

ANALYSIS OF A METHOD FOR OPTIMUM
DESIGN OF WATERJET PROPULSION
SYSTEMS.

Dennis Keith Kruse

LIBRARY
NAVAL POSTGRADUATE SCHOOL
MONTEREY, CALIF. 93940

ANALYSIS OF A METHOD FOR OPTIMUM DESIGN
OF WATERJET PROPULSION SYSTEMS

by

Dennis Keith Kruse

B.S., U. S. Naval Academy

(1965)

SUBMITTED IN
PARTIAL FULFILLMENT
OF THE REQUIREMENTS FOR THE
DEGREE OF OCEAN ENGINEER
AND MASTER OF SCIENCE

at the

MASSACHUSETTS INSTITUTE OF
TECHNOLOGY

June, 1973

ANALYSIS OF A METHOD FOR OPTIMUM DESIGN
OF WATERJET PROPULSION SYSTEMS

by

Dennis Keith Kruse

Submitted to the Department of Ocean Engineering on May 11, 1973, in partial fulfillment of the requirements for degrees of Ocean Engineer and Master of Science in Mechanical Engineering.

ABSTRACT

An existing computerized method for optimum design of waterjet propulsion systems for sub-cavitating hydrofoil craft is analyzed for sensitivity to variations in normally fixed parameters and for sensitivity to variations in the starting points for the search used in the optimization procedure. A compatible method for off design evaluation of waterjet propulsion systems is developed and incorporated into the optimization program in a manner which permits off design evaluation to be performed separately or in conjunction with design. The evaluation routine requires that system geometry, craft characteristics and pump characteristics be specified. System drag and losses are calculated to determine required flow rate and pump head and the corresponding pump speed, efficiency and required power are determined. Results of design optimization for a series of similar craft are presented and show a strong sensitivity to the input estimate of the take-off drag. Sensitivity to starting values of the independent variables was noted in some cases and appears to be due to the fact that jet velocity ratio dominates the other independent variables as an influence on total system weight. A FORTRAN computer listing and sample inputs and outputs for both design and evaluation routines are included.

Thesis Supervisor: A. Douglas Carmichael

Title: Professor of Power Engineering

TABLE OF CONTENTS

Title Page	1
Abstract	2
Table of Contents	3
List of Figures	4
List of Symbols Used in Text	5
Chapter 1. Introduction	8
Chapter 2. Methodology Used for Evaluation	11
2.1 Assumed General System Configuration	11
2.2 General Procedure for Evaluation	12
2.3 Specific Evaluation Methodology	13
2.4 Cavitation Considerations	24
Chapter 3. Design Methodology	27
Chapter 4. Discussion of Results	32
Chapter 5. Conclusions and Recommendations	36
Bibliography	38
Figures	39
Appendix A. List of Symbols Used in Program	42
Appendix B. Program Listing	81
Appendix C. Users' Manual	162

LIST OF FIGURES

<u>Figure No.</u>	<u>Title</u>	<u>Page</u>
1	General Ducting Configuration	39
2	Propulsion Weight Fraction vs. Installed Power Propulsive Coefficient at Take-off	40
3	Cruise Speed Propulsive Coefficient vs. Installed Power Propulsive Coefficient at Take-off	41

SYMBOLS USED IN TEXT

<u>Symbol</u>	<u>Definition</u>
A	area
A_s	nacelle surface area
C	nacelle diffuser expansion loss factor
C_d	nacelle drag coefficient
C_{ds}	strut drag coefficient
C_{dsp}	spray drag coefficient
C_f	Schoenherr friction coefficient
C_s	Factor for strut diffuser expansion loss
c	strut chord
D	diameter
d_m	mean diameter
f	Moody pipe friction factor
g	acceleration of gravity
h	head loss
H_a	atmospheric head
H_{sv}	net positive suction head
K_e	strut diffuser expansion loss coefficient
K_j	junction mixing loss coefficient
K_t	form loss coefficient
L	nacelle forebody length
L_d	nacelle diffuser length
N	pump RPM
P	total pressure
p	static pressure

<u>Symbol</u>	<u>Definition</u>
pv	vapor pressure
Q	volume flow rate
q	dynamic pressure
RPR	ram pressure recovery factor
R	bend radius of elbow centerline
RO	elbow internal duct radius
S	suction specific speed
T	thrust
t	strut thickness
V	velocity
α	angle of attack
β	nozzle depression angle
θ	diffuser half angle
λ	divergence loss factor
ρ	density of salt water
σ	cavitation number
ϕ	junction angle

<u>Subscript</u>	<u>Definition</u>
aux	auxiliary inlet
c	combined main and auxiliary flow
i	main inlet
j	jet
n	nacelle
t	nozzle throat
o	free stream condition

SubscriptDescription

1	nacelle inlet
2	nacelle diffuser exit
6	pump inlet

1. INTRODUCTION

Waterjet propulsion has attracted growing interest in recent years due, in part, to the trend toward the development of faster and more non-conventional vehicle types such as hydrofoils and surface-effect ships. These vehicles have made the potential advantages of waterjet propulsion systems more attractive, while at the same time lessening some of its disadvantages. Among the possible advantages are:

- a. Reduced vibrations and avoidance of propeller cavitation noise
- b. Fewer system components
- c. Elimination of underwater appendages
- d. Simpler transmission machinery
- e. Capable of use in shallow water operations
- f. Steering control directly from the propulsor

Additionally, the low overall system efficiency which causes waterjet propulsion to compare unfavorably with conventional propulsion systems at lower speeds is less of a penalty at higher speeds.

In light of the increasing interest waterjet propulsion, a number of studies have been made reviewing the important parameters involved in the selection of system components; as well as considerations for overall system design. This led naturally to efforts to computerize the design process and the most recent, and most comprehensive, of these efforts is the computerized method for design optimization of waterjet propul-

sion systems for hydrofoils developed by R.P. Gill, B.T. Conner, and R.C. Percival (references 1, 2 and 3).

It is desirable to have a compatible method of evaluating existing or proposed systems which can be used in conjunction with such a design method. This will enable existing systems, as well as alternative design proposals, to be evaluated on a common basis with the computer produced optimum design. It would also permit the performance of the computer produced design, or any other design, to be predicted at speeds other than the design speeds.

This report presents a revision of the computerized design method developed by Gill, Conner and Percival which permits it to be used for system evaluation as well as for its original purpose of design. The inclusion of the evaluation program within the design program permits common portions of the program to be used for both evaluation and design and also permits the off-design performance to be predicted in conjunction with the design computations. Additionally, it will permit the accuracy of the program to be analyzed since it can be used to predict the performance of existing systems and the predicted performance can then be compared with measured performance. This could lead ultimately to refinements and improvements in both the design and evaluation portions of the program.

This report describes the methodology used in the program, presents an analysis of results obtained by using the program, and provides a users' manual as an aid in utilizing the program

and interpreting the output obtained from the program. The methodology used for evaluation is presented in detail, and the design routine is summarized briefly. The design routine has been analyzed for sensitivity of a basic design to variations in normally fixed parameters and to variations in initial values of the independent variables. The evaluation routine has been tested by evaluating the performance of a system designed by the program at design and off-design conditions. The users' manual, contained in appendices to this report, describes the use of the program for design as well as for evaluation; and it contains listings of required and optional inputs, sample controlling programs, and sample outputs.

2. METHODOLOGY USED FOR EVALUATION

2.1 ASSUMED GENERAL SYSTEM CONFIGURATION

The general configuration of the waterjet propulsion system used for the evaluation routine is the same as that which is used in the design routine, and is depicted in Figure 1. The components of the system, in order from inlet to output, are:

1. The nacelle or ram inlet, which includes the auxiliary inlet, if any, and which is assumed to be of a sub-cavitating type for the calculations.
2. The strut elbow which derects flow from the nacelle into the strut and which generally contains either splitter vanes or turning vanes to reduce internal losses.
3. The strut which supports the foils and carries the water into the hull, and which is generally used as a diffuser in order to avoid cavitation in the pump inlet piping.
4. The hull elbow which directs the flow from the strut to the pump inlet piping.
5. The pump inlet piping which consists of one or two elbows and either a junction, divergence or straight pipe depending upon the total number of pumps used.
6. The pump which is driven by a gas turbine engine through a reduction gear.
7. The nozzle or jet which discharges the water through

either the transom or the bottom of the hydrofoil at a high speed thereby producing the required thrust.

The program assumes that the usual two strut configuration is utilized.

2.2 GENERAL PROCEDURE FOR EVALUATION

The evaluation procedure assumes that the geometry of the system is defined in its entirety and that a reliable estimate of ship drag, ie. that portion of total drag not directly attributable to propulsion system components, is available for the speeds at which the system will be evaluated. Estimated values of those components of drag due to the propulsion system; ie. nacelle drag, strut drag and spray drag, are added to the ship drag to obtain the estimated total drag and, therefore, the estimated required thrust. The required flow rate and jet velocity are related to the required thrust by the equation,

$$T = \rho Q (V_j \cos(\beta) - V_o) \quad (1)$$

The required flow rate and required jet velocity, as well as the required nacelle inlet velocity, can thus be determined by using equation (1) together with the one-dimensional continuity equation for an incompressible fluid,

$$A_i V_i = A_j V_j = Q \quad (2)$$

The flow rate and inlet and jet velocities, together with the system geometry, are sufficient to permit calculation of all of the head losses in the system ducting. The total system head loss and the flow rate can then be used to calculate the required pump speed and shaft horsepower.

The required flow rate and inlet velocity, together with ship speed and system geometry, are also used to calculate revised estimates of nacelle drag, strut drag and spray drag. These values are added to ship drag to obtain a new estimate of total drag. If the revised total drag differs by more than one per-cent from the previous estimate of the total drag, the new estimate is used to calculate a new flow rate and new inlet and jet velocities. The procedure is then repeated until a total drag estimate is obtained which differs by less than one-percent from the previous estimate. The calculated system performance is then output by the program.

The evaluation routine also calculates the structural weight of the system components and the weight of water entrained in the system using the same methods as those used in the design routine. This provides an additional check to ensure that the evaluated system is the same as the designed system. Additionally, when an existing system is evaluated, the calculated weights may be compared with actual weights in order to provide a check of the weight prediction methods used in the program. This could eventually result in improvements in the weight estimation models used.

2.3 SPECIFIC EVALUATION METHODOLOGY

The evaluation routine is begun by inputting the required parameters to the program and calculating the required flow rate and jet and inlet velocities in the manner described in the previous section. The parameters which must be input to the

program are:

- a. Displacement
- b. Prime mover
- c. Cross sectional areas and shapes of all system components
- d. Lengths of all system components
- e. Bend angles of all elbows in the system
- f. Nozzle depression angle
- g. Pump type and number of stages
- h. Pump blade tip diameter
- i. Pump characteristics at design specific speed
- j. Depth of submergence of nacelle
- k. Height of pump centerline above waterline
- l. Craft speeds for which system is to be evaluated
- m. Craft drag at evaluation speeds

A more detailed list of both required and optional inputs, including the required dimensions and the variable names used in the program, is contained in Appendix C.

Since it is assumed that the system is of a subcavitating design, an important consideration in the evaluation of system performance is a check to determine whether cavitation is occurring within the system at the evaluation speed. Checks for cavitation are made for those points in the system where it is deemed most likely to occur. Within the program, cavitation calculations are performed concurrently with other calculations; but, for the sake of clarity and continuity in this report, they will be described separately in section 2.4.

Head loss, drag and estimated weight calculations are performed for each component consecutively in the order in which the flow passes through them, thus the first component to be evaluated is the nacelle. Losses in the nacelle are calculated as three separate components; lip losses, pipe losses and diffuser losses.

When the nacelle subroutine is entered during evaluation, the first operations performed are the calculation of inlet velocity ratio, V_i/V_o from the input values of V_i and V_o ; and calculation of free stream static pressure, free stream dynamic pressure, incipient cavitation number, and total pressure for the evaluation speeds.

$$q_o = \frac{1}{2} \rho V_o^2$$

$$\sigma_{oi} = (p_o - p_v) / q_o$$

$$P_o = p_o + q_o$$

After checking for external cavitation and for cavitation on the inlet lip, as described in section 2.4, the subroutine proceeds with the calculation of lip losses. Lip losses are calculated differently depending on whether or not a positive indication of inlet lip cavitation was obtained. If cavitation was indicated, the auxiliary inlet area which must be used to avoid cavitation is calculated. To accomplish this, the first step is determining the maximum permissible inlet velocity ratio by interpolating in the data tables.¹

¹The data tables referred to in discussion of the nacelle eval-

uation are the same as those used in the design routine and are tabulations of the information contained in Tables 1 through 12 and Figures 4 through 20 of reference 2.

$$(V_i/V_o)_{\max} = f(L/D_m, \sigma_{oi}, \alpha)$$

The required auxiliary flow rate is then calculated.

$$Q_{\text{aux}} = Q_c - Q_i$$

Where

$$Q_i = A_i (V_i/V_o)_{\max} V_o$$

and Q_c is the required total flow per nacelle.

The ram pressure recovery of the lip is a tabulated function of the inlet velocity ratio, now limited to $(V_i/V_o)_{\max}$.

The total pressure inside the lip is thus

$$P_i = \text{RPR}_{\text{lip}} q_o + p_o$$

and the static pressure inside the lip is

$$P_i = P_i - \frac{1}{2} \rho V_i^2.$$

Pressure recovery of the auxiliary inlet was assumed to be 0.8. The total pressure inside the auxiliary is then given by

$$P_{\text{aux}} = \text{RPR}_{\text{aux}} q_o + p_o$$

The static pressure inside the auxiliary inlet must be the same as that previously calculated for inside the main inlet lip.

Therefore, the velocity through the auxiliary inlet can be determined by

$$V_{\text{aux}} = [2(P_{\text{aux}} - P_i)/\rho]^{1/2}.$$

The auxiliary inlet area required is then

$$A_{\text{aux}} = Q_{\text{aux}}/V_{\text{aux}}.$$

If this area is less than the total available auxiliary

inlet area, the total pressure of the combined flow is calculated as the mass weighted average of the combining flows.

$$P_c = (Q_{aux}P_{aux} + Q_iP_i)/Q_c$$

The average velocity of the combined flow is calculated from the dynamic pressure

$$V_i = [2(P_c - p_i)/\rho]^{1/2}$$

and the losses are then obtained directly from the pressure recovery coefficient to this point, which is

$$RPR_c = (P_c - p_o)/q_o.$$

If lip cavitation was not indicated initially, then no auxiliary inlet area is required to be used, and the ram pressure recovery coefficient of the lip is determined directly as a tabulated function of V_i/V_o .

$$\Delta P_d = CK_t q_i$$

where C is the diffuser expansion loss factor and K_t is the form loss coefficient.

$$C = 3.19 \times 10^{-3} \theta^2 + 8.452 \times 10^{-4} \theta$$

$$K_t = (1 - A_1/A_2)^2$$

θ is the diffuser half angle.

$$\theta = \tan^{-1} [(D_2 - D_1)/2L_d]$$

Pipe loss in the diffuser is

$$\Delta P = f(L_d/d_m)q_i$$

where f is the Moody pipe friction factor which is calculated within the routine as a function of Reynold's number.

Total internal losses in the nacelle are then

$$\Delta P_n = (1 - P_c)(\frac{1}{2}\rho V_o^2) + [CK_t + f(L_d/d_m) + fL_{aux}] \frac{1}{2}\rho V_i^2.$$

The nacelle drag coefficient is given by

$$C_D = C_f \left[1 + 1.5(D_m/L_N)^{3/2} + 7(D_m/L_N)^3 \right]$$

where C_f is the Schoenherr friction coefficient which is calculated as a function of Reynold's number in a separate subroutine.

The total external drag is then

$$D = C_D \frac{1}{2} \rho V_o^2 A$$

where A is the external surface area of the nacelle, calculated in the same way as in the design routine, reference 2.

Calculations for the strut elbow are performed by the same subroutine that handles all of the elbows in the system and calculations for all of the elbows are performed in the same manner as in the design. For purpose of the calculations an elbow is fully defined by specifying the following parameters: width, depth, cross sectional area, bend angle, and radius ratio. The radius ratio, R/R_0 , is the ratio of the radius of the centerline of the bend to the internal radius of the duct. The inlet and outlet cross sectional areas are assumed to be equal.

In order to maintain compatibility with the design routine, the shapes of the elbows are assumed to be the same as the shapes which are assumed in the design routine. Specifically, the strut elbow is assumed to be rectangular with the depth equal to twice the width, the hull elbow is assumed to be rectangular with the depth equal to one half the width, and the pump elbow is assumed to be circular in shape. The radius ratios and bend angles may be specified in the calling program; but,

if they are not, they will be assigned the same values as in the design program. The cross sectional areas must be specified.

The strut subroutine requires that the thickness to chord ratio, the thickness and chord at the root, the thickness and chord at the tip, and the inlet and outlet areas be specified. The shape of the strut internal ducting is assumed to be the same as that of the strut elbow. The strut length and the diffuser equivalent angle, 2θ , need not be specified as they are calculated within the subroutine from other specified parameters.

The head losses within the strut diffuser are a combination of friction losses and expansion losses. The friction loss coefficient is the Moody friction factor, calculated in the same manner as in the nacelle subroutine. The expansion loss coefficient is given by

$$K_e = C_s (1 - A_i/A_o)^2$$

where C_s is dependent upon the diffuser equivalent angle, 2θ , and is determined by interpolation in data tables contained in the subroutine.

Two external drag components, strut drag and spray drag, are attributable to the strut and are calculated in the strut subroutine. The drag coefficients for both components are calculated as functions of the thickness to chord ratio. The strut drag coefficient is given by

$$C_{ds} = 2C_{fs} [1 + 2(t/c) + 60(t/c)^4]$$

and the spray drag coefficient is

$$C_{dsp} = 0.03(t/c).$$

The hull elbow losses are determined in the same manner as previously described for the strut elbow. It may be noted, however, that the assumed shape for the hull elbow gives it a more favorable aspect ratio than that of the strut elbow. This, together with the lower inflow velocity due to diffusion in the strut, will result in lower losses in the hull elbow than in the strut elbow.

The pump inlet piping is that portion of the ducting which connects the hull elbow to the pump inlet. It consists of a straight pipe, the pump elbow, and a transition piece which is either a junction, divergence, or another straight pipe depending on whether there are one, four, or two pumps respectively.

The geometry of the pump inlet piping is basically determined by previous assumptions and previously required inputs with the only remaining variables being cross sectional area of the piping and the junction angle, ϕ . The assumption is made that the mean velocity is constant throughout the pump inlet piping, thus the areas of the ducting are fixed by specifying the area of the hull elbow. The junction angle is assumed to be zero unless otherwise specified. The divergence angle need not be specified since it is calculated within the routine as a function of the distance between the specified pump location and the specified strut location.

The athwartships pipe makes a transition from a rectangular shape to a circular shape, but losses due to change of

shape are neglected in the calculations. Losses in all straight pipe sections are calculated using the Moody friction factor in the manner previously described. Losses for the pump elbow are calculated in the same manner as for the other elbows in the system.

The mixing loss coefficient is dependent on the junction angle, θ , and the divergence loss factor, λ , and is determined by

$$K_j = 1 + \lambda - 2\cos(f(\phi))$$

where

$$f(\phi) = 1.4\phi - 0.00583\phi^2.$$

The divergence loss factor is computed by interpolation in a data table which is taken from Figure 15 of reference 1. When a divergence is used, another elbow is required at the pump entrance. This elbow is treated in the same manner as the other elbows except that the angle of the bend is equal to the divergence angle.

The input parameters required for the nozzle calculations are throat area, jet area, and nozzle depression angle. Nozzle length is calculated from the pump exit position and the nozzle depression angle. Nozzle head loss is calculated in the same manner as in the design routine, where head loss is given by

$$h_n = [f(L_j/2)/(D_t - D_j) - 16/(1 + D_t)^4] v_j^2 / D_j 2g$$

when D_j is greater than 1. For D_j less than 1, head loss is assumed to be 1.5% of dynamic head.

With all head losses in the system having thus been deter-

mined; they are then summed, the required pump head is calculated and the pump subroutine is entered for calculation of pump performance. The required pump head is given by

$$H_p = (V_j^2 - V_o^2)/2g + h_{\text{elev}} + \Delta h_{\text{total}}.$$

The calculations of pump performance are performed in essentially the same manner as that used by the design routine for calculating performance at cruise speed. These methods are described in detail in reference 3 and will only be summarized here, with emphasis placed on the points where the evaluation routine differs.

The input parameters to the pump subroutines for evaluation are pump length, pump type, number of impellers, impeller tip diameter, and a set of performance characteristics which are representative of the design condition; ie. a pump speed, pump head and flow rate which are representative of the design specific speed of the pump. Additionally, the subroutine utilizes the required flow rate, required pump head, and various craft characteristics which are either input or calculated elsewhere in the evaluation routine.

As in the design routine, the pump characteristic head and efficiency curves are assumed to be parabolic and off-design pump speed and efficiency are calculated on that basis. The off-design pump speed is obtained from the equation

$$H/H_D = A(Q/Q_D)^2 + B(Q/Q_D)(N/N_D) + C(N/N_D)^2$$

where subscript D indicates the design condition. The equation is solved for N, which is the off-design pump speed. The coef-

ficients, A, B and C, are determined for the particular pump type in the same manner as in the design routine. The off-design efficiency is calculated in an identical manner after the design point efficiency has first been determined as a function of impeller tip diameter in the same way as in the design routine.

In the design routine, the flow coefficient, head coefficient and suction specific speed are determined and output for the design condition only. In the evaluation routine these parameters are determined and output for the evaluated conditions in order to more fully describe the pump performance at these conditions.

During design, the pump subroutine calls the gear and fuel subroutines which are used to calculate reduction gear weight and weight of fuel required for specified endurance at cruise speed. These subroutines are also called during evaluation, but their use is somewhat different than in design. For evaluation the gear ratio must be specified whereas in design it is calculated as the ratio of prime mover design speed to pump design speed. The type of reduction gear may be either specified, or selected on the basis of gear ratio and number of inputs and outputs in both design and evaluation routines. The fuel subroutine does not calculate a fuel weight during evaluation, but instead, it calculates the fuel consumption rate at the evaluated speed, assuming the craft is at specified displacement. If the fuel weight is desired to be included in the

total system weight output at the end of evaluation, it must be specified in the inputs.

2.4 CAVITATION CONSIDERATIONS

There are six points, two external and four internal, where cavitation is most likely to occur, and an important part of the evaluation of a system is a determination of whether cavitation is expected to occur at any of these points at the evaluated speeds. The six points which must be checked for cavitation are:

1. The exterior surface of the nacelle.
2. The exterior surface of the strut.
3. Inside the nacelle inlet lip.
4. Inside the strut elbow.
5. At the pump inlet.
6. On the pump impeller.

The check for cavitation on the exterior of the nacelle is accomplished by interpolating in a data table to obtain the forebody length to maximum diameter ratio, L/D_m , for a nacelle having an external pressure coefficient equal to the negative of the free stream incipient cavitation number. The inlet to maximum diameter ratio, D_i/D_m , corresponding to this L/D_m is determined by interpolating in another data table. This value is compared with the actual D_i/D_m and if the actual value is greater cavitation can be expected to occur and an indication to that effect is printed in the output data.

The cavitation number for flow around a strut can be approximated by

$$\sigma = [1.15(t/c) + 1]^2 - 1.$$

The incipient cavitation number for flow around the strut is

$$\sigma_i = (H_a - p_v)g / \frac{1}{2}(1.137V_o)^2.$$

If $\sigma_i < \sigma$ at the evaluated speed, then cavitation is likely and is so indicated in the output data.

The check for cavitation on the inside of the nacelle inlet lip is accomplished by interpolating in a data table to determine the limiting inlet velocity ratio beyond which cavitation will occur inside the lip.

$$(V_i/V_o)_{\max} = f(L/D_m, \sigma_{oi}, \alpha)$$

If V_i/V_o at the evaluated condition is greater than $(V_i/V_o)_{\max}$ cavitation will occur unless sufficient auxiliary flow is provided to reduce the effective inlet velocity to that corresponding to $(V_i/V_o)_{\max}$.

The auxiliary inlet area required to provide sufficient auxiliary flow to avoid cavitation is calculated in the manner described in Section 2.3. If the required auxiliary inlet area is greater than the available auxiliary inlet area, then cavitation can not be avoided and the output of the program will indicate that inlet lip cavitation will occur at the evaluated speed. If cavitation can be avoided only by using some portion of the auxiliary inlet area, then the output will indicate the auxiliary inlet area which is required.

The incipient cavitation number on the turning vanes in the strut elbow is given by

$$\sigma_{tvi} = (p_2 - p_v) / (\frac{1}{2}\rho V_2^2)$$

where subscript 2 indicates conditions at the exit of the nacelle

diffuser. The incipient cavitation number, σ_{tvi} , is a characteristic number which has been determined experimentally, thus the critical velocity at the diffuser exit can be determined.

$$V_{crit} = [\sigma_{oi} + 1 - P_{loss}/(\frac{1}{2}\rho V_o^2)]^{\frac{1}{2}} [V_o/(1 + \sigma_{tvi})^{\frac{1}{2}}].$$

These two values are compared. If V_2 max is greater than V_{crit} cavitation is likely to occur and will be indicated in the program output.

Cavitation is assumed to occur in the pump inlet if the local static pressure, p_6 , is less than vapor pressure, p_v ; or more precisely, if local dynamic head is less than local total head minus vapor head, ie. $(P_6 - p_6) < (P_6 - p_v)$,

where $P_6 - p_6 = (Q/A_6)^2/2g$

and $P_6 - p_v = V_o^2/2g + H_a - P_{loss} - P_v$.

If this condition exists at the evaluated speed, an indication to that effect is printed in the program output.

Cavitation on the pump impeller is not intrinsically determined by the program, but the suction specific speed, S , at which the pump is operating at the evaluated speed is determined and printed out. The suction specific speed at which cavitation occurs may vary depending upon the pump design; but, by knowing the suction specific speed at which the pump is required to operate, the user can readily determine if the limiting value is exceeded. The suction specific speed at the evaluated condition is determined by

$$S = NQ^{\frac{1}{2}}/H_{sv}^{0.75}$$

where H_{sv} , the net positive suction head, is given by

$$H_{sv} = V_o^2/2g + H_a - p_v - P_{loss}.$$

3. DESIGN METHODOLOGY

The design methodology used in the design portion of the program is described in detail in references 1, 2, and 3, and is summarized here to lend continuity and completeness to this report.

The design routine uses a directed pattern search to select a system which is optimized from the standpoint of having the lowest overall system weight of any satisfactory system. The system weight includes all structural and machinery weight attributable to the propulsion system, as well as the weight of all water entrained in the system above the waterline and the weight of fuel required to provide the specified endurance at cruise speed. The craft displacement is held constant during the optimization routine, thus the selection of a minimum weight propulsion system maximizes the payload for that particular craft.

The variable parameters used in the design optimization are jet velocity ratio, V_j/V_o , inlet velocity ratio, V_i/V_o , and nacelle inlet to maximum diameter ratio, D_i/D_m . The pattern search subroutine selects different values of these parameters for each design iteration and retains the values which produced the minimum weight system. When the search pattern has been completed, V_j/V_o , V_i/V_o , and D_i/D_m are reset to the values corresponding to the minimum weight system and the design routine is entered for a final time in order to reproduce that system.

In addition to V_j/V_o , V_i/V_o , and D_i/D_m ; the design routine requires that at least the following information be specified to define the craft for which the propulsion system is being designed:

- a. Displacement
- b. Range
- c. Prime mover
- d. Craft speed at take-off and cruise
- e. Craft drag at take-off and cruise
- f. Depth of submergence of foil
- g. Distance of strut from transom
- h. Distance of pump exit from transom
- i. Height of pump centerline above waterline

The design routine begins by using the inlet and jet velocity ratios chosen by the pattern search subroutine, together with the specified craft speed, to calculate the required inlet and jet velocities at the cruise condition. The optimum nozzle depression angle is then determined by

$$\beta = \tan^{-1}(D_{\text{total}}/\Delta).$$

These values are then used to calculate the required flow rate and the required inlet and jet areas.

The design routine then proceeds to size the components of the system in the same sequence as previously described for the evaluation routine; and, after each component has been sized, losses and drags are determined in the same manner as in the evaluation routine.

The nacelle inlet to maximum diameter ratio, as selected by the pattern search subroutine, determines the amount of diffusion possible prior to the strut elbow. Since the inlet diameter has already been determined, the maximum diameter is therefore fixed by the diameter ratio. The length of the nacelle forebody is determined by selecting the length to maximum diameter ratio which is just at incipient cavitation on the external surface at cruise speed. The ratio of inlet to maximum diameter which corresponds to the forebody shape thus chosen is compared with that ratio previously specified. If the previously specified ratio exceeds the maximum ratio allowed by cavitation, the forebody is resized using the maximum allowable ratio. An iteration is then performed to determine the overall nacelle length which will result in the least total power loss due to external drag and diffusion in the ducting.

The inlet area of the strut elbow is set equal to the exit area of the nacelle diffuser. The elbow shape, radius ratio and bend angle are determined according to assumptions previously described for the evaluation routine. The number of splitters or guide vanes is then determined to minimize losses in the elbow.

The strut diffuser inlet area and shape is set equal to the exit of the strut elbow, and the diffuser area ratio is chosen to avoid cavitation in the pump inlet piping. The external surface of the strut is sized as necessary to enclose the ducting, and the shape is chosen to avoid external cavi-

tation.

The nacelle, strut and spray drags which have been calculated during this iteration are compared with previous estimates of these drags. If a significant difference exists, the drag estimates are revised and calculations begin anew; otherwise the rest of the system is then designed.

The hull elbow is designed in the same manner as the strut elbow, and the configuration of the pump inlet piping is selected according to the number of pumps. The pump inlet piping begins with a transition from the shape of the hull elbow to a circular shape, and the rest of the piping is circular in shape. The size of the pump inlet piping is chosen to maintain a constant velocity throughout its length.

The nozzle is assumed to have an inlet area equal to the pump outlet area. This and the jet area which has been previously determined are used to calculate losses in the nozzle.

The ducting losses prior to the pump are used to determine net positive suction head, and nozzle losses are added to determine the required pump head. Two basic pump designs are considered; axial with an inducer impeller, and centrifugal with double suction impellers. The alternative pumps are sized by using assumed specific speeds, flow coefficients and impeller diameter ratios for each alternative. The number of axial stages is determined by cavitation criteria, while the number of stages in the centrifugal pump is chosen to minimize combined pump, gear and fuel weight. The required gear ratio is determined

as the ratio of prime mover normal operating speed to required pump speed at take-off and this ratio, together with the required shaft horsepower at take-off, determines the type and size of gear to be used. The fuel weight required to provide the specified endurance at cruise speed is calculated for each alternative design, and that pump design which requires the lowest combined weight of pump, gear and fuel is then chosen as the pump to be used by the design.

4. DISCUSSION OF RESULTS

Sample outputs for both the evaluation and the design routine are contained in Appendix C. The format used for output from the evaluation routine is essentially the same as that used for design. This permits maximum compatibility within the output subroutine; and, additionally, facilitates comparison of data produced by the design routine with that produced by the evaluation routine for the same or any other system.

The design routine was analyzed for sensitivity of a basic design to variations in initial values of the independent variables and to variations in normally fixed parameters. The basic design used in the analysis was for a 750 ton hydrofoil with a take-off speed of 30 knots, a cruise speed of 45 knots, and an endurance of 2000 miles at cruise speed. Two LM2500 gas turbines were the specified prime movers, and the specified number of pumps was two.

In most cases there was little or no variation in the resulting system design when the starting values of the independent system parameters were varied. In some cases, however, significant variations occurred with total system weight varying by as much as 3.5%. In those cases where results did vary, the jet velocity ratio of the resulting design deviated very little from that of the basic design. The inlet velocity ratio and nacelle diameter ratio both varied over a considerable range, however, consequently the changes in system weight resulted primarily from changes in the nacelle and strut designs. This suggests

that the optimum design is primarily defined by the jet velocity ratio and that the inlet velocity ratio and nacelle diameter ratio become important only after the optimum jet velocity ratio has been determined. This may also explain the apparent lack of correlation between inlet velocity ratio and other parameters in designs produced by the program. Further study of this problem is needed and it may be necessary to revise the design routine so that once an initial solution is obtained, the jet velocity ratio is fixed at the resulting value while further variations in the other two system parameters are explored.

