ -0 \bigcirc (11x11 CUBIC SPLINE SLOPE MATRIX.) [A] $0.577 - 0.15$ 0.041 -0.01 0.002 -0.00 0.000 -0.00 0.000015 -0.00 0.000 -0.15 0.309 -0.08 0.022 -0.00 0.001 -0.00 0.000 -0.00003 0.000 -0.00 0.041 -0.08 0.290 -0.07 0.020 -0.00 0.001 -0.00 0.000107 -0.00 0.000 -0.01 0.022 -0.07 0.288 -0.07 0.020 -0.00 0.001 -0.00040 0.000 -0.00 $0.002 - 0.00$ $0.020 - 0.07$ 0.288 -0.07 0.020 -0.00 0.001495 -0.00 0.000 -0.00 0.001 -0.00 0.020 -0.07 0.288 -0.07 0.020 -0.00558 0.001 -0.00

 $0.000 - 0.00$ 0.001 -0.00 0.020 -0.07 0.288 -0.07 0.020832 -0.00 0.002 -0.00 0.000 -0.00 0.001 -0.00 0.020 -0.07 0.288 -0.07774 0.022 -0.01 $0.000 - 0.00$ 0.000 -0.00 0.001 -0.00 0.020 -0.07 0.290163 -0.08 0.041 -0.00 0.000 -0.00 0.000 -0.00 0.001 -0.00 0.022 -0.08290 0.309 -0.15 0.000 -0.00 0.000 -0.00 0.000 -0.00 0.002 -0.01 0.041451 -0.15 0.577 (INVERTED 11x11 MATRIX FROM ABOVE.)

 $E A J^{\sim} - 1$

THE FORMULA FOR CALCULATING THE SPLINE FUNCTIONS:

 $[AA^{\wedge} - 1 \times \Im [U] = [S]$

[U] IS THE INPUT MATRIX [S] IS THE RESULTING SLOFE MATRIX [A] IS THE CUBIC SPLINE SLOPE MATRIX

 $\overline{1}$

 $-B5-$

TOTAL:

 $3.2E-18$

TOTAL:

--------------- $5.8E-17$

 $\begin{tabular}{ll} \tt TOTAL: & & 4.1E-17 \end{tabular}$

TOTAL: 3.563441

TOTAL: 51-21251

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TOTAL: 51.21251

TOTAL: 3.563441

TOTAL:

 $4.2E-17$

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"HERMFAST.WK1" USED TO FIND THE INTERPOLATING POLYNOMIALS AND SURFACE AREA OF A BODY OF REVOLUTION.

The inputs are the station radii and slopes, and interstation distances, The outputs are cubic polynomials which describe the radius of the body between stations, and are used to calculate the body surface area. The generated polynomials and their first derivatives are CONTINUOUS over the length of the body, which is ideal for submarine surface area calculation.

CAUTION: IF THE ACTUAL INPUT BODY HAS DISCONTINUOUS FIRST DERIVATIVES, THE LOCATION OF THE SLOPE DISCONTINUITY SHOULD BE TREATED AS TWO VERY CLOSE STATIONS, EACH WITH ITS OWN SLOPE! PROCEDURE

(1) INPUT ACTUAL RADII, SLOPES, AND STATION SPACINGS.

(2) HIT CF9D TO CALCULATE INTERPOLATING COEFFICIENTS, AND AREAS.

