AIR ATOMIZATION OF FUEL OIL

SUBMITTED BY: LT. P.F. ERKENBRACK, U.S.NAVY LT. R.J. ZOELLER, U.S.NAVY

THESIS SUPERVISOR: PROFESSOR H.C. HOTTEL

JANUARY 16,1948



AIR ATOMIZATION OF FUEL OIL

by

Lieutenant Phillip Frederick ErkenBrack, U. S. Navy B.S., U. S. Naval Academy, 1942

Lieutenant Robert Joseph Zoeller, U. S. Navy B.S., U. S. Naval Academy, 1942

THERE

Submitted in Partial Fulfillment of the

Requirements for the Degree of

MASTER OF SCIENCE

in

NAVAL CONSTRUCTION AND ENGINEERING

from the

MASSACHUSETTS INSTITUTE OF TECHNOLOGY

IID ANT TO MOIPATAOTA WIL

Yd

Lieutenant Phillip Preserick FrienBrack, U. S. Have B.S., U. S. Davel Academy, 1942

Lieutemant Robert Tomerh Scaller, U. B. Murr B.S., U. S. Maval Account, 1942

Thesis

EG

adt to mountifier faiture at bettimfue

insquirements for the legites of

MANTER OF SCIENCE

11

DATE CONSTRUCTION ANT EXCLUSION DAVIS

ods Borr

MASSACROSTIN INSTITUTE OF THE STORE OF ASSAM

Cambridge, Massachusetts, January 16, 1948.

Professor J. S. Newell, Secretary of the Faculty, Massachusetts Institute of Technology, Cambridge, Massachusetts.

Dear Sir:

In accordance with the requirements for the Degree of Master of Science in Naval Construction and Engineering, we submit herewith a thesis entitled "AIR ATOMIZATION OF FUEL OIL".

Respectfully,

0.- 0

The last of the state of the

Cantonican, famousta, famousta, famousta, famousta, famousta, famousta, famousta, 1946.

Professor J. S. Newell, Secretary of the rould, Massachuset a Institute of Technilog,

.asteudouseaM .eghindmaD

Peer Sir:

In accordance with the recultements for the Degree of Master of Belence in Masal Construction and Englanding, we submit herewith a bhesis entitled "AIN ATOMICANTED OF TOXL OIL".

Capootrully,

ACKNOWLEDGMENT

The authors wish to express their appreciation for the assistance and advice of Professor Hoyt C. Hottel who suggested the subject, and under whose supervision the investigation was conducted.

In L'DI'M ROALOA

The authors wish to express their expresistion for the admission and defen of Professor Hoyt C. Hottel who sugmented the audject, and under whole supervision the investication was conducted.

the property of the level of the second data and the second data a

TABLE OF CONTENTS

																						Page
SUM	LARY	•••			•	•	•	•		•	•	•	•	•	•	•			•	•		1
INTH	RODU	ICTIO	N.	•	•	•	•	•	•		•	•	•	•	•				•	•		3
PROC	CEDU	RE .	•	•	•	•	•	•			•	•	•				•		•	•	•	6
RESU	JLTS		•	•	•		•	•	•			•	•	•	•	•	•			•	•	13
DISC	USS	BION	of	RI	ISU	ILI	'S	•		•	•					•				•		15
CONC	lus	IONS	•	•	•	•	•		•	•		•	•	•	•	•	•		•	•	•	27
RECO	MAG	NDAT	IOI	VS	•	•	•	•	•	•	•	•	•	•	•	•	•	•			•	29
APPI	INDI	X																				
A	۱.	SYMB	OL	3.				•		•	•			•						•		34
F	3.	SUPP	LEI	Œŀ	VTA	R	r :	EN!	rre	ODI	JC	FIC	ON	•			•		•		•	35
C	3.	equi	PM	en 1	e.			•			•		•		•	•	•	•	•	•	•	39
I).	NOZZ	LE	DA	T		M) (DI	LI	PR	OPI	ER!	PI)	ES					•	•	41
I	5.	CALI	BR	TI	101	1 0	F	E	U.	IP	ME	NT		•	•					•		43
F	r.	DATA	Al	ND	CA	IC	U	LA!	ri(ON	s.	•		•			•					49
G	¥.	SAMP	LE	CA	LC	UI	.A.	FI (ON	s.	•	•		•			•			•	•	55
F	I.	LITE	RA	FUI	RE	CI	1	T	IOI	NS												58

TABLE OF CONTENTS

anis																														
1					÷								•		2				•				•		•	7	N.A	NCH.	IVE	2
3		•				ø	4			4				4		•	R						. 1	01	T	00	icto	ar	MJ	
- 9			2				•	z		-	-														3	TO	132	00	19.9	And
13	1												-1		5				3							ċ	2.1	Jue:		Ł
15	•	э						a		ļ							2	1) 1	IL	121	23)	1	2.0	-	ta	EI	670	48	I	-
27					a			a					•			•				4				ax	10	18	10.5	ю.	00	2
29			•	2	4	•		9				•					•		•		S	M	OI	T.	d	83	DE	uo).J.	
																										XI	T	15	51	
31							•	•					•						•	•		S	IO	83	1	00		A		
35	,							a	10	01	[1]	JC	11	0,1	1	-13	[Y	A.	dri	12	20	rel		U)	E.		E		
39		-					•				Ċ								18		FM	170	1	IU	J ()	E		C		
43.					•		2	1.5	77	23	0	22		11	0	(M	IA	A		DA		L	2	201	8		a		
43						a				7	57	30	12	IU	J.,	z	1	0	1	0	I'I	1	10	IJ	[A]	С		E		
49								0				.8	11	01	1	Ai	IU	ra	14	C	a	W	6	FA	£.	D		T		
55							•		2				-	27	10	T	A	I	UC	L	10	1	1	I	LA	21	•	G		
58								9					31	NO.	12	21	1T	I	O	23	IU	T.	R	87	T	1		H		

SUMMARY

This thesis presents a macroscopic study of the effect of orifice diameter, fuel rate, air velocity and type of injection on the characteristics of a spray of fuel oil atomized by a high velocity air stream. The qualitative results were obtained from a close examination of photographs taken both by normal exposure technique and by use of the Edgerton high speed spark-lighting technique. The sprays investigated were those of U. S. Navy Diesel oil injected into an air stream in three ways: (a) Parallel to and in the direction of air stream flow; (b) perpendicular to the direction of air stream flow; and (c) parallel to and counter to the direction of air stream flow.

The results show that normal photographic procedure with time exposure to portray a spray envelope is of little value in studying atomization characteristics and, in fact, leaves erroneous impressions. Spark photography, on the other hand, gives excellent qualitative information and has possibilities for some quantitative development.

It was found that:

(a) For increasing orifice diameter, drop size and uniformity were not materially affected, dispersion increased.

(b) For increasing air velocity, drop size and dispersion decreased and uniformity increased.

(c) For increased fuel rate, uniformity and dispersion decreased and drop sizë increased at low air velocities and was not affected materially at high velocities.

STREAM

"his thesis prearnts a meroscopic study of the effect of crifice diameter, fund rate, air valocity and type of injection on the characteristics of a spray of fuel oil atomized by a high valodity air stream. The qualitative remults were obtained from a close examination of photographs taken both by mernal expensive technique and by use of the Edgerton high speed apart-lighting technique and by use of the Edgerton high speed of U. S. Mayr bluesh oil injected into an air stream in three ways: (a) familed to and in the direction of size stream flow; (b) permulsular to the direction of air stream flow; parallel to and direction of air stream flow; (b) permulsular to the direction of air stream flow.

The results she that normal photographic procedure with the appoint to portray a spray ervelope is of litel value in studying atomization characteristics and, in feet, laws erromenn is pressions. Spark photography, on the other band, gived excellent qualitative information and has possibilities for set qualitative development.

:tedt buvo_ tt

(a) For increasing orifice dismeter, drop size and uniformity were not materially affected, dispersion increased.

(W) for increasing air velocity, frop size wad dispersion decreased and uniformity increased.

(a) For increased fuel rate, uniformity and dispersion decreased and drop size increased at low air valueities and was not affected materially at bight velocities.

(d) For type of injection, spray characteristics were not materially affected.

Perpendicular and upstream injection offer serious disadvantages in the way of fuel nozzle distortion of the air stream. From all considerations, downstream injection from large orifices affords the best atomization. This is fortunate for in application to modern high rate combustion chambers, it means maximum flexibility with moderate pump size. (d) for type of injection, array Sharacteristics were not materially afforted.

Termendicular and opetrees injection offer errises disadvantages in the way of fuel nosale distortion of the air stream. Free all considerations, downstress injection from harge orifices effords the bast stoutsation. This is fortunate for is analisation to solver high rate contraction examples, it means maximum floritility with motorate sum after.

Rough Barry Barry Brit

Int reactions are and the second of the strength of the second stren

- I'd an expansion and an end of the local state of the s

- All and the property of the part of the last of the

S

INTRODUCTION

With recent increased interest in, and development of, high rate combustion chambers, studies of the atomization of liquid fuels by a high velocity air stream have assumed new importance. Jet, and turbo-jet engines, and gas turbines have available a high velocity air stream as an inherent part of the design which is most efficiently used as a fuel atomizing force. At present, insufficient knowledge of the variables and controlling factors of air atomization prevents a wholly scientific attack on the design problem with the consequent result that much of the combustion planning is done on a trial and error, or rule of thumb, basis.

Fuels are atomized mechanically by "solid injection", or by a gas stream. In the former, the liquid is atomized by forcing it under high pressure through a small orifice of special design into a stagnant gas. In the latter, the liquid is atomized by the shearing action of a high velocity gas stream on the surface of the liquid column as it is pumped from an orifice under just sufficient pressure to give the desired fuel rate.

Until quite recently, air atomization has always given way to pressure atomization in the combustion of fuel oils because of the high efficiency of pressure systems, the relatively simple problem of putting fuel under pressure and preheating it, and the unnecessarily complicated design

WOTFOUJOERIL

Nith rooms increment interest in, and persistents of, high mass anabustion of mhore, studies of the stanization of light fuels by a high velocity sit stream have secure new importance. Jet, and turbo-jet and her stream have secure bines mays available a high velocity sir stream as an inbarent part of the design which is now efficiently used as a feel stanizing force. At present, insufficiently used as of the variables and controling formers of sir storization prevents a wholly scientific streak on the design problem with the consequent result that and of the constant planning is done on a ortal and arror, or rule of thems, bash, bash

Just are stonized mechanically by "moils injoction", or by a gas strown. Is the former, the liquid is stonized by foroize it under high pressure througe a small orifice of special design into a stegment ges. In the intter, the liquid is stomized by the shearing wouldn of a high velocity gas stream on the surface of the liquid oclumn as it is pumped from an orifice uncar just mufficient promute to size the desired fuel rate.

Until quite recently, air stoningtion has siveys given way to pressure atomination in the emphation of fuel cils because of the high efficiency of pressure systems, the ruletively simple problem of putwing fuel under pressure and preheating it, and the unnecessarily complicated design problem of compressing large quantities of air and controlling the velocities required for proper atomization. However, the demand of modern power systems for compactness, lightness, simplicity, dependability, and most important, extreme flexibility, has shifted attention to atomization by an air stream. As the range of fuel rates increases in a pressure atomization system, the pressure required (and, consequently, the size and weight of pump) increases in far greater proportion - the fuel rate being proportional to the square of the fuel oil pressure.

4

Because of the tremendous scope of the field and the pressing need for specific information, a great deal of the research work on the subject of atomization pertains to commercial arrangements tested under fixed conditions. Also, because the characteristics of a liquid spray are so difficult to measure experimentally with accuracy, most of the work is of a qualitative nature. The meager quantitative data available to date is empirical in nature and investigators are generally in poor agreement. Some theoretical considerations have been made, but these, too, are meager.

In the literature, information is extremely sparse on the effect of orifice diameter on the spray characteristics of an atomized liquid. Longwell (11) has shown that drop size increases with increasing orifice diameter and decreases with fuel velocity at the orifice, the velocity being a function of the pressure, but this applies only to solid injection using swirl-type nozzles. It is reasoned that penetration problem of compressing inFree (muntilize of sir and sontrolling the velocities required for proper acompaction. Nonever, the demand of modern power systems for compactment, itentance, simplicity, departentility, and and tempertent, arbrane flexibility, has shifted attention to available by an sir abroam. As the range of fuel ruces intromumen in a pressure acomization system, the pressure required (due, consequently, the size and weight of pury) intromumen in greater proportion - the fuel ruce being proportional to the arguments of the fuel ruce being proportional to the

4

Because of the treassing same of the risis he the pressing need for specific information, a great call of the second work as the swoject of somiastion perfering to connercial mrangements ranked under fixed somiations. Also, because the characteristics of a light synce are so diffould to measure experimentally with some of the work is of a qualitably outputs in source, most of the date available to date is expirited in source, nost of the batter are passful in poor agreement, tone theoretical considerations have been made, but mean, too, are mader.

In the literature, information is extremely sparae on the affect of orlites discuster on the aproxy descentation of an about red liquid. (conquell (11) not shown hist trop size increates with factorating orifice elimentar and hearened with fuel velocity at the orifice, the velocity wing a function of the presence, but this applies only to wolid infection using swirt-type nounles. It is reasoned that panetration increases with decreasing orifice diameter, but even qualitative substantiation is lacking. It is known that orifice geometry is the most controlling factor for dispersion and spray intensity, but there is no information as to how orifice size affects them. It is felt, then, that an investigation of even a qualitative nature could add much to the knowledge of atomization in general, and to air atomization of liquid fuels in particular.

With this in mind, this thesis is concerned with studying the effect of orifice diameter on the characteristics of a spray of diesel oil formed by air atomization under varying and controlled conditions of air and fuel rate. For this purpose, a series of nozzles were photographed by the Edgerton Spark technique and, where feasible, by time exposure on the spray envelope at each of six conditions of fuel and air rate, and the results macroscopically compared.

increases with decreasing artrice diameter, hat area qualttative substantication is incking. It is known that origina generity is the most controlling fractor for diamersia and apray intensity, but there is no information as to how onlrice size affects than. It is full, then, that an investiquited of even a qualitative mature could not much to the boomledge of stomization in general, and to all similaritan of it quid fuels in particulur.

aith this in mind, this theals is concerned with studying the effect of critice dismatur on the observated all of a spray of discel dil formed by sir stomismition under verying and controlled conditions of sir and fuel rate. For this purpose, a veries of consiss were photographed by the digerton Spare tochnique and, where fousible, by time arposure on the spray envelope at each of all conditions of fuel and sir rate, and the remains marcacopically commend.

memory as a carrier shorts an access

A DESCRIPTION OF THE OWNER AND DESCRIPTION OF

A set to a first the set of the s

And a second sec

PROCEDURE

Description of Apparatus

The apparatus used was originally designed and constructed by Geoffrey Robillard (14), later modified by Robert Maxwell of the M.I.T. Combustion Research Laboratory, and finally modified for this thesis by the authors. It is designed to take high speed photographs of a liquid spray. A schematic arrangement of the apparatus is shown in Figure I. Figure II shows all the actual apparatus, with the exception of the air compressor, while Figure III is a close-up of the chamber.

The focus of the investigation is on a diesel oil spray contained in a glass chamber, and for obtaining and photographing this spray, three systems are necessary: the air system, the fuel system, and the photographic system.

The Air System: Air from a 100 psi, 533 cfm Allis Chalmers "NoTwin" gear type compressor flows through a twoinch pipe past a one-inch orifice for measuring air rate. The air then flows through a diffuser in which is a four-inch square section containing three fifty mesh screens in series which minimize turbulence and maximize a uniform velocity front. The diffuser exit is reduced through a nozzle to a one square inch cross section. The nozzle outlet is directly connected to the spray chamber. This chamber consists of two one-quarter inch thick optical flats and two one-quarter inch thick milled steel plates. These plates and flats form a

in sit pailin's

Description of Americaed

The apparetus uses was originally declared and youextracted by Geoffrey Hobilizer (14), later mairies by moment warwelt of the stirt. Genession Hasserah isterators, and finally modified for this these is up the authors, is in the signed to take high speed photographs of a light server. A sobesatic arrangement of the apparetus is shown in Stevre 1. Figure II shows all the setual spectaretus, with the secondium of the air compressor, while Figure II is a close-up of the obumber.

