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A computational study of the effect of cross wind on the flow of fire fighting agent

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NAVAL POSTGRADUATE SCHOOL

MONTEREY, CALIFORNIA

THESIS

A COMPUTATIONAL STUDY OF THE EFFECT OF CROSS WIND ON THE FLOW OF FIRE FIGHTING AGENT

by

Alexandra Myers

June 2004

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Thesis Co-Advisor: G.V. Hobson

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A COMPUTATIONAL STUDY OF THE EFFECT OF CROSS WIND ON THE FLOW OF FIRE FIGHTING AGENT

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ABSTRACT

This research will be used to evaluate the feasibility of robotically, or remote-controlled firefighting nozzles aboard air-capable ships. A numerical model was constructed and analyzed, using the program CFD-ACE, of a fire hose stream being deflected by the influence of a crosswind, tailwind, or headwind. The model is intended to predict the reach of the fire hose stream, indicate the distribution pattern, and estimate the volume of fire fighting agent available at the end of the stream. Preliminary results for a two fluid cross flow model have been obtained.

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I. INTRODUCTION

A. BACKGROUND

The Navy is presently attempting to reduce the cost of operating and maintaining the fleet, and manpower is a major area under scrutiny. There is great concern with the concept of reduced manpower and its effect on the damage control capabilities aboard ship. In order to reduce manpower the ability to automate ship systems is essential.

Within damage control automation the technologies that are being sought and implemented are fixed automatic firefighting systems, improved damage control communications, self-reconfiguring systems, flood control systems, and fixed boundary cooling. It is believed that automated firefighting systems will be beneficial to controlling fires since the crucial time to achieve control is within the first 3-5 minutes of the start of the fire. Automated systems activate more quickly than a human response team.

In order to obtain optimal systems and procedures for ships with reduced manpower the Office of Naval Research (ONR) is sponsoring two Naval Research Laboratory programs, the Damage Control Automation for Reduced Manning (DC-ARM) and Integrated Survivability Fleet Evaluation (ISFE). In September 1998, the first successful demonstration of a 35% reduction in damage control manpower on the test platform, ex-USS Shadwell (LSD 15), was performed. The next goal is to reach a 60% reduction in manpower. [Ref. 1]

The concept of fire extinguishment is based in the idea of the fire triangle where a fire consists of three sides of the triangle representing fuel, oxygen, and heat.

If either of the three is removed then the fire will be extinguished. Firefighting agents can achieve this through physical or chemical means. There are four basic physical means: 1) Smothering the fire, where the fuel and air are separated. This concept is behind foam extinguishers. 2) The removal of heat. Agents with high heat capacities can provide the means for heat removal by absorbing the heat from the fire. 3) Forcing a high velocity gas over the flame to separate the fuel and air or the fuel and heat. 4) Flame radiation blockage, where the agent absorbs thermal radiation between the surface of the fuel and the flame for liquid or solid fuels. Fire extinguishment via chemical means occurs by using an agent that interferes with the chemical reaction that sustains combustion. [Ref. 2]

Presently, the Navy uses a variety of firefighting agents including Halon 1301, Halon 1211, Aqueous Film-Forming Foam (AFFF), CO_2, and potassium bicarbonate powder (PKP). Water mist systems are the likely candidate for future ships. Halon 1301 is used in enclosed spaces by gas-phase catalytic interruption of combustion reactions. Halon 1211 is used for streaming applications such as fires in engines that result from the pooling of fuel when an aircraft engine does not start, as well as large three-dimensional cascading flight deck fires. AFFF is 6% Foam and 94% water. It is used to extinguish two-dimensional pool fires. AFFF, CO_2, and PKP are all used within portable extinguishers for first-response fire fighting. [Ref. 2]

Halon has been found to damage the ozone layer and production of this material was halted on 31 December 1993 due to international treaties and US legislation. The Navy

maintains a strategic reserve of Halon since it is mission critical for the Navy. [Ref.3] Therefore, it is necessary for the Navy to find a suitable replacement where Halon is used. No substance with the same qualities that Halon possesses has been found. However, there are a few possibilities for Halon 1301 replacement. One being fine-aerosol generation, which is where a solid propellant is burned and a fine, fire-fighting aerosol is released. It retains the same fire-fighting capability as Halon 1301 with better weight and space requirements. Fine-aerosol generation is not preferable because it is difficult to manage the "high temperature of the burning propellant and the non-clean-agent residue that can be both toxic to humans and corrosive to shipboard systems". [Ref. 4, p.109] Another possibility for replacement is heptafluoropropane (HFP). The drawback for using HFP is a greater requirement for space and weight than Halon 1301. A third possibility is hydrofluorocarbons such as HFC-227 (FM200). FM200 can be utilized in occupied as well as unoccupied spaces. However, its drawbacks include the inability to be piped over long distances and it has a low boiling point. [Ref. 5] The final possibility is water mist technology. The only replacement for Halon 1211 being considered is halocarbons. There is still a great deal of testing that needs to be done to determine toxicity to humans and possible environmental impacts. [Ref.6]

Water mist systems are defined by producing a droplet size smaller than 500 microns. These types of systems are a desirable alternative because they use a small amount of water and are lower in weight. [Ref. 7]

> Water mist systems can use as little as 2% of the water normally used by conventional water systems. Water mist extinguishes fires quickly, cools radiant heat from surrounding equipment to eliminate any risk of re-ignition, and permits almost immediate access to an affected space. Overall, water mist is probably the best alternative to halon and all other gaseous systems. [Ref. 8,p. 1]

> On May 26, 1981, an EA-6B aircraft crashed into several parked F-14's while attempting to land on the USS NIMITZ (CVN-68). As a result of the crash and the ensuing fire and explosions, 14 persons were killed and 42 injured. [Ref 9, p. 1]

The Board of Investigation found the level of disaster to be due to several deficiencies within equipment and techniques. This drove the Naval Research Laboratory to investigate improvements to firefighting tactics on flight deck fires by evaluating various firefighting techniques such as different firefighting agents, the application of those agents under various wind conditions, and their ability to extinguish a variety of fires. Specifically, seawater versus AFFF, 1-1/2 in. and 2-1/2 in hand lines in various wind conditions, effective range of monitors (water cannons having flow capabilities from 500 to 12,000 gpm) in various wind conditions, running fuel fires, and debris pile fires. [Ref. 9]

One of the greatest inhibiting factors is being able to "approach an area where the incident heat flux level exceeds the protection of a fire proximity suit."[Ref. 10, p. 1] Due to these inhibitions, Remote Controlled Firefighting Platforms (RCFP) have been developed and tested. Another advantage of the RCFP is the ability to approach a fire from the downwind side. Two prototypes

produced by the Naval Surface Weapons Center are the Firecat, a battery-powered, tracked vehicle, and the Firefox, a gasoline driven, skid steer vehicle. After testing the vehicles in various wind conditions and comparing the results to hand lines it was found that in certain situations the hand lines extinguished the fire more quickly due to the ability to employ a firefighting technique of rapidly moving the stream of foam back and forth. However, the RCFP's proved to be an asset in certain situations. "The vehicles were able to maneuver into close proximity (less than 8m) and extinguish or control 34 of the 45 test fires in 150 seconds or less. These fires would have been difficult or impossible for unprotected hose line crews to extinguish, particularly in a downwind approach."[Ref. 10, p. 25]

Regardless of the firefighting agent in use, wind conditions will always be a significant factor in their application on a flight deck. The amount of deflection of the agent being applied due to wind conditions is essential information in applying firefighting tactics. The amount and direction of deflection will determine how and where the firefighting agent needs to be applied for optimal extinguishing effects. Computational Fluid Dynamics (CFD) modeling can provide the insight of wind effects upon the application of firefighting agents. Many different wind conditions and options for the application of firefighting agents can be tested and evaluated at a low cost using CFD modeling. Field-testing still needs to be performed to evaluate the validity of the CFD modeling. The findings from this type of modeling and testing can be used on future flight decks of aircraft carriers with respect to

the types of deliverance methods of firefighting agents and their placement around the flight deck in order to optimize firefighting capabilities under various wind conditions. This study uses CFD modeling to evaluate the jet stream flow development and deflection at the exit of a nozzle in a cross flow of wind.

B. PROBLEM DESCRIPTION

The purpose of this study is to construct a Computational Fluid Dynamics (CFD) model to analyze a fire hose stream being deflected by the influence of a crosswind, tailwind, or headwind. The model is intended to predict the stream reach, indicate the distribution pattern, and estimate the volume of fire fighting agent available at the end of the stream. This study concentrates on a model of the flow development and deflection for various jet stream velocities and cross flows of wind near the exit of the nozzle.

CFD-GEOM, CFD-ACE, and CFD-VIEW version 2003, commercial CFD programs produced by the Computational Fluid Dynamics Research Corporation (CFDRC) were used in the construction of the model and analysis. The modeling and analysis was done on Micron Pro Desktop computer, with a 400 MHz processor, 384 megabytes of RAM and a 12-gigabyte internal hard drive.

II. METHODOLOGY AND FORMULATION

A. MODEL GEOMETRY

1. Grid Formation

The model was constructed to represent a fire hose with a jet stream of water exiting parallel to the flight deck. The model represents a small volume of area near the nozzle exit. The three-dimensional volume consists of the pipe, with a total length of 27.559-in (0.7-m), protruding into a box with a total length of 39.37-in (1.0-m). The pipe protrudes into the box a length of 7.874-in (0.2-m). The plan view of the volume without the grid is shown in Figure 1. The plan view of the volume with the grid is shown in Figure 2. Images of the volume from the side view can be found in Appendix A.

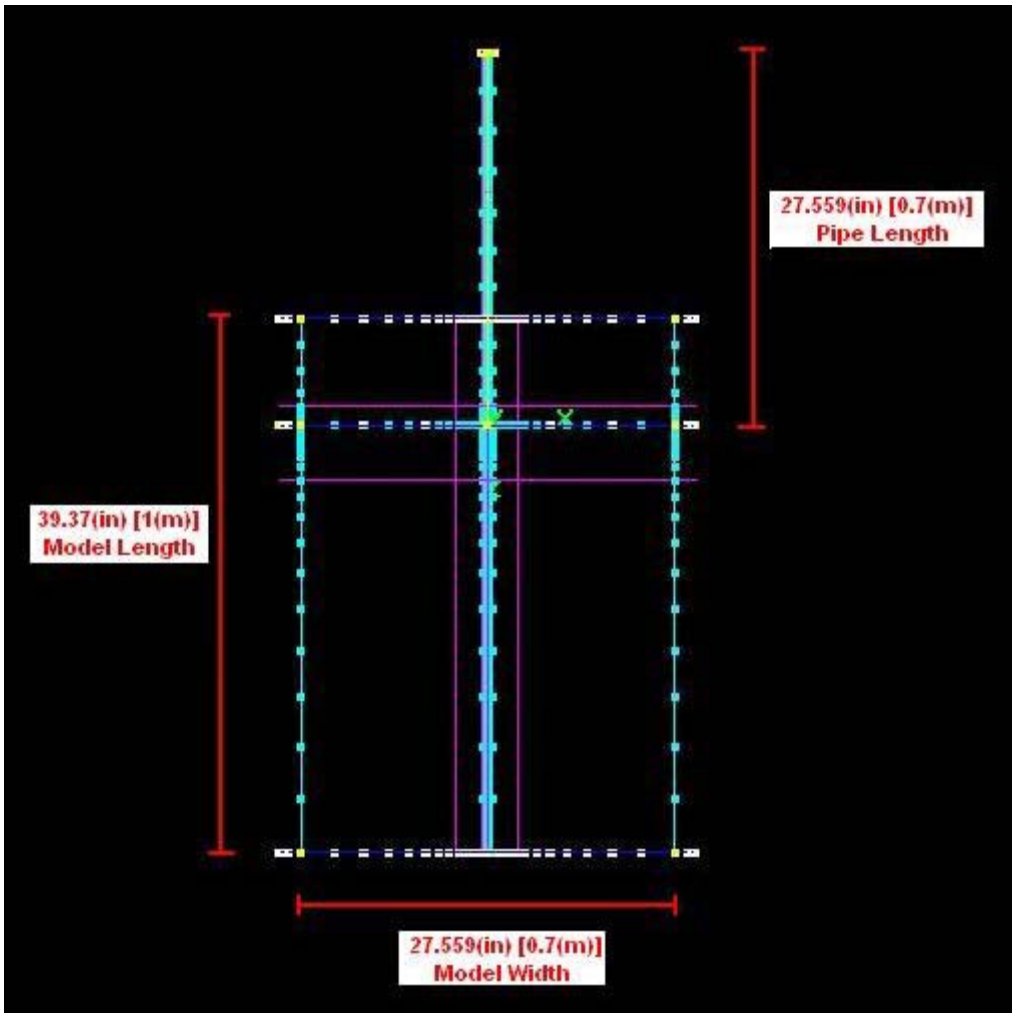

Figure 1. Plan View without Grid.

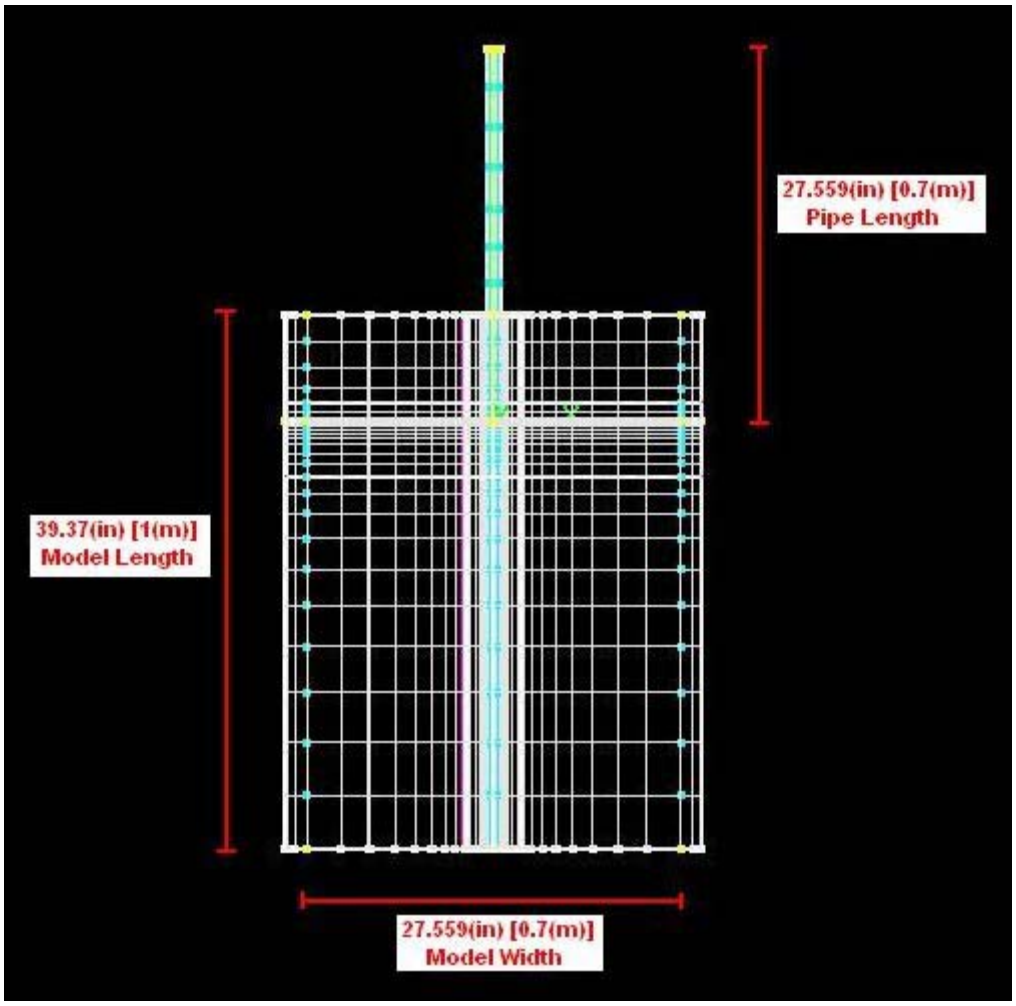

Figure 2. Plan View with Grid.

The grid spacing varies over the length of the volume and finer grid spacing is in place closer to the nozzle exit, which is necessary for the solver to resolve a solution. The grid spacing increases with increasing distance from the nozzle. The complete volume contains 16,044 cells.

The construction of the model began with a two-dimensional cross section of the volume within CFD-GEOM. The two-dimensional cross section was then extruded to create the three-dimensional volume. The complete two-dimensional cross section of the grid is shown in Figure 3.

The sides of the two-dimensional shape are representative of the flight deck, crosswind, and environment on the three-dimensional volume. The sides are rounded to maintain the grid shape except on the bottom, which is necessary to be flat as it represents the flight deck.

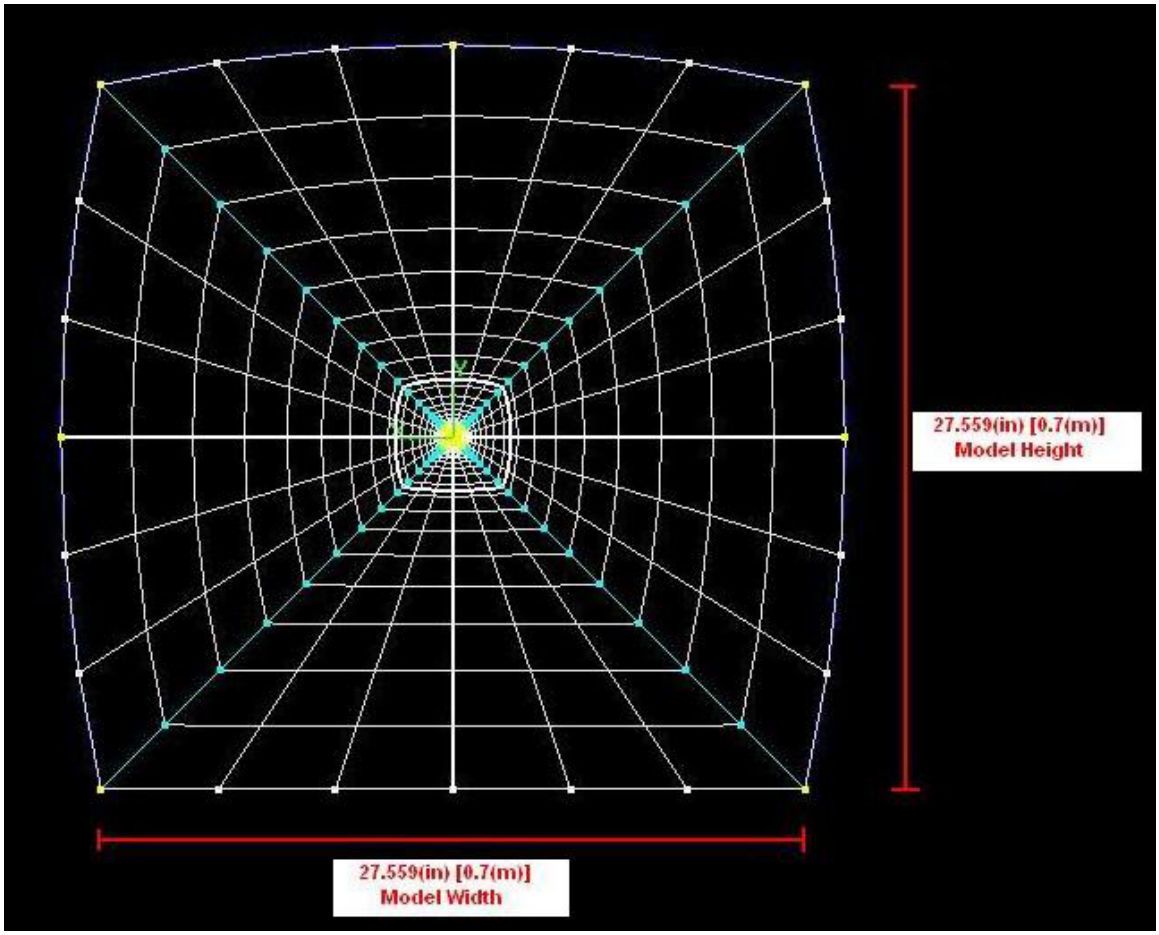

Figure 3. Two-Dimensional Cross Section.

At the center of the cross section the one-inch nozzle was constructed. A smaller box with rounded sides within the one-inch nozzle was constructed to prevent grid deformation. Figure 4 is an image of the one-inch nozzle with grid spacing.

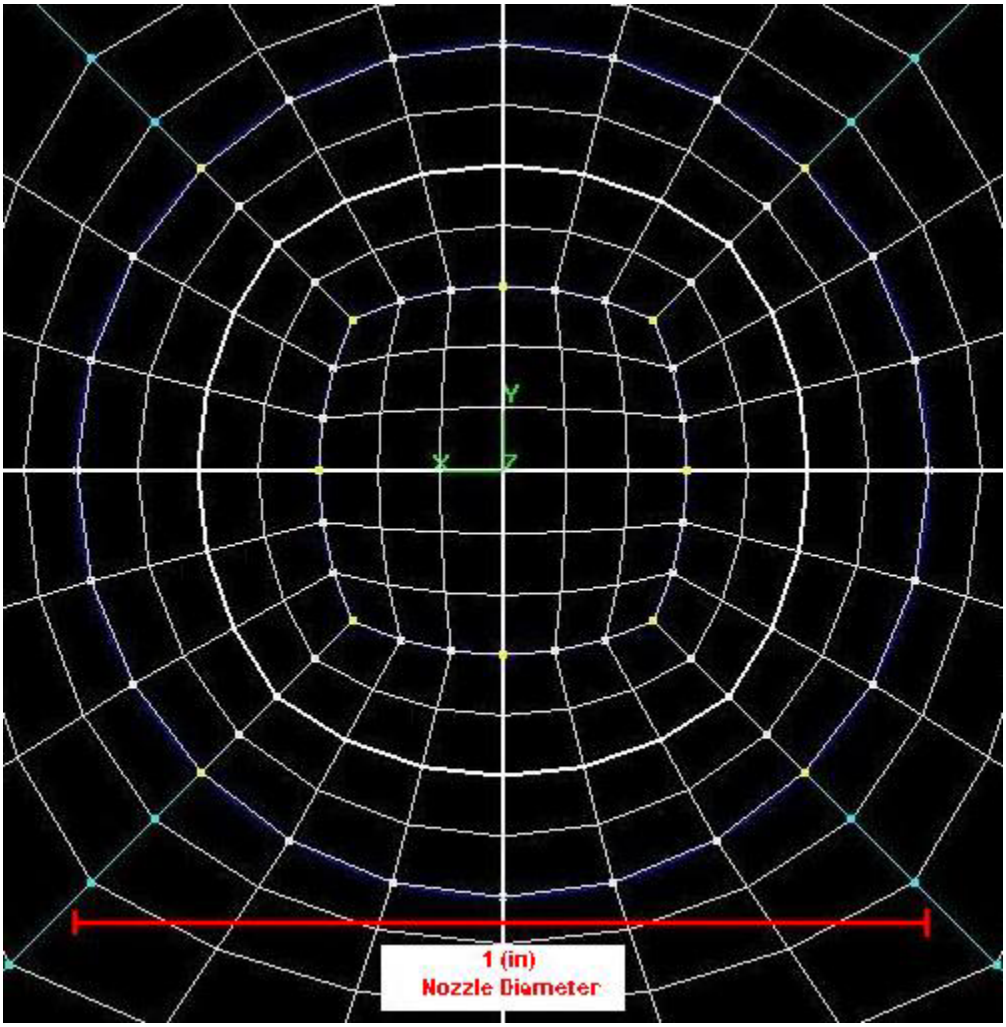

Figure 4. 1 (in) Nozzle Grid Spacing.

The same methodology was applied for creating the grid formation for the two-inch and three-inch nozzles as well as the volumes for the models with a two and three inch nozzle. The only difference between the models is the size of the pipe and nozzle diameter. Images of the grids for these nozzles are located in Appendix A.

2. Boundary Conditions

The boundary conditions have been set on the volume to simulate the environment at the jet stream exit from the nozzle. The crosswind flows in the positive x-direction

across the flight deck. The wind velocity was set to constant velocities of fifteen [7.716 (m/s)] and thirty [15.64 (m/s)] knots for various trials. The crosswind boundary is highlighted in red in Figure 5.

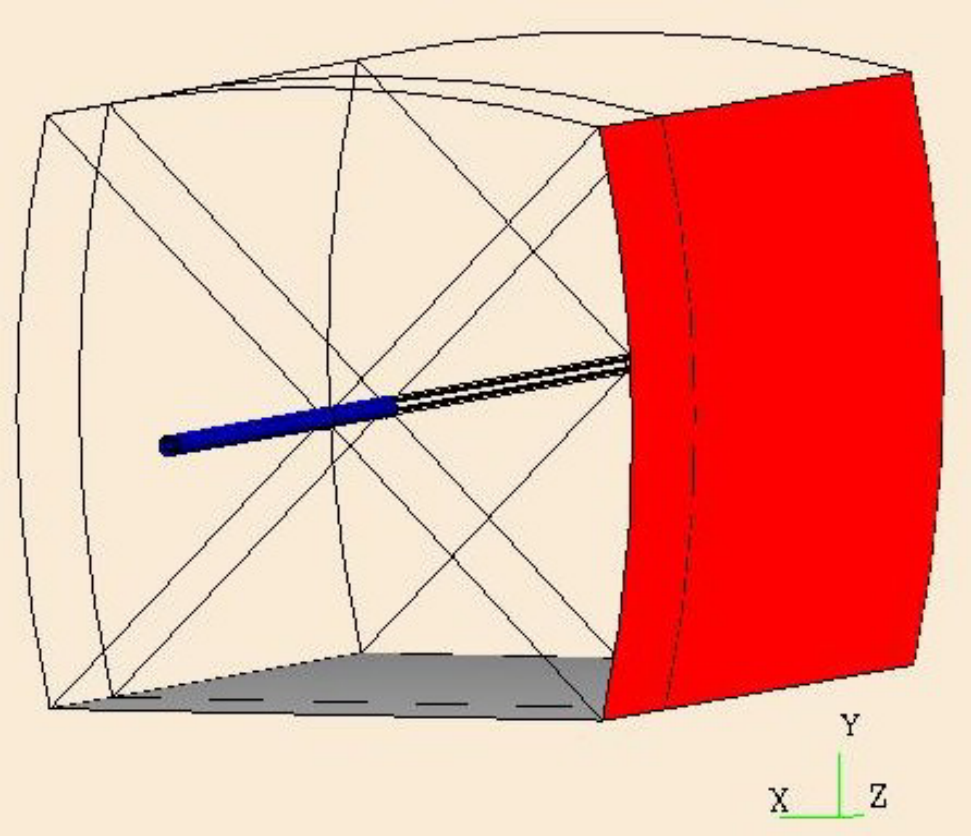

Figure 5. Crosswind Boundary in Red.

All of the other boundaries were set to a constant environmental pressure of 101325 (N/m^2). These boundaries are highlighted in red in Figure 6.