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TOTAL: 17. 15679

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-B18-

 1 TO 2 : matrix input: LF1 $X1:$ $\mathbf{1}$ $\mathbf{1}$ $\mathbf{1}$ $\mathbf{1}$ $\mathbf{1}$ 2 [F]= $\sqrt{2}$ \mathbf{B} -4 $X2:$ $\mathbf{1}$ -2 $\overline{1}$ \Box \circ LOCAL INFUTS: $12 -4$ -1 $\ddot{\Omega}$ 1.073 $UI:$ DET LF1 = -1 $U2: 1.984$ $S1:$ 1.3 -2 $S2:$ 0.9 -2 $\overline{1}$ $\overline{1}$ $CF3^{\sim}-1$ -9
12 $\overline{9}$ -5
8 -4 LOCAL COEFFICIENTS: -12 - 5 a: 0.378 -4 5 -4 -2 b: -1.90 3.971 C.E. $d:$ -1.37 SUB-a bc d 1 TO 2 STATION 0.378 -1.90 3.971 -1.37 ------------------------AVG SURF NUMBER: $\mathbf{u} \times \mathbf{n}$ x^43 x^42 x^41 x^40 U(x) L(i) RADIUS AREA $\overline{1}$ $1 \t 1 \t 1 \t 1 \t 1.073979$ $1.331 \quad 1.21 \quad 1.1$ 1 1.196679 0.158288 1.135329 1.129149 1.1 1.2 1.728 1.44 1.2 1 1.306293 0.148375 1.251486 1.166724
1.3 2.197 1.69 1.3 1 1.405093 0.140575 1.355693 1.197431
1.4 2.744 1.96 1.4 1 1.495351 0.134708 1.450222 1.227465
1.5 3.375 2.25 1.5 1 1.579336 0.130589 1.537343 1.2 1.7 4.913 2.89 1.7 1 1.737579 0.126981 1.698451 1.355101
1.8 5.832 3.24 1.8 1 1.816379 0.127316 1.776979 1.421498
1.9 6.859 3.61 1.9 1 1.897993 0.129077 1.857186 1.506206
2 8 4 2 1 1.984693 0.132351 1.941343 1.614398

TOTAL: 13.18228

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 2 TO $3:$ matrix input: [F] (local) \mathbf{B} $4¹$ -2 $X2:$ $2 \mathbf{1}$ 3 $[Fig12]$ $27[°]$ $9[°]$ -3- $X3:$ $\mathbf{1}$ $\overline{1}$ $12₁$ -4 \circ LOCAL INPUTS: $27[°]$ -6 $\overline{1}$ \ddot{O} $U2:$ 0.660 1.019 DET $[F] =$ U3: -1 $S2:$ 0.9 $S3:$ 0.01 -2 -2 -2 ~ 1 $\overline{1}$ 15 -8 -7 $CFJ^{\wedge}-1$ -15 $\overline{36}$ -36 -21 LOCAL COEFFICIENTS: 16 $\overline{28}$ -18 a: 0.191 -12 -27 -1.88 $b:$ 6.133 CE 11 -5.60 $d:$ SUB-a b c d 2 TO 3 STATION 0.191 -1.88 6.133 -5.60 ------------------------AVG SURF NUMBER: x^2 x^2 x^2 x^2 x^2 x^2 x^2 y^2 y^2 y^2 y^2 y^2 y^2 y^2 y^2 y^2 z^2 z^2 z^2 y^2 z^2 z^2 $\mathbf{u} \times \mathbf{u}$ 8 4 2 1 0.660441 \mathbb{Z}^2 $2.1 -$ 1 0.743305 0.129871 0.701873 0.572731 2.1 9.261 4.41 10.64 4.84 2.2 1 0.812666 0.121699 0.777985 0.594896
12.16 5.29 2.3 1 0.869673 0.115107 0.841169 0.608371 2.2 2.3 13.82 5.76 2.4 1 0.915478 0.109991 0.892576 0.616856 2.4 2.4 13.82 5.76 2.4 1 0.915478 0.109991 0.892576 0.616856
2.5 15.62 6.25 2.5 1 0.951233 0.106199 0.933355 0.622802
2.6 17.57 6.76 2.6 1 0.978087 0.103543 0.964660 0.627588
2.7 19.68 7.29 2.7 1 0.997192 0.101808 0.987639 0.6

TOTAL: 6.188598

TOTAL: 6.981447

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TOTAL: 0.776238

5 TO 6: matrix input: [F] (local) $5\overline{a}$ 125 25. 5 $X5:$ $\overline{1}$ XE: $6 - 1$ $[FF] =$ 216 - 36 - $6²$ $\overline{1}$ 10 - 75 $\overline{1}$ \circ $\overline{1}$ LOCAL INPUTS: $12⁷$ \ddot{O} 108 $U5:$ 0.471 $DEF E =$ $UE:$ 0.450 -1 **S5:** \sim 0. -2 **S6:** -2 -0.02 \sim 1 \blacksquare -17 $EF1^{\sim}-1$ -33 33 -16 -180 LOCAL COEFFICIENTS: 180 - 96 -85 $325 - 180$ a: 0.021 -324 -150 -0.36 $b:$ 2.035 $C =$ d : -3.27