The found of the investigation is on a diesel of appar contained in a glass chamber, and for obtaining and photographics this spray, three systems are necessary: the air system, the fuel system, and the photostruphic system.

The Air Lysten: Air from a 100 pmi, 533 ofm Ailis Chaimers "AoTwin" dear type compressor flows through a twoinch pipe peat a una-inch orifice for measuring air rate. The air then flows through a diffuser is which is a four-inch aquere section containing three fifty much streams in astim which simistize turbulance and maximize a uniters valouity front. The diffuser sais is reduced through a morale to a one symme inch store section. The morale outlet is directly one-quarter has blow outlosi flate and names of two one-quarter is also plates. These plates and two one-quarter inch these milled store plates. These plates and two one-quarter inch square duct one inch on a side, inside dimension, and six inches in length, two opposing walls of which are perfectly transparent. The outlet of the spray chamber is connected to a two-inch exhaust line.

The spray chamber walls, when secured by thumb screws into aluminum blocks at each end, form a rigidly intact unit which slides into brass guide blocks secured to the nozzle exit and exhaust duct. The chamber is then secured in place by raising the lower guide block by means of a threaded collar.

Air temperature is measured at a thermometer well preceding the diffuser. Static pressure in the section between diffuser and nozzle is measured by manometer and calibrated against chamber pressure, as described in Appendix E. Static pressure downstream of the metering orifice and differential pressure across the orifice are measured by manometer.

A by-pass line from a point preceding the metering orifice to the exhaust duct contains a stop valve by means of which air rate is controlled. Air velocities from 125 to 830 feet per second can be attained in the chamber.

<u>The Fuel System</u>: Fuel is supplied from a five gallon reservoir by a "Gerator" gear pump capable of 150 psi and equipped with internal by-passes. The fuel is metered through a 0.025-inch orifice in half-inch brass tubing and measured by a fuel-over-mercury manometer independently calibrated, as described in Appendix E. aquare duct one imak on a alde, imaide dimension, and air inches in length, two uncould walls of which are perfectly transparent. The soliet of the spray unmour is connected to a two-inch eshamat line.

The spray shapper wills, when secured by thush server late alvainan blocks at each out, form a rigidly (start out which aligned into pross guide blocks ascared to the worned will and anhaust dust. The chamber is then secures in place by rateing the lower guide block by means of a threaded coller.

All feagerature is accepted at a thermometer wall precediar the diffurer. Static presence in the section between diffuer and courts is measured by menometer and culturated against abreader pressure, as described in Appendix I. Static pressure downstream of the seconds culture and differential pressure across the orifice are measured by menometer.

A 10-pare line from a point presenting the metering ortfiom to the exhaust dust contains a stop valve by means of which sir rate is controlled. Air valgeities from 125 to 830 feet per motored out he atteined in the ebentur.

The fact Drates: Number is supplied from a five suiton reservair by a "Corator" cost pump capable of 150 pul and outipped atta internal by-passes. The fuel is actored through a, 0.025-indu cylfine is calf-ince breas subing and measured by a fuel-ever-servary measured integradently calibrated, as described in appadin 5.

The fuel is introduced into the chamber through a brass adapter which holds the nozzle under investigation. One end of the adapter accommodates the fuel line; the other end screws into a tapped hole in one of the metal walls of the chamber, as indicated in Figure IV. The adapter is held securely in place by means of two lock washers and a nut.

The two sets of five nozzles, ranging in inside diameter from 0.023 to 0.105 inches, are Stainless steel tubing of the type used for hypodermic needles. The word nozzle is used only for convenience, and carries no implications of having converging or diverging sections, as no attempt was made to alter the character of flow at the discharge end of the fuel line other than that dictated by the differences in inside diameter. Each nozzle was silver soldered into the adapter, bent, ground and polished, as deseribed in Appendix D.

Fuel rate is controlled by a globe valve preceding the metering orifice.

The Photographic System: Photographs are taken with a Voightlander 9 x 12 cm. film, f 4.5 pack camera equipped with a 7.5 cm. focal length lens and double extension bellows. The lighting and camera arrangement is shown in Figure III. Light is provided by discharging across a one-half inch stainless steel spark gap a 0.01 microfarad condenser charged to 15,000 volts by a simple half-wave rectifier using 60 cycle

The fast is introduced into the chamber through a brass scatter and a bailer the occals and a investigation. The end of the schubber scoomoletes the fuel line; the other and sorewe into a targed bels in one of the motal walls of the chamber, as indicated in figure IV. The adopter is held secural; is pince by same of two lock washed and a but.

The two sets of five nomilar, ranging in inside diameter from 0.023 to 0.105 innues, are dusiniess steel tubing of the type thed for hypodermic meeties. The word nomils is used only for converging or diverging modiles, an plicetions of having converging or diverging meetions, an at solidation was made to altor the observator of flow at the the differences in inside distributer. Hash notals was wirth able differences in inside distributer. Hash notals was all worldated in topolder, head, ground and polyaned, as de-

Fuel rate to controlled by a gloue valve preceding the

<u>The Protocratitie Setten</u>: Mutocratite are teacher with a modeled wher 3 x 12 m. film, f 5.3 pack coners woulged with * 7.5 mm. ford least land and counte matanaion ballows. The listifies and comers arrangement is shown in Figure 111. Light is provided or discontrains serons a fam-balf losh state. Light is provided or discontrains serons a fam-balf losh state.

C



LEGEND FOR FIGURES II AND III

9

A .	Air orifice meter.		
B .	Power pack.		
C.	Thermometer well.		
D.	Air by-pass valve.		
E.	Fuel control valve.		
F.	Fuel orifice meter.		
G.	Air trap.		
H.	Fuel pump by-pass valve and	pressure	gage.
I.	Fuel pump.		
J.	Fuel Reservoir.		
K.	Diffuser.		
L.	Air rate manometer.		
M.	Fuel rate manometer.		
N.	Diffuser and Orifice Static	pressure	manometer.
0.	Optical Bench and adjusting	jacks.	
P.	Exit Duct.		
Q.	Variac.		
R.	Air Control valve.		
S.	Chamber.		
T.	Spark Gap.		
U.	Condenser.		
V.	Condensing lenses.		
W.	Mirror.		
X.	Fuel adapter and nozzle.		
V	Light-proof cloth		

Z. Camera.

III CAA II SUUDIE SOE ORADAJ

	, TEJED COLLICO TI	· A
	Power pack.	В.
	Ther ont r ell.	. D
	Air by-pass valve.	D.
	eviev fortrop leur	.E
	Fuel orifice meter.	P
	Air tran.	-D
RAILTA FAFA.	Fuel numb by-pass valve and pre-	H
.0000 0	First print the state of the state	T
	Rual Receivate	T.
	Diffinary	
	ter rate manager	T
	End rete manometer	* 5.4
antomore arma	ruce rate manometer,	•
. 190 SECTION SINCE	and around portran mus tenting	. 11
. G.A	are arreacher and warred rearedo	.0
	DATE DUCE.	.2
	Variade.	• 9
	. SVIEV LOTINOJ TIA	• 71
	Champer.	8.
	Spark Gap.	.1
	Condenser.	. U
	Condensing lenses.	. V
	. Toril	.7
	luel adapter and nonale.	.X.
	Light-proof cloth.	.Y
		100

L. Campra.






115 volt building power. This spark technique and the power pack design is fully described in (14). Previous measurements have shown the spark duration to be less than 0.25 x 10^{-7} seconds.

The light from the spark is directed by two five-inch condensing lenses and a 3×6 inch mirror into the camera through the chamber so that the spray is photographed in semi-silhouette. The light is diffused and centered by manipulation of the lenses.

The camera, spark gap, mirror and lenses are supported on a 24 x 24 inch bench with telescoping legs, which can be adjusted vertically by means of two screw-type jacks so that any part of the chamber may be photographed. The base is bolted rigidly to the floor.

A light-proof cloth cloaks the chamber, nearest condensing lens, and the camera lens so that the camera shutter may be opened, the condenser discharged, and the shutter closed, eliminating the necessity for spark-shutter synchronization.

Experimental Procedure

The oil spray was obtained with three types of fuel injection: downstream flow, perpendicular flow and upstream flow.

For downstream flow, the bent nozzle to be studied was secured in a tapped hole near the top of one metal wall, the chamber assembled and installed. The alignment of the nozzle in the plane of the adapter was checked when the nozzle 115 volt building power. Whis spark technique and the power pack design is fully described in [16]. Frevious manuauraachts have shown the spark duration to be less than 0.25 x 10⁻⁷ accouts.

The light from the spark is directed by two five-inch condensing inness and a 3 x 6 imen mirror into the carers through the chember so that the spray is photographed in memi-milbouette. The light is tirfused and sentered by memipulation of the leases.

The camera, spark gap, mirror and leaves are supported on a 24 x 24 inch bench with telescoping legs, which can be adjusted vertically by means of two acressive type jacks so that any part of the charber may be partographed. The back is folted rigidly to the floor.

A light-groof cloth clocks the church, nearest contenning long, and the contern lens so that the compre shutter may be opened, the condensar discharged, and the shutter closed, elisimating the necessity for spark-shutter synchronigation.

Experimental Procedure

The oil spray was obtained with three types of fuel injection: downstream flow, perpendicular flow and upstream flow.

For downstreem flow, the best sozzle to be studied was secured in a tapped hole sear the top of bas metal well, the chamber assembled and installed. The sligument of the sozzle in the plane of the somptor was abached when the securic and adapter were made (See Appendix D). The alignment in the line of sight of the camera was checked by opening a plugged hole in the opposite metal wall that was closest to the nozzle tip.

For perpendicular flow, disassembly of the chamber was not necessary and the nozzles could be inserted by merely screwing them into a tapped hole near the top until the forward face of the adapter was flush with the milled inner surface of the wall.

For upstream flow, the procedure was the same as for downstream flow except that the metal wall was reversed, top-to-bottom, with the nozzle pointing up into the chamber.

For each type of flow, and for each nozzle run at a specified condition of air and fuel rate, the procedure was as follows:

- Camera and lenses were aligned and the bench was adjusted to proper height.
- The air compressor was started and allowed to build up pressure until stable conditions existed in the surge tank with the by-pass open.
- The fuel pump was started and pressure adjusted
 by internal by-pass to 90 psi.
- The desired air rate was set by adjusting the by-pass.
- 5. The desired fuel rate was set by adjusting the fuel valve.
- 6. The light-proof cloth was adjusted around the chamber.

and adoptor ware more (See Appandiz D). The digment in the line of sight of the camera was decoired by opening a ploated mole in the opposite recal well who was donor to the normals tip.

You margandicalar flow, disammently of the plumeet was not assessed and the normine could be inserted by wordly seresing these face date as find bole near the us; wordli the forware face of the adaptar was fluck with the milled inner surface of the wall.

For sparress fibs, the procedure was the sums at for downstream flow southy that the sets will was reversed, usp-to-sottwa, with the notals pointing up into the shares.

ity acak type of flow, and for each nounle run at a spacified condition of air and fuel rate, the procedure was as follows:

Funder and lances were allened and the banch
 Fund adjusted to proper noight.
 The dir comprehent was started any allowed to
 The lir comprehence weth stable completions

astated in the surge and with the by-pass open. 3. The fuel pump was shirted and pressure adjusted by internal by-pass to 90 psi.

The desired air rate was set by adjusting the of-page.

5. Its dealed that rete was not in allesting the fact relat.

. Ine light-proof sloth was soluted around and

- 7. The shutter was opened, condenser discharged, and shutter closed.
- 8. All manometers were read.

Each nozzle was photographed at a high and a low fuel rate and at a low, intermediate, and high air rate for each fuel rate.

For downstream flow, the spray envelope was photographed at each condition with reflected light from two Super Flood lamps placed 18 inches behind the camera lens and at the maximum angle permitted by the chamber walls. The exposure time was 2-4 seconds at f3.5. The shifter was opened, conformer discharged, and shifter alored.

. ALL maximuters were reat.

inch normie was photographed at a high and a low fuel rete and at a low fuel for anoh . Such shall at rate for anoh . Such fats.

top fore from the strongers and the second strong the second strong and the second strong the stron

The Restory of Conception of States in the second second

RESULTS

The results are presented as a series of plates, Figures V through XXXIII, portraying the spray by two independent photographic techniques. The instantaneous pictures were taken by means of the spark-lighting apparatus while the spray envelopes shown were obtained by time exposure with reflected light.

Each figure represents the full series of nozzles, increasing in size from left to right, at one condition of air rate and fuel rate, and for one type of injection. The number beneath each picture refers to the experimental run listed in Appendix F. The plates are arranged as to type of injection and for each type, they are arranged in order of increasing air rate, and at each air rate, two plates are arranged in order of increasing fuel rate. Pictures of the spray envelope were obtained for downstream flow only because of the generally poor character of the spray for the other types of injection which would make time exposures of little constructive value.

For each individual photograph, a knowledge of what is shown is necessary to permit correct analysis. The magnification is approximately 2.8. The actual magnification can be obtained by measuring the nozzle tip in the photograph and comparing it with the actual diameter for that nozzle given in the appendix. The depth of focus is about 1.5 mm. and the center of focus is on the diameter of the fuel orifice or axis of the spray. The negatives were cut down for mounting but no vital information was lost since the nozzle tip was in the center of

OTJUE TS

the readits are presented as a series of plates, "igures w through millit, partraying the spray of two independent photographic tooiniques. The icatante cour platures were taken by mans of the spare-lighting apparetus while the spray anuniques shown mere obtained of time exponent with reflected light.

Made figure represents the full center of neceler, inevential in alte readent for right, at one condition of air rate and fuel rate, and for one type of injection. The number bound th each distant refere to the experimental run listed in appendix 7. The plates are arranged as to type of injection and for each type, they are arranged in order of injection als rate, and at each air rate, two plates are arranged in order of incortant fuel rate. Flotures of the spray movelope are obtained for some transform only because of the generally poor character of the spray movelope are obtained for towners and the other type of injection which would take time appearers of little constants of the generally

Nor and individual photograph, a knowledge of what is above is necessary to permit correct analysis. The manification is approximately 5.8. The actual magnification can be obtained by measuring the mosale tip in the photograph and comparing it with the actual dismetor for that nossile given in the appendix. The depth of focus is about 1.6 me, and the denter of focus is on the dismetor of the fact for that nossile given in the spray. The magnification of the fact has a senter appendix. The magnification of the fact has been been of the information are lost since the normal signal of the denter of the chamber in all cases and the chamber walls may be reconstructed knowing the magnification. In some photos, the chamber wall shows as a black strip to the right or left and was indicated where impingement was present. The nozzle tip is shown in the prints on the top for down stream, from the left for perpendicular injection and from below for upstream injection. Out-of-focus imperfections are unavoidable, particularly at low air velocities due to fuel impingement on the optical flats.

Fuel rates of 1.08 and 3.34 grams per second and air rates of 40, 80, and 180 grams per second were chosen bearing in mind that the change in air rate is logarithmic in character and that the intermediate range might be expected to prove of greatest interest. The metric system was used here to keep numbers large enough to handle, and weight rates were used because the metering was more feasible in terms of weight.

the chamber is all cases and the chamber walls may be resonatructed knowing the magnification. In some motes, the chamber will shows as a black strip to the right or left and was indicated simte inpingement was present. The namely the is shown in the prints on the top for down stream, from the left for perpendicular injection and from being for mattering at dut-of-form imperieshes are unavoidable, particularly at her air velocities for to fuel impingement on the optical flats.