The results showed a considerable sensitivity to variations in craft drag; particularly that component of drag which is not directly attributable to the propulsion system, referred to here as ship drag. Ship drag is a craft characteristic which is independent of the propulsion system design and it is held constant at the input value throughout the design calculations. Relatively small variations in ship drag produced comparatively large changes in total system weight. This sensitivity is apparently due to the fact that the cases studied for this report fall in a region where the selection of a system design is constrained by a power limitation imposed by the specified prime movers. If there were no power limitation, the optimum system would have a very high jet velocity ratio and consequently a low volume flow rate thereby reducing both ducting weight and the weight of entrained water. If take-off drag, and therefore the required take-off thrust, is so high that a power limita-

tion is imposed; then the jet velocity must be reduced in order to reduce the required pump head sufficiently to bring required power back within limits. The reduction in thrust due to the lowered jet velocity must then be compensated for by increasing the flow rate. This requires larger and heavier ducting and results in a larger volume of entrained water. This suggests that the optimum combination of craft and propulsion system would be one in which the propulsion system design selected by the design routine just bordered on being power limited.

In view of the above discussion, it appears that the propulsion coefficient based upon installed horsepower is an important system parameter since it is basically a measure of the fraction of installed horsepower required to overcome drag.

$$PC_i = (D_{total} V) / (550 SHP_i)$$

The propulsion weight fraction and the propulsive coefficient based on used horsepower at cruise speed are plotted against the installed power propulsive coefficient at take-off in Figures 2 and 3 respectively. The curves are plotted for designs in which ship drag was the only parameter varied. The other points plotted represent designs in which other parameters were varied.

The results also showed some sensitivity to variations in the strut length. This was to be expected since reducing the strut length would reduce the required pump head as well as reducing the amount of ducting and entrained water in the system. The sensitivity was not nearly as pronounced as the sensitivity to ship drag.

The evaluation routine was tested by evaluating systems designed by the design routine at both design and off design speeds. The results were satisfactory and, when the evaluation conditions were the same as design conditions, the operational characteristics of the system as determined by the evaluation routine were nearly identical to those calculated by the design routine.

5. CONCLUSIONS AND RECOMMENDATIONS

A computer program to evaluate the performance of a waterjet propulsion system has been developed and incorporated into the program for design optimization of waterjet propulsion systems. The evaluation routine can be used successfully to evaluate a wide array of possible system configurations. A lack of available data on the configuration and performance of existing systems precluded the use of the evaluation routine to analyze the performance of an actual system for this report. When such information is available, it is recommended that this be done. The results obtained from the evaluation routine could then be compared with actual performance data to further evaluate the accuracy of the methods used in the program.

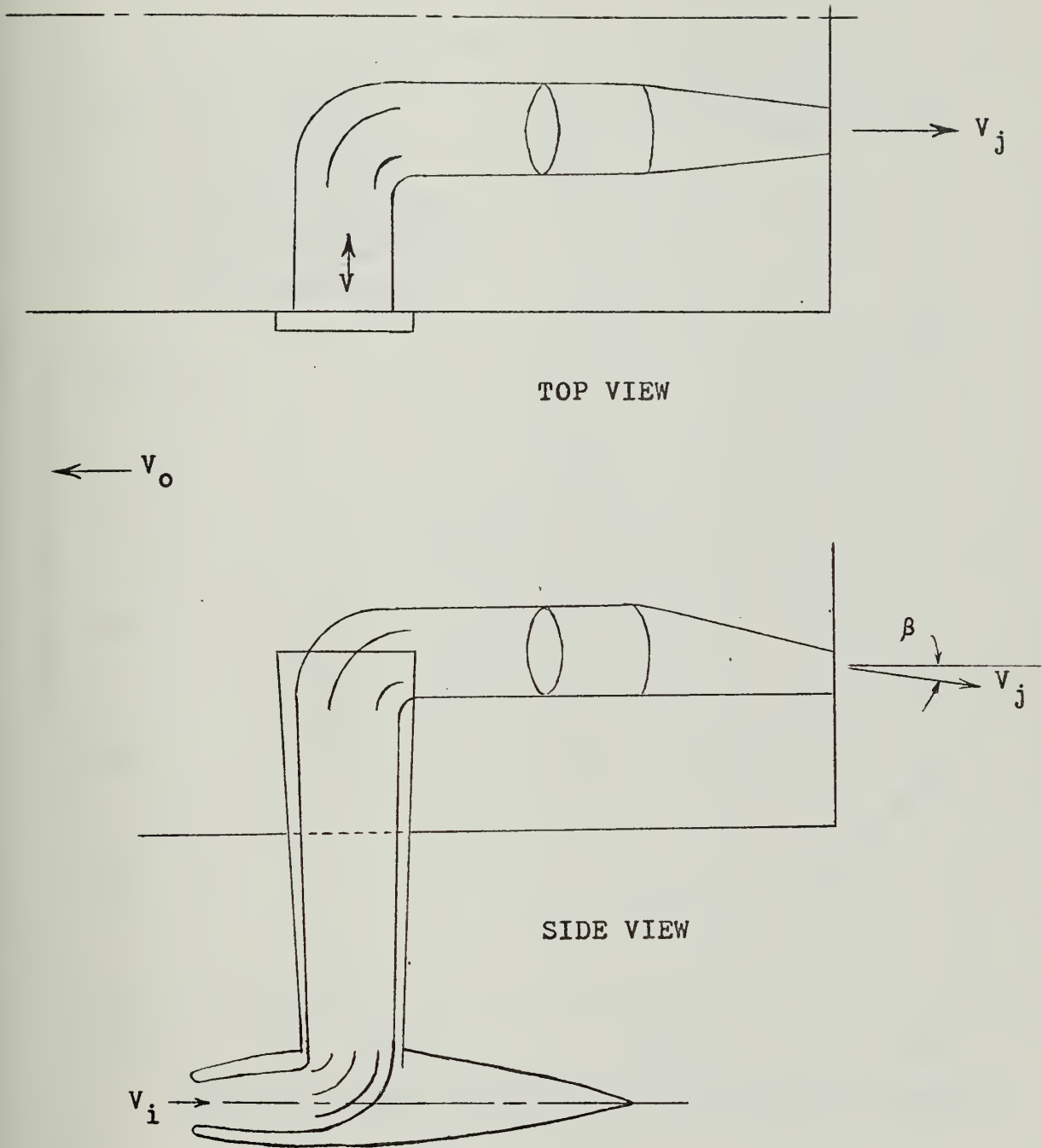
The sensitivity of the design routine to initial values of the independent variables requires further study. The use of either a random search or an additional pattern search is suggested to explore further variations in inlet velocity ratio and nacelle diameter ratio once the best jet velocity ratio has been determined in the manner currently used in the program.

The results of the design routine are highly sensitive to variations in ship drag when the available power is barely adequate for the design conditions. This indicates that the total hydrofoil design should be carefully tailored to the prime mover so that the total power required at design conditions is somewhat less than the total available power in order to allow for growth. If the hydrofoil is designed so that maximum

available horsepower is required at either take-off or cruise condition, the specified craft conditions must include an adequate margin for growth. If not, a serious degradation of performance would occur when the inevitable growth took place because, if maximum power is already being used, there would be no way of increasing available thrust to compensate for increased drag without exceeding the power limitations of the prime mover.

BIBLIOGRAPHY

1. Gill, R.P., Design Optimization of Waterjet Propulsion Systems for Hydrofoils, Ocean Engineer Thesis, M.I.T., May 1972.
2. Conner, B.T., A Study of Ram Type Intake Parameters, Ocean Engineer Thesis, M.I.T., June 1972.
3. Percival, R.C., Optimization of Waterjet Propulsion Pumps for Hydrofoil Applications, Ocean Engineer Thesis, M.I.T., May 1972.
4. Brandau, J.H., Performance of Waterjet Propulsion Systems - A Review of the State of the Art, Journal of Hydronautics, vol. 2, no. 2, pp. 61-73, April 1968.
5. Arcand, L. and Comolli, C., Optimization of Waterjet Propulsion for High Speed Ships, Journal of Hydronautics, vol. 2, no. 1, pp. 2-8, 1968.
6. Chironis, E.P. ed., Gear Design and Application, McGraw-Hill Book Co., New York, 1967.
7. Hatte, R. and Davis, H., Selection of Hydrofoil Waterjet Propulsion Systems, AIAA Paper 66-732, August 1966.
8. Hoerner, S.F., Fluid Dynamic Drag, published by author, 1965.
9. Johnson, V.E., Waterjet Propulsion for High Speed Hydrofoil Craft, AIAA Paper 64-306, June 1964.
10. Levy, J., The Design of Waterjet Propulsion Systems for Hydrofoil Craft, Journal of Marine Technology, vol. 2, no. 1, pp. 15-26, January 1965.
11. Traskel, J. and Beck, W.E., Waterjet Propulsion for Marine Vehicles, AIAA Paper 65-245, March 1965.



TOP VIEW

SIDE VIEW

FIGURE 1: GENERAL DUCTING CONFIGURATION

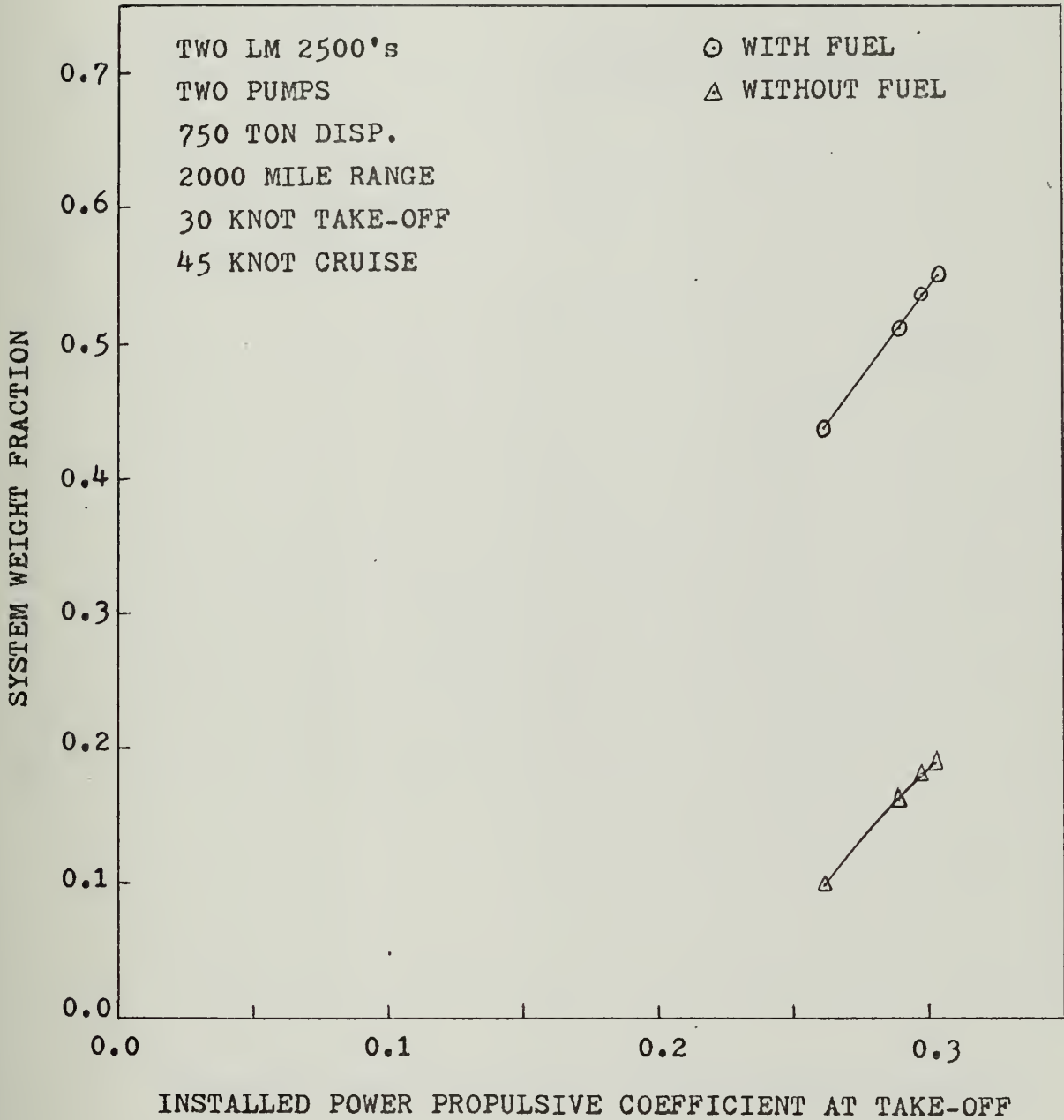


FIGURE 2: PROPULSION SYSTEM WEIGHT FRACTION VS. INSTALLED POWER PROPULSIVE COEFFICIENT AT TAKE-OFF

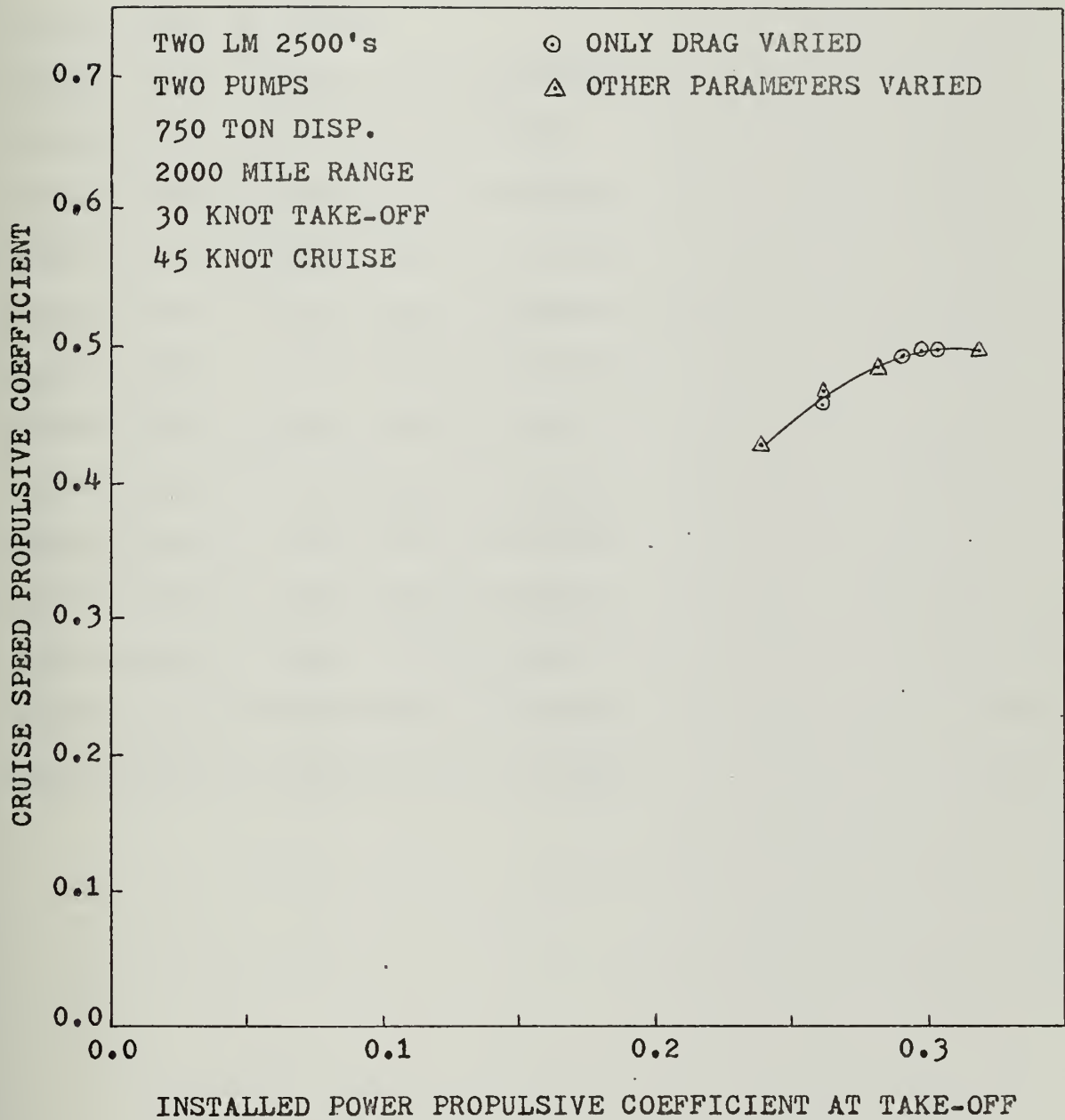


FIGURE 1: CRUISE SPEED PROPULSIVE COEFFICIENT VS. INSTALLED
 POWER PROPULSIVE COEFFICIENT AT TAKE-OFF

APPENDIX A

LIST OF SYMBOLS USED IN PROGRAM

SYMBOLS USED IN COMMON	43
SYMBOLS USED IN H2OJT NOT IN COMMON	56
SYMBOLS USED IN FCT NOT IN COMMON	58
SYMBOLS USED IN NACEL NOT IN COMMON	60
SYMBOLS USED IN ELBOW NOT IN COMMON	65
SYMBOLS USED IN STRUT NOT IN COMMON	67
SYMBOLS USED IN JUNCT NOT IN COMMON	69
SYMBOLS USED IN PIPE NOT IN COMMON	70
SYMBOLS USED IN DIVRG NOT IN COMMON	71
SYMBOLS USED IN NOZZL NOT IN COMMON	73
SYMBOLS USED IN PUMP NOT IN COMMON	74
SYMBOLS USED IN FUEL NOT IN COMMON	77
SYMBOLS USED IN PTRN NOT IN COMMON	78
SYMBOLS USED IN OUTPUT NOT IN COMMON	79

LIST OF SYMBOLS

USED IN COMMON

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
PARMS			System parameters
	VJVO		Jet velocity ratio
	VIVO		Inlet velocity ratio
	DIDM		Inlet diameter to maximum nacelle diameter ratio
DRAG			Drag estimates
	TDRAG(I)	I	Total craft drag at condition I, pound force
		1	Cruise condition
		2	Take-off condition
		3-5	Evaluation conditions
	STRTD(I)		Strut drag at condition I, pound force
	POD(I)		Nacelle drag at condition I, pound force
	SPRAY(I)		Spray drag at condition I, pound force
	REST(I)		TDRAG(I)-SPRAY(I)-STRTD(I)-POD(I), pound force
	VO(I)		Craft velocity at condition I, foot per second
	TRIM(I)		Craft trim angle at condition I, degree
FLOW			Flow characteristics
	Q(I)		Volume flow rate at condition I, cubic foot per second

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
	AIN		Inlet area, square foot
	AJET		Jet area, square foot
	AREA(ICOMP)		Ducting cross-sectional area at station ICOMP, square foot
	VJ(I)		Jet velocity at condition I, foot per second
	VI(I)		Inlet velocity at condition I, foot per second
ELBW			Elbow data
	XK(IELB)	IELB	Radius ratio of elbow IELB
		1	Strut elbow
		2	Hull elbow
		3	Pump elbow
		4	Divergence elbow
	RO(IELB)		Duct radius at elbow IELB, degree
	THATA(IELB)		Angle of bend of elbow IELB, foot
	WIDTH		Width of duct at elbow inlet
	DEPTH		Depth of duct at elbow inlet
	TYPE(3,ITYPE)		Contains name of elbow shape ITYPE
CHARS			System characteristics
	WGTS(LS,LC)		Weight of component LC, pound

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
		LS	
		1	Structure
		2	Water (or fuel)
		LC	
		1	Nacelle
		2	Strut elbow
		3	Strut diffuser
		4	Hull elbow
		5	Athwartships length
		6	Pump elbow
		7	Transition piece
		8	Pump
		9	Nozzle
		10	Reduction gear
		11	Fuel
		12	Prime mover
		13	Lift from nozzle depression
		14	Total weight
		15	Spare location
	CGS(LG,LC)		Centers of gravity of component LC, excluding fuel, gearbox, or prime mover, foot
		LG	
		1	Structure vertical center of gravity (from keel)

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
		2	Structure longitudinal center of gravity (from transom)
		3	Water vertical center of gravity, (from keel)
		4	Water longitudinal center of gravity, (from transom)
	DELH(I,ICOMP)		Total head loss up to and including component ICOMP at condition I, foot
		ICOMP	Same as defined except
		8	Nozzle head loss only
		9	Total system head loss (including elevation), DELH(I,9)=DELH(I,8)+ DELH(I,7)
		10-15	Spare locations
	CGSX		Total longitudinal center of gravity system excluding gearbox, prime mover and fuel, from transom, foot
	CGSZ		Total vertical center of gravity of system excluding gearbox, prime mover and fuel, from keel, foot
H2O			Sea water (3.5% salinity) properties
	TEMP		Temperature, degree Fahrenheit
	PV		Vapor head, foot
	RHOW		Density, pound force second squared per foot to the fourth

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
	GNU		Viscosity, foot squared per second
	HA		Atmospheric head, foot
TOLER			
	DELTA		Check for optimum system
PSUB			Pump system
	GERAT(NSTG)		Gear ratio required for pump NSTG
	SHP(I,NSTG)		Shaft horsepower per prime mover required at condition I for pump NSTG, horsepower
	RPM(I,NSTG)		Axial speed of pump NSTG at condition I, revolution per minute
	PERF(L,IENGN)		Prime mover IENGN Characteristics (L)
		L	
		1	Maximum normal horsepower at design speed, horsepower
		2	Maximum intermittent horsepower at design speed, horsepower
		3	Specific fuel consumption (SFC) at design speed and maximum normal horsepower, pound fuel per horsepower hour
		4	Design speed, RPM
		5	Prime mover weight, without auxiliaries, pound
	ETAP(I,NSTG)		Efficiency of pump NSTG at condition I

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
		I	Same as defined except
		7	Reduction gear
		6,8	Spare locations
SHIP			Craft characteristics
	DISP		Craft displacement, pound
	RANGE		Endurance, nautical mile
	BEAM		Beam, foot
	HS		Depth of submergence of foil, foot
	HE		Height of pump centerline above mean water, foot
	HCL		Height of pump centerline above keel, foot
	XLS		Distance of centerline of strut root from transom, foot
	XLPE		Distance of pump exit from transom, foot
	XLP		Length of pump, foot
NACLL			Nacelle characteristics
	DRAT		Diameter ratio, DI/DM
	DM		Maximum external diameter, foot
	AI		Inlet area per nacelle, square foot
	AIAUX		Auxiliary inlet area per nacelle, square foot
	ELEXT		Length of forebody, foot
	ELENT		Length of lip, foot

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
	ELAUX		Length due to auxiliary inlet, foot
	ELDIF		Length of diffuser, foot
	ELN		Length of nacelle, foot
	AAX(I)		Portion of auxiliary inlet area in use at condition I, square foot
CONST			Constants
	PI		3.14159265
	G		Acceleration of gravity, 32.174, foot per second squared
	RHOD		Density of steel, 480, pound per cubic foot
STRTC			Strut characteristics
	TC		Thickness to chord ratio
	T		Thickness at root, foot
	C		Chord at root, foot
	T1		Thickness at tip, foot
	C1		Chord at tip, foot
	CFM		Chord at flying water-line, foot
INDEX			Indices for program control
	IEVAL		Design/evaluation index
		<0	Evaluate at IEVAL points, no design
		=0	Design at cruise and take-off conditions only

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
		> 0	Design at cruise and take-off conditions and evaluate at IEVAL points
	IEQPT		
		0	No equipments or configuration entered; total system design/evaluation
		1	Number of gas turbines is specified
		3	Number of gas turbines and number of pumps are specified
	ISTRT		Initial program condition
		1	Design and/or evaluate
		3	Evaluate only
	NUMB		Final program condition
		2	Design only
		2+ IEVAL	Design and/or evaluate
	IENGN		Prime mover type
		1	TF 35
		2	TF 40
		3	Proteus, 1500 rpm
		4	Proteus, 1000 rpm
		5	Tyne 1A
		6	Tyne 1C
		7	FT12A
		8	LM 1500
		9	LM 2500
		10	FT4A-2C

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
		11	FT4A-12
		12	FT4C-2
	ITYPE		Elbow shape
		1	Ellipse
		2	Circle
		3	Rectangle
		4	Square
	ICOMP		Component index
		1	Nacelle outlet
		2	Strut elbow
		3	Strut diffuser exit
		4	Hull elbow
		5	Athwartships length
		6	Pump elbow
		7	Transition piece
		8	Pump inlet
		9	Nozzle throat
	NPUMP		Number of pumps
	NGT		Number of gas turbines
	IGEAR		Type of reduction gear
		1	Single reduction with idler
		2	Planetary
		3	Double reduction, double branch
ITABL			Interpolation parameters

<u>Label</u>	<u>Variable Names</u>	<u>Subscript</u>	<u>Description/Usage</u>
CDRAG			Computed drags
	CSTRT(I)		Computed strut drag at condition I, pound force
	CPOD(I)		Computed nacelle drag at condition I, pound force
	CSPRY(I)		Computed spray drag at condition I, pound force
WEGT			Pump, gear and fuel weights
	XWD(NSTG)		Pump NSTG dry weight, pound
	XWW(NSTG)		Pump NSTG wet weight, pound
	XWG(NSTG)		Gearbox weight associated with pump NSTG, pound
	XWF(NSTG)		Fuel weight associated with pump NSTG, pound
PUMM			Pump characteristics
	QQ(I)		Flow rate per pump at condition I, cubic foot per second
	D1S(NSTG)		Inlet tip diameter of pump NSTG, foot
	D2S(NSTG)		Exit tip diameter of pump NSTG, foot
	XNS(NSTG)		Specific speed of pump NSTG, cfs units
	SM(I,NSTG)		Suction specific speed of pump NSTG at condition I, cfs units
	PLP(NSTG)		Length of pump NSTG, foot
	NSTG		Indicator of pump type
	1		Axial pump with single inducer impeller

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
		2	Axial pump with inducer and one impeller stage
		3	Axial pump with inducer and two impeller stages
		4	Centrifugal pump with maximum of ten parallel double suction impellers
		5	Spare location
	SHI(I,NSTG)		Head coefficient of pump NSTG at condition I
	XIM		Number of parallel double suction impellers for centrifugal pump
			Number of impellers, not including inducer, for centrifugal pump
HEAD			Pump flow characteristics
	HPP(I)		Pump head at condition I, foot
	HSV(I)		Net positive suction head at condition I, foot
	THOM(I)		Thoma's cavitation index at condition I
	PHI(I,NSTG)		Flow coefficient of pump NSTG at condition I
	WF		Working variable for fuel weight, pound
	WG		Working variable for gear weight, pound
	FRATE(I)		Fuel consumption rate at condition I, pounds per hour
PINLP			Pump inlet piping parameters

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
	ALPHA		Junction angle
WARN			Warning generated during evaluation
	CAV(I,J)	J	Cavitation indicator at condition I at location J
		1	Nacelle exterior
		2	Strut exterior
		3	Nacelle inlet lip
		4	Strut elbow
		5	Pump inlet
		6	Pump impeller
NOZL			Nozzle characteristics
	JANGL		Nozzle depression angle
	*****		Unlabeled common variables
	IFUEL		Logical variable for fuel calculation
		.TRUE.	Make fuel calculation
		.FALSE.	Do not make fuel calculation
	IPUMP		Logical variable for pump calculation
		.TRUE.	Pump design parameters have been calculated/specified
		.FALSE.	Pump design parameters have not been calculated/specified
	TYGER		Logical variable for gear calculation
		.TRUE.	Gear type is specified

<u>Label</u>	<u>Variable Name</u>	<u>Subscript</u>	<u>Description/Usage</u>
		.FALSE.	Gear type is to be selected by program

VARIABLES USED IN H20JT
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
PARM(3)	Working system parameters
ENGN(3,IEGN)	Contains name of engine IENGN for output
EHP(I)	Effective horsepower at condition I, horsepower
DEL(3)	Initial step sizes of system parameters
DELMIN(3)	Minimum step sizes of system parameters
VK(I)	Craft speed at condition I, knot
IPRNT	Print data set reference number
DISPL	Craft displacement, long ton
IN	Estimate of minimum number of gas turbines required to power craft
MAX	Estimate of maximum number of gas turbines needed to power craft
XJ	Working variable for number of gas turbines
IK	Working variable for craft condition I
IL	Working variable for IN
KM	Minimum number of pumps to be used for XJ gas turbines
K1	Maximum number of pumps to be used for XJ gas turbines
WEIGT	Propulsion system weight, long ton
IM	Working variable for prime mover characteristic L
DISPL	Craft displacement, long ton
I	Index for do loops
IJ	Working variable for check on power adequacy
J	Index for do loop

<u>Variable</u>	<u>Description</u>
ISAVE	Working variable for MAX
N	Working variable for output of prime mover type
XMIN	Factor for minimum acceptable prime mover SHP operation
MK	Working variable for prime mover characteristic L
JNUM	Working variable for NUMB

LIST OF VARIABLES USED IN FCT
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
APOD(I)	Difference between present and previous nacelle drag calculations at conditions I, pound force
ASTRT(I)	Difference between present and previous strut drag calculations at condition I, pound force
ASPRY(I)	Difference between present and previous spray drag calculations at condition I, pound force
C(I)	Margin factor for thrust at condition I
TSUM	Working variable for propulsion system weight, pound
DSIG	Working variable for SIGMA
TOTAL	Total head less vapor head, foot
SIGMA	Head above vapor head, foot
SUMZ	Working variable for moment in vertical plane, due to weights, foot pound force
SUMX	Working variable for moment on longitudinal plane due to weights, foot pound force
SUM	Working variable for system weights, pound
ZSUM	Total moment in vertical plane due to weights, foot pound force
XSUM	Total moment on longitudinal plane due to weights, foot pound force
SPD	Working variable for jet velocity calculation, foot per second
J	Working variable for NUMB
JDRAG	Indicator of acceptable drag accuracy
1	New drag calculation is within 5% of previous calculation
2	New drag calculation is greater than 5% of previous calculation, redesign strut and nacelle

<u>Variable</u>	<u>Description</u>
ANGLE	Optimum nozzle depression angle, radian
COEF	Factor for nacelle drag calculation
0	Strut must be resized due to cavitation
1	Use nacelle design
KOUNT	Working variable for craft condition I
I	Craft condition
PARM(K)	Working variable for system parameters
K	
1	VJVO
2	VIVO
DDRAG(I)	Previous acceptable total drag calculation at condition I, pound force
DSPRY(I)	Previous acceptable spray drag calculation at condition I, pound force
DSTRT(I)	Previous acceptable strut drag calculation at condition I, pound force
DPOD(I)	Previous acceptable nacelle drag calculation at condition I, pound force

LIST OF VARIABLES USED IN NACEL
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
ZK	Percentage of auxiliary inlet area permitting flow
SPO	Static pressure at inlet, pound per square foot
PVP	Vapor pressure, pound per square foot
SIGTV	Incipient turning vane cavitation number, referenced to diffuser exit velocity
JNUMB	Working variable for NUMB
I	Craft condition, also index for do loops
CPEX	Peak external pressure coefficient
K	Working index for do loop for ELEFT/DM values
J	Working index for do loop for VIVO values
XD	Length of forebody to maximum diameter ratio, ELEFT/DM
DIDMX	Maximum permissible diameter ratio
QI	Flow rate per strut, cubic foot per second
DI	Nacelle inlet diameter, foot
CPIN	Peak internal pressure coefficient
QIN	Take-off flow rate per nacelle through inlet, cubic foot per second
QC	Take-off flow rate per nacelle, total, cubic foot per second
QAUX	Auxiliary flow rate required to avoid cavitation, cubic foot per second
KDEX	Counter for iterations on diffuser length
VR	Velocity ratio at take-off condition, based on flow through inlet
KNUMB	Working variable for NUMB