SUB-a b c d 5 TO 6 STATION 0.021 -0.36 2.035 -3.27 ----------------AVG SURF NUMBER: x^2 x^2 x^2 x^2 x^2 x^2 y^2 y^2 L(i) RADIUS AREA $\frac{H_X H}{2}$ -5 $\begin{array}{cccc} 125 & 25 & 5 & 10.471698 \\ 132.6 & 26.01 & 5.1 & 10.471296 \end{array}$ 1 0.471296 0.100000 0.471497 0.296253 5.1 5.1 132.6 26.01 5.1 1 0.471296 0.100000 0.471497 0.296253

5.2 140.6 27.04 5.2 1 0.470179 0.100006 0.470738 0.295792

5.3 148.8 28.09 5.3 1 0.468475 0.100014 0.469327 0.294929

5.4 157.4 29.16 5.4 1 0.468475 0.100023 0.467 ---------------------------

TOTAL: 2.909633

TOTAL: 2.852143

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TOTAL: 2.065701

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TOTAL: 0.981191

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-----NUMBER : A SURF AND SURF) (but in the RADIUS of AREA 9.1 753.5 82.81 9.1 $1-0.02025$ 0.102029 0.010125 0.006490 -0.032 0.100687 0.026125 0.016527 9.3 804.3 86.49 9.3 $1 - 0.03675$ 0.100112 0.034375 0.021622 9.4 830.5 88.36 9.4 1 -0.036 0.100002 0.036375 0.022855 9.5 857.3 90.25 9.5 1 -0.03125 0.100112 0.033625 0.021151 9.6 884.7 92.16 9.6 $1 -0.024$ 0.100262 0.027625 0.017402 9.7 912.6 94.09 9.7 1 -0.01575 0.100339 0.019875 0.012530 9.8 941.1 96.04 9.8 1 -0.008 0.100299 0.011875 0.007483 9.9 970.2 98.01 9.9 $1 - 0.00225$ 0.100165 0.005125 0.003225 10 1000 100 10 100 0.100025 0.001125 0.000707 ----------------

TOTAL: 0. 129997

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APPENDIX C: CALCULATION OF SHAFT HORSEPOWER WHILE SUBMERGED

Essential to the calculation of endurance range and indiscretion rate is the calculation of shaft horsepower (SHP) required to make a given speed. The following formulas are taken from Reference (16), and is used to determine the required SHP at various speeds in submerged operating mode.

```
SHP = EHP/PC EQN C-1
```
EHP = 0.00872 (V^3) * [WS* (Cf+Ca+Cr) + (Ss*Cds) + (Sa*Cda)] $EON C-2$

where:

Table CI lists the values of required SHP for each of the submarines, for speed ranges of which each is capable.

TABLE C1: Calculated values of required shaft horsepower (SHP) as a function of speed. The lower part of the table indicates the correlation of the calculated data with that found in the literature. (Sheet one of two).

APPENDIX D: CALCULATION OF ADDED RESISTANCE AND REQUIRED SHAFT HORSEPOWER WHILE SNORKELING

When the submarine operates near the free surface of the ocean, it generates gravity waves. Generating the gravity waves requires power, which must be supplied by the submarine if it is to remain at the same speed as it had when transiting more deeply submerged. The power increase is not great, unless the submarine is operating at Froude numbers greater than about 0.6, or submergence depth less than one tenth of its length. Reference (16) lists a chart and provides a methodology for determining the added resistance co efficient due to operating close to the surface, Cw, as a function of Froude number, length-to-diameter ratio, and submergence ratio (operating depth divided by overall length) . The calculations for the computation of Cw are as follows:

> (1) . Enter chart with submergence ratio and Froude number

(2).
$$
Cw =
$$
 $-- (-L/D) -1.3606] * (L/D) -2 $QN D-1$$

(3). SHPw = 0.00872 (V~3) (WS) (Cw) EQN D-2

where:

The results of the calculations for each of the submarines are as listed in Table Dl

operating at snorkel depth. (Sheet one of two).