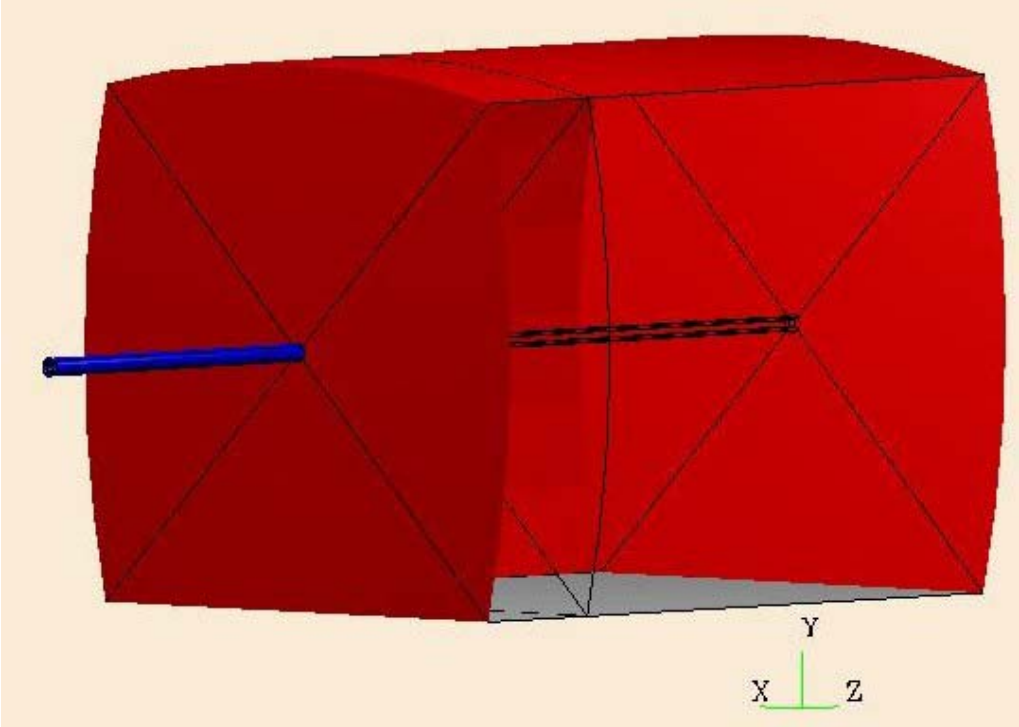

Figure 6. Environmental Pressure Boundaries in Red.

3. Volume Conditions

The Volume of Fluid Problem Type under transient conditions was set within CFD-ACE. The volume of fluid representing the jet stream exiting the nozzle is water with a constant density of 1000(kg/m^3) and a constant kinematic viscosity of 1E-6 (m^2/s). It is exiting the nozzle into a volume of air with a constant density of 1.1614 (kg/m^3) and a constant dynamic viscosity of 1.846E-5 (m^2/s). Refer to Appendix B for sample inputs to CFD-ACE from Trial 1.

B. CALCULATIONS

The analysis done on the models used flow rates of 125, 250, and 500 (GPM). The flow rates were then used to calculate velocities for the jet stream at the nozzle exit. These were the inputs used for CFD-ACE. Reynolds numbers

were also calculated for each of the flow rates. These calculations are summarized in Table 1.

$$V = \frac{m}{\rho A} \qquad \mathrm{Re} = \frac{\rho V D}{\mu}$$

V=velocity

m=mass flow rate

ρ=density

A=area

D=Diameter

μ=viscosity

NOZZLE DIAMETER (in)	NOZZLE DIAMETER (ft)	AREA (ft^2)
1	0.0833	0.0055
2	0.1667	0.0218
3	0.2500	0.0491

	FLOW RATE (GPM)		
	125	250	500
	FLOW RATE (ft^3/s)		
NOZZLE DIAMETER (in)	0.2785	0.5569	1.1139
	VELOCITY (ft/s)		
1	25.6565	51.3131	102.6261
2	6.4141	12.8283	25.6565
3	2.8507	5.7015	11.4029
NOZZLE DIAMETER (in)	VELOCITY (m/s) (1m/s)=(1ft/s)(0.3048)		
1	7.8201	15.6402	31.2804
2	1.9550	3.9101	7.8201
3	0.8689	1.7378	3.4756
NOZZLE DIAMETER (in)	Reynolds Numbers		
1	169510.3057	339020.6115	678041.223
2	84755.15287	169510.3057	339020.6115
3	56503.43525	113006.8705	226013.741

Table 1. Jet Stream Velocity and Reynolds Numbers.

III. RESULTS

For each variable such as the velocity components and pressure, residuals are calculated for each cell, weighted by the cell volume, and then summed over the entire model. A residual reduction of five orders of magnitude is desired to ensure that solution convergence is achieved. [Ref. 11] The velocity components in the first trial achieved a residual reduction of six orders of magnitude and the pressure variable achieved a residual reduction of five orders of magnitude. Similar residuals were achieved for all subsequent trials in this study. The residuals achieved for the first trial is depicted in Figure 7, and is representative of the results for all the trials in this study.

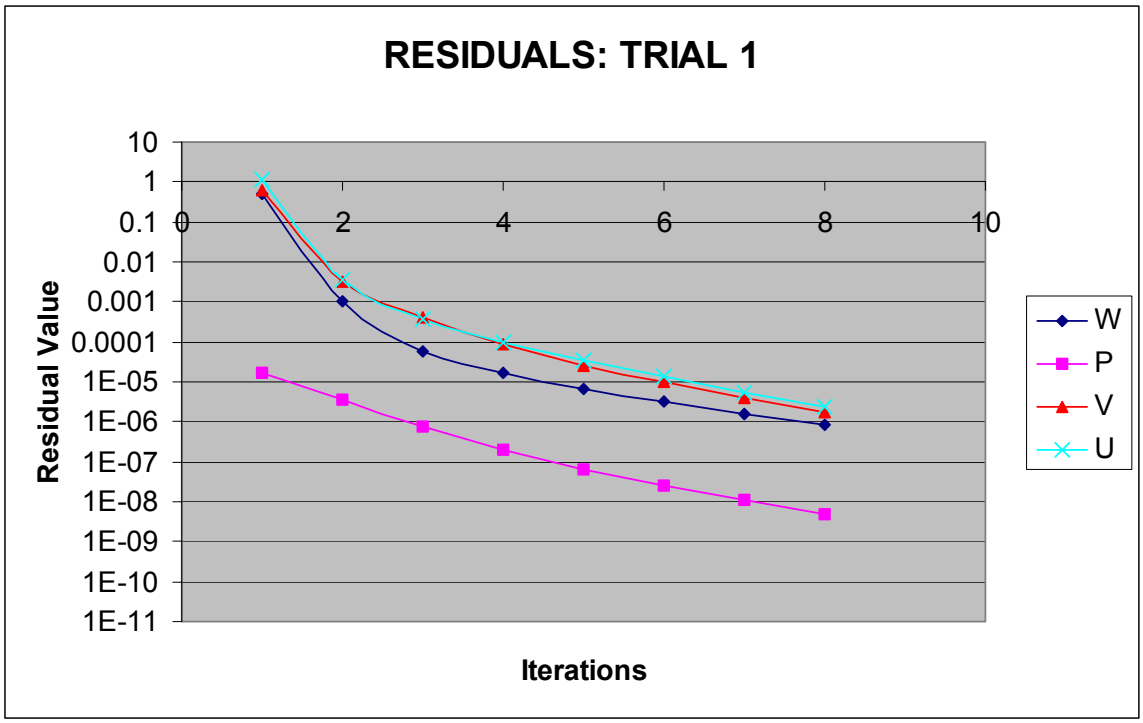

Figure 7. Residuals: Trial 1.

Trial 1 was tested for a 1 (in) nozzle with a flow rate of 125 (GPM) in cross winds of 15 and 30 knots. The amount of deflection of the jet stream due to a cross wind of 15 (kts) is displayed in the plan view of the volume of fluid flow in Figure 8. A side and angled view are also displayed in Figures 9 and 10 to obtain an improved visualization of the wind effects.

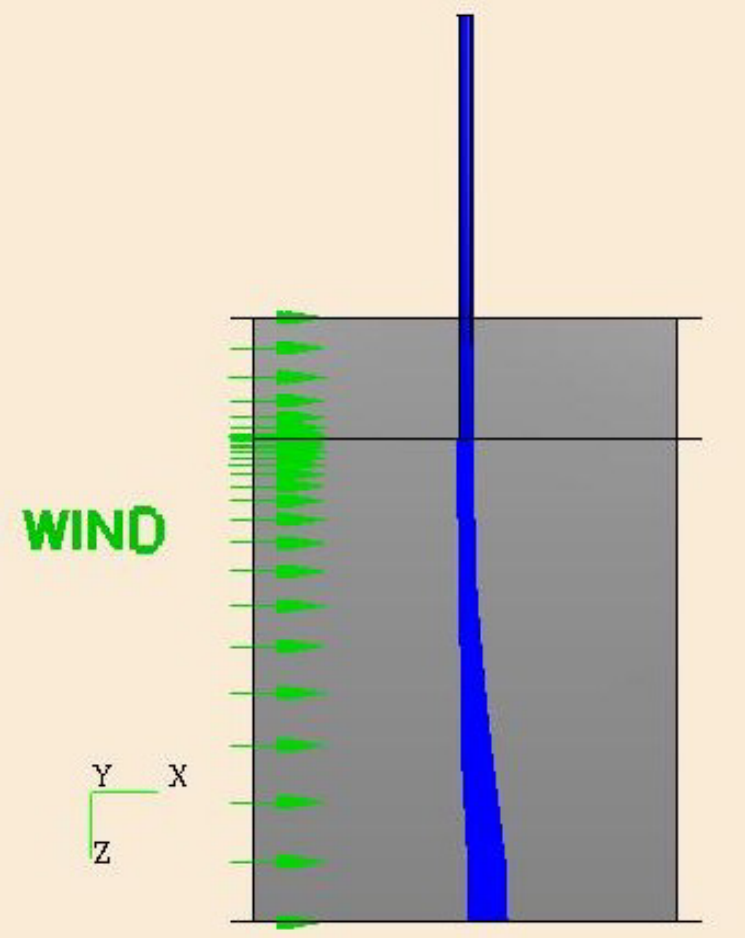

Figure 8. Plan View: Trial 1, 15 (kts).

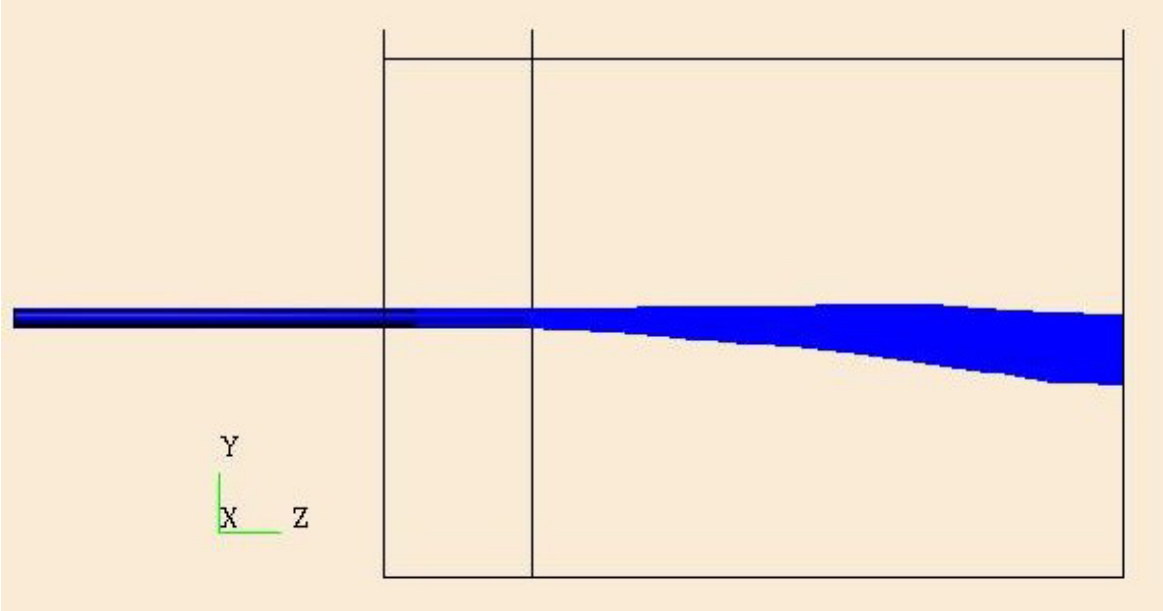

Figure 9. Side View: Trial 1, 15 (kts).

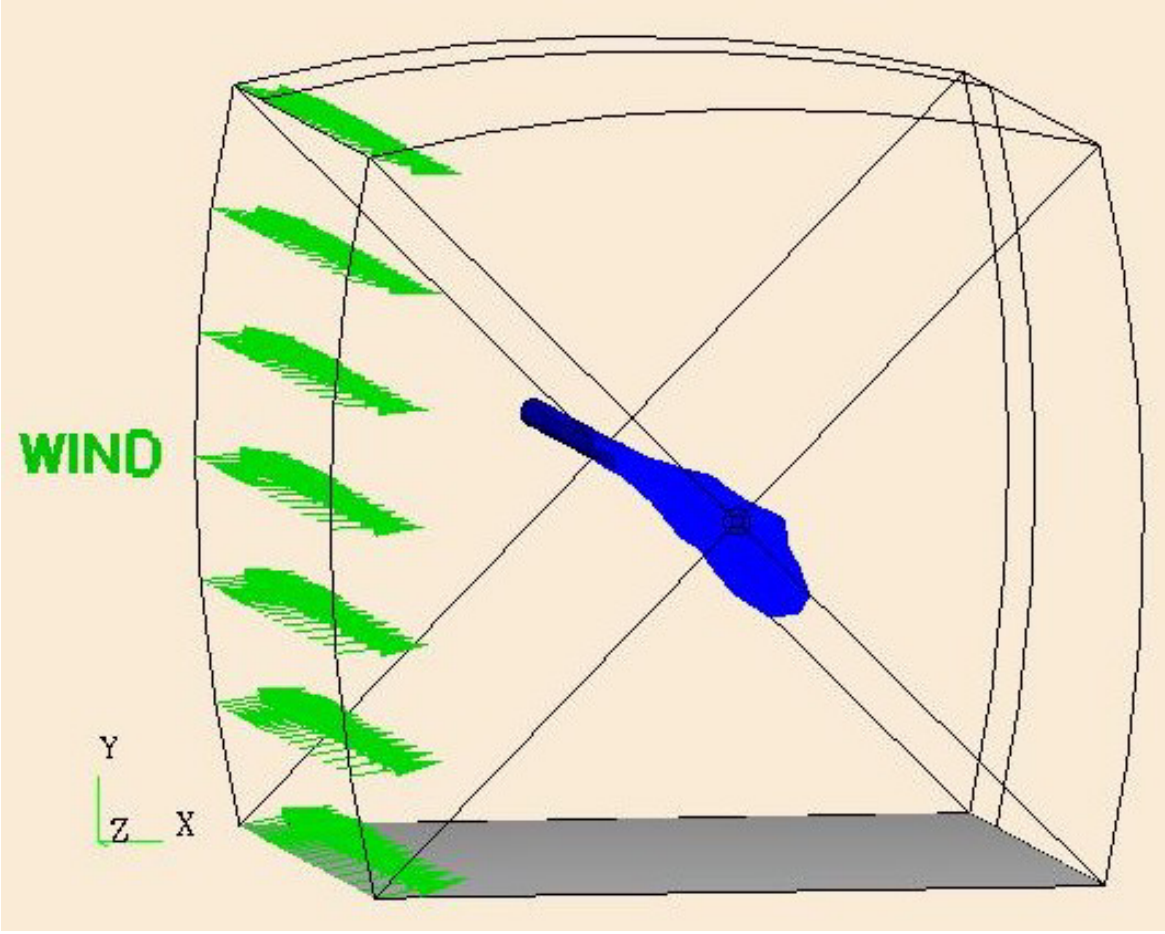

Figure 10. Angled View: Trial 1, 15 (kts).

The wind effects for 30 (kts) of wind are displayed in the following Figures 11, 12, and 13.

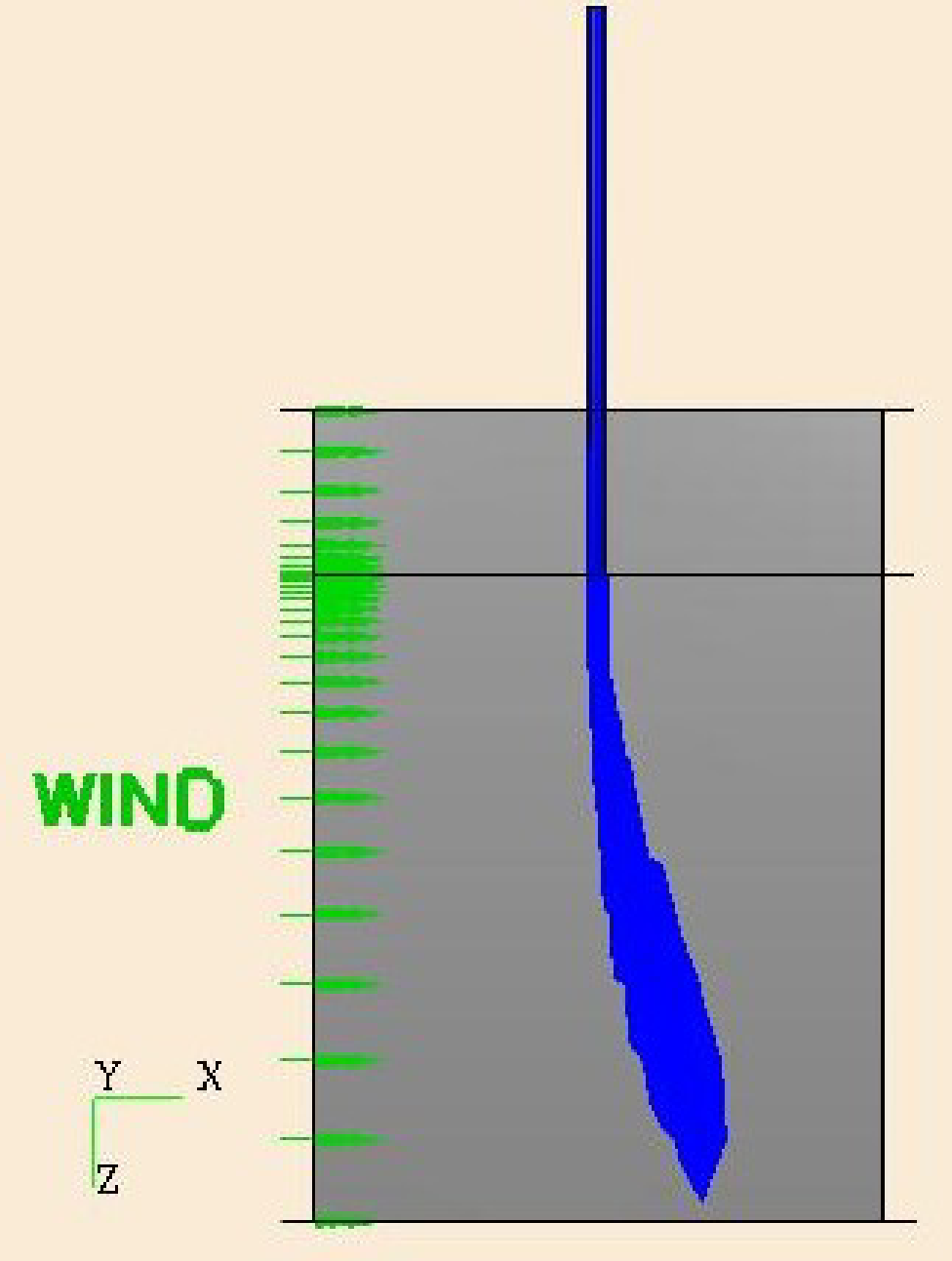

Figure 11. Plan View: Trial 1, 30 (kts).

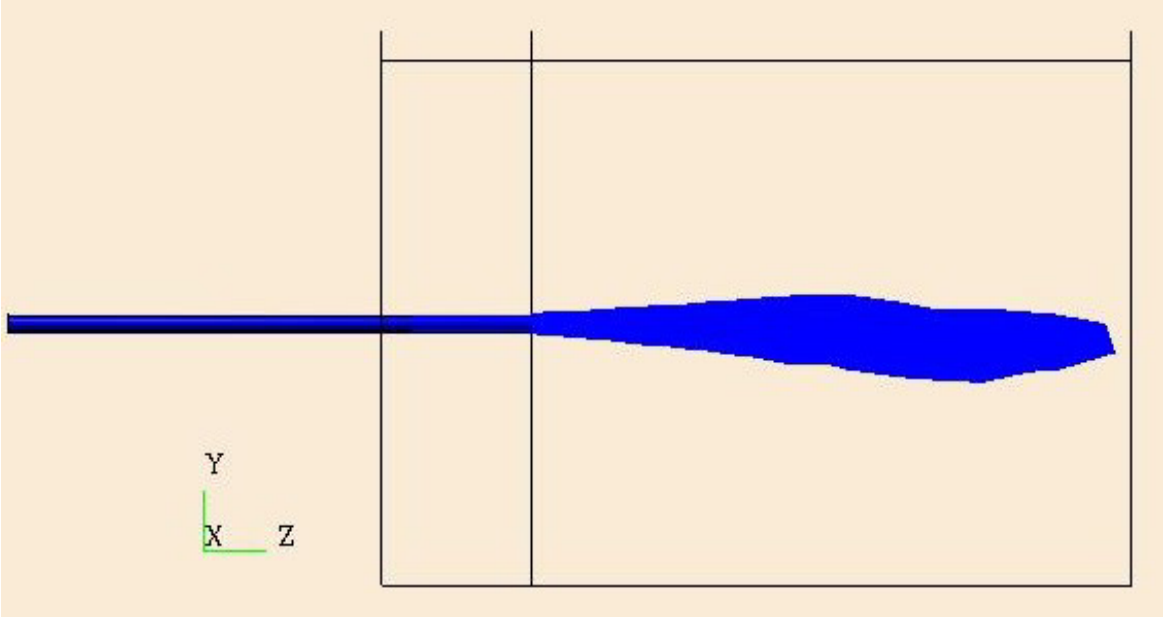

Figure 12. Side View: Trial 1, 30 (kts).

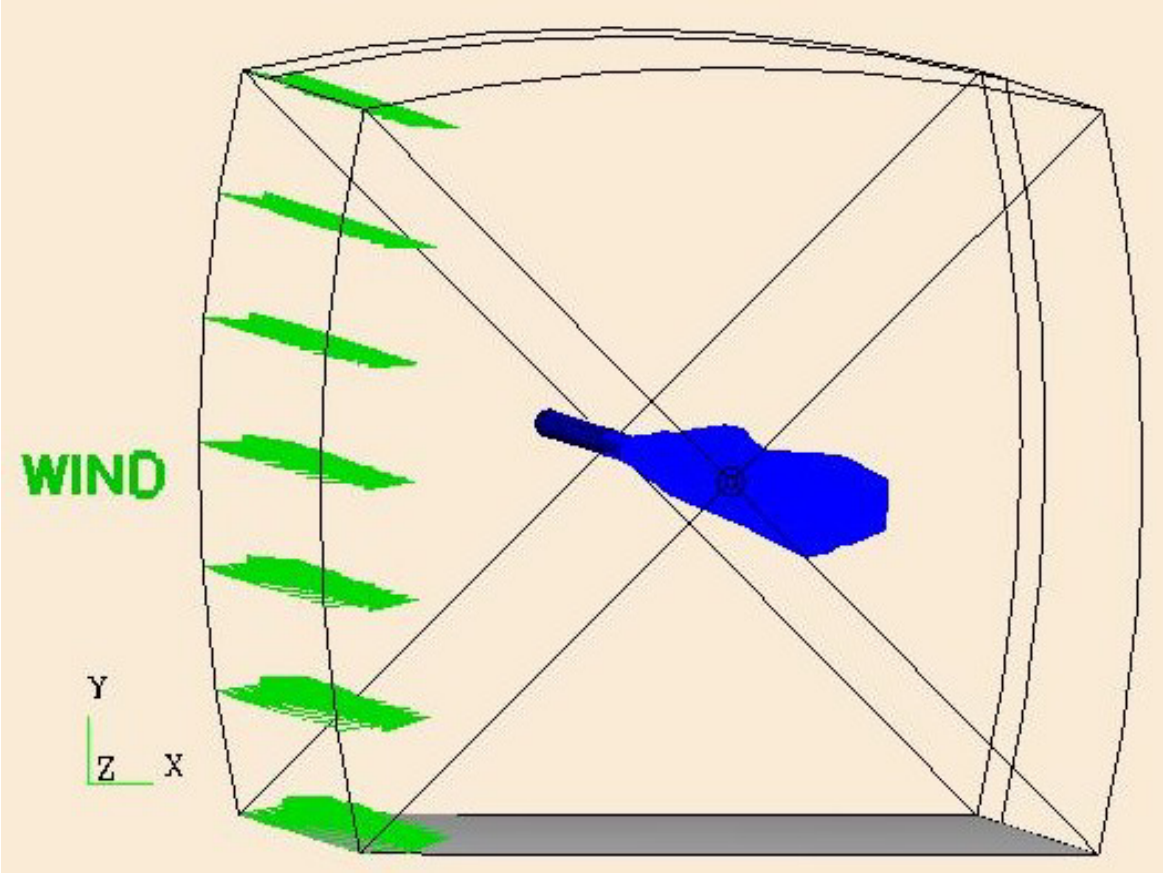

Figure 13. Angled View: Trial 1, 30 (kts).

Each of the flow rates 125 (GPM), 250 (GPM), and 500 (GPM) were tested exiting from 1 (in), 2 (in), and 3 (in) nozzles with cross winds of 15 (kts) and 30 (kts). The images displaying the resulting jet stream deflection due to the wind effects for the remaining trials are located in appendices C through J. They are summarized in Table 2.

Appendix	Trial	Flow Rate (GPM)	Wind Speed (kts)	Nozzle Size (in)
C	2	125	15, 30	2
D	3	125	15, 30	3
E	4	250	15, 30	1
F	5	250	15, 30	2
G	6	250	15, 30	3
H	7	500	15, 30	1
I	8	500	15, 30	2
J	9	500	15, 30	3

Table 2. Summary of Trials.

As is to be expected greater deflection and wind effects can be seen in the tests with higher wind speeds and larger flow rates.

IV. CONCLUSIONS

The results from this study display the wind effects on stream deflection at the fire hose nozzle exit. Field-testing needs to be performed to validate the model results.

The modeling performed in this study is computationally intensive and each trial took from two to three days to evaluate the mean stream deflection near the nozzle exit. Prediction of flow patterns and the available volume of fire fighting agent at the end of the stream would require expanding the model volume. Obtaining more detailed spray patterns would require a finer mesh within the volume. Both expanding the model volume and creating a finer mesh within the present model would significantly increase the time and RAM necessary to obtain valid results.

Based on the results of this study, computational fluid dynamics is an effective method of determining wind effects on a fire hose stream.

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V. RECOMMENDATIONS

The following recommendations are made in continuation of this study:

- Develop a model with a larger volume to obtain the results of full stream flow and reach.
- Estimate the volume of fluid available at the end of the stream.
- Model fluids of different densities such as AFFF, which are used for firefighting.
- Model the effects of various wind angles upon the fire hose stream.
- Vary the angle of the nozzle exit into the wind to possibly reduce wind effects.
- Develop a model with a finer mesh to resolve more detailed flow patterns such as spray and atomization.