<u>Variable</u>	<u>Description</u>
VIJ	Inlet velocity at evaluation condition, based on flow through inlet
PRLJ	Pressure recovery coefficient of lip at evaluation condition
VI2	Inlet velocity at take-off condition, based on flow through inlet
PRL2	Pressure recovery coefficient of lip at take-off
PTI	Inlet stagnation pressure immediately aft of lip at take-off, pound per square foot
SPI	Inlet static pressure immediately aft of lip at take-off, pound per square foot
VIAUX	Inlet velocity in auxiliary inlet, foot per second
PRAUX	Pressure recovery of auxiliary inlet
PTAUX	Stagnation pressure inside auxiliary inlet, pound per square foot
DYP	Net dynamic pressure immediately aft of lip, pound per square foot
PC	Average inlet stagnation pressure of combined flow, pound per square foot
QDIF	Working variable for diffuser flow rate, cubic foot per second
PHI	Equivalent angle of forebody, radian
PHS	Sine of equivalent angle of forebody
X	Length of auxiliary inlet, foot
D2	Diffuser exit diameter, foot
D1	Inlet diameter, foot
ELMAX	Maximum permissible length of diffuser, foot
ELMIM	Minimum permissible length of diffuser, foot
II	Working variable for craft condition I

<u>Variable</u>	<u>Description</u>
EL	Working variable for diffuser length, foot
DEL	Working variable for change in diffuser length, foot
ELD	Working variable for nacelle length, foot
ELL	Working variable for nacelle length, foot
ELFAC	Working variable for excess in nacelle length due to diffuser, foot
DDM	Average diffuser diameter, foot
XKT	Form loss coefficient of diffuser
REL	Reynolds number, based on nacelle length
RED	Reynolds number, based on inlet diameter
DL	Ratio of maximum external diameter to nacelle length
CDRG	Computed drag coefficient
ANGL	Equivalent half angle of diffuser, degree
CDIF	Diffuser expansion factor
POW	Power loss due to drag and duct loss of diffuser, horsepower
POWI	Previous power loss calculation for diffuser length, horsepower
EM	Factor in wetted surface calculation
AEXN	Wetted surface area, square foot
REND	Reynolds number, based on inlet diameter
DDIF	Total pressure loss in diffuser, pound per square foot
PLOSS	Total pressure loss in nacelle, pound per square foot
VAOUT	Average exit velocity, foot per second
SQUAR	Factor in critical velocity calculation

<u>Variable</u>	<u>Description</u>
VCRIT	Critical velocity in strut elbow, foot per second
VMAX	Maximum velocity at nacelle exit, foot per second
RENL	Reynolds number, based on nacelle length
NL(2)	Interpolation parameters
ML(2)	Interpolation parameters
KL(2)	Interpolation parameters
JL(2)	Interpolation parameters
IL(2)	Interpolation parameters
VRT(6)	Data array of velocity ratios
XDT(10)	Tabulated forebody length to inlet diameter ratios
PRLT(6)	Tabulated lip pressure recovery coefficients
SIGI(I)	Free stream cavitation index at condition I
PTO(I)	Stagnation pressure at craft condition I, pound per square foot
XDTT(J)	Tabulated ELEFT/DM ratios for trim angle J
CDUMX(K)	Dummy array of peak pressure coefficients for VIVO K
CDUMY(L)	Dummy array of velocity ratios for ELEFT/DM L
CD(I)	Drag coefficients at craft condition I
DIDMT(J)	Tabulated DI/DM values for ELEFT/DM J
VRTT(J)	Dummy array of velocity ratios for angle of attack (J), internal
DLIP(I)	Lip loss coefficient at craft condition I
QO(I)	Free stream dynamic pressure at craft condition I, pound per square foot
VELR(I)	Inlet velocity ratio at craft condition I

<u>Variable</u>	<u>Description</u>
TLIP(I)	Working variable for DLIP(I)
VRTEX(I)	Same as VRTT(I) but external
VRMAX(I)	Maximum permissible velocity ratio at craft condition I
FRICT	Function statement, calculates Moody friction factor for smooth pipe, based on Reynolds number

VARIABLES USED IN ELBOW
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
SHAPE(3,ITYPE)	Contains name of shape ITYPE
THETA(J)	Data array of elbow angles
XLOSS(J)	Data array of elbow loss coefficients with thin, circular arc turning vanes
ROA(10)	Outside radius of splitters, foot
RE	Reynolds number of duct
FRICT	Function statement, calculates Moody friction factor for smooth pipe, based on Reynolds number
CORR	Function statement, calculates head loss correction factor for different Reynolds numbers
REMAX	Maximum Reynolds number permitted for splitter loss equation
KOUNT	Working variable for craft condition I
IELB	Index indicating which elbow is being designed/evaluated
1	Strut
2	Hull
3	Pump
4	Divergence
FACTR	Factor used in splitter loss calculation
RIN	Inside radius of bend, foot
ROUT	Outside radius of bend, foot
RATIO	Desired radius ratio, 4.3
XN	Number of subdivided elbows required to achieve RATIO=4.3
N	Number of subdivided elbows used

<u>Variable</u>	<u>Description</u>
N1	Number of splitters corresponding to N
SUM	Working variable of sum of head times subdivided elbow area, foot cubed
RIA	Inside radius of subdivided elbow, foot
V	Average velocity, foot per second
HGT	Height of subdivided elbow, foot
AA	Equivalent cross-sectional area of subdivided elbow based on HGT, square foot
RAD	Equivalent radius of subdivided elbow, foot
XCORR	Ratio of head loss correction factors for differing Reynolds numbers
XKT	Head loss coefficient of subdivided elbow
DIAM	Equivalent diameter, foot
VOLV	Volume of splitters, cubic foot
VOL	Volume of splitters and elbow structure, cubic foot
AREA1	Duct area per elbow, square foot
IJ	Working variable for shape determination
IK	Working variable for shape determination
I	Working variable for number of subdivided elbows N
RATEO	Ratio of inside radius to outside radius of subdivided elbows
HEAD	Average head loss in elbow, foot

LIST OF VARIABLES USED IN STRUT
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
THET2	Array of data of equivalent angle of diffuser, degree
EXPAN	Array of data of expansion coefficient
ARATO	Area ratio of strut diffuser
VL	Local maximum velocity, external, foot per second
SIGMA	Local cavitation number, external
SIGMI	Incipient cavitation number, external
CM	Mean chord, foot
DEIN	Strut inlet equivalent diameter, foot
WIDE	Width of duct at strut exit, foot
DEOUT	Strut exit equivalent diameter, foot
DEAVE	Average equivalent diameter, foot
STRT	Vertical strut length, foot
XLONG	Actual strut length, foot
STAN	Arctangent of equivalent diffuser angle
THETA	Equivalent angle of diffuser, 2θ , degree
ECOEF	Diffuser expansion factor
FORML	Diffuser expansion loss coefficient
VELIN	Average inlet velocity, foot per second
VLOUT	Average exit velocity, foot per second
RES	Strut Reynolds number, based on mean chord
CDS	Strut drag coefficient
CDSP	Spray drag coefficient
RE	Duct Reynolds number, based on inlet velocity

<u>Variable</u>	<u>Description</u>
PIPEL	Duct friction loss coefficient
TOTAL	Total loss coefficient
HEAD	Head loss of diffuser, foot
KOUNT	Working variable for craft condition I
HGT	Elevation of strut, foot
CGWS	Vertical center of gravity of duct, foot

LIST OF VARIABLES USED IN JUNCT
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
DIAM	Duct diameter, foot
FRCTL	Friction loss coefficient for athwartships length
KOUNT	Working variable for craft condition I
AJCT	Length of fore and aft ducting to pump inlet, foot
FRCTJ	Friction loss coefficient for junction
XLAMD	Working variable for AMIXL calculation
FLONG	Length of athwartships ducting, foot
AJCTL	Total loss coefficient of junction
AMIXL	Mixing loss coefficient of junction
V	Average velocity, foot per second
XPUMP	Number of gas turbines + 1
RE	Reynolds number of duct
BETA(J)	Data array of junction angles, degree
ALAMD(J)	Data array of mixing loss coefficient corresponding to BETA(J)
FRICT	Function statement, calculates Moody friction factor for smooth pipe, based on Reynolds number

LIST OF VARIABLES USED IN PIPE
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
KOUNT	Working variable for craft condition I
V	Average velocity, foot per second
DIAM	Duct diameter, foot
RE	Reynolds number of duct
APIPE	Length of fore and aft pipe to pump inlet, foot
FRCTL	Friction loss coefficient for athwartships length
XPUMP	Number of gas turbines
XKT	Friction loss coefficient of fore and aft length
XLONG	Length of athwartships pipe, foot
FRICT	Function statement, calculates Moody friction factor for smooth pipe, based on Reynolds number

LIST OF VARIABLES USED IN DIVRG
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
FRICT	Function statement, calculates Moody friction factor for smooth pipe, based on Reynolds number
KOUNT	Working variable for craft condition I
RE	Reynolds number of duct
DIVL	Divergence loss coefficient, not including friction
DWGT1	Duct weight of divergence angle, pound
DWGT2	Duct weight of divergence length, pound
DWGT3	Duct weight of pump inlet angle, pound
WWGT1	Water weight of divergence angle, pound
WWGT2	Water weight of divergence length, pound
WWGT3	Water weight of pump inlet angle, pound
CGWX1	Longitudinal center of gravity of divergence angle, from transom, foot
CGWX2	Longitudinal center of gravity of divergence length, from transom, foot
CGWX3	Longitudinal center of gravity of pump inlet angle, from transom, foot
DIAM	Duct diameter, foot
FRCTL	Friction loss coefficient for athwartships length
DIVLC	Total divergence loss coefficient
XPUMP	Number of gas turbines + 1
ANGLE	Divergence angle, also pump inlet angle, radian
HEADL	Total divergence head loss, foot
FLONG	Athwartships length, foot

<u>Variable</u>	<u>Description</u>
XLONG	Fore and aft length, foot
V	Average velocity, foot per second
ADIV	Divergence length, $ADIV = XLONG / \cos(\text{ANGLE})$, foot
THETA(J)	Data array of divergence angles, degree
COEF(J)	Data array of divergence loss coefficients without friction, corresponding to THETA(J)

LIST OF VARIABLES USED IN NOZZL
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
KOUNT	Working variable fro craft condition I
XLPS	Dummy nozzle length, foot
XFAC	Check on whether nozzle exits through bottom or stern
XPUMP	Number of pumps
ANOZ	Optimum nozzle depression angle, radian
XLNOZ	Nozzle length, foot
DT	Nozzle throat diameter, foot
DJ	Nozzle jet diameter, foot
XCORR	Nozzle head loss factor
RE	Reynolds number, based on average diameter and velocity
AREA1	Throat pipe area, square foot
AJET1	Jet pipe area, square foot
QQ	Flow rate per nozzle, cubic foot per second
FRICT	Function statement, calculates Moody friction factor for smooth pipe, based on Reynolds number

LIST OF VARIABLES USED IN PUMP
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
K	Cruise condition indicator
J	Take-off condition indicator
I	Index for do loops
XPUMP	Number of pumps
THOMI	Lower limit of Thoma's criterion for single inducer axial pump
DRAT	Hub to tip diameter ratio
XXLP	Factor for pump length
CW	Weight coefficient
QX	Ratio of cruise flow rate to take-off flow rate
CA	Factor for RPM calculation
CB	Factor for RPM calculation
HX	Ratio of design pump head to off design head
RX	Ratio of design to off design pump RPM
PHRAT	Ratio of off design to design flow coefficients
XNGT	Number of gas turbines
ETAPP	Product of pump and gearbox efficiencies
HP	Inducer head for one and two stage axial pump designs, foot
THOMS	Thoma's cavitation criterion for inlet to axial stage
HHP	Axial stage pump head, foot
BETA2	Exit blade angle, radian
JNUMB	Working variable for numb
ETAX	Ratio of off design to design pump efficiency

<u>Variable</u>	<u>Description</u>
XNNS	Non-dimensional specific speed
BD	Impeller exit width ratio
CC	Factor in flow coefficient calculation
AA	Factor in flow coefficient calculation
IMPL	Maximum number of impellers permitted for centrifugal pump, = 10
M	Working variable for number of centrifugal pump impellers
N	Working variable for NSTG
PC(K,J)	Inducer head curve coefficients
PCA(K,J)	Inducer plus axial stage head curve coefficients
PCC(K,J)	Centrifugal pump head curve coefficients
XRPM(M)	Working variable for off design RPM of centrifugal pump with M impellers, RPM
XD1(M)	Working variable for inlet tip diameter, D1S, foot
RPK(M)	Working variable for design RPM of centrifugal pump with M impellers, square foot
XPUP(M)	Working variable for centrifugal pump area with M impellers, foot squared
YLP(M)	Working variable for centrifugal pump length with M impellers, foot
XERAT(M)	Working variable for gear ratio required for centrifugal pump with M impellers
APUP(NSTG)	Inlet area of pump NSTG, square foot
WRAT(NSTG)	Weight ratio of pump NSTG, including pump dry and wet weight, gearbox and fuel
WD(M)	Working variable for XWD(NSTG) for centrifugal pump with M impellers
WW(M)	Working variable for XWW(NSTG) for centrifugal pump with M impellers

VariableDescription

WWG(M)	Working variable for XWG(NSTG) for centrifugal pump with M impellers
QQ(I)	Flow rate per pump at condition I, cubic feet per second

LIST OF VARIABLES USED IN FUEL
NOT IN COMMON

<u>Variables</u>	<u>Description</u>
CFS	Constant for SFC calculation
CA	Cruise condition drag to lift ratio
CD	1 + total system head loss coefficient, based on jet velocity
N	Number of intervals endurance is divided into + 1
XN	Number of intervals endurance is divided into
TI	Time to cover one range interval at constant VO(1), hour
IJK	Index for shifting SFC curves
XJ	Factor to convert SFC if SHP is less than 70% of design SHP
I	Working index for N
H	Head required, foot
SHPP	Total thrust required, horsepower
SHNG	Total thrust required per engine, horsepower
M	Working index for N
WT(ITIME)	Weight of fuel used in time increment ITIME, pound
DIS(ITIME)	Displacement at time increment ITIME, pound
VJJ(ITIME)	Jet velocity at time increment ITIME, foot per second
ENN	Working variable for XJ

LIST OF VARIABLES USED IN PTRN
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
PSI(N)	Current basepoint coordinate of parameter N
THETA(N)	Previous basepoint coordinate of parameter N
PHI(N)	Present exploratory point coordinate of parameter N
DEL(N)	Current step size of parameter N
DELMIN(N)	Minimum step size of parameter N
DIR(N)	Last successful direction of parameter N
SAVE(N)	Working variable for PHI(N)
S	Working variable for function value
SPHI	Working variable for function value at PHI coordinates
SPSI	Current best function value at PSI coordinates
NUMB	Counter for minimum step size check
RHO	Step size change factor
ICALL	Indicator of current point move
K	Index for do loop
I	Index for do loop
N	Number of parameters of search
SIGN(N)	Directed step size of parameter N

LIST OF VARIABLES USED IN OUTPUT
NOT IN COMMON

<u>Variable</u>	<u>Description</u>
IPRNT	Print data set reference number
J	Working variable for condition I
SUM	Total duct head loss, excluding elevation, foot
I	Index for do loop
DRATO	Strut diffuser area ratio
IK	Index for number of elbows in system
XNGT	Number of gas turbines
K	Index for do loop
L	Index for implied do loop in output statement
LL	Working variable to point to correct head loss for output
KLL	Working variable to point to correct format statement for head loss output
M	Index for implied do loop in output statement
NIMP	Number of impellers in pump
TFM	Strut thickness at flying waterline, foot
WTRAT	Total propulsion system weight ratio
WRATF	Propulsion system weight ratio, excluding fuel
ENGN(IENGN)	Contains name of engine IENGN
VJRAT(I)	Jet velocity ratio at craft condition I
VIRAT(I)	Inlet velocity ratio at craft condition I
HEADL(I,ICOMP)	Head loss of component ICOMP at craft condition I
CONDS(2,I)	Label for craft condition I
ELBWS(2,IK)	Label for elbow IK

<u>Variable</u>	<u>Description</u>
PC(I)	Propulsive coefficient at craft condition I
LABEL(5,M)	Labels for output
VK(I)	Craft speed at condition I, knots
LOCAT(6,J)	Labels for cavitation locations

APPENDIX B
PROGRAM LISTING

SAMPLE INPUT	82
SUBROUTINE H2OJT	84
SUBROUTINE FCT	92
SUBROUTINE ELBOW	97
FUNCTION CFS	102
FUNCTION TABLE	103
SUBROUTINE STRUT	106
SUBROUTINE JUNCT	111
SUBROUTINE PIPE	114
SUBROUTINE DIVRG	116
SUBROUTINE NOZZL	120
SUBROUTINE PUMP	122
SUBROUTINE GEAR	133
SUBROUTINE FUEL	136
SUBROUTINE NACEL	138
SUBROUTINE PPTRN	149
SUBROUTINE OUTPUT	153
BLOCK DATA	160

Note: Statement numbers in right hand margin indicate statements retained from design program listed in reference 1.


```

LOGICAL IFUEL, IPUMP, TYGER
REAL JANGL
COMMON /PINLP/ALPHA
COMMON /PARMS/VJVO, VIVO, DIDM
COMMON /WARN/CAV(5,6)
COMMON /NOZL/JANGL
COMMON /CHARS/WGTS(2,15), CGS(4,15), DELH(5,15), CGSX, CGSZ
COMMON /DRAG/DRAG(5), STRTD(5), POD(5), SPRAY(5), REST(5), VO(5),
1 TRIM(5)
COMMON /NAELL/ERAT, DM, AI, AIAUX, ELEFT, ELAUX, ELDIF, ELN, AAX(5)
COMMON /H2O/TEMP, PV, RHCW, GNU, HA
COMMON /SHIP/DISP, RANGE, BEAM, HS, HE, HCL, XLS, XLPE, XLP
COMMON /CONST/PI, G, PHCD
COMMON /FLOW/Q(5), AIN, AJET, AREA(11), VJ(5), VI(5)
COMMON /ELBW/XK(4), PQ(4), THATA(4), WIDTH, DEPTH, TYPE(3,4)
COMMON /INDEX/IEVAL, IEQPT, ISRT, NUMB, IENGN, ITYPE, ICOMP, NPUMP, NGT,
AIGEAR
COMMON /ITABL/L(2)
COMMON IFUEL, IPUMP, TYGEP

SAMPLE INPUT
VC(1)=45.*1.6889
VC(2)=3J.*1.6889
BEAM=36.
RANGE=2000.
HE=35.5
HS=12.
XLPE=12.
IENGN=9
XLS=32.
HCL=17.5
TRIM(1)=0.
TRIM(2)=0.
TYGER=.FALSE.
DISP=750.*2240.

```

```

0002
0003
0004
0005
0006
0007
0008
0009
0011
0013
0014
0015

```



```
POD(1)=11436.1
STRTD(1)=11653.7
SPRAY(1)=8474.2
POD(2)=POD(1)*VO(2)/VO(1)**2
STRTD(2)=STRTD(1)*VO(2)/VO(1)**2
SPRAY(2)=SPRAY(1)*VO(2)/VO(1)**2
REST(1)=76956.
REST(2)=129971.6
CALL H2OJT
STOP
END
```

```
0034
0035
```


H20J0001

SUBROUTINE H20JT
 LOGICAL IFUEL, IPUMP, TYGER
 REAL JANGL
 COMMON /PINLP/ALPHA
 COMMON /NOZL/JANGL
 COMMON /WARN/CAV(5,6)
 COMMON /PARMS/ VJVO,VIVO,DIDM
 COMMON /DRAG/DRAG(5), STRTD(5), POD(5), SPRAY(5), REST(5), VO(5),
 1 TRIM(5)
 COMMON /ELRW/XK(4),RO(4),THATA(4),WIDTH,DEPTH,TYPE(3,4)
 COMMON /CHARS/WGTS(2,15),CGS(4,15),DELH(5,15),CGSX,CGSZ
 COMMON /SHIP/ SHIP,RANGE,BFAM,HS,HE,HCL,XLS,XLPE,XLP
 COMMON /H2O/TEMP,PV,RHOW,GNU,HA
 COMMON /TOLER/DELTA
 COMMON /PSUB/GERAT(5),SHP(5,5),RPM(5,5),PERF(5,12),ETAP(8,5)
 COMMON /CONST/ PI,6,RHOD
 COMMON /STRTC/TC,T,C,T1,C1,CFM
 COMMON /INDEX/IEVAL,IEOPT,ISTRT,NUMB,IENGN,ITYPE,ICOMP,NPUMP,NGT,
 AIGEAR

H20J0015

COMMON /ITABL/L(2)
 COMMON IFUEL,IPUMP,TYGER
 EQUIVALENCE (VJVO,PARM(1))
 DIMENSION PARM(3),ENGN(3,12),EHP(5),DEL(4),DLMIN(4),PMIN(5)
 DIMENSION VK(5)
 EXTERNAL FCT
 DATA IPRNT/6/
 DATA ENGN/4HTF35,2*4H ,4HTF40,2*4H ,4HPRCT, 4HEUS ,4H1500,
 A 4HPRCT,4HEUS ,4H1000, 4HTYNE,4H 1A ,4H ,4HTYNE,4H 1C ,4H
 B 4HFT12,4HA ,4H ,4HLM15,4H00 ,4H ,4HLM25,4H00 ,4H ,
 C 4HFT4A,4H-2C ,4H ,4HFT4A,4H-12 ,4H ,4HFT4C,4H-2 ,4H /
 DATA DLMIN/4*.01/
 IFUEL=.TRUE.
 IPUMP=.FALSE.
 DISP=DISP/2240.
 DETERMINE ENGINE TYPE
 ENGN CONTAINS CODING OF GAS TURBINE MODEL

C
C

H20J0003

H20J0004

H20J0005

H20J0006

H20J0007

H20J0008

H20J0009

H20J0010

H20J0011

H20J0012

H20J0013

H20J0017

H20J0018

H20J0019

H20J0020

H20J0021

H20J0022

H20J0023

H20J0024

H20J0025

H20J0026

H20J0027

H20J0028

H20J0029

H20J0030

H20J0031

H20J0032
H20J0033
H20J0034
H20J0035
H20J0036
H20J0037
H20J0038
H20J0039
H20J0040
H20J0041
H20J0042
H20J0043
H20J0044
H20J0045
H20J0046
H20J0047
H20J0048
H20J0049
H20J0050
H20J0051

IEGN GAS TURBINE
1 TF35
2 TF40
3 PROTEUS 1500 RPM
4 PROTEUS 1000 RPM
5 TYNE 1A
6 TYNE 1C
7 FT12A
8 LM1500
9 LM2500
10 FT4A-2C
11 FT4A-12
12 FT4C-2

DETERMINE WHICH POINTS ARE INPUT

ISTRT=1

NUMB=2

IF(IEVAL.EQ.0) GO TO 3

IF(IEVAL.LT.0) ISTRT=3

IF(IEVAL.LT.0) NUMB=2+IABS(IEVAL)

3 JNUM=2+IABS(IEVAL)

DC 1 I=ISTRT, JNUM

VK(I)=V0(I)*0.5921

1 TDRAG(I)=POD(I)+STRTD(I)+SPRAY(I)+REST(I)

H20J0054
H20J0055
H20J0056
H20J0057
H20J0058
H20J0059
H20J0060
H20J0061
H20J0062
H20J0063

IEVAL.LT.0 IMPLIES NO CRUISE OR TAKE-OFF POINTS SPECIFIED,
ABS(IEVAL) INDICATES HOW MANY POINTS FOR PERFORMANCE ESTIMATION
IEVAL.EQ.0 IMPLIES DESIGN AT CRUISE AND TAKE-OFF ONLY
IEVAL.GT.0 IMPLIES DESIGN AT CRUISE/TAKE-OFF AND ESTIMATE PERFORM.
AT IEVAL POINTS

IF ENTERED, CRUISE POINTS ARE IN FIRST POSITION IN ARKAY, TAKE-OFF
POINTS IN SECOND AND PERFORMANCE ESTIMATION POINTS IN REMAINING
POSITIONS

C IEQPT NON-ZERO IMPLIES SPECIFIC TYPE EQUIPMENT TO BE INPUT BY USERH20J00064
 C IEQPT.EQ.1 IMPLIES NUMBER OF GAS TURBINES IS SPECIFIED BY USER
 C IEQPT.EQ.3 IMPLIES NUMBER OF GAS TURBINES AND NUMBER OF PUMPS
 C ARE SPECIFIED BY USER

C TYGER.EQ..TRUE. IMPLIES GEAR TYPE, IGEAR, IS SPECIFIED BY USER
 C TYGER.EQ..FALSE. IMPLIES GEAR TYPE TO BE SELECTED BY PROGRAM
 C IGEAR TYPE OF GEAR
 C 1 SINGLE REDUCTION WITH IDLER
 C 2 PLANETARY
 C 3 DOUBLE REDUCTION, DOUBLE BRANCH

IF(IIRST.NE.1) GO TO 4
 WRITE(IPRNT,5)
 5 FORMAT(1H1,10X,58H *** WATERJET PROPULSION SYSTEM DESIGN AND PERFO
 RMANCE *** ,//)
 WRITE(IPRNT,31)
 WRITE(IPRNT,6)
 6 FORMAT(12H OPERATIONAL,/) H20J00066
 WRITE(IPRNT,7) H20J00067
 7 FORMAT(30X,6HCRUISE,15X,8HTAKE-OFF,/) H20J00068
 WRITE(IPRNT,8) (VK(N),N=1,2) H20J00069
 8 FORMAT(16H VELOCITY, KNOTS,14X,F5.1,17X,F5.1) H20J00070
 WRITE(IPRNT,9) (TDRAG(N),N=1,2) H20J00071
 9 FORMAT(16H TOTAL DRAG, LBS,13X,F7.0,15X,F7.0) H20J00072
 WRITE(IPRNT,11) (TRIM(N),N=1,2) H20J00073
 10 FORMAT(25H ANGLE OF ATTACK, DEGREES,6X,F4.1,18X,F4.1,/) H20J00074
 IF(IEVAL.EQ.0) GO TO 11 H20J00075
 WRITE(IPRNT,12) H20J00076
 12 FORMAT(/,48H PERFORMANCE EVALUATION AT THE FOLLOWING POINTS,/) H20J00077
 GO TO 13 H20J00078
 4 WRITE(IPRNT,14) H20J00079
 14 FORMAT(1H1,10X,47H *** WATERJET PROPULSION SYSTEM PERFORMANCE *** ,H20J00086
 A ///) H20J00087
 13 IM=IABS(IEVAL) H20J00088
 WRITE(IPRNT,6) H20J00089


```

DO 15 I=1, IM
J=I+2
15 WRITE(IPRINT,16) VK(J),TDRAG(J),TRIM(J)
16 FOPMAT(16H VELOCITY, KNOTS,14X,F5.1,/,16H TOTAL DRAG,LBS,13X,F7.0,H20J0093
A /,25H ANGLE OF ATTACK, DEGREES,6X,F4.1,/)
WRITE(IPRINT,31)
31 FOPMAT(22H CRAFT CHARACTERISTICS,/)
11 WRITE(IPRINT,17)
17 FOPMAT(14H CONFIGURATION,/)
WRITE(IPRINT,18) BEAM,DISPL,RANGE,(ENGN(N,IENGN),N=1,3)
18 FOPMAT(28H AVERAGE BEAM, FEET....., F7.1,/,28H DISPLACEMENT, LH20J0100
AONG TONS....., F7.0,/,28H ENDURANCE, NM.....,F7.0,/,28H GAH20J0101
BS TURBINE PLANT.....,3X,3A4,/)
WRITE(IPRINT,19) HS,HE,HCL,XLS,XLPE
19 FOPMAT(48H DEPTH OF SUBMERGENCE OF NACELLE.....,F5.1,2XH20J0104
A,4HFEET,/,48H HEIGHT OF PUMP CENTERLINE ABOVE MEAN WATER.....,F5.1H20J0105
B,2X,4HFEET,/,48H HEIGHT OF PUMP CENTERLINE ABOVE KEEL....., H20J0106
CF5.1,2X,4HFEET,/,48H DISTANCE OF STRUT FROM TRANSOM.....H20J0107
D.,F5.1,2X,4HFEET,/,48H DISTANCE OF PUMP EXIT FROM TRANSOM.....H20J0108
E.....,F5.1,2X,4HFEET,/)
WRITE(IPRINT,20)
20 FOPMAT(10X,17H WATER PROPERTIES,/,10X,45H (ASSUMES STANDARD(3.5% SH20J0111
SALINITY) SALT WATER),/)
C
C
C
TEMP IS THE TEMPERATURE OF THE WATER IN DEGREES FAHRENHEIT
25 PV=.11413E-7*TEMP**4+.12435E-6*TEMP**3+.49859E-4*TEMP*TEMP+.29719EH20J0116
A-2*TEMP+.032335
GNU=EXP(-.6201E-6*TEMP**3+.1749E-3*TEMP*TEMP-.02796*TEMP+1.414)*
A 1.E-5
RHOW=(-.79965E-6*TEMP**4+.18714E-3*TEMP**3-.015982*TEMP*TEMP+.5817H20J0120
A*TEMP+56.509)/G
HA=2117./(RHOW*G)
WRITE(IPRINT,21) TEMP,RHOW,GNU,PV
21 FOPMAT(32H TEMPERATURE, DEGREES FAHRENHEIT,5X,F4.0,/,
A 28H DENSITY, LRF-SEC**2/FEET**4,9X,F6.0,/,

```



```

B 31H VISCOSITY, *10**5, FEET**2/SEC,6X,5PF6.3,/,
C 21H VAPOR PRESSURE, FEET,16X,OPF6.3,/(/
WRITE(IPRNT,22) G
22 FORMAT(36H ACCELERATION OF GRAVITY, FT/SECS**2,6X,F7.3,/(/
IF(IEOPT.NE.1) GO TO 23
IF(IYGER) GO TO 23
WRITE(IPRNT,24)
24 FORMAT(68H NO EQUIPMENTS OR CONFIGURATIONS SPECIFIED. GENERATED AS
A PER PROGRAM,/,I11)
GO TO 32
33 WRITE(IPRNT,34) (ENGN(I,IENGN),I=1,3)
34 FORMAT(16H MORE THAN FOUR ,3A4,3)H REQUIRED. INVALID PRIME MOVER)
RETURN
23 WRITE(IPRNT,26)
26 FORMAT(I11,' THE FOLLOWING EQUIPMENT, ETC. SPECIFIED.....')
IF(IYGER) GO TO(101,102,103),IGEAR
GO TO 104
101 WRITE(IPRNT,112)
112 FORMAT(/,51H SPECIFIED GEAR TYPE IS SINGLE REDUCTION WITH IDLER,/
A/)
GO TO 104
102 WRITE(IPRNT,113)
113 FORMAT(/,33H SPECIFIED GEAR TYPE IS PLANETARY,/(/
GO TO 104
103 WRITE(IPRNT,114)
114 FORMAT(/,55H SPECIFIED GEAR TYPE IS DOUBLE REDUCTION, DOUBLE BRAN
ACH,/(/
104 IF(IEOPT.EQ.1) GO TO 32
IF(IEOPT.NE.1) GO TO 131
WRITE(IPRNT,121) NGT
121 FORMAT(/,37H SPECIFIED NUMBER OF GAS TURBINES IS ,I2,/(/
GO TO 141
131 WRITE(IPRNT,122) NGT,NPUMP
122 FORMAT(/,37H SPECIFIED NUMBER OF GAS TURBINES IS ,I2,/,30H SPECIF
AIED NUMBER OF PUMPS IS ,I2,/(/
GO TO 141

```


COMMENCE DESIGN/PERFORMANCE PREDICTION

H20J0144
H20J0145
H20J0146
H20J0147
H20J0148
H20J0149
H20J0150
H20J0151
H20J0152
H20J0153
H20J0154
H20J0155
H20J0156
H20J0157
H20J0158
H20J0159
H20J0160
H20J0161
H20J0162
H20J0163
H20J0164
H20J0165
H20J0166
H20J0167
H20J0168
H20J0169