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APPENDIX E: CALCULATION OF ENDURANCE RANGE

The fuel endurance range is calculated at the snorkel depth, and depends upon the following factors:

> (1) Diesel engine specific fuel consumption. (2) Bunker fuel load.
(3) Transit speed. (3) Transit speed.
(4) Speed vs Power (4) Speed vs Power relation. (5) Hotel electric load. Complement. (7) Water temperature, (affects heating and/or A/C load). (8) Sea State, (to the extent that it affects snorting).

All of these factors may play a part in determining the endurance range of the diesel-electric submarines, but only items (1) through (5) above can be estimated with a degree of accuracy given the data available in the literature.

It should be noted that in the matter of endurance range, "bigger is better". The reason for this economy of scale is that the drag force on a submerged body is proportional to its wetted surface area, but the fuel capacity is proportional to its internal volume.

The following relation, adapted from Reference (3), may be used to calculate endurance range:

 $Rf = 2240[(F)(0.8)(Nem)(V)]/[SFC)(SHP + 1.34*Lh)]$ EON E-1

where:

 $E-1$

Table E-l lists the fuel endurance range, calculated at snorkel depth.

"It may be mathematically proven that the maximum endurance range occurs at the speed for which the shaft horsepower required is exactly half of the hotel electric power requirement, (for ^a constant hotel power load)." - Harry Jackson, P.E., CAPT USN, (Ret.) 24 April, 1988

The following is one way of proving CAPT Jackson's statement:

$$
Pt = (SHP) + (Lh).
$$
 EQN E-2

where:

Pt = Total power required at a given speed, HP. SHP = Shaft Horsepower required at a given speed, HP. Lh = Hotel electric load, assumed constant, HP.

Equation E-2 quantifies the power required at a given speed. The SHP is a function of speed, as shown by Equations C-l and C-2, and can be approximated by the following expression:

$$
SHP = (Kv)(V2)
$$
 EQN E-3

 $where:$ K

$$
Kv = A constant, determined from EQN C-1 and C-2.
$$

Equation E-3 is suitably accurate within a suitable neighborhood of the point of evaluation of Kv. The maximum endurance range will occur at the speed for which the energy expenditure per mile is the least. The energy per mile is related to the power output by this expression:

$$
E/Nm = Pt/V = (Kv)(V2) + (Lh)/V EQN_4
$$

where: E

= Total Energy required by the ship at some speed, HP-Hour. Nm = Nautical miles.

The energy per nautical mile will have a minima where its slope is zero, a point which may be found by:

$$
(E/Nm) = 2(Kv)V - (Lh)/(V2) = 0.
$$
 EQN E-5

Rearranging Equation E-5:

$$
Lh = 2(Kv)(V2) = 2(SHP).
$$
 EQN E-6

Which was to be proved.

loadout, at snorkel depth. For PUBIS, values reflect range on emrgency diesel. (Sheet one of two).

TABLE E1: Calculated values of endurance range based upon bunker fuel loadout, at snorkel depth. MIDGET 100 calculated at deep submerged depth. (Sheet two of two).

APPENDIX F: LEAD-ACID BATTERY POWER AND ENERGY CHARACTERISTICS

F.l. Type of Battery

It is commonly held that the type of secondary storage batteries in modern diesel-electric submarines are of the lead-acid variety, although the literature did not confirm this. Some of the reasons for the popularity of lead-acid cells in submarines are as follows, from Reference (20):

- (1) Lowest cost (by a factor of ten) per KW-Hr of all storage batteries.
- (2) Reasonably high power to volume density.
- (3) The weight is beneficial to stability.
- (4) Maintainance is available throughout the world.
-

(5) Good safety record.

Reasons for the popularity of lead-acid batteries in submarine propulsion.

F.2. Power and Energy Capacity

The energy available from a lead-acid cell is dependent upon the rate of energy extraction - the power demanded of the cell relative to the cell's capacity. This is because the internal electrical resistance of the lead plates and the internal fluid resistance of the ions in the acid electrolyte both increase with increasing power demands. This effect has been minimised in stateof-the-art batteries manufactured by such firms as Varta and Hagen of West Germany, and Gould of the U.S. Below are listed the most important factors concerning the battery capacity are, from Reference (20) :

- (1) Battery nominal energy capacity.
- (2) Battery design, (geometry and structure)
- (3) Maintainance state of the battery.
- (4) Number of previous deep-discharge cycles on the battery.
- (5) Battery internal resistance.
- (6) Battery power vs energy relation.
-

(7) Temperature of discharge.