The resulting information from the continuation of this study will assist engineers and ship designers to predict the best location for fire hose nozzles and the best techniques for reducing wind effects when combating fires.

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APPENDIX A. GRID FORMATION

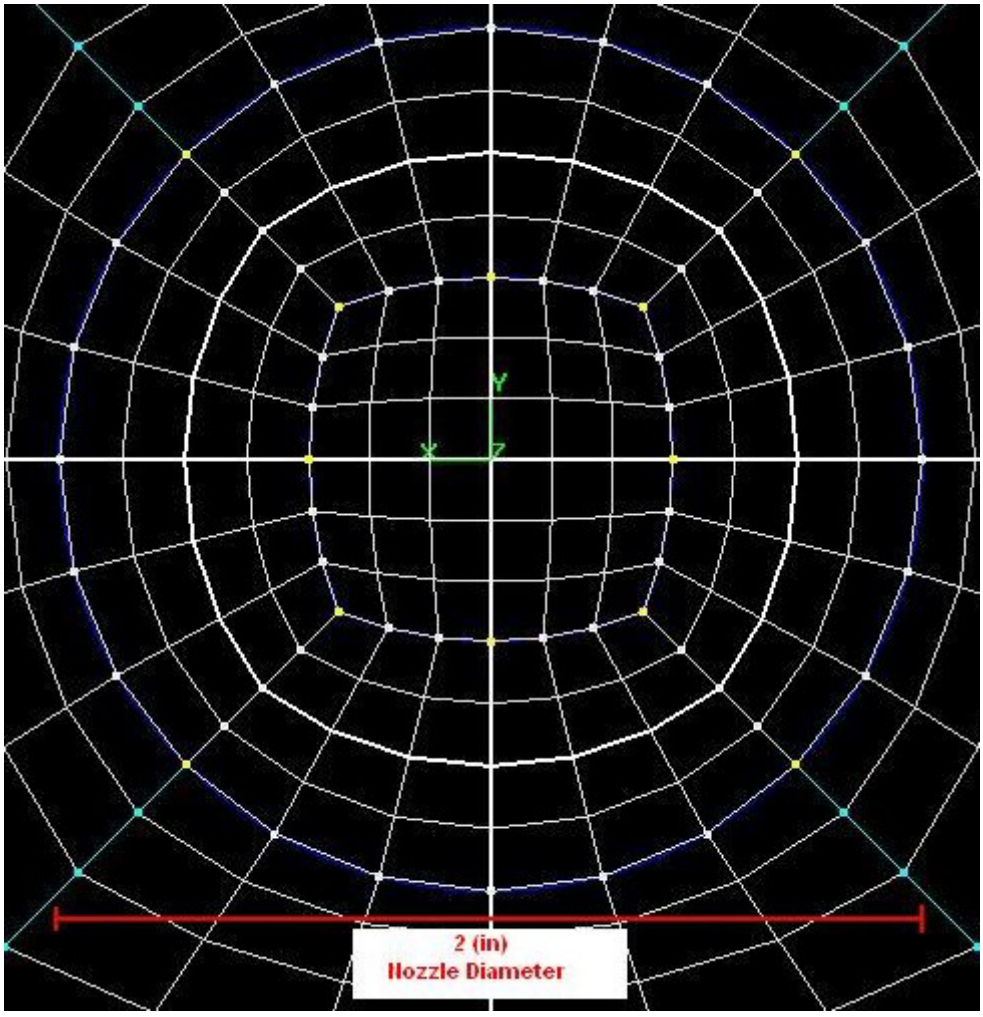

Figure 14. 2 (in) Nozzle Grid Spacing

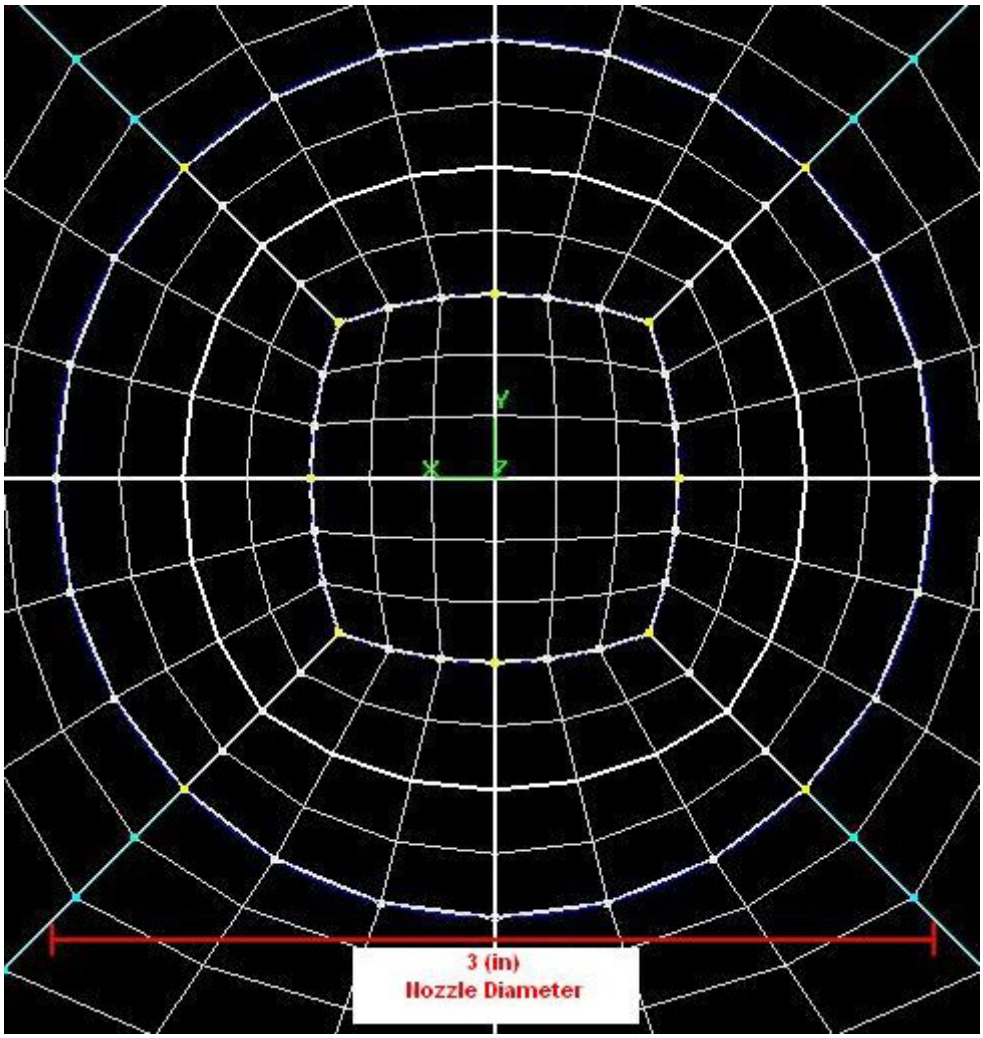

Figure 15. 3 (in) Nozzle Grid Spacing

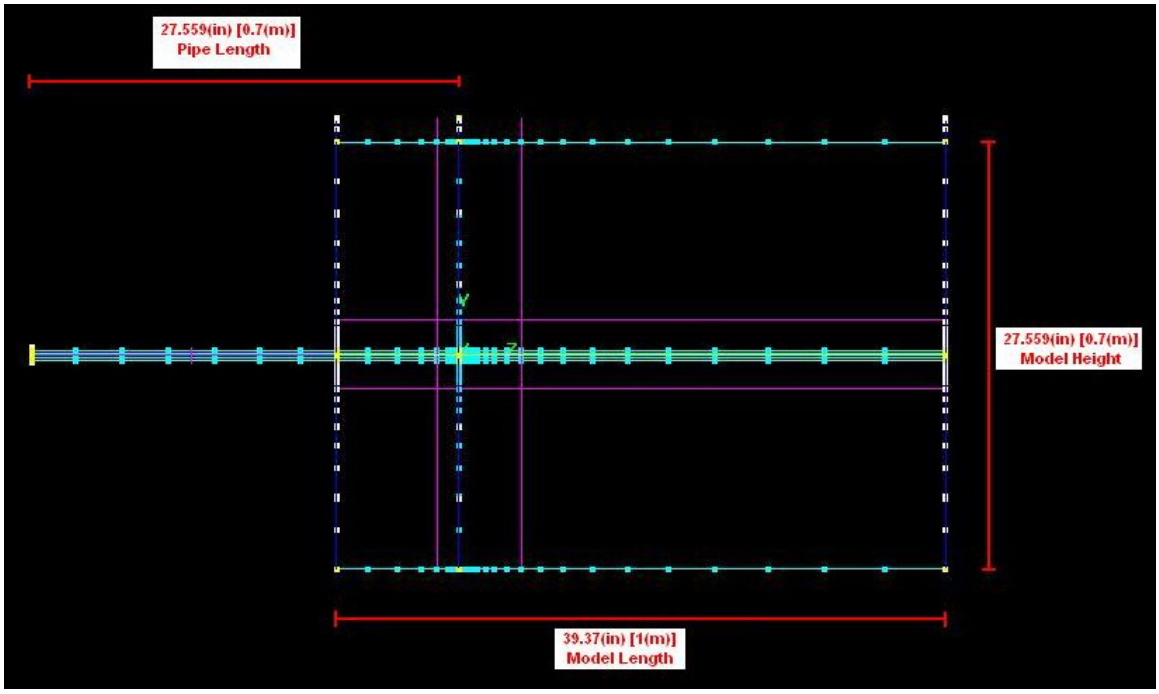

Figure 16. Side View without Grid

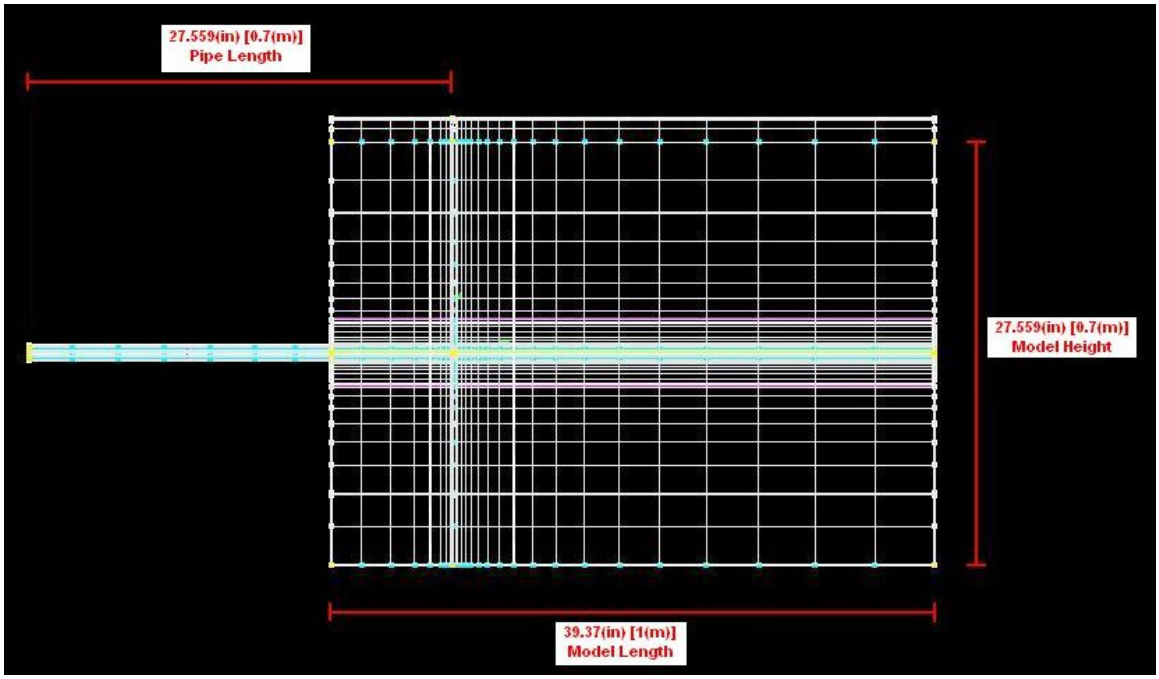

Figure 17. Side View with Grid

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APPENDIX B. SAMPLE INPUTS CFD-ACE (TRIAL 1)

PT: Modules->Flow, Free Surface (VOF)

MO: Shared->

Simulation Description->Title->Re=678041.2230

Transient Conditions->

Time Dependence: Transient

Transient Time Step: Auto Time Step, Start Time:0, End Time:10, Target

Target CFL:0.2,Minimum dt:0, Maximum dt:1, Initial dt:1E-6

Time Accuracy->Euler (1^{st} Order)

Body Forces->Gravity->Gravity in Y-Direction: Constant -9.81(m/s^2)->Ref. Density: User Specify 1.16(kg/m^3)

Flow->Flow->Reference Pressure: 101325(N/m^2)

VC: Group Fluid VCs->Activate Secondary Fluid

Phys->Density: Constant 1.1614(kg/m^3)

Fluid->Viscosity: Constant(Dynamic) 1.846E-5(kg/m-s)

VOF->Name: Water, Density: Constant 1000(kg/m^3), Viscosity: Constant(Kinematic) 1E-6(m^2/s)

BC: Group Outlet->BC Setting Mode->General

BC Type->Outlet

Flow->Flow->Subtype->Fixed Pressure

Group Pipe (Outside Volume)-> BC Setting Mode->General

BC Type->Wall

VOF->Fluid Volume Fraction->Fluid 1(Fraction=0)

Group Pipe(Protruding into Volume)->BC Setting Mode->Thin Wall

BC Type->Thin Wall->Set

BC Setting Mode->General

Group Inlet Wind-> BC Setting Mode->General

BC Type->Inlet

Flow->Flow->Subtype->Fix Vel. (Normal)

Normal Velocity: Constant 7.716(m/s)

VOF->Fluid Volume Fraction->Fluid 1(Fraction=0)

Group Flight Deck->BC Setting Mode->General

BC Type->Wall

Group Inlet Water->BC Setting Mode->General

BC Type->Inlet

Flow->Flow->Subtype->Fix Vel. (Normal)

Normal Velocity: Constant 31.2804(m/s)

VOF->Fluid Volume Fraction->Fluid 2(Fraction=1)

IC: IC Option->Constant->IC Applied: Volume by Volume

Group Volume within Pipe->BC Setting Mode->General

VOF->VOF->All Fluid 1

SC: Iter->Shared->Max. Iterations: 50

Solvers->Velocity->CGS+Pre:Sweeps=50, Criterion=0.0001

P Correction->AMG:Sweeps=50, Criterion=0.1

Relax->Inertial Relaxation->Velocities: 0.2,

Linear Relaxation->Pressure: 0.9

Adv->VOF->Activate Remove Flotsam and Jetsam->Removal Frequency: 1

OUT: Output->Output Frequency->Constant Time Step->Starting Timestep: 0, Ending Timestep: 10000000, Timestep Frequency: 20

Print->Activate Mass Flux Summary

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APPENDIX C. TRIAL 2

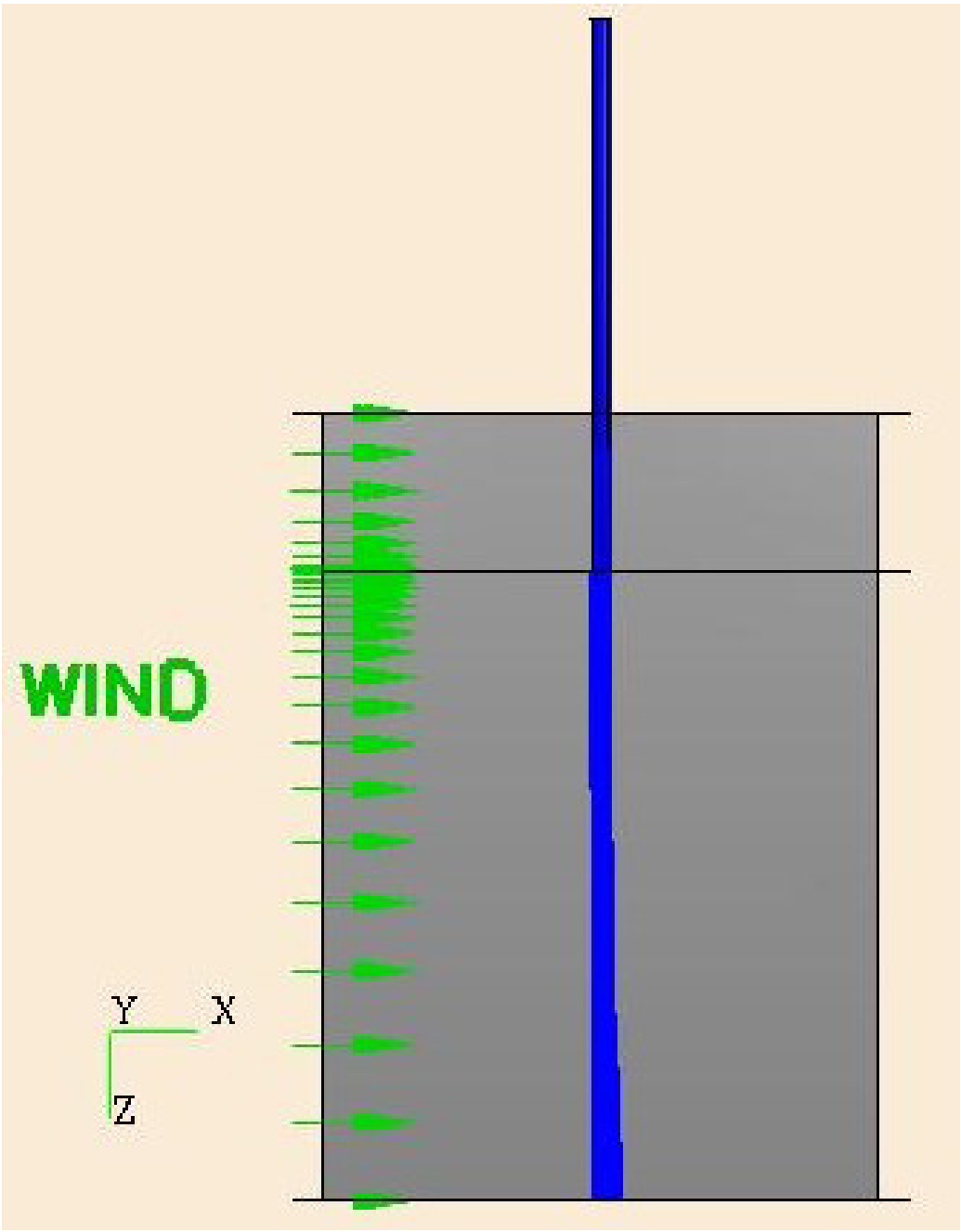


Figure 18. Plan View: Trial 2, 15 (kts).

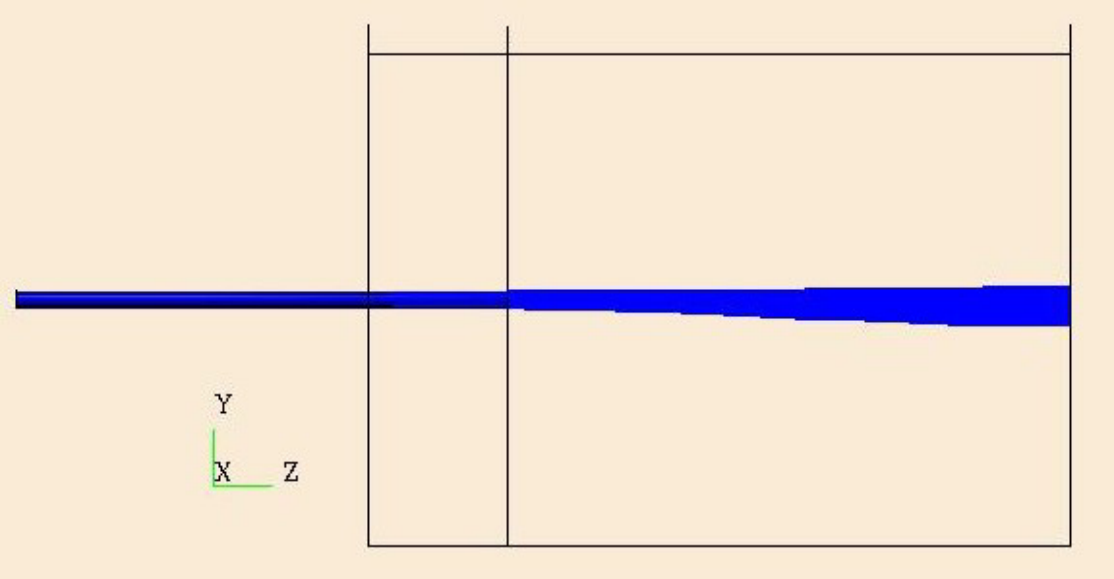


Figure 19. Side View: Trial 2, 15 (kts).

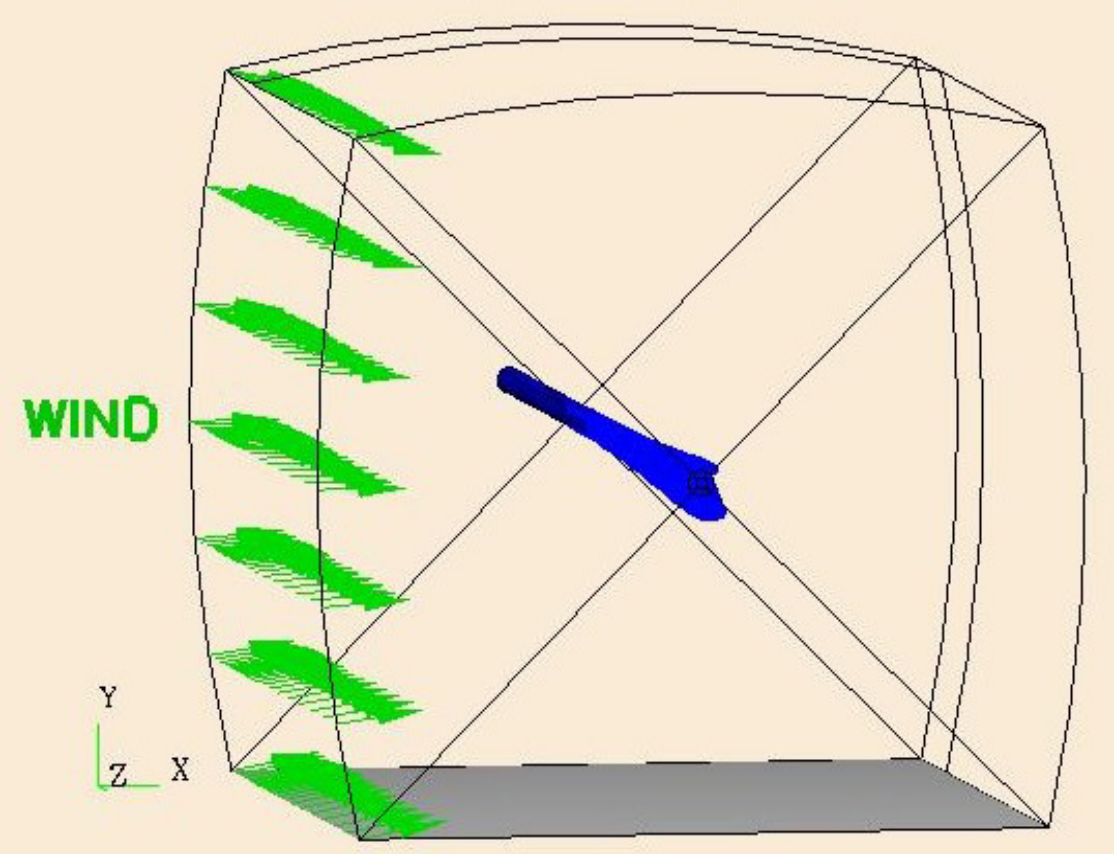


Figure 20. Angled View: Trial 2, 15 (kts).

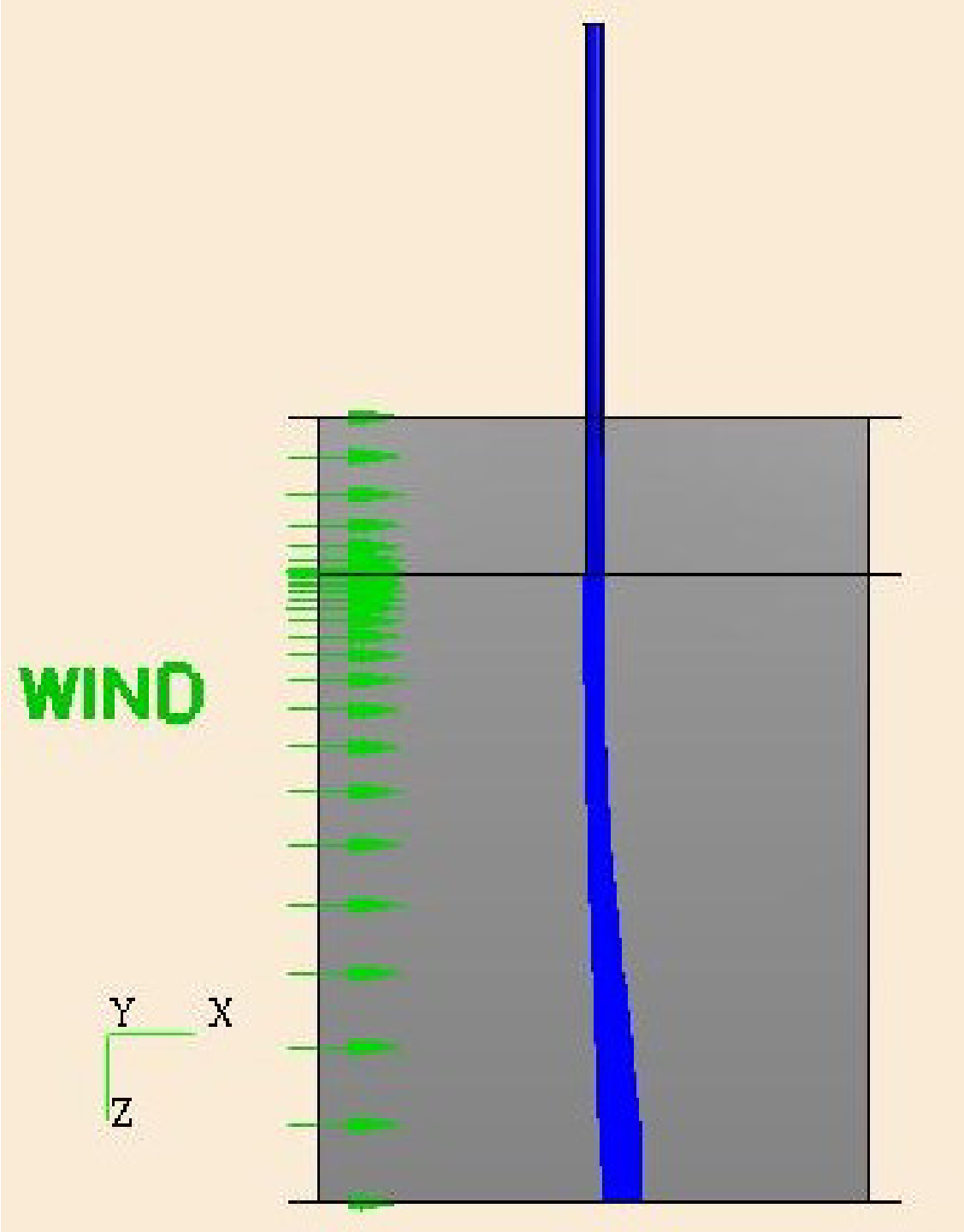


Figure 21. Plan View: Trial 2, 30 (kts).