Fuel rates of 1.00 and 3.34 gruns per second and als rates of 40, 80, and 100 grans per secone were chosen could be that that the obanne in air rate is legarithmic in cheracies and that the interaction of a life is a second to prove at provot interact. The metric system was need into to have agronged into eaouth to handle, and weight rates are and here to have agronged the second to handle, and weight rates are and here to have a the actual and the second in terms of weight rates are also be and and the second to handle, and weight rates are and here to be a second to here the second to handle, and weight rates are also be a second of the actual and the second in terms of weight.

DISCUSSION OF RESULTS

There is a question as to the value of the photographs regarding their reproducibility. In all, 228 runs were made. mostly all concerned with downstream flow, and the best negative chosen for each condition. For the instantaneous pictures, no marked misrepresentation was noted in any case. However, to substantiate their value further, three pictures were taken in rapid succession for each nozzle at an air and fuel rate known to give acceptable atomization. Two of the three taken are shown in Figures XVII, XVIII, XIX, XX and XXI. They may be compared with the third, which was inserted in its proper place in the series, Figure X. The Spray Envelopes, however, are not so easily validated. The density of the negative is a function of the density and reflectability of the spray itself. At the spray cone edges, where drops are small and density is light, definition is not good in the negative and it is bettered in developing only at the expense of losing some of the edge and thus not exactly portraying the true cone. In printing negatives of varying density simultaneously, the loss of definition is constant, but the ones of lighter density suffer more in order to bring out the heavier ones. This is most readily illustrated in the pictures at low air velocities where spray density was quite light.

Two important features of the instantaneous photographs must be noted in order to understand what is actually recorded.

ET AUGUST TO DUISSON IN

Indemandance eas he sulus ads of the noiseour a ci steal reservice their reproducibility. In all, 226 runs warm ands, mostly all concerned with downstress flow, and the best negative shopen for and condition. For the instantaneous miclurve, no worked of second states to the was noted in any oste. However, to anberantiate their value further, three plotores ware taken in reply succession for such possis at an air and tool rate known to wive accepted a cont; tion. "wo of the then all of a star abown in Dignitas IVII, IVIII, MIL, IN and MIL. They say be despared with the third, which was inserted in its proper place in the series, Tistre T. The Porty Davelopen, however, are not so ausily validated. The density of the necetive to a function of the density and reflect mility of the spray itself. It the spray come schos, where arons one small and density is light, declation is not good in the nagative and it is befored in developing only at the expense of laning some of the addre and thus not exactly portenying the true cone. In printing sections of varying density similaneously, the icas of definition is constant, but the ones of lighter density suffer sore is order to oving out the provier case. This is anilipolar the woll de constain and hi hedresailly things such where spray density was quite light.

The important fastared of the instantaneous bistographs subt be noted in order to underwhend what is security recorded. First, because the picture was not truly taken "instantaneously", though the exposure time is very small, there will be a greater ratio of large drops to small drops than actually exists, due to the fact that the smallest drops are accelerated faster and cannot be "stopped". Secondly, the exposure time limits the size of drop that can be "stopped". In order for a drop to be "stopped", it must not travel its own diameter during the time of exposure. Knowing the time of exposure to be less than 0.025 microseconds, it may be calculated that the spark will "stop" drops of size greater than 5 microns for an air velocity of 660 feet per second and 2 microns for an air velocity of 275 feet per second, and probably "stops" even smaller ones.

The nature of the equipment limits the accuracy with which conditions may be reproduced. Consequently, a representative value of fuel rate and air velocity was chosen for labelling each figure. This is an acceptable procedure for a qualitative investigation. The actual data and calculations for each run are given in Appendix F for reference.

Eastman Plus X Fine Grain film packs were used throughout, with an aperture of f 3.5. Negatives through Run 80 were developed in Eastman Microdol Developer, over-developed 100% to give maximum contrast and good grain for future enlargement. It was then realized that a more compact presentation of results would be desirable, obviating the use of enlargements, so all subsequent negatives were given normal development in Eastman D-11 Developer. It was also physically impossible to duplicate exactly the light intensity on the film when the lens

Thret, because the distinge mer not truly taken "instantaneously", though the exponents that is very small, there will be a greener ratio of large drops to small drops that sevenily exists, due to the face that the smallest drops are conferented frater and cannot be "dopped". Secondly, the seponers the limits the size of drop that can be "stopped". In order for a drop to be "stopped", it must not travel its own dimensar during the time of exposure. Invelous the time of exponents to spark will "stop" drops of all greater than 5 microns for an a park will "stop" drops of all greater than 5 microns for an air velocity of bob feet per second and 2 microns for an walceivy of 275 feet per second, and probably "stops" even smaller ones.

The mature of the equipment limits the scouracy with which conditions by teproduced. Concequently, a representative value of twol mits and sir elocity as observe for lebelling each finure. This is an acceptable proceedary for each run investigation. The scient date and culculations for each run

Mastaan Flos T Fine Grain film packs mure used throughout, with an aperture of f J.5. Megutives through Run NO were developed in Fratana Microdel Leveloper, over-developed 100% to give serieus controst and good grain for future colsrgement. It was then remlied that a more compact presentation of results would be desirable, obvicting the use of colorgannits, so all methement negatives were given normal development in mastas D-li Developer. It was also paymonity impossible to duplicate eractly the light intensity on the film when the lens arrangement was disturbed. For these reasons, the density of the instantaneous pictures was not constant in all negatives leading to non-uniform results in printing. Insufficient time prevented using a more suitable but more tedious technique of printing each negative to best results, stripping the prints in a plate and rephotographing the plate.

The following exceptions from standard conditions are noted:

- 1. Fig. VI Run 79 Nozzle tip just out of picture.
- Fig. XXIII Run 144 Fuel rate 2.50 gms/sec. to prevent impingement.
- Fig. XXV Run 149 Fuel rate 3.05 gms/sec. to prevent impingement.
- Fig. XXIX Run 91 Tip of spray 4.00 inches above nozzle tip.
- Fig. XXIX Run 99 Tip of spray 3.64 inches above nozzle tip.

It is apparent from a comparison of the Spray Envelope pictures with the instantaneous pictures, that the time exposure technique is incapable of telling the true story of atomization. In many instances, the time exposures give an illusion of a good mist formation, but the instantaneous pictures show incomplete atomization, often with fuel mass concentrations. This is easily explained by the fact that the negative receives many traces during exposure instead of recording a physical mass position. When the fuel becomes well atomized, the dispersion of drops causes a diffusion of the arrangement was disturbed. For these reards, the density of the instantaneous pictures was not constant in all meratives leading to non-unifers results in primiting. Insufficient the prevented using a nore suitable but more taking nique of priming and negative to best results, stripping the primits in a plate and results the place.

The following exceptions from standard constitions are noted:

- Tig. VI Mar 79 Houris the just out of picture.
 Z. Vig. IXEEI Fun LA: Fuel rate 2.50 gms/ams. to provent impiagement.
 - 3. Mig. KIV Mun 14.4 Mul rute 3.05 ma/sen. to
- A. Fig. MIX Suu 91 Tip of spray 4.00 igolas
- 5. Min. 111 Run 99 The of spray 3.64 inches
- ADDTS BOTLE LID.

It is apparent from a comparison of the Spray Envelope pictures with the instantenerse pictures, that the time arponure biominicie is inseptible of celling the true story of stankestion. In many instances, the time exposures give on illusion of a good what formation, but the instanteneous pictares and incomplete stanission, orten with fuel mass contares and incomplete stanission, orten with fuel mass concontractions. This is easily explained by the fact that the negative receives any truces during exposure instant of renording shybical mass position. When the fact becomes wall atomized, the dispersion of drops censes a diffusion of the

light with the result that a fading is evident in pictures where atomization is fine and drops are well dispersed. This fading effect is small in comparison with the differences encountered in developing and printing, and, in fact. is largely dependent on them. Also, this fading effect is not uniform with increasing air or fuel rate because the film cannot differentiate between a concentration of small droplets and a solid mass concentration during long exposure. Another feature of the time exposure is that it represents the summation of positions occupied by the spray during the exposure with the consequence that spray fluctuations cause a false impression of spray volume and cone angle on the negative. It must be concluded that photographing the spray envelope with time exposure does not offer a picture valid enough for even the roughest qualitative analysis of the spray characteristics.

On the other hand, the instantaneous photographs lend themselves well to detailed analysis provided the air stream has not been disturbed sufficiently to disrupt formation of a good spray cone. An inspection of the figures indicates that Downstream flow lends itself best to analysis because of least interference from the nozzle. Only three spray characteristics can be studied with any degree of certainty: Drop Size, Dispersion, and Uniformity.

<u>DROP SIZE</u>: For velocities sufficient to produce acceptable atomization, no effect can be noted for increasing fuel rate or increasing orifice diameter at one air rate, and for

antutoid at the live of mains a fand simple as as a shelf more - and is in the and droup are will depended. This must offeet to see i to conversion with the differance according in developing and printing, and, in faut, i tooll albe idt oel io i de base ylogui i milt and assume and include the solar toni die motion don -conh lime to multari hogeo a submid sitista to Tib tonano lets and molife mass concern ration during tone exposure. antine faultre of the time exposure is that it reveales the summion of conttion occurs of by the stray will be and a solution of a state of the second of a state of the second s false inpression of spray volume and come angle on the onestive. It was be concluded that abtopraching the array envilore with time exponent does not offer a picture will stough for even the routhest suchtasive analysis of the avery charac-. . of first

In the other hand, the instantaneous theterraphe land thesealves well to detailed enalysis provided the sir stream has not been distarbed sufficiently to disrupt formation of a good sorey come. In inspection of the figures indicates that lownstream flow lette theolf best to analysis because of loost interference from the normale. Only three sprey thereforieties out he studied with any degree of certainty: Drop him, Disporator, and Raitoraty.

able atomication, no effect can be noted for increasing Fuel rate or increasing orifice diameter at one sir rate, and for

one type of flow. At low air velocities, the velocity of the fuel is a greater percentage of the air velocity than at high, with the consequence that the change in fuel velocity imposed by change of orifice diameter or increase of fuel rate has a more noticeable effect on the relative velocity of fuel to air, and a decrease in drop size is observed for increased nozzle size and decreased fuel rate. At high air velocities, the change in relative fuel to air velocity with change in fuel rate and nozzle size is so small that the effect on drop size is not noticeable. For one fuel rate and one nozzle size, increased air velocity has a marked effect on the decrease in drop size. As the type of injection is shifted from downstream. through perpendicular to upstream, the relative velocity of fuel to air becomes greater, other variables constant, with the result that drop size decreases. The effect is more noticeable with large nozzles than with small, which leads to the conclusion that the influence of type of injection on drop size as photographed is exaggerated by the fact that there is more physical interference from the large nozzle, causing more fuel to be pulled out of the spray along the nozzle. This amounts to an appreciable decrease in fuel being atomized.

<u>DISPERSION</u>: Dispersion decreases with an increase in air velocity for all types of injection, due to increased stability and increased resistance to distortion of streamlineses air velocity increases. It decreases with increased fuel rate for downstream flow, probably because of the greater stability of a more rapidly moving liquid column at the same air velocity. For perpendicular injection, the dispersion increases because

At low mir velocitian, the velocity of the one type of flow. runi is meaning perovision of the air velocity that at here, with the consetuence that the change in full relating throught by change of orifics dispeter of intrase of fuel rate has a nor anti-sectie effect on the relative velocity of real to air, and a decrease in arou airs is observed for insteamed gentioner ale she stat feel tate. At high air relooities, the change in relation fact to air valuelly with change in their rate and markle size is an amali that is affect on Aren size is not acticocolic. For and fuel rete and one months size, increased air velocity has a marked offered on and two feareases in drop size. As the type of injestion is shifted from dometress, shrough perpendicular to upstran, the rolative velocity of ful to alk recorded grouter, other variables countrat, with the result that aron size Secrement. The sizes to motioesble with large powsies then with small, which leads to the consisaion that the influence of type of in estion on drap size as photographed is enurgerated by the fuct that flats is more physiccal interference from the large nemale, canaling care fash to be pulled out of the name slong the ontain. This should be the appreciate decrease in fant bring stationd.

<u>DI CTABLICA</u>: Dispersion degresses with an increase in air velegity for all types of injectics, due to increased avability and increased resistance to distortion of atreamlinesses air velocity increases. It degresses with increased fuel rete for downstrong flow, probably because of the grester stability. of a more repidly moving liquid solume at the same air velocity.

the increase in fuel velocity, with increase in fuel rate and decrease in orifice size, serves only to shoot the fuel column farther across the air stream, affording more area upon which the air can work. For upstream injection, the increased fuel velocity shoots the fuel column farther upstream. The farther the fuel column goes, the more it loses its stability and becomes dispersed, thus presenting a greater area of impact to the onrushing air.

Increasing nozzle size increases dispersion for all types of injection, though the effect is not well illustrated for perpendicular injection. As nozzle size increases, the perimeter of the fuel column increases, providing more surface on which the air may act. For upstream injection, larger nozzles provide a greater area of impact to the velocity front.

UNIFORMITY: In atomization, the limit of drop size is one infinitely small, and this is approached asymptotically, with the force required to obtain it increasing in a like manner, i.e., an infinite shearing force being required to give an infinitely small drop size. When the atomizing force increases, fewer large drops will occur in relation to the number of small. Thus, with a general decrease in mean drop size, the decrease is at the expense of large drops being broken up. This amounts to saying that the variation of drop size from the mean is less with a decrease in mean drop size, and this variation is the definition of uniformity. This reasoning is confirmed by the photographs. Increased air rate gives better uniformity for all types of injection. Increased fuel rate

the increase is fuel velocity, is h increase in fuel rate and decrease in orifice size, werves only to shoot the rul column fartues screet the siz distant, affording and seren upon which the siz can work. For anotreas hjection, the increased fuel velocity shoots the fuel column tarbles upstream. The first or the fuel column fore, the sore it ices its scability and becomes dispersed, then presenting a treater area of ing of to the emushing wir.

Increasing notice size increases dispersion for all types of injection, though the effect is not wall illustrated for perpendicular injection. Is notified increases, the portmater of the fuel column increases, providing erre surface on which the dir any act. For upstress injection, larger notales provide a greater area of impact to the velocity front.

<u>WHITOMALER</u>: In standartion, the limit of arop size is one infinitely scall, and this is approached approximity, with the force required to obtain it increasing in a like manner, i.e., an infinite snearing force being required to the an infinitely small drop size. When the stokizing force the of scall. Thus, with a reserve being rolein to the nuber of scall. Thus, with a reserve being around the to the nuthe decrease is at the expanse of intge around hells to the decrease is at the expanse of intge around hells, and with the decrease is at the expanse of intge around hells, and with the decrease is at the expanse of intge around hells, and with the decrease is at the expanse of intge around hells, and with the anomate to soften that the vertection of arop aims from waringtion is the definition of antionality. This remaining in confirmed by the decrease of injection. Increased all rates given here and fraction is the definition of antionality. This remaining in

decreases the uniformity. The effect of nozzle size is not marked except in upstream flow where the same illusion occurs as explained in the discussion of drop size regarding the amount of fuel drawn out of the column by the larger nozzle.

In line with these observations, an attempt was made to apply the formulation of Nukiyama and Tanisawa (15) in order to predict the effect of the variables on Mean Drop Size. These investigators tested a small nozzle, using liquid fuels injected into the throat of a venturi atomizer with air as the atomizing agent. From their tests they determined that:

$$D_{m} = \frac{585\sqrt{\sigma}}{\sqrt{\sqrt{\rho}}} + 597 \left(\frac{\mu}{\sqrt{\rho\sigma}}\right)^{0.45} \left(\frac{1000}{\varphi_{a}}\right)^{1.5}$$
[1]

where:

D_m = Volume-surface mean diameter in microns. V = Relative velocity of air to liquid - meters/second. O = Surface tension of liquid - dynes/cm. P = Density of liquid - grams/cc. M = Viscosity of liquid - dynes-sec./sq.cm. Q = Volume rate of liquid - cc./sec. Q = Volume rate of air - cc./sec.