H20J0171
H20J0172
H20J0173

FIRST CHECK ON MINIMUM NUMBER OF GAS TURBINES REQUIRED, GIVEN THE TYPE AND EHP. THEN CYCLE THROUGH THE PUMP COMBINATIONS POSSIBLE.

```

32 IF(ISTRT.NE.1) GO TO 38
   IN=1
   MAX=3
   ISAVE=4
   XMIN=.25
   DO 37 IK=ISTRT,NUMB
     IL=1
     EHP(IK)=TDRAG(IK)*VO(IK)/550.
     DO 37 IJ=1,3
       XJ=IJ+IJ/3
       IM=IK/3
       MK=IK-IM*(IK-2)
       IF(1.5*EHP(IK).GT.XJ*PERF(MK,IENGN)) IL=IJ+1
       IF(IK.GT.1) GO TO 39
       IF(EHP(IK)/XJ.LT.XMIN*PERF(1,IENGN)) ISAVE=XJ*.5
       IF(ISAVE.LT.MAX) MAX=ISAVE
       IF(MAX.EQ.0) MAX=1
39 IF(IL.GT.3) GO TO 33
37 IF(IN.LT.IL) IN=IL
   TWGTS=1.F30
   GO TO 144
141 IN=1
   IF(ISTRT.NE.1) GO TO 38
   MAX=1
144 DO 30 IK=IN,MAX
   IF(IEOPT.NE.) GO TO 145
C   PUT MIN NUMBER GT'S REQUIRED IN HERE
C   NGT=IK+IK/3
145 KM=NGT/4+1

```



```

H20J0175
KI=2*IK-NGT+1
IF(IEQPT.LE.1) GO TO 143
KI=KM
143 DO 28 I=KM,KI
IF(IEQPT.GT.1) GO TO 142
NPUMP=I+I/3
142 DELTA=.05
DEL(1)=.5
DEL(2)=.2
DEL(3)=.3
VJVO=1.8
VIVC=.7
DIDM=.6
CALL PTRN(PARM,WEIGT,3,FCT,DEL,DLMIN)
DELTA=0.0
WEIGT=FCT(PARM)/2240.
IF(WEIGT.GT.TWGTS) GO TO 52
DO 53 JK=1,3
53 PMIN(JK)=PARM(JK)
PMIN(4)=NGT
PMIN(5)=NPUMP
52 WRITE(IPRNT,50) WEIGT
50 FORMAT(10X,21H SYSTEM WEIGHT IS ,F10.2,9H TONS
WRITE(IPRNT,51) NGT,NPUMP
51 FORMAT(/,27H NUMBER OF GAS TURBINES IS ,I2,/,20H NUMBER OF PUMPS
AIS ,I2,/)
CALL OUTPUT
28 CCNTINUE
30 CONTINUE
IF(IEVAL.EQ.0) RETURN
IF(IEVAL.NE.0) ISTRT=3
NUMB=2+IARS(IEVAL)
38 DELTA=0.0
I PUMP=.TRUE.
WEIGT=FCT(PARM)/2240.
WRITE(IPRNT,50) WEIGT
H20J0181
H20J0178
H20J0179
H20J0180
H20J0182
H20J0183
H20J0184
H20J0185
H20J0186
H20J0187
H20J0188
H20J0189
H20J0190
H20J0191
H20J0192
H20J0193
H20J0194
H20J0195
H20J0196
H20J0197
H20J0198
H20J0199
H20J0200
H20J0201
H20J0207
H20J0208

```


WRITE(IPRNT,51) NGT, NPUMP
CALL OUTPUT
RETURN
END

H20J0209
H20J0210
H20J0213
H20J0214


```

FUNCTION FCT(PARM)
LOGICAL IFUEL, IPUMP, TYGER
REAL JANGL
COMMON /NOZL/JANGL
COMMON /WARN/CAV(5,6)
COMMON /PINLP/ALPHA
COMMON /CHARS/WGTS(2,15), CGS(4,15), DELH(5,15), CGSX, CGSZ
COMMON /PSUR/GERAT(5), SHP(5,5), RPM(5,5), PERF(5,12), ETAP(8,5)
COMMON /H2O/TEMP, PV, PHOW, GNU, HA
COMMON /FLBW/XK(4), PO(4), THATA(4), WIDTH, DEPTH, TYPE(3,4)
COMMON /SHIP/ DISP, RANGE, BEAM, HS, HE, HCL, XLS, XLPE, XLP
COMMON /NACLL/CRAT, DM, AI, AIAUX, ELEFT, FLENT, FLAUX, ELDIF, ELN, AAX(5)
COMMON /FLOW/Q(5), AIN, AJET, AREA(11), VJ(5), VI(5)
COMMON /CONST/PI, G, RHOD
COMMON /PUMN/QQ(5), DIS(5), D2S(5), XNS(5), SM(5,5), PLP(5), NSTG, SHI(5,
A5), XIM
COMMON /DRAG/IDRAG(5), STRTD(5), POD(5), SPRAY(5), REST(5), VO(5),
1 TRIM(5)
COMMON /CDRAG/CDSTR(5), CPOD(5), CSPRY(5)
COMMON /TOLER/ DELTA
COMMON /INDEX/IEVAL, IEQPT, ISTRT, NUMB, IENGN, ITYPE, ICOMP, NPUMP, NGT,
AIGEAR
COMMON /ITABL/L(2)
COMMON IFUEL, IPUMP, TYGER
DIMENSION PARM(4), APOD(5), ASTRT(5), ASPRY(5), C(5), DDRAG(5), DSPRY(5)
A ,DSTRT(5), DPOD(5)
DATA C/L.,1.,25.,3*1./
IF(ISTRT.NE.1) GO TO 101
IF(.NOT.IPUMP) XLP=5.
TSUM=1.E9

POP BACK TO OPTIMIZATION IF BCUNDS EXCEEDED

IF(PARM(1).GT.5..AND.DELTA.GT.1.E-9) GC TC 8
IF(PARM(1).LT.1.1) GO TO 8
IF(PARM(2).LT..5) GC TC 8

```

FCT 0001

FCT 0003

FCT 0004

FCT 0005

FCT 0006

FCT 0007

FCT 0009

FCT 0010

FCT 0013

FCT 0014

FCT 0015

FCT 0016

FCT 0018

FCT 0020

FCT 0021

FCT 0022

FCT 0023

FCT 0024

FCT 0025

FCT 0026

FCT 0027

FCT 0028

FCT 0029

FCT 0030

C

C

C


```

IF(PARM(2).GT.1.5) GO TO 8
IF(PARM(3).GT.9) GO TO 8
IF(PARM(3).LE.J..AND.DELTA.GT.1.E-9) GC TO 8
DRAT=PARM(3)
101 DO 15 J=I,NUMB
DSTRT(J)=STRTD(J)
DPOD(J)=POD(J)
DSPRY(J)=SPRAY(J)
DDRAG(J)=TDRAG(J)
IF(ISTRT.NE.1) GO TO 102
VJ(1)=VJ(1)+PARM(1)
VI(1)=VJ(1)+PARM(2)
10 ANGLE=ATAN(TDRAG(1)/DISP)
JANGL=ANGLE
IF(TDRAG(1)/DISP.GT..2.AND.DELTA.GT.1.E-9) GO TO 8
IF(TDRAG(1)/DISP.LT..05.AND.DELTA.GT.1.E-9) GC TO 8
CDEF=1.
Q(1)=TDRAG(1)/(RHO*(VJ(1)*COS(ANGLE)-VO(1)))
AIN=Q(1)/VI(1)
AJET=Q(1)/VJ(1)
I=2
102 IF(ISTRT.EQ.3) I=3
IF(ISTRT.FO.3) ANGLE=JANGL
DO 5 J=I,NUMB
SPD=.5*VO(J)/CCS(ANGLE)
VJ(J)=SPD+SQRT(SPD*SPD+C(J)*TDRAG(J)/(RHO*AJET*COS(ANGLE)))
Q(J)=AJET*VJ(J)
5 VI(J)=Q(J)/AIN
ICOMP=1
CALL NACFL
IF(WGTS(1,1).GT.DISP.AND.DELTA.GT.1.E-9) GC TO 8
IF(DELTA.LT.1.F-5) PARM(3)=DRAT
ICOMP=2
DEPTH=SQRT(AREA(1))
WIDTH=.5*DEPTH
CALL ELBOW
FCT 0031
FCT 0032
FCT 0033
FCT 0034
FCT 0036
FCT 0037
FCT 0038
FCT 0039
FCT 0040
FCT 0041
FCT 0042
FCT 0043
FCT 0044
FCT 0045
FCT 0046
FCT 0047
FCT 0048
FCT 0049
FCT 0051
FCT 0052
FCT 0053
FCT 0054
FCT 0055
FCT 0056
FCT 0057
FCT 0058
FCT 0059
FCT 0060
FCT 0061
FCT 0062
FCT 0063

```



```

CGS(1,ICOMP)=HCL-HE-HS
CGS(2,ICOMP)=XLS
IF(THATA(2).NE.90.) CGS(2,ICOMP)=CGS(2,ICOMP)+(HE-RO(1)*XK(1)+HS)/
A TAN(THATA(2)*.0174533)
WGTS(1,ICOMP)=WGTS(1,ICOMP)*((RHOD-RHCW)/RHOD
WGTS(2,ICOMP)=0.
12 ICOMP=3
CALL STRUT
IF(WGTS(1,ICOMP).GT.DISP) GO TO 8
J=NUMR
IF(ISTR.EQ.1) J=2
C
C SEE IF NEWLY COMPUTED DRAG DIFFERS SIGNIFICANTLY (I.E. GREATER
C THAN 5%) FROM PREVIOUS ESTIMATE. IF SO, RECCMPUTE ON BASIS OF NEW
C DRAG. OTHERWISE CONTINUE.
C
JDRAG=1
DEV=.05
IF(DELTA.LT.1.E-9) DEV=.01
DO 9 I=ISTR,J
APOD(I)=COFF*(CPOD(I)-POD(I))
ASTRT(I)=CSTRT(I)-STRTD(I)
ASPRY(I)=CSPRY(I)-SPRAY(I)
IF(ABS(APOD(I)+ASTRT(I)+ASPRY(I)).GE.DEV*TDTRAG(I)) JDRAG=2
SPRAY(I)=CSPRY(I)
STRTD(I)=CSTRT(I)
POD(I)=CPOD(I)
9 TDTRAG(I)=REST(I)+PCD(I)+STRTD(I)+SPRAY(I)
IF(JDRAG.EQ.2.AND.ISTR.EQ.3) GO TO 102
IF(JDRAG.EQ.2) GO TO 10
ICOMP=4
CALL ELBOW
CGS(1,ICOMP)=HCL
CGS(2,ICOMP)=XLS
CGS(3,ICOMP)=CGS(1,ICOMP)
CGS(4,ICOMP)=XLS

```

```

FCT 0064
FCT 0065
FCT 0066
FCT 0067
FCT 0068
FCT 0069
FCT 0070
FCT 0071
FCT 0072
FCT 0073
FCT 0074
FCT 0075
FCT 0076
FCT 0077
FCT 0078
FCT 0079
FCT 0080
FCT 0081
FCT 0086
FCT 0082
FCT 0083
FCT 0084
FCT 0085
FCT 0087
FCT 0088
FCT 0089
FCT 0090
FCT 0091
FCT 0092
FCT 0093
FCT 0094
FCT 0095
FCT 0096
FCT 0097
FCT 0098

```



```

ICOMP=5
GO TO (1,2,3,3),NPUMP
1 CALL JUNCT
GO TO 4
2 CALL PIPE
GO TO 4
3 CALL DIVRG
4 ICOMP=9
DSIG=4.
DC 13 KOUNT=ISTR, NUMB
TOTAL=VO(KOUNT)*2/G*.5-DELH(KOUNT,6)-PV+HA
SIGMA=TOTAL-(Q(KOUNT)/AREA(6))**2*.5/G
IF(SIGMA.GT.0.) GO TO 13
CAV(KOUNT,5)=1.
IF(ISTR.EQ.3) GO TO 13
IF(SIGMA.GT.DSIG) GO TO 13
COEFF=0.
DSIG=SIGMA
AREA(4)=1.05*Q(KOUNT)/SQRT(2.*G*TOTAL)
13 CONTINUE
IF(DSIG.NE.4.) GO TO 12
14 CONTINUE
CALL NOZZL
IF(WGTS(1,ICOMP).GT.DISP.AND.DELTA.GT.1.E-9) GO TO 8
ICOMP=8
IF(ISTR.NE.3) NSTG=0.
CALL PUMP
IF(ISTR.EQ.3) GO TO 17
IF(IPUMP) NSTG=NSTG
IF(IPUMP) GO TO 12
IF(NNSTG.EQ.NSTG.OR .DELTA.GT.1.E-9) GO TO 17
IPUMP=.TRUE.
NNSTG=NSTG
GO TO 12
17 DO 11 J=ISTR, NUMB
11 DELH(J,9)=DELH(J,7)+DELH(J,8)

```

```

FCT 0099
FCT 0100
FCT 0101
FCT 0102
FCT 0103
FCT 0104
FCT 0105
FCT 0106
FCT 0107
FCT 0108
FCT 0109
FCT 0110
FCT 0111

```

```

FCT 0112
FCT 0113
FCT 0114
FCT 0115
FCT 0116
FCT 0117
FCT 0118
FCT 0119
FCT 0120
FCT 0121

```

```

FCT 0122

```

```

FCT 0123
FCT 0124
FCT 0125
FCT 0126
FCT 0127
FCT 0128
FCT 0129
FCT 0130

```


FCT 0131
 FCT 0132
 FCT 0133
 FCT 0134
 FCT 0135
 FCT 0136
 FCT 0137
 FCT 0138
 FCT 0139
 FCT 0140
 FCT 0141
 FCT 0142
 FCT 0143
 FCT 0144
 FCT 0145
 FCT 0146
 FCT 0147
 FCT 0148
 FCT 0149
 FCT 0150
 FCT 0151
 FCT 0152
 FCT 0153
 FCT 0154
 FCT 0155
 FCT 0156
 FCT 0157
 FCT 0158
 FCT 0159
 FCT 0160
 FCT 0161
 FCT 0162

```

ZSUM=0.
XSUM=0.
TSUM=0.
WGTS(1,12)=PERF(5,IENGN)*FLOAT(NGT)
DO 6 J=1,2
SUM=0.
SUMX=0.
SUMZ=0.
DO 7 I=1,12
SUMZ=SUMZ+WGTS(J,I)*CGS(2*J-1,I)
SUMX=SUMX+WGTS(J,I)*CGS(2*J,I)
7 SUM=SUM+WGTS(J,I)
WGTS(J,14)=SUM
CGS(2*J-1,13)=SUMZ/SUM
CGS(2*J,13)=SUMX/SUM
TSUM=TSUM+SUM
XSUM=XSUM+SUMX
ZSUM=ZSUM+SUMZ
CGSX=XSUM/TSUM
CGSZ=ZSUM/TSUM
WGTS(2,13)=-RHOW*Q(1)*VJ(1)*SIN(ANGLE)
WGTS(2,14)=WGTS(2,14)+WGTS(2,13)
TSUM=TSUM+WGTS(2,13)
8 FCT=TSUM
IF(FCT.LT.DISP.OR.DELTA.LT.1.E-5) RETURN
DO 16 I=1,STRT,NUMB
SPRAY(I)=DSPRY(I)
POD(I)=DPOD(I)
STRD(I)=DSTRT(I)
16 TDRAG(I)=DDRAG(I)
RETURN
END
  
```


ELBOW000
 ELBOW001
 ELBOW002
 ELBOW003
 ELBOW004
 ELBOW005
 ELBOW006

ELBOW008
 ELBOW009
 ELBOW010
 ELBOW011
 ELBOW012
 ELBOW013
 ELBOW014
 ELBOW015
 ELBOW016
 ELBOW017
 ELBOW018
 ELBOW019
 ELBOW020
 ELBOW021
 ELBOW022
 ELBOW023
 ELBOW024
 ELBOW025
 ELBOW026
 ELBOW027
 ELBOW028
 ELBOW029
 ELBOW030
 ELBOW031
 ELBOW032
 ELBOW033
 ELBOW034

SUBROUTINE ELBCW
 COMMON /H2O/TEMP,PV,RHOW,GNU,HA
 COMMON /SHIP/ DISP,RANGE,BEAM,HS,HE,HCL,XLS,XLPE,XLP
 COMMON /CHARS/WGTS(2,15),CGS(4,15),DELH(5,15),CGSX,CGSZ
 COMMON /ELBW/XK(4),RO(4),THATA(4),WIDTH,DEPTH,TYPE(3,4)
 COMMON /FLOW/Q(5),AIN,AJET,APEA(11),VJ(5),VI(5)
 COMMON /CONST/PI,G,RHOD
 COMMON /INDEX/IEVAL,IEQPT,ISTRT,NUMB,IFENGN,ITYPE,ICOMP,NPUMP,NGT,
 AIGEAR

CCOMMON /ITABL/L(2)
 ELBOW PERFORMANCE AND DESIGN

REQUIRED INPUT
 WIDTH - WIDTH OF DUCT AT ELBOW INLET
 DEPTH - HEIGHT OF DUCT AT ELBOW INLET
 XK - RADIUS RATIO, RATIO OF THE RADIUS OF THE CENTERLINE OF
 BEND TO THE INTERNAL RADIUS OF THE DUCT
 Q - FLOW RATE, IN CUBIC FEET PER SECCND
 AREA - CROSS SECTIONAL AREA OF DUCT AT THE ELBOW, PRESUMED
 SAME AT THE INLET AND THE OUTLET
 THATA - ANGLE OF BEND, FROM HORIZONTAL TO OUTSIDE EDGE
 GNU - VISCOSITY OF STANDARD (35 PER CENT SALINITY) SALT WATER
 PI - 3.14159265
 ICOMP - INDEX INDICATING WHICH COMPONENT IS BEING LOOKED AT
 ISTRT - INDEX NOTING WHICH MODE OF OPERATION FOR THE HYDROFOIL
 NUMB - INDEX NOTING HOW MANY MODES TO BE CCNSIDERED DURING
 THIS PASS

DIMENSION SHAPE(3,4),THETA(11),XLOSS(11),ROA(10)
 DATA SHAPE/4H ELL,4HIPSE,4H ,4H CIR,4HCLE ,4H ,4H REC,
 A 4HTANG,4HLE ,4H SCU,4HARE ,4H /
 DATA THETA/0.,10.,20.,30.,40.,50.,60.,70.,80.,90.,95./
 DATA XLOSS/0.,0.029,0.059,0.08,0.107,0.133,0.156,0.176,0.198,
 A0.198/
 FRICT(RE)=(.86859#ALOG(RE/(1.964#ALOG(RE)-3.8215)))**(-2)

C
 C
 C
 C
 C
 C
 C
 C
 C
 C
 C
 C
 C
 C
 C
 C
 C


```

CORR(RE)=1.00057-.16892*ALCG(RE*1.E-5)+.0145385*ALOG(RE*1.E-5)**2
A -.16948E-2*ALCG(RE*1.E-5)*#3
REMAX=4.E+5
KOUNT=I*STRT
IELB=ICOMP-ICOMP/2
ARFA(ICOMP)=AREA(ICOMP-1)
AREAL=.5*ARFA(ICOMP)
IF(IELB.EQ.4) AREAL=.5*AREAL
C
DETERMINE SHAPE
C
IJ=0
IK=1
IF(WIDTH.EQ.DEPTH) IJ=1
IF(ABS(WIDTH*DEPTH-AREAL).LT..05*AREAL) IK=3
ITYPE=IJ+IK
C
ITYPE SHAPE
C 1 ELLIPSE
C 2 CIRCLE
C 3 RECTANGLE
C 4 SQUARE
C
FACTR=(.00229+.041452*XK(IELB)**(-1.96))*XK(IELB)**.84
C
DETERMINE EQUIVALENT RADII
C
RO(IELB)=.5*DEPTH
IF(ITYPE.EQ.1) RO(IELB)=WIDTH*DEPTH*SQRT(2./(WIDTH**2+DEPTH**2))
C
TRANSFER SHAPE TO CALLING PROGRAM
C
DO 1 N=1,3
1 TYPE(N,IELB)=SHAPE(N,ITYPE)
C

```

```

ELBOW035
ELBOW036
ELBOW037
ELBOW038
ELBOW039
ELBOW040
ELBOW041
ELBOW042
ELBOW043
ELBOW044
ELBOW045
ELBOW046
ELBOW047
ELBOW048
ELBOW049
ELBOW050
ELBOW051
ELBOW052
ELBOW053
ELBOW054
ELBOW055
ELBOW056
ELBOW057
ELBOW058
ELBOW059
ELBOW060
ELBOW061
ELBOW062
ELBOW063
ELBOW064
ELBOW065
ELBOW066
ELBOW067
ELBOW068
ELBOW069
ELBOW070

```



```

C ELBOW071
C ELBOW072
C ELBOW073
C ELBOW074
C ELBOW075
C ELBOW076
C ELBOW077
C ELBOW078
C ELBOW079
C ELBOW080
C ELBOW081
C ELBOW082
C ELBOW083
C ELBOW084
C ELBOW085
C ELBOW086
C ELBOW087
C ELBOW088
C ELBOW089
C ELBOW090
C ELBOW091
C ELBOW092
C ELBOW093
C ELBOW094
C ELBOW095
C ELBOW096
C ELBOW097
C ELBOW098
C ELBOW099
C ELBOW100
C ELBOW101
C ELBOW102
C ELBOW103
C ELBOW104
C ELBOW105
C ELBOW106

C FIND BEND RADII
C RIN=RO*(IELB)*(XK(IELB)-1.)
C ROUT=RIN+2.*RO(IELB)
C IF XK=R/RO IS LE. 1. DO NOT USE SPLITTERS METHOD.....
C USE THIN TURNING VANES CALCULATION
C IF(XK(IELB).LE.1.) GO TO 100
C DETERMINE NUMBER OF VANES REQUIRED FOR MINIMUM LOSS
C RATIO IS THE OPTIMUM RATIO OF THE INBOARD AND OUTBOARD RADII FOR
C MINIMUM LOSS IN THE BEND
C RATIO=4.3
C XN=ALOG((XK(IELB)-1.)/(XK(IELB)+1.))/ALCG((RATIO-1.)/(RATIO+1.))
C N=XN+.5
C IF(N.EQ.0) N=1
C N1=N-1
C IF(N1.LE.0) N1=1
C FIND THE RATIO OF THE INSIDE RADIUS TO OUTSIDE RADIUS OF ANY OF
C SUBDIVIDED ELBOWS
C RATEO=(RIN/ROUT)**(1./FLOAT(N))
C N = NUMBER OF SUBDIVIDED ELBOWS
C N1 = NUMBER OF SPLITTERS
C COMPUTE HEAD LOSSES FOR EACH SUBDIVIDED ELBOW, STARTING FROM INSIDE
C NOTE THE MAXIMUM REYNOLDS NUMBER THE EQUATION IS GOOD FOR, AND
C CORRECT FOR THE ACTUAL REYNOLDS NUMBER IF ABOVE THE MAXIMUM
C N.B. THE REYNOLDS NUMBER IS CALCULATED FROM THE SUBDIVIDED ELBOW,
C NOT THE ORIGINAL ELBOW
C SUM=0.

```


ELBOW107
 ELBOW108
 ELBOW109
 ELBOW110
 ELBOW111
 ELBOW112
 ELBOW113
 ELBOW114
 ELBOW115
 ELBOW116
 ELBOW117
 ELBOW118
 ELBOW119
 ELBOW120
 ELBOW121
 ELBOW122
 ELBOW123
 ELBOW124
 ELBOW125
 ELBOW126
 ELBOW127
 ELBOW128
 ELBOW129
 ELBOW130
 ELBOW131
 ELBOW132
 ELBOW133
 ELBOW134
 ELBOW135
 ELBOW136
 ELBOW137
 ELBOW138
 ELBOW139
 ELBOW140
 ELBOW141
 ELBOW142

```

RIA=RIIN
V=Q(KOUNT)/AREA(ICOMP)
DO 3 I=1,N
  ROA(I)=RIA/RATED
  HGT=ROA(I)-RIA
  AA=PI*HGT*HGT*.25
  IF(ITYPE.EQ.3.OR.ITYPE.EQ.4) AA=WIDTH*HGT
  RAD=HGT*PI*.25
  IF(ITYPE.EQ.3.OR.ITYPE.EQ.4) RAC=.5*(HGT+WIDTH)
  RE=AA*V/(RAD*GNU)
  XCORR=1.
  IF(PE.LT.REMAX) GO TO 7
  XCORR=CORR(RE)/CORR(REMAX)
  RE=REMAX
  7 XKT=XCORR*FACTR*THATA(IELB)*RE*(-.17)
  KIA=ROA(I)
  3 SUM=SUM+XKT*AA
  HEAD=SUM/AREAL+.5*V*V/G
  GO TO 5
  C
  C THIN,CIRCULAR ARC TURNING VANE CALCULATIONS
  C
  100 XCORR=1.
  DIAM=2.*RO(IELB)
  IF(ITYPE.EQ.3.OR.ITYPE.EQ.4) DIAM=2.*WIDTH*DEPTH/(WIDTH*.2+DEPTH**2+DEPTH**2)
  A 2)
  RE=DIAM*V/GNU
  IF(RE.GT.1.E+5) XCORR=CORR(RE)/CORR(1.E+5)
  XKT=TABLE(THETA,XLOSS,THATA(IELB),L)
  HEAD=(FRICT(RE)*RO(IELB)*XK(IELB)*THATA(IELB)*.0174533/DIAM+XKT)
  A *.5*V*V/G*XCORR
  5 DELH(KOUNT,ICOMP)=DELH(KOUNT,ICOMP-1)+HEAD
  IF(IELB.EQ.2) DELH(KOUNT,ICOMP)=DELH(KOUNT,ICOMP)+RO(2)*XK(2)
  IF(KOUNT.EQ.2.OR.KOUNT.EQ.4) KOUNT=KOUNT+1
  IF(XK(IELE).GT.1.) GO TO 6

```


ELBOW143
ELBOW144
ELBOW145
ELBOW146
ELBOW147
ELBOW148
ELBOW149
ELBOW150
ELBOW151
ELBOW152
ELBOW153
ELBOW154
ELBOW155
ELBOW156
ELBOW157

```
V=Q(KOUNT)/AREA(ICOMP)
GO TO 100
C
C   ELBOW STRUCTURE WEIGHT CALCULATIONS
C
4 VOLV=0.
  IF(XK(IELB).LE.1.) GO TO 9
  DO 8 I=1,N1
8 VOLV=VOLV+8.7267E-4*THATA(IELB)*ROA(I)*RIN*WIDTH
9 VOL=VOLV+THATA(IELB)*RO(IELB)*XK(IELB)*AREA(ICOMP)/(134.56*RHOD)
  WGTS(1,ICOMP)=VOL*PHOD*G
  WGTS(2,ICOMP)=AREA(ICOMP)*RHOW*G*THATA(IELB)*RO(IELB)*XK(IELB)*
  A .0174533
  RETURN
  END
```


CFS 000
CFS 001
CFS 002
CFS 003
CFS 004
CFS 005
CFS 006
CFS 007
CFS 008
CFS 009

```
FUNCTION CFS(R)  
C SCHDENHERR SKIN FRICTION CCEFFICIENT  
C  
C     CFS=0.004  
1 DCFS=(0.242/ALOG10(R*CFS))**2-CFS  
CFS=CFS+DCFS  
IF(DCFS.GT.1.E-6) GO TO 1  
RETURN  
END
```

C
C
C


```

C      FUNCTION TABLE(XTAB,YTAB,XIN,L)
C      DIMENSION XTAB(2),YTAB(2),X(5),Y(5),A(5),B(5),L(2)
C
C      L(1) - NUMBER OF PAIRS OF DATA POINTS ENTERED
C      L(2) - DEGREE OF FIT, MAXIMUM IS FOUR
C      XTAB - DATA ARRAY CF X VALUES
C      YTAB - DATA ARRAY CF Y VALUES
C      XIN  - INDEPENDENT VARIABLE
C      TABLE - DEPENDENT VARIABLE CORRESPONDING TO XIN
C
C      NPPTS=L(1)
C      K=L(2)+1
C      IF(K.GT.NPPTS) K=NPPTS
C
C      BRANCH TO TEN IF X IS INCREASING
C      BRANCH TO 160 IF X IS DECREASING
C      IF XTAB(1).EQ.XTAB(2) ABCRT RUN
C
C      IF(XTAB(1)-XTAB(2)) 10,290,160
C      10 IF(XTAB(1)-XIN) 20,140,200
C      20 DO 120 IX=2,NPPTS
C
C      FIND XTAB VALUES BRACKETING XIN
C
C      IF(XTAB(IX).LE.XTAB(IX-1)) GO TO 290
C      IF(XTAB(IX)-XIN) 120,150,40
C      120 CONTINUE
C      GO TO 130
C      40 CONTINUE
C
C      IF XIN LIES BETWEEN EITHER END POINT OF THE XTAB ARRAY AND ITS
C      ADJACENT POINT, THE INTERPOLATION IS LIMITED TO A SECOND DEGREE
C      FIT
C      IF(IX.GT.2) GO TO 60
C      IF(K.GT.3) K=3
C      60 IF(IX.LT.NPPTS) GO TO 80

```

TABLE000
TABLE001
TABLE002
TABLE003
TABLE004
TABLE005
TABLE006
TABLE007
TABLE008
TABLE009
TABLE010
TABLE011
TABLE012
TABLE013
TABLE014
TABLE015
TABLE016
TABLE017
TABLE018
TABLE019
TABLE020
TABLE021
TABLE022
TABLE023
TABLE024
TABLE025
TABLE026
TABLE027
TABLE028
TABLE029
TABLE030
TABLE031
TABLE032
TABLE033
TABLE034
TABLE035


```

IF(K.GT.3) K=3
30 NDX=IX-K/2
IF(IX.LT.NPTS) GO TO 120
NDX=NPTS-K+1
100 DO 110 IL=1,K
C
C XTAB AND YTAB VALUES FOR THE XTAB VALUES BRACKETING XIN ARE
C T TRANSFERRED TO THE LAGRANGIAN EQUATION
C
X(IL)=XTAB(NDX)
Y(IL)=YTAB(NDX)
NDX=NDX+1
110 CONTINUE
GO TO 210
130 CONTINUE
C
C TO GET PAST STATEMENT NUMBER 120, XIN IS LARGER THAN THE LARGEST
C VALUE OF X IN XTAB. EXTRAPOLATION IS NECESSARY TO FIND TABLE AT
C XIN
C TABLE=((YTAB(NPTS)-YTAB(NPTS-1))/(XTAB(NPTS)-XTAB(NPTS-1)))*
A (XIN-XTAB(NPTS))+YTAB(NPTS)
RETURN
140 IX=1
150 TABLE=YTAB(IX)
RETURN
160 IF(XIN-XTAB(1)) 170,140,200
170 DO 190 IX=2,NPTS
C
C XTAB IS SEARCHED TO FIND THE VALUE CLOSEST TO XIN
C
IF(XTAB(IX).GE.XTAB(IX-1)) GO TO 290
IF(XIN-XTAB(IX)) 190,150,40
190 CONTINUE
GO TO 130
C
C TO GO TO STATEMENT NUMBER 130 INDICATES XIN IS SMALLER THAN THE

```

```

TABLE036
TABLE037
TABLE038
TABLE039
TABLE040
TABLE041
TABLE042
TABLE043
TABLE044
TABLE045
TABLE046
TABLE047
TABLE048
TABLE049
TABLE050
TABLE051
TABLE052
TABLE053
TABLE054
TABLE055
TABLE056
TABLE057
TABLE058
TABLE059
TABLE060
TABLE061
TABLE062
TABLE063
TABLE064
TABLE065
TABLE066
TABLE067
TABLE068
TABLE069
TABLE070
TABLE071