Factors contributing to battery capacity.

For a typical lead-acid storage cell of the type used in submarines, the total energy capacity, when discharged at the 100-hour rate, is approximately 23.5 KW-Hrs, calculated from Reference (16) . The available energy capacity of the battery is reduced for faster discharge rates. A numerical curve-fit describing the

energy capacity of this typical cell, as a function of servicelife, may be descibed by a sum of first-order transients, given by Equation $F-1$.

where: E

- Eb = The energy in the entire battery, KW-Hrs.
Td = Service life, Hrs. The time required to $=$ Service life, Hrs. The time required to discharge the battery to a given end-voltage at a given discharge rate.
- #C = The number of standard cells in the battery.

Table Fl gives the calculated values of battery energy capacity for each of the submarines, at discharge rates equal to that necessary to maintain the corresponding speed. Figure F-l graphically displays the information of Table Fl.

F.3. Other Factors

A well-designed battery will minimize internal resistance and allow more complete energy utilization at high discharge rates. One way of reducing the internal resistance is to use sandwich anode and cathode plates, which have internal cores of copper which is about fifteen times more conductive than lead at room temperature, References (20) and (22). This decreased resistance is very important in reducing the ampere heating of the lead plates at high discharge currents. The battery may also be provided with its own cooling system to prevent overheating. The battery room should also be equipped with a separate ventilation system, to safely duct away any evolved hydrogen during charging periods

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Available Battery Energy

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APPENDIX G: CALCULATION OF BATTERY ENDURANCE RANGE

Once the battery power and energy capacities are determined, the submerged endurance as a function of speed may be calculated. The battery range falls off with increasing speed even more quickly than the fuel endurance range, because the submarine is fighting the non-linear dissipative effects of battery internal resistance as well as the non-linear external resistance of the sea.

 $-G1$

The maximum range which a diesel-electric submarine may achieve depends upon elements (1) through (8) detailed in Appendix E, and also upon factors which relate to the submarine's battery capacity. The following relations, adapted from References (3) and (20), may be used to calculate battery endurance range:

where:

Note that because Eb, Td, and Mb are interdependent, Equations G-l through G-3 must be solved iteratively. The results of the iterative calculations are listed in Table G-l.

The procedure is to iterate beween Equations G-2 and F-l to solve for Td, then evaluate G-l to find the range.

Table Gl lists the numerical values resulting from this procedure, for the appropriate speed ranges for each submarine.

batteries alone, 80% depth of discharge. (Sheet one of two).

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TABLE Gl: Calculated values of each submarine's endurance range on batteries alone, 807. depth of discharge. (Sheet two of two)

APPENDIX H: CALCULATION OF INDISCRETION RATE AND INDISCRETION INTERVAL

The necessity to charge the storage batteries requires the dieselelectric submarine to operate, for some portion of time, either on the surface or snorting near the surface. The ratio of the time spent on or near the surface to the total time spent in transit is called the indiscretion rate, and it is desirable to keep it as low as possible.

> (1) Electric generator/alternator power capacity. (2) Number and size of cells in the battery. (3) Type of battery.
(4) Ability to contr Ability to control the charging parameters for optimal charging profile, (closed-loop control). (5) Vessel transit speed and associated SHP. (6) Hotel electric load.

Factors affecting indiscretion rate.

Calculation of the indiscretion rate may be performed as follows:

IR = $Tr/(Tr + Td)$ $EON H-1$ (DoD) (Eb) $Tr =$ ------------------------------- EON H-2 $(Pdg - (SHP/1.34) - Lh)(Nbc)$

where:

The calculated values of indiscretion rate are listed in Table Hi. Note that the Shaft horsepower used in Equation H-2 is the "near the surface" value, since the submarine is operating near the surface when snorkeling to recharge its batteries. The

 $H-1$

speed range Listed in Table Hi extends naturally, only to each submarine's max snorkel speed.