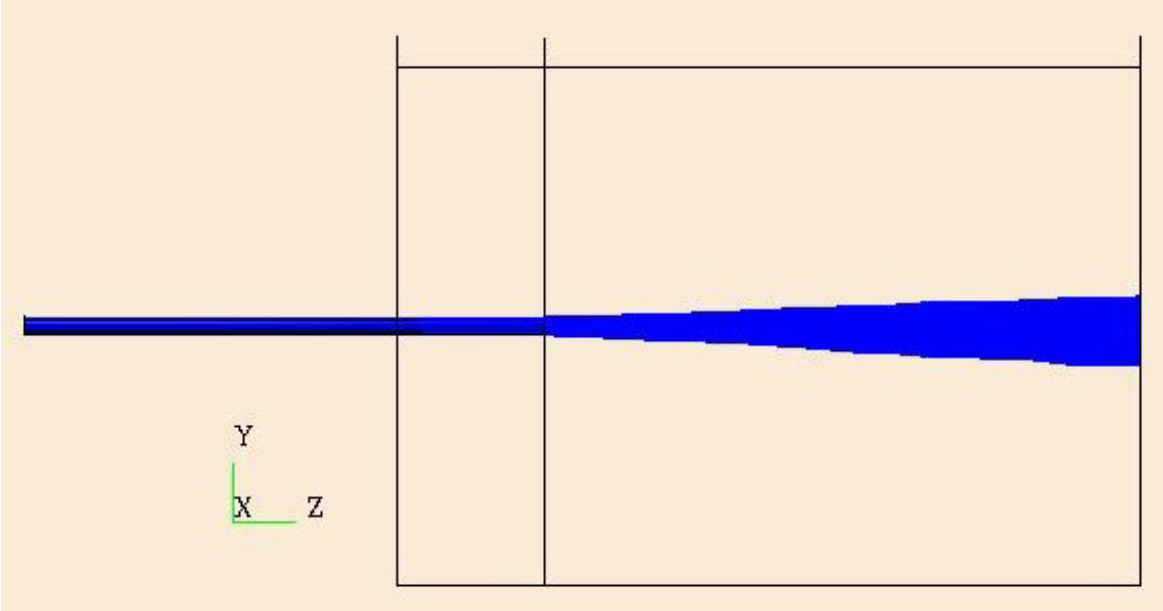


Figure 22. Side View: Trial 2, 30 (kts).

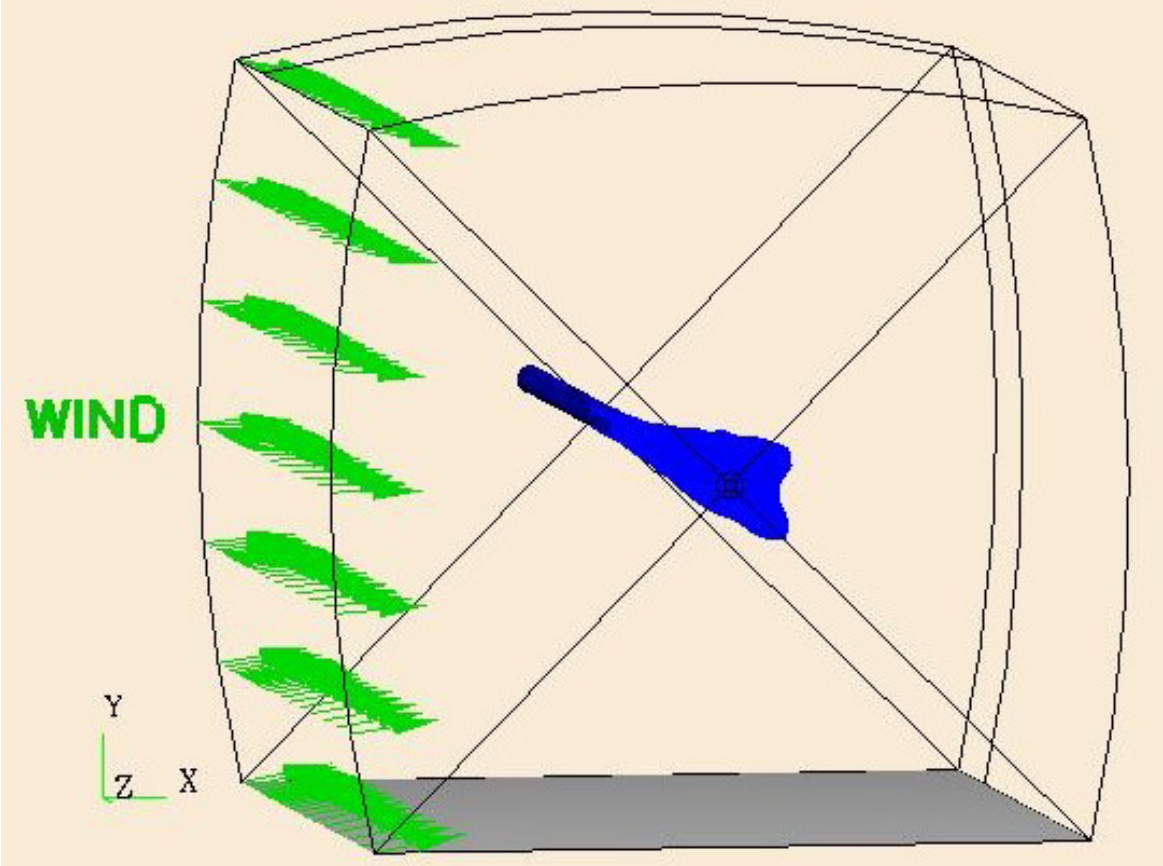


Figure 23. Angled View: Trial 2, 30 (kts).

APPENDIX D. TRIAL 3

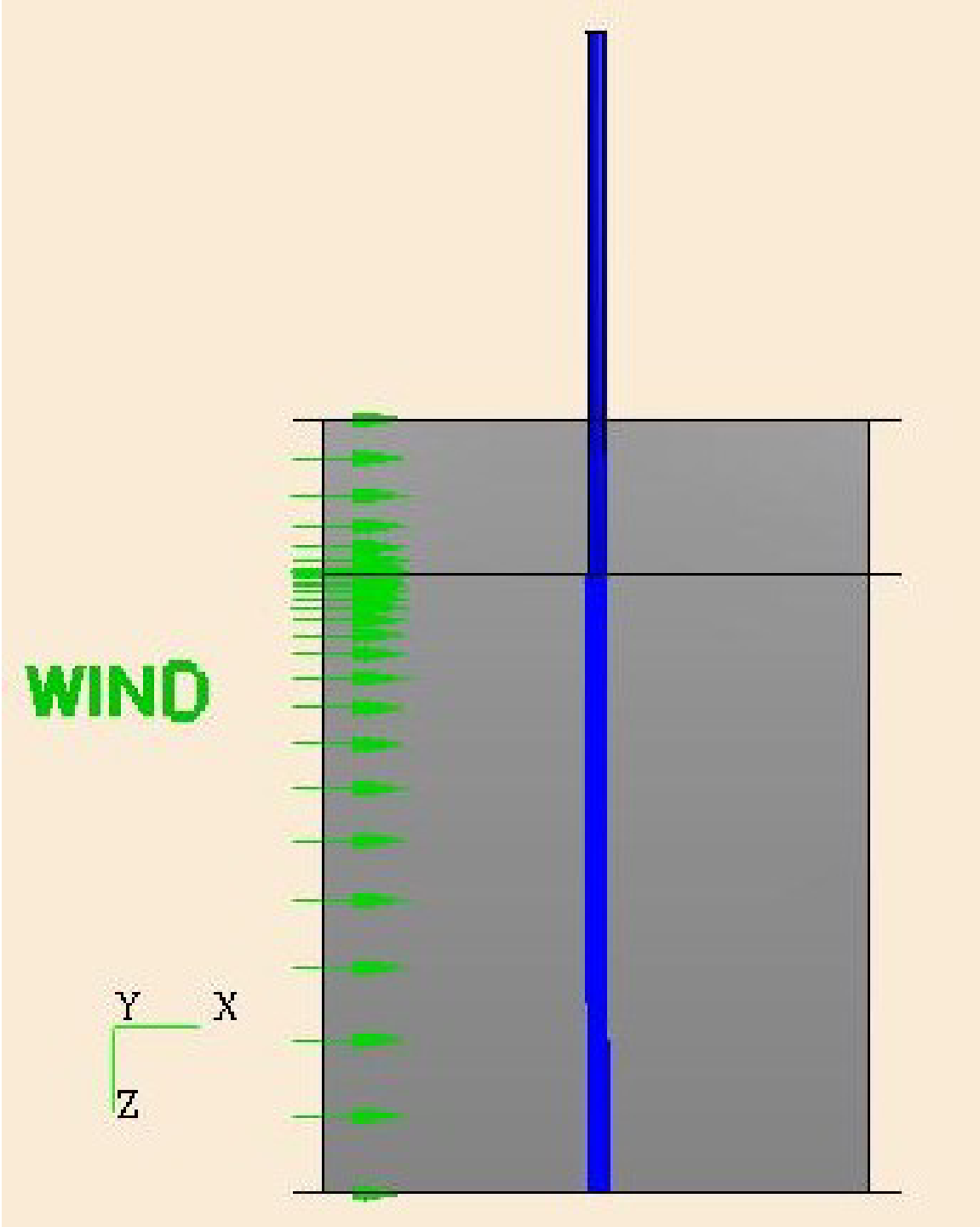


Figure 24. Plan View: Trial 3, 15 (kts).

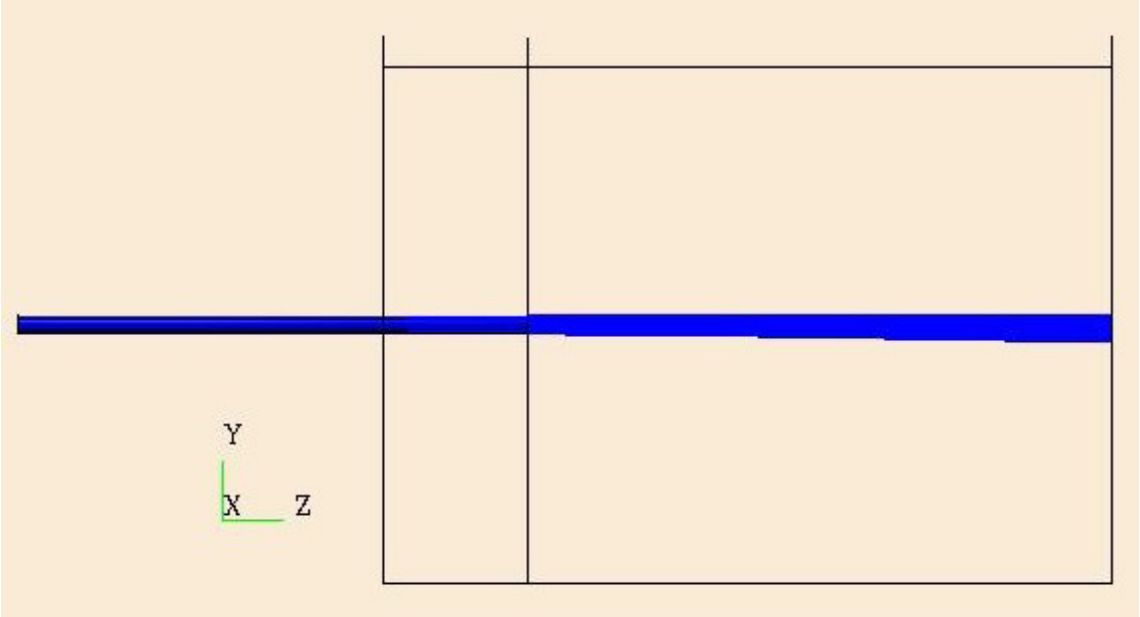


Figure 25. Side View: Trial 3, 15 (kts).

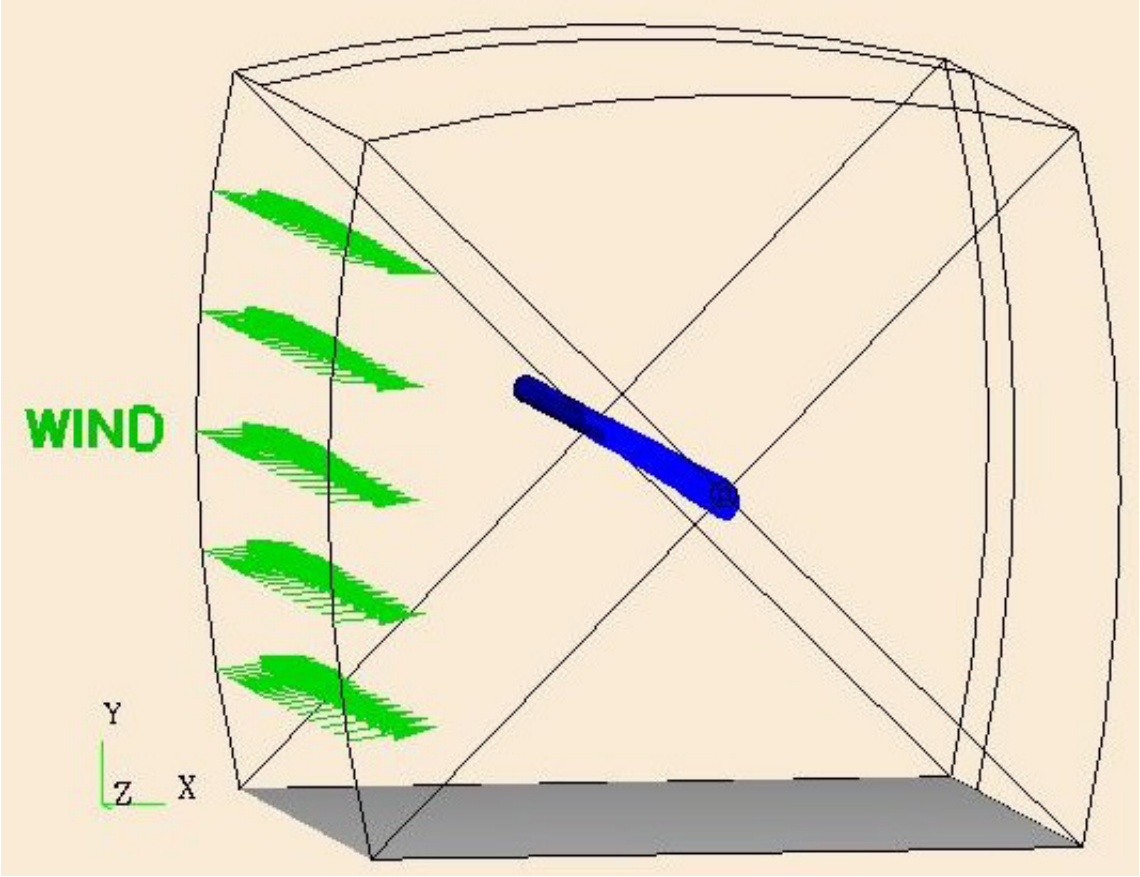


Figure 26. Angled View: Trial 3, 15 (kts).

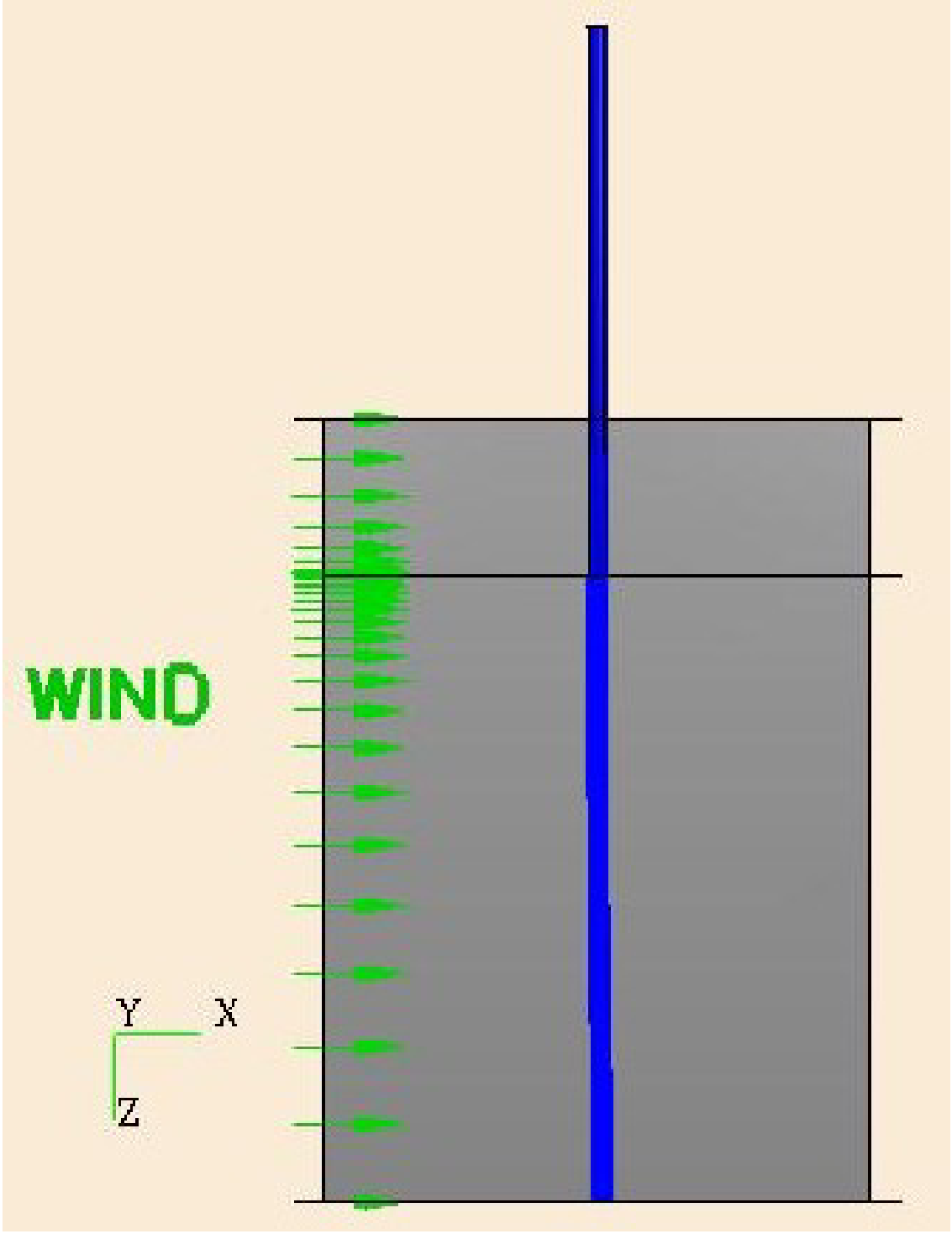


Figure 27. Plan View: Trial 3, 30 (kts).

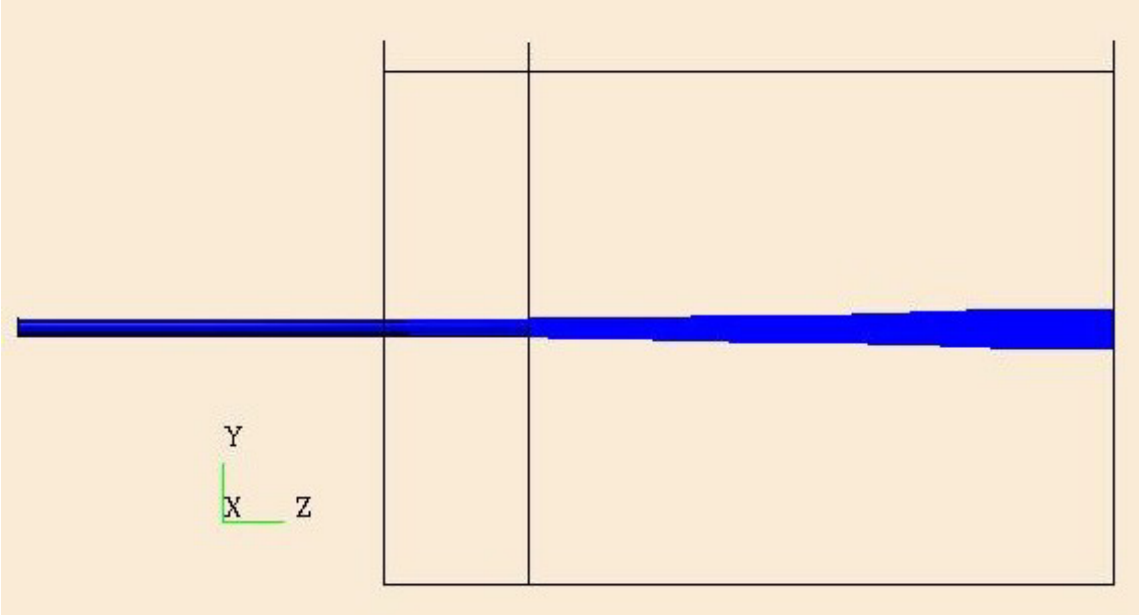


Figure 28. Side View: Trial 3, 30 (kts).

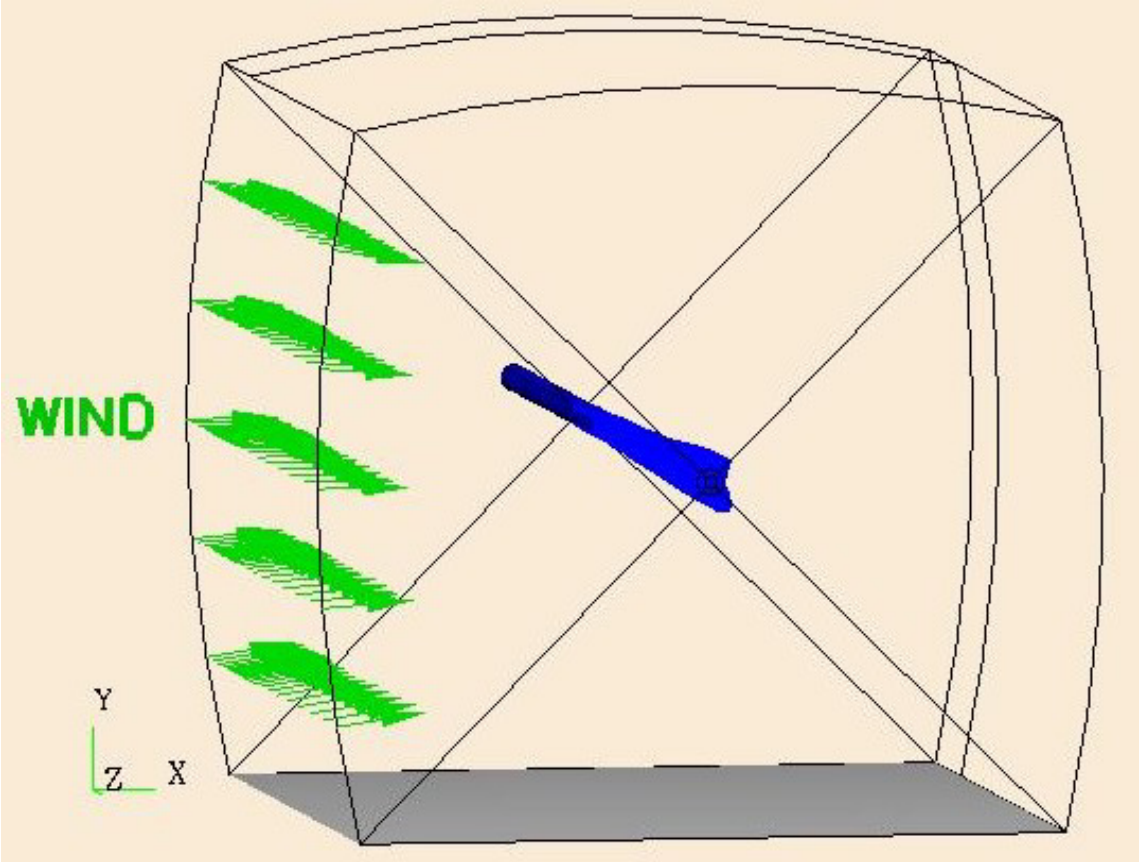


Figure 29. Angled View: Trial 3, 30 (kts).

APPENDIX E. TRIAL 4

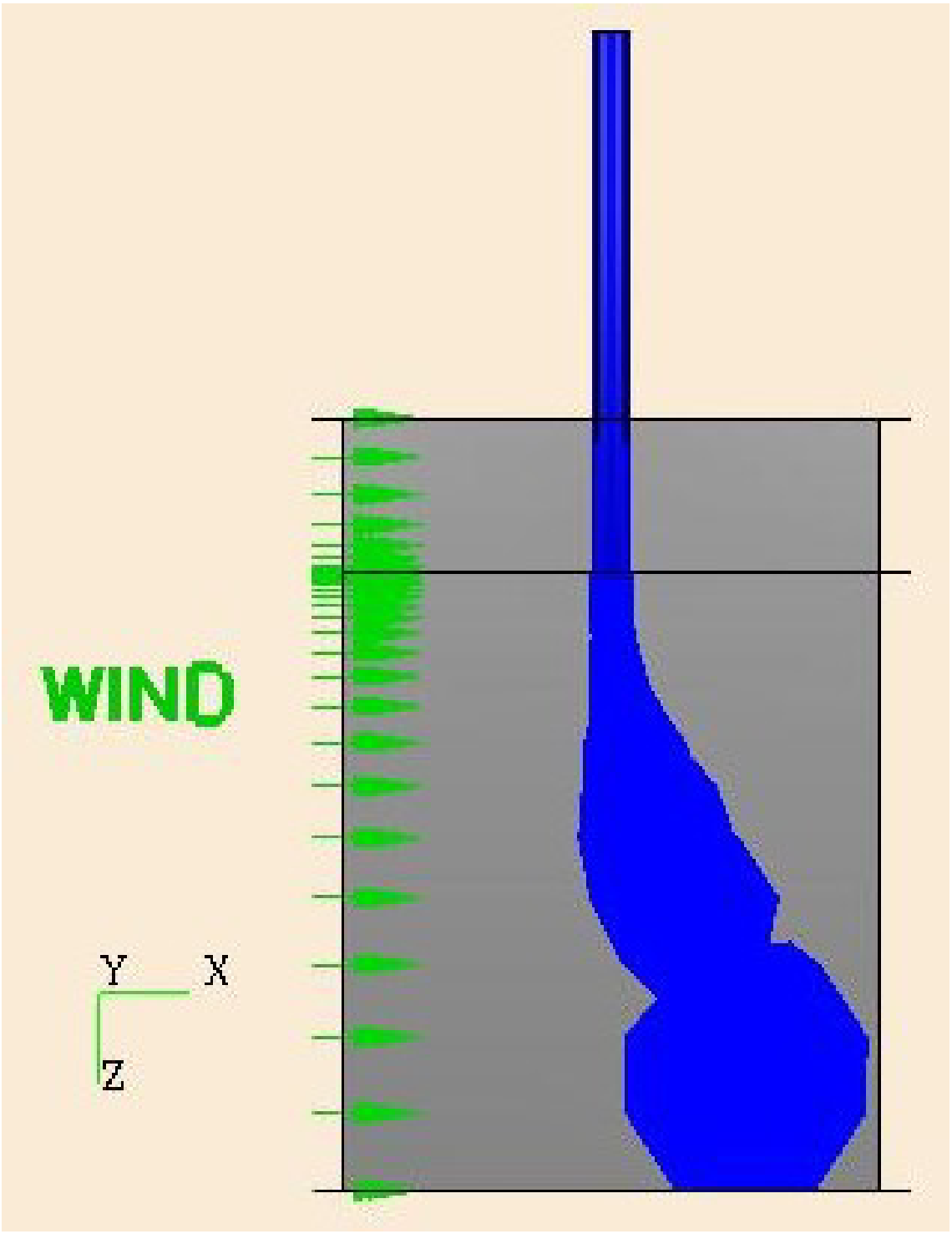


Figure 30. Plan View: Trial 4, 15 (kts).