```



```

C   SMALLEST VALUE OF X IN XTAB AND EXTRAPOLATION IS NECESSARY TO FIND
C   TABLE FOR XIN
C   200 TABLE=((YTAB(2)-YTAB(1))/(XTAB(2)-XTAB(1)))*(XIN-XTAB(1))+YTAB(1)
      RETURN
C   210 DO 220 LL=1,K
      A(LL)=1.
C   220 B(LL)=1.
      P=J.
C
C   PERFORM LAGRANGIAN INTERPOLATION
C
C   DO 280 N=1,K
C   DO 270 J=1,K
      AA=XIN-X(J)
      IF(J.EQ.N) GO TO 240
      A(N)=A(N)*AA
C   240 BB=X(N)-X(J)
      IF(BB.EQ.0.) GO TO 270
      B(N)=R(N)*BB
C   270 CC=1/BB
      C=A(N)/B(N)*Y(N)
C   280 P=P+C
      TABLE=P
      RETURN
C
C   EQUAL CONSECUTIVE OR NON-MONOTONIC VALUES OF X ENCOUNTERED IN XTAB
C   290 TABLE=1.E30
      RETURN
      END

```



```

DIMENSION THET2(22),EXPAN(22)
DATA THET2/0.,2.,4.,6.,8.,10.,12.,15.,17.5,20.,25.,27.,30.,32.,
1 34.,39.,45.,50.,60.,70.,80.,90./
DATA EXPAN/0.,.005,.015,.03,.05,.09,.17,.3,.4,.5,.7,.8,.91,0.97,
A 1.,1.033,1.04,1.04,1.037,1.035,1.033,1.03/
FRICT(RE)=(.86859*ALOG(RE/(1.964*ALOG(RE)-3.8215)))**(-2)
IF(ISTRT.EQ.3) GO TO 10

C
C FIND EFFECTIVE LENGTH FOR EITHER GENERALIZED RECTANGLE OR ELLIPSE
STRUT040
STRUT041
WGT5(1,ICOMP)=1.E9
AREA(ICOMP)=2.*AREA(ICOMP-1)
IF(.NOT.IPUMP) GO TO 5
AREA(ICOMP)=AREA(8)
IF(AREA(8).LT.AREA(7).AND.AREA(4).GT.AREA(7)) AREA(ICOMP)=AREA(4)
5 IF(AREA(ICOMP).LT.AREA(ICOMP-1)) AREA(ICOMP)=AREA(ICOMP-1)
ARATO=AREA(ICOMP)/AREA(ICOMP-1)
STRUT043
STRUT044
STRUT045
STRUT046
STRUT047
STRUT048
STRUT049
STRUT050
STRUT051
STRUT052
STRUT053
STRUT054
STRUT055
STRUT056
STRUT057
STRUT058
STRUT059
STRUT060
STRUT061
STRUT062
STRUT063
STRUT064
STRUT065

C
C SIZE STRUT EXTERNAL DIMENSIONS
VL=1.137*VO(1)
SIGMA=(HA*G-PV*G)/(.5*VL*VL)
TC=(SQRT(1.+SIGMA)-1.)/1.15
IF(TC.LT..12) TC=.12
C=SQRT(AREA(ICOMP)/TC)
T=TC*C

C
C T AND C ARE THE THICKNESS AND CHORD AT THE STRUT EXIT
STRUT059
STRUT060
C1=SQRT(AREA(ICOMP-1)/TC)
T1=TC*C1

C
C C1 AND T1 ARE THE CHORD AND THICKNESS AT THE STRUT INLET
STRUT063
STRUT064
STRUT065
GO TO 12
IF ISTRT=3, CHECK FOR CAVITATION
10 DO 11 I=ISTRT,NUMR

```



```

VL=1.137*VO(I)
SIGMA=(HAG-PV*G)/(.5*VL*VL)
SIGMI=(1.15*IC+1.)**2-1.
IF(SIGMA.LT.SIGMI) CAV(I,4)=1.
ARATO=AREA(ICCMP)/AREA(ICCMP-1)
11 CONTINUE
12 CM=.5*(C+C1)
C
C
C
FIND EQUIVALENT DIAMETERS
IF(ITYPE/2.EQ.0) GO TO 1
DEIN=WIDTH*DEPTH/(WIDTH+DEPTH)
WIDTH=SQRT(AREA(ICCMP))*.5
WIDE=2.*WIDTH
DEOUT=WIDE*WIDTH/(WIDE+WIDTH)
RC(2)=.5*WIDTH
GO TO 2
1 DEIN=WIDTH*DEPTH*SQRT(2./(WIDTH*WIDTH+DEPTH*DEPTH))
WIDTH=SQRT(AREA(ICCMP)/PI)
WIDE=2.*WIDTH
DEOUT=WIDTH*WIDE*SQRT(2./(WIDE*WIDE+WIDTH*WIDTH))
RC(2)=.5*DEOUT
2 DEAVE=DEIN+DEOUT
STR=HE+HS-RO(1)*XK(1)-RO(2)*XK(2)
IF(STR.LE.0.) RETURN
XLONG=STRP/SIN(THATA(1))*174533)
C
C
C
FIND EQUIVALENT ANGLE OF DIFFUSER, TWO THETA
STAN=(WIDE-DEPTH)*.5/XLONG
THETA=2.*ATAN(STAN)*57.29578
C
C
C
FIND EXPANSION COEFFICIENT
ECOEFF=TABLE(THETA,EXPAN,THETA,L)
C

```

```

STRUT067
STRUT068
STRUT069
STRUT070
STRUT071
STRUT072
STRUT073
STRUT074
STRUT075
STRUT076
STRUT077
STRUT078
STRUT079
STRUT080
STRUT081
STRUT082
STRUT083
STRUT084
STRUT085
STRUT086
STRUT087
STRUT088
STRUT089
STRUT090
STRUT091
STRUT092
STRUT093
STRUT094
STRUT095

```



```

C C FORM THE EXPANSION LOSS COEFFICIENT
C C FORML=ECOEFF*(1.-1./ARATO)**2
C C
C C DETERMINE INLET AND OUTLET VELOCITIES
C C
C C DEPTH=WIDTH
C C WIDTH=WIDTH
C C KOUNT=15TRT
C C 3 VELIN=O(KOUNT)/AREA(ICOMP-1)
C C VLOUT=VELIN/ARATO
C C
C C DETERMINE STRUT DRAG FROM HOERNER
C C DETERMINE SPRAY DRAG FROM SHERMAN AND LINCCLN
C C
C C RES=CM*VO(KOUNT)/GNU
C C CDS=2.*CFS(PES)*(1.+2.*TC+60.*TC**4)
C C CFM=HS/(HS+HE-PQ(2))*XK(2))*(C-C1)+C1
C C CSTRT(KOUNT)=CDS*RHO*VO(KOUNT)**2*HS/SIN(THATA(1))*0.0174533)**0.5
C C A*(CFM+C1)
C C CDSP=.03*TC
C C CSPRY(KOUNT)=CSTRT(KOUNT)*(CDSP/CDS)
C C
C C CALCULATE THE STRAIGHT PIPE FRICTION COEFFICIENT
C C
C C RE=VELIN*DEIN/GNU
C C PIPEL=FRICT(RE)*XLONG/DEIN
C C
C C LOSS COEFFICIENT WHICH IS PROPORTIONAL TO VELOCITY HEAD AT INLET
C C
C C TOTAL=FORML+PIPEL
C C
C C HEAD LOSS DUE TO DIFFUSER
C C
C C HEADL=.5*TOTAL*VELIN*VELIN/G
C C

```

```

STRUT096
STRUT097
STRUT098
STRUT099
STRUT100
STRUT101
STRUT102
STRUT103
STRUT104
STRUT105
STRUT106
STRUT107
STRUT108
STRUT109
STRUT110
STRUT111
STRUT112
STRUT113
STRUT114
STRUT115
STRUT116
STRUT117
STRUT118
STRUT119
STRUT120
STRUT121
STRUT122
STRUT123
STRUT124
STRUT125
STRUT126
STRUT127
STRUT128
STRUT129
STRUT130
STRUT131

```


STRUT132
 STRUT133
 STRUT134
 STRUT135
 STRUT136
 STRUT137
 STRUT138
 STRUT139
 STRUT140
 STRUT141
 STRUT142
 STRUT143
 STRUT144
 STRUT145
 STRUT146
 STRUT147
 STRUT148
 STRUT149
 STRUT150
 STRUT151
 STRUT152
 STRUT153

```

C TOTAL HEAD LOSS DUE TO DIFFUSER AND ELEVATION
C
C DELH(KOUNT,ICOMP)=DELH(KCUNT,ICOMP-1)+HEADL+HE-RO(2)*XK(2)
C IF(KOUNT.EQ.2.OR.KCUNT.EQ.NUMB) GO TO 4
C KCUNT=KOUNT+1
C GO TO 3
C
C CENTER OF GRAVITY CALCULATIONS
C
C 4 HGT=XLONG-HS+RO(1)*XK(1)
C CGWS=HGT*(3.*ARATC+2.*SQRT(ARATC)+1.)*.25/(ARATC+SQRT(ARATC))+1.)
C CGS(3,ICOMP)=-CGWS
C CGS(4,ICOMP)=XLS+CGWS*CO TAN(THATA(1)*.0174533)
C CGS(1,ICOMP)=CGS(3,ICOMP)
C CGS(2,ICOMP)=CGS(4,ICOMP)
C
C WEIGHT CALCULATIONS
C
C WGT(1,ICOMP)=.25)**(T+T1)*CM*XLCNG*RHOW*G
C WGT(2,ICOMP)=HGT/3.*AREA(ICOMP)*((ARATC+1.+SQRT(ARATC))*RHOW*G
C RETURN
C END

```



```

SUBROUTINE JUNCT
COMMON /PINLP/ALPHA
COMMON /H2O/TFMP,PV,RHOW,GNU,HA
COMMON /SHIP/DISP,RANGE,BEAM,HS,HE,HCL,XLS,XLPE,XLP
COMMON /CHARS/WGTS(2,15),CGS(4,15),DELH(5,15),CGSX,CGSZ
COMMON /CONST/PI,G,RHOD
COMMON /ELBW/XK(4),RJ(4),THATA(4),WIDTH,DEPTH,TYPE(3,4)
COMMON /FLOW/Q(5),AIN,AJET,ARFA(11),VJ(5),VI(5)
COMMON /INDEX/IEVAL,IEOPT,ISTRT,NUMB,IENGN,ITYPE,ICOMP,NPUMP,NGT,
AIGEAR
COMMON /ITABL/L(2)
DIMENSION BETA(19),ALAMD(19)
DATA ALAMD/.975,.97,.967,.963,.96,.957,.953,.95,.948,.945,.94,.93,.93,
A .92,0.9,0.85,0.81,0.75,0.69,0.61/
DATA BETA/0.,5.,10.,15.,20.,25.,30.,35.,40.,45.,50.,55.,60.,65.,70.
A.,75.,80.,85.,90./
FRICT(RE)=(.86859*ALOG(RE)/(1.964*ALCG(RE)-3.8215))**(-2)
AREA(ICOMP)=AREA(ICOMP-1)

ALPHA IS ANGLE THAT ONE BRANCH MAKES WITH END PIPE IN JUNCTION
KOUNT=ISTRT

MIXING LOSS COEFFICIENT OF JUNCTION

XLAMD=TABLE(BETA,ALAMD,ALPHA,L)
AMIXL=1.+XLAMD-2.*COS(.024434*ALPHA-.00583*(ALPHA*.0174533)**.2)
IF(AMIXL.LT.0.) AMIXL=0.

XPUMP=NGT+1

ASSUMES NO LOSS DUE TO CHANGE OF SHAPE TO CIRCULAR

DIAM=1.414*SQRT((AREA(ICOMP)/PI)
RO(3)=.5#DIAM
AJCT=XLS-RO(3)*XK(3)-XLP-XLPE
IF(AJCT.LT.0.) AJCT=0.

```

JUNCT000

JUNCT001

JUNCT002

JUNCT003

JUNCT004

JUNCT005

JUNCT006

JUNCT008

JUNCT009

JUNCT010

JUNCT011

JUNCT012

JUNCT013

JUNCT015

JUNCT016

JUNCT017

JUNCT018

JUNCT019

JUNCT020

JUNCT021

JUNCT022

JUNCT023

JUNCT024

JUNCT025

JUNCT026

JUNCT027

JUNCT028

JUNCT029

JUNCT030

JUNCT031

JUNCT032

JUNCT033

JUNCT034

111

C

C

C

C

C

C

C

C

C

C

C

C

C

C

JUNCT035
 JUNCT036
 JUNCT037
 JUNCT038
 JUNCT039
 JUNCT040
 JUNCT041
 JUNCT042
 JUNCT043
 JUNCT044
 JUNCT045
 JUNCT046
 JUNCT047
 JUNCT048
 JUNCT049
 JUNCT050
 JUNCT051
 JUNCT052
 JUNCT053
 JUNCT054
 JUNCT055
 JUNCT056
 JUNCT057
 JUNCT058
 JUNCT059
 JUNCT060
 JUNCT061
 JUNCT062
 JUNCT063
 JUNCT064
 JUNCT065
 JUNCT066
 JUNCT067
 JUNCT068
 JUNCT069
 JUNCT070

FLONG=BEAM/XPUMP-RO(3)*(1.+XK(3))-RO(2)*XK(2)+RO(2)
 DEPTH=DIAM
 WIDTH=DEPTH
 2 ICOMP=5
 C
 C
 C
 FIND HEAD LOSS FOR ATHWARTSHIPS LENGTH
 V=Q(KOUNT)/AREA(ICOMP)
 RE=V*DIAM/GNU
 FRCTL=FRICT(RE)*FLONG/DIAM
 DELH(KOUNT,ICOMP)=DELH(KOUNT,ICOMP-1)+FRCTL*.5*V*V/G
 ICOMP=6
 C
 C
 C
 FIND HEAD LOSS IN PUMP ELBCW
 CALL ELBOW
 RE=RE*1.414
 C
 C
 C
 FRICTION LOSS COEFFICIENT IN JUNCTION
 FRCTJ=.3535*FRICT(RE)*AJCT/DIAM+FRCTL*AJCT/FLCNG*.5
 C
 C
 C
 TOTAL JUNCTION LOSS CCEFFICIENT
 AJCTL=AMIXL+FRCTJ
 C
 C
 C
 HEAD LOSS OF JUNCTION
 ICOMP=7
 DELH(KOUNT,ICOMP)=DELH(KOUNT,ICOMP-1)+AJCTL*.5*V*V/G
 IF(KOUNT.EQ.2.OR.KCOUNT.EQ.NUMB) GO TO 1
 KCOUNT=KOUNT+1
 GO TO 2
 1 AREA(ICOMP)=AREA(ICOMP-1)
 C
 C
 C
 CENTER OF GRAVITY AND WEIGHT CALCULATIONS OF COMPONENTS OF PUMP
 INLET PIPING.

JUNCT071
JUNCT072
JUNCT073
JUNCT074
JUNCT075
JUNCT076
JUNCT077
JUNCT078
JUNCT079
JUNCT080
JUNCT081
JUNCT082
JUNCT083
JUNCT084
JUNCT085
JUNCT086
JUNCT087
JUNCT088
JUNCT089
JUNCT090
JUNCT091

CGS(1,6)=HCL
CGS(2,6)=XLS
CGS(3,6)=HCL
CGS(4,6)=XLS
WGTS(1,5)=13.7*AREA(5)*FLCNG
WGTS(2,5)=AREA(5)*FLONG*RHOW*G
CGS(1,5)=HCL
CGS(2,5)=XLS
CGS(3,5)=HCL
CGS(4,5)=XLS
WGTS(1,ICOMP)=13.7*AJCT*AREA(ICOMP)
WGTS(2,ICOMP)=AJCT*AREA(ICOMP)*RHOW*G
CGS(1,ICOMP)=HCL
CGS(2,ICOMP)=XLS-RD(3)*XK(3)*SIN(THATA(3)*.008757)-.5*AJCT
CGS(3,ICOMP)=HCL
CGS(4,ICOMP)=CGS(2,ICOMP)
DEPTH=1.414*DIAM
WIDTH=DEPTH
RETURN
END

PIPE0036
 PIPE0037
 PIPE0038
 PIPE0039
 PIPE0040
 PIPE0041
 PIPE0042
 PIPE0043
 PIPE0044
 PIPE0045
 PIPE0046
 PIPE0047
 PIPE0048
 PIPE0049
 PIPE0050
 PIPE0051
 PIPE0052
 PIPE0053
 PIPE0054
 PIPE0055
 PIPE0056
 PIPE0057
 PIPE0058
 PIPE0059
 PIPE0060
 PIPE0061
 PIPE0062
 PIPE0063
 PIPE0064
 PIPE0065

ICOMP=7
 XKT=FRIC(TRE)*APIPE/DIAM
 DELH(KOUNT,ICOMP)=DELH(KCUNT,ICOMP-1)+.5*XKT*V*V/G
 IF(KOUNT.EQ.2.OR.KOUNT.EQ.NUMB) GO TO 1
 KOUNT=KOUNT+1

GO TO 2

1 AREA(ICOMP)=AREA(ICOMP-1)

CENTER OF GRAVITY AND WEIGHT CALCULATIONS OF COMPONENTS OF PUMP
 INLET PIPING

WGT(1,5)=13.7*XLONG*AREA(5)
 WGT(2,5)=AREA(5)*XLCNG*RHOW*G

CGS(1,5)=HCL

CGS(2,5)=XLS

CGS(3,5)=HCL

CGS(4,5)=XLS

CGS(1,6)=HCL

CGS(2,6)=XLS

CGS(3,6)=HCL

CGS(4,6)=XLS

WGT(1,ICOMP)=13.7*APIPE*AREA(ICOMP)

WGT(2,ICOMP)=AREA(ICOMP)*APIPE*RHOW*G

CGS(1,ICOMP)=HCL

CGS(2,ICOMP)=XLS-RO(3)*XK(3)*SIN(THATA(3)*.008727)-.5*APIPE

CGS(3,ICOMP)=HCL

CGS(4,ICOMP)=CGS(2,ICOMP)

RETURN

END

C

C

C

C


```

SUBROUTINE DIVRG
LOGICAL IFUEL, IPUMP
COMMON /H2O/TEMP, PV, RHO, GNU, HA
COMMON /SHIP/ DISP, RANGE, BEAM, HS, HE, HCL, XLS, XLPE, XLP
COMMON /CONST/PI, G, RHOD
COMMON /CHARS/WGTS(2,15), CGS(4,15), DELH(5,15), CGSX, CGSZ
COMMON /ELBW/XK(4), RO(4), THATA(4), WIDTH, DEPTH, TYPE(3,4)
COMMON /FLOW/O(5), AIN, AJET, AREA(11), VJ(5), VI(5)
COMMON /INDEX/IEVAL, IEQPT, ISTRT, NUMB, IENGN, ITYPE, ICCMP, NPUMP, NGT,
AIGEAR
COMMON /ITABL/L(2)
COMMON IFUEL, IPUMP
DIMENSION THETA(12), COEF(12)
DATA THETA/0.,10.,20.,30.,40.,50.,60.,70.,80.,90.,100.,110./
DATA COEF/0.005,0.04,0.18,0.36,0.57,0.77,0.955,1.14,1.3,1.42,1.5,
1 1.57/
FRIC( RE) = (.86859 * ALOG( RE / (1.964 * ALOG( RE ) - 3.8215 ) ) ) * * (-2)
KOUNT = ISTRT
AREA( ICOMP ) = AREA( ICOMP - 1 )
DIAM = 1.414 * SORT( AREA( ICOMP ) / PI )
RO( 3 ) = .5 * DIAM
XPUMP = NGT + 1
FLONG = 1.5 * BEAM / XPUMP - RO( 2 ) * XK( 2 ) + RO( 2 ) - RO( 3 ) * XK( 3 ) * SIN( THATA( 3 ) )
A = .008727 )
CENTER OF GRAVITY AND WEIGHT CALCULATIONS FOR ATHWARTSHIPS LENGTH
WGTS( 1, ICOMP ) = 13.7 * FLONG * AREA( ICOMP )
WCTS( 2, ICOMP ) = AREA( ICOMP ) * FLONG * RHO * G
CGS( 1, ICOMP ) = HCL
CGS( 2, ICOMP ) = XLS
CGS( 3, ICOMP ) = HCL
CGS( 4, ICOMP ) = XLS
RO( 4 ) = .3535 * DIAM
3 ICOMP = 5

```

DIVRG000
DIVRG001
DIVRG002
DIVRG003
DIVRG004
DIVRG005
DIVRG006
DIVRG007

DIVRG009
DIVRG010
DIVRG011
DIVRG012
DIVRG013
DIVRG014
DIVRG015
DIVRG016
DIVRG017
DIVRG018
DIVRG019
DIVRG020
DIVRG021
DIVRG022
DIVRG023
DIVRG024
DIVRG025
DIVRG026
DIVRG027
DIVRG028
DIVRG029
DIVRG030
DIVRG031
DIVRG032
DIVRG033
DIVRG034

C
C
C
C


```

C      WIDTH=.707*WIDTH
C      DEPTH=WIDTH
C      IF(KOUNT.EQ.2) ISTRT=2
C
C      ELBOW JUST PRIOR TO PUMP INLET
C
C      CALL ELBOW
C      DELH(KOUNT,ICOMP)=DELH(KOUNT,ICOMP)+HEADL
6      IF(KOUNT.EQ.2.OR.KOUNT.EQ.NUMB) GO TO 4
C      KOUNT=KOUNT+1
C      GO TO 3
4      CGS(1,6)=HCL
C      IF(KOUNT.FQ.2) ISTRT=1
C
C      CENTER OF GRAVITY AND WEIGHT CALCULATIONS OF PUMP ELBOW AND
C      TRANSITION PIECE(DIVERGENCE)
C
C      CGS(2,6)=XLS
C      CGS(3,6)=HCL
C      CGS(4,6)=XLS
C      CGS(1,ICOMP)=HCL
C      CGS(3,ICOMP)=HCL
C      DWGT1=WGTS(1,ICOMP)
C      DWGT2=13.7*XLONG*AREA(ICOMP)
C      DWGT3=DWGT1
C      WWGT1=WGTS(2,ICOMP)
C      WWGT2=XLONG*AREA(ICOMP)*.5*RHOW*G
C      WWGT3=WWGT1
C      IF(ANGLE.EQ.0.) RETURN
C      CGWX1=XLS-RO(3)*XK(3)-SIN(.5*ANGLE)*RO(4)*XK(4)
C      CGWX2=XLS-RO(3)*XK(3)-.5*XLCNG*SIN(ANGLE)-RO(4)*XK(4)*SIN(ANGLE)
C      CGWX3=XLP+XLPF+SIN(.5*ANGLE)*RO(4)*XK(4)
C      CGS(2,ICOMP)=(DWGT1+DWGT2*CGWX2+DWGT3*CGWX3)/(DWGT1+DWGT2+DWGT3)
C      IGT3)
C      CGS(4,ICOMP)=(WWGT1*CGWX1+WWGT2*CGWX2+WWGT3*CGWX3)/(WWGT1+WWGT2+
1      WWGT3)

```

```

DIVRG070
DIVRG071
DIVRG072
DIVRG073
DIVRG074
DIVRG075
DIVRG076
DIVRG077
DIVRG078
DIVRG079
DIVRG080
DIVRG081
DIVRG082
DIVRG083
DIVRG084
DIVRG085
DIVRG086
DIVRG087
DIVRG088
DIVRG089
DIVRG090
DIVRG091
DIVRG092
DIVRG093
DIVRG094
DIVRG095
DIVRG096
DIVRG097
DIVRG098
DIVRG099
DIVRG100
DIVRG101
DIVRG102
DIVRG103
DIVRG104
DIVRG105

```


DIVRG106
DIVRG107
DIVRG108
DIVRG109

WGTS(1, ICOMP) = DWGT1 + DWGT2 + DWGT3
WGTS(2, ICOMP) = WWGT1 + WWGT2 + WWGT3
RETURN
END


```

NOZZL000
NOZZL001

NOZZL002
NOZZL003
NOZZL004
NOZZL005
NOZZL006
NOZZL007
NOZZL008

NOZZL010
NOZZL011
NOZZL012
NOZZL013
NOZZL014
NOZZL015
NOZZL016
NOZZL017
NOZZL018

NOZZL020
NOZZL021
NOZZL022
NOZZL023
NOZZL024
NOZZL025
NOZZL026
NOZZL027
NOZZL028
NOZZL029
NOZZL030
NOZZL031

SUBROUTINE NOZZL
LOGICAL IFUEL, IPUMP
REAL JANGL
COMMON /NOZL/JANGL
COMMON /DRAG/TDRAG(5), STRTD(5), POD(5), SPRAY(5), REST(5), VO(5),
1 TRIM(5)
COMMON /FLOW/Q(5), AIN, AJET, AREA(11), VJ(5), VI(5)
COMMON /CHARS/WGTS(2,15), CGS(4,15), DELH(5,15), CGSX, CGSZ
COMMON /CONST/PI, G, RHOD
COMMON /SHIP/DISP, RANGE, BEAM, HS, HE, HCL, XLS, XLPE, XLP
COMMON /H2O/TEMP, PV, RHOV, GNU, HA
COMMON /INDEX/IEVAL, IEQPT, ISTRT, NUMB, IENGN, ITYPE, ICCOMP, NPUMP, NGT,
AIGEAR
COMMON IFUEL, IPUMP
DIMENSION QQ(5)
FRICT(RE)=(.86859**ALOG(RE)/(1.964**ALOG(RE)-3.8215))**(-2)
KCOUNT=ISTRT
XLPS=XLPE
XFAC=1.4**HCL/XLPE
XPUMP=NPUMP
IF(.NOT.IPUMP) AREA(8)=AREA(7)
AREA(ICOMP)=AJET
IF(ISTRT.EQ.1) ANOZ=ATAN(TDRAG(1)/DISP)
IF(ISTRT.NE.1) ANCZ=JANGL

C
C
C
XFAC.LT.1. INDICATES NOZZLE EXITS THROUGH BOTTOM

IF(XFAC.LT.1.) XLPS=1.4**HCL
XLNOZ=XLPS/COS(ANOZ)
DT=2.*SQRT(AREA(ICOMP-1)/(PI*XPUMP))
DJ=2.*SQRT(AJET/(PI*XPUMP))
IF(DT.LT.DJ) GC TC 3
1 QQ(KOUNT)=Q(KOUNT)/XPUMP
XCORR=XLNOZ/(DT-DJ)*(1.-(2./(1.+DT/DJ))**4)*VJ(KOUNT)/G**NOZZL029
1.25
RE=8.*QQ(KOUNT)/(PI*(DT+DJ)*GNU)

```



```

C      K = CRUISE POINT , J = TAKE OFF PCINT
C      PUMP DESIGN POINT,J, IS AT TAKE OFF BASED ON THOMA'S CRITERION
C      OFF DESIGN POINT,K, IS ANY OTHER PCINT ALONG THE DRAG TO SPEED CUR
PUMP 034
PUMP 035
PUMP 036
PUMP 037
PUMP 038
PUMP 039
PUMP 040
PUMP 041

      K=1
      J=2
      DO 4 I=1,5
      WPAT(I)=0.
      XPUMP=FLOAT(NPUMP)
      QQ(J)=Q(J)/XPUMP
      JNUMB=NUMB
      IF(ISTRT.EQ.1) JNUMB=2
      DO 2 I=ISTRT,JNUMB
      QQ(I)=Q(I)/XPUMP
      HPP(I)=(VJ(I)*VJ(I)-VO(I)*VO(I))/(2.*G)+DELH(I,7)+DELH(I,8)
      HSV(I)=HA-DELH(I,7)+VO(I)*VO(I)/(2.*G)-PV
      THOM(I)=HSV(I)/HPP(I)
2      CONTINUE
      IF(ISTRT.EQ.3) GO TO(200,401,401,300),NSTG
      NSTG=1
      IF(HSV(J).LT.0.005) GO TO 10

C      FOR THE VALUE ON THOMA'S CAVITATION CRITERION LESS THAN THE CUT
C      OFF POINT, THOMI, THE INDUCER PLUS ONE AXIAL STAGE IS REQUIRED
C
C      THOMI=0.055
C      IF(THOM(J).LT.THOMI) GO TO 201

C      SINGLE STAGE INDUCER DESIGN
C      PUMP CHARACTERISTICS
C      XNS IS THE SPECIFIC SPEED (CFS) FOR THE FIRST STAGE IN A MULTI-STAPUMP
C      DESIGN OR FOR ONE IMPELLER IN A MULTI-DOUBLE SUCTION IMPELLER DESI
PUMP 051
PUMP 052
PUMP 053
PUMP 054
PUMP 055
PUMP 056
PUMP 057
PUMP 058
PUMP 059
PUMP 060
PUMP 061
PUMP 062
PUMP 063
C      SM IS THE MAXIMUM SUCTION SPECIFIC SPEED AT TAKE OFF .
200 XNS(NSTG)=149.5
      IF(ISTRT.NE.3) SM(2,1)=XNS(1)*(1.0/THCM(J))*0.75
      PHI(2,1)=0.11

```



```

DRAT=0.30
XXLP=1.79
CW=347.1*XPUMP
IF(ISTRT.EQ.3)GO TO 402
C
C
C
DESIGN POINT PUMP SPEED
RPM(J,NSTG)=XNS(NSTG)*HPP(J)**0.75/SQRT(QQ(J))
C
C
C
PUMP INLET TIP DIAMETER
DIS(1)=(240.*QQ(J))/(9.4748*PHI(2,1)*(1.-DRAT**2)*RPM(J,1))**(1./3
1.)
SHI(2,1)=G*HPP(J)/(PI*RPM(J,1)*DIS(1)/60.)***2
IF(SHI(2,NSTG).GT.0.41)GO TO 201
402 ETAP(J,NSTG)=1.-((3.666/DIS(NSTG)**3).165*(1.-.915))
JNUMB=NUMB
IF(ISTRT.EQ.1)JNUMB=1
DO 403 K=ISTRT,JNUMB
QX=QQ(K)/QQ(J)
ETAX=PC(2,1)*QX**2+PC(2,2)*QX+PC(2,3)
IF(ETAX.LE.0.)ETAX=0.001
ETAP(K,NSTG)=ETAP(J,NSTG)*ETAX
C
C
C
OFF DESIGN POINT PUMP RPM
CA=PC(1,2)/PC(1,3)*0.5
CR=PC(1,1)/PC(1,3)
HX=HPP(K)/HPP(J)
RX=-CA*QX-SQRT(QX*QX*(CA*CA-CB)+HX/PC(1,3))
RPM(K,NSTG)=RPM(J,NSTG)*RX
C
C
C
PHRAT IS THE OFF DESIGN TO DESIGN FLOW COEFFICIENT RATIO, PHI(K)/PPUMP 091
PHRAT=QX/RX
C