The indiscretion interval at a given speed is the duration of time a submarine may transit at the speed, while completely sub- 'merged and without snorkeling. It is actually another name for the battery service life at the given speed. Table H2 lists the calculated values of indiscretion interval for each submarine's submerged speed range.

Note that the average electrical-to-chemical energy conversion efficiency is used. It is suitable for comparison purposes, but assumes a constant charging rate. The actual charging rate and charging efficiency is a function of the recharge power, since the same internal resistance factors are at work in the recharging process as in the discharging process. So shorter recharge times are less efficient (and hence longer) than indicated in this first-order calculation.

-H5-

 $\hat{\boldsymbol{\epsilon}}$

Table H2: Calculated values of indiscretion interval as a function of submerged speed. This is the same as the battery discharge time at a given speed. The values given for the MIDGET 100 are termed the battery discharge times only since this subdoes not snorkel. (Sheet two of two).

APPENDIX I: ESTIMATION OF WEIGHT GROUPS

The estimation of the weight groups from open literature is dif-
ficult. The following empirical relations have been developed to The following empirical relations have been developed to generate weight values for functional groups which were not found in the literature.

where:

Equations 1-1 and 1-2 are adapted from Reference (15). Equations 1-3 through 1-7 were based loosely on the weight values for the BARBEL, with consideration of the variables which contribute to the weight of a functional group.

After the above equations were evaluated for each submarine, the formula-calculated submerged displacement is compared to the reference submerged displacement, and all of the calculated weights are scaled by a common coefficient in order to bring the calculated displacement equal to the reference displacement.

The computed values for each submarine weight group are shown in Table 8-1

APPENDIX J: CALCULATION OF ANAEROBIC DIESEL FUEL/OXYGEN LOAD

The anaerobic diesel cycle developed by Sub Sea Oil Services and used in the MIDGET 100 is truly an engineering accomplishment. According to the literature, the submerged endurance range of the MIDGET 100 is unmatched by all vessels within ten times its displacement. The reason for this is the relative energy-to-weight and energy-to-volume densities of fuel oil/oxygen as compared to lead/acid storage batteries.

Since the propulsion plant is "anaerobic" as far as the atmosphere is concerned, the combustion oxygen must be carried along with the submarine.

The following analysis is taken from Reference (3) , and shows that the weight of the required oxygen loaded is approximately three-and-a-half times the weight of the fuel oil loaded.

For the combustion of a typical long-chain saturated hydrocarbon, the process balance between reactants and products is:

$$
(C_{N}H_{2N}) + (3N/2) (O_{2}) = (N) (CO_{2}) + (N) (H_{2}O)
$$
 EQN J-1

where:

 C_NH_{2N} = One mole of long-chain hydrocarbon.

 $0₂$ = One mole of oxygen.

 $CO₂ = One mole of carbon dioxide.$

 $H_2O = One$ mole of water.

 $N = A constant$.

Equation J-l shows that (3N/2) moles of oxygen are needed for the combustion of one mole of fuel oil. Moles may be converted into weights by the following relations:

 $J-1$

One mole of $C_N H_{2N}$ weighs: N[(12) + 2*(1)] = 14N

 $3N/2$ moles of O_2 weigh: $(3N/2)$ [(2) (16)] = 48N

$48M/14N = 3.428 =$ Relative weight of required oxygen to fuel.

The minimum weight of the oxygen is 3.428 times that of the fuel oil. Assuming that the amount of oxygen used in combustion is five-percent above the amount needed for stoichiometric balance:

> $3.428 * 1.05 = 3.600 =$ Relative weight of loaded oxygen to fuel.

APPENDIX K: FACTORS AFFECTING CREW ENDURANCE

Some of the most important factors which affect crew endurance are listed below. the factors can be grouped into two categories: Vessel-related, and Personnel related.

- 1. Number of crewmembers, (complement).
- 2. Quantity of provisions loaded.
- 3. Fresh water tank capacity.
4. Existence of fresh water d
- 4. Existence of fresh water distillers.
5. Air purification capability and effe
- 5. Air purification capability and effectiveness.
6. Space and volume per crewmember.
- Space and volume per crewmember.
- 7. Quality of provisions loaded.
8. Crew discipline and morale.
- 8. Crew discipline and morale.
9. Crew training state as indi
- Crew training state as individuals and as a team.
- 10. Crewmember's ages.
- 11. Crewmember's previous experience with similar situations.
- 12. Crewmember' s psychological profiles and temperament.