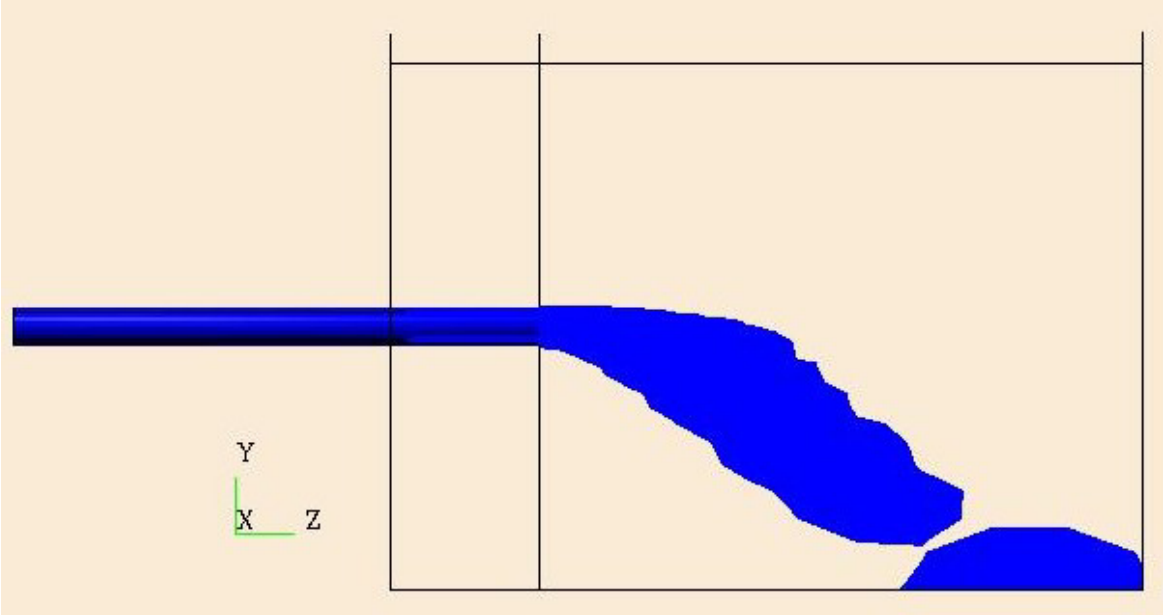


Figure 31. Side View: Trial 4, 15 (kts).

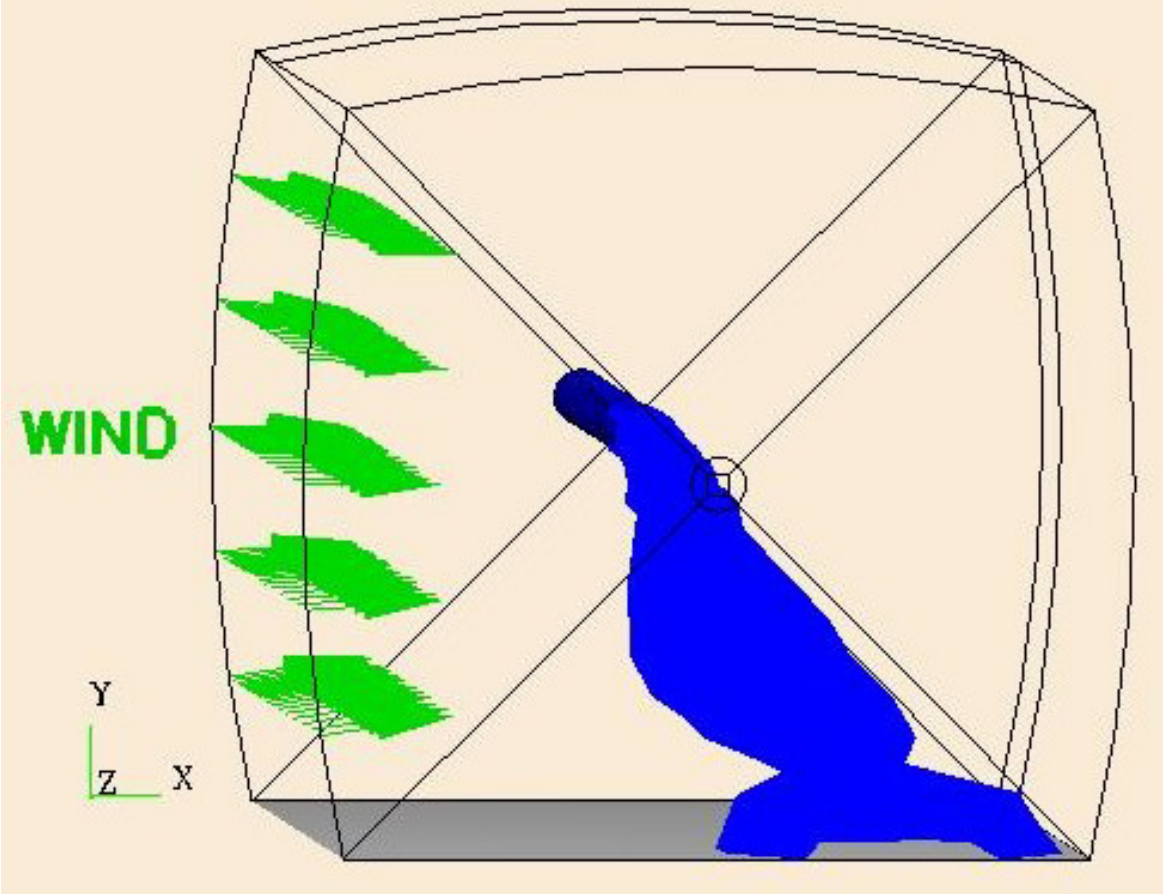
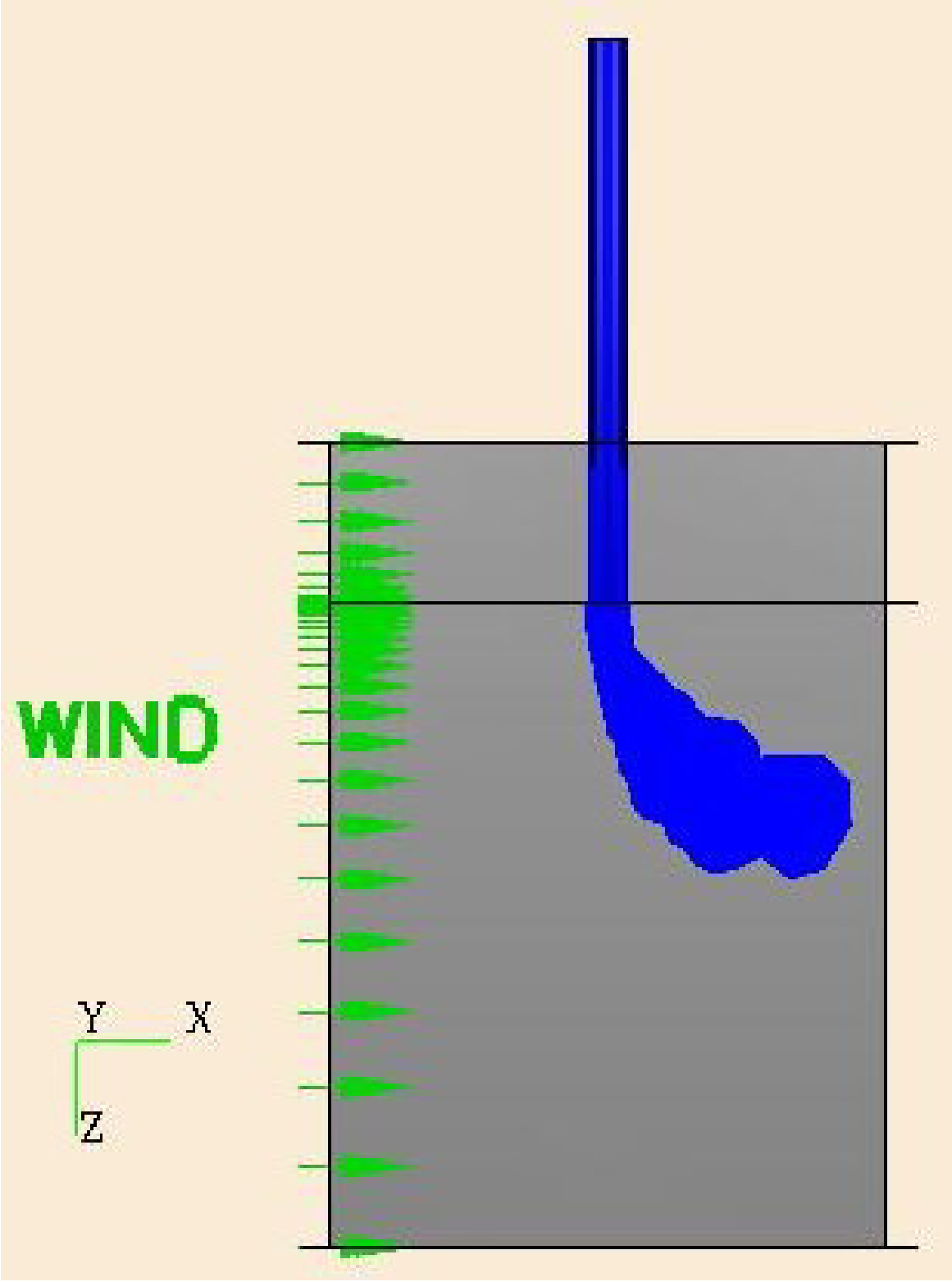


Figure 32. Angled View: Trial 4, 15 (kts).



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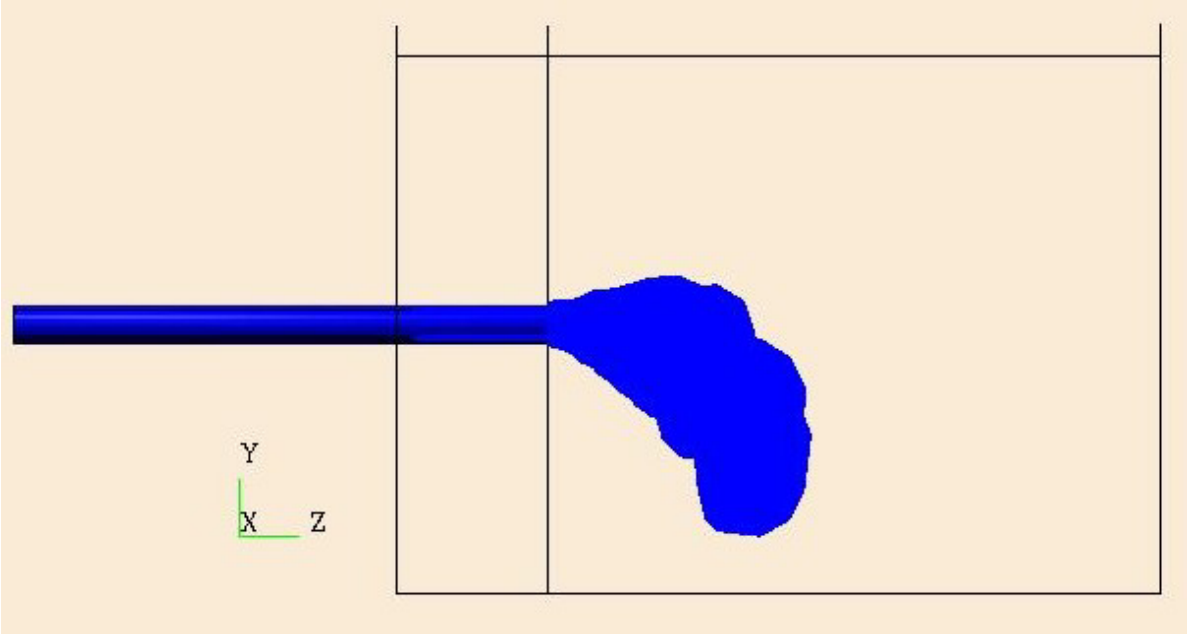


Figure 34. Side View: Trial 4, 30 (kts).

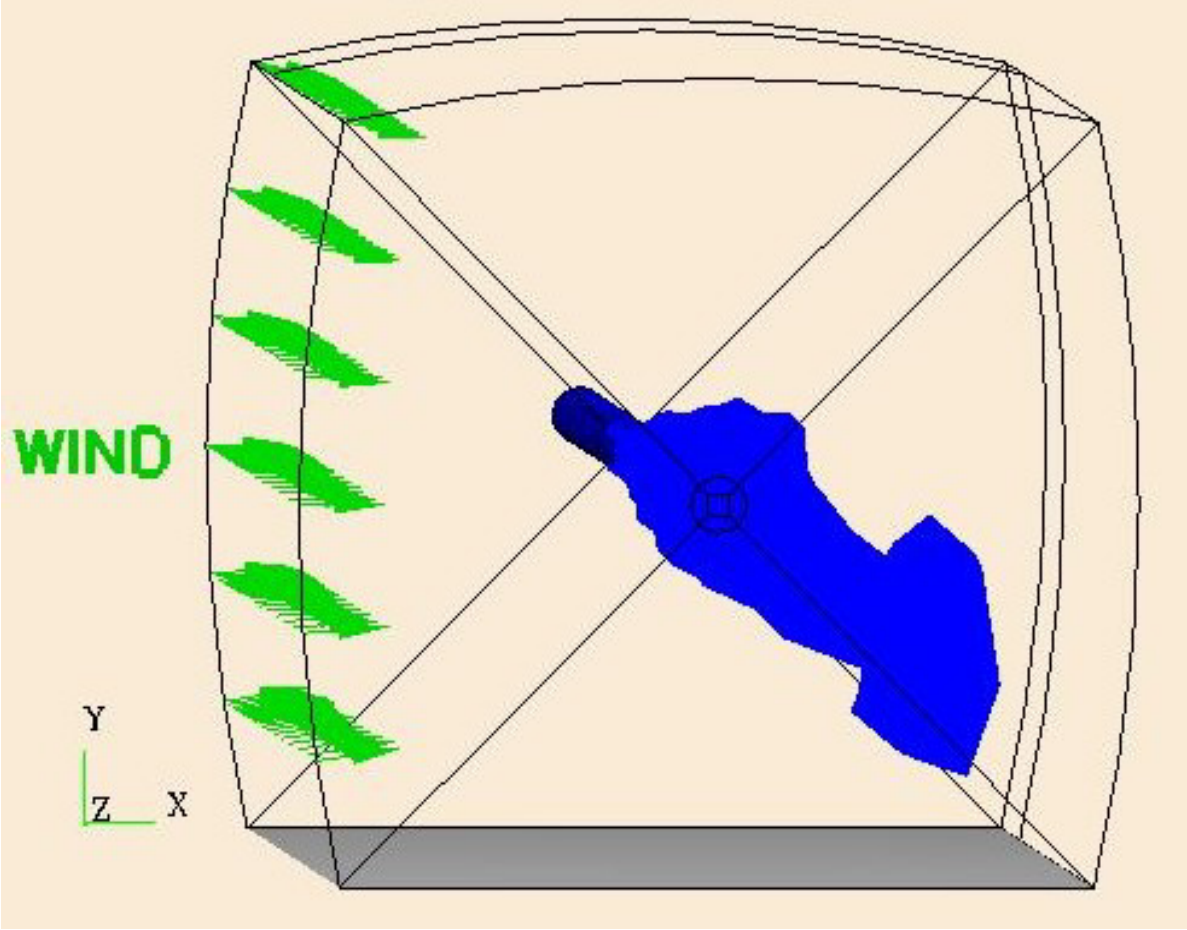


Figure 35. Angled View: Trial 4, 30 (kts).

APPENDIX F. TRIAL 5

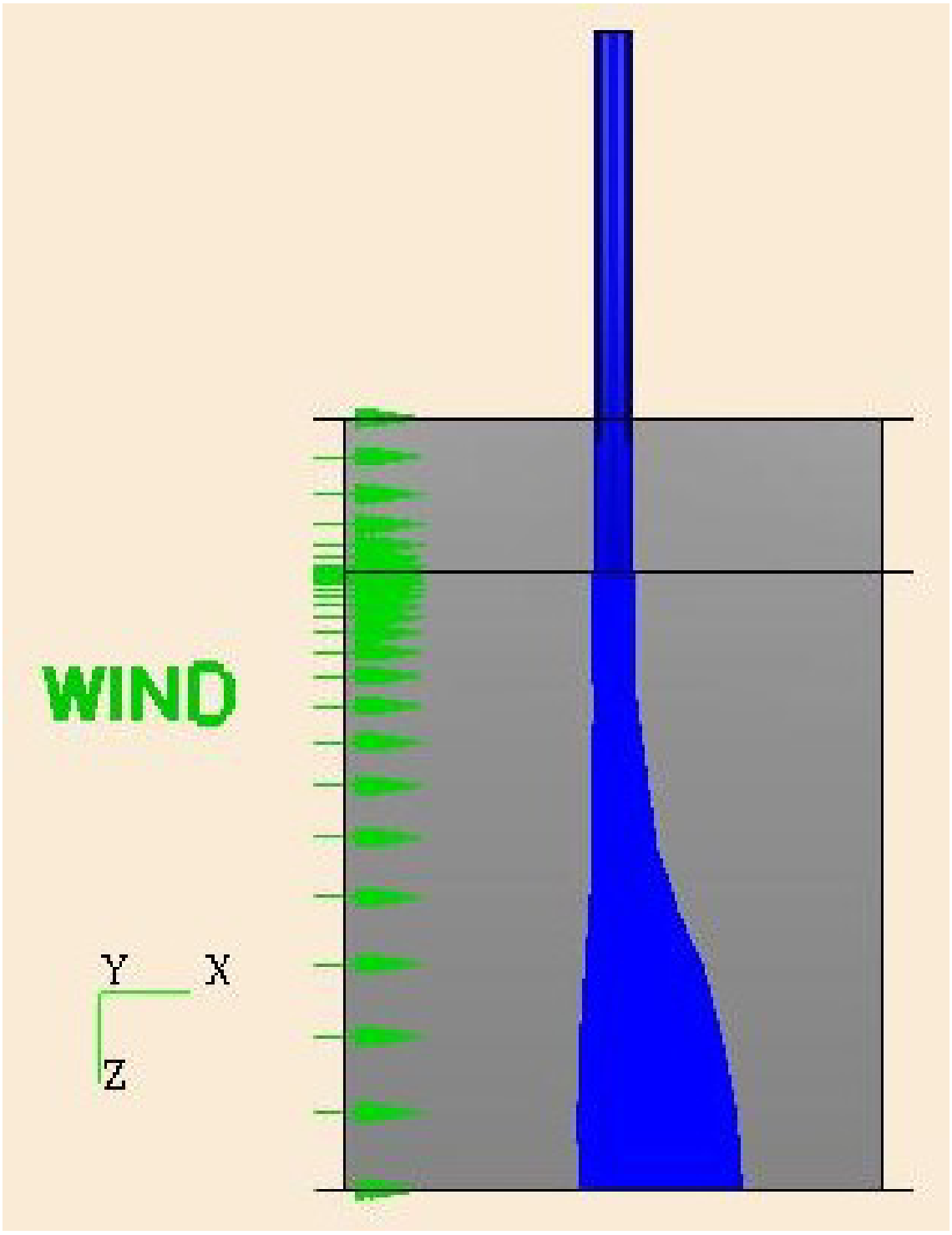


Figure 36. Plan View: Trial 5, 15 (kts).

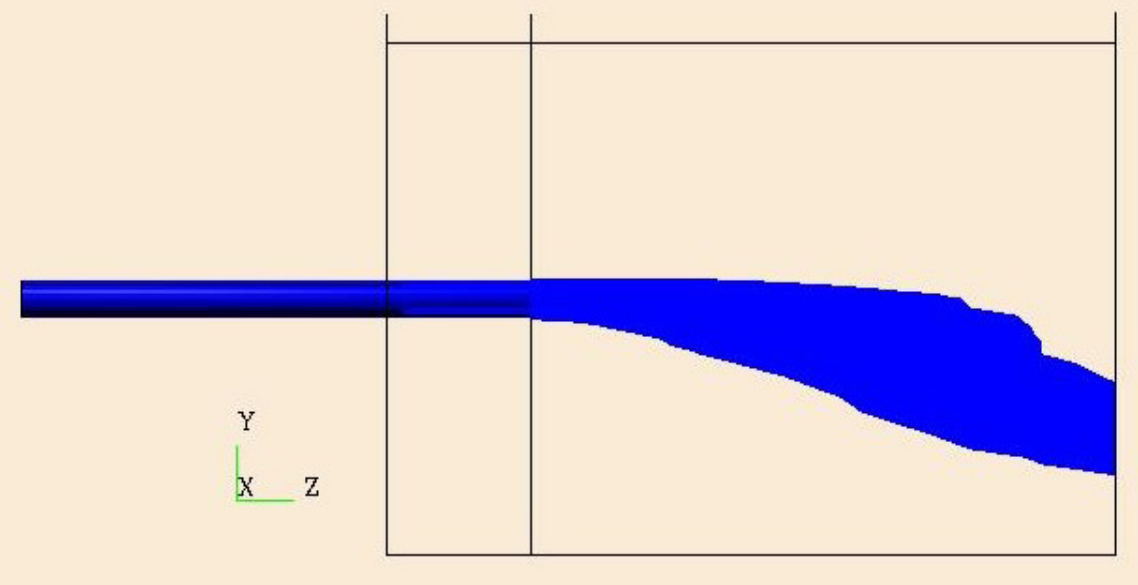
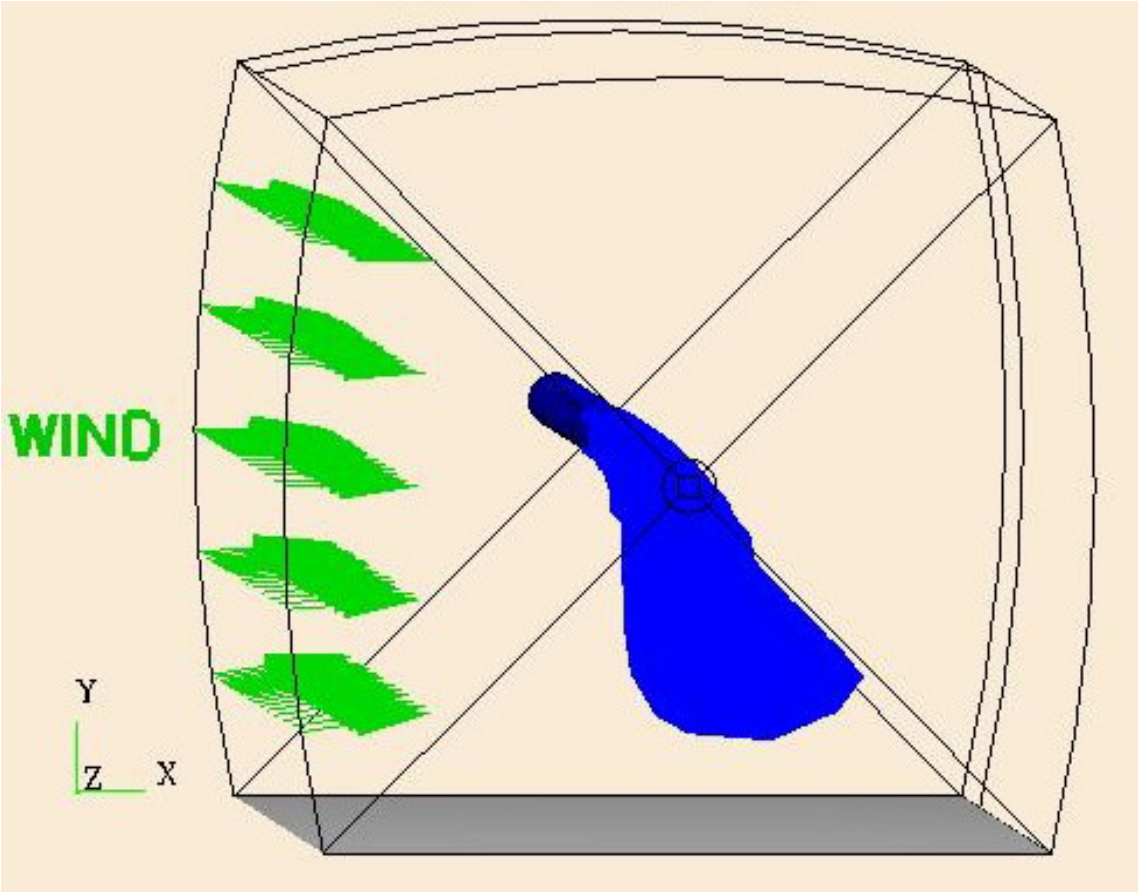


Figure 37. Side View: Trial 5, 15 (kts).