```

PUMP 066
PUMP 067
PUMP 068

PUMP 069
PUMP 070
PUMP 071
PUMP 072
PUMP 073
PUMP 074
PUMP 075

PUMP 080

PUMP 082
PUMP 083
PUMP 084
PUMP 085
PUMP 086
PUMP 087
PUMP 088
PUMP 089
PUMP 090
PUMP 091
PUMP 092
PUMP 093
PUMP 094


```

IF(ISTR.EQ.1)GO TO 403
SHI(K,NSTG)=G#HPP(K)/(PI#RPM(K,NSTG)*DIS(NSTG)/60.)*#2
PHI(K,NSTG)=PHRAT*PHI(2,NSTG)
SM(K,NSTG)=RPM(K,NSTG)*SQRT(QQ(K))/HSV(K)*#0.75
403 CONTINUE
C DRY PUMP WEIGHT
PUMP 095
PUMP 096
PUMP 097
PUMP 098
PUMP 099
PUMP 100
PUMP 101
PUMP 102
PUMP 103
PUMP 104
PUMP 105

C
C
C ENTRAINED WATER WEIGHT
XWW(NSTG)=0.523*APUP(NSTG)*PLP(NSTG)*RHOW*G*XPUMP
XNGT=NGT
IF(ISTR.EQ.3)GO TO 404
SHP(2,NSTG)=RHOW*G*Q(2)#HPP(2)/(550.*XNGT*ETAP(2,NSTG)*.98)
GERAT(NSTG)=PERF(4,IENGN)/RPM(J,NSTG)
IF(GERAT(NSTG).LT..95)GO TO 16
404 IF(ISTR.EQ.3)SHP(2,NSTG)=PERF(1,IENGN)
CALL GEAR
XWG(NSTG)=WG
JNUMB=NUMB
IF(ISTR.EQ.1)JNUMB=2
DO 3 I=ISTR,JNUMB
ETAPP=ETAP(I,NSTG)*ETAP(7,NSTG)
SHP(I,NSTG)=RHOW*G*Q(I)#HPP(I)/(550.*XNGT*ETAPP)
3 IF(SHP(I,NSTG).GT.PERF(I,IENGN).AND.DELTA.GT.1.E-9)XWW(NSTG)=1.E9PUMP 112
PUMP 113
PUMP 114
PUMP 115
PUMP 116
PUMP 117
PUMP 118
PUMP 119

CALL FUEL
XWF(NSTG)=WF
WKAT(NSTG)=(XWD(NSTG)+XHW(NSTG)+XWG(NSTG)+XWF(NSTG))/DISP
IF(ISTR.EQ.3)GO TO 413
GO TO 300
16 XWG(NSTG)=1.E9

```


PUMP 121
 PUMP 122
 PUMP 123
 PUMP 124
 PUMP 125
 PUMP 126
 PUMP 127
 PUMP 128
 PUMP 129
 PUMP 130
 PUMP 131
 PUMP 132
 PUMP 133
 PUMP 134
 PUMP 135

PUMP 138
 PUMP 139
 PUMP 140

PUMP 141
 PUMP 142
 PUMP 143
 PUMP 144
 PUMP 145
 PUMP 146
 PUMP 147

```

GO TO 300
INDUCER PLUS ONE AXIAL STAGE DESIGN
201 THOMI=0.058
    NSTG=2
    HP=HSV(J)/THOMI
THOMS IS THOMAS'S CAVITATION CRITERION FOR THE AXIAL STAGE
BASED ON A MAXIMUM S OF 10,000 AND NS=3619
THOMS=0.258
INDUCER SPECIFIC SPEED AND FLOW COEFF.
401 XNS(NSTG)=149.5
    PHI(2,2)=0.15
    DRAT=C.3
    XLPL=1.71
    CW=393.5*XPUMP
    IF(NSTG.EQ.3) XLPL=2.03
    IF(NSTG.EQ.3) CW=439.5*XPUMP
    IF(ISTR.EQ.3)GO TO 405
DESIGN PUMP SPEED
RPM(J,NSTG)=XNS(NSTG)*HP**0.75/SQRT(OO(J))
PUMP INLET TIP DIAMETER
DIS(2)=(240.*OO(J)/(9.4748*PHI(2,2)*(1.-DRAT**2)*RPM(J,2)))**(1./
13.)
405 ETAP(J,NSTG)=1.0-((3.666/DIS(NSTG))**0.165*(1.-.915))
JNUMB=NUMB
IF(ISTR.EG.1) JNUMB=1
DO 406 K=ISTR,JNUMB

```



```

RPM(J,3)=RPM(J,2)
RPM(K,3)=RPM(K,2)
ETAP(J,3)=ETAP(J,2)
ETAP(K,3)=ETAP(K,2)
IF(((HSV(J)+HP)/HHP).LT.THCMS) GO TO 10
GO TO 203
C
C
C
MULTI PARALLEL IMPELLER DESIGN
C
300 NSTG=4
SM(2,NSTG)=424.5
C
C
PUMP CHARACTERISTICS.
C
THE CENTRIFUGAL IMPELLER DESIGN IS BASED ON A CONSTANT IMPELLER
C
EXIT ANGLE OF 22 DEGREES AND A HYDRAULIC EFFICIENCY OF 0.90
C
C
DRAT=0.50
BETA2=2.477
IF(ISTR.NE.3)XNS(4)=SM(2,4)*THOM(J)**C.75
IF(ISTR.EQ.3) XNS(4)=RPM(J,NSTG)*SQRT(QQ(J))/HPP(J)**0.75
XNS=XNS(4)/811.3
C
C
THE IMPELLER EXIT WIDTH RATIO IS ASSUMED TO BE LINEAR WITH NS
C
C
BD=0.001*XNS(4)-).025
IF(XNS(4).LT.50.) BD=0.025
CC=SQRT(BD/PI)
PHI(2,4)=0.06
14 AA=CC*SQRT(PHI(2,4))/(0.9*(1.-PHI(2,4)+BETA2)**0.75
IF((XNNS-AA).LE.0.005) GC TC 15
PHI(2,4)=PHI(2,4)+0.005
GO TO 14
15 SHI(2,4)=0.90*(1.-PHI(2,4)+2.414)
C
C
C
SPECIFIC SPEED AND PUMP CHARAC. ARE FOR A SINGLE IMPELLER
C
C
C

```

```

PUMP 177
PUMP 178
PUMP 179
PUMP 180
PUMP 181
PUMP 182
PUMP 183
PUMP 184
PUMP 185
PUMP 186

PUMP 188
PUMP 189
PUMP 190
PUMP 191
PUMP 192
PUMP 193
PUMP 194

PUMP 196
PUMP 197
PUMP 198
PUMP 199
PUMP 200
PUMP 201
PUMP 202

PUMP 205

PUMP 207

PUMP 209
PUMP 210
PUMP 211

```



```

PCC(1,3)=XNS(4)/283.85-.17
PCC(1,2)=2.5-PCC(1,3)
IF(ISTR.EQ.3)GO TO 407

C
IMPL=10
DO5 M=1,IMPL
XIM EQUALS THE NUMBER OF DOUBLE SUCTION IMPELLERS
XIM=FLOAT(M)
QQ(J)=Q(J)/(2.0*XIM*XPUMP)
QQ(K)=Q(K)/(2.0*XIM*XPUMP)
QX=CQ(K)/QQ(J)
XRPM(M)=XNS(4)*HPP(J)**).75/SGRT(QQ(J))
XD1(M)=(240.*QQ(J)/(9.4748*0.2500*(1.-DRAT**2)*XRPM(M))***(1./3.))
XD2(M)=60.0*SGRT(GHPP(J)/SHI(2,4))/(PI*XRPM(M))
OFF DESIGN POINT PPM
CA=PCC(1,2)/PCC(1,3)*0.5
CB=PCC(1,1)/PCC(1,3)
HX=HPP(K)/HPP(J)
RX=-CA*QX+SGRT(QX*QX*(CA*CA-CB)+HX/PCC(1,3))*(CA/ARS(CA))
RPM(M)=XRPM(M)*RX

C
ETAP(J,4)=1.-((2.333/XD2(M))**0.165*(1.-.880))
ETAX=PCC(2,1)*QX**2+PCC(2,2)*QX+PCC(2,3)
IF(ETAX.LE.0.) ETAX=0.001
ETAP(K,NSTG)=ETAP(J,NSTG)*ETAX

C
PUMP WEIGHT
XPUP(M)=101.91*G(J)/(XRPM(M)*XD1(M)*XPUMP)
YLP(M)=SGRT(XPUP(M)/(XIM*0.866))*(1.+XIM)
CW=4.66
WD(M)=(XIM*.725+.275)*CW*XD2(M)*XD2(M)*XNS(4)*XPUMP
WX(M)=0.55*WD(M)

C
SHP(2,NSTG)=RHGM*G*0(2)*HPP(2)/(550.*XNGT*ETAP(2,NSTG)*.98)

```

```

PUMP 212
PUMP 213

PUMP 214
PUMP 215
PUMP 216
PUMP 217
PUMP 218
PUMP 219
PUMP 220
PUMP 221
PUMP 222
PUMP 223

PUMP 225
PUMP 226
PUMP 227
PUMP 228
PUMP 229
PUMP 230
PUMP 231
PUMP 232

PUMP 234
PUMP 235
PUMP 236
PUMP 237
PUMP 238
PUMP 239
PUMP 240
PUMP 241
PUMP 242
PUMP 243
PUMP 244

```


PUMP 245
 PUMP 246
 PUMP 247
 PUMP 248
 PUMP 249
 PUMP 250
 PUMP 251
 PUMP 252
 PUMP 253
 PUMP 254
 PUMP 255
 PUMP 256
 PUMP 257
 PUMP 258
 PUMP 259
 PUMP 260
 PUMP 261
 PUMP 262
 PUMP 263
 PUMP 264
 PUMP 265
 PUMP 266
 PUMP 267
 PUMP 268
 PUMP 269
 PUMP 270
 PUMP 271

```

XERAT(M)=PERF(4,IENGN)/XRPM(M)
GERAT(4)=XERAT(M)
CALL GEAR
WVG(M)=WG
DO7 J=1,2
  ETAPP=ETAP(I,NSTG)*ETAP(7,NSTG)
  SHP(I,NSTG)=RHCW*Q(I)*HPP(I)/(55)*XNGT*ETAPP
7 IF(SHP(I,NSTG).GT.PERF(I,IENGN).AND.DELTA.GT.1.E-9) WW(M)=1.E9
  IF(M.EQ.1) GO TO 5
  IF(THOM(J).LT.0.046) GO TO 12
  IF(XFRAT(M).LE.1.0) GO TO 6
  IF((WD(M)+WW(M)+WVG(M)).GT.(WD(M-1)+WW(M-1)+WVG(M-1))) GO TO 6
5 CONTINUE
M=IMPL+1
GO TO 6
12 WD(M-1)=1.E9
6 XWD(4)=WD(M-1)
  XWW(4)=WW(M-1)
  XWG(4)=WVG(M-1)
  XIM=M-1
  RPM(J,4)=XRPM(M-1)
  DIS(4)=XD1(M-1)
  D2S(4)=XD2(M-1)
  kPM(K,4)=PPK(M-1)
  APUP(4)=XPUP(M-1)
  PLP(4)=YLP(M-1)
  GERAT(4)=XEPAT(M-1)
  IF(GERAT(4).LT..95) XWG(4)=1.E9
  IF(ISTRT.NE.3)GO TO 408
4J DO 4,9 K=ISTRT,NUMB
  QQ(K)=QQ(K)/(XIM**2.0)
  QQ(J)=QQ(J)/(XIM**2.0)
  QX=QQ(K)/QQ(J)
  RPM(J,NSTG)=XNS(NSTG)*HPP(J)**0.75/SQRT(QQ(J))
  CA=PCC(1,2)/PCC(1,3)**0.5
  CB=PCC(1,1)/PCC(1,3)

```



```

HX=HPP(K)/HPP(J)
P X=-CA*QX+SQRT(QX*QX*(CA*CA-CB)+HX/PCC(1,3))*(CA/ABS(CA))
RPM(K,NSTG)=RPM(J,NSTG)*RX
ETAP(J,NSTG)=1.0-((2.333/D2S(NSTG))**Q).165*(1.-.88J)
ETAX=PCC(2,1)*QX**2+PCC(2,2)*QX+PCC(2,3)
IF(ETAX.LE.0.) ETAX=0.001
ETAP(K,NSTG)=ETAP(J,NSTG)*ETAX
PHRAT=QX/RX
PHI(K,NSTG)=PHRAT*PHI(2,NSTG)
SHI(K,NSTG)=G*HPP(K)/(PI*RPM(K,NSTG)*D2S(NSTG)/60.)***2
439 CONTINUE
APUP(NSTG)=101.91*Q(J)/(RPM(J,NSTG)*DIS(NSTG)*XPUMP)
PLP(NSTG)=SQRT(APUP(NSTG)/(XIM*0.866))*(1.+XIM)
CW=4.66
XWD(NSTG)=(XIM*.725+.275)*CW*D2S(NSTG)*D2S(NSTG)*XNS(NSTG)*XPUMP
XWW(NSTG)=0.55*XWD(NSTG)
SHP(2,NSTG)=PERF(1,IENGN)
CALL GEAR
XWG(NSTG)=WG
DO 411 I=1,ISTPT,NUMB
ETAPP=ETAP(I,NSTG)*ETAP(7,NSTG)
SHP(I,NSTG)=RHCW*G*Q(I)+HPP(I)/(550.*XNGT*ETAPP)
411 CONTINUE
408 CONTINUE
CALL FUEL
XWF(4)=WF
C
C
C
TOTAL WEIGHT TO DISPLACEMENT RATIO
WRAT(NSTG)=(XWD(NSTG)+XWW(NSTG)+XWG(NSTG)+XWF(NSTG))/DISP
IF(ISTRT.EQ.3)GO TO 410
N=4
D08 M=1,3
IF(WRAT(M).NE.0.) GO TO 9
8 CONTINUE
9 CONTINUE
PUMP 273
PUMP 274
PUMP 275
PUMP 276
PUMP 277
PUMP 278
PUMP 279
PUMP 280
PUMP 281
PUMP 282
PUMP 283

```



```

IF(WRAT(M).GT.WRAT(N)) M=N
NSTG=M
410 M=NSTG
XLP=PLP(NSTG)
IPUMP=.NOT.IPIUMP
WGTS(1,8)=XWD(M)
WGTS(2,8)=XWW(M)
WGTS(1,11)=XWG(M)
WGTS(2,11)=XWF(M)
CGS(1,ICOMP)=HCL
CGS(2,ICOMP)=XLPE+XLP*.5
CGS(3,ICOMP)=HCL
CGS(4,ICOMP)=CGS(2,ICOMP)
AREA(8)=APUP(NSTG)**XPUMP
IF(NSTG.NE.4) XIM=NSTG-1
RETURN
10 WGTS(1,8)=1.E9
RETURN
END

```

```

PUMP 284
PUMP 285

PUMP 286
PUMP 287
PUMP 288
PUMP 289
PUMP 290
PUMP 291
PUMP 292
PUMP 293
PUMP 294
PUMP 295
PUMP 296
PUMP 297
PUMP 298

PUMP 300
PUMP 301

```



```

SUBROUTINE GEAR                                GEAR 000
LOGICAL IFUEL, IPUMP, TYGER
COMMON /INDEX/IEVAL, IEQPT, ISTRT, NUMB, IENGN, ITYPE, ICCOMP, NPUMP, NGT,
AIGEAR
COMMON /CONST/PI, G, RHOD
COMMON /PSUB/GERAT(5), SHP(5,5), RPM(5,5), PERF(5,12), ETAP(8,5)
COMMON /PUMM/QQ(5), DLS(5), D2S(5), XNS(5), SM(5,5), PLP(5), NSTG, SHI(5,
A5), XIM
COMMON /HEAD/HPP(5), HSV(5), THCM(5), PHI(5,5), WF, WG, FRATE(5)
COMMON IFUEL, IPUMP, TYGER

C
C GEAR PUMP ARRANGEMENT. IF NO. OF PUMPS = S NO. OF GTS , A PLANETARGEAR 008
C WILL BE USED. IF TWO PUMPS PER GT OR TWO GTS PER PUMP A COMBININGGEAR 009
C REDUCTION GEAR WITH IDLER WILL BE USED.
C
GRAT=GERAT(NSTG)
IF(TYGER) GO TO(20,101,12), IGEAR
IF(GRAT.GT.12.) GO TO 12
IF(NGT-NPUMP) 100,101,102
20 IF(NGT-NPUMP) 100,106,102

C
C SURROUTINE FOR SINGLE REDUCTION GEAR WITH IDLER
C
100 FAC=1.7
GO TO 3
106 FAC=1.0
GO TO 3
102 FAC=1.3
3 IF(GRAT.GT.1.0) GO TO 1
GRAT=1.0
1 C=GPAT*GPAT+1.0
D=SQRT(C*C/64.0+0.00463)
DX=C*.25-D
X=(C*.25+D)**(1.0/3.0)+DX/ABS(DX)**ABS(DX)**(1.0/3.0)
FD=(1.0+GRAT**2.0)**(1.0+1.0/X)+X**(1.0+X)
WG=FAC**88.2*FD**SHP(2,NSTG)/PERF(4,IENGN)**ABS(FLOAT(NGT-NPUMP))

C
C GEAR 007
C GEAR 008
C GEAR 009
C GEAR 010
C GEAR 011
C GEAR 012
C GEAR 013
C GEAR 014
C GEAR 015
C GEAR 016
C GEAR 017
C GEAR 018
C GEAR 019
C GEAR 020
C GEAR 021
C GEAR 022
C GEAR 023
C GFAP 024
C GEAR 025
C GEAR 026
C GEAR 027
C GEAR 028

```



```

ETAP(7,NSTG)=0.98
GO TO 2
C
C SUBROUTINE FOR PLANETARY GEAR WEIGHT
C ASSUMED. K FACTOR = 500
C ALLOCATION FACTOR = 0.35
C
1)1 IF(GRAT.GT.1.5) GO TO 8
WG=0.0
ETAP(7,NSTG)=1.0
GO TO 2
8 IF(GRAT.GT.2.05) GO TO 9
GRAT=2.05
9 IF(GRAT.GT.4.0) GO TO 10
B=6.0
GO TO 11
10 B=PI/AR SIN((GRAT-2.0)/GRAT)
11 C=2.4*(GRAT-1.0)*(GRAT-1.0)
F=(C+1.0)/B*0.5
D=SQRT(.25*(.25/27.-E)**2-1./43856.)
DX=-(.25/27.-E)**0.5-D
X=(-(.25/27.-E)**.5+D)***(1./3.)+DX/ABS(DX)**ABS(DX)***(1./3.)
FD=1./B*(1.+1./X)**(1.+X*X*B+C)
WG=88.2*FD*SHF(2,NSTG)/PERF(4,IENGN)*FLOAT(NPUMP)
ETAP(7,NSTG)=0.98
GO TO 2
C
C DOUBLE REDUCTION DOUBLE BRANCH GEAR DESIGN
C GEAR BASED ON A K FACTOR OF 300
C
12 IF(NGT-NPUMP) 103,104,105
103 FAC=1.7*FLOAT(NPUMP-NGT)
GO TO 13
104 FAC=1.0*FLOAT(NPUMP)
GO TO 13
105 FAC=1.3*FLOAT(NGT-NPUMP)

```

```

GEAR 029
GEAR 030
GEAR 031
GEAR 032
GEAR 033
GEAR 034
GEAR 035
GEAR 036
GEAR 037
GEAR 038
GEAR 039
GEAR 040
GEAR 041
GEAR 042
GEAR 043
GEAR 044
GEAR 045
GEAR 046
GEAR 047
GEAR 048
GEAR 049
GEAR 050
GEAR 051
GEAR 052
GEAR 053
GEAR 054
GEAR 055
GEAR 056
GEAR 057
GEAR 058
GEAR 059
GEAR 060
GEAR 061
GEAR 062
GEAR 063
GEAR 064

```



```

13 A=GRAT/(GRAT+1.0)
   C=(GRAT*GRAT+1.0)**0.25
   D=SQRT((2.0**A**3/27.0-A*C)**2**0.25-A**6/729.0)
   E=-2.0**A**3/27.0+A*C
   DX=(E-D)**0.5*(1./3.)
   X=(E+D)**0.5*(1./3.)+DX/ABS(DX)**ABS(DX)**0.5*(1./3.)
   FD=.5*(1.+1./X+4.0**X**2.0**X**X*(1.+1./GRAT))+2.0*GRAT*GRAT/X+GRAT)
   WG=FAC**147.0*FD**SHP(2,NSTG)/PERF(4,IENGN)
   ETAP(7,NSTG)=0.93
2 CONTINUE
  RETURN
  END

```

```

GEAR 065
GEAR 066
GEAR 067
GEAR 068
GEAR 069
GEAR 070
GEAR 071
GEAR 072
GEAR 073
GEAR 074
GEAR 075
GEAR 076

```



```

SUBROUTINE FUEL
FUEL WEIGHT AT CONSTANT SPEED ASSUMING SFC = CSF*SHP TO 1/4 AND EFUEL
COMMON /CHARS/WGTS(2,15),CGS(4,15),DELH(5,15),CGSX,CGSZ
COMMON /H2O/TEMP,PV,RHOW,GNU,HA
COMMON /DRAG/TERAG(5),STRID(5),POD(5),SPRAY(5),REST(5),VO(5),
1 TRIM(5)
COMMON /SHIP/ DISP, RANGE, BEAM, HS, HE, HCL, XLS, XLPE, XLP
COMMON /CONST/PI,G,RHCD
COMMON /FLOW/G(5),AIN,AJET,AREA(11),VJ(5),VI(5)
COMMON /INDEX/IEVAL,IEQPT,ISTRT,NUMR,IFENGN,ITYPE,ICOMP,NPUMP,NGT,
AIGEAR
COMMON /PSUB/GERAT(5),SHP(5,5),RPM(5,5),PERF(5,12),ETAP(8,5)
COMMON /PUMM/QQ(5),DIS(5),D2S(5),XNS(5),SM(5,5),PLP(5),NSTG,SHI(5,
A5),XIM
COMMON /HEAD/HPP(5),HSV(5),THCM(5),PHI(5,5),WF,WG,FRATE(5)
DIMENSION WT(21),DIS(21),VJJ(21)
IF(ISTRT.EQ.3)GO TO 14

FUEL CALCULATIONS BASED ON CONSTANT SPEED THROUGHOUT RANGE. DURATI
TRAVEL DIVIDED INTO 20 SEGMENTS AND FUEL WEIGHT CALCULATED ON REVI
SHP REQUIPEMENTS DUE TO WEIGHT DECREASE

CFS=PERF(3,IFENGN)*PERF(1,IFENGN)**0.25
CA=TDRAG(1)/DISP
CD=1.0+(DELH(1,8)+DELH(1,7))*2.0*G/(VJ(1)*VJ(1))
WT(1)=0.0
DIS(1)=DISP
N=21
XN=20.
TI=RANGE/(VO(1)*XN)*1.689
IJK=0
XJ=0.75
DO1 I=2,N
DIS(I)=DIS(I-1)-WT(I-1)
VJJ(I)=VJ(1)/2.0+SORT(VO(1)**2/4.0+DIS(I)**CA/(RHOW*AJET))
H=CD*VJJ(I)*VJJ(I)/(2.*G)-VO(1)*VO(1)/(2.*G)+HE

```

```

FUEL 000
EFUEL 001
FUEL 002
FUEL 003
FUEL 004
FUEL 005
FUEL 006
FUEL 007
FUEL 008
FUEL 010
FUEL 014
FUEL 015
FUEL 016
FUEL 017
FUEL 018
FUEL 019
FUEL 020
FUEL 021
FUEL 022
FUEL 023
FUEL 024
FUEL 025
FUEL 026
FUEL 027
FUEL 028
FUEL 029
FUEL 030
FUEL 031
FUEL 032
FUEL 033

```

C

C
C
C
C
C

FUEL 034
 FUEL 035
 FUEL 036
 FUEL 037
 FUEL 038
 FUEL 039
 FUEL 040
 FUEL 041
 FUEL 042
 FUEL 043
 FUEL 044
 FUEL 045

```

SHPP=(RHOW*G*AJET*VJJ(I)*H)/(550)*ETAP(1,NSTG)*ETAP(7,NSTG)
SHNG=SHPP/FLOAT(NGT)
IF(SHNG.GT.0.7*PERF(1,IENGN).OR.IJK.EQ.1) GO TO 4
CFS=CFS*SQRT(SHNG)
IJK=1
XJ=0.25
4 WT(I)=CFS*TI*SHPP**XJ*FLOAT(NGT)**(1.-XJ)
1 CONTINUE
DO2 M=2,N
2 WT(M)=WT(M)+WT(M-1)
WF=WT(N)
RETURN
10 DO 11 I=1,ISTR1,NUMB
CFS=PERF(3,IENGN)*PERF(1,IENGN)**.25
ENN=.75
IF(SHP(I,NSTG).LT.(0.7*PERF(1,IENGN)))CFS=CFS*SQRT(SHP(I,NSTG))
IF(SHP(I,NSTG).LT.(.7*PERF(1,IENGN)))ENN=.25
FRATE(I)=CFS*FLOAT(NGT)*SHP(I,NSTG)**ENN
WF=WGTS(2,11)
11 CONTINUE
RETURN
END

```


SUBROUTINE NACEL
COMMON /WARN/CAV(5,6)
COMMON /H2O/TEMP,PV,RHOW,GNU,HA
COMMON /SHIP/DISP,RANGE,BEAM,HS,HE,HCL,XLS,XLPE,XLP
COMMON /CONST/PI,G,RHOD
COMMON /CHARS/WGTS(2,15),CGS(4,15),DELH(5,15),CGSX,CGSZ
COMMON /FLOW/Q(5),AIN,AJET,AREA(11),VJ(5),VI(5)
COMMON /INDEX/IFVAL,IEQPT,ISTRT,NUMB,IENGN,ITYPE,ICOMP,NPUMP,NGT,
AIGFAR
COMMON /DRAG/TDRAG(5),STRTD(5),POD(5),SPRAY(5),REST(5),VO(5),
1 TRIM(5)
COMMON /CDRAG/CSTRT(5),CPOC(5),CSPRY(5)
COMMON /TOLEK/DELTA
COMMON /NACLL/DRAT,DM,AI,AIAUX,ELEXT,ELENT,ELAUX,ELDIF,ELN,AAX(5)
COMMON /ELBW/XK(4),RO(4),THATA(4),WIDTH,DEPTH,TYPE(3,4)
DIMENSION TLIP(5)
DIMENSION NL(2),ML(2),KL(2),JL(2),IL(2)
DIMENSION AA(30),AB(30),AC(30),AD(30),AE(30),AF(30),AG(30),AH(30)
DIMENSION BA(30),BB(30),BC(30),BD(20),BE(30),BF(30),BG(30),BH(30)
DIMENSION ALFAT(4),VRT(5),XDT(10),CPINT(6,10,4),CPEXT(6,10,4),
1PRLT(6),SIGI(5),PTU(5),XDTT(4),CDUMX(10), CDUMY(10), AT(2),CD(5)
2,DIDMT(10),VRTT(4),DLIP(5),GO(5),VELP(5),VRTEX(4),VRMAX(5)
EQUIVALENCE (CPINT(1,1,1),AA(1)),(CPINT(1,6,1),AB(1)),(CPINT(1,1,2),
1),AC(1)),(CPINT(1,6,2),AD(1)),(CPINT(1,1,3),AE(1)),(CPINT(1,6,3),
2AF(1)),(CPINT(1,1,4),AG(1)),(CPINT(1,6,4),AH(1)),(CPEXT(1,1,1),BA(
31)),(CPEXT(1,6,1),BB(1)),(CPEXT(1,1,2),BC(1)),(CPEXT(1,6,2),BD(1))
4,(CPEXT(1,1,3),BF(1)),(CPEXT(1,6,3),BE(1)),(CPEXT(1,1,4),BG(1)),
5(CPEXT(1,6,4),BH(1))
EQUIVALENCE (AT(1),ALFAT(1)),(DRAT,DIDM)
DATA NL,ML,KL,JL/6, 2,2,2,10,3,6,3/IL/4,2/
DATA PRLT/0.973,0.969,C.962,0.945,0.928,C.909/
DATA ALFAT/0.,2.,4.,6.,8.,
1 XDT/0.25,0.50,0.75,1.0,1.25,1.50,1.75,2.0,2.25,2.5/,DIDMT/0.810,
2 0.695,0.627,0.577,0.533,0.490,0.439,0.365,0.260,0.090/
3, VRT/.,7.,8.,9.,1.,05,1.,15,1.,25/
DATA AA/

A	0.170	, -0.150	, -0.540	, -1.300	, -1.980	, -2.770	,	NACL0035
B	0.267	, 0.010	, -0.300	, -0.860	, -1.330	, -1.925	,	NACL0036
C	0.335	, 0.125	, -0.145	, -0.605	, -0.965	, -1.415	,	NACL0037
D	0.378	, 0.175	, -0.055	, -0.470	, -0.775	, -1.120	,	NACL0038
E	0.403	, 0.212	, -0.005	, -0.383	, -0.665	, -0.975	/	NACL0039
DATA AB/								
A	0.413	, 0.233	, 0.026	, -0.333	, -0.510	, -0.887	,	NACL0040
B	0.417	, 0.245	, 0.053	, -0.302	, -0.506	, -0.832	,	NACL0041
C	0.419	, 0.254	, 0.071	, -0.278	, -0.502	, -0.790	,	NACL0042
D	0.420	, 0.262	, 0.082	, -0.259	, -0.499	, -0.752	,	NACL0043
E	0.421	, 0.271	, 0.090	, -0.241	, -0.480	, -0.720	/	NACL0044
DATA AC/								
A	0.000	, -0.360	, -0.800	, -1.570	, -2.430	, -3.900	,	NACL0045
B	0.157	, -0.125	, -0.475	, -1.118	, -1.680	, -2.750	,	NACL0046
C	0.260	, 0.018	, -0.258	, -0.788	, -1.075	, -1.670	,	NACL0047
D	0.328	, 0.102	, -0.130	, -0.583	, -0.930	, -1.250	,	NACL0048
E	0.365	, 0.160	, -0.060	, -0.464	, -0.770	, -1.020	/	NACL0049
DATA AD/								
A	0.386	, 0.196	, -0.018	, -0.385	, -0.664	, -0.902	,	NACL0050
B	0.398	, 0.220	, 0.011	, -0.330	, -0.595	, -0.836	,	NACL0051
C	0.407	, 0.238	, 0.034	, -0.287	, -0.545	, -0.780	,	NACL0052
D	0.414	, 0.252	, 0.055	, -0.258	, -0.507	, -0.766	,	NACL0053
E	0.417	, 0.263	, 0.071	, -0.222	, -0.482	, -0.742	/	NACL0054
DATA AE/								
A	-0.215	, -0.590	, -1.140	, -2.150	, -3.150	, -4.450	,	NACL0055
B	0.020	, -0.295	, -0.700	, -1.465	, -2.185	, -3.240	,	NACL0056
C	0.160	, -0.105	, -0.408	, -0.980	, -1.450	, -2.000	,	NACL0057
D	0.255	, 0.010	, -0.255	, -0.710	, -1.070	, -1.455	,	NACL0058
E	0.310	, 0.088	, -0.145	, -0.554	, -0.860	, -1.180	/	NACL0059
DATA AF/								
A	0.345	, 0.152	, -0.075	, -0.450	, -0.725	, -1.024	,	NACL0060
B	0.367	, 0.200	, -0.030	, -0.383	, -0.643	, -0.930	,	NACL0061
C	0.386	, 0.225	, 0.012	, -0.328	, -0.589	, -0.880	,	NACL0062
D	0.401	, 0.243	, 0.048	, -0.280	, -0.541	, -0.850	,	NACL0063
E	0.412	, 0.258	, 0.060	, -0.245	, -0.509	, -0.820	/	NACL0064
DATA AG/								