They found no effect on drop size from changing the size of venturi, within small limits. Lewis and Edwards (17) have also shown that the equation gives good results for venturis of any size. In particular, they tested perpendicular injection from a small nozzle into the throat of a relatively large venturi. This constitutes point injection into a reasonably uniform air stream, and they found that equation (1) still held. decreases the unifordity. The wifest of adjuste that is not warked encopt in uppersus then where the nume linedon occurs as explained in the disculate of the obtain of the prime respecting the amount of the fraction of the column by the integer undels. The integer undels to the formulation of the column by the integer undels to the solute of the column by the integer undels to the solute of the column by the integer undels to the solute of the column by the integer undels to the solute of the column by the integer undels to the solute of the column by the integer undels to the solute of the column by the integer undels to the solute of the column by the integer undels to the solute of the sol

: 970

De - Volume-surfice sein divestor 19 sierona.

V = Meletive velocity of air to liquid - catera/awoond. C = surface tension of liquid - orace/em. P = Tensity of liquid - grasss/co. $Q_t = \text{viscosity of liquid - dynam-moo./mq.tm.}$ $Q_t = \text{Volume rate of liquid - ct./mac.}$

These found no effect on those size from changing the size of venturi, within mult limits. Lewis and Mawards (17) nere also abown took the sometion gives good results for venturis of any size. In particular, they bested perpendicular injection from a small moscle into the throat of a relatively large venturi. This constitutes point injection into a relatively introim sir stroam, and they found that sources (1) attil noid.

19

TABLE I

For Diesel 011: $Dm = \frac{3220}{V} + 41.5 (\frac{1000}{V} + 0.1.5)$

14.6 16.4 DB "D" 10000f 1.5 Dimensions given in Appendix R. 47 234 .23 .23 18878 224724 224724 B 27400 26400 26400 51900 53400 128500 130000 53400 53400 53500 55800 54600 54800 54800 313000 28500 3 .87 1.30 .95 L.31 23 14.6 16.2 3220 85.5 DOWNSTREAM 39 194.1 201.0 188.0 203.3 PERPENDIC 199.0 199.0 199.0 220.0 227.0 from #1 to #5. 6. 7160022022007200 15.1 5.1 4.8 15.0 TY Nozzle size decreases Va 201 201 204 204 216 196 Nozzle No. Run No. 152 865 161 10018021 20018022 145 Note: XXXIII XXXXIII ILVXX F16. IIIA IA IA LIX LIX LIX LIX

a Da
Vbb TOTX
111
ST V St
Dimensiona
. 21
04
T.
TLOW
gettarges
BIZO
1 7 0

:0101

0.25			ß
		and the second division of the local divisio	(
.53	6t-0 • •	ARTER SEAL SANDAR JA	TOODET
130000	T58200 T50000	131300 131300 131000 131000 20100 20100 20100 20100 20100 20100 20100 20100 20100 20100	0 (T) J.
10.4	0E.1	MOMOMMEDELLE OFACM	2 (<u>T</u> bot
14.8	1.0.	R	<u>3550</u> 0 + 17.
550.0 550.0	0.001	20102 10110 1000000	4 1 1 11 2555
0.01	1. cr	715 0 150 55 0 0 0 0 0 0 0 0 0 0 0 0 0 0	T : LIC
505 510	1733 Jàe	1055 1055 1055 1055 1055 1055 1055 1055	AT
נה נה	NINI	HATANDONEATNOTANDA	TOT .ev
02.02	140 GAT	1772270005000770770 750000000000000000000	.011 JUN
II SUC	IV25	TAX ANY ANY ANY ANY ANY ANY ANY ANY ANY ANY	

I AJEAT

Table I includes the calculations for sufficient pertinent runs to provide a basis for correlation.

The conclusions drawn from this analysis generally agree with those observed in the photographs. The variables investigated in this thesis affect the formula in the following ways: (a) Increase of nozzle size decreases the fuel velocity, thus affecting the relative velocity V: (b) Increased fuel rate increases the fuel velocity, thus affecting relative velocity, and increasing Qf; (c) Increased air rate increases the relative velocity V and increases Qa; (d) Type of injection affects relative velocity V - fuel and air velocities being subtracted for downstream, added for upstream, and fuel velocity being neglected for perpendicular injection. The general observation first made is that the change in mean drop size due to nozzle size, fuel rate and type of injection at air velocities sufficient to give acceptable atomization is of the same order of magnitude as the experimental error involved in this thesis. The predicted effect on mean drop size from equation (1) is most notable at low air velocity for nozzle size and fuel rate, and most apparent for type of injection at high fuel rates, as previously explained. The change in drop diameter at higher air velocities is sufficiently small, with all the studied variables except air velocity itself, to escape detection by macroscopic examination of the photographs. The variables are all significant at low air velocities, but this does not contribute much to the knowledge of atomization as regards its application to the proper combustion of fuels, since atomization is not acceptable. The

Table I includes the celculations for sufficient pertinent runs to provide a bhuls for sofralation.

The conclusions drawn this will an algorithm and solid and solid and the second with those observed in the nio ographe. The verial all and galwollof and at alumo? and teath alands shut at beterit (a) Increase of Lozilo sire desreaments will valon ty, Way :: int manorani (a) : vitalev vi for edi mitseft erit rute increases the fuel sulocity thus are o in relative velocity, and increasing Cf; (c) Increased air rate increases the rolative velocity V and increased (a) "rod of thingand inclose the fact - V ticolev evitel a storia acit being austracted for down trans tor unstanting anied relocity being neglected for perpendicular in estate. SAL none of example out that is that farif not for abana in cool drop size due to murale size, feel rate and when of this with at air velocitie miliciant a cine at a second a second Torne Lataenireque est na edui inage fo rebro emes suit le si is olved in this thesis. The redicted erret on man from size from equation (1) to more possible at low sir velocity for notale give and fael rate, and most append for type of in sotion at hich fuol rates, as previously explained. "ne chance in drop district is reining of the chance of the chance -31 witholaw ile topon and ine biller biller it is it is it is self, to matapa detection by maproscopio equitantion of the photogramha. The variables are all significant at low all velocities, but this cost on the said the mon to the morieder of attained and of all and in the real attained and the proper to bu tion of furin, minor atomination is not adaptable. maria

correlation does show that the formula is correct as to general trend, but its validity has been more comprehensively established by other investigators.

The photographs afford a good comparison of the types of injection and the advantages and disadvantages offered by each. The effects of each on the spray characteristics have been previously explained. A consideration of other features is also feasible. It is seen that streamlining that portion of the nozzle that extends into the air stream is desirable for downstream flow, and a definite necessity for the other types. The flow might be expected to be more asymmetrical for downstream injection than is indicated by the photographs, and probably would be in a less restricted gas duct where streamlines would have more freedom to distort laterally. Nozzle size in the range tested does not seem to be an important factor with downstream flow, but it is quite important with perpendicular and upstream injection. For the perpendicular type, the large nozzles form a definite low pressure area on the downstream side, although the tapered tip did prevent fuel from running back along the nozzle (a condition experienced by Robillard with the blunt tip). In upstream flow, general atomization was very poor because the fuel collected on all exposed portions of the nozzle and was blown off in large droplets. Also, from the combustion standpoint, it does not seem possible that a nozzle would stand up well with the flame enveloping it. In summary, perpendicular and upstream injection do not offer enough advantage in atomi-

correlation dose apor that the formula is correct on to general wrond, out its velicity has been more comprehensively wstablished by other investigatory.

The photographic attore a good competion of the types of interior and the savariages and itssound tages offered in fucu. The offects of anon on the star churateristen features is the factore. It is some bal a reaching meaning the add and appears and all on why to notitor sent is desirable for downers and inc. and a definite wederity age stricel for domastrege in action on is indicated by the anotographic, and reably ould be in a loss restricted And duct ober etremeline would have har treated to dietort laterally. Norshe dise in the page tested does not seen to be an apportant factor it i domeste a lio. but it is with involtant with part and outer and unstream injection. For the parsencicular type, the large nousles form a definite Low pressure area on the desastream side, sithough the tapared -nos a) sizzou out mole asso gaimar nort four two yours bib wit -go al . (all sable out asks bindlight of beominers molifie strong flor, conord, todiation was mary more because the fue onlicted on all exposed portions of the norship and was blown off in large tropiess. Al. o. To the combustion would out have bloom winner a fait aid take more that sould bi , think well with the task enveloping it. In summary, perpendents -lests at system to day of the inough drawtage in a test

zation to counterbalance the inherent disadvantages due to interference of the nozzle. Perpendicular injection does offer the easiest means of nozzle replacement, however.

The instantaneous photographs, particularly those of downstream flow, illustrate nicely the mechanism of atomization and are in good agreement with the theories expounded by Castleman (8), Rayleigh (10), and Haenlein (18), discussed in Appendix "B". It is not the purpose of this thesis to elaborate on this agreement, but attention should be called to certain points pertaining to the effect of nozzle size.

At the same fuel rate and air rate, atomization starts sooner but takes longer for completion with the large nozzle than with the small. It starts sooner because of the greater relative velocity, and takes longer for completion because of the greater stability of a fuel column of greater diameter. The fuel column from the small nozzle shatters as soon as it starts to atomize. These facts are more evident at the lowest air velocity. At high air velocities, the effect is small but still noticeable.

The mechanics of atomization are illustrated by observing the principal action of surface tension at low air velocities and the combined action of surface tension and ligament formation by shearing action at high air velocities. No evidence of the action of turbulence in the fuel column can be observed because the highest Reynolds' Number encountered was 220.

Other observations made agree with the literature in that atomization occurs closer to the orifice with increasing air

zation to usasterbalance the inherent disadventages due to interforence of the neezle. Fergendicular islestion does offer the ausiest seame of notale replicement, however.

The instantaneous motographs, particularly house of downstreem flow, likestrate aloady the mechanism of atmization and are in good estrement with the theories aryounded by Gastleman (8), Royleigh (10), and Beanisin (18), Alsoursed in Appendix "B". It is not the rerested of this basels to slaborate on this schronout, but attentic about the slice to certain points parteining to the affect of actual size.

At the same fuel rate and dir rate, storigation starts sooner but takes lower for completion with the large neuris then with the small. It starts roomer breaks of the greater relative velocity, and takes longer for completion because of the greater stability of a fuel column of greater discreter. The fuel column from the small workle distance with the lowest starts to stability. At birn field are not starter as soon as it air velocity. At birn dir velocities, the street is small but still notice bir.

The modulation of aromination are illumbrated by observing the principal action of surface considents iow wir velocities and the complete estion of surface tempion and lightent formation by absuring estion at high air velocities. To avidence of the action of turbelence in the fuel column can be observed because the highest Romber succurtered was 220.

Other observations and e oftee with the literature in that atomization occurs closur to the orifics with instanting air velocity, and tend to agree with Scheubel and Sauter (1) in that ligaments cannot be observed above relative velocities of 10,000 - 12,000 cm/sec. It is further noted that atomization occurs closer to the orifice, with increasing orifice size. velocity, and tend to apres with Schwubel and Samter (1) in thes illements common be observed apove relative velocition of 10,000 - 12,000 cm/sec. It is farther muted that alonization occurs closer to the orifice, with increasing orifice size.

Constant of the

CONCLUSIONS

From the observations and discussion it may be concluded that:

- Spark photography affords a good representation of the degree of atomization and general spray characteristics, and that the results so obtained are reproducible.
- Correct analysis of the atomization of a fluid cannot be made from relatively long exposure pictures of the spray.
- 3. With increasing orifice diameter:
 - (a) No appreciable effect is noted on drop size at air velocities sufficient to give acceptable atomization. At low air velocities, drop size decreases.
 - (b) Dispersion increases.
 - (c) Uniformity not materially affected.
- 4. With increased air velocity:
 - (a) Drop size decreases.
 - (b) Dispersion decreases.
 - (c) Uniformity increases.
- 5. With increased fuel rate:
 - (a) No appreciable effect on drop size at air velocities sufficient to give acceptable atomization. At low air velocities, drop size increases.

DINGLOWSEDING

From the event tions all discussion it may be due-

- 1. Opark photography affords a good representation of the degrae of atomization and general apray observativition, and that the results so obtained are reproducible.
- Correct sealysis of the statistion of a fluid consut be adde from reletively long exposure picturge of the spray.
 - 3. At increasing or fice diameter:
- (.) > a preciable effect is noted on drop also at in elocities sallicien, to five scoptble acomization. It los air velosities, drop at a cerrore.
 - (a) Dispersion increases.
 - (c) White mity not meterially affected.
 - 4. itu innrease ir vioci :
 - (u) Trop alse coreaes.
 - (a) Eleptrelon decreases.
 - (c) Uniformity inorecaes.
 - 5. With instaused funl rate:
 - (a) so apprediable affast on drop size at air velocition sufficient to give acceptable effection. At low air velocition, drop also increases.
- (b) Dispersion decreases for downstream injection, and increases for perpendicular and upstream injection.
- (c) Uniformity decreases.
- 6. For type of injection:
 - (a) Spray characteristics are not materially affected.
 - (b) Perpendicular and upstream injection have serious nozzle design problems, with large nozzles showing the greatest disadvantages.
- Within the limits tested, air velocity was the only variable that materially affected the fineness of atomization.
- 8. From all considerations; downstream injection with large nozzles offers most to proper and complete atomization.

- (b) Edimension commenses for downwirean injection, and increases for perpendicular and upatroam injection.
 - (a) Unitority decreases.
 - 6. ser bue of intertion:
 - (a) Sprog characteristics are not an erially with the second states and the second states are not second state
- (b) Perpendicular and upstream injection have serious concle design problems, with large normal showing the greatest disadvantarow.
- 7. Ithin the limits tested, sir velocity was the only variable that metally affected the finances of ecclesization.
- 8. Free 11 considerations; nownstream injection with large scales offers and to proper and complete statistics.

RECOMMENDATIONS

- An attempt be made with this apparatus to atomize Bunker "C" fuel oil under carefully controlled conditions to determine the character of atomization and what air velocities are necessary for acceptable spray formation.
- 2. An investigation into the effect of orifice perimeter/ area ratio on atomization might prove of value.
- 3. With the chamber redesigned to permit good side lighting, and using a much shorter exposure time and smaller lens aperture, it might be possible to correlate quantitatively cone angle with nozzle size and other variables.
- 4. The effect of preheat on highly viscous fuels could be investigated on this equipment.
- 5. Specific nozzle designs could be analyzed with this apparatus. This is particularly true for attempts to improve perpendicular injection.
- 6. If the apparatus is to be used for further investigation, the chamber assembly could be further improved to prevent air leaks by assuring positive contact between glass and metal walls. This could be done by drilling countersunk holes in the glass every inch along its length next to both sides and using fine thread screws into tapped holes in the metal walls. A stop valve should also be placed in the exhaust line before it joins the by-pass to permit removal of the chamber without shutting off the air compressor. Also, a simple but efficient separator should be installed on the exhaust line if heavy liquids are to be used.

REALIZED MALE AND A

- An attempt be sede with this approximation to stong a Runker "C" fuel all under carefully controlled conditions to determine the operation of foulation and what air velocition are represently for scorptable spray formation.
 - 2. In investigation into the street of writics periator/ area ratio on nomination wight prove of value.
- 3. With the observer redesigned to parmit gues also lighting, and using a coor morter exposure time and mailer lens and using a coor morter exposure time and mailer lens tively cone angle with source size and other variables.
 - 4. The effect of prehet on highly victors fuels could be investigated on this equipment.
 - 5. Specific norale designs could be easigned with this upperatus. This is particularly true for attangue to inprove perpendicular injection.
- 6. If the apparatur is to be used for further improved to provent the chamber assembly oculd be further improved to provent air leaks by assuring positive contect between glass and metal wells. This sould be done by drilling countermuck holes in the glass every inch along its length next to in the metal wells. A stop valve should also be placed in the antal wells. A stop valve should also be placed pressors of the observe without shutting off the sir compressor. Also, a simple but efficient separator abould be installed on the subarst line if acay jugaths are to be used.