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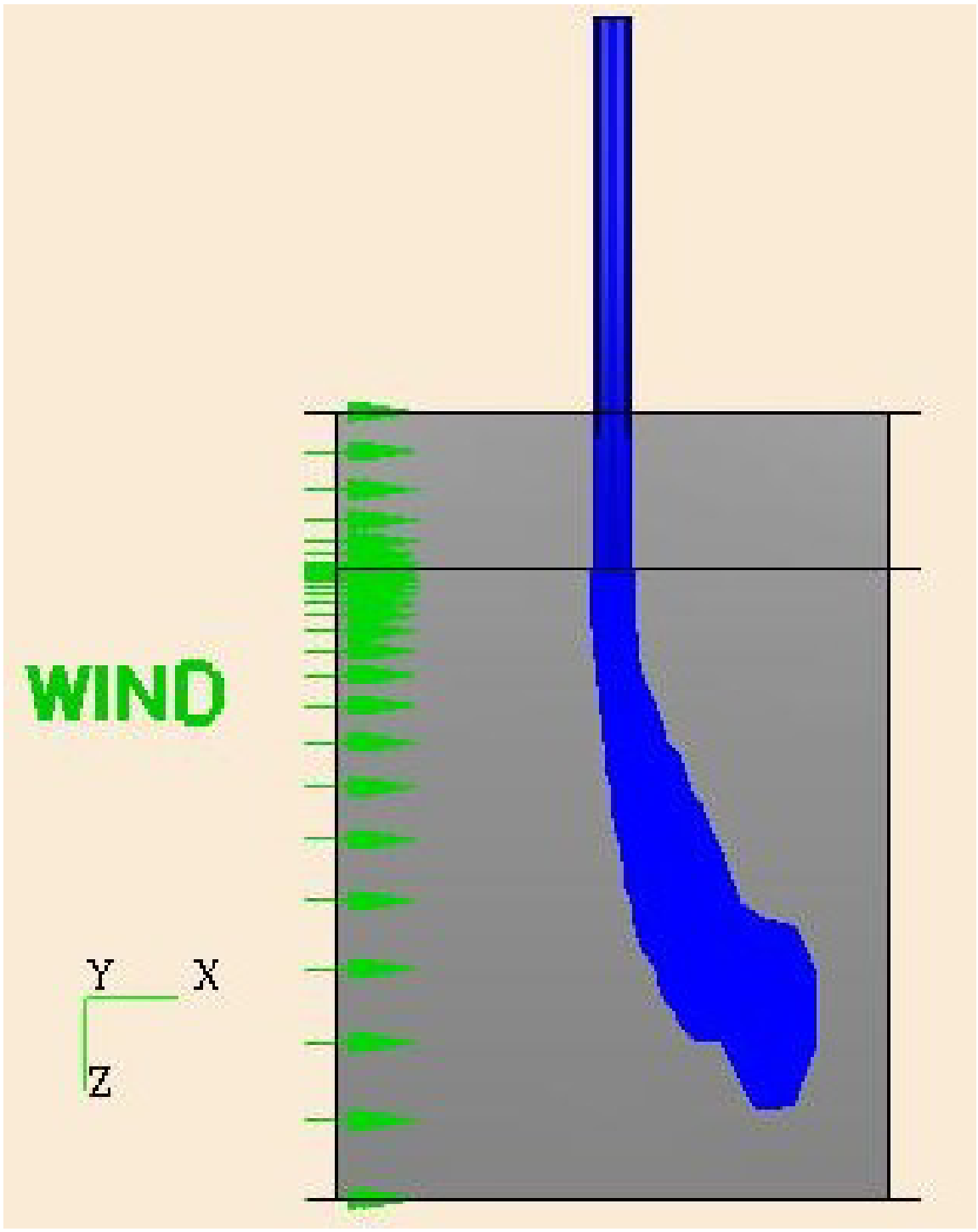

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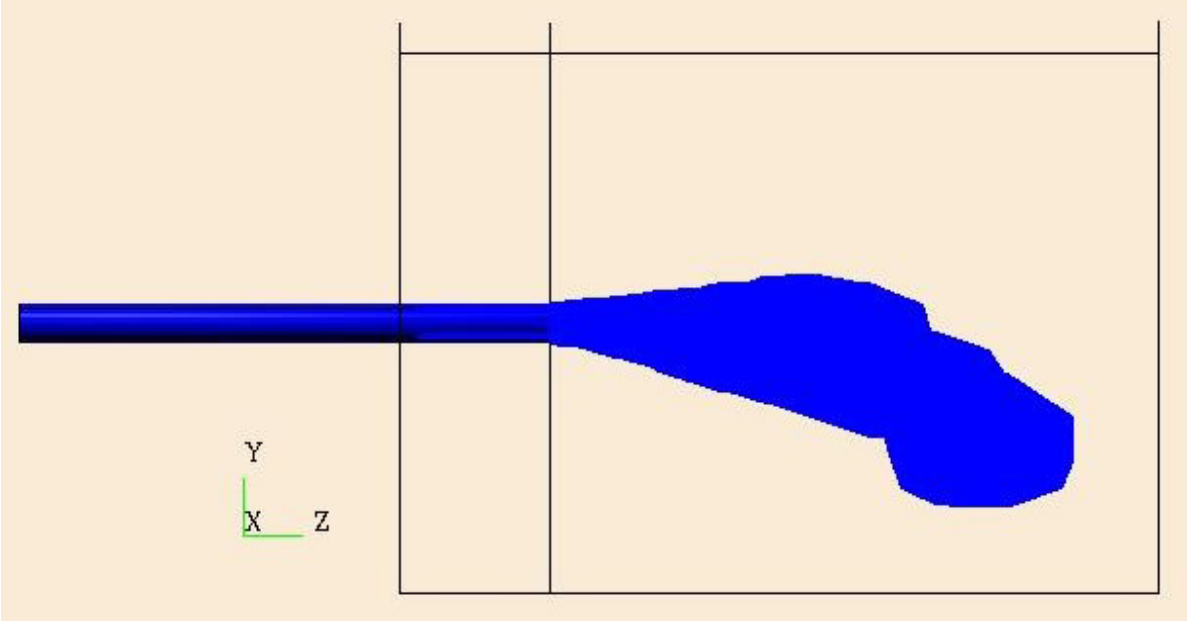

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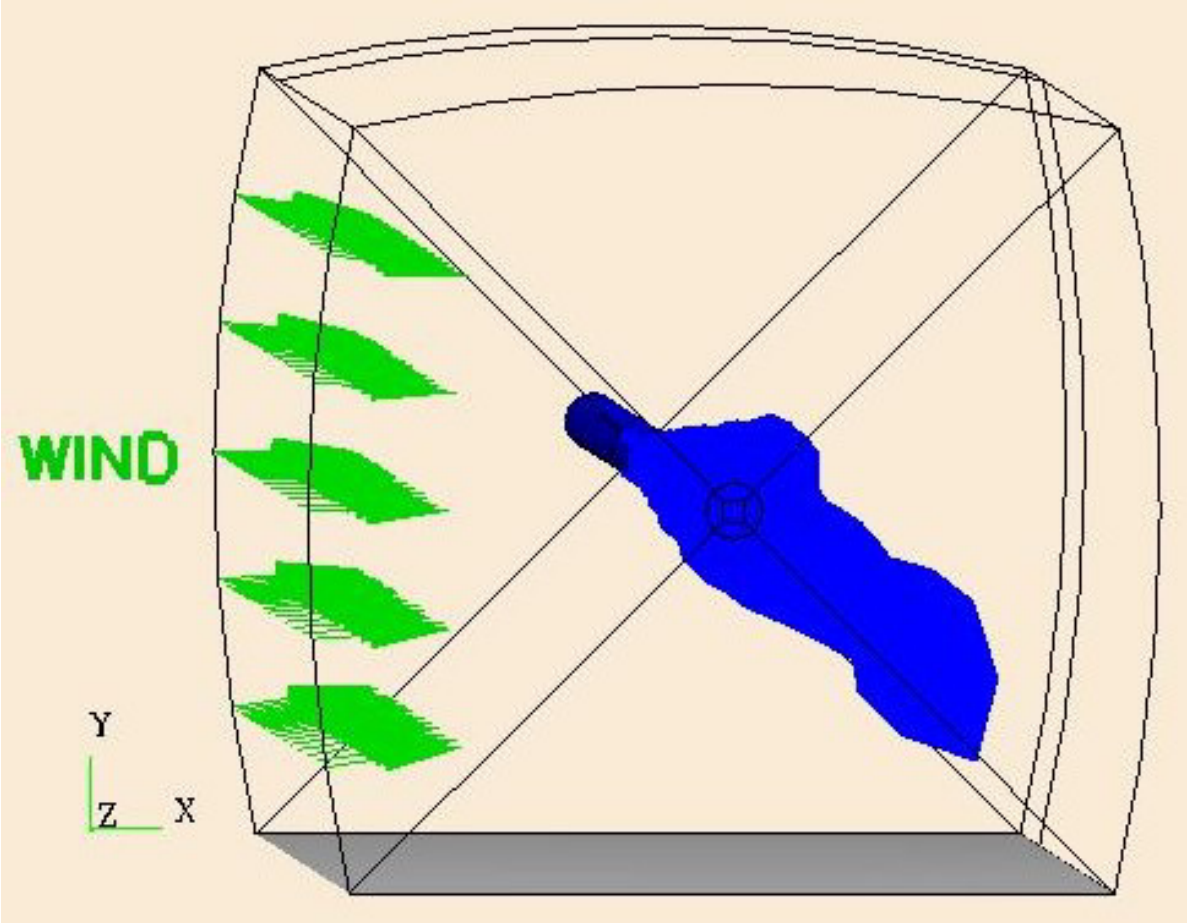


Figure 41. Angled View: Trial 5, 30 (kts).

APPENDIX G. TRIAL 6

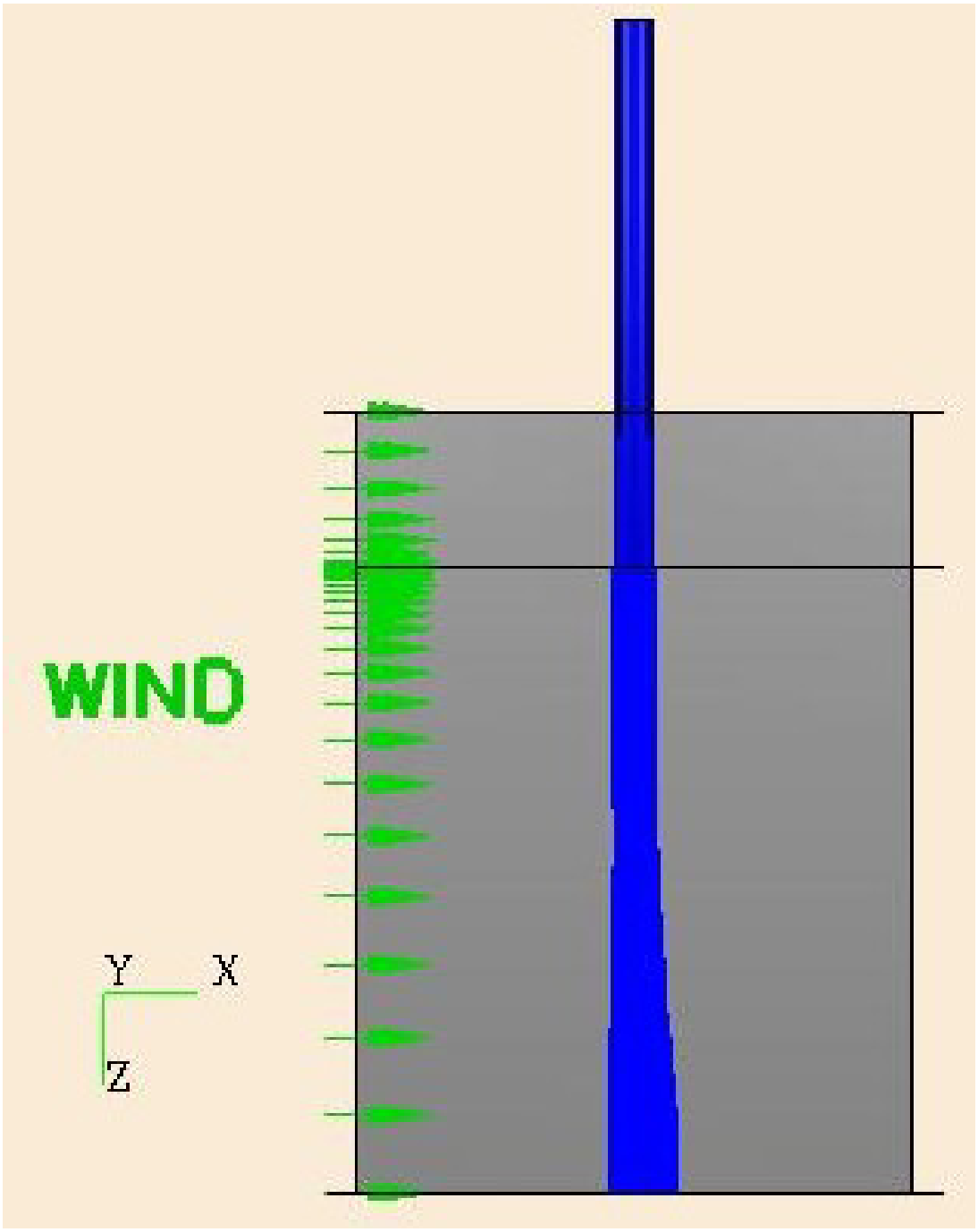


Figure 42. Plan View: Trial 6, 15 (kts).

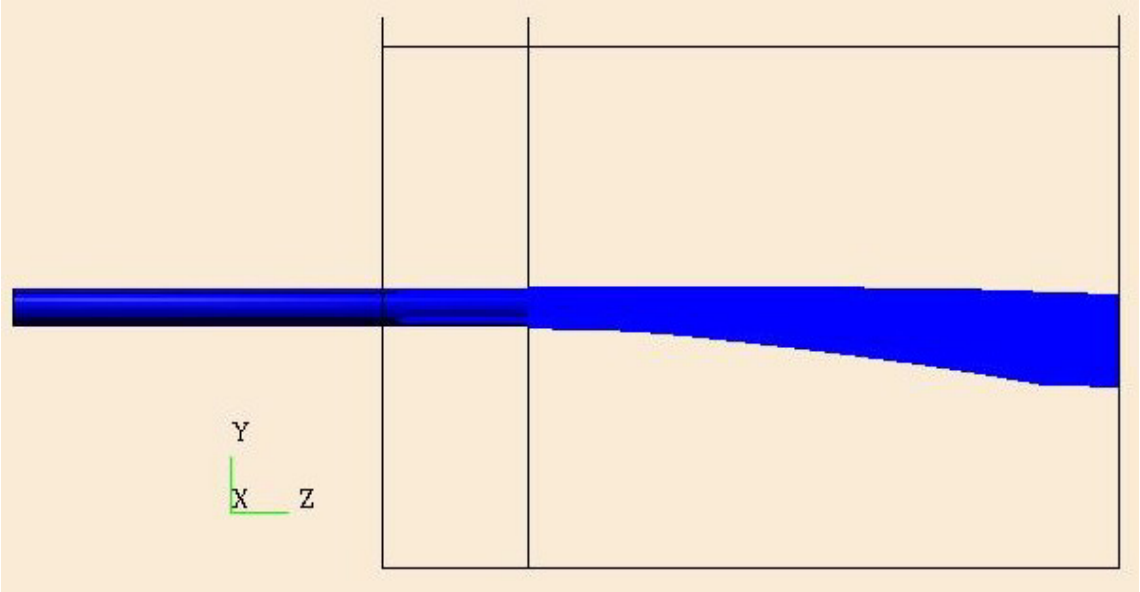


Figure 43. Side View: Trial 6, 15 (kts).

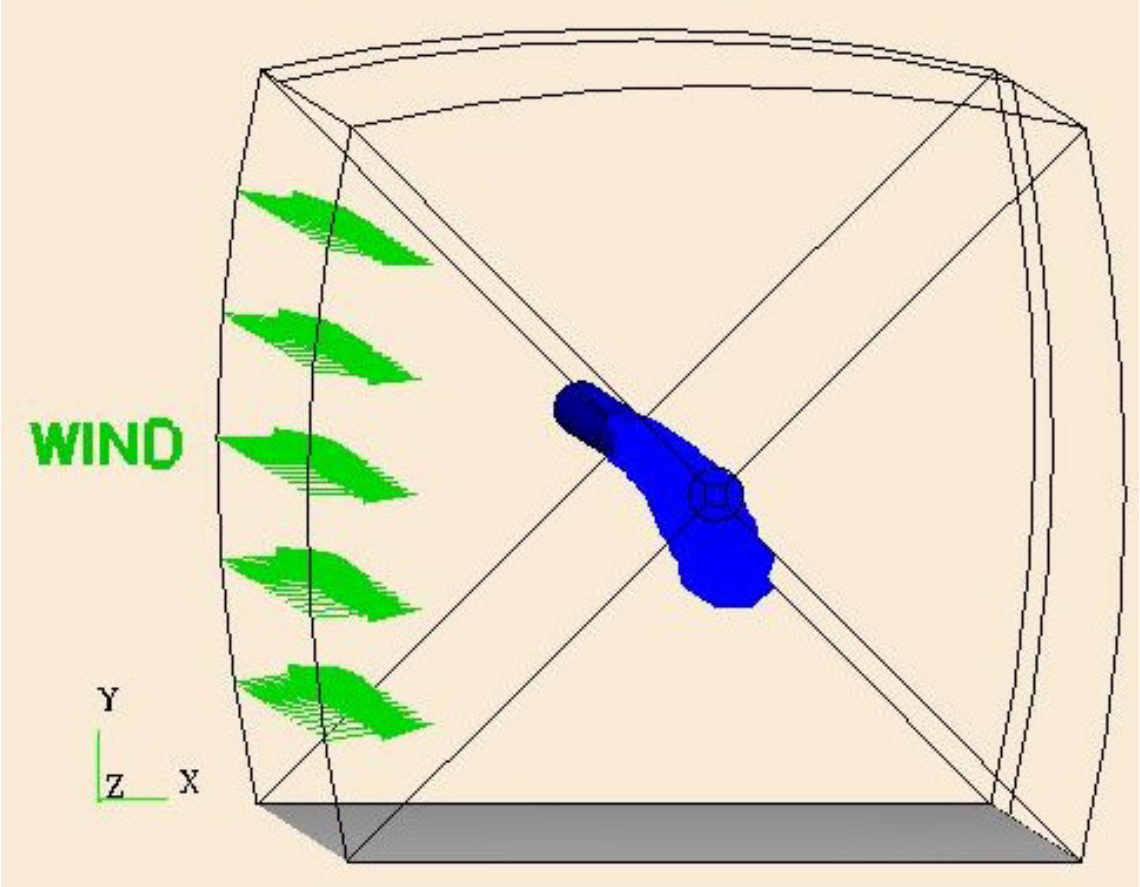
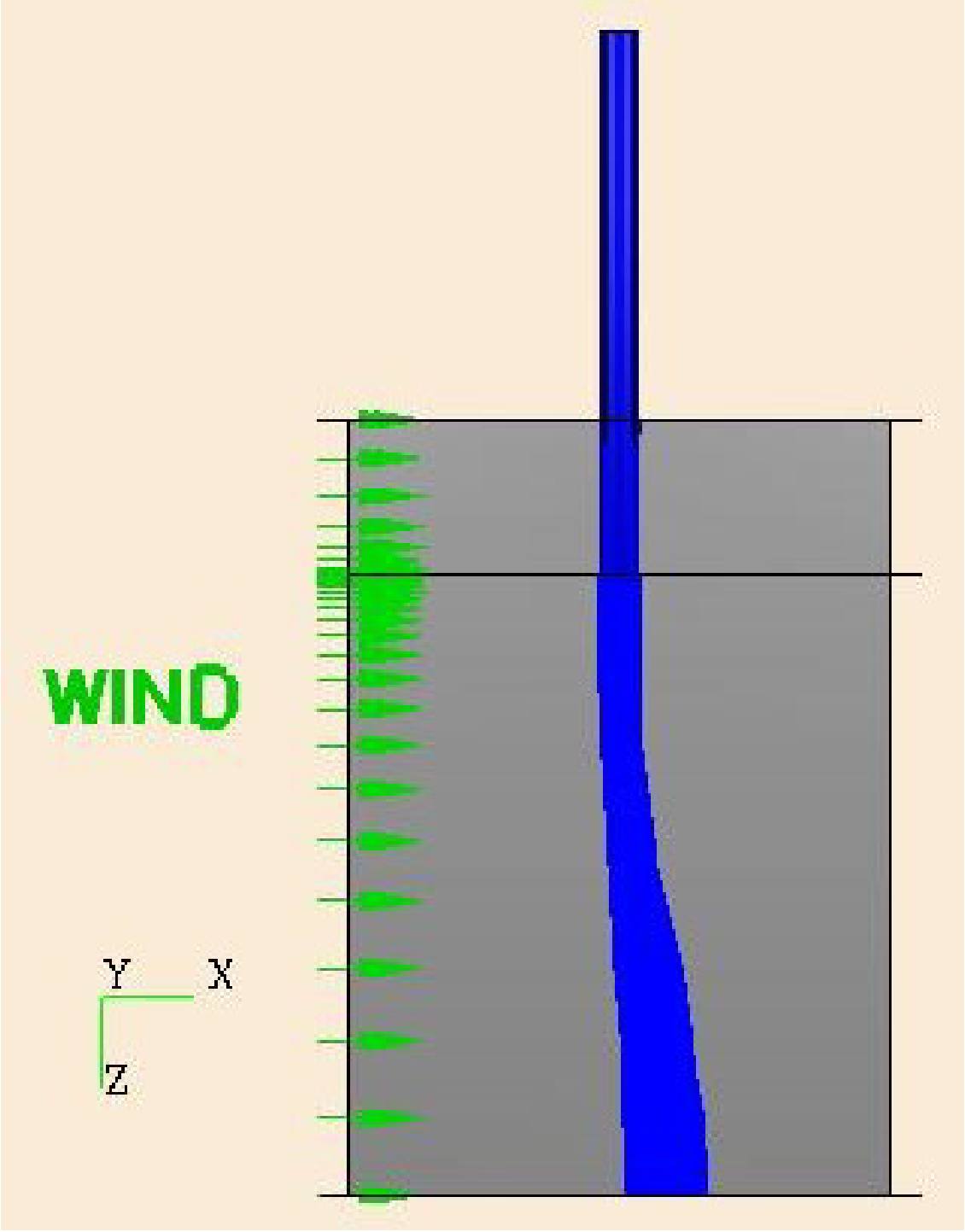


Figure 44. Angled View: Trial 6, 15 (kts).



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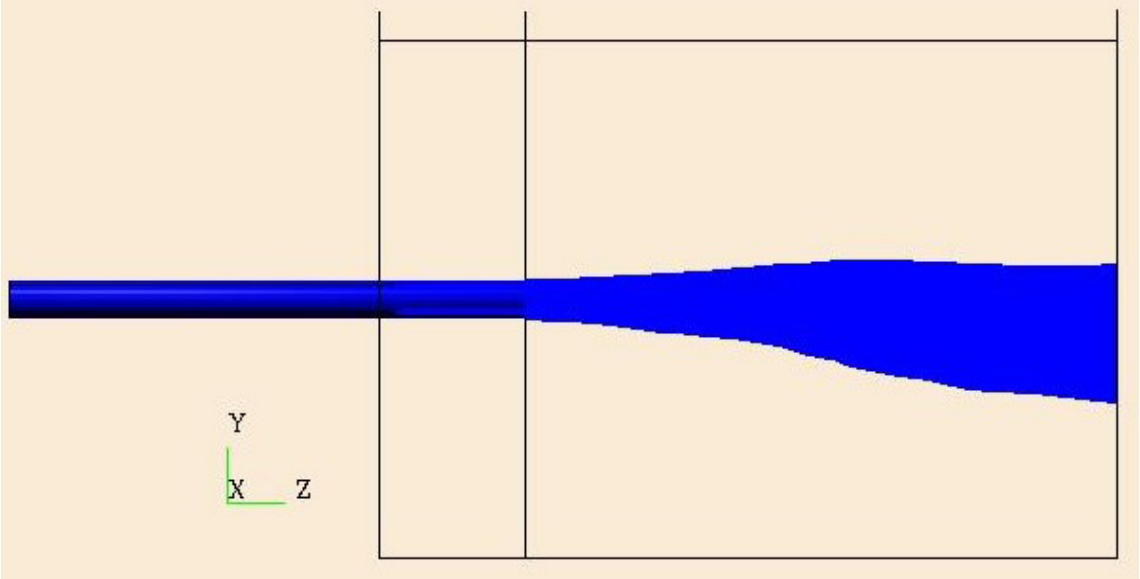

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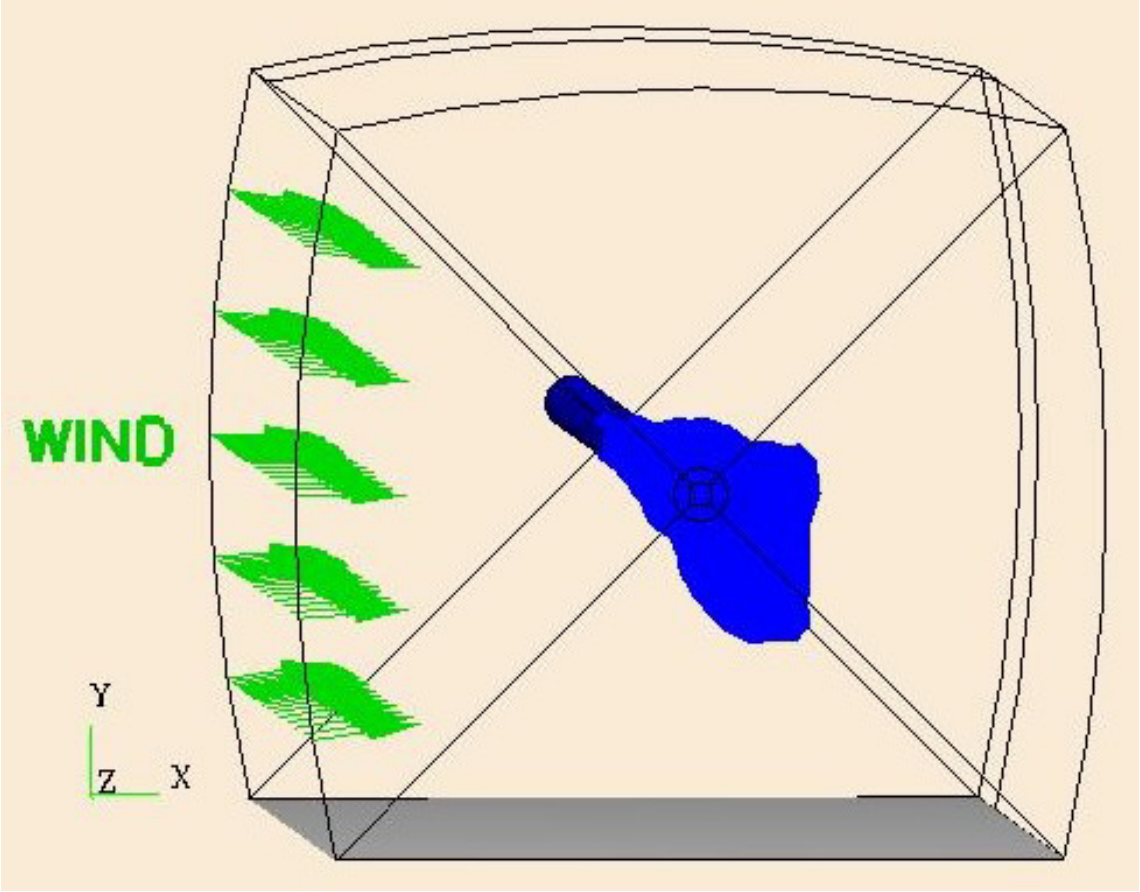

Figure 47. Angled View: Trial 6, 30 (kts).