The focus of this study is necessarily directed to items (1) through (6) . Hard data for the above factors is only available with consistency for item (1) in the literature, and even then, the sources often disagree. As such, much of the other data is gleaned from drawings that may be provided in the literature, with the author's best estimate of the unprovided data items.

APPENDIX L: VULNERABILITY AND SURVIVABILITY FACTORS

Submarine vulnerability is directly proportional to its detectability of the submarine to acoustic, magnetic, thermal, and visual sensors. The stealth capability of the submarine is its greatest asset as a military device, although it would be pointless militarily to build a submarine which was merely stealthy, without enough endurance to transit to where needed, enough sensor capability to be effective, and enough speed and weaponry to accomplish its mission.

So, the idea, it would seem, is to build a submarine as invulnerable as possible, which means primarily as quiet as possible but also encompasses the ability to defend itself or to escape in the eventuality that it is detected.

Not all submarines have been designed to this philosophy. Some submarines seem to have been designed to withstand the damage from an attack, and continue. This is the concept of survivability. The U.S.S.R. in particular seems to have taken this approach, with and perhaps not only for the purpose of surviving wartime attack, given the poor peacetime safety record of Soviet submarines, Reference (27).

Survivability may be improved by keeping these concepts in mind during the design of the submarine.

1. Redundant and separated vital systems and components: (a) . MBT blow valves. (b) . Propulsion motors. (c) . Electric power cables, (d) . Diesel generator sets, (e) . Battery banks. 2. Strength and toughness of hull material. 3. Using double-hull design. 4. Dividing the pressure hull into watertight compartments 5. Using fire-resistant materials within the submarine. 6. Adequate and appropriate fire-extinguishing gear. 7. High state of crew training. 8. High state of equipment readiness. 9. Emergency air-breathing apparatus.

Finally, there is the issue of crew survivability and escape in the event that the submarine is sunk. In several instances, the crew would be unable to survive attacks severe enough to sink their submarine. The humane approach is certainly to provide the crew with an escape pod, in the event its use becomes neccessary. Still, some countries prefer not to have an escape pod mounted, due to thier philosophy that the crew will perform more vigorously if the submarine itself is the only ticket home, Reference (15).

APPENDIX M: ESTIMATION OF HOTEL ELECTRIC LOAD

The open literature gives no empirical formulas for hotel electric load. The following relation is the author's best attempt to parameterize this item which is an important factor in the en durance calculations, primarily so that the hotel load of each submarine is calculated by the same formula, rather than some equally arbitrary but non-uniform basis.

 $Lh = 1.5$ (Vmob) + 4(Vc3i) + 1.5(Vss) + 1(Vwep) EQN M-1

where:
Lh

 $=$ Hotel electric load, KW. $Vmob = Volume of the mobility machinery, cu ft.$ Vc3i = Volume of C3I equipment spaces, cu ft. Vss = Volume of ship support spaces, cu ft. Vwep = Volume of weapons spaces, cu ft.

APPENDIX N: DIESEL ENGINE DATA

The power output of each submarine's prime mover (diesel engine/ alternator set, except for RUBIS) is of great importance in the calculation of sustained maximum speed while surfaced or snorkeling, and in the calculation of the indiscretion rate. The power output of the main propulsion electric motor used in each submarine is important in the calculation of maximum submerged speed.

The open-literature data on the diesel engines and main-propulsion motors is listed in Table Nl

SUBMARINE NAME

TABLE Nl: Diesel engine and electric propulsion motor data.

Abbreviations:

===============

GE - General Electric. GMT - Grandi Motori Trieste. JEU-SCHNDR - Jeumont-Schneider. MTU - Motoren-und Turbinen-Union Friedrichshafen G.m.b.H. SSOS - Sub Sea Oil Services of Micoperi S.p.A.

APPENDIX 0: DATA ON OTHER MODERN SUBMARINES

The literature contained information on several other submarines which were not included in the detailed study. The information shown in Table 01 is taken primarily from References (5), (6), (7), and (17). The individual data entries are not attributed to the specific reference since most entries could be susbstantiated by multiple references.