DOWNSTREAM INJECTION

- And the second s
- And have been been balled an element that the balled with the second sec
 - MO ITTHE LAIL MANOR WOOD
 - No. In which we have a set of the second of
- and being been a better and the bar the set of the best being a set by the set of the se
- And we are and and the second back when a second many many and been areas and and the second back we have an area and and the the second which when there with a first the second the second the second second back back of a second the second the second second second back second is and all the second second second the second second second second second second second second the second the second second second second second the second second second the second second second second second the second second second second the second second second second second the second second second second the second second second second second the second second second second the second second second second the second second second second second the second second second second to the second second second second second the second second second second to the second the second second second second second second second second the second second second second to the second the second second















n2















. .







-



.






PERSONAL AND A THE REPORT ON



*

.

























*













APPENDIX A

SYMBOLS

AC	-	Chamber area - in. ²
AI	-	Nozzle area, inside - in. ²
đr	-	Density of fuel - 0.83 gms./cm. ³
Di	-	Nozzle diameter, inside - in.
Do	-	Nozzle diameter, outside - in.
∆ Pa	-148	Differential air pressure - in. hg.
Pc	-	Chamber pressure - in. hg., gage.
Pca	-	Chamber pressure - psia.
Pð	-	Diffuser outlet pressure - in. hg., gage.
Pr	-	Fuel pump pressure - psi.
△Pf	-	Differential fuel pressure - in. hg.
Po	8	Static pressure downstream of air nozzle - in. hg., gage.
Pòa	-	Static pressure downstream of air nozzle - psia.
R	-	Gas constant for air - 53.34 ft. 1b./1b. Fabs.
Re	-	Reynolds' number.
ta	-	Air temperature before diffuser - ^O F.
Ta		Air temperature before diffuser - ^O Fabs.
Va	-	Air velocity in chamber - ft./sec.
Vſ	-	Fuel velocity at nozzle tip - ft./sec.
Wa	-	Air weight rate - 1b/min. or gms./sec., as indicated.
Wf	-	Fuel weight rate - gms./sec.

A TI PLITA

+ 34

.0303 ...

hg,

. 70

. 00 . .

.

. BCA 70

norsie - in. hg.,

./ sec., as indicated.

norkie - pais.

15./1b. "Talks.

<u>E.IOED</u>	600
- Chamber eres - in. 2	0.0
- lozzle area, insi - in.	Ł
- Tensity of fool - 0.03 gas./cm.	2.0
- No zle di str, luide - in.	EL
- mozzle diameter, autaide - in.	DO
- Differential sir pressure - in.	AP
- Obamber pressure - in. bc., sage	Po
a - Chumber presente - pain.	Pe
- Diffuser outlet presente - in. h	Þ?
- lusi pump preusure - pri.	21.
- Static pressure domairean of si	01
a no meetic pressure domastreem of al	p/I
- Gas constant for air - 53.34 ft.	15.
- Reynolds' number.	84
- Air temperature before diffuser	33
- Air temperature bofore diffuser	æ
- Air velocity in chamber - 12./30	NT.
- The velocity at norsis tip - f	ev.
a - Alr woight rate - highlin. or go	10
r - Juei weight retu - gas./sec.	W

APPENDIX B

SUPPLEMENTARY INTROPUCTION

The atomization of liquids is important for many uses, such as spraying insecticides and paints, laying military smoke screens, and in drying and evaporation operations; but perhaps the most important use is in fuel burning power devices. In the latter it is essential that the fuel be finely atomized to permit intimate mixing of the fuel with as great a surface/volume ratio as possible, and that the fuel be mixed as rapidly as possible with the proper amount of air for combustion.

The characteristics nost often used to describe a spray are:

- Prop size diameter of the individual particles in the spray.
- 2. Uniformity deviation of the drop size from the mean.
- Intensity weight rate of flow of fluid per steradian.
- Dispersion Ratio of spray volume to liquid volume.
- Distribution Ratio of weight of air to fuel at any point in the spray.
- 6. Penetration Farthest distance from the orifice along the axis of the spray reached by the spray.

S ALTRES D

MOTOR DO. I TO TAME FROM

The staminution of licuids is important for sany uses, soot as arreging insoor oldes and paints, laging allitary anoke servers, and in trying in evaporation spendelone; out partning the bort in Ptant was in in thel burning power devices. In the latent if is passantial that the fael be finaly stoniged to persit invite sitter of the fuel with as grant a urf co/volume a fur a road ole no that the their be is a spicit a point of the proper would be at . 101 COC DUSA 101

the state of the second party land and the second s

seioling Isphiwine: wir to televisio - orig gout -. [

. T. MEGH BALL ...

matrorates - devieties of the drop size from the -AMERICAN AND THE REAL POINT AND ADDRESS OF THE POINT AMERICAN × 11.0-010

Intensity - weight into of flow of rimid per . 8 -teretau.

Cispersion - seals of spiny volume to liguid THE REAL PROPERTY AND ADDRESS OF THE PARTY O · DELILOV

many of a side at an or of the side of nistribution - main of watche of all to toutate . C

any polat in the apray.

solitie on mort solutely sentire - nois-rimant .0 along the write of the spray reached by the spray.

- Penetration rate Velocity of the spray tip along its axis.
- Cone angle The total plane angle between the sides of the spray cone at its apex.

The physical variables affecting the spray characteristics are liquid nozzle geometry, spray container geometry, gas duct geometry, liquid characteristics and gas characteristics. Some theoretical considerations have been made to determine the effect of the above variables on the spray, but quantitative relations are lacking for want of sufficient generalized data.

The generally accepted theory of atomization is that proposed by Castleman (8), who assumes that atomization is the same for solid injection and air injection systems, depending only upon the relative velocity of the gas and liquid. His ligament theory is that droplets form as a consequence of small filaments of liquid being drawn out by the action of air on the main stream of the fuel jet. According to an earlier investigation by Rayleigh (10), the stability of a cylinder of liquid being drawn out and decreasing in diameter for any reason whatsoever decreases as the length of the cylinder is increased in comparison to the diameter of the cylinder. At the point where the length/diameter ratio becomes greater than the circumference of the cylinder, a decided instability is present and the action of surface tension is enough to cause the cylinder of liquid to collapse into droplets. At low air velocities drops are formed directly

- 7. Functration rate Velocity of the spray tip
- B. Cone angle The total plane angle between the oldes of the spray cone at its spox.

The physical variables affecting the energy characteristics are liquid nosale popertry, optor container geometry, gas dues geometry, liquid characteristics and and and and other istics. Some theoretical considerations have been adde to determine the effect of the doove variables on the aproy, but quantifative relations are lacking for what of mufficient someralized data.

The removally accepted theory of worldwiton is that period of service and the paint of the period of the service of th proposed by Cantlanam (8), who secures that close is the the sume for solid in setion and sir injection systems, dompLadram Barry and and the anding only upon the relative velocity of the cas and liveld. 10 sonsureant theory is that droplate for we some succes of with the other states in the state mail filmonts of liquid seing drawn out to the newler of air on the sals stream of the fool jet. According to an arlier investigation by deviate (10), the stability of a sylinder of liquid bains drawn ant and tearesting in diameter Minister of section with the for any reason shataoaver decreases as the leadth of the out to retempth of the source in second to the diameter of the the sharest new wide the second At the point when the long bhidismotor ratio becelin ... and the second second comes preater than the sirousformment the cylinder, a deolded instability is present and the spolen of suffees ten ion where and the summary summary and the state of is shough to cause the splitsdar of Light to collepse into droplate. At low air velocities drops are formed cirectly

from the jet by surface tension alone, at high velocities are formed by the combined action of ligament formation by the air and surface tension, while at high orifice Reynolds' numbers drop formation is further augmented by the turbulence of the liquid stream.

Haenlein (18) found that drop formation occurs by four separate mechanisms. At low relative velocity of liquid and air, the air does not appreciably affect the jet. Here the major factor in drop formation is the liquid surface tension, under the influence of which, rotationally symmetrical disturbances are set up in the column which increase until drops are formed. As air velocity increases, the amplitude of the disturbance increases, due to the high air velocity in the peaks and the low air velocity in the troughs of the liquid column. When the velocity is further increased, the initial disturbances become one-sided due to the augmented influence of the air on the column. The surface tension in this case retards wave formation since it tends to return the liquid column to its original form. At this point, the Castleman effect can be seen in liquids of low viscosity. Filaments are torn from the main stream and small drops are formed. With still further increase of velocity, filaments are formed closer and closer to the nozzle until all that can be seen is a cloud of droplets issuing directly from the orifice.

from the jet by surrect teacton lone, at its velocities are formed by the combined action of ligenesit formation by the air and surrest tendlon, while at his printer heynoids' numbers drop formation is further sugrested by the turbulence of the light streen.

Machiela (13) found that apop formallon occurs in four separate means the selection relative valoater of light and sir, the sir open not appreciably front the jet. More that -ne sector block out at tota int goth at to out tota sion, under the influence of which, routinnelly symmetrical Litau anesteal dolaw maules out at an los of a constantelo drops are formed. as air valocity increases, whe amplitude or the listuriance increases, and to the sir velocity in the pasks and the low all valuality in the trouchs of the ident column. When the velocity is further increased, the intois determines become one-sided due to the sugested al noisnest coultus ant . manico and so the end lo consultat this or a shad at shall notion of ever ship of the tetter eit , talo eint inni fro eri os moulos bindi ent Castlanan affool out be soon in liquide of ion viscosity. Alesats are tora from the main stream and small drops are torestin atthe still further increase of velopity, filenesses and the firms alson of the blance of the possib beriot at out be seen is a cloud of dreplets Letuing firedly from the and and and the second s which an appropriate and intervant for some of the specific dataset and which we will

1.C

The generally accepted theory of atomization, then, is that at low air velocities drops are formed directly from the jet by surface tension alone; at high air velocities drops are formed by the combined action of ligament formation by the air and of surface tension; while at high orifice Reynolds' numbers drop formation is further augmented by the turbulence of the liquid stream. "Le caterally usaevied theor of stemizstion, uten, is that at her air velocities drops are formed dirachly fro the lat by murface travice libre; at high air velocities drops are formed by the occhined action of lignment formetion by the sir and of surface tension; will at high sriftce terminate of the libre tormation is furtaer equemied by the terminate of the libre tormation is furtaer equemied by the

and some the part of the part

APPENDIX C

EQUIPMENT DATA

Compressor:

"Ro-Twin" - Allis Chalmers Manufacturing Co., Milwaukee, Wisconsin.

Delivery - 533 cubic feet per minute at 100 psi gage.

Speed - 700 RPM

Compressor Motor:

General Electric Induction Type.

220 v., 60 cycle, 3 phase, 180 amp., 75 H.P.

860 RPM

Fuel Pump:

"Gerator" gear type.

Fuel Pump motor:

General Electric Split Plase Resistance

1 H.P., 110 v., 60 cycle, 7.5 amp., 1725 RPM.

Camera:

"Voightlander" f 4.5, 9 x 12 cm. film pack.

Lens: Wirgin, f 3.5, 7.5 cm. focal length.

Film: Eastman "Plus-N".

Spark Fower Pack:

- Rectifier Transformer: Sola Gas Tube Transformer, 115 volt input, 15,000 volt output, 825 v.e., 60 ma.
- Filament Transformer: "Thordarsen Multivolt" Electric Manufacturing Company, Chicago. Type 1-11F61, 63 watt.

Condenser: .01 mfd.

0

ATAT TERMITORS

100010001:

- "Ro-Twie" Alio Mulimers Manarsotwelle Co., -
- Telivery 533 oute reat per single at 100 put

opeed - 700 EEN

COMPLEMENT LOSOF:

Conversi Pleetric Induction Type.

220 T., 50 03030, 3 71050, 100 amp., 75 0.2.

BOD BURK

iquoi Davi

"Gers tor" gear trac.

Fuel Tung sokers

Guastaless sould side oftroot laters

(H.F., 110 *., 40 Gyole, 7.5 4 1725 HT.

Cone 100

"Wolgh blan ler' T 4.5, 9 x 12 cm. Tilm pack.

Louis: Tria, 13.5, 7.5 cm. room longth.

. The pitter stands and in fits

HUNTE TOWNY TOURS

Reatifier Transic I and by the Ges up Trans-Iorist, 113 volt Lands, 15,000 volt outhut, 625 ver., 56 ad.

History Transformer: "Doriginou Miltivolt" Heatric Doutering Co pany, Calenco.

. olm IO. : Ineneogoo)
Rectifier Tube: Raytheon RKR-72.

Resistances: 5 megohm in each lead to condenser. Spark Gap: $\frac{1}{2}$ " stainless steel rods with $\frac{1}{2}$ " gap. Rectifier Tube: Raytheon AKR-72. Resistances: 5 magona in each load to condener. Spark Gap: i" stainless steel rods with " gap.

advert and the second s

AND A CONTRACT OF A CONTRACT O

Card and the state of the second

"WIN DOT I VILLAND

APPENDIX D

NOZZLE DATA

Each nozzle consists of a polished stainless steel tube tapered at the end and silver soldered to special brass adapters which screw into one of the metal walls, being held there by a nut and two lock washers, as shown in Figure IV. The other end of the adapters connects to a one-half inch brass fuel line. The nozzles were aligned in the adapters so that when the end of the adapter was flush with the metal wall, the nozzle tip was in the center of the chamber crosssection, and in the case of the bent nozzles, the bent portion was also aligned vertically, parallel to the chamber walls.

The nozzle tips were tapered to give a minimum interference between the air and fuel streams at the tip due to wall thickness. The taper was gradual, starting approximately one-half inch from the end, and the maximum taper of any one nozzle did not exceed 12 degrees, thereby assuring almost uni-directional flow of air along the nozzle at the taper.

No attempt, other than polishing, was made to streamline that part of the nozzles which lay across the air stream. In the case of the five nozzles bent for upstream and downstream injection, that portion of the nozzle which lay along the air stream was made sufficiently long to allow the disturbed air, caused by the cross-stream part of the nozzle, to regain its uni-directional flow once again before reaching the tip.

C TREASURE - CALIFORNIA

Mach nozzle consists of a polished at itless steel tube tapered at the end and silver soldered to weetal brace depters which screw into one of the stall wills, being hild there by a nut and two look westers, as shown in figure IV. The other end of the enapters connects to a one-half inch brass fuel line. The nozzles were align in the adopters so that when the end of the daspter as flue with the metal well, the nozzle tip as in the center of the chamber erosssection, and in the case of the bent nozile, the bant portion was also aligned vertically, persiled to the chamber walls.

The nozzle tips were tapered to give a minimum interference between the air and fuel streams at the tip due to wall thickness. The taper was gradual, starting approxiately one-half inch from the end, and the maximum taper of any one nozzle did not exceed in degrees, thereby assuring almost uni-directional flow of ir along the mozale at the taper.

No attempt, other than polishing, was made to atreamline that part of the noszles which by screek the wir stream. In the case of the five rozzles bent for upstream and downstream injection, that portion of the nexule which by along the air stream was made sufficiently long to allow the disturned air, caused by the cross-stream part of the nexule, to regain its uni-directional flow once again before reaching the tip. Each tip was measured on a metallograph at a magnification of 10.3 to 1. The outside diameters before the taper were measured with a micrometer.