A	-0.550	, -0.950	, -1.600	, -2.940	, -3.500	, -5.200	,	NACL0071
B	-0.250	, -0.570	, -1.020	, -1.900	, -2.650	, -3.920	,	NACL0072
C	0.000	, -0.275	, -0.600	, -1.230	, -1.770	, -2.385	,	NACL0073
D	0.150	, -0.045	, -0.340	, -0.950	, -1.365	, -1.960	,	NACL0074
E	0.255	, 0.020	, -0.200	, -0.800	, -1.215	, -1.800	/	NACL0075
DATA AH/								NACL0076
A	0.315	, 0.120	, -0.110	, -0.685	, -1.130	, -1.715	,	NACL0077
B	0.355	, 0.173	, -0.050	, -0.614	, -1.087	, -1.665	,	NACL0078
C	0.375	, 0.195	, 0.000	, -0.575	, -1.060	, -1.640	,	NACL0079
D	0.392	, 0.225	, 0.032	, -0.545	, -1.035	, -1.625	,	NACL0080
E	0.405	, 0.248	, 0.060	, -0.520	, -1.017	, -1.617	/	NACL0081
DATA RA/								NACL0082
A	-0.485	, -0.460	, -0.450	, -0.410	, -0.410	, -0.385	,	NACL0083
B	-0.335	, -0.330	, -0.315	, -0.300	, -0.295	, -0.280	,	NACL0084
C	-0.240	, -0.239	, -0.230	, -0.225	, -0.220	, -0.215	,	NACL0085
D	-0.175	, -0.135	, -0.170	, -0.170	, -0.165	, -0.164	,	NACL0086
E	-0.140	, -0.145	, -0.135	, -0.140	, -0.135	, -0.135	/	NACL0087
DATA RB/								NACL0088
A	-0.115	, -0.120	, -0.110	, -0.115	, -0.110	, -0.115	,	NACL0089
B	-0.100	, -0.105	, -0.100	, -0.100	, -0.100	, -0.100	,	NACL0090
C	-0.093	, -0.095	, -0.095	, -0.095	, -0.092	, -0.095	,	NACL0091
D	-0.090	, -0.090	, -0.090	, -0.090	, -0.090	, -0.090	,	NACL0092
E	-0.088	, -0.088	, -0.088	, -0.088	, -0.088	, -0.085	/	NACL0093
DATA RC/								NACL0094
A	-0.540	, -0.500	, -0.450	, -0.460	, -0.410	, -0.435	,	NACL0095
B	-0.365	, -0.350	, -0.335	, -0.320	, -0.305	, -0.305	,	NACL0096
C	-0.270	, -0.265	, -0.245	, -0.235	, -0.230	, -0.230	,	NACL0097
D	-0.205	, -0.205	, -0.185	, -0.175	, -0.175	, -0.180	,	NACL0098
E	-0.165	, -0.160	, -0.145	, -0.140	, -0.140	, -0.145	/	NACL0099
DATA RD/								NACL0100
A	-0.140	, -0.130	, -0.120	, -0.115	, -0.115	, -0.120	,	NACL0101
B	-0.125	, -0.110	, -0.105	, -0.100	, -0.100	, -0.105	,	NACL0102
C	-0.115	, -0.095	, -0.095	, -0.095	, -0.092	, -0.095	,	NACL0103
D	-0.110	, -0.090	, -0.090	, -0.090	, -0.090	, -0.090	,	NACL0104
E	-0.105	, -0.085	, -0.085	, -0.085	, -0.088	, -0.085	/	NACL0105
DATA BE/								NACL0106

A	-0.595	,	-0.535	,	-0.535	,	-0.460	,	-0.445	,	-0.445	,	NACL0107
B	-0.420	,	-0.390	,	-0.370	,	-0.350	,	-0.335	,	-0.330	,	NACL0108
C	-0.330	,	-0.310	,	-0.275	,	-0.260	,	-0.260	,	-0.260	,	NACL0109
D	-0.275	,	-0.265	,	-0.215	,	-0.205	,	-0.205	,	-0.195	,	NACL0110
E	-0.250	,	-0.230	,	-0.175	,	-0.165	,	-0.165	,	-0.160	/	NACL0111
DATA BF/													
A	-0.235	,	-0.205	,	-0.145	,	-0.145	,	-0.140	,	-0.138	,	NACL0112
B	-0.235	,	-0.185	,	-0.135	,	-0.130	,	-0.125	,	-0.125	,	NACL0114
C	-0.240	,	-0.170	,	-0.125	,	-0.120	,	-0.118	,	-0.118	,	NACL0115
D	-0.250	,	-0.155	,	-0.120	,	-0.115	,	-0.115	,	-0.113	,	NACL0116
E	-0.265	,	-0.145	,	-0.118	,	-0.112	,	-0.114	,	-0.110	/	NACL0117
DATA BG/													
A	-0.670	,	-0.610	,	-0.570	,	-0.525	,	-0.475	,	-0.480	,	NACL0118
B	-0.485	,	-0.460	,	-0.415	,	-0.380	,	-0.365	,	-0.360	,	NACL0119
C	-0.395	,	-0.373	,	-0.335	,	-0.293	,	-0.285	,	-0.280	,	NACL0120
D	-0.348	,	-0.322	,	-0.285	,	-0.235	,	-0.232	,	-0.225	,	NACL0121
E	-0.331	,	-0.300	,	-0.255	,	-0.200	,	-0.195	,	-0.190	/	NACL0122
DATA BH/													
A	-0.337	,	-0.295	,	-0.238	,	-0.176	,	-0.171	,	-0.168	,	NACL0123
B	-0.360	,	-0.310	,	-0.231	,	-0.163	,	-0.157	,	-0.154	,	NACL0124
C	-0.400	,	-0.310	,	-0.230	,	-0.157	,	-0.150	,	-0.145	,	NACL0125
D	-0.447	,	-0.323	,	-0.229	,	-0.155	,	-0.146	,	-0.139	,	NACL0126
E	-0.499	,	-0.336	,	-0.230	,	-0.153	,	-0.144	,	-0.135	/	NACL0127
FRICT(RE)=(.36859*ALOG(RE)/(1.964*ALOG(RE)-3.8215))*(-2)													

C ZK IS THE DECIMAL PART OF THE ANNULUS OCCUPIED BY THE
C AUXILIARY INLET THAT IS ACTUALLY OPENING. THE REMAINDER IS
C STRUCTURE.

ZK=.8
SPO=(HS+HA)*RHOW*G
PVP=PV*RHOW*G

C SIGTV IS THE INCIPIENT CAVITATION NO. ON THE ELBOW TURNING
C VANES REFERENCED TO DIFFUSER EXIT PRESSURE AND VELOCITY.

NACLO144
NACLO145
NACLO146
NACLO147

NACLO148
NACLO149
NACLO150
NACLO151
NACLO152
NACLO153
NACLO154

NACLO155
NACLO156
NACLO158
NACLO159
NACLO160
NACLO161

NACLO162
NACLO163
NACLO164
NACLO165
NACLO166
NACLO167
NACLO168
NACLO169
NACLO170
NACLO171
NACLO172
NACLO173

```
SIGTV=0.4  
JNUMB=2  
IF(ISTRT.EQ.3)JNUMB=NUMB  
DO 10 I=ISTRT,JNUMB  
VELR(I)=VI(I)/VG(I)  
QQ(I)=.5*RHOW*VD(I)*VO(I)  
  
C SIGI(I) IS THE INCIPIENT CAVITATION NC. REFERENCED TO FREE STREAM  
C CONDITIONS.  
C  
C SIGI(I)=(SPO-PVP)/QQ(I)  
C PTO(I)=SPO+QQ(I)  
C IF(ISTRT.EQ.1) GO TO.10  
C IF(TRIM(I).GT.3) TRIM(I)=3.  
C  
C 10 CONTINUE  
C IF(TRIM(I).GT.3.)TRIM(I)=3.  
  
C INTERPOLATE IN THE DATA TABLE TO FIND THE INLET WITH THE DESIRED  
C PRESSURE COEFFICIENT.  
C  
C KNUMB=NUMB  
C IF(ISTPT.EQ.1) KNUMB=ISTRT  
C DO 211 IF=ISTRT,KNUMB  
C CPEX=-SIGI(IE)  
C DO 610 I=1,2  
C DO 609 K=1,10  
C DO 608 J=1,6  
C CDUMX(J)=CPEXT(J,K,I)  
C  
C 608 CONTINUE  
C CDUMY(K)=TABLE(VRT,CDUMX,VELR(1),NL)  
C 609 CONTINUE  
C XDIT(I)=TABLE(CDUMY,XDT,CPEX,KL)  
C 610 CONTINUE  
C ML(I)=2  
C XD=TABLE(AT,XDIT,TRIM(1),ML)  
C DIDMX=TABLE(XDT,DIDMT,XD,KL)
```



```

IF(DIDMX.LT.DIDM) CAV(IE,I)=1.
IF(ISTRT.EQ.3) GO TO 201
WGTS(1,1)=1.59
C
C
C
C
C
C
IF THE TRIAL NACELLE HAS LESS FRONTAL AREA THAN THE MINIMUM
REQUIRED TO AVOID CAVITATION, REJECT THE TRIAL VALUE. IF NON-
CAVITATING, CALCULATE INLET DIMENSIONS.
IF(DIDMX.LT.DIDM*(1.+DELTA)) CIDM=DIDMX
QI=.5*Q(1)
AI=OI/VI(1)
DI=SQRT(AI)*1.12838
11 CONTINUE
DM=DI/DIDM
XD=TABLE(DIDMT,XDT,DIDM,KL)
EEXT=DM**XD
ASSUMING THE AUX. INLETS ALLOW FLOW TO ENTER BEFORE THE DIFFUSER,
CALCULATE LOSSES AND TOTAL PRESSURE OF THE COMBINED FLOW.
2J1 IF(ISTRT.EQ.1) CPIN=-SIGI(2)
C
C
C
C
C
C
INTERPOLATE IN THE DATA TABLE TO DETERMINE THE MAXIMUM VELOCITY
RATIOS AT CRUISE AND TAKE-OFF.
DO 710 I=1,4
DO 709 K=1,6
DO 708 J=1,10
CDUMX(J)=CPINT(K,J,I)
708 CONTINUE
CDUMY(K)=TABLE(XDT,CDUMX,XD,KL)
709 CONTINUE
IF(ISTRT.EQ.1) VRTT(I)=TABLE(CDUMY,VRT,CPIN,IL)
VRTEX(I)=TABLE(CDUMY,VRT,CPEX,IL)
710 CONTINUE
ML(1)=4
VRMAX(IE)=TABLE(ALFAT,VRTEX,TRIM(IE),ML)
IF(ISTRT.EQ.1) VRMAX(2)=TABLE(ALFAT,VRTT,TRIM(2),ML)
NACL0174
NACL0175
NACL0176
NACL0177
NACL0178
NACL0179
NACL0180
NACL0181
NACL0182
NACL0183
NACL0184
NACL0185
NACL0186
NACL0187
NACL0188
NACL0189
NACL0191
NACL0192
NACL0193
NACL0194
NACL0195
NACL0196
NACL0197
NACL0198
NACL0199
NACL0200
NACL0201
NACL0203
NACL0204
NACL0205

```


NACLO208
NACLO209
NACLO210

C CHECK FOR LIP CAVITATION. IF CAVITATING, RETURN
C IF(VELR(IE).GT.VRMAX(IE).AND.DELTA.GT.1.E-9) RETURN
C CAV(IE,2)=0.
C IF(VELR(IE).GT.VRMAX(IE)) CAV(IE,2)=2.

211 CONTINUE

IF(ISTR.EC.3) GO TO 203

C DETERMINE MAX. FLOW RATE AT TAKE-OFF AND COMPARE WITH REQUIRED
C FLOW RATE. AN AUXILIARY INLET MUST BE SIZED TO ACCEPT ANY EXCESS
C REQUIRED FLOW.

QIN=AI+VRMAX(2)*VO(2)

QC=0.5*Q(2)

QAUX=QC-QIN

IF(QAUX.LE.0.)QAUX=0.

QIN=QC-QAUX

C CALCULATE STATIC PRESSURE IMMEDIATELY AFT. OF THE LIP.

VI2=QIN/AI

VR=VI2/VO(2)

PRL2=TABLE(VRT,PRLT,VR,JL)

PTI=PRL2*CO(2)+SPC

SPI=PTI-0.5*RHOW*VI2*VI2

C CALCULATE COMBINED FLOW PRESSURES AND AUX. INLET AREA.

AIAUX=0.

VIAUX=0.

PTAUX=0.

IF(QAUX.EQ.0.)GO TO 12

PRAUX=0.90

PTAUX=PRAUX*QN(2)+SPC

DYP=PTAUX-SPI

VIAUX=SQRT(2.*DYP/RHOW)

AIAUX=QAUX/VIAUX

12 CONTINUE

C THE TOTAL PRESSURE OF THE COMBINED FLOW IS CALCULATED AS THE

NACLO213
NACLO214
NACLO215
NACLO216
NACLO217
NACLO218
NACLO219
NACLO220
NACLO221
NACLO222
NACLO223
NACLO224
NACLO225
NACLO226
NACLO227
NACLO228
NACLO229
NACLO230
NACLO231
NACLO232
NACLO233
NACLO234
NACLO235
NACLO236
NACLO237
NACLO238
NACLO239
NACLO240

NACL0241
 NACL0242
 NACL0243
 NACL0244
 NACL0245
 NACL0246
 NACL0247
 NACL0248
 NACL0249
 NACL0250
 NACL0251
 NACL0252
 NACL0253
 NACL0254
 NACL0255
 NACL0256
 NACL0257
 NACL0258
 NACL0259
 NACL0260
 NACL0261
 NACL0262
 NACL0263
 NACL0264
 NACL0265
 NACL0266
 NACL0267
 NACL0268
 NACL0269
 NACL0270
 NACL0271
 NACL0272
 NACL0273
 NACL0274
 NACL0275
 NACL0276

```

C MASS WEIGHTED AVERAGE OF THE COMBINING FLOWS.
C
PC=(PTI*QIN+PTAUX+QAUX)/QC
VI(2)=QC/AI
DLIP(2)=1.-(PC-SPO)/QO(2)
QDIF=QC

C CALCULATE THE INTERNAL LENGTHS TO THE DIFFUSER ENTRY.
C
ELENT=ELEXT/9.
PHI=ATAN(.5*(DM-DI)/ELEXT)
PHS=SIN(PHI)
X=.5*(SQRT(DI**2+1.27324*AI*AU*PHS/ZK)-DI)/PHS
ELAUX=X/COS(PHI)
SIZE THE DIFFUSER
D2=0.9*DM
D1=DI
IF(D2.LT.D1)D2=D1
ELMAX=9.22339*(D2-D1)
ELMIN=2.836075*(D2-D1)
C DECIDE WHICH CONDITION GOVERNS THE DIFFUSER.
IF(QI.GT.QC)GO TO 13
II=2
GO TO 14
13 CONTINUE
II=1
QDIF=QI
14 CONTINUE
EL=0.45*(ELMAX+ELMIN)
ELD=ELENT+EL+3.54491*D2+ELAUX
ELL=5.5*DM
ELN=ELD
IF(ELL.GE.ELD)ELN=ELL
DOM=0.5*(D2+D1)
XKT=(1.-(D1/D2)**2)**2
ANGL=ATAN((C2-D1)/(2.*EL**2))**57.2958

```


NACL0277
 NACL0278
 NACL0279
 NACL0280
 NACL0281
 NACL0282
 NACL0283
 NACL0284
 NACL0285
 NACL0286

NACL0288
 NACL0289
 NACL0290
 NACL0291

C CDIF=3.19E-3*ANGL*ANGL+8.452E-4*ANGL
 C ELDIF IS THE DIFFUSER LENGTH REQUIRING THE LEAST TOTAL POWER
 C FOR THE DESIRED DIFFUSION RATIO.
 C

C ELDIF=EL
 C
 C CALCULATE THE LIP LOSSES FOR EACH SITUATION.
 C

203 DLIP(1)=1.--TABLE(VRT,PRLT,VELR(1),JL)
 IF(NUMB.LT.3) GO TO 113
 DO 15 J=3,NUMB
 VI(J)=.5*Q(J)/AI
 VELR(J)=VI(J)/VC(J)
 DLIP(J)=1.--TABLE(VRT,PRLT,VELR(J),JL)
 IF(CAV(J,2).EQ.0.) GO TO 15
 AAX(J)=0.
 VIAUX=0.
 PTAUX=0.
 QIN=AI*VRMAX(J)*VO(J)
 QC=0.5*Q(J)
 QAUX=QC-QIN
 IF(QAUX.LE.0.) GO TO 15
 VIJ=QIN/AI
 VR=VIJ/VO(J)
 PRLJ=TABLE(VRT,PRLT,VR,JL)
 PTI=PRLJ*QC(J)+SPC
 SPI=PTI-0.5*RHOW*VIJ**2
 PRAUX=0.8
 PTAUX=PRAUX*QC(J)+SPC
 DYP=PTAUX-SPI
 VIAUX=SQRT(2.*DYP/RHOW)
 AAX(J)=QAUX/VIAUX
 PC=(PTI*QIN+PTAUX*QAUX)/QC
 TLIP(J)=1.--(PC-SPC)/QC(J)
 IF(AAX(J).GT.AIAUX) CAV(J,2)=1.


```

IF(CAV(J,2).NE.1.) DLIP(J)=TLIP(J)
15 CONTINUE
113 JNUMB=2
IF(ISTR.NE.3) GO TO 251
JNUMB=NUMB
D1=DI
D2=SQRT(AREA(1)/(PI*.5))
DDM=0.5*(D2+D1)
XKT=(1.-(D1/D2)**2)**2
ANGL=ATAN((D2-D1)/(2.*ELDIF))*57.2958
CDIF=3.19F-3*ANGL*ANGL+8.452E-4*ANGL
251 CONTINUE
DO 17 I=ISTR,JNUMB
C
C CALCULATE THE DIFFUSER AND PIPE LOSSES FOR EACH SITUATION AND
C ADD TO THE LIP LOSSES.
C
REND=D1*VI(I)/GNU
DDIF=(CDIF*XKT+FRICT(REND)*ELDIF/DDM)*0.5*RHOW*VI(I)*VI(I)
PLOSS=DLIP(I)*QQ(I)+DDIF+FRICT(REND)*ELAUX/DI*.5*RHOW*VI(I)*VI(I)
IF(I.EQ.2) PLOSS=DLIP(2)*QQ(2)+DDIF
IF(CAV(I,2).EQ.2.) PLCSS=DLIP(I)*QQ(I)+DDIF
VAOUT=Q(I)*0.63662/(D2*D2)
SQUAR=SIGI(I)+1.-PLOSS/QQ(I)
C
C DETERMINE THE CRITICAL LOCAL VELOCITY AT THE DIFFUSER EXIT AT
C WHICH CAVITATION ON THE TURNING VANES OCCURS.
C
VCRIT=SQRT(SQUAR)*VG(I)/SQRT(1.+SIGTV)
C
C ESTIMATE THE MAXIMUM LOCAL VELOCITY AT THE DIFFUSER EXIT.
C
VMAX=1.5)*VAOUT
C
C IF CAVITATION OCCURS, REJECT CN DESIGN, INDICATE ON EVALUATION.
C
C

```

NACL0292
NACL0293

NACL0295
NACL0297
NACL0298
NACL0299
NACL0300
NACL0301
NACL0302
NACL0304
NACL0305
NACL0306
NACL0307
NACL0308
NACL0309
NACL0310
NACL0311
NACL0312
NACL0313
NACL0314
NACL0315
NACL0316
NACL0317
NACL0318


```

IF(VMAX.GT.VCRIT)CAV(I,3)=1.
IF(VMAX.GT.VCRIT.AND.ISTRT.EQ.1.AND.DELTA.GT.1.E-9) RETURN
C
C
C
C
AT THIS POINT THE DIFFUSER HAS BEEN SIZED TO AVOID CAVITATION AT
BOTH TAKE/OFF AND CRUISE. INTERNAL FLOW LOSSES ARE DETERMINED
C
DELH(I,1)=PLCSS/(RHOW*G)
RENL=ELN*VC(I)/GNU
C
C
C
CALCULATE THE DRAG COEFFICIENTS.
C
C
C
CD(I)=CFS(RENL)*(1.+1.5*(DM/ELN)**(3/2)+7.*(DM/ELN)**3)
C
CALCULATE WETTED SURFACE AND DRAG.
C
C
C
EM=SQRT(1.+4.*(2.*ELEXT)/DM)**2)
AEXN=1.0472*DM*DM*(EM+1./(EM+1.))+PI*DM*(ELN-2.*ELEXT)
CPOD(I)=2.*QC(I)*AEXN*CD(I)
C
17 CONTINUE
AREA(1)=PI*D2*D2*0.5
CGS(1,1)=HS+HE-HCL
CGS(2,1)=XLS+.5*(ELN-3.54491*02)
IF(THATA(1).GE.90.)GC TO 18
CGS(2,1)=CGS(2,1)+CGS(1,1)/TAN(THATA(1))*0.0174533)
18 CGS(3,1)=0.
CGS(4,1)=0.
WGTS(1,1)=.11*DM*AEXN*(.5*PHOD-RHOW)+15.07*AREA(1)*(ELENT+ELAUX+
A EL)
RETURN
END
NACLO320
NACLO321
NACLO322
NACLO323
NACLO324
NACLO325
NACLO326
NACLO327
NACLO328
NACLO329
NACLO330
NACLO331
NACLO332
NACLO333
NACLO334
NACLO335
NACLO336
NACLO337
NACLO338
NACLO339
NACLO340
NACLO341
NACLO342
NACLO343
NACLO344
NACLO345
NACLO346
NACLO347
NACLO348

```



```

C .....
C DIMENSION PSI(1), THETA(9), PHI(9), DEL(1), DLMIN(1), DIR(9), SAVE(9)
C DATA RH7/.5/
C
C EVALUATE THE FUNCTION AT THE INITIAL PCINT
C
C SPSI=FCT(PSI)
C
C SET THE BASEPOINT
C
C 1 S=SPSI
C DO 10 I=1,N
C 10 PHI(I)=PSI(I)
C
C ICALL=1 INDICATES THE BASEPCINT HAS JUST BEEN UPDATED
C
C ICALL=1
C
C MAKE EXPLORATORY MOVES FROM THE BASEPOINT
C
C GO TO 99
C
C STORE PREVIOUS POINTS
C
C 2 SPSI=S
C DO 11 I=1,N
C THETA(I)=PSI(I)
C PHI(I)=PHI(I)
C
C MAKE PATTERN MOVE (I.E. SIMULTANEOUSLY MOVE THE DISTANCE FROM THE
C BASEPOINT TO THE PRESENT PCINT IN EACH COORDINATE)
C
C 11 PHI(I)=2.*PHI(I)-THETA(I)
C SPSI=FCT(PHI)
C S=SPHI
C
PTTR0037
PTTR0038
PTTR0039
PTTR0040
PTTR0041
PTTR0042
PTTR0043
PTTR0044
PTTR0045
PTTR0046
PTTR0047
PTTR0048
PTTR0049
PTTR0050
PTTR0051
PTTR0052
PTTR0053
PTTR0054
PTTR0055
PTTR0056
PTTR0057
PTTR0058
PTTR0059
PTTR0060
PTTR0061
PTTR0062
PTTR0063
PTTR0064
PTTR0065
PTTR0066
PTTR0067
PTTR0068
PTTR0069
PTTR0070
PTTR0071
PTTR0072

```



```

C          ICALL=2 INDICATES PATTERN MOVE JUST MADE
C          ICALL=2
C          MAKE EXPLORATORY MOVES FROM RESULTING POINT OF PATTERN MOVE
C          GO TO 99
C          DECREMENT STEP SIZE BY A FACTOR OF RHO
C          3 NUMB=0
C          DO 31 I=1,N
C          IF(DEL(I).GT.DLMIN(I)) GO TO 31
C          NUMB=NUMB+1
C          31 DEL(I)=RHO*DEL(I)
C          IF ALL STEP SIZES ARE LESS THAN MINIMUM, RETURN TO CALLING PROGRAM
C          OTHERWISE, START CFF FROM BASEPOINT WITH SMALLER STEP
C          IF(NUMB.FQ.N) RETURN
C          GO TO 1
C          MAKE EXPLORATORY MOVES
C          99 DO 95 K=1,N
C          SAVE(K)=PHI(K)
C          SIGN=DEL(K)
C          PHI(K)=SAVE(K)+SIGN
C          SPHI=FACT(PHI)
C          TSPHI=SPHI
C          PHI(K)=SAVE(K)-SIGN
C          SPHI=FACT(PHI)
C          IF(TSPHI.GT.SPHI) GO TO 98
C          SPHI=TSPHI
C          PHI(K)=SAVE(K)+SIGN
PTTR0073
PTTR0074
PTTR0075
PTTR0076
PTTR0077
PTTR0078
PTTR0079
PTTR0080
PTTR0081
PTTR0082
PTTR0083
PTTR0084
PTTR0085
PTTR0086
PTTR0087
PTTR0088
PTTR0089
PTTR0090
PTTR0091
PTTR0092
PTTR0093
PTTR0094
PTTR0095
PTTR0096
PTTR0097
PTTR0098
PTTR0099
PTTR0100
PTTR0101
PTTR0102
PTTR0103
PTTR0104
PTTR0105
PTTR0106
PTTR0107
PTTR0108

```


PTTR0109
 PTTR0110
 PTTR0111
 PTTR0112
 PTTR0113
 PTTR0114
 PTTR0115
 PTTR0116
 PTTR0117
 PTTR0118
 PTTR0119
 PTTR0120
 PTTR0121
 PTTR0122
 PTTR0123

```

98 IF(SPHI.LT.S) GO TO 96
   PHI(K)=SAVE(K)
   GO TO 95
96 S=SPHI
95 CONTINUE

C
C IF A EXPLORATORY MOVE IS SUCCESSFUL TRY A PATTERN MOVE IN THAT
C DIRECTION. IF ALL EXPLORATORY MOVES ARE UNSUCCESSFUL, DECREASE
C STEPSIZE AND RESET BASEPOINT
C
C FOR NORMAL USE OF PTTRN, THE FOLLOWING STATEMENT SHOULD READ
C IF(S.GT.SPSI) GO TO (3,1),ICALL
C IF(S.GT.SPSI-100.) GO TO (3,1),ICALL
   GO TO 2
   END
  
```


SUBROUTINE OUTPUT
INTEGER ENGN(3,12)
LOGICAL IFUEL, IPUMP, TYGER
COMMON /PINP/ALPHA
COMMON /WARN/CAV(5,6)
COMMON /DRAG/IDRAG(5), STRTD(5), POD(5), SPRAY(5), REST(5), VO(5),
1 TRIM(5)
COMMON /CONST/ PI, G, PHOD
COMMON /FLOW/Q(5), AIN, AJET, AREA(11), VJ(5), VI(5)
COMMON /SHIP/ DISP, RANGE, REAM, HS, HE, HCL, XLS, XLPE, XLP
COMMON /HPQ/TFVP, PV, RHCW, GNU, HA
COMMON /CHARS/WGTS(2,15), CGS(4,15), DELH(5,15), CGSX, CGSZ
COMMON /STRIC/IC, T, C, TL, CL, CFM
COMMON /PUMM/PO(5), DIS(5), D2S(5), XNS(5), SM(5,5), PLP(5), NSTG, SHI(5,
A5), XIM
COMMON /PSUB/GERAT(5), SHP(5,5), RPM(5,5), PERF(5,12), ETAP(8,5)
COMMON /HEAD/HPP(5), HSV(5), THCM(5), PHI(5,5), WF, WG, FRATE(5)
COMMON /ELBW/XK(+), KO(4), THATA(4), WIDTF, DEPTH, TYPE(3,4)
COMMON /NACLL/DRAT, DM, AI, AIAUX, ELEFT, ELAUX, ELCIF, ELN, AAX(5)
COMMON /INDEX/IFVAL, IEQPT, ISTART, NUMB, IFENGN, ITYPE, ICCMP, NPUMP, NGT,
AIGEAR
COMMON IFUEL, IPUMP, TYGER
DIMENSION VK(5)
DIMENSION VJRAT(5), VIRAT(5), HEADL(5,15), CCNDS(2,5), ELBWS(2,4), OUTPT019
APC(5), LABEL(5,14), LOCCAT(6,6)
DATA COND5/4H CRU, 4HISE, 4H T/C, 4H , 4H EVA, 4HL 1, 4H EVA, 4HL 2
A, 4H EVA, 4HL 3 /
DATA ELRWS/4H STR, 4HUT, 4H HUL, 4HL , 4H PUM, 4HP , 4H DIV, 4HERGEOUTPT022
A /
DATA HEADL/75*0. /
DATA LABEL/4HNACE, 4HLE, 3*4H , 4HSTRU, 4HT EL, 4HBOW, 2*4H ,
A 4HSTRU, 4HT DI, 4HFFUS, 4HER, 4H , 4HHULL, 4H ELB, 4HOW, 2*4H ,
B 4HATHW, 4HARTS, 4HHIPS, 4H LEN, 4HGTH, 4HPUMP, 4H ELB, 4HOW, 2*4H ,
C 4HFORE, 4H AND, 4H AFT, 4H LEN, 4HGTH, 4HPUMP, 4*4H , 4HNOZZ, 4HLE ,
D 3*4H , 4HREDU, 4HCTIO, 4H GE, 4HAP, 4H , 4HFUEL, 4*4H ,
E 5*4H , 4HJET, 4HLIFT, 3*4H , 4HTCTA, 4HLS, 3*4H /


```

DATA ENGN/4HTF35,2*4H ,4HTF40,2*4H ,4HPROT,4HEUS,4H1500, OUTPT031
A 4HPROT,4HEUS,4H1000,4HTYNE,4H1A,4H ,4HTYNE,4H1C,4H ,OUTPT032
B 4HFT12,4HA,4H ,4HLM15,4H00,4H ,4HLM25,4H00,4H ,OUTPT033
C 4HFT4A,4H-2C,4H ,4HFT4A,4H-12,4H ,4HFT4C,4H-2,4H / OUTPT034
DATA LOCAT/4H NAC,4HELLE,4H CUT,4HSIDE,4H ,4H ,4H NAC,4HELLE
A,4H INL,4HET L,4HIP,4H ,4H NAC,4HELLE,4H TUR,4HNING,4H VAN,4H
RES,4H STR,4HUT O,4HUTSI,4HDE,4H ,4H ,4H PUM,4HP IN,4HLET
C,4H ,4H /
I PRNT=6
WRITE(IPRNT,1)
1 FORMAT(5X,4H ***: WATERJET PROPULSION SYSTEM OUTPUT DATA ***: ///)
A )
DO 7 J=1,NUMB
VJRAT(J)=VJ(J)/VC(J)
VIRAT(J)=VI(J)/VC(J)
HEADL(J,1)=DELH(J,1)
HFADL(J,2)=DELH(J,2)-DELH(J,1)
HEADL(J,3)=DELH(J,3)-HE+RC(2)*XK(2)-DELH(J,2)
HEADL(J,4)=DELH(J,4)-PO(2)*XK(2)-DELH(J,3)
SUM=)
DO 8 I=5,8
HEADL(J,I)=DELH(J,I)-DELH(J,I-1)
IF(I.EQ.8) HEADL(J,8)=DELH(J,8)
8 SUM=SUM+HEADL(J,I)
7 HEADL(J,9)=SUM+HEADL(J,1)+HEADL(J,2)+HEADL(J,3)+HEADL(J,4)
DRATO=AREA(3)/AREA(2)
WRITE(IPRNT,3) AIN,DRATO,AJET
3 FORMAT(27H INLET AREA, TOTAL, FEET**2,F8.2,/,
A 27H STRUT DIFFUSER AREA RATIO,F8.2,/,
B 25H JET AREA, TCTAL, FEET**2,2X,F8.2,/)
IK=NPUMP+2-NPUMP/2
XNGT=NGT
DO 3) K=1,ISTR,NUMP
30 PC(K)=TDRAG(K)*VO(K)/(550.*SHP(K,NSTG)*XNGT)
WRITE(IPRNT,12)
12 FORMAT(24X,3HJET,15X,5HINLET)