(Sheet one of five).

 $-0.2 -$

(Sheet two of five).

(Sheet three of five).

WEAPONS SYSTEMS

(Sheet five of five).

APPENDIX P: DIESEL ENGINE SPECIFIC FUEL CONSUMPTION VARIABLES

The specific fuel consumption of modern diesel engines is in the range of approximately 0.30 to 0.35 lbs/HP-Hour, Reference (3). This exceptionally-low SFC is achieved when the engine is run on the test stand, and under the "best" conditions of engine speed and power loading. For other speeds and other power loadings, the SFC is generally greater than this value. The SFC at the actual condition of loading may be approximated graphically by the generic diagram of Figure P-l. Figure P-l shows that the SFC will increase at power levels other than approximately the 90% power level, and will also increase at engine RPM other than the approximate optimum of 90% of rated maximum RPM.

The specific fuel consumption of the installed diesel engines, while snorkeling, will be greater still than the values predicted by Figure P-l, because of flow resistance in the snorkel intake and uptake.

For the anaerobic diesel engines in the MIDGET 100, the exact SFC under any conditions is not known, since Sub Sea Oil Services has not published the details of their technology. It is reasonable to assume that the SFC of the anaerobic diesel cycle, as a whole, is greater than that of a comparable conventional diesel cycle, since the carbon-dioxide exhaust gas produced, (after any startup transients) must be discharged overboard at ambient pressure.

If the assumption is made that the operators of the diesel engine will operate the engine at the optimum engine speed/power point, then the specific fuel consumption of the installed marine diesel engines, as described by Figure P-l, may be approximated by the following

 $SFC = 0.40 + 0.30(BHP90\% - BHPop) / (0.65*BHPr)$ EQN P-1

where:

SFC = Specific fuel consumption at the actual operating point, lbs/HP-Hr. BHP90 = The power output of the engine at its assumed optimum efficiency operating point: 90% of rated power, HP. $BHPop = The power output of the actual operating point,$ HP. BHPr = The rated power output of the engine, HP .

Assumed values of the SFC for each engine, calculated from Equation P-l, are listed in table Pi.

FIGURE P1: DIESEL ENGINE SIECIFIC FUEL CONSUMPTION GENFEIC PROFILE. [FROM Reference (3).]

fuel consumption dependency with engine loading of Equation P-l, that the battery is not being charged, and that the diesel is supplying power for propulsion and hotel electricity. (Sheet one of two).

function of speed. This calculation assumes the specific fuel consumption dependency with engine loading of Equation P-l, that the battery is not being charged, and that the diesel is supplying power for propulsion and hotel electricity. NOTE: For MIDGET 100, the diesel is providing propulsive power only. (Sheet two of two).

APPENDIX Q: CALCULATION OF PROVISIONING ENDURANCE

The provision endurance range is based primarily upon the foodstores loadout for the crew. It is a linear function with speed, being directly proportional to the speed. For this reason, the actual endurance range of a submarine may be far less than had been calculated from the consideration of the bunker fuel loadout alone. At low speeds, the submarine range is limited by the loadout of foodstores, since those are exhausted prior to the exhaustion of bunker fuel.

The provisioning endurance range may be expressed as:

$$
Mpr = (\#P)(V)(24) \t\t EQN Q-1
$$

where:

The comparison of fuel endurance range to provision endurance range is shown in Figures in Chapter X.

 $\hat{\mathcal{A}}$

 $-Q2-$

 $\mathcal{L}^{\mathcal{L}}(\mathcal{A})$.

APPENDIX R: ESTIMATION OF PRISMATIC COEFFICIENT

The prismatic coefficient of each submarine is found by first calculating the envelope volume, which will have tha dispalcement of the submarine while submerged, plus free flood, which may be assumed to be approximately five percent. The volume ^s of the appendages and sail are then estimated from pictures in the literature, and are subtracted from the envelope volume. The result of this calculation is the bare-hull volume.

The ratio of the volume of the bare hull to the volume described by the product of the maximum section area and the length overall is the prismatic coefficient.

The calculated values for prismatic coefficient are shown in Table Rl

SUBMARINE NAME

Table R-l: Calculation of prismatic coefficient.

 $419 - 500$