APPENDIX H. TRIAL 7

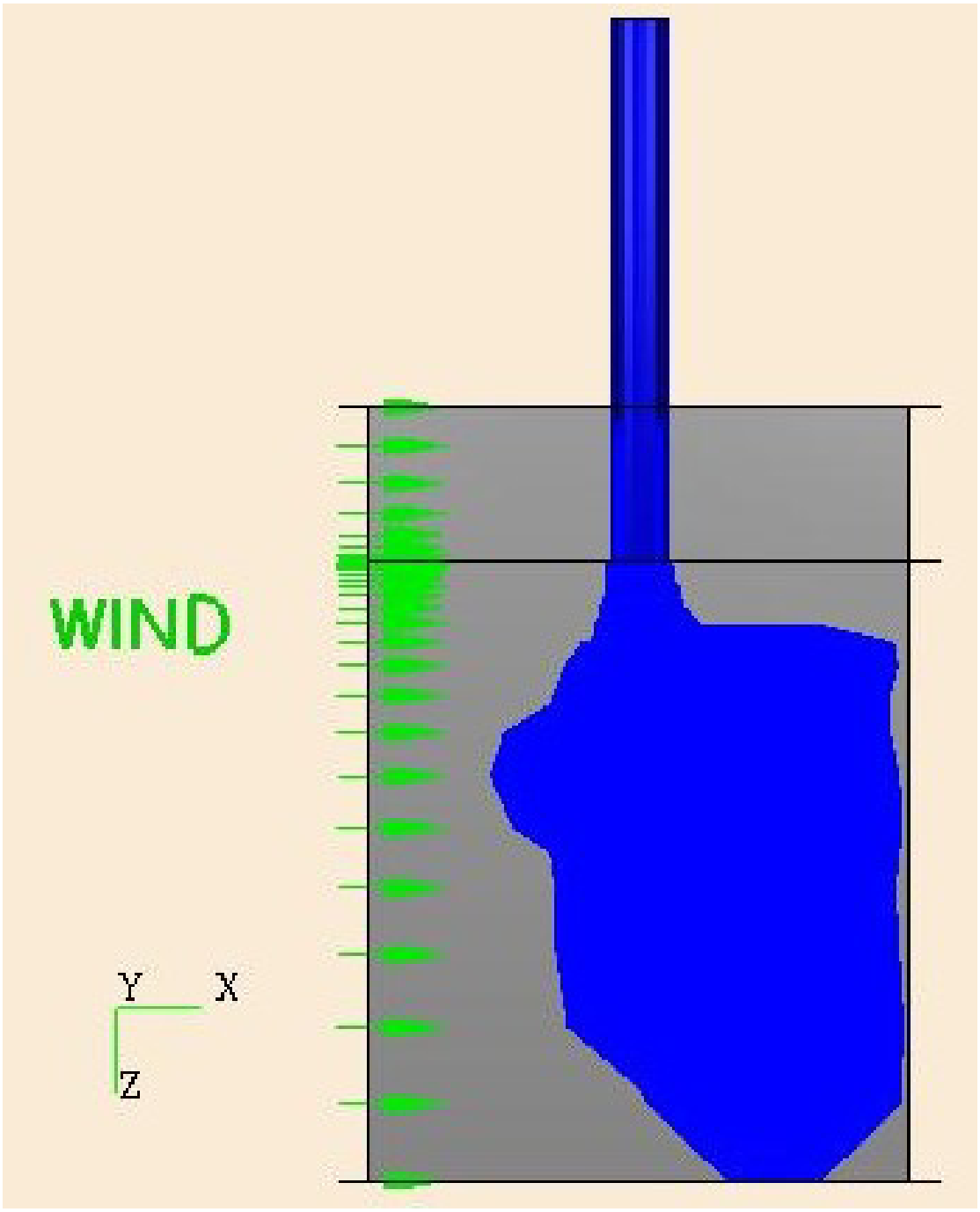

Figure 48. Plan View: Trial 7, 15 (kts).

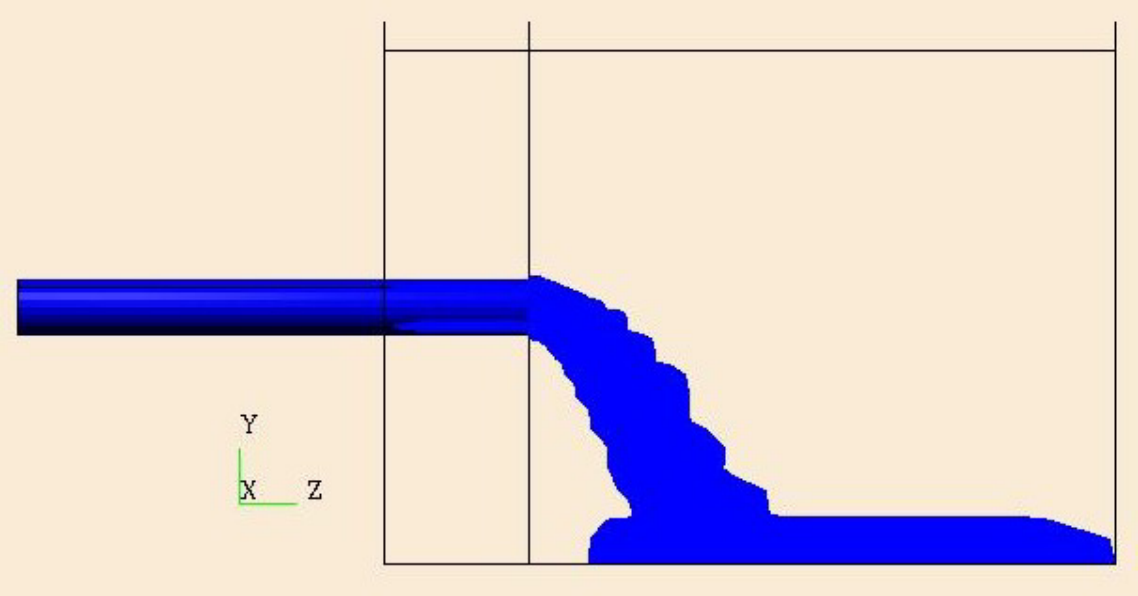

Figure 49. Side View: Trial 7, 15 (kts).

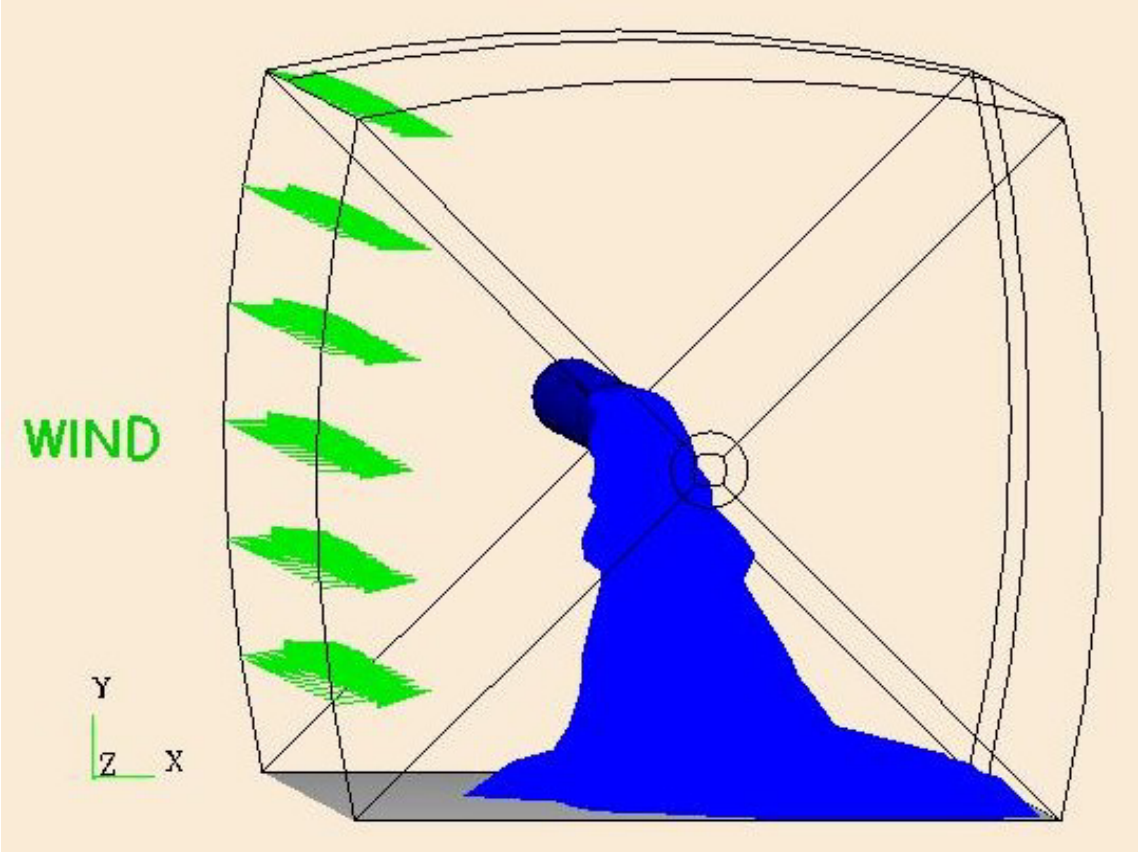

Figure 50. Angled View: Trial 7, 15 (kts).

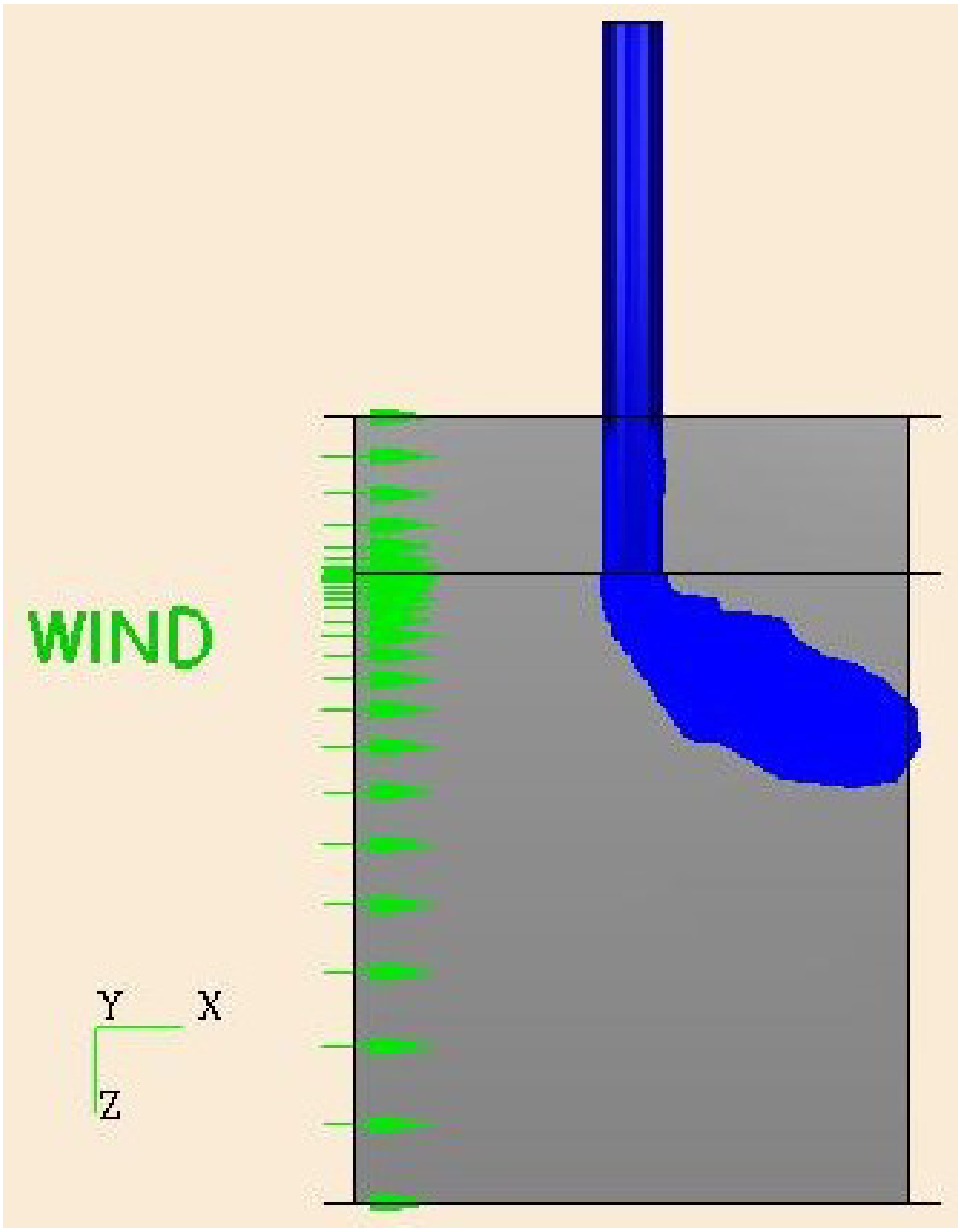

Figure 51. Plan View: Trial 7, 30 (kts).

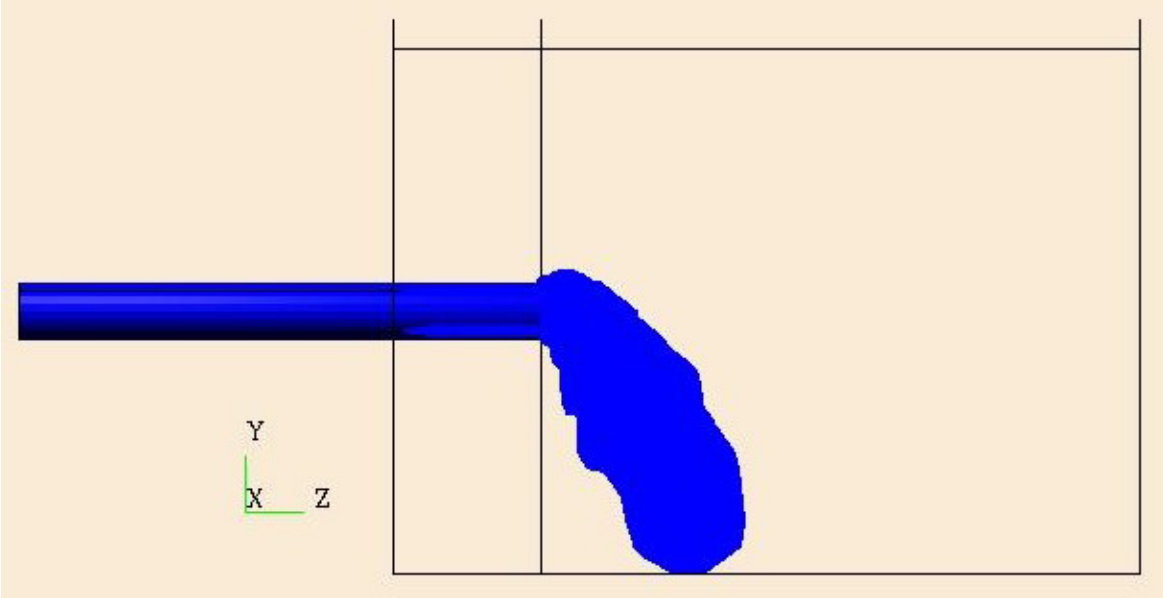

Figure 52. Side View: Trial 7, 30 (kts).

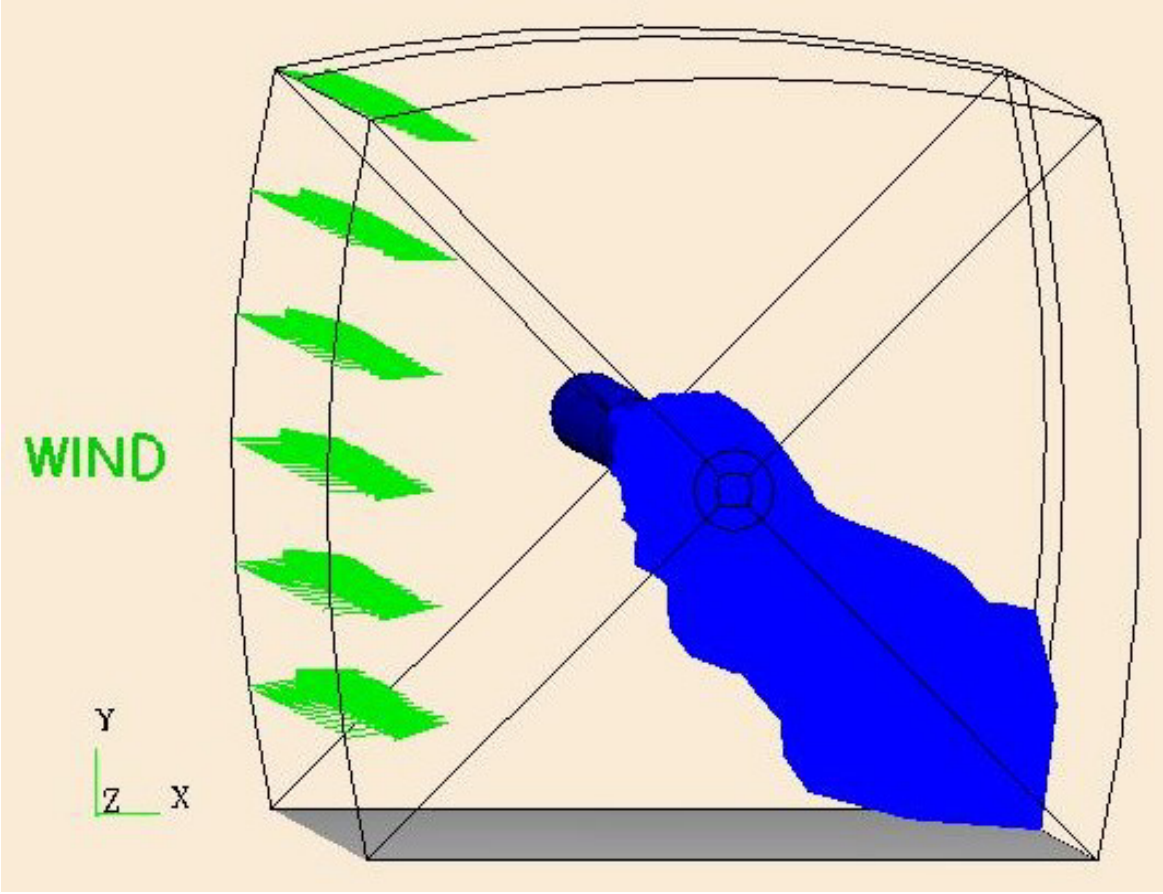

Figure 53. Angled View: Trial 7, 30 (kts).

APPENDIX I. TRIAL 8

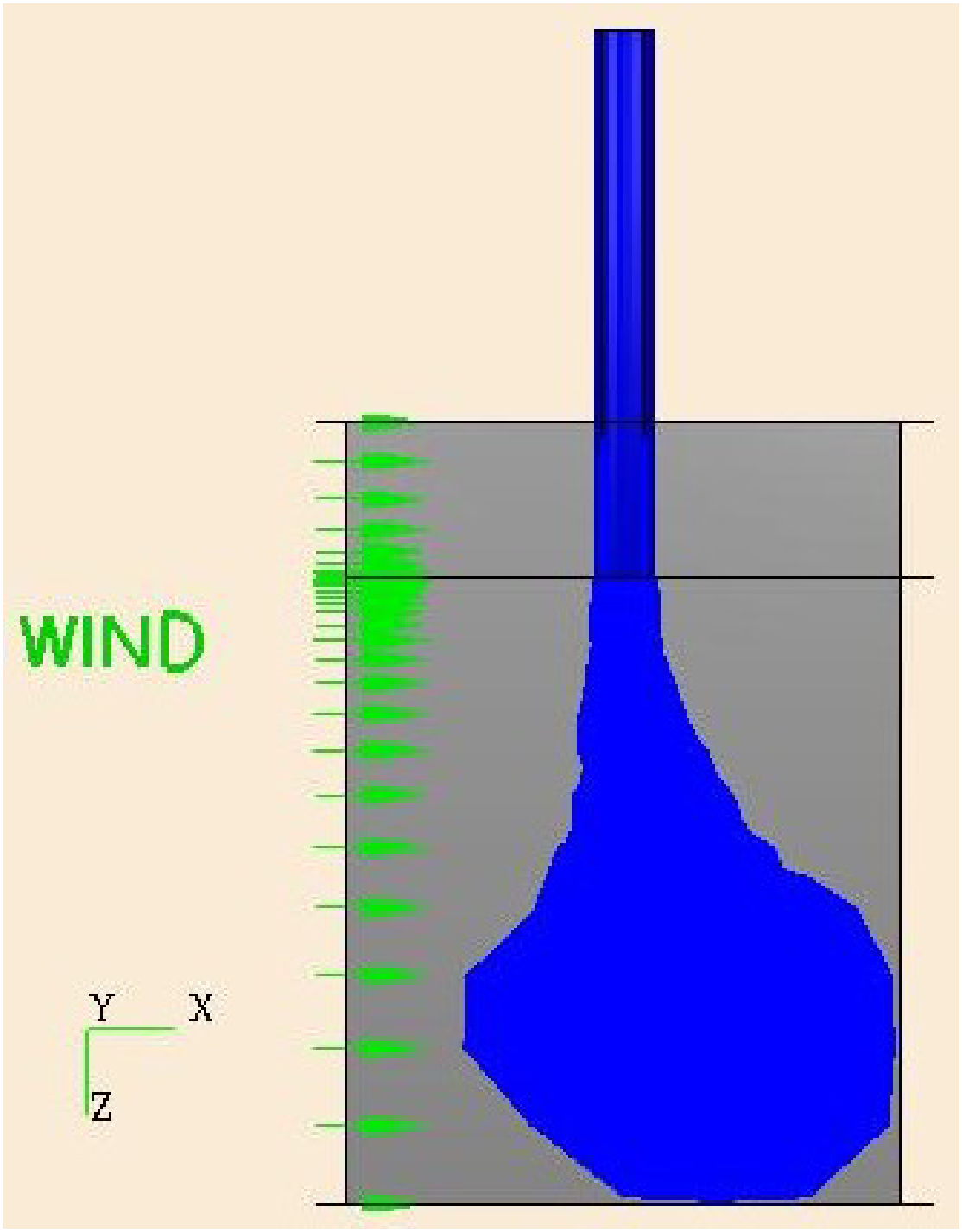

Figure 54. Plan View: Trial 8, 15 (kts).

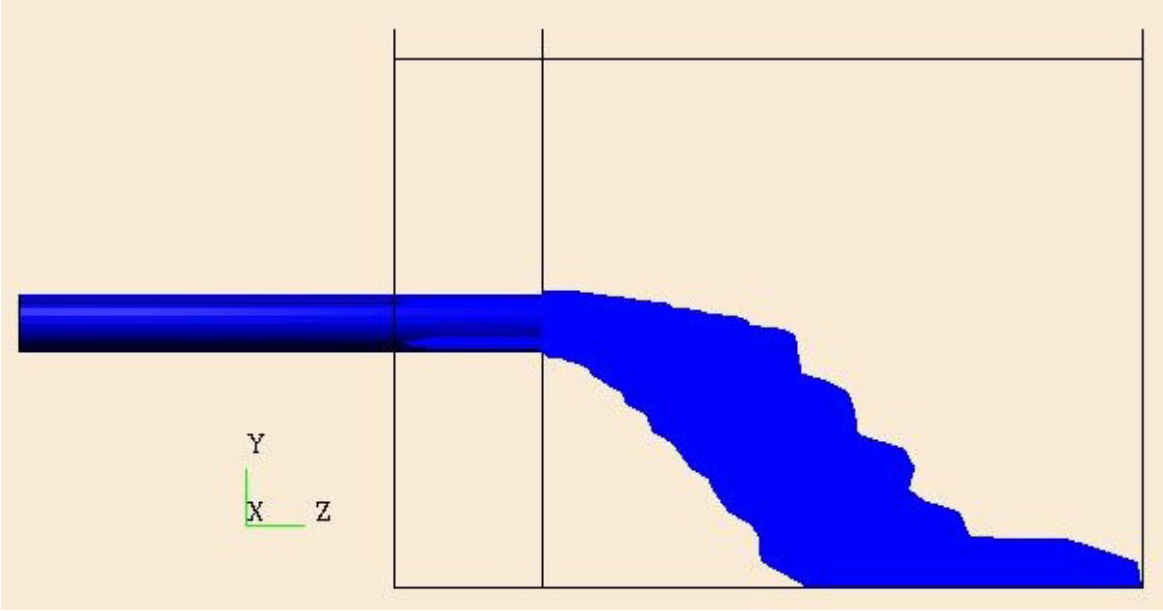

Figure 55. Side View: Trial 8, 15 (kts).

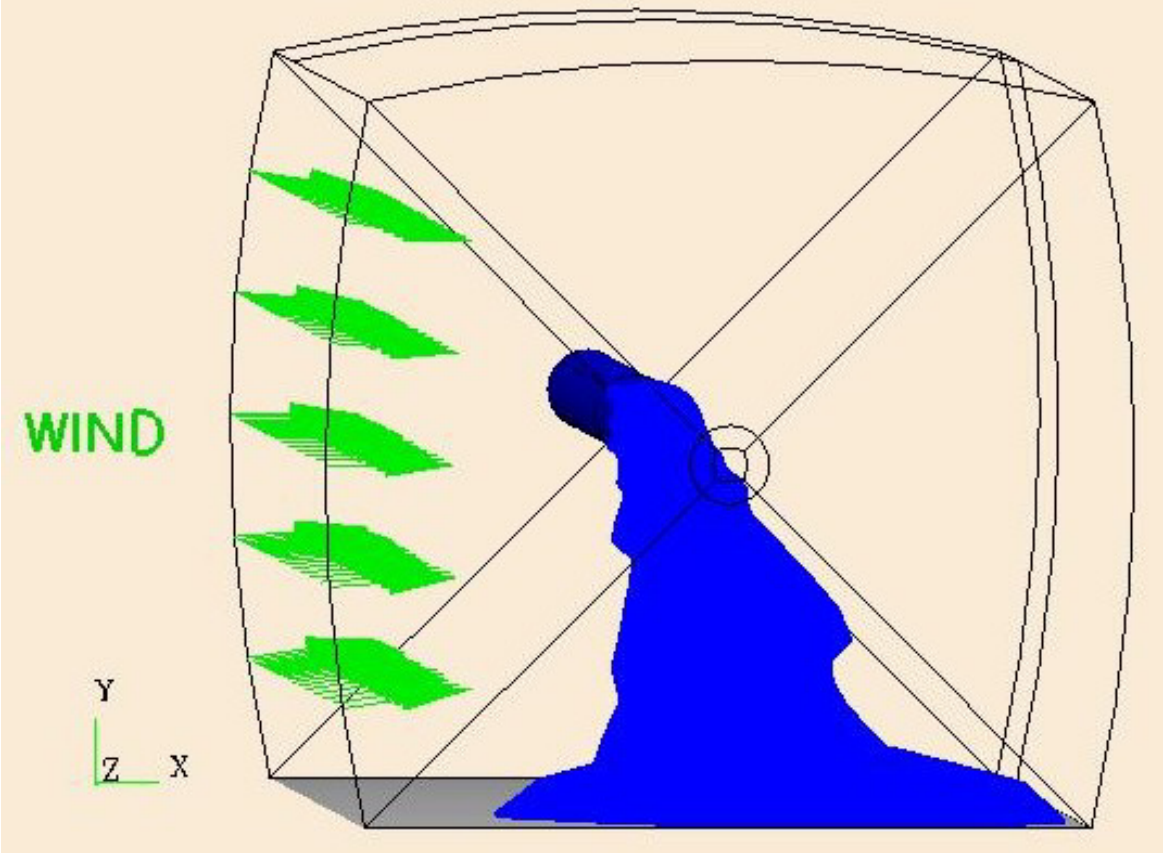

Figure 56. Angled View: Trial 8, 15 (kts).