N	Nozzle Gage		Diameter Outside <u>at Tip</u>	Diameter Inside at Tip	Wall Thickness <u>at Tip</u>	Diameter Outside before taper
11	Bent	10	0.1080"	0.1050"	0.0015"	0.1330"
	Perp.	10	0.1100	0.1020	0.0040	0.1280
22	Bent	12	0.0910	0.0845	0.0033	0.1070
	Perp.	12	0.0950	0.0860	0.0045	0.1030
33	Bent	15	0.0610	0.0550	0.0030	0.0710
	Perp.	15	0.0650	0.0550	0.0050	0.0710
44	Bent	18	0.0400	0.0340	0.0030	0.0470
	Perp.	18	0.0380	0.0350	0.0015	0.0460
55	Bent	20	0.0320	0.0230	0.0045	0.0340
	Perp.	20	0.0320	0.0230	0.0045	0.0340

PROPERTIES OF U. S. NAVY DIESEL OIL

Flash point, closed cup, min	• •		•	•	•	•	150 ^o f
Pour point		•					0 °F
Cloud point		•		•	•	•	10 °F
Viscosity, Saybolt Seconds Universal		•			•		40
Water and sediment, max		•	•		•	•	trace
Total sulphur, max		•	•	•	•	•	1.00%
Carbon residue, on 10% bottoms, max.						•	0.20%
Ash, max	• •	•				•	0.01%
Corrosion at 212°F, copper strips .		•			•	•	Passable
90% distillation temperature, max.	• •	•	•		•	•	675 ^o f
Color, max		•	•	•	•	•	5
Ignition quality, min. Centane number			•		•	•	50
Density, gms./cm. ³	•••	•	•	•	•	•	0.83
Surface tension, dynes/cm				•		•	25.0

Such the was hear and a second of a magnification of 10.3 to 1. The outside diameters before the toper ware measured with a microsoft.

1 <u>100</u>	otomsil obistu Mj Toled	ffe as u olf ql' to	Tet mill ebianI <u>gif Ja</u>	Diatura Distu 112 Ja	<u>e 115</u>	ROTTE
	0.1330" 0.1280	0.0015	0.1050	0.1060° 0.1100	10 10	i Pont 1 Port
	0.1070	0.0031	0.0860	0.0910	12	2 Font 2 Forp.
	0.0710	0.00.0	0.0550	0.0610	15 15	3 Part.
	0.0470 0.0460	0.0030	0.0340 0.0350	0.0400 0.0380	3.8 1.6	4 Sont 4 Serp.
	0.0340 0.0340	0.0045 2400.0	0.0230	0.0320 0,0320	20 20	5 Bent

PROFERIES OF U. S. MAVY LIESSL OIL Flood point, dibsed dup, min. 150 °F Viscosity, arboi "scould July real 1.00 Carbon residue, on 100 bos one, may. 0.205 Borroalen et 21207, corper wirige Foreable 90 AL TILLATION TO DEFUTURE, MAL. D75 °F Igaition quality, wie, Gentune number 50 Aurises tension, Graes/cm. 25.0

APPENDIX E

CALIBRATION OF MANCHETERS

Fuel Manometer

The fuel mass rate was measured with a fuel over meroury manometer across a .025 sharp edged orifice in standard half-inch brass tubing. Differential pressure in inches of hg. across the orifice was plotted against fuel rate in grams per second. The data for this curve, Figure XXXIV, was obtained by weighing the fuel accumulated in a tared beaker in a given amount of time.

Measured Wt. Grams	Time of Run Seconds	Manometer In. of Hg.	Fuel Rate Gms./Sec.
93.90	181.1	.98	.518
179.22	170.2	3.84	1.052
140.68	90.4	7.74	1.557
137.00	68.4	11.87	2.002
163.20	71.0	15.75	2.30
186.25	71.2	20.11	2.620
181.00	63.0	24.21	2.870
177.85	54-7	30.52	3.245
184.50	52.0	35.67	3.545

Air Mass Rate

The air rate was measured by a standard one-inch sharpedged orifice. Mass rate calibrations were made for this orifice for air by Dr. R. S. Bevans of the M.I.T. Staff, and Figure XXXV shows curves taken from his data. Differential pressure across the orifice was measured in inches of mercury with an air over mercury manometer. The static pressure down-

A XIMILIYA

Teleocal fert

The fuel man rate as measured its fiel over setcary mnometer across a .025 shurp edge orifice in standard helf-inch brass tubing. Hifferential presence in inches of hg. across the orifice was plotted against fuel rate in grans per second. The core for this durve, rights TXLV, was obtained by seighted the fuel accumulated in a tered bauter in a given amount of time.

1001 1001 Cas./ 100.	In. of Mg.	nun to entre	Messared Mt.
.518	62.	1.181	93.90
1.052	3.04	170.2	179.22
1.557	7.76	1.00	140.68
200.E	21.67	4.80	137.00
02.5	15.75	71.0	163.20
023.5	20.11	71.2	106.25
2.670	24.21	0.00	181.00
3.245	30.52	54.7	177.85
3.545	.5.67	52.0	184.50

The air rate was noteward by a standard one-inch sharpedged orlifice. Mass rate engineerions were made for this orlifice for air by IT. 7. 5. Sevens of the ".I.T. Staff, and Figure LLTV shows curves taken from his data. Fifterential pressure seroes the orlifice was manufed in inches of arroury with an air over notedar manufed. The static creaters downstream from the orifice was measured with an open end mercury manometer in inches. The air temperature was measured at the entrance to the diffuser and the correction to the mass rate made as indicated on the figure.

Chamber Pressure

Because the chamber had to be removed frequently in order to change nozzles and clean the transparent walls, it was deemed impractical to install a permanent static pressure tap in the chamber for measuring chamber pressure. Instead, calibration curves (Figure XXXVI) were constructed between a permanent pressure tap in the 4" x 4" section at the outlet from the air diffuser and a temporary pressure tap replacing the fuel nozzle in the spray chamber. Two calibration curves were necessary for determining chamber pressure because the chamber exhaust piping was lengthened starting with Run 121, thereby influencing both the chamber pressure and the diffuser outlet pressure.

Original Chamber Exhaust Piping

Diffuser	pressure (In.Hg.)	Chamber pressure (In.Hg.)
	10.1	9.8
	10.9	10.0
	11.7	10.0
	12.4	9.9
	13.0	9.8
	13.7	9.6
	14.5	9.3
	15.4	9.1
	16.3	8.9
	17.8	8.7
	18.5	8.3
	19.8	8.0
	23.0	7.8

stream from the orifice we receased alth an open and marcury anometer in inches. The sir terps stars was measured at the entrance to the diffuser and the correction to the mass rate ande as indicated on the figure.

Charber Freueure

Because the charker had to be reacted frequently in order to charge nonside and clean the treasport tail, it was deced imprectical to install a permenent static presure tap in the charper for measurin charter pressure. Instead, calieration curves (rigure XIVI) were constructed between a permanent pressure top in the 6° x 4° socilou et the outle from the sir diffuser and a terrater ry pressure tap reducing the fuel nozale in the spray onerbor. Two onlibration curves were necessary for determining church pressure because the charter exhaust pipting was leagthand starting with fun 121, thereby influencing both the charber pressure and the diffuser outlet pressure.

Original Canader Traust Fight

(.al.al) erparent T	SIMPLES	(.a	pressure	10201210
9.8 10.0 9.8 9.8			10.1 10.9 11.7 12.4 13.0	
9.3 9.1 8.9 8.9 7.0 7.0			13.7 15.5 15.1 17.8 18.5 19.8 23.0	

Diffuser pressure (In.Hg.)	Chamber pressure (In.Hg.)
7.4	7.0
8.1	7.2
9.0	7.3
9.8	7.3
10.5	7.3
11.3	7.2
12.3	7.1
13.0	6.9
14.2	6.6
15.4	6.4
16.8	6.1
19.4	5.8
22.8	5.9

Modified Chamber Exhaust Piping

Spark Timing

An attempt was made by Robillard to measure the duration of the spark and, consequently, the exposure time for the "instantaneous" photographs of the spray. He indicates that the exposure time is less than $.25 \times 10^{-7}$ seconds. No further attempt at physical measurement of the spark time was made by the authors, since it was felt that more accurate information than that given was not vital to a qualitative investigation.

(.m.H.a.I)	DIGRAMITO	1-010-0	() 10-	I II BOILU	10001710
	7.0				1.1	
	7.2				1.8	
	7.3				30.5	
	7.2				£.11	
	1.1				13.	
	0.6				14.2	
	6.1	and the			16.8	
	5.9				22.8	

Maight Jauales and cons be filler

an attempt was such by Robillard to measure the duration of the spark and, consequently, the exposure time for the "inetertanous" photographs of the spray. He indicates that the exposure the is less than, 5 x 10⁻⁷ seconds. He further at-

tempt at minical measurement of the spark time was made by the authors, since it was fait that for accurate information that that siven was not vital to a multistive investigation.









FIGURE XXXV CALIBRATION CURVE FOR ONE INCH AIR ORIFICE FROM DATA OF R.S. BEVANS STATIC PRESSURE DOWNSTREAM OF ORIFICE -AIR TEMPERATURE 75F*



I have a set of the second set of the second s





APPENDIX F

DATA AND GALCULATIONS

A TIMOTA

BHOITAIJOLAD TRA ATAT

	Δ Pa	A Pf	Po	Þđ	Pc	Poa	Pca	Wa	ME	Va	JA
NOL	in.hg.	in.hg.	in.hg. gage	in.hg.	in.hg. gage	psia	psia	<u>gms</u> . 860.	gns.	ft. 800.	ft. 800.
17	25.0	32.6	14.2	16.1	6.2	21.7	17.7	180.5	3.37	665	49.5
22	82. N	32.2	14.6	16.4	6.1	21.9	17.7	182.5	3.35	673	8.7
50	0.0	32.2	0.1	100	2.3	18.1	16.3	78.0	3.35	278	0
29	1.7	32.1	0.0	- C	7.1	17.9	18.2	40.4	3.34	145	8.0
34	26.0	4.0	12.8	16.7	6.1	21.0	17.7	183.0	1.08	675	00 • •
39	6.2	33.2	8.2	0.6	7.3	18.7	18.3	79.5	3.40	284	22.8
40	25.1	32.2	11.9	16.1	6.2	20.5	17.7	179.0	3.35	660	22.5
50	25.0	31.1	14.1	16.1	6.2	21.6	17.7	179.0	3.27	660	3.5
51	6.2	30.8	6.8	9.0	7.3	19.0	18.3	80.1	3.26	286	3.5
Sw	1.7	3.8	7.4	7.8	7.1	16.3	18.2	41.0	1.05	147	1.1
54	1.7	31.0	8.0	7.8	7.1	18.6	18.2	41.0	3.27	147	3.5
74	25.0	32.0	14.2	16.1	6.2	21.7	17.7	181.5	3.34	699	2.4
22	5.9	32.0	8.9	8.9	7.3	19.1	18.3	78.0	3.34	278	2.4
LL	1.6	3.65	7.5	7.8	7.1	18.4	18.2	41.8	1.05	150	0.7
29	1.6	4.0	7.1	7.4	7.0	18.2	18.1	38.8	1.10	140	16.2
80	1.6	32.0	7.0	7.4	7.0	18.1	18.1	38.8	3.34	140	49.0
32	27.4	3.3	13.7	17.5	6.0	21.4	17.6	191.1	.96	709	14.1
36	24.8	32.3	13.7	16.7	6.1	21.4	17.7	179.1	3.35	661	49.2
37	5.6	32.3	9.8	9.3	7.4	19.5	18.3	76.8	3.36	274	49.3
00	6.2	3.00	8.3	9.3	7.4	18.8	18.3	79.0	1.05	282	15.4
16	3.2	31.6	7.6	8.9	7.3	18.4	18.3	55.1	3.31	197	48.6
92	3.3	3.5	7.8	8.2	7.2	18.5	18.2	55.1	10.1	198	14.8
76	25.2	31.0	13.6	17.0	6.0	21.4	17.6	180.0	3.28	667	22.0
35	26.1	3.00	13.3	16.8	6.1	21.2	17.7	185.1	1.05	683	7.1
96	6.2	3.9	8.4	9.3	7.4	18.8	18.3	79.0	1.07	282	7.2
26	6.0	32.0	0.1	9.2	7.4	18.7	18.3	77.5	3.34	276	22.4
98	1.6	3.00	7.6	7.8	7.1	18.4	18.2	41.6	1.05	149	7.1
66	1.8	32.0	7.4	7.8	7.1	18.3	18.2	41.6	3.34	149	22.4
101	1.6	33.1	7.4	7.6	7.1	10.5	18.2	38.8	3.40	139	20
104	6.8	4.0	8.2	9.4	7.4	18.7	16.3	82.8	1.08	295	2.8

8.5	B.#	4.25	1.1	2.88	2.7	1.1	53.0	14.11	0.04	2.2		E.0.	5.04	1.1	D. P.A	70.2	1.0		2.5	R + 1			50	10.5	0150	10.0	d. 5	7 . I.	7.8	2.01	-000	N.
295	PE.F	14	140	576	SSS	623	Sau	TANK			2201	100	60T	203	110	TTO	120	1-12	600	141	721	280	000	Dog 1	180	250	112	518	673	200	-021	-
.50 · L	3.70	tr. P	20.1	3.39	10.1	20.1	85.5	10.1	1.34	200	210	ر در در ۱	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~	00.	1.2.	7.10	20.1	127	* 5 * 6	7.5	20.1	0.80	3.57	1000	04.6	30.I	3.34	3.32	3+32	3.35	1004 1004	1
6.58	8.86	3.I.1	6.11		0.27	1. 202	760.00	23+5			01.00	8.07	7-4-7	191.1	00 · 8 (*)	311-11	3. 24	0.37	5.101	0.14	0.14	1.08	7.13.0	0.071	2.01	0.281	1.04	0.85	2.351	10.2	. 595	58
E.01	S.81	S.81	S. BI	· · ·	E-MI	1. 5 50	0.71	N. BI	C.01		10/10/1	2.31	1.75	0.71	1.81	1.81	S. BI	£.81	1.12	10.	18.5	E + 81	1.11	7.72	5.82	2.51	S.81	23.3	1.71	27.7	<u>nated</u>	Dog
10.7	TR'2	C	1.35	1.24	5.0I	12.25	から	50.25		1 0 1		19.0	2.1.	5.1.1	18.1	.81	1.01	1.01	5.15	0.1	(+ 3 m	0.91	51.0	20.0	7.81	0.15	0.55	1	51.0	5.15	pilag	Fou
4.5	1.7	1.7	1.1	2.7	4.5	1.0	0.0		(· ·)	3		1.7	1.0	0.0	7.0	0.5	1.1	M.1	10.0	1.1	1	n.1		0.01	(m=-)	1.0	5.7	2.2	1.0	6.2	. Ju. al	To
1.0	3.5	0.5	01	52.00	/m *	3.01	15.0	N.B.	4.0	0/.	5	er. 0	7.01	1.71	4.7	2.7	0	2.0	1.01	8.7	8.7	0.2	7.07	7.01	0.0	7.01	8.7	Q. W	10.4	1.01	- 34.01	2
5.8	4-7	· · · · · · · · · · · · · · · · · · ·	d.P	1.8	4.8	10,1	73.0	8.1	0.1	1	20	00.0	22.2	2.1.2	0.5	1.7	2.1	12.0	S . 11	0.8	4.5	0.0	7.47	N.TT	5.00	FS.	0.0	0.1	0.11	5.41	10.08.	0.
0.4	1.00	35 + 0	3.8	35.0	3.6	3.8	0.10	17.	7.0			6.31	10.10	~· ·	0.92	D.4	0.1	0,50	0.55	D.12	0.8	30.8	1.1	25.56	33.5	0.1	1.52	2.18	5.00	32.0		A T &
0.0	D.1	3.1	e Jon	0.0	3.0	1.05	52.5	5.5	24	0.00	a la	d.7	0.15		0.1	0.1	7.8	2.07	0.55	7.L		2.0	0.25	1.25	5.0	20.0	5.1	0.0	1.25	0.20	. 5d . at	A
TOF	101	10	NO	No	90	20	34	AS	N.	-	20	100	80	28	60	13	1	135	- MA	10	52	22	20	04	23	AL	29	67	25	TA	1. Uy	