```



```

GO TO (31,32,33,34,35,36),KLL
31 WRITE(IPRNT,25) (LABEL(M,K),M=1,5),(WGTS(I,K),I=1,2),(HEADL(J,LL),
A J=I,NUMB)
25 FORMAT(1X,5A4,5X,2F10.1,2F8.2)
GO TO 5
32 WRITE(IPRNT,37) (LABEL(M,K),M=1,5),(WGTS(I,K),I=1,2),(HEADL(J,LL),
A J=I,NUMB)
37 FORMAT(1X,5A4,5X,2F10.1,2F8.2,8X,F8.2)
GO TO 5
33 WRITE(IPRNT,38) (LABEL(M,K),M=1,5),(WGTS(I,K),I=1,2),(HEADL(J,LL),
A J=I,NUMB)
38 FCRMAT(1X,5A4,5X,2F1).1,2F8.2,4X,2F8.2)
GO TO 5
34 IF(ISTRT.EQ.1) GO TO 40
WRITE(IPRNT,39) (LABEL(M,K),M=1,5),(WGTS(I,K),I=1,2),(HEADL(J,LL),
A J=I,NUMB)
39 FORMAT(1X,5A4,5X,2F1).1,24X,F8.2)
GO TO 5
40 WRITE(IPRNT,41) (LABEL(M,K),M=1,5),(WGTS(I,K),I=1,2),(HEADL(J,LL),
A J=I,NUMB)
41 FORMAT(1X,5A4,5X,2F10.1,5F8.2)
GO TO 5
35 WRITE(IPRNT,42) (LABEL(M,K),M=1,5),(WGTS(I,K),I=1,2),(HEADL(J,LL),
A J=I,NUMB)
42 FORMAT(1X,5A4,5X,2F10.1,20X,2F8.2)
GO TO 5
36 WRITE(IPRNT,43) (LABEL(M,K),M=1,5),(WGTS(I,K),I=1,2),(HEADL(J,LL),
A J=I,NUMB)
43 FORMAT(1X,5A4,5X,2F10.1,16X,3F8.2)
GO TO 5
9 IF(K.EQ.9) LL=8
IF(K.EQ.14) LL=9
IF(K.EQ.14) WRITE(IPRNT,26)
26 FORMAT( 26X,60(1H-))
IF(LL.NE.K) GO TO 4
WRITE(IPRNT,25) (LABEL(M,K),M=1,5),(WGTS(I,K),I=1,2)

```



```

5 CONTINUE
  IF(ISTRT.NE.3)GO TO 100
  DO 101 I=ISTRT,NUMB
    VK(I)=VO(I)*0.5921
  DO 102 J=1,6
    IF(CAV(I,J).EQ.1.)WRITE(IPRNT,200) VK(I),(LOCAT(K,J),K=1,6)
  200 FORMAT(/,39H *** WARNING CAVITATION OCCURRING AT ,F5.1,6H KNOTS,
    A/,23H AT FOLLOWING LOCATION:,5X,6A4,/)
102 CONTINUE
101 CONTINUE
100 CONTINUE
  WRITE(IPRNT,6)
  IF(NSTG.EQ.4) NIMP=XIM
  IF(NSTG.NE.4) NIMP=NSTG-1
  WRITE(IPRNT,21)
  21 FORMAT(15X,9HPUMP DATA,/)
  IF(NSTG.EQ.4) GO TO 48
  WRITE(IPRNT,45) NIMP
  45 FORMAT(38H AXIAL PUMP WITH INDUCER IMPELLER AND ,I2,17H ADDITIONAL
    A STAGE,/)
  GO TO 49
  48 WRITE(IPRNT,50) NIMP
  50 FORMAT(23H CENTRIFUGAL PUMP WITH ,I2,34H DCUBLE SUCTION IMPELLERS
    APER PUMP,/)
  49 WRITE(IPRNT,51)
  51 FORMAT(5X,4HHEAD,4X,4HNPSH,2X,5HTHOMA,5X,3HRPM,3X,10HEFFICIENCY)
  WRITE(IPRNT,52) (HPP(I),HSV(I),THOM(I),RPM(I,NSTG),ETAP(I,NSTG),
    A (CONDS(J,I),J=1,2),I=ISTRT,NUMB)
  52 FORMAT(4X,F6.1,2X,F5.1,2X,F5.3,3X,F6.1,4X,F5.3,5X,2A4)
  IF(ISTRT.EQ.3)GO TO 201
  WRITE(IPRNT,53) XNS(NSTG),SM(2,NSTG),PHI(2,NSTG),SHI(2,NSTG),DIS(N
    ASTG),O2S(NSTG),XLP
  53 FORMAT(/,19H SPECIFIC SPEED,CFS,16(1H.),F7.1,/,
    A 27H SUCTION SPECIFIC SPEED,CFS,8(1H.),F7.1,/,
    B 17H FLOW COEFFICIENT,13(1H.),F7.3,/,17H HEAD COEFFICIENT,18(1H.),OUTPT160
    C F7.3,/,24H INLET TIP DIAMETER,FEET,11(1H.),F7.2,/,

```

OUTPT139

OUTPT141
 OUTPT142
 OUTPT143
 OUTPT144
 OUTPT145
 OUTPT146
 OUTPT147
 OUTPT148
 OUTPT149
 OUTPT150
 OUTPT151
 OUTPT152
 OUTPT153
 OUTPT154
 OUTPT155

OUTPT158
 OUTPT159
 OUTPT160
 OUTPT161

OUTPUT162
OUTPUT163
OUTPUT164
OUTPUT165

```
D 23H EXIT TIP DIAMETER,FEET,12(1H.),F7.2,/,  
F 17H PUMP LENGTH,FEET,18(1H.),F7.2)  
WRITE(IPRNT,57) GERAT(NSTG)  
57 FORMAT(11H GEAR RATIO,24(1H.),F7.2,///  
GO TO 208  
201 WRITE(IPRNT,202)  
202 FORMAT(/,15X,15HPUMP PARAMETERS,/) )  
WRITE(IPRNT,203)XNS(NSTG),DLS(NSTG),D2S(NSTG),XLP  
203 FORMAT(/,19H SPECIFIC SPEED,CFS,16(1H.),F7.1,/,24H INLET TIP DIAME  
ATER,FEET,11(1H.),F7.2,/,23H EXIT TIP DIAMETER,FEET,12(1H.),F7.2,/,  
R17H PUMP LENGTH,FEET,18(1H.),F7.2)  
WRITE(IPRNT,204)  
204 FORMAT(/,50H PUMP OPERATING PARAMETERS AT EVALUATED CONDITIONS,/) )  
WRITE(IPRNT,205)(SM(I,NSTG),I=1,ISTR,NUMB)  
205 FORMAT(/,27H SUCTION SPECIFIC SPEED,CFS,8(1H.),3F10.1)  
WRITE(IPRNT,206)(PHI(I,NSTG),I=1,ISTR,NUMB)  
206 FORMAT(/,17H FLOW COEFFICIENT,18(1H.),3F1.3)  
WRITE(IPRNT,207)(SHI(I,NSTG),I=1,ISTR,NUMB)  
207 FORMAT(/,17H HEAD COEFFICIENT,18(1H.),3F10.3)  
208 WRITE(IPRNT,24)  
24 FORMAT(15X,12HNACELLE DATA,/) )  
WRITE(IPRNT,54) DRAT,DM,AI,AIAUX,ELEXT,ELENT  
54 FORMAT(21H DIAMETER RATIO,DI/DM,19(1H.),F6.3,/,  
A 25H MAXIMUM DIAMETER,DM,FEET,15(1H.),F6.2,/,  
R 29H INLET AREA PER NACELLE,FT**2,11(1H.),F6.2,/,  
C 39H AUXILIARY INLET AREA PER NACELLE,FT**2,F7.2,/,  
D 21H FOREBODY LENGTH,FEET,19(1H.),F6.2,/,  
E 16H LIP LENGTH,FEET,24(1H.),F6.2)  
WRITE(IPRNT,55) ELDIF,EUN  
55 FORMAT(21H DIFFUSER LENGTH,FEET,19(1H.),F6.2,/,  
A 20H NACELLE LENGTH,FEET,20(1H.),F6.2,///  
TFM=TC*CFM  
WRITE(IPRNT,47) TC,T,C,T1,C1,TFM,CFM  
47 FORMAT(10X,19HSTRUT CONFIGURATION,///  
A 3X,3HT/C,5X,9HTTHICKNESS,5X,5HCHORC,/,  
B 1X,F5.3,7X,F4.1,3X,F4.1,5X,4HROOT,/,
```

OUTPUT167
OUTPUT168
OUTPUT169
OUTPUT170
OUTPUT171
OUTPUT172
OUTPUT173
OUTPUT174
OUTPUT175
OUTPUT176
OUTPUT177
OUTPUT178
OUTPUT179
OUTPUT180
OUTPUT181
OUTPUT182


```

C 13X,F4.1,8X,F4.1,5X,3HTIP,/,
D 13X,F4.1,8X,F4.1,5X,9HWATERLINE,/
WRITE(IPRNT,56) (TDRAG(I),POD(I),STRTD(I),SPRAY(I),
A (CONDS(J,I),J=L,2),I=ISTRT,NUMB)
56 FORMAT(10X,14HDPAG ESTIMATES,/,4X,5HTOTAL,5X,7HNACELLE,5X,5HSTRUT,
A 5X,5HSPRAY,/, (1X,F8.1,5X,F7.1,4X,F7.1,3X,F7.1,5X,2A4))
WTRAT=(WGTS(1,14)+WGTS(2,14))/DISP
WRITE(IPRNT,23) WTRAT
23 FORMAT(/,30H TOTAL SYSTEM WEIGHT RATIO IS ,F6.4)
IF(ISTRT.NE.3)GO TO 300
DO 302 I=ISTRT,NUMB
WRITE(IPRNT,301) VK(I),FRATE(I)
301 FORMAT(/,25H FUEL CONSUMPTION RATE AT,F5.1,10H KNOTS IS ,F9.1,17H
A POUNDS PER HOUR,/)
302 CONTINUE
300 CONTINUE
WRATE=(WGTS(1,14)+WGTS(2,14)-WGTS(2,11))/DISP
WRITE(IPRNT,46) WRATE
46 FORMAT(/,37H SYSTEM WEIGHT RATIO WITHOUT FUEL IS ,F6.4)
WRITE(IPRNT,6)
6 FORMAT(1H1)
RETURN
END

```

OUTPT183
 OUTPT184
 OUTPT185
 OUTPT186
 OUTPT187
 OUTPT188
 OUTPT189
 OUTPT190
 OUTPT191

OUTPT192
 OUTPT193
 OUTPT194
 OUTPT195
 OUTPT196
 OUTPT197
 OUTPT198


```

BLKDT000
COMMON /PINLP/ALPHA
COMMON /WARN/CAV(5,6)
COMMON /PARMS/ VJVO,VIVO,DIDM
COMMON /STRTC/TC,T,C,T1,C1,CFM
COMMON /PUMN/OQ(5),PLS(5),D2S(5),XNS(5),SM(5,5),PLP(5),NSTG,SHI(5,
A5),XIM
COMMON /H2O/TEMP,PV,PHOW,GNU,HA
COMMON /SHIP/DISP,RANGE,BEAM,HS,HE,HCL,XLS,XLPE,XLP
COMMON /CONST/PI,G,RHCD
COMMON /CHARS/WGTS(2,15),CGS(4,15),DELH(5,15),CGSX,CGSZ
COMMON /ELRW/XK(4),RO(4),THATA(4),WIDTH,DEPTH,TYPE(3,4)
COMMON /TOLER/DELTA
COMMON /FLOW/Q(5),AIN,AJET,AREA(11),VJ(5),VI(5)
COMMON /PSUB/GERAT(5),SHP(5,5),RPM(5,5),PEPF(5,12),ETAP(8,5)
COMMON /NACLI/DRAT,D4,AI,AIAUX,ELEXT,ELENT,ELAUX,ELDIF,ELN
COMMON /INDEX/IEVAL,IEQPT,ISTRT,NUMB,IENGN,ITYPE,ICOMP,NPUMP,NGT,
AIGEAR
COMMON /ITABL/L(2)
DATA IGEAR/99/
DATA TEMP/59./
DATA HS,HE,HCL,XLS,XLPE,XLP/5.,10.,2.,20.,2.,0./
DATA PI,G,RHOD/3.14159,32.174,14.92/
ENGINE PERFORMANCE DATA
DATA PERF/222.,2840.,.59,14500.,1350.,2850.,3060.,.55,14500.,
11050.,2300.,3510.,.63,1500.,3200.,2800.,3510.,.63,1000.,3300.,
23320.,4250.,.49,3110.,2800.,4160.,5300.,.47,3110.,2800.,
32220.,2840.,.79,9000.,1010.,12500.,14000.,.575,5500.,7500.,
422200.,22500.,.41,3400.,10500.,19150.,24200.,.52,3600.,14200.,
52175.,26950.,.52,3610.,14200.,27600.,34400.,.48,3600.,14200./
DATA WGTS,CGS,DELH/15*0./
DATA DRAT,DM,AI,AIAUX,ELEXT,ELENT,ELAUX,ELCIF,ELN/9*1./
DATA RO/4*1./,XIM/1./
DATA TC,T,C,T1,C1,CFM/6*1./
DATA THATA,XK/3*90.,30.,3*1.5,2./
DATA SHP/25*1000./,DELTA/.05/
BLKDT001
BLKDT002
BLKDT005
BLKDT006
BLKDT007
BLKDT008
BLKDT009
BLKDT010
BLKDT011
BLKDT012
BLKDT013
BLKDT015
BLKDT016
BLKDT017
BLKDT018
BLKDT019
BLKDT020
BLKDT021
BLKDT022
BLKDT023
BLKDT024
BLKDT025
BLKDT026
BLKDT027
BLKDT028
BLKDT029
BLKDT030
BLKDT031

```

C

DATA VJVD,VIVQ,DI DM/1.8,.7,.7,.7/
DATA Q,VJ,VI,AREA/15*0.,11*1./
DATA IEQPT,IEGN,IEVAL/0.8,0/
DATA L/11,1/
DATA TYPE/12*4H /
DATA NSTG/1/
DATA CAV/30*0./
DATA ALPHA/0./
END

BLKDT032
BLKDT033
BLKDT034
BLKDT035
BLKDT036
BLKDT037

BLKDT038

APPENDIX C
USERS' MANUAL

REQUIRED INPUTS FOR OPTIMIZATION	163
OPTIONAL INPUTS FOR OPTIMIZATION	164
SAMPLE INPUT FOR OPTIMIZATION	165
SAMPLE OPTIMIZATION OUTPUT	167
REQUIRED INPUTS FOR EVALUATION	170
OPTIONAL INPUTS FOR EVALUATION	172
SAMPLE INPUT FOR EVALUATION	173
SAMPLE EVALUATION OUTPUT	176

REQUIRED INPUTS FOR OPTIMIZATION

<u>Symbol</u>	<u>Required Units</u>
VO(1)	feet per second
VO(2)	feet per second
REST(1)	pounds
REST(2)	pounds
STRTD(1)	pounds
STRTD(2)	pounds
SPRAY(1)	Pounds
SPRAY(2)	pounds
POD(1)	pounds
POD(2)	pounds
TRIM(1)	degrees
TRIM(2)	degrees
RANGE	nautical miles
DISP	pounds
BEAM	feet
TYGER	(none)

OPTIONAL INPUTS FOR OPTIMIZATION

<u>Symbol</u>	<u>Required Units</u>	<u>Default Value</u>
XK(1)	(none)	1.5
XK(2)	(none)	1.5
XK(3)	(none)	1.5
XK(4)	(none)	2.0
THATA(1)	degrees	90.
THATA(2)	degrees	90.
THATA(3)	degrees	90.
THATA(4)	degrees	30.
TEMP	degrees Farenheit	59.
HS	feet	5.
HE	feet	10.
HCL	feet	2.
XLS	feet	20.
XLPE	feet	2.
IEQPT	(none)	0
IENGN	(none)	8
IEVAL	(none)	0
NGT	(none)	calculated
NPUMP	(none)	calculated
IGEAR	(none)	calculated
ALPHA	degrees	0.


```

LOGICAL IFUEL, IPUMP, TYGER
REAL JANGL
COMMON /PINLP/ALPHA
COMMON /PARMS/VJVO, VIVO, DIDM
COMMON /WARN/CAV(5,6)
COMMON /NOZL/JANGL
COMMON /CHARS/WSTS(2,15), CGS(4,15), DELH(5,15), CGSX, CGSZ
COMMON /DRAG/TCRAG(5), STRTD(5), POD(5), SPRAY(5), REST(5), VO(5),
1 TRIM(5)
COMMON /NACLL/DRAT, DM, AI, AIAUX, ELEXT, ELEN, ELAUX, ELDIF, ELN, AAX(5)
COMMON /H2O/TEMP, PV, RHCW, GNU, HA
COMMON /SHIP/DISP, PANGE, BEAM, HS, HE, HCL, XLS, XLPE, XLP
COMMON /CCNST/PJ, G, RHOD
COMMON /FLOW/Q(5), AIN, AJET, AREA(11), VJ(5), VI(5)
COMMON /ELBH/XK(4), RD(4), THATA(4), WIDTH, DEPTH, TYPE(3,4)
COMMON /INDEX/IEVAL, IEQPT, ISTR, NUMB, IENGN, ITYPE, ICOMP, NPUMP, NGT,
AIGFAP.
COMMON /ITABL/L(2)
COMMON IFUEL, IPUMP, TYGER

SAMPLE INPUT FOR DESIGN OPTIMIZATION

VO(1)=45.*1.6889
VO(2)=30.*1.6889
BEAM=36.
RANGE=2000.
HE=35.5
HS=12.
XLPE=12.
IENGN=9
XLS=32.
HCL=17.5
TRIM(1)=0.
TRIM(2)=0.
TYGER=.FALSE.
DISP=75).*2240.

```

C
C
C


```
POD(1)=11436.1
STRTD(1)=11653.7
SPRAY(1)=8474.2
POD(2)=POD(1)*((VO(2)/VO(1))**2)
STRTD(2)=STRTD(1)*((VC(2)/VO(1))**2)
SPRAY(2)=SPRAY(1)*((VO(2)/VO(1))**2)
REST(1)=76956.
PEST(2)=129971.6
CALL H20JT
STOP
END
```


CRAFT CHARACTERISTICS

OPERATIONAL

	CRUISE	TAKE-OFF
VELOCITY, KNOTS	45.0	30.0
TOTAL DRAG, LBS	108520.	144000.
ANGLE OF ATTACK, DEGREES	0.0	0.0

CONFIGURATION

AVERAGE BEAM, FEET..... 36.0
 DISPLACEMENT, LONG TONS..... 750.
 ENDORANCE, NM..... 2000.
 GAS TURBINE PLANT..... LM2500

DEPTH OF SUBMERGENCE OF NACELLE.....	12.0	FEET
HEIGHT OF PUMP CENTERLINE ABOVE MEAN WATER.....	35.5	FEET
HEIGHT OF PUMP CENTERLINE ABOVE KEEL.....	17.5	FEET
DISTANCE OF STKUT FROM TRANSUM.....	32.0	FEET
DISTANCE OF PUMP EXIT FROM TRANSUM.....	12.0	FEET

WATER PROPERTIES
 (ASSUMES STANDARD(3.5% SALINITY) SALT WATER)

TEMPERATURE, DEGREES FAHRENHEIT	59.
DENSITY, LBF-SEC**2/FEET**4	1.989
VISCOSITY, *10**5, FEET**2/SEC	1.279
VAPOR PRESSURE, FEET	0.545

ACCELERATION OF GRAVITY, FT/SECS**2 32.174

SPECIFIED NUMBER OF GAS TURBINES IS 2
 SPECIFIED NUMBER OF PUMPS IS 2

*** SYSTEM WEIGHT IS 401.64 TONS ***

NUMBER OF GAS TURBINES IS 2
 NUMBER OF PUMPS IS 2

*** WATERJET PROPULSION SYSTEM OUTPUT DATA ****

INLET AREA, TOTAL, FEET**2 13.41
 STRUT DIFFUSER AREA RATIO 1.15
 JET AREA, TOTAL, FEET**2 4.89

FLOW RATE CFS	JET VELOCITY	JET RATIO	INLET VELOCITY	INLET RATIO	INLET VELOCITY	INLET RATIO	SHIP PER TURBINE	PROPULSIVE COEFFICIENT	CRUISE T/U
738.75	151.05	1.99	55.10	0.72	55.10	0.72	15130.	0.5007	0.2985
804.32	164.46	3.25	59.99	1.18	59.99	1.18	22393.	0.5007	0.2985

1600

RADIUS RATIO	DUCT RADIUS	ANGLE DEGREES	LOCATION	SHAPE
1.50	2.93	90.00	STRUT	RECTANGLE
1.50	1.57	90.00	HULL	RECTANGLE
1.50	2.50	90.00	PUMP	CIRCLE

	STRUCTURE		WATER		CRUISE		EVALUATION	
	WEIGHTS (POUNDS)	WEIGHTS (POUNDS)	WEIGHTS (POUNDS)	WEIGHTS (POUNDS)	DUCT LOSSES (FEET)	DUCT LOSSES (FEET)	DUCT LOSSES (FEET)	DUCT LOSSES (FEET)
NACELLE	6555.4		0.0		5.04		6.13	
STRUT LLBCW	3764.2		0.0		1.32		1.53	
STRUT DIFFUSER	48006.0		89649.5		0.78		0.92	
HULL ELBOW	2669.6		9319.0		1.10		1.28	
APPARTSHIPS LENGTH	4619.1		21578.8		0.08		0.09	
PUMP ELBOW	4553.2		14868.8		0.42		0.49	
FORE AND AFT LENGTH	5694.4		17258.6		0.06		0.07	
PUMP	31386.6		12345.7					
NOZZLE	4497.1		14888.5		4.09		4.92	
REDUCTION GEAR	5329.3		0.0					
FUEL	0.0		56197.3					
LM2500	21000.0		0.0					
JET LIFT	0.0		-14479.0					
TOTALS	136054.7		763627.4		12.90		15.44	

AXIAL PUMP WITH INDOUER IMPELLER AND 0 ADDITIONAL STAGE

HEAD	NPSH	THOMA	RPM	EFFICIENCY	
313.2	78.0	0.249	620.2	0.908	CRUISE
431.4	26.4	0.301	705.6	0.520	T/O

SPECIFIC SPEED,CFS..... 149.5
 SUCTION SPECIFIC SPLD,CFS..... 1214.6
 FLOW COEFFICIENT..... 0.110
 HEAD COEFFICIENT..... 0.370
 INLET TIP DIAMETER,FEET..... 5.24
 EXIT TIP DIAMETER,FEET..... 5.24
 PUMP LENGTH,FEET..... 9.39
 GEAR RATIO..... 4.82

NACELLE DATA

DIAMETER FATIC,DI/DM..... 0.562
 MAXIMUM DIAMETER,DM,FEET..... 5.19
 INLET AREA PER NACELLE,FT**2..... 6.70
 AUXILIARY INLET AREA PER NACELLE,FT**2 0.0
 FOREBODY LENGTH,FEET..... 5.61
 LIP LENGTH,FEET..... 0.62
 DIFFUSER LENGTH,FEET..... 9.51
 NACELLE LENGTH,FEET..... 28.57

STRUT CONFIGURATION

T/C	THICKNESS	CHORD	ROOT	SPRAY
C.120	2.2	16.1	TIP	
	2.0	10.9	WATERLINE	
	2.1	17.2		

DRAG ESTIMATES

TOTAL	NACELLE	STRUT	SPRAY	
109639.3	10755.8	10924.8	8474.1	CRUISE
145115.2	5082.7	5179.4	3766.3	T/O

TOTAL SYSTEM WEIGHT RATIO IS 0.5355

SYSTEM WEIGHT RATIO WITHOUT FULL IS 0.1795

REQUIRED INPUTS FOR EVALUATION

<u>Symbol</u>	<u>Required Units</u>
STRTD(I)	pounds
POD(I)	pounds
SPRAY(I)	pounds
REST(I)	pounds
VO(I)	feet per second
TRIM(I)	degrees
AIN	feet squared
AJET	feet squared
AREA(1)	feet squared
AREA(3)	feet squared
AREA(8)	feet squared
RANGE	nautical miles
DISP	pounds
BEAM	feet
TYGER	(none)
DRAT	(none)
DM	feet
AIAUX	feet squared
ELEXT	feet
ELENT	feet
AI	feet squared
ELAUX	feet
ELDIF	feet
ELN	feet

<u>Symbol</u>	<u>Required Units</u>
TC	(none)
T	feet
C	feet
T1	feet
C1	feet
IEVAL	(none)
IEQPT	(none)
NPUMP	(none)
NGT	(none)
NSTG	(none)
RPM(2,NSTG)	revolutions per minute
QQ(2)	cubic feet per second
D1S(NSTG)	feet
D2S(NSTG)	feet
PLP(NSTG)	feet
HPP(2)	feet
JANGL	radians

OPTIONAL INPUTS FOR EVALUATION

<u>Symbol</u>	<u>Required Units</u>	<u>Default Value</u>
XK(1)	(none)	1.5
XK(2)	(none)	1.5
XK(3)	(none)	1.5
XK(4)	(none)	2.0
THATA(1)	degrees	90.
THATA(2)	degrees	90.
THATA(3)	degrees	90.
THATA(4)	degrees	30.
TEMP	degrees Fahrenheit	59.
HS	feet	5.
HE	feet	10.
HCL	feet	2.
XLS	feet	20.
XLPE	feet	2.
IENGN	(none)	8
IGEAR	(none)	calculated
ALPHA	degrees	0.
WGTS(2,11)	pounds	0.


```

LOGICAL IFUEL, IPUMP, TYGER
REAL JANGL
COMMON /PINLP/ALPHA
COMMON /PARMS/VJVO, VIVO, DIDM
COMMON /WARN/CAV(5,6)
COMMON /NOZL/JANGL
COMMON /CHARS/WGTS(2,15), CGS(4,15), DELH(5,15), CGSX, CGSZ
COMMON /DRAG/TDRAG(5), STRTD(5), POD(5), SPRAY(5), REST(5), VO(5),
1 TRIM(5)
COMMON /NACLL/CRAT, DM, AI, AIAUX, ELEFT, ELAUX, ELDIFF, ELN, AAX(5)
COMMON /H2O/TEMP, PV, RHO, GNU, HA
COMMON /SHIP/DISP, RANGE, BEAM, HS, HE, HCL, XLS, XLPE, XLP
COMMON /CONST/PI, G, RHOD
COMMON /FLOW/C(5), AIN, AJET, AREA(11), VJ(5), VI(5)
COMMON /ELBN/XK(+), RO(4), THATA(4), WICHT, DEPTH, TYPE(3,4)
COMMON /INDEX/IEVAL, IEQPT, ISIRT, NUMB, IENGN, ITYPE, ICCMP, NPUMP, NGT,
AIGEAR
COMMON /ITABL/L(2)
COMMON IFUEL, IPUMP, TYGER
DIMENSION SDRG(5), TDISP(5)

SAMPLE INPUT FOR EVALUATION

VO(1)=45.*1.6889
VO(3)=40.*1.6889
VO(4)=45.*1.6889
VO(5)=50.*1.6889
BEAM=36.
RANGE=2000.
HE=35.5
HS=12.
XLPE=12.
IENGN=9
XLS=32.
HCL=17.5
TRIM(1)=0.

```

C
C
C


```

TRIM(2)=0.
TRIM(3)=0.
TRIM(4)=0.
TRIM(5)=0.
TYGER=.TRUE.
IGEAP=1
IFVAL=-3
DISP=750.*2240.
IFOPT=3
NGT=2
NPUMP=2
SDRG(1)=76956.
REST(1)=SDRG(1)
POD(1)=11436.1
STRTD(1)=11653.7
SPRAY(1)=8474.2
SDRG(2)=129971.6
DO 901 I=3,5
REST(I)=REST(1)*(VO(I)/VO(1))**2
STRTD(I)=STRTD(1)*(VO(I)/VO(1))**2
SPRAY(I)=SPRAY(1)*(VO(I)/VO(1))**2
POD(I)=POD(1)*(VO(I)/VO(1))**2
901 CONTINUE
NSTG=1
RPM(2,1)=782.9
QQ(2)=368.89
HPP(2)=467.7
DIS(1)=4.92
D2S(1)=4.92
PLP(1)=8.81
AIN=14.79
AJET=4.31
AREA(1)=22.4676
AREA(3)=34.6
APEA(8)=34.5837
DPAT=0.731

```

901

DM=4.20
AIAUX=0.0
AI=7.395
ELAUX=0.0
ELDIF=3.84
ELN=23.08
ELENT=0.19
ELEXT=1.67
TC=0.12
T=2.0
C=17.0
T1=1.6
C1=13.7
JANGL=.06021
CALL H20JT
STOP
END

OPERATIONAL

VELOCITY, KNOTS 40.0
TOTAL DRAG, LBS. 80272.
ANGLE OF ATTACK, DEGREES 0.0

VELOCITY, KNOTS 45.0
TOTAL DRAG, LBS. 101276.
ANGLE OF ATTACK, DEGREES 0.0

VELOCITY, KNOTS 50.0
TOTAL DRAG, LBS. 124689.
ANGLE OF ATTACK, DEGREES 0.0

CRAFT CHARACTERISTICS

CONFIGURATION

AVERAGE BEAM, FEET..... 36.0
DISPLACEMENT, LONG TONS.... 750.
ENDURANCE, NM..... 2000.
GAS TURBINE PLANT..... LM2500

DEPTH OF SURGERGE OF NACELLE..... 12.0 FEET
HEIGHT OF PUMP CENTERLINE ABOVE MEAN WATER..... 35.5 FEET
HEIGHT OF PUMP CENTERLINE ABOVE KEEL..... 17.5 FEET
DISTANCE OF STRUT FROM TRANSOM..... 32.0 FEET
DISTANCE OF PUMP EXIT FROM TRANSOM..... 12.0 FEET

WATER PROPERTIES
(ASSUMES STANDARD(3.5% SALINITY) SALT WATER)

TEMPERATURE, DEGREES FAHRENHEIT 59.
DENSITY, LBF-SEC**2/FFFT**4 1.989
VISCOSITY, *10**5, FFFT**2/SEC 1.279
VAPOR PRESSURE, FEET 0.545

ACCELERATION OF GRAVITY, FT/SECS**2 32.174

PUMP DATA

AXIAL PUMP WITH INDUCER IMPELLER AND 0 ADDITIONAL STAGE

HEAD	NPSH	THOMA	RPM	EFFICIENCY	EVAL 1
265.9	59.7	0.225	603.4	0.951	
326.2	76.6	0.235	672.6	0.900	EVAL 2
393.3	95.5	0.243	742.3	0.919	EVAL 3

PUMP PARAMETERS

SPECIFIC SPEED, CFS	149.5
INLET TIP DIAMETER, FEET	4.92
EXIT TIP DIAMETER, FEET	4.92
PUMP LENGTH, FEET	8.81

PUMP OPERATING PARAMETERS AT EVALUATED CONDITIONS

SUCTION SPECIFIC SPEED, CFS	481.3	472.0	465.1
FLOW COEFFICIENT	0.114	0.115	0.115
HEAD COEFFICIENT	0.354	0.349	0.346

MACELLE DATA

DIAMETER RATIO, DI/DO	0.731
MAXIMUM DIAMETER, CM, FEET	4.20
INLET AREA PER MACELLE, FT*2	7.40
AUXILIARY INLET AREA PER MACELLE, FT*2	0.0
FOREBODY LENGTH, FEET	1.67
LIP LENGTH, FEET	0.19
DIFFUSER LENGTH, FEET	3.84
MACELLE LENGTH, FEET	23.08

STRUT CONFIGURATION

T/C	THICKNESS	CHORD	ROOT
0.120	2.0	17.0	TIP
	1.6	12.7	WATERLINE
	1.7	14.5	

DRAG ESTIMATES

TOTAL	MACELLE	STRUT	SPRAY	EVAL 1
80271.9	6511.5	7423.9	5531.9	
101275.5	9094.9	9223.9	7001.3	EVAL 2
124689.4	9836.2	11202.4	8643.6	EVAL 3

TOTAL SYSTEM WEIGHT RATIO IS 0.5104

FUEL CONSUMPTION RATE AT 40.0 KNOTS IS 10690.9 POUNDS PER HOUR,

FUEL CONSUMPTION RATE AT 45.0 KNOTS IS 13027.0 POUNDS PER HOUR,

FUEL CONSUMPTION RATE AT 50.0 KNOTS IS 15961.7 POUNDS PER HOUR,

3 OCT 74

22321

Thesis
K876

Kruse

Analysis of a method
for optimum design of
waterjet propulsion sys-
tems.

16 OCT 73
3 OCT 74

145654
DISPLAY
22321

Thesis
K876

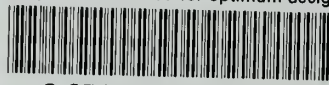
Kruse

Analysis of a method
for optimum design of
waterjet propulsion sys-
tems.

145654

thesK876

Analysis of a method for optimum design



3 2768 002 11552 9

DUDLEY KNOX LIBRARY