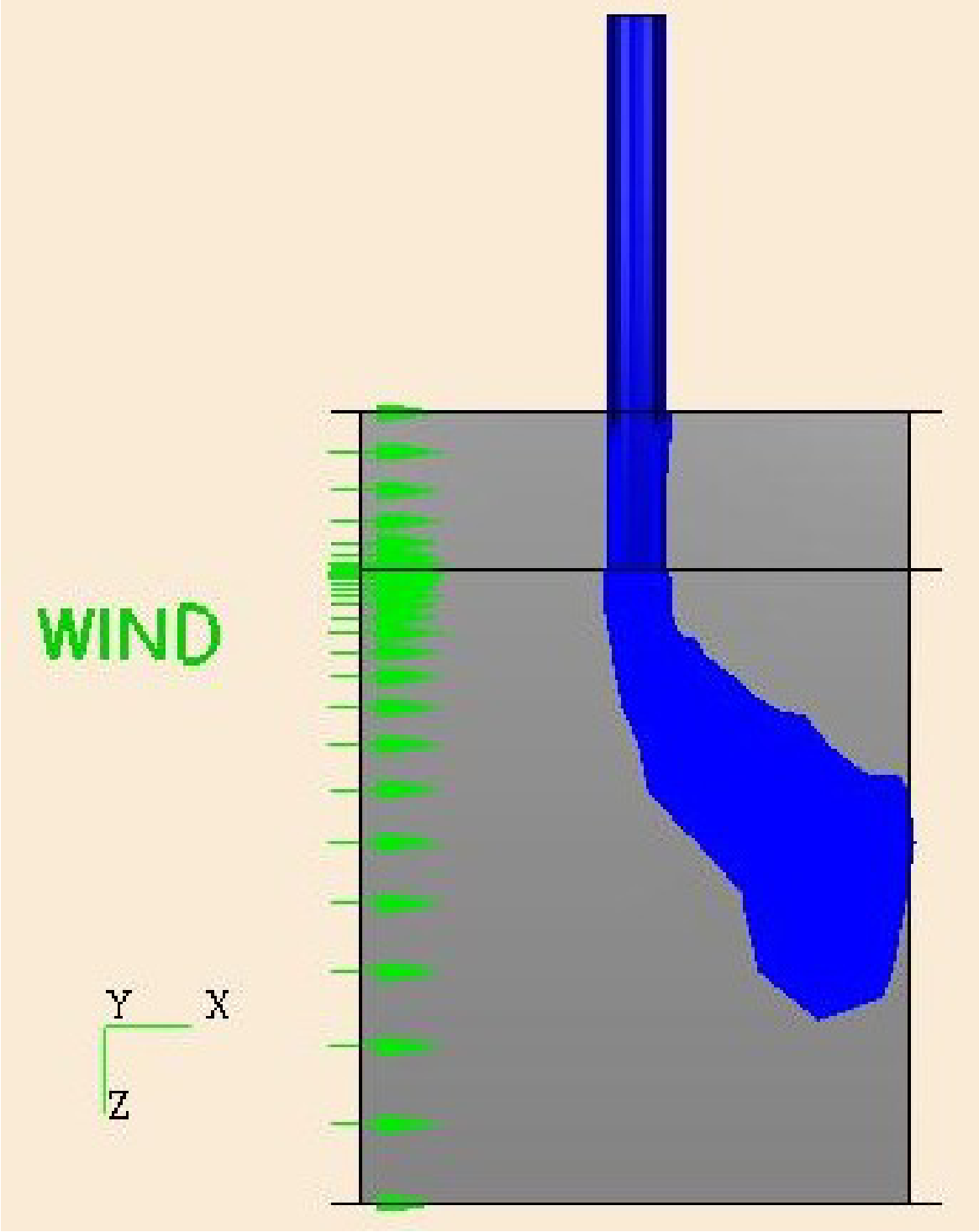

Figure 57. Plan View: Trial 8, 30 (kts).

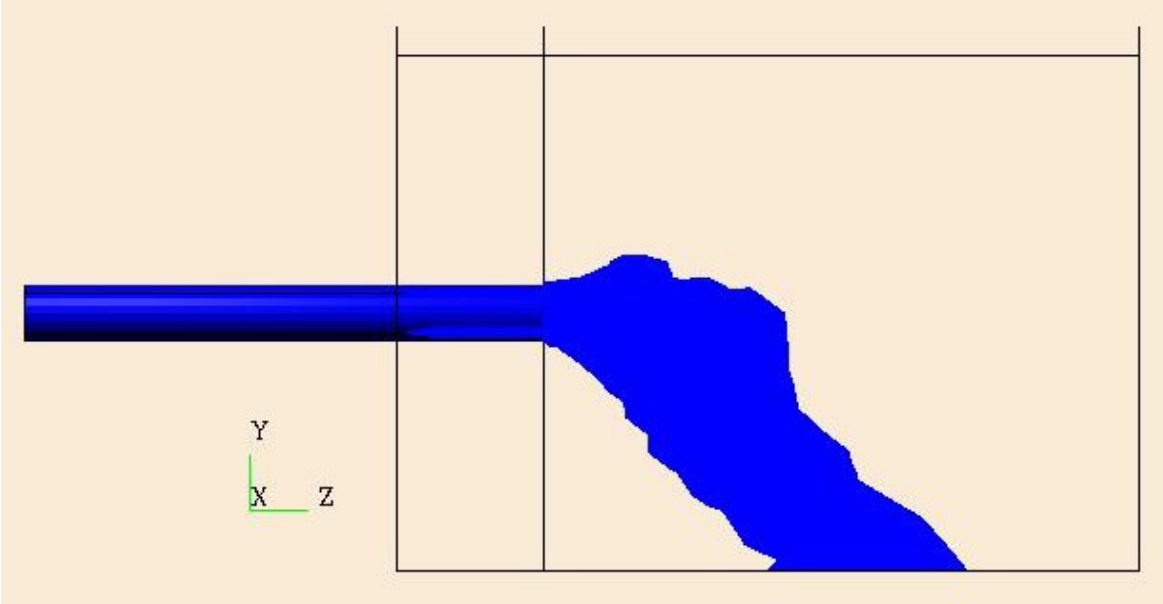

Figure 58. Side View: Trial 8, 30 (kts).

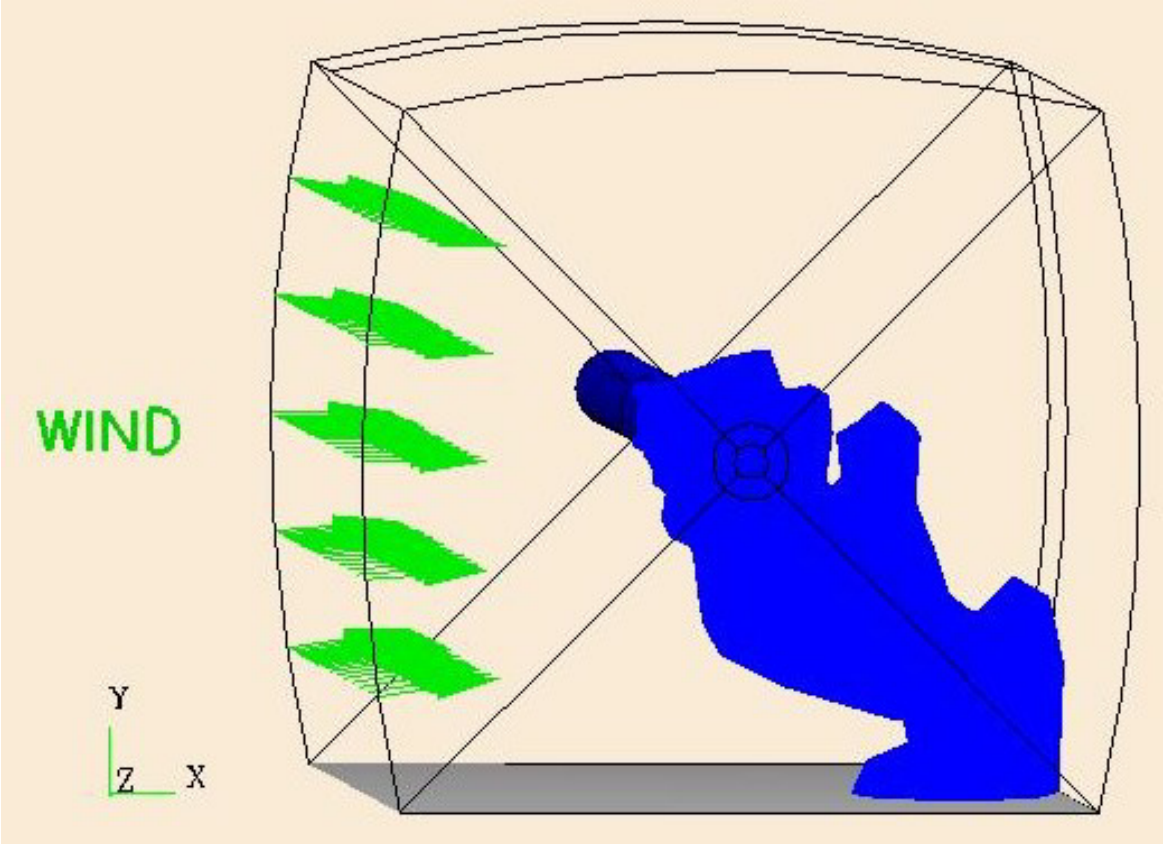

Figure 59. Angled View: Trial 8, 30 (kts).

APPENDIX J. TRIAL 9

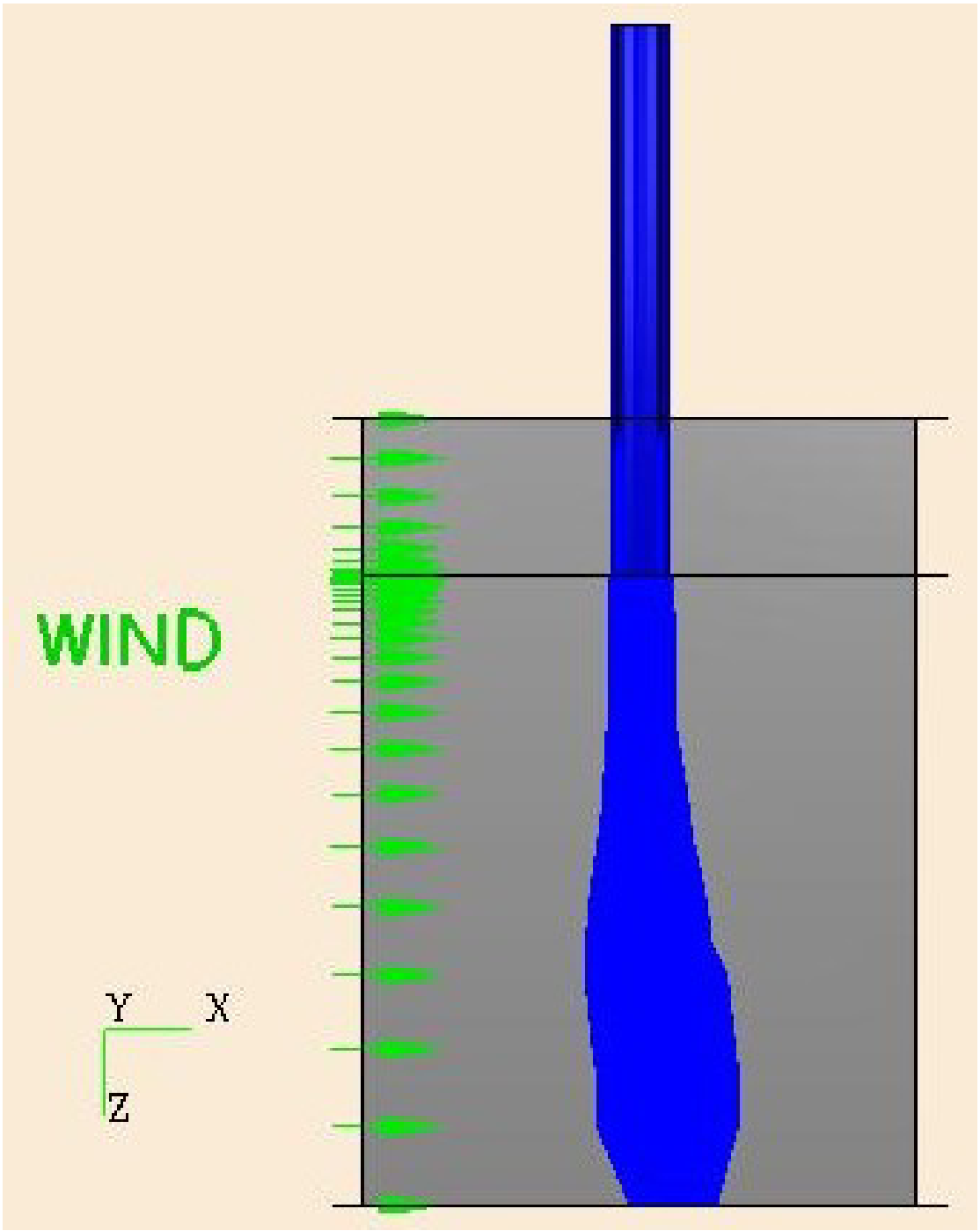

Figure 60. Plan View: Trial 9, 15 (kts).

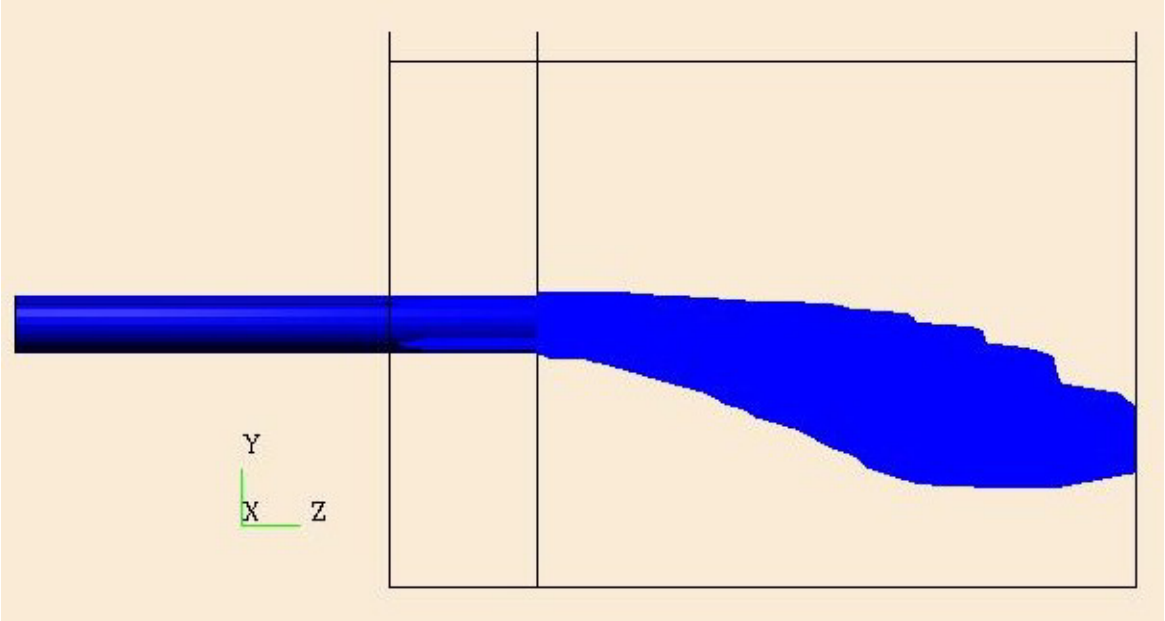

Figure 61. Side View: Trial 9, 15 (kts).

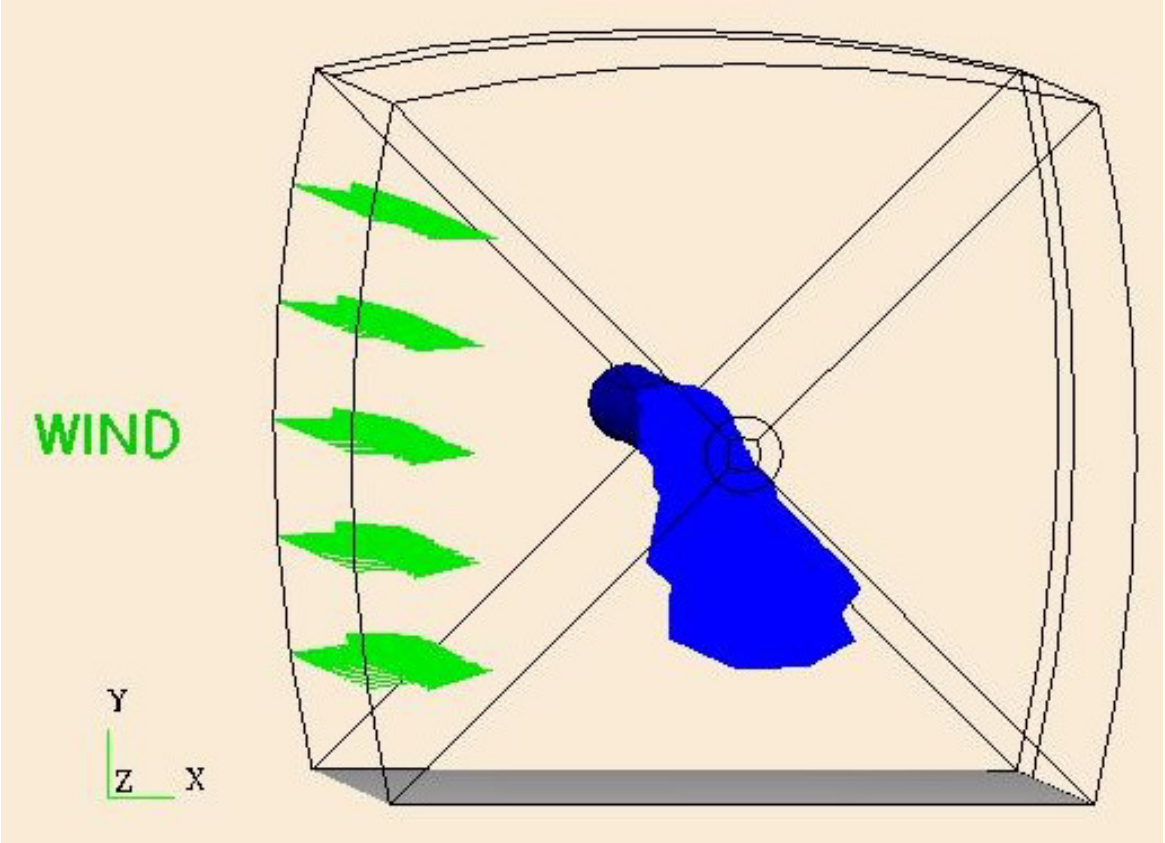

Figure 62. Angled View: Trial 9, 15 (kts).

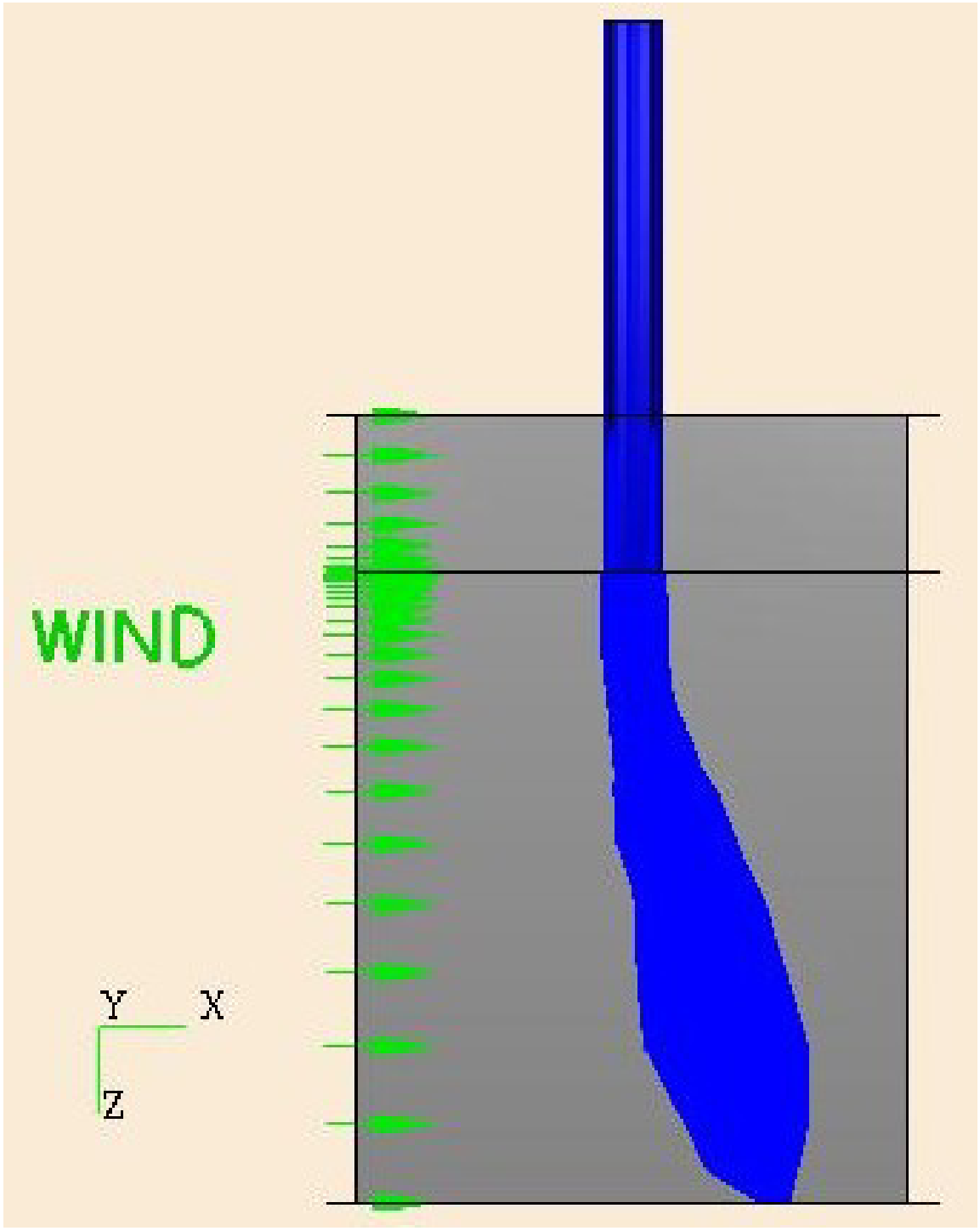

Figure 63. Plan View: Trial 9, 30 (kts).

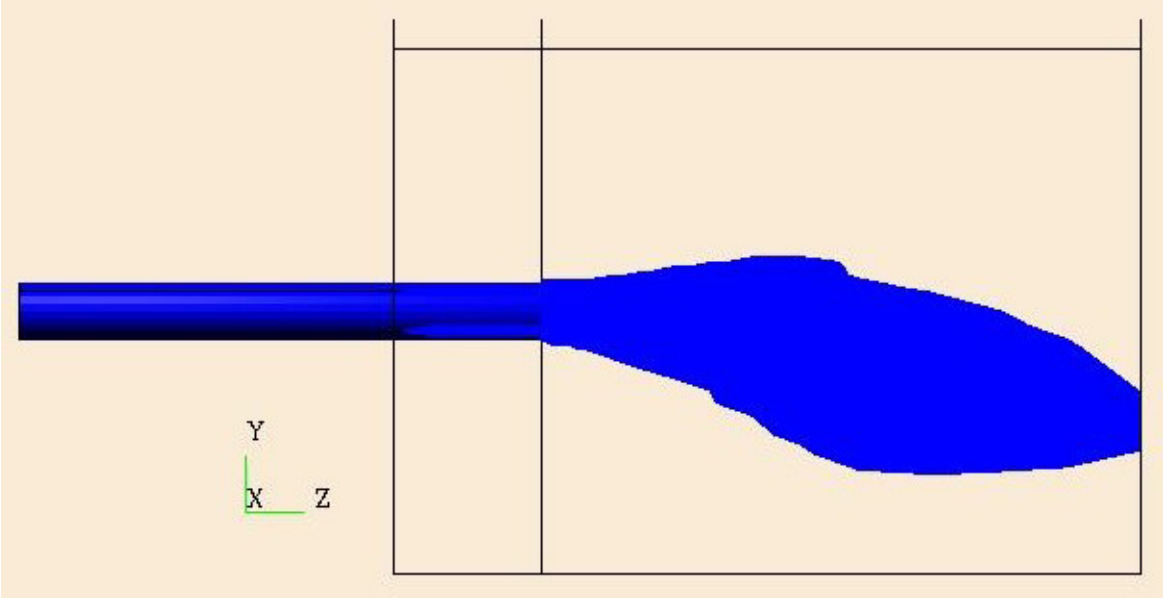

Figure 64. Side View: Trial 9, 30 (kts).

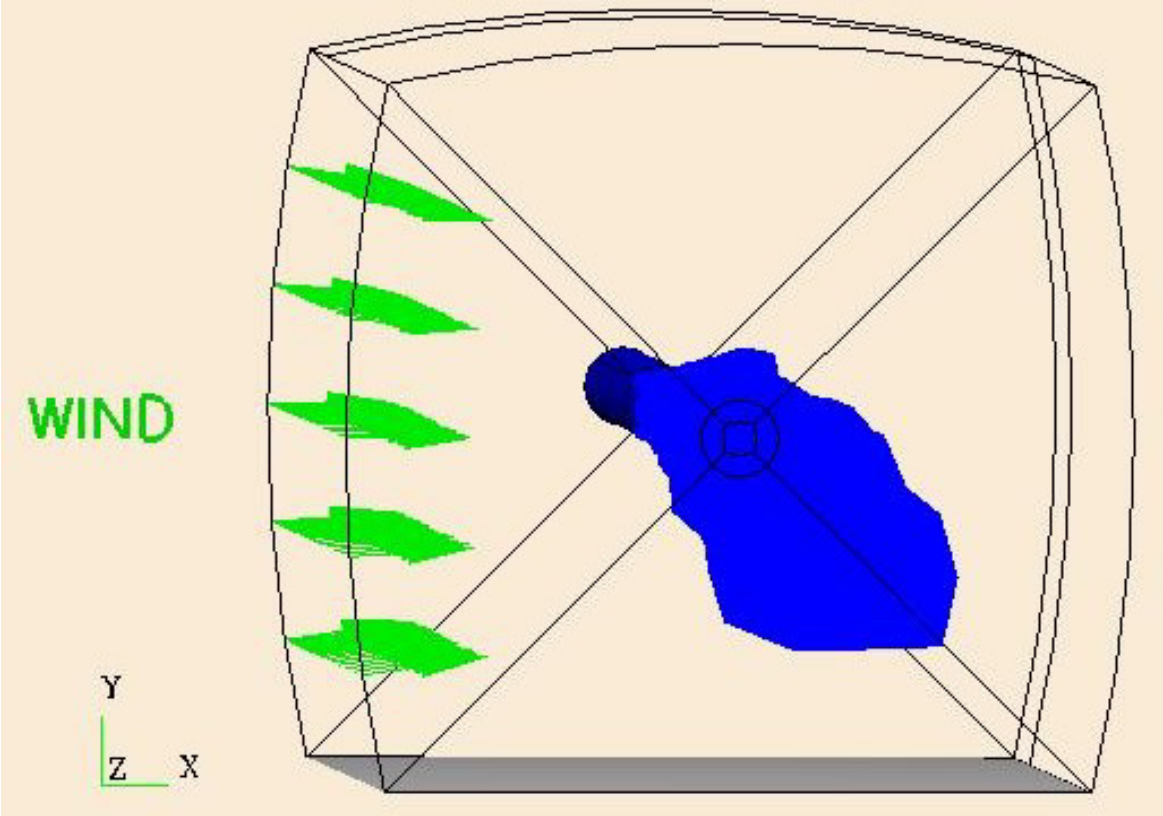

Figure 65. Angled View: Trial 9, 30 (kts).

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