IA	ft. 800.	2.8	03 - N	2.7	1.l	3.0		3.7		301	N	0.7	0.8	5.3	0.8	2.4		0.8	2.2	0.8	2.5	0.0	2.5	1.1	5.00	1.1	3.51	1.1	3.5	00 :	0.6
Aa	ft. 800.	171	169	650	656	679	200	279	647	647	750	145	276	276	685	685		653	653	270	276	139	139	040	643	266	271	734	129	621	979
JM	<u>586</u> .	1.08	3.23	1.05	toot	3.36	1.02	14.6	10.7	いなった	3.34	1.01	1.07	3.30	1.03	3.32		1.05	3.31	1.06	3.37	1.06	3.36	1.05	3.33	1.05	3.35	1.05	3.36	1.05	3.38
Ma	528. 090.	47.6	185.1	178.2	178.2	183.0	80.0	78.2	Do Th	0.14	9-74	40.2	7.7.5	77.5	184.6	184.6		186.1	126.1	83.0	83.0	42.7	41.7	183.1	183.1	80.0	81.4	40.4	200	179.0	185.6
Poul	psia	10.2	17.5	17.7	17.7	17.6	18.3	- 2 · - 2		70.27	10.71	18.2	10.3	18.3	17.6	17.6	MILT.	18.6	18.6	19.6	19.6	19.6	19.6	18.7	18.6	19.6	19.6	19.6	19.6	18.8	18.7
Poa	psia	18.8	23.2	21.9	21.1	22.4	19.1	18.7	い。こう	0.00 10 10	16.3	18.2	18.6	18.6	21.5	21.5	GAUTT 1	22.9	22.6	20.9	20.8	20.1	19.9	22.3	22.3	20.4	20.5	19.7	19.7	22.1	22.4
Pc	in.hg.	7.2	5.3	6.1	6.1	0.0	4.6	4.2	1.1	T. /	7.0	7.1	7.3	7.4	ú.0	6.0	CHANDER ID	8.0	8.0	6.6	6.6	10.01	10.0	3.5	8.0	10.0	6.6	6.6	6.6	8.3	8.1
M	In.hg.	8.0	20.0	16.7	10.5	17.1			1.0	1.00	7.04	7.6	0.6	9.1	17.6	17.6	ODIVIED	20.0	19.7	12.7	12.6	11.3	11.1	19.0	19.6	12.1	12.7	10.4	10.3	16.4	19.2
Po-	In . hg . Lege	2.6	17.2	14.6	13.1	13.7	67 1 10 1	00 1	1.1	1.1	7.3	7.2	6.0	0.8	13.9	13.9		16.6	16.0	12.7	12.04	11.0	10.7	15.4	15.4	11.5	11.7	10.2	10.1	15.1	15.7
∆ P T	in.hg.	4.0	30.2	3.8	3.6	32.6	4.0	33.4		33.5	31.7	3.6	3.9	31.3	4.0	31.8		3.8	31.6	3.9	31.7	3.9	32.3	3.8	31.9	3.8	32.2	3.9	32.4	3.03	32.7
D Pe	in.hg.	2.4	24.9	24.5	24.5	25.4	5.	0.5	20 1 		1.8	1.7	6.1	6.1	25.6	25.5		25.4	26.0	6.2	6.2	1.7	1.7	25.6	25.4	20.00	6.0	1.6	1.5	24.0	26.0
	RUN	105	107	108	109	110	TTT	112	ETT	114	115	116	117	118	119	120		121	122	123	124	125	126	127	128	129	130	131	132	133	134

-0107		~ ~ ~	20		8,0	1.8	0.0	1.0	141	100	17.6		the state	ile i	法
018 750 1750	175	mai	130			280	1918	120		070	050	199	1WI	11	-
20.2	n ni		0 	24.21	e	3. 25	20.4	10-1	131	150 a 1	1001	CR.C	- 2042	is	t,
8.281 8.281	1-10	7.001	7.7.	1.082	L'ANT.	2.151		1.0.1	0. 24	100	2.001	20.14	A STREET	-	N/m
18.5 1.6 18.0	0.01	2.81 2.01		0.11	Th the	0.71	1. S.	1		0.01 1			tages	1	2.02
1.22	1.05	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	2,02	64110 64110	TEDA EX		Tar.	19.2	192	1.02		1191	日日	1	204
1000 2000	8.9 70.0	C. CT	0.02	8.0	Curry Party in the	9.0	2-2-	0.1	1	1.0	1.0		-	- Mal . 311	The second
19.92	1.1	, . 7 . T	5.12	7	and Linn	0,71	and a second	1-5	tino		10.02	0.1	- ANT	and and	5
1011	and a a a buy play for	7 J.	17.0	0.01	A STATE		0.0	C-3	1.30		1	-0-1-1	- and	19	
101.00	1.1	5.55		0.17		1.1. 1.1.		23.5	1. A.	p a - a	0.00	0.10	-104-14		
00000	Ph	5. F	200	151	0.00	0.25	7-1 1-0	a bis			10.10	44	201.01	Δ	
	1280	- dest	101	RE	Ven	3770	100		E	100	22	192	周		

JA	ft. 800.	2.9	2.7	8.9	21.3	6.5	21.4	6.9	15.9	15.9	49.5	15.6	44.8	16.5	49.5	1.2	1.1	1.1	1.2	0.0	0.8	10°0	0.8	2.3	0.7	2.7	2.8	2.8	2.8	7.3
Va	ft. 800.	274	140	140	642	266	276	139	137	642	653	268	274	139	137	271	271	271	661	659	262	271	271	135	278	271	271	271	135	264
TW	EMS.	1.09	1.03	3.33	3.36	1.03	3.37	1.09	2.50	1.08	3.37	1.06	3.05	1.12	3.37	1.09	1.05	1.05	1.10	1.10	1.09	1.09	1.08	3.28	1.05	1.06	1.10	1.09	1.09	1.06
Wa	<u>gas</u> . 80C.	82.1	42.0	42.0	184.0	79.2	82.9	61.7	41.1	184.0	186.1	80.6	82.0	41.7	41.1	81.4	4.18	81.4	158.8	187.9	78.5	81.4	81.4	40.4	83.6	81.4	81.4	81.4	40.4	79.2
Pca	psia	19.6	19.6	19.6	18.7	19.5	19.6	19.6	19.6	18.7	18.6	19.6	19.6	19.6	19.6	19.61	19.6	19.6	18.6	18.6	19.6	19.6	19.6	19.6	19.6	19.6	19.6	19.6	19.6	19.6
Poa	psia	20.4	20.4	20.3	22.5	20.6	20.7	20.1	19.9	22.4	23.0	20.8	20.3	20.1	19.9	20.2	20.2	20.2	22.8	23.0	20.5	20.1	20.1	19.7	20.3	19.9	19.9	19.9	19.7	19.9
Po	in.hg.	9.9	10.0	10.0	8.1	2.6	6.6	10.0	10.01	-•0	7.9	9.6	9.6	10.0	10.0	10.0	10.01	10.0	0.0	7.9	10.01	10.0	10.0	6.6	10.01	10.0	10.0	10.01	10.0	10.0
pd	in . hg . gage	12.5	10.9	10.8	19.1	- - 	12.7	11.2	10.8	19.2	20:4	12.7	12.7	11.2	11.2	12.3	12.3	12.3	19.9	20.4	12.4	12.0	12.0	10.3	12.3	11.7	11.7	11.7	10.5	11.7
Po	in.hg.	11.5	10.5	11.3	15.8	12.0	12.2	10.9	10.7	15.7	16.8	12.5	11.3	11.0	10.6	11.2	11.2	11.2	16.5	16.8	11.8	11.0	11.0	10.3	11.3	10.6	10.7	10.7	10.2	10.7
A PT	in.hg.	4.1	3.7	32.6	32.3	3.7	32.5	1.4	18.4	0.4	32.5	3.9	27.0	4.3	32.5	4.2	3.8	3.8	4.2	4.2	4.1	4.1	4.0	31.0	3.0	3.9	4.2	4.2	4.1	4.0
A Pa	in.hg.	6.2	1.7	1.7	25.6	2.0	0.0	1.7	1.7	25.7	25.5	5.0	6.2	1.7	1.7	6.0	6.1	6.1	26.0	25.9	5.6	6.0	6.0	1.6	6.6	6.2	6.2	6.2	1.6	6.1
	RUN	136	137	138	0411	141	142	143	144	145	146	148	149	150	151	152	153	154	155	157	158	159	160	161	165	167	168	169	170	171

	10.0	1.5	2/2	5.0	0. 1	C7 .	0			-	C. P.4	* * *	07 	0-1-2-1-2-1-2-1-2-1-2-1-2-1-2-1-2-1-2-1-	. 12		2.2	いたい	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	C.18	C.8	1.	0.0	0.2	- FT	100	2
t- n+	272	17-	2001	17	517	202	050	100	200	375	135	QCI	たつい	208	620	5		100	NOON	512	THE	CAL	173	ans	· 0 · 4	-46-	
80.1 80.1	01.	00.5	10	7.03	90. L	20,1	01.1			100.4	15.5	AL.	20.1	1.00	12.0	00.1	00.0		うべいた	0	5° 33	£0.1	2:0	1.09	讀	- alt	
5.07 5.01	4. L3	1.18	とういう	N.IE .	3		0.731		1.1.	4.10	1. [4]	Y. 14	80	0.38	1.361	0.411		7	2.07	70.101	0.54	N5.0	5.3	. 08	Rub.	100	
9.61	2.4.L	9.02	0. F	21.0	0.91	9.91	0.81	0.37	2.0.4	19.61	7.0	0.01	0.01	C+01	3.0.5	7.51	0.01	2. 5	1.2		9.47	0.01	13.0	d.e.t	ming	102	
10.9 10.9	10.01	0.25		1.05	1.05	20.00	0.65	101	0 00	20.2	8.91	3.02	E.05	20.0	0.05	-	10.00	1.00	10.0	2.55	8.05	5.05	20.92	20.4	BING	100	-
0.01	10.0	0.01	2.9	0.01	0.01	10.0	R	0.4	0.01	0.01	10.01	0.0I	10.10	0.9		1.0	0.01	2,0		3 2-07	10.01	0-01	0.01	0	and, us	10	Contra la
1.01	7.IL		E.01	0.31	01	1. SI	1.02	0.01	1	S***	277.5	33.8		Y. 54	1.05		0		20.01	1.04	0.01	P.05	15.8	14.2	-pd-ut	100	in the second
10.7	1.01	0.05	2.0L	0.11	C.LI	5.50	0.01	19.91	1	17 * 5	9.01	O.LL	E.IL	A	8.61	12.4	101	0.00	· · · ·	22.2	21.3	10.2	1.15	a free	· ssi.at	10	-
01.1		0.2	0.12	0.4	ショー	F. B.	0	10.1		1.1	2. XE	2.0	0.12	0.6	رم) رم	0.1	1- 1- 1- 1- 1- 1- 1- 1- 1- 1- 1- 1- 1- 1	1 - 1	2.4	C.SE	G- 58	* *	11. 12	I.A.		- 4	>
1.0	10	17 0	20.1	0.0	0.0	6.67	0.22	0	2.0	0.0	T.L	P.1	12.0	د د د	10-5	1.25	2.0	The second	0.0	53.0	T=1	5.2	10.0	5.0	-164. 11.2	0 10	
7120	128	197	Let	160	123	126	122	227	194	125	727	120	DAC	BAL	345	11	100	20%	Ter	TNO	138	TEL	120	201			

ZC

JA	ft.	7.3	7.3	7.3	7.4	00.00	16.0	15,9	15.9	49.2	200 sml	15.8	49.4	49.2	15.9	15.9	49.2	7.3	22.6	7.3	02 4	7.3	22.4	0.00	0.0	83 83	8.6	C3	9.0	3.0	1.2
Va	ft. <u>56C</u> .	266	266	665	144	137	272	272	272	272	653	268	270	624	640	129	129	273	273	640	624	139	139	618	610	272	272	120	139	129	273
100 Mar	gms.	1.09	1.09	1.09	1.10	3.41	1.09	1.08	1.08	3.35	1.07	1.08	3.36	3.35	1.08	1.08	3.35	1.09	3.36	1.08	3.33	1.09	3.33	1.09	3.34	1.08	3.34	1.05	3.32	3.34	1.10
がる	gns . 360.	8.67	79.9	189.8	43.1	41.0	61.9	81.9	61.9	61.9	186.0	80.5	81.2	178.8	162.1	38.7	38.7	82.0	82.0	182.1	178.8	41.6	41.6	180.0	177.8	82.0	62.0	41.6	41.6	38.7	82.0
Pca	psis	19.6	19.6	18.6	19.6	19.6	19.6	19.6	19.6	19.6	18.6	19.6	19.6	16.7	18.6	19.6	19.6	19.6	19.6	18.6	18.7	10.6	19.6	19.0	19.0	19.6	19.6	19.6	19.6	19.6	19.6
Pos	psia	19.9	19.9	23.1	20.2	19.8	20.1	20.1	20.1	20.1	22.03	20.1	20.2	22.1	22.4	19.7	19.7	20.3	20.2	22.4	20° J	20.1	20.1	21.4	20.4	20.0	20.3	19.8	19.9	19.7	20.7
PC	in.hg.	10.0	10.0	8.0	10.0	10.01	10.0	10.0	10.0	10.0	8.0	10.0	9.0	9.1	8.0	0.0	8.8	9.9	10.0	8.0	8.1	10.0	10.0	8.7	B •8	10.0	9.8	10.0	10.0	8°8	9.9
pd	1n.hg. Kage	11.7	11.7	20.1	11.3	10.7	11.5	11.5	11.5	11.5	19.7	12.3	12.4	19.3	20.1	10.3	10.3	12.7	12.3	20.2	19.3	11.3	11.3	17.0	16.7	11.8	12.7	6.01	10.9	10.5	12.7
Po	In.hg.	10.7	10.7	17.0	11.1	10.3	0.11	11.0	11.0	11.0	16.4	11.0	11.1	15.1	15.7	10.1	10.1	11.3	11.2	15.7	15.1	11.0	11.0	13.6	13.5	10.7	11.3	10.5	10.5	10.1	12.2
APE	in.hg.	4.1	4.1	4.1	6. H	33.2	4.1	4.0	4.0	32.1	6.0	4.0	32.2	32.1	4.0	4.0	32.1	4.1	32.2	4.0	31.8	4.1	31.8	4.1	32.0	4.0	32.0	3.8	31.6	32.0	4.2
A Pa	1n.hg.	6.1	6.1	26.0		1.7	6.2	6.2	0.0	6.2	26.7	5.9	6.0	24.8	25.0	1.5	50 10 10	6.2	6.1	25.0	84.8	7.7	1.7	80°8	24.9	6.0	0.0	7.7	1.7	1.5	Q.9
	RUN	172	173	174	175	177	178	179	180	181	182	184	135	186	188	190	191	192	193	194	195	197	198	199	200	201	202	203	204	206	207

	3.5	- B.B	10.0	2.24	-4-0	240	58.	1.50	10.7	35.0	P P	10.8	30.01	30.8	- 10.05	1312	1.1	11.4		No. of P	0.01	- P. 32	2.5	2.5		-	1000	44.
	SAS.	1085	NASE SAN	10112	- STO	ors	123	021	090	EAN	5/12	150	10.0	010	et. 4	240	2000	600	EANS To be	Cishos -	SAN S	NET	335	2003	AND -	-	1	No.
	7-76-2	御湯	1.06	T*Uel	19840	11000	100.0	1.000	11000	62.8	00.1	2.20	-bo.1	10.1	60.46	08.0	50-1	20.1	02.1	2+10	T+DA-	21.81	T-TO	1.03	100.1	No.		d.
	P-102	0.13	31.15	10,00	B.WDI	100.0	a la	D-DA	1881	0.10	0.10	N. A.S.	19.02	R. wdt	3. 72	2.18	0.08	D. MAL	81.5	tor . D	2.11	AL'S	1.3	8.844	100	0.04	- 945 544	-TION
	Da vil	0. AL	0.01	To-0	0.01	10.02	10.6	To all	4.4	NA P	3.41	Piris.	20.02	3.31	TU A	8. 02	19.9		0.07		4. 4.	Ta'e	78.0	0. SL	19.81		Mas	Now.
	10.44	4.85	0.00	0.02	5402	91.10	F.OR	5-02	H = 30	E010	20.02	1.07	Nº 11	14 12	7.20	10.00	F. 05	0.00	1.08	- ANN	1.05	18.0	8.08	1.00	10.02	0.00	alter	101
A Real Property lies	8.9	10.01	0.97	2004	10-0-	× + 1	in al	I. H	8.0.	30.00	10.00	10 10	0.0	3.0	1.8	8.4	0.01	0.8	10.0	10.01	0.01	20.00	0.01	8,0.	10.01		turbe.	he
	70-5-	10.61		11.8	78.4	14.00	10121		30.18	76.55	T.S.C	10.	10.5	1.02	3618	15.4		Je N	1. 1.	1		THE	32.20	1.02		-	-94-111	bul.
	Tion .	1.01	-04.11	T074-	0.11		0.71	1.11	- V. O.E	2.11	17.8-11	C. T. OL	10.8		12.12	17.11	11.10		1000	10×24	0.11	10.02	1.22	17.0	71.12	- SHOW	10158-	las.
	- 0.80	22.6	0.82	0. 5	- 0.95	1.7		0.10	0.2	21.92		1.95	0.4	- 0.8	T. 2.95		0.1	0.00		1.00	-1212	- 2.20	1000	1.0			. mr. nt	2
		1.4	10.40	P. 2.	9.15	2.12	1.1		0.0	113	2.4	a	1.1	0.38	20.00	0	0.1	W.00			10.0	1.V	E	0.00	1.0		Turest.	A
	COL.	100	008	foe	800	100	1-1-1	Tap	104	1001	101	Ini.	100-	1000		108	Line .	- 607	- Hart	- NUL	TADI	Lan	742	347	TAR	- inter	視	

	A Pa	2 Pr	Po	pd	pc	Poa	Pca	Ma	了温	Va	JA
RUN	1n.hg.	<u>in.hc.</u>	in-hg. gage	in.hg.	in.hg.	psis	psia	gms . sec .	5708 . 360 .	ft. sec.	ft. 380.
208	6.0	31.8	11.7	12.7	6.9	20.5	19.6	80.5	3 • 33	268	3.6
203	200 - 1	32.3	15.8	19.2	6.1	22.5	18.7	186.0	3.36	650	3.6
510	24.6	4.0	13.1	16.6	8.8	21.2	19.0	176.8	1.08	614	1.2
211	1.6	4.1	10.3	10.4	9.9	19.8	19.6	39.4	1.09	131	0.8
212	1.0	32.1	10.1	10.3	9.9	19.7	19.6	36.7	3.35	129	2.4
214	(1) (1)	32.2	12.5	12.7	9°9	20.9	19.6	80.5	3.36	263	2. 4 to
215	24.8	4.0	15.1	19.3	8.1	22.1	18.7	178.8	1.08	624	0.8
516	25.0	32.1	15.7	20.2	8.0	22.4	18.6	182.1	3.35	640	8. S
220	25.3	4.0	15.6	19.1	8.1	20.4	18.7	186.0	1.08	650	6.9
228	1.7	4.1	10.5	10.9	10.0	19.9	19.6	41.6	1.09	139	1.2

DESERTION DESERTION	10.	H.
	* 01	410
Secondenses Andreadhan	. 967.1	14
A. BALL A. BALL	200.	-
	pain	10m
07.000 07.000 00.001 00.00000000	11 2	BOR
Choracome Despectate	• 3//• nl	24
	Tinue Linue	BA
	treas 12-pc-	P.o.
Torophy and the second se	· Wrant	ΥA
	-nding	4

APPENLIX G

SAMPLE CALCULATIONS

Run 180 - Nozzle 5 Downstream with modified chamber exhaust piping.

- Calculate: (1) Fuel weight rate, (2) fuel velocity, (3) air weight rate, and (4) air velocity.
- Assume: (1) Compressible, steady state air flow in the constant area chamber. This assumption is valid for the maximum Mach number encountered, approximately 0.6.

(2) The temperature of the air preceding the air diffuser constant and equal to the temperature of the air in the chamber. The variation in temperature preceding the diffuser in all runs was no greater than four degrees and the maximum temperature difference between it and the chamber temperature was 10°F, this maximum occurring only at the highest air rate. To simplify tedious calculations, a mean temperature of 95°F was used for all runs. The maximum error in air weight rate by this assumption was 2% at the highest air rate, less for lower air rates. Since the use of Figure XXXV gave an error of approximately 5%, this assumption was justified.

S. ALIMS. STA

SHOLL CALCULATIONS

Hun 180 - Nomile 5 Fownstream with modified chemier canvest piping.

Celeul to: (1) Fuel weight rate, (2) fuel velocity, (3) air wight are, and (4) air velocity.

(1) Compressible, steady state air flow in the constant errs chamber. This assumption is valid for the maximum flack number encountered, approximetely 0.6.

(2) The temperature of the is proceeding the air diffuser constant and equal to the temperature of the air in the charber. The variation in temperature preceding the diffuser in all range was no greater than four degrees and the militum temperature difference between it and the militum temperaporature was 10°, this mainum constring only at the highest air rate. To simplif, tations calcuistions, a mean imperature of 95° was used for all runs. The mainum error in sir weight rate of this mean tion was 20 at the digment air rate. XXXV ave an error of apportancely 56, this assumption was furtified.

(Cont'd) $t_a = 95F$ Ta = 555F abs. Given: Pf = 90 psi. $\Delta Pf = 4.0$ in. hg. $\Delta Pa = 6.2$ in. hg. Pd = 11.5 in. hg. gage Po = 11.0 in. hg. gage Poa = 20.1 psia. $d_{P} = 0.83 \text{ gms./cm.}^{3}$ Di = 0.023 in. $Ai = 4.155 \times 10^{-4}$ in? for nozzle 5 Downstream $Ac = 1.0 in^2$ Solution: (1) with APf = 4.0, enter curve, Figure XXXIV, read Wf = 1.08 gms./sec. (2) $Vf = \frac{Wf}{Af de}$ $\frac{1.08 \text{ gms./sec.}}{4.155 \text{ x } 10^{-4} \text{ in?}} 0.83 \frac{\text{gms}}{\text{cm.}^3} 6.45 \frac{\text{cm.}^2}{\text{in?}} 30.48 \frac{\text{cm.}}{\text{ft.}}$ = 15.9 ft./sec. (3) With APa = 6.2 and Poa = 20.1 psia, enter curve, Figure XXXV, read Wa = 11.0 lb./min. Wa (corrected for temperature) = Wa (1.06 - 0.0008 t_BF) $= 11.0 (1.06 - 0.008 \times 95)$ = 10.81 lb./min. = 10.81 lb./min 453.6 gms./lb. 60 sec./min. = 81.9 gms./sec.

(b' frod)

Given: t = 95F = = 555T abs. Pf = 90 pst. AFT = 4.0 1n. h. APu = 6.2 in. 1g. Te = 11.5 in. bg. gage 034 .3d .ni 0.11 = 01 . . L . CS = NO1 de = 0.83 mas/c. 3 D1 = 0.023 in. $A1 = 4.155 = 10^{-4}$ in? for nozrio 5 om cream .c = 1.6 1n.2 Solution: (1) Mith APT = 4.0, Later curve, 1 are XXIIV, re 1 1 = 1.05 (8.8. / 80. and a second description of (2) --== 1.98 223 80.1 = 4.1.5 x 10-4 1n. 0.63 1 6.45 cm. 30.48 200 = 1:.9 It./800. (3) With ATe - 5.2 and Fou = 20.1 pais, enter ou.v., Fiur M.V. read . - 11.0 1b./ 1n. (corristed for to neritars) at (1.º 0000.0 - 00.1) HW = ADDRESS TALL ADDRESS TO THE OWNER. = 11.0 (1.00 - 0.008 x 95) = 10.41 1b./min. 10.81 10./ala 453.6 gas./1b. . 988 . Amg 9.18 =

SAMPLE CALCULATIONS (Cont'd)

(4) With Pd = 11.5, enter curve, Figure XXXVI, modified chamber exhaust piping, read Pc = 10.0 in. hg. gage, Pca = 19.6 psia. Then Va = $\frac{Wa \ R \ Ta}{Pca \ Ac}$ where R is the gas constant for air = 53.34 ft. lb./lb. F abs.

-	01.4	Sec.	1b.Fabs.	222 8	aps.
	19.6	$\frac{1b}{1n.2}$	1.0 in. ²	453.6	gms. lb.

= 272 ft./sec.

SAPPIN CALOULATIOND (Contid)

- 1 L

(L) WITH Pa = 11.5, enter curve, 1 are XIXVI. 2-2 = 721 ber , minig tenne reduce ballion 70 = 10.0 In. Mg. gago, cu = 19.6 mia. 1 . . . ang odd al H statw DEDA DI 1:51 = 11 TOI JELJELOO 53.34 10. 10./1b. F abs. 53.36 26.3.g. · BER 555 P 405. 81.9 . 388 The There a 10 10. 1.9 12.2 453.6 · 213 19.6 S. I

= 272 It./ 800.
APPENDIX H

LITERATURE CITATIONS

- 1. Sauter, J., "Determining Size of Drops in Fuel Mixture of Internal Combustion Engines", NACA T.R. 390, 1926.
- 2. Sauter, J., "Investigation of Atomization in Carburetors", NACA T.M. 518, 1929.
- 3. Lee, D. W., "The Effects of Nozzle Design and Operating Conditions on the Atomization and Distribution of Fuel Sprays", NACA T.R. 424, 1932.
- 4. Lee, D. W., "Photomicrographic Studies of Fuel Sprays", NACA T.R. 454, 1933.
- 5. Lee, D. W., "A Comparison of Fuel Sprays from Several Types of Injection Nozzles", NACA T.R. 520, 1935.
- 6. De Juhasz, K. J., "Results of Recent Oil Spray Research", Trans. ASME OGP-51-9, 1929.
- De Juhasz, Zahn & Schweitzer, "On the Formation and Dispersion of Oil Sprays", Bulletin 40, Penn. State College Eng. Experimental Station, 1932.
- 8. Castleman, R. A., Jr., "The Mechanism of the Atomization of Liquids", Research Paper 281, Bar Standard Journal Research, March 1931.
- 9. Castleman, R. A., Jr., "Mechanism of Atomization Accompanying Solid Injections", NACA T.R. 440, 1932.
- Rayleigh, Lord, "On the Instability of Jets", Proc. London Math. Soc., Vol. X, pp 4-13, 1878.
- Longwell, J. P., "Fuel Oil Atomization", Sc.D. Thesis, M.I.T., Course X, 1941.
- 12. Kolupaev, P. G., "Atomization of Heavy Fuel Oil", Sc.D. Thesis, M.I.T. Course X, 1941.
- Snuggs, J. F., "Atomization of Heavy Fuel Oils", S.M. Thesis, M.I.T. Course X, 1938.
- 14. Robillard, G., "Atomization of Liquid Fuels in an Airstream", S.M. Thesis, M.I.T., Course X, 1947.

APPENTIX H

EMOIPATIO BEUTAS PPIL

- 1. Suter, J., "Leter daine Size of Drove in Turk intere of Internal Co Sustion Engines", MICA T.R. 390, 1926.
- 2. Suter, J., "Invaligation of tomization in Carburstors", MACA T.M. 518, 1929.

and the state of the

- 3. Lee, I. ., ""In fifets of oral freigh and Operating Conditions on the storic time and list ibu ion of Fuel Spray.", MGA T.H. 42, 193.
- 4. Lee, I. W., "Protonicrographic Studies of Tel Sprays", NAC T.E. 454, 1933.
 - 5. Les, D. R., "A Comparison of Fuel Sprys fro Everal Types of Injetion forsis", MACA 1.1. 520, 1935.
- 6. De Juhasz, . J., "Realts of Recent Oll Fray Recourds, Trans. 15 202-51-9, 1929.
- 7. De Jukesz, Zahn Cobreitzer, On the loration an Dipersion of Oil Sprys", Salletin 0, 1mm. State Colle e Lag. Experimental Easton, 1932.
- 8. Castleman, R. A. Jr., "Inc Schanlar of the Tomization of Liquids", Honoarch Part 281, Bar Handard Journal Research, March 1931.
 - Castleman, E. ., Tr., "echanim of Application cocompanying Solid Injections", Mat. T. 140, 1932.
 - 10. Raylei h, lord, "On the instability of Jac", Proc. London with. Soc., Vol. , p. 4-13, 14 8.
 - 11. iongwell, T. P., Pull 011 contration, M.F. Inemia.
 - 12. Kolupav, F. G., "Atomization of Mary Fiel Dil", Sc.D. Themis, M.I.T. Course I, 1941.
 - 13. Snuger, J. T., "toristics of heavy Fuel Oils", P. ..
 - 14. Mobillard, G., "Atomisation of Linkid Tuels in an Airstroum", 5.M. Tuesia, W.I.T., Course X, 1917.

LITERATURE CITATIONS (Cont'd)

- Nukiyama & Tanisawa, "An Experiment on the Atomization of Liquid", Report 1-6, Trans. Soc. Mech. Eng., Japan, 1938-1940.
- 16. British Fuel Research Board Report for the year ending March 31, 1938.
- 17. Lewis, Edwards, Goglia, Rice and Smith, "A Study of the Atomization of Liquids", National Defense Research Committee, Div. 10, OSRD No. 6345, Oct. 10, 1945.
- 18. Haenlein, "Disintegration of a Liquid Jet", NACA T.M. 659, 1932.
- 19. Bishko, H. & Freudenthal, B. R., "Atomization of Fuel Oil", M.S. Thesis, M.I.T., Course X, 1925.
- 20. Leising & Rice, "Flame Propogation in Fuel Oil Sprays", M.S. Thesis, M.I.T., Course XIII-A, 1946.
- 21. Pfieffer, Murati & Engel, "Mixing of Gas Streams", M.S. Thesis, M.I.T., Course XIII-A, 1945.
- Esso Laboratories Progress Report on "Study of Combustors for Supersonic Ram-Jet", Period April 1, 1946 - June 30, 1946. (CONFIDENTIAL).

MOITALL U SALL

- 15. Mukiyana & Taniana, "In Experient on the Monigation of Liquid", Meport 1-5, Tans. 200. Mech. Edg., Japan, 1938-1940.
 - British Fuel Sessare Board Roport for the rear shoing 16. arch 31, 1933.

Study of the New rold	DATE DUE			Lois, Eivel Lois, Los Constee,	17.
M.CA T.M. 059,	1 Feb '49 16 Feb '49			Heenlein, * 1932.	18.
ton of Fuel				Bishko, B. 011", F.J.	19.
011 Sprays",			Ð	L anialod Biasof	.05
treamer, M.S.				fleffer, M	21.
y of Combustors 946 - June 30,				1050 21072 101 Cuper o 1946. (0012	22.
				SALING SALING	
sente pre				- MARADON	
Part				1	
				A CONTRACTOR	
				A College	+8.6



Thesis 6501 E6 Erkenbrach Air atomization of fuel oil